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Volume VI. Wind Tunnel Test of a Powered Tilt-Rotor Dynamic Model on a Simulated Free Flight Suspension System

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FOREWORD

This report was prepared by the Boeing Company, Vertol Division, Philadelphia, Pennsylvania, for the Air Force Flight Dynamics Laboratory, Wright-Patterson Air Force Base, Ohio, under Phase II of Contract F33615-69-C-1577. The contract objective is to develop design criteria and aerodynamic prediction techniques for the folding tilt rotor concept through a program of model testing and analysis.

The contract was administered by the Air Force Flight Dynamics Laboratory with Mr. Daniel E. Fraga (FV) as Project Engineer.

This report covers the period from January to July 1971.

The reports published under this contract for Design Studies and Model Tests of the Stowed Tilt Rotor Concept are:

Volume I	Parametric Design Studies
Volume II	Component Design Studies
Volume III	Performance Data for Parametric Study
Volume IV	Wind Tunnel Test of the Conversion Process of a Folding Tilt Rotor Aircraft Using a Semi-Span Unpowered Model
Volume V	Wind Tunnel Test of a Powered Tilt Rotor Performance Model
Volume VI	Wind Tunnel Test of a Powered Tilt Rotor Dynamic Model on a Simulated Free Flight Suspension System
Volume VII	Wind Tunnel Test of the Dynamics and Aerodynamics of Rotor Spinup, Stopping and Folding on a Semi-Span Folding Tilt Rotor Model
Volume VIII	Summary of Structural Design Criteria and Aerodynamic Prediction Techniques
Volume IX	Value Engineering Report

This report has been reviewed and is approved.

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ABSTRACT

This report presents the results of a wind tunnel test on a powered dynamic model of the Boeing M-160 tilt rotor aircraft with 5.5 foot diameter rotors. The model was tested in the Boeing V/STOL 20 X 20 foot wind tunnel during January-February 1971 and was supported to simulate free flight conditions with mount frequencies much lower than the dynamic aircraft frequencies. Blade loads, wing loads, flying qualities and skittishness in ground effect data were obtained.

SUMMARY

The test was conducted to obtain data in several different technical categories and the summary of the results is given below.

Rotating blade frequencies for the first three modes have been measured and correlate very well with predictions. (Figure 5.2, Section 5).

Blade load data were obtained in hover, transition, and cruise attitudes. The hover results show that the sensitivity of the blade loads to cyclic pitch are not affected by ground effect, but ground effect does increase the minimum blade load at zero cyclic. Increased collective pitch increased the blade load sensitivity to cyclic pitch. (Figures 6-1 to 6-58, Sections 6.1 through 6.4).

Blade alternating loads were essentially unaffected by differential collective between the two rotors but waveforms changed considerably (Section 6.1, Figure 6-12).

Low amplitude stall flutter inception occurred at $\theta_{.75} = 11°$, but torsional blade loads were low up to the highest blade angle tested of 14° (Section 6.1, Figures 6-19 and 6-20).

Results show that in transition the minimum blade alternating chord bending load increases with increasing dynamic pressure. Flap bending loads were not significantly affected by dynamic pressure. Increased collective pitch decreased the minimum alternating chord bending load. Fuselage pitch and yaw caused large changes in alternating chord bending load with little effect on flap bending. (Section 6.2, Figures 6-22 through 6-44).

Alternating blade loads in cruise attitude were significantly lower than those encountered in hover and transition. However, the alternating chord bending sensitivity to cyclic was greater than in hover. (Section 6.3, Figures 6-45 through 6-58).

At a dynamic pressure of 6.65 psf corresponding to full-scale cruise of 140 knots, 5 degrees of yaw produced alternating chord bending loads equivalent to that due to 0.85 degrees of cyclic in hover. Alternating flap bending loads were insensitive to yaw. (Section 6.3.1 and 6.3.2).

Whirl flutter, static divergence, ground and air resonance were not encountered over the range of conditions tested. However, near zero damping occurred in the blade chordwise bending motion at tilt angles tested (0°, 40°, 60°, 90°) at low dynamic pressures and near zero thrust conditions. Further analysis of these data are being conducted. (Section 7.3.2, Figures 7-18 through 7-23).

The model was very stable in its rigid body modes. The rotors increased rigid body stability in hover and cruise attitude (Section 8, Figures 8-1 through 8-7).

Skittishness in ground effect was found to exist, but motion is non-divergent, low amplitude ($\pm 2°$) and very low frequency. Results indicate that skittishness on the full scale aircraft can be adequately stabilized by stability augmentation system required for normal flight conditions (Section 9.0, Figures 9-1 and 9-2).

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LIST OF SYMBOLS AND SIGN CONVENTION

c	-	Blade chord	- ft or inch
$c_{.75}$	-	Blade chord at .75R	- ft
$C_{M_{\theta_2}}$	-	Pitching moment coefficient due to cyclic pitch	-
C_T	-	Thrust coefficient = $\frac{T}{\pi \rho V_T^2 R^2}$	-
D	-	Rotor Diameter	- ft or inch
f	-	Frequency	- cps
h	-	Height of rotor disc plane above ground level	- ft
IGE	-	In ground effect	-
i_N	-	Nacelle tilt angle	deg
M	-	Pitching moment	- lb in.
N	-	Number of blades	-
n	-	Rotor speed	- RPS
OGE	-	Out of ground effect	-
q	-	Freestream dynamic pressure	- lb/ft^2
q_m	-	Freestream dynamic pressure, model scale	- lb/ft^2
q*	-	$q (\cos^2 i_N)$	- lb/ft^2
R	-	Blade radius	- ft or inch
r	-	Radial distance from rotor center to blade station	- ft or inch

LIST OF SYMBOLS AND SIGN CONVENTION

T	-	Thrust	- lb
V	-	Velocity	- ft/sec or knots
V_m	-	Freestream velocity, model scale	- ft/sec
V_T	-	Rotor tip velocity	- ft/sec
α	-	Fuselage pitch angle (positive-nose up)	- deg
γ	-	Fuselage roll angle (positive-right wing down)	- deg
δ	-	Horizontal stabilizer from angle (positive - leading edge up)	- deg
$\theta_{.75}$	-	Blade collective pitch angle (positive - blade leading edge up)	- deg
θ_2	-	Blade cyclic pitch angle (positive-blade leading edge rotated down on the advancing blade) (See Figure 3-3)	- deg
$\bar{\zeta}$	-	Viscous damping coefficient = c/c_c	-
ρ	-	Air density	- slugs/ft^2
σ	-	$\frac{Nc}{\pi R}$	-
ψ	-	Rotor blade azimuth (See Figure 3-3)	- deg
ω	-	Frequency	- cpm or rad/sec
ω_n	-	Blade nth natural coupled frequency	(n = 1,2,---) CPS or CPM
ω_α	-	Blade torsion frequency	- rad/sec
φ	-	Fuselage yaw angle (positive-nose right)	- deg
Ω	-	Rotor speed	- RPM

LIST OF SYMBOLS AND SIGN CONVENTION

LOAD SIGN CONVENTION*

- Blade and wing flapwise bending moment - positive, compression on upper surface

- Blade and wing chordwise bending moment - positive, compression on trailing edge

- Blade and wing torsion - positive, leading edge rotated nose up

*With respect to strain gages

1.0 INTRODUCTION

VTOL aircraft with forward tilting rotors mounted on nacelles at the wing tips experience large aerodynamic changes as the rotors are tilted from the hover attitude through the transition regime to cruise flight. Freedom of the aircraft to move and to elastically deform under these changing aerodynamic conditions can have an effect on the rotor blade loads, flying qualities, and aeroelastic stability of the aircraft.

These aspects must be examined in sufficient detail so that design criteria may be developed which account for aeroelastic effects. In addition, existing analytical methods must be verified so that full scale aircraft designs can proceed with technical confidence.

One step toward the achievement of the above objectives is to perform wind tunnel tests on a full span dynamically scaled model on a mount that permits some freedom of motion. Boeing-Vertol Wind Tunnel Test No. 047 of the VR054D model mounted on a pole support conducted in January 1970, demonstrated the feasibility of this type model testing, the results of which are reported in Reference 1.

This document contains the results of wind tunnel tests conducted during January and February 1971 by Boeing-Vertol on the same model, with certain refinements, in support of the Phase II contract for the Design Studies and Model Tests of the Stowed Tilt Rotor Concept.

2.0 OBJECTIVES

The test program was performed to obtain data under conditions ranging from hover through tilt transition and low speed cruise. The general objectives were to:

a) Provide blade and wing loads data, both steady and dynamic, throughout the transition flight envelope.

b) Explore the flutter and divergence boundaries of the rotor and wing including the whirl mode.

c) Obtain data which can be used to calculate the effects of gust penetration and maneuvers.

d) Provide aerodynamic data which include aeroelastic effects.

The specific program objectives are listed below:

Blade Dynamics

1) Determine the non-rotating and rotating blade frequencies and damping values. Check against technical predictions.

   - Data have been obtained from baffle tests and are reported in Section 5.0 with analysis correlation.

Simulated Free Flight Suspension System

2) Determine the effect of the support system (vertical guide, umbilical, and restraining cables) on the model behavior.

   - Prior to the power and wind on runs with the model installed in the wind tunnel test section, tests were performed to determine the effect of the umbilical and snubber cables on the rigid body modes. Corresponding data are presented in Section 4.0. These effects are small.

Hover

3) Determine blade loads as a function collective pitch, cyclic pitch and ground proximity.

   - Data are presented in Section 6.1.

4) Determine blade loads, collective pitch and C_T/σ at stall flutter.

   - Data are presented in Section 6.1.

5) Determine the tendency for the model to have air resonance.

   - No air resonance was encountered, as predicted. See Section 7.

6) Measure the response of the model to rigid body disturbances.

   - A limited amount of data indicated that, in hover, the rotor substantially increases the rigid body stability (damping) over the unpowered condition. See Section 8.0.

7) Determine the effect of IGE on rotor control derivatives.

   - Results presented in Section 8.3 indicate ground effect to be negligible.

Transition

8) Determine rotor stall limits as a function of airspeed.

   - No stall limits were encountered during this test.

9) Determine blade loads in the transition region.

   - Details are presented in Section 6.2.

10) Determine if aeroelastic instabilities occur in the transition region.

- No whirl flutter or air resonance was found during this test.

- Low frequency oscillations associated with the blade chordwise bending mode occurred at various conditions near zero thrust. Details are discussed in Section 7.3.2.

11) Determine model response to rigid body disturbances.

- A limited amount of data indicates stability in the rigid body modes. See Section 8.4.

Cruise

12) Measure blade loads in cruise and their change with q collective pitch, cyclic pitch, aircraft pitch and yaw.

- Details are presented in Section 6.3.

13) Determine if aeroelastic instabilities occur in the cruise attitude.

- See answer given to Question (10).

Maneuver and Gust Penetration

14) Measure blade loads responses to rapid aircraft attitude changes.

- Details are presented in Section 6.4.

3.0 TEST DESCRIPTION

Run numbers are designated as run XX (Y). XX indicates run number and the number noted in the brackets designates the area of the oscillograph tape analyzed.

3.1 MODEL DESCRIPTION

The model shown in Figure 3-1 is a powered, 1/10 scale full span dynamic model aerodynamically representative of the M-160,55' rotor diameter tilt rotor aircraft. Its construction consists of scaled stiffness beams in all members with segmented balsa structure providing the required aerodynamic contours. Additional properties are contained in References 1, 2 and 3 and dimensions are listed in Table 3-1.

The collective and monocyclic pitch of the rotor blades and the incidence angle of the horizontal tail are remotely adjustable. The nacelle tilt angle is manually adjustable. An internal rider assembly permits the model to be mounted on a single vertical cable in the wind tunnel. The model is supported by a soft spring (bungee) to carry its total weight when shut down. This mounting system provides vertical, pitch, roll and yaw freedoms. Additional significant features are:

a. Nacelles

- Manually tiltable from 25° below horizontal reference plane to 15° aft of vertical plane by using fixed pre-set links.
- Tilt axis is located at 40% wing chord.

b. Rotor

- 5.5 foot diameter.
- Distance between rotor centers is 6.78 feet.
- Power is provided by an air turbine motor located in the fuselage which transmits power through a chain drive to a shaft interconnecting the rotors.
- Hover RPM (Froude scaling) - 825 RPM, V_T = 237.5 fps Full scale values are - 261 RPM, V_T = 750 FPS.

c. Blades

- o Soft in-plane, segmented
- o Dynamically-scaled with monocoat covering across segments (See Appendix B for aerodynamic and aeroelastic properties).
- o Calculated blade natural frequencies per Figure 3-2.
- o Mass center, pitch axis and shear center are located at 25% chord.
- o Collective pitch - remotely controllable between -5° to 40°
- o Monocyclic pitch - remotely controllable between $+15^{\circ}$ to -15° with input azimuth angle at 108° (See Figure 3-3).

d. Model Mass and Stiffness Properties

- o See Appendix A and B

e. Miscellaneous

- o The fuselage is flexible but not directly scaled.
- o The tail surfaces were stiffened from the original design with a layer of 1/32" balsa. Also, 45 gms was added to each tip of the horizontal tail (Run 29) to eliminate the tail shake.
- o The horizontal tail surface is remotely adjustable for pitch trim.
- o The model was equipped with pitch and roll axis springs to simulate a simple feedback control system.

3.2 DYNAMIC SCALE RELATIONSHIPS

The model was designed to maintain a Froude No. ratio (model to full scale) = 1.0 so that resulting loads contain the proper gravitational influences. Figure 3-4 shows the relationship between the model dynamic pressure and full scale velocity. The following table lists the scale factors for model to full scale:

Parameter	Scale Factor
Length	$L = 1/10$
Mass	L^3
Time	$L^{1/2}$
Frequency	$L^{-1/2}$
Froude No.	1.0
Force	L^3
Moment	L^4
Pressure	L
Stiffness (EI & GJ)	L^5
Density (Sea Level)	1.0

3.3 MODEL INSTRUMENTATION

A complete list of the measurements made with their respective locations is presented in Table 3-2. Yaw position was measured with a wind vane mounted in a nose boom (Figure 3-1). Pitch and roll displacements were measured utilizing position potentiometers attached between the model and the vertical cable. Two oscillographs were used to measure right blade and wing loads. All measurements were recorded on the computer system. One blade on each rotor was instrumented. See Figure 3-3. The full scale velocity to model dynamic pressure relationships and the nacelle tilt angle reached during the tests are presented in Figure 3.4.

Since the blade strain gages were bonded to the blade spar, resulting blade flapwise and chordwise motions construed from bending moments will always be with respect to the blade and not with respect to the rotor disc plane. For instance, at cruise collective angles ($0.75 \approx 30^{o}$), the chordwise bending moment would result primarily from out-of-disc plane deformation, and flapwise bending moment would result primarily from in-disc plane motion. This is illustrated in Figure 3-5.

3.4 AIR JET SHAKER

Solenoid controlled air jet shakers were mounted on each nacelle 4 inches from the tilt axis (See Figure 3-6) with the air hoses running from the umbilical to the shaker through the wing. These were developed specifically for this model test to provide a source of excitation which would not be affected by model rigid body motion or offer restraints to rigid body freedom. Calibrations based on frequency and air pressure were conducted before the test. The results are presented in Figure 3-7.

3.5 MODEL WIND TUNNEL INSTALLATION

The model was installed as shown in Figure 3-1 in the center of the test section. The vertical guide was a 1/4 inch diameter multiple strand cable anchored at the test section ceiling and weighted with 400 lbs. underneath the test section floor to provide the required fore-aft and lateral mounted frequency (1.0 cps). Snubber cables were installed to serve the dual purpose of restraining the model for safety purposes, and to displace the model during test as required. The umbilical was installed as shown in Figure 3-1 to minimize its effect on the total system mass stiffness and damping. For the skittishness test a platform extending across the test section floor at an elevation of 4 feet was installed. The model was lowered to the desired "ground" proximity for these tests. Figure 3-8 contains the design flight conditions and the general range of those at which the model was tested.

TABLE 3-1

MODEL DIMENSIONS

Rotor	
Number of blades	3
Radius	2.750 ft
Solidity	.0857
Effective Disc Area	23.750 ft^2
Blade Area	2.034 ft^2
Airfoil	See Fig. B-7
Wing	
Airfoil	NACA $63_4$21 (Modified)
Span (Nacelle C_L to Nacelle C_L)	6.78 ft
Chord (Constant)	.858 ft
Thickness Ratio, constant root to tip	.21
Area	5.85 ft^2
Aspect Ratio	7.93
Nacelle Pitch Axis	40% Chord
Wing Angle of Attack with Respect to Fuselage Waterline	2.5 Deg
Horizontal Tail	
Airfoil	NACA 0015
Root Chord	.708 ft
Tip Chord	.416 ft
Span	3.00 ft
Area	1.68 ft^2
Aspect Ratio	5.34
Vertical Tail	
Airfoil	NACA 0015
Root Chord	1.45 ft
Tip Chord	.94 ft
Span	1.04 ft
Aspect Ratio	0.87

TABLE 3-2

MEASUREMENT LIST - LOADS

MEASUREMENT AND LOCATION		RECORDER		
		CEC #1	CEC #2	COMPUTER
ROTOR BLADE LOADS				
FLAPWISE AND CHORDWISE BEND MOMENT	.10R	R		L,R
	.25R			L,R
TORSION	.15R	R		L,R
WING LOADS				
FLAPWISE AND CHORDWISE BEND MOMENT	B.L. 6.62"		R	L,R
DIFFERENTIAL TORSION	B.L. 8.50"		R	L,R

NOTE: ALL DATA CHANNELS HAD FREQUENCY RESPONSE GREATER THAN 100 HERTZ FOR LOADS
ALL DATA ON ON-LINE PRINT-OUT GREATER THAN 50 HERTZ FOR ACCELERATION
L = LEFT, R = RIGHT

MEASUREMENT LIST - ACCELERATION

MEASUREMENT AND LOCATION		RECORDER		
		CEC #1	CEC #2	COMPUTER
NACELLES - NORMAL AND AXIAL	5.5" FWD OF PIVOT			L,R
WING - VERTICAL - WING TIP				L,R
FUSELAGE NOSE AND CG - VERTICAL AND LATERAL				X
FUSELAGE CG - LONGITUDINAL				X

TABLE 3-2 (CONTD.)

MEASUREMENT LIST - MISCELLANEOUS

MEASUREMENT	RECORDER			FREQ. RESPONSE
	CEC #1	CEC #2	COMPUTER	
COLLECTIVE AND CYCLIC 4 — LEFT ROTOR			L,R	O
COLLECTIVE AND CYCLIC 4 — RIGHT ROTOR	C,R	L,R	L,R	O
FUSELAGE POSITION	PITCH, YAW	PITCH ROLL YAW	PITCH ROLL YAW	5
HORIZONTAL STABILIZER ANGLE			X	O
ROTOR SPEED	X	X	X	
1/REV LOCATOR	X	X	X	20
VERTICAL SUPPORT LOAD	X	X	X	5
SHAKER FREQUENCY	X	X	X	20
DYNAMIC PRESSURE	X		X	O

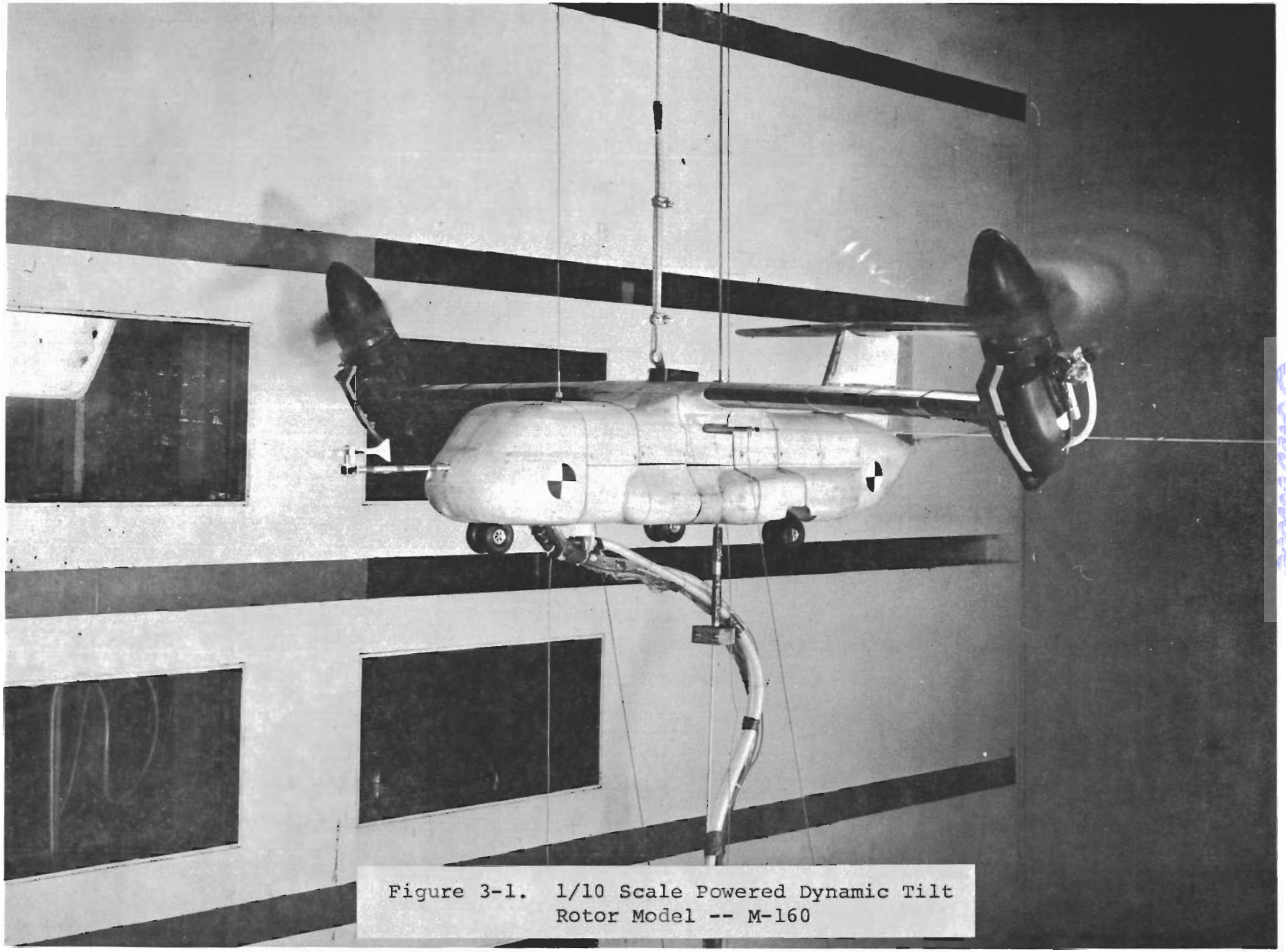


Figure 3-1. 1/10 Scale Powered Dynamic Tilt Rotor Model -- M-160

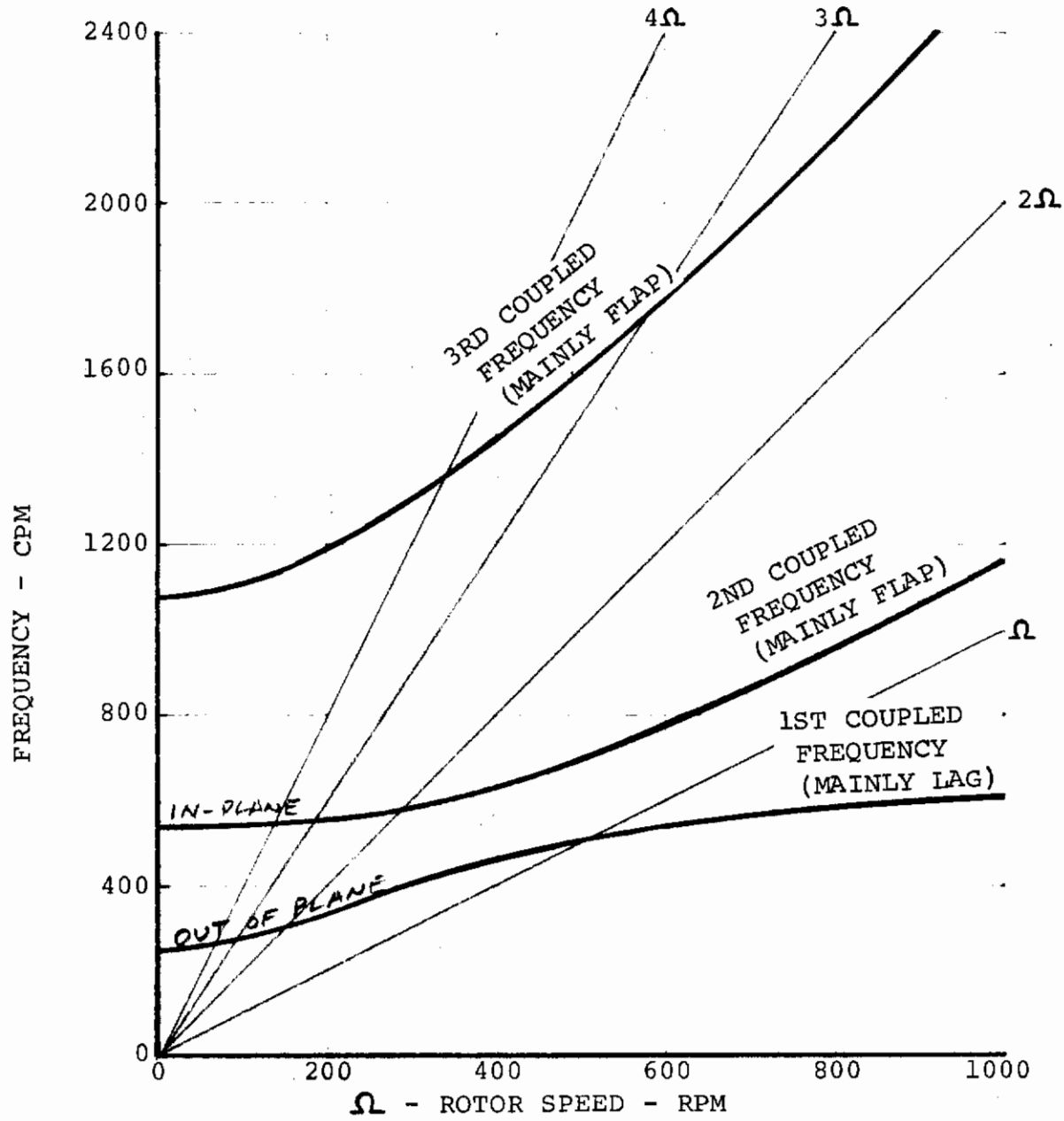


FIGURE 3-2. CALCULATED BLADE FREQUENCY SPECTRUM
$\theta_{.75}=3°$

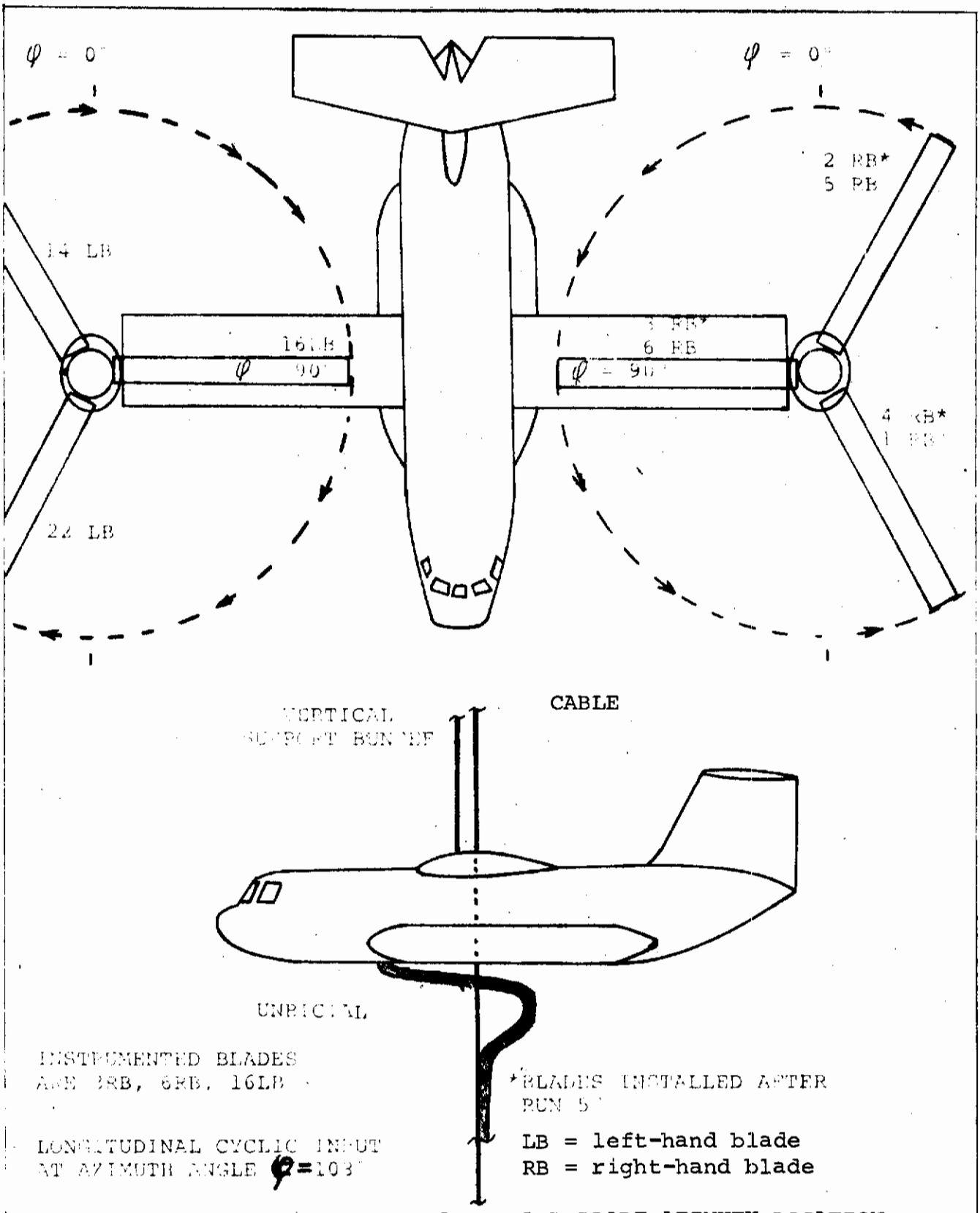


FIGURE 3-3 MODEL INSTALLATION AND BLADE AZIMUTH LOCATION AT TIME OF 1/REV LOCATOR

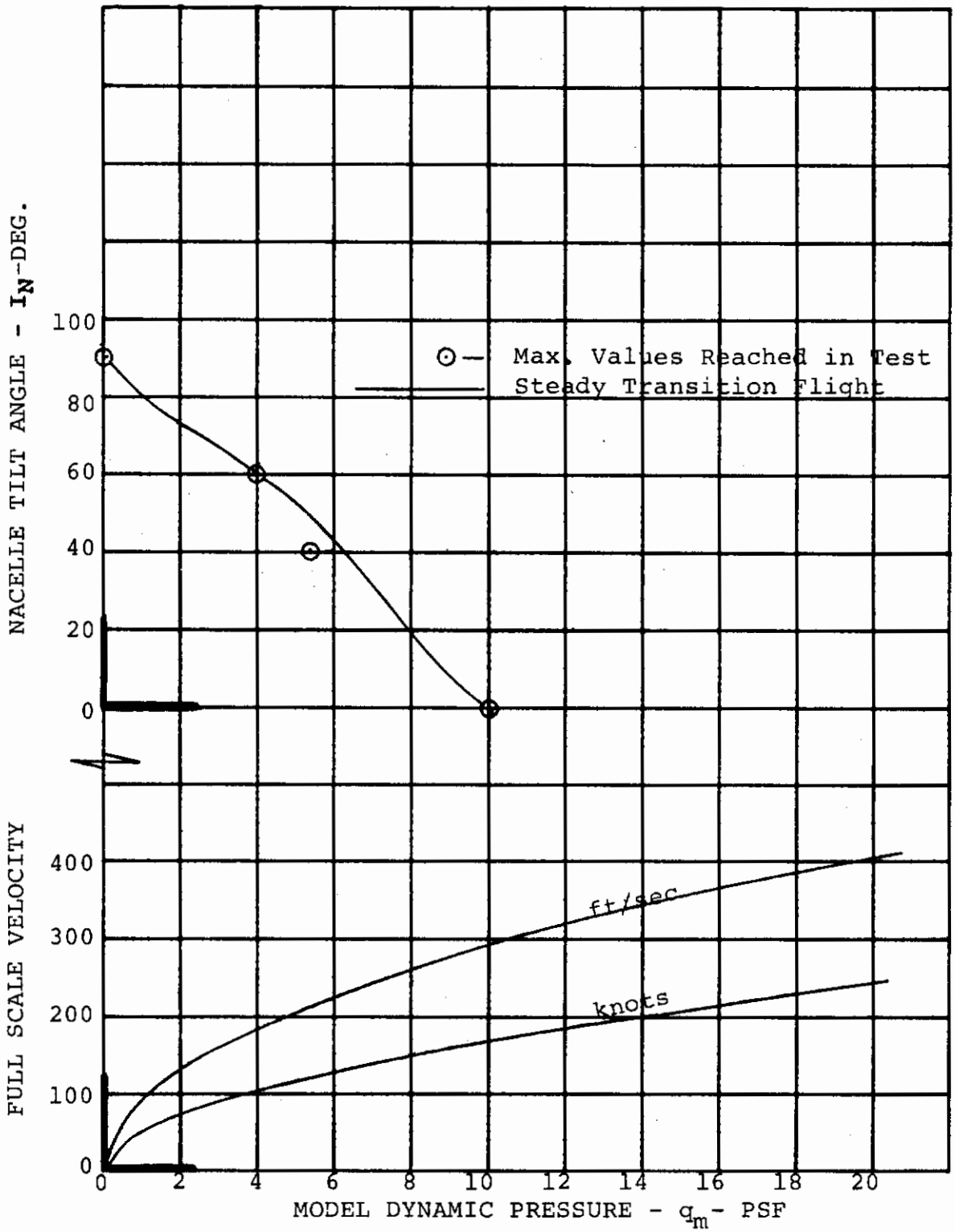


FIGURE 3-4. - FULL SCALE VELOCITY - MODEL DYNAMIC PRESSURE RELATIONSHIP

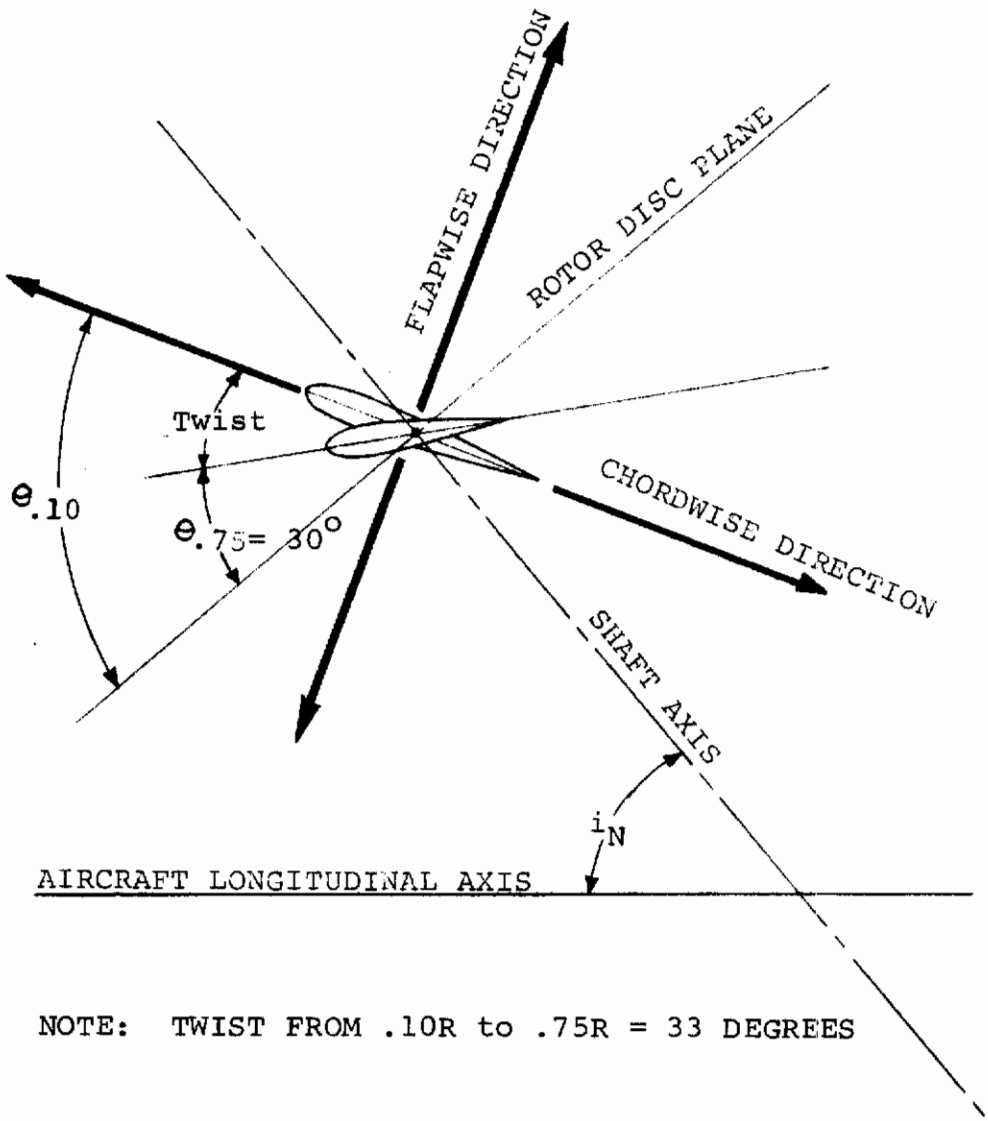


FIGURE 3-5. ILLUSTRATION OF BLADE FLAPWISE AND CHORDWISE MOTION WITH RESPECT TO DISC PLANE BASED ON BLADE BENDING MOMENT STRAIN GAGES AT 0.10R

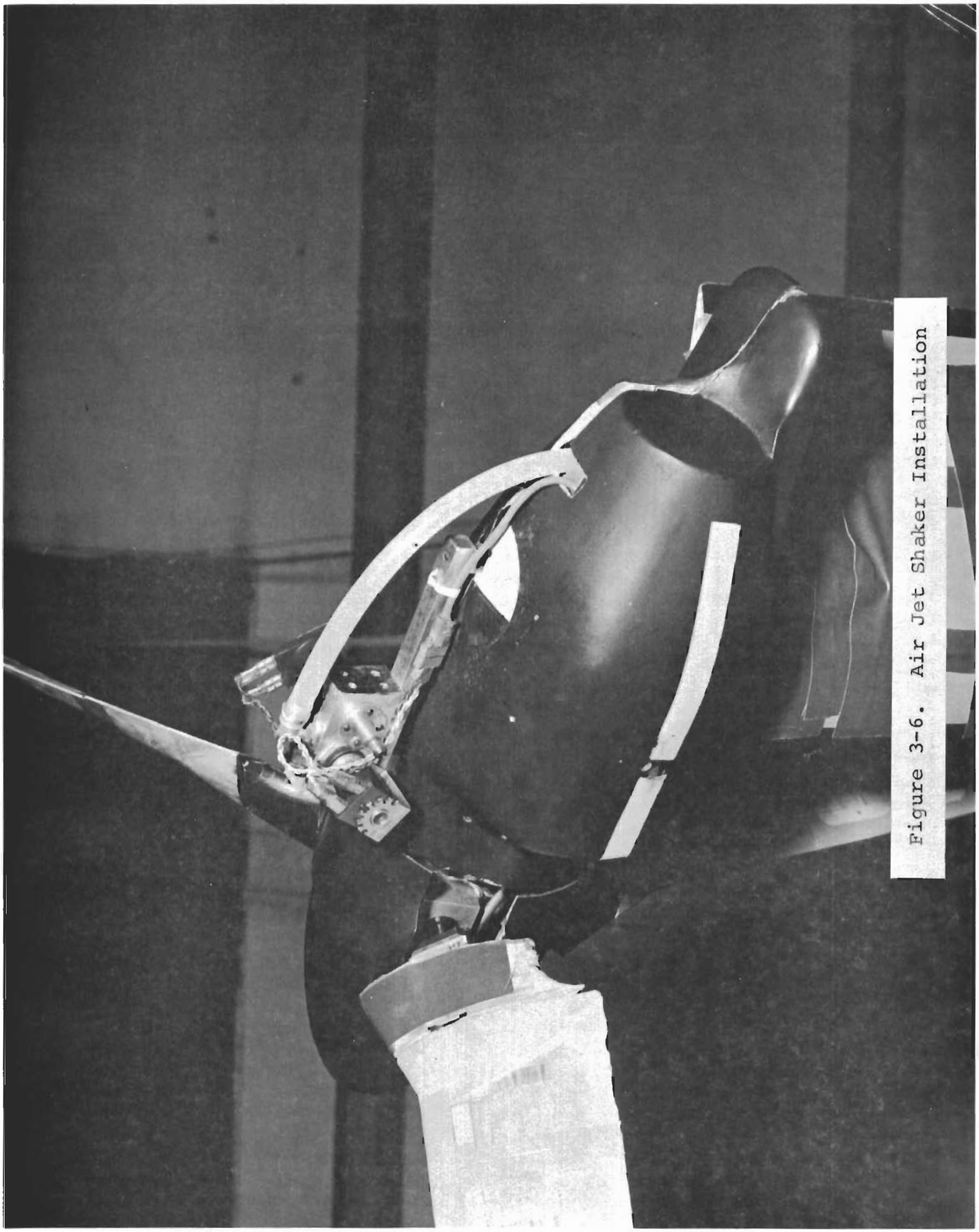


Figure 3-6. Air Jet Shaker Installation

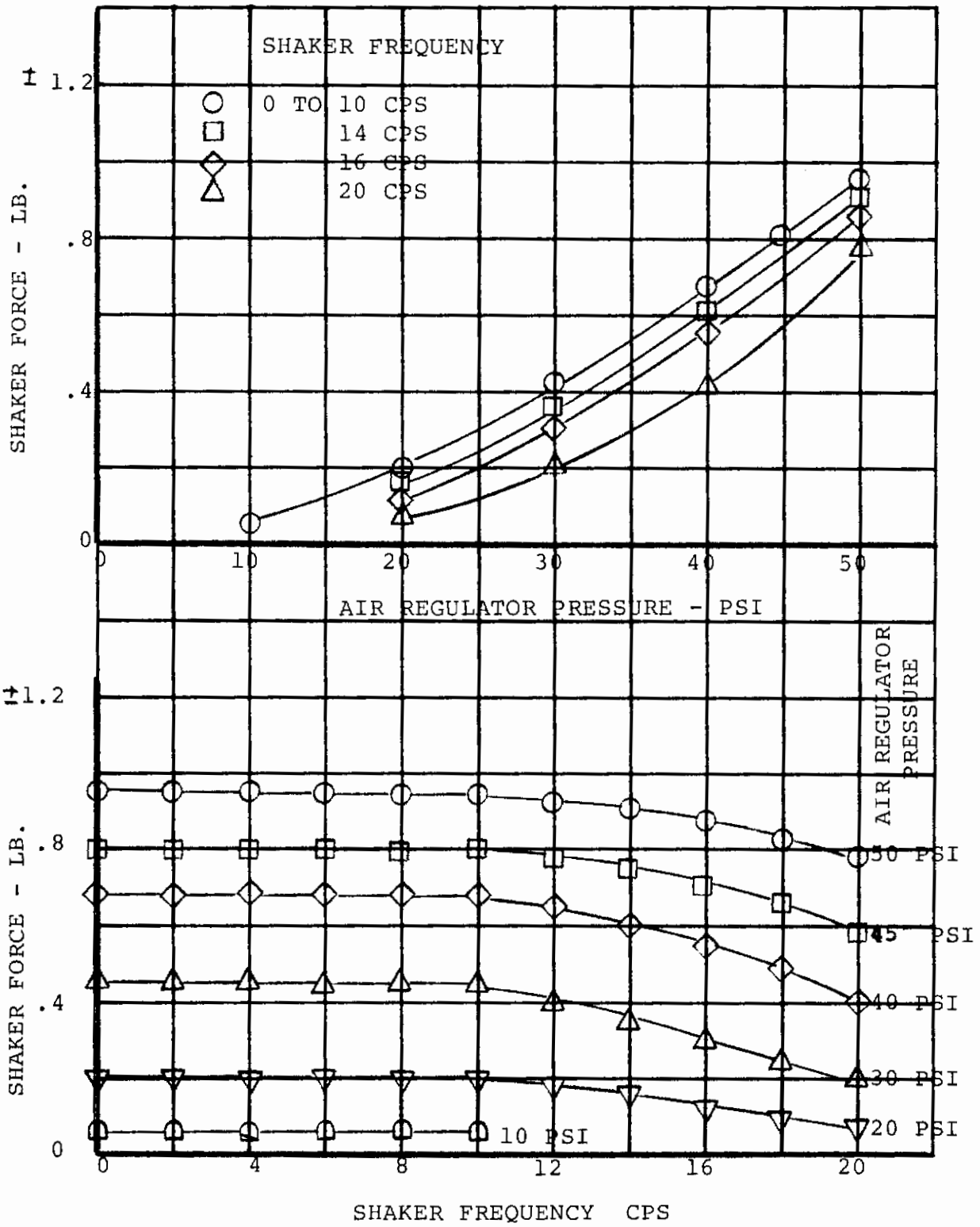


FIGURE 3-7. CALIBRATION OF AIR POWERED SHAKERS

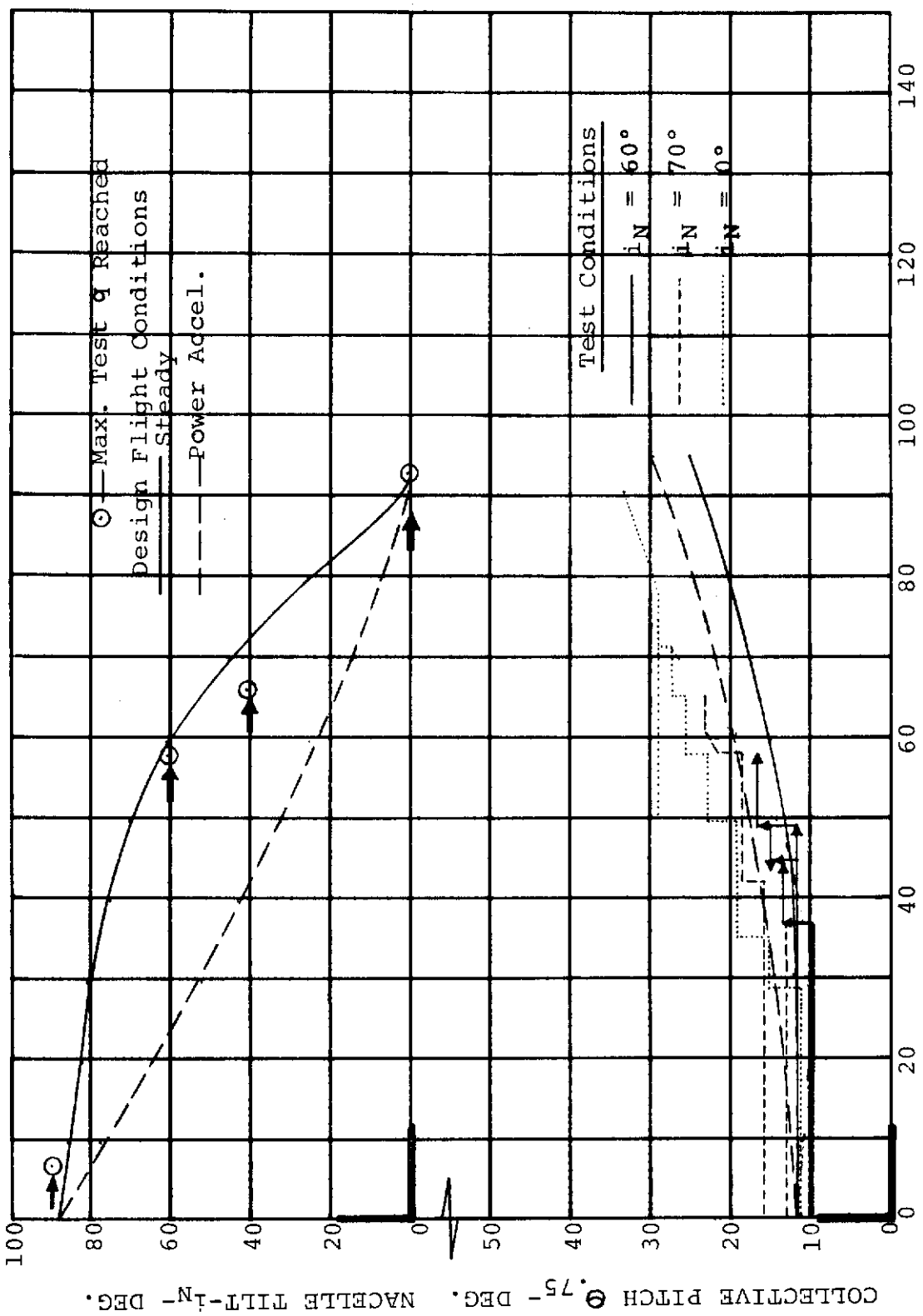


FIGURE 3-8. - TEST CONDITION RANGE

4.0 MODEL UNPOWERED FREQUENCY AND DAMPING DATA

The model was tweak tested by manually bending or pushing the model to an initial deflection to approximately each mode shape, and suddenly releasing the model to measure frequency and decay. This was performed periodically throughout the program and the corresponding results are listed in Table 4-1. The data corresponding to the run numbers in Table 4-1 with pre as a prefix are data taken immediately before the run and with a post prefix are taken immediately after a run. Inconsistency among the frequency and damping values may be due in part to the manner in which the model was excited. Some modes were difficult to excite without causing others to respond also. All tweak data were reduced from the CEC records. The damping values are equivalent viscous damping coefficients. In general, the model was highly damped in all modes (rigid body as well as elastic) relative to typical values for full-scale structures. Although the higher damping is not expected to affect blade loads resulting from forced conditions, it could cause unconservatism where system stability is involved.

4.1 RIGID BODY

A summary of the rigid body frequency and damping data including the mount effects are provided in Table 4.2. Prior to the powered tests, Runs 6 through 10 were conducted which found the umbilical (which consisted of two air hoses and a wire bundle) to have no effect on the model rigid body frequencies and damping as mounted, therefore the data is essentially free of mount effects. Also, the addition of the snubber cables, when loose, were found to have no effect. The snubber cables when tightened increased the rigid body frequencies. The magnitude of restraint depended upon the operator (different ones during different test periods) as well as the manner in which the snubbing load was applied. Vertical support was provided with a bungee cable anchored to the test section ceiling from the model center of gravity. No problems were encountered with this arrangement during the test up to the maximum test dynamic pressure of 10.0 psf.

4.2 BLADE FREQUENCIES

During the course of testing it was noted that the monocoat material on the blades was cracking. Corresponding reduction in blade tweak frequencies also resulted as shown in Figure 4-1. This suggests that the monocoat contributed significantly to the blade stiffness.

SN-13 blade was tweak-tested on 3/10/71 (after the completion of wind tunnel tests) to determine the effects of monocoat on blade frequency. This blade was cantilevered at its shoulder (without pitch shaft) and the following results were obtained:

	Fully Monocoated	Outer Five Segments Cut	All Segments Cut
Flap Bending	3.6 cps	3.6 cps	3.5 cps
Chord Bending	13.2 cps	12.3 cps	10.6 cps

As indicated above, the monocoat had little effect on flap bending frequency but a large effect on the chord bending frequency. This was caused by the fact that the monocoat was loose between segments, and since the difference from the blade flap bending elastic axis to the monocoat was small there was no change in the flap bending frequencies. A large change in the chordwise bending frequency resulted because of its greater distance from the neutral axis. Blade damping coefficient variation with time shows considerable scatter but no basic variation with time, Figure 4-2.

TABLE 4-1

TWEAK TEST RESULTS

RUN	RIGID BODY			BLADE				WING		
	PITCH	ROLL	YAW	FB	CB	2FB	T	FB	CB	T
PRE 1 (2)				4.15	8.7	19.6				
BAFFLE TEST				.019	.014	.026				
PRE 3				4.1	9.0	18.4				
BAFFLE TEST				.017	.004	-				
PRE 11				4.15	9.2	19.2		5.15	11.0	18.0
				.017	-	-		.027	-	-
11				4.2	9.2				10.9	17.6
									.017	.012
PRE 14	1.5	.54						5.1		18.
	.06	.016						.03		-
POST 14				4.1	9.1	19.6				
				.014	-	-				
POST 28	1.5	.54		4.0	9.0	18.7		5.1	10.9	
	.027	.028		.016	-	.01		.031		
PRE 30				4.0	8.8	18.5				
				.022	.012	-				
POST 30	1.5									
	.09									
PRE 35				3.9	8.8	18.5	73.	5.2	10.5	15.3
				.03	.01	.018	.027	-	-	-
PRE 39				4.0	8.8	18.5	75.	5.1	10.3	
				.031	.017			.032	.017	
PRE 44	1.5	.55		3.9	8.4	17.9	69.	5.1	10.5	17.
	.07	.02		.022	.027	.022	.027	.041	.015	-
PRE 56	1.4	.53		3.9	8.4	17.6	69.	5.1	10.5	
	.065	.024		.021	.01	.032	.03	.026	-	
PRE 57	1.5	.60		BLADES OFF				5.4	10.7	16.8
	.05	.02						.022	.012	.010
POST 60	1.5			3.9	8.3	18.3	70.	5.1	10.3	18.4
	.046			.03	.013	.017	-	.04	-	-

NOTE: (1) 1st No. in Box is Frequency - CPS; 2nd is Damping Coef. c/c_c
(2) Pre means immediately prior to run. Post means immediately after run.

TABLE 4-1 (continued)

TWEAK TEST RESULTS

RUN	RIGID BODY			BLADE				WING		
	PITCH	ROLL	YAW	FB	CB	2FB	T	FB	CB	T
POST 62	.87	.30								
	.14	.10								
POST 67		.40		3.9	8.3	17.6		5.1	10.4	17.0
		-		.017	.015	.02		.055	-	-
POST 60 LEFT BLADE					8.5	19.3				
POST 62 MODEL SNUBBED	3.0	1.2								
POST 67 LEFT BLADE					8.0					

TABLE 4-2

PRE-TEST RIGID BODY FREQUENCY AND DAMPING -- POWER AND WIND OFF

MODE	WITHOUT UMBILICAL WITHOUT SNUBBERS				WITH UMBILICAL No Snub.		Snub Loose		Snub Tight	
	i_N = 90°		i_N = 0°		i_N = 0°		i_N = 0°		i_N = 0°	
	f	ζ	f	ζ	f	ζ	f	ζ	f	ζ
	Run 6		Run 7		Run 8		Run 9		Run 10	
PITCH	1.5	.027	1.5	.026	1.55	.023	1.5	.032	3.5	-
ROLL	.56	.012	.56	.014	.58	.014	.59	.014	1.5	-
YAW	.50	.015	.48	.014	.47	.013	.47	.014	.50	-
VERTICAL	.57	>.10	.57	>.10	.57	>.10				
LATERAL	1.02	.004	1.02	.004	1.02	.005	1.02	.004	1.04	-
LONGITUDINAL	SAME AS LATERAL									

f = frequency - CPS

ζ = damping coefficient - %

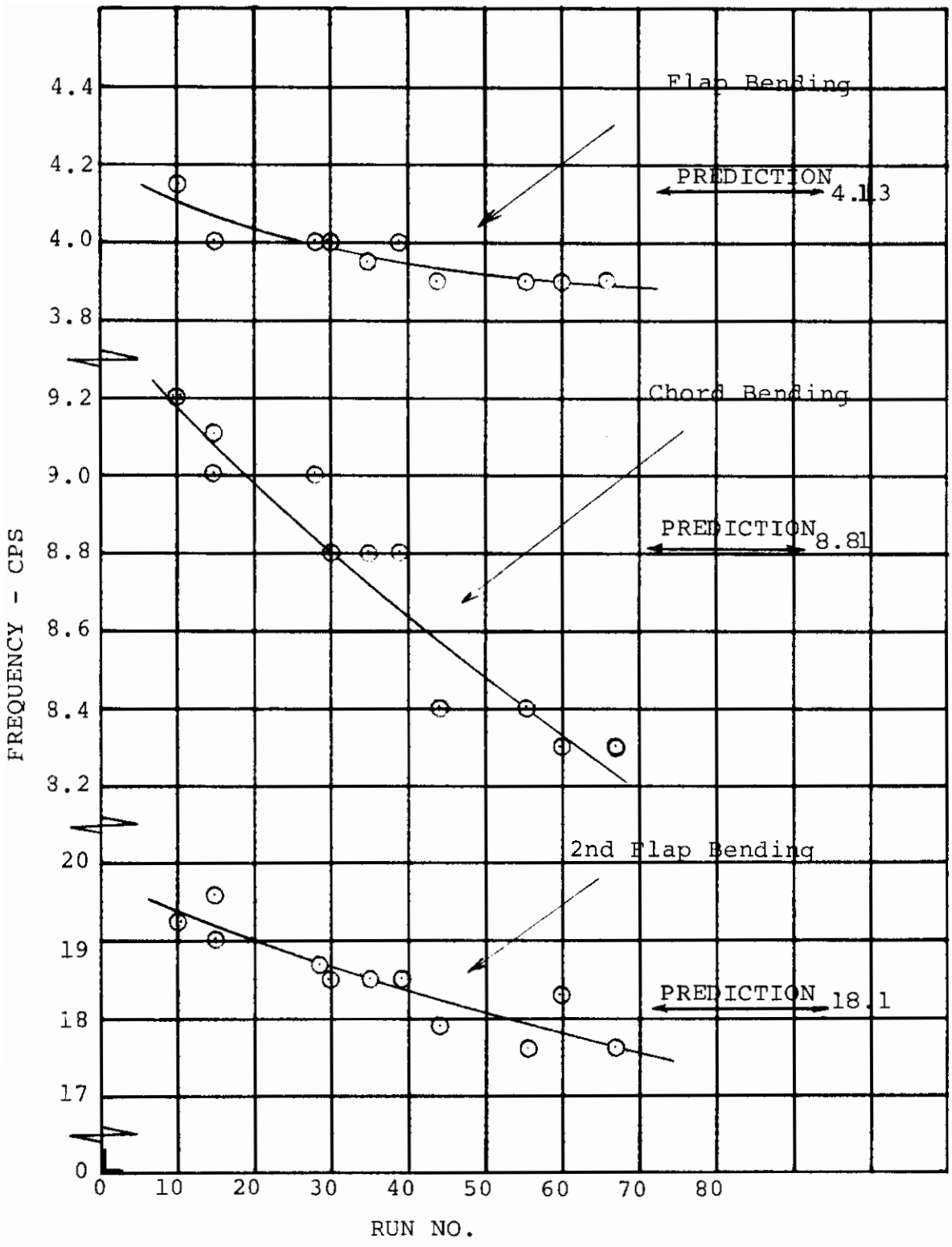


FIGURE 4-1. - VARIATION OF BLADE FREQUENCY WITH RUNS. ROTOR RPM = 0.

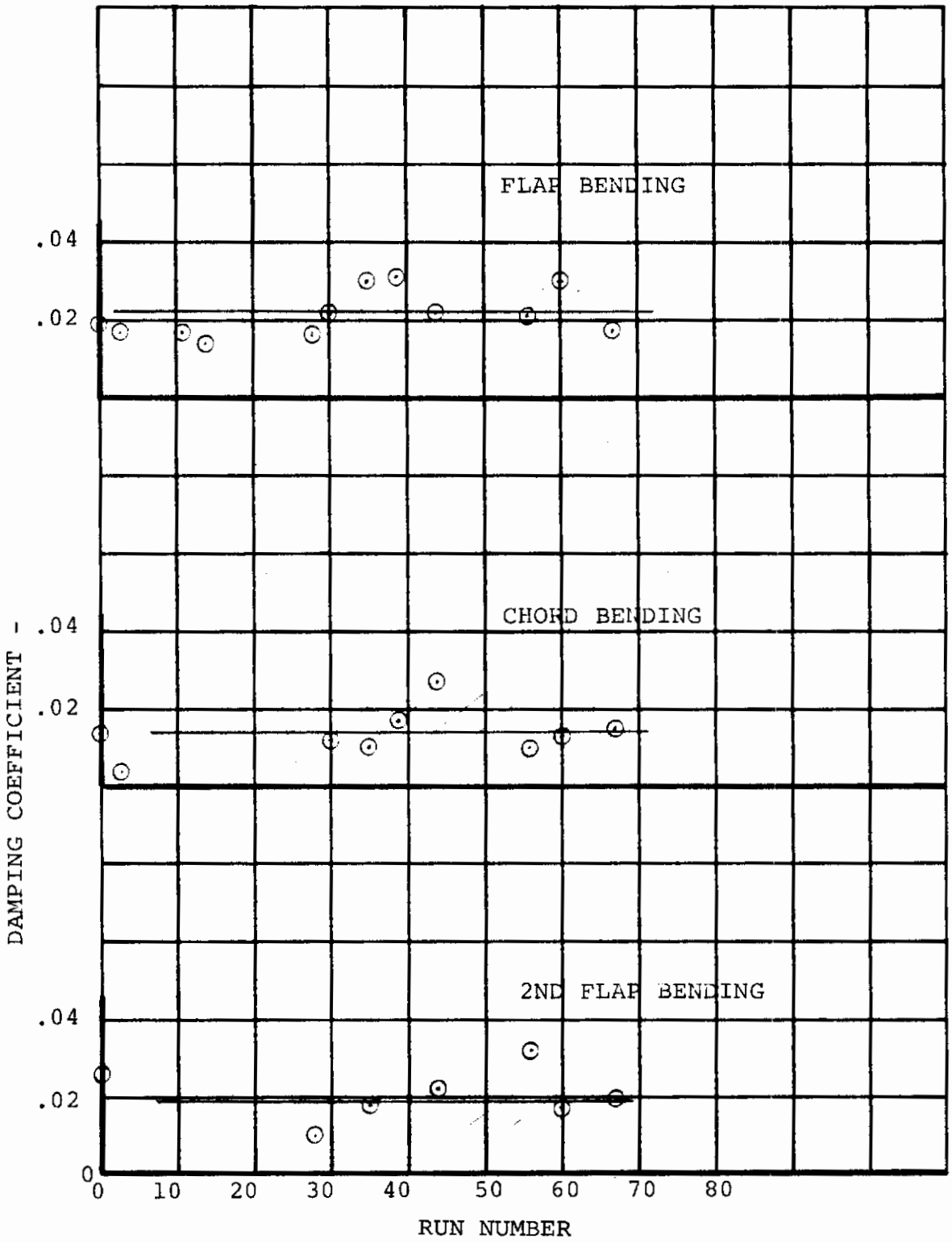


FIGURE 4-2. BLADE MODAL DAMPING FROM TWEAK TESTS
$\Omega = 0$

5.0 ROTATING BLADE FREQUENCY MEASUREMENTS

One of the test objectives was to determine the rotating blade frequencies in hover and correlate the results with a coupled blade frequency analysis. Test results show good agreement with analysis for the first three blade modes over the RPM range tested.

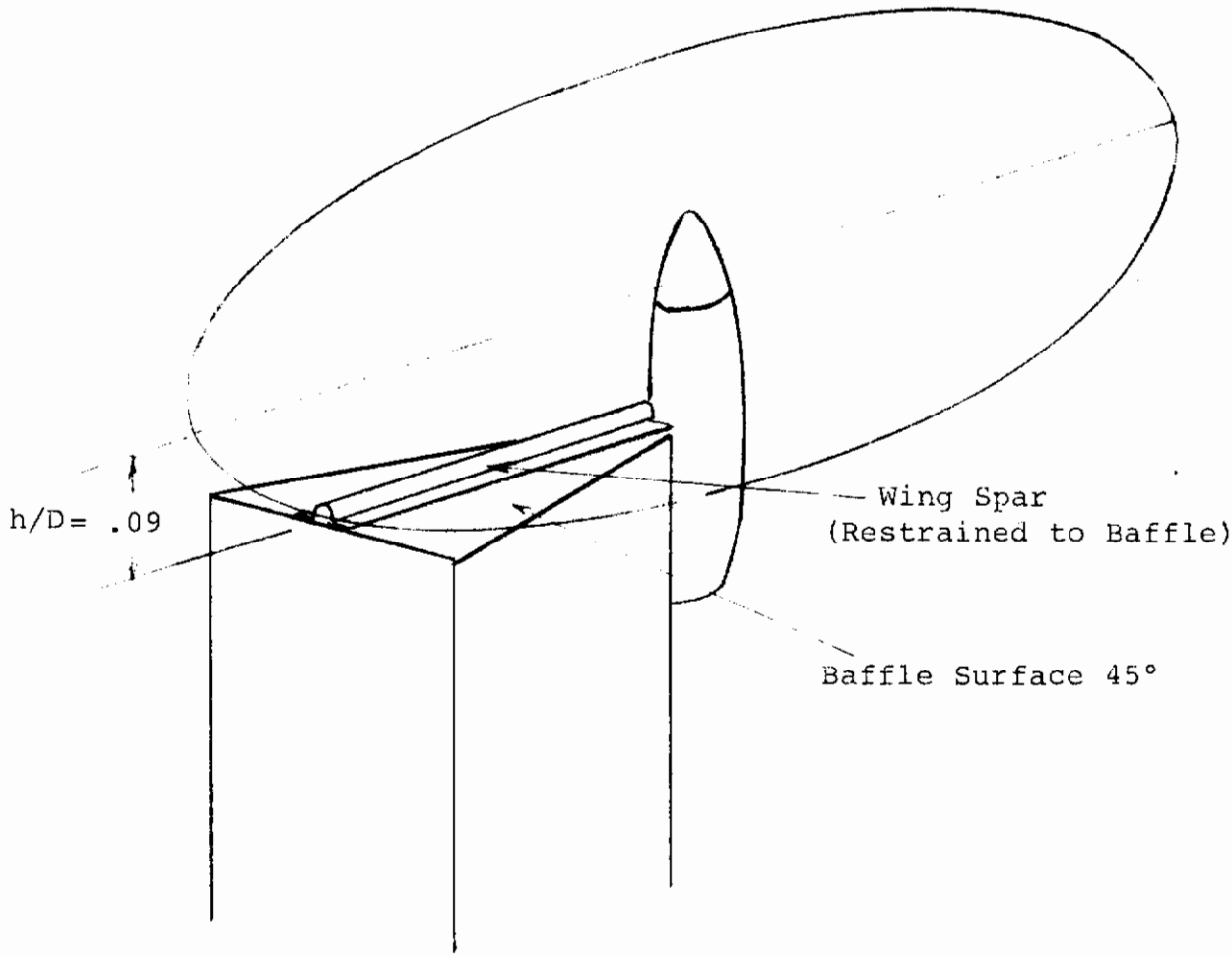
Substantiation of the rotating blade frequencies was accomplished through the use of a baffle configuration used on a previously tested rotor for blade frequency measurement. The baffle arrangement is shown in Figure 5-1. Baffle combinations of 1, 2, 3 and 4 were used.

A comparison of the baffle test results with prediction is shown in Figure 5-2 and shows good agreement with prediction for the first three modes. The test data used to determine the 1, 2, 3, 4 and 5 per rev blade frequencies is shown in Figures 5-3 through 5-14. The alternating chord and flap bending data in Figures 5-3 and 5-4 show a high response at 550 RPM. Alternating chord bending response in Figures 5-5 and 5-6 show the same amplification near 550 RPM. The blade loads were near fatigue allowables at 550 RPM and only oscillograph records were taken in this RPM range since harmonic analysis of the data required additional running time per data point. The second harmonic test data in Figures 5-7 and 5-8 show a sharp response to the two baffle configurations at 130 and 280 RPM. Analysis results indicate the first two modes to be highly coupled in this RPM range but the 2 per rev crossing at 130 RPM was predicted to be predominently a flapping mode and the 2 per rev crossing at 280 RPM predominently a chordwise mode. The high chord bending response at 280 RPM and high flap bending at 130 RPM substantiate the predicted 2 per rev blade frequencies and also the predicted modal predominance.

The third harmonic blade bending response to the baffles is shown in Figures 5-9 and 5-10. The chord bending data show a large response at 100 and 170 RPM to the 3 per rev baffle configuration and the flap bending data shows a response to 3 baffles at 90 RPM. The largest third harmonic flap bending response occurred between 500 and 600 RPM being twice as large as the second harmonic flap bending response between 500 and 600 RPM as shown in Figure 5-8 and six times as large

as the fourth harmonic flap bending response as shown in Figure 5-10 for the same RPM range. This indicates that the third harmonic flap bending response is due to a 3 per rev frequency crossing on a higher flapping mode and not due to the one per rev chordwise frequency crossing at 550 RPM. Although the third harmonic flap bending data is sparce between 500 and 600 RPM, the RPM corresponding to the highest flapping response (570 RPM) was determined to be the 3 per rev crossing for the second flap mode.

The fourth harmonic blade bending response is shown in Figures 5-11 and 5-12. The chord bending response shows a large amplification at 135 RPM and 360 RPM for the 4 per rev baffle configuration and the flap bending response shows little amplification at 135 RPM and a large amplification at 360 RPM. The large response at 135 RPM is due to a 4 per rev crossing on the second mode (predominently chordwise) and the flap bending response at 360 RPM is due to a 4 per rev crossing on the third mode (primarily second flap). Although a 5 per rev baffle configuration was not tested, the test results show a significant fifth harmonic response. The fifth harmonic blade bending response is shown in Figures 5-13 and 5-14 and show a chord bending amplification at 110 RPM and 265 RPM and a flap bending amplification at 265 RPM. The response to 110 RPM is due to a 5 per rev frequency crossing on the second mode and the response at 265 RPM is due to a 5 per rev frequency crossing on the third mode.



Note: Model was partially assembled.

Tests were conducted with
1, 2, 3 and 4 baffles.

Figure 5-1. Schematic of Baffle Test Set-Up

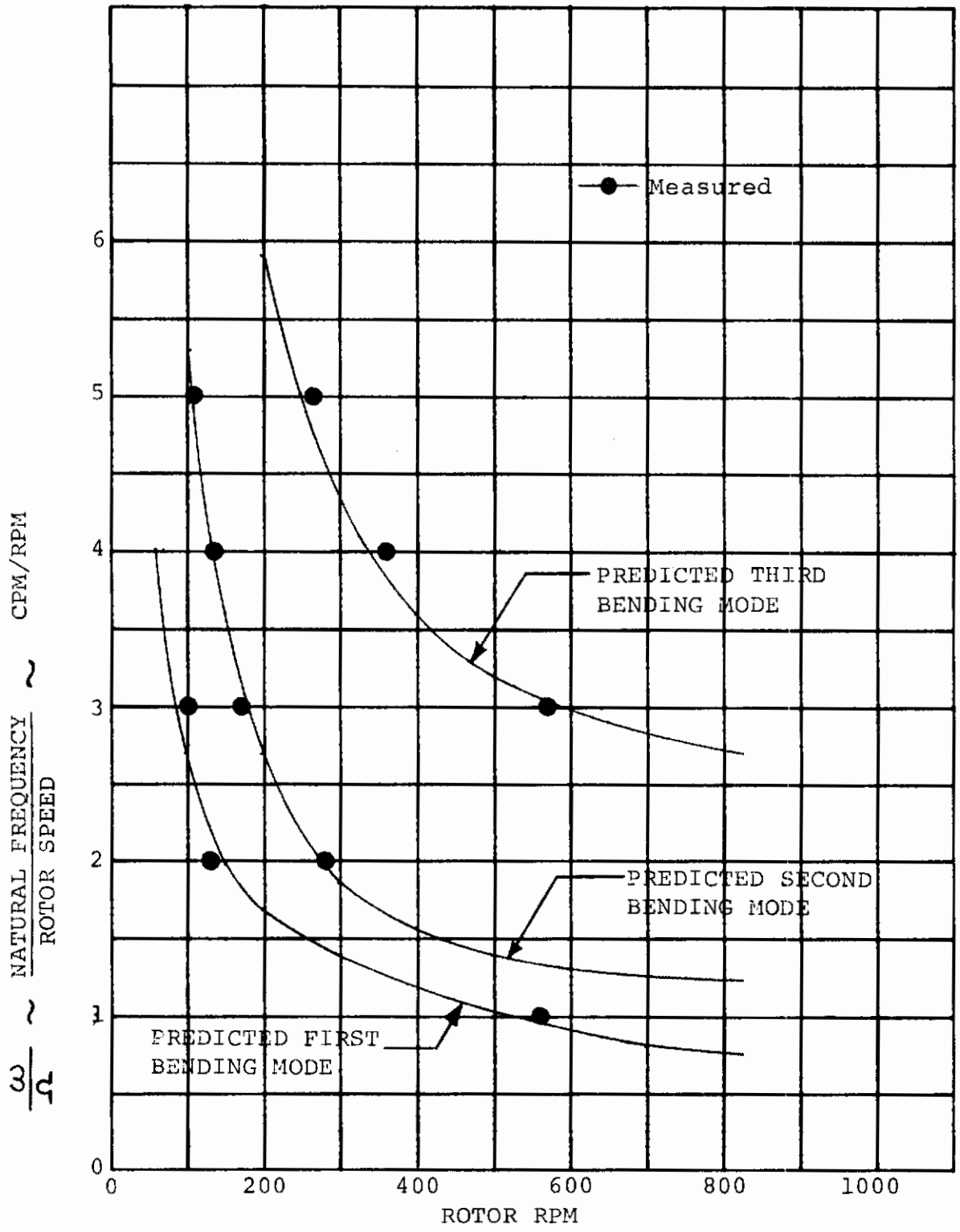


FIGURE 5-2. PREDICTED AND MEASURED ROTOR BLADE NATURAL FREQUENCIES FOR $\Theta_{.75}$ = 3 DEG.

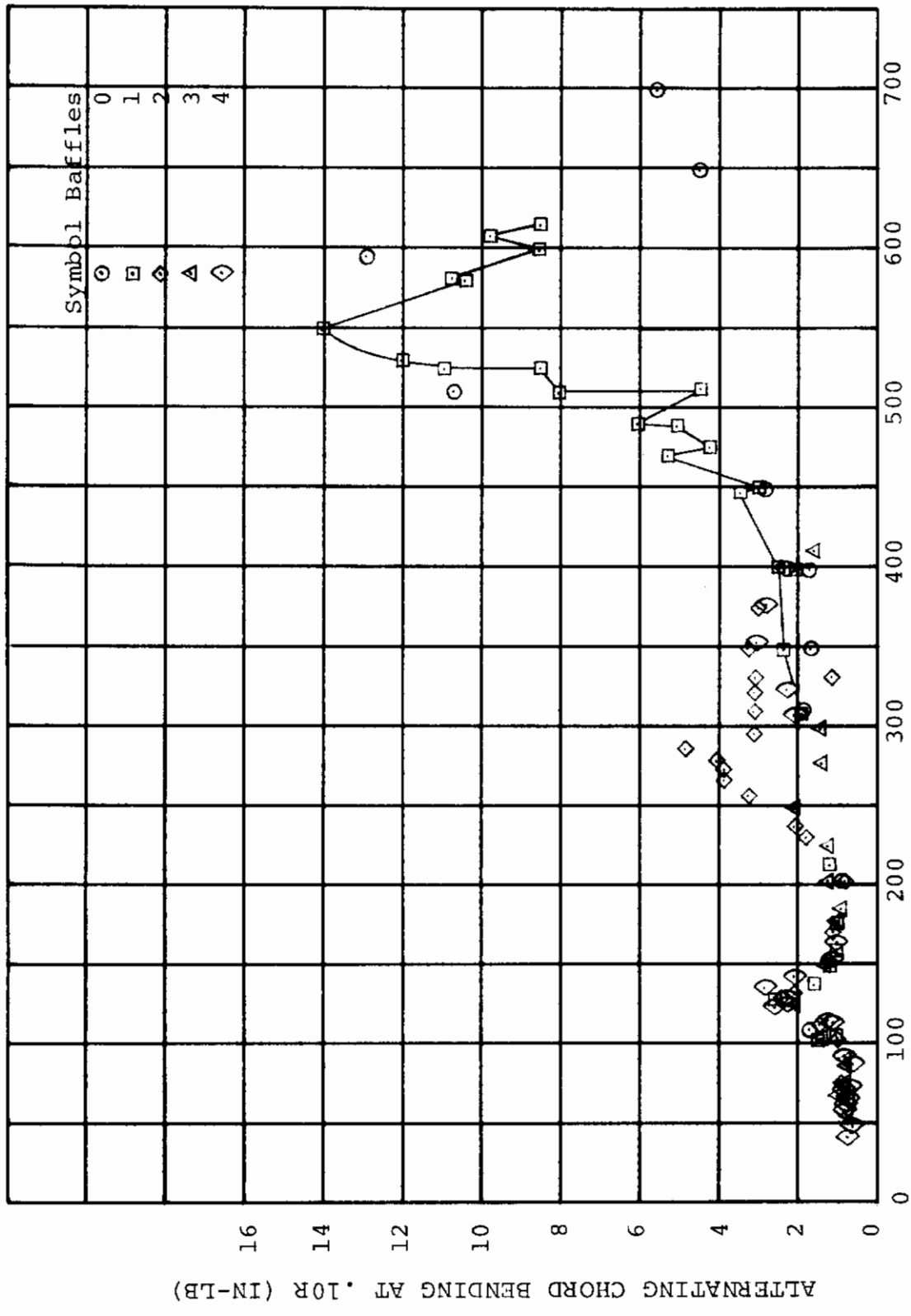


FIGURE 5-3. ALTERNATING CHORD BENDING RESPONSE TO BAFFLES

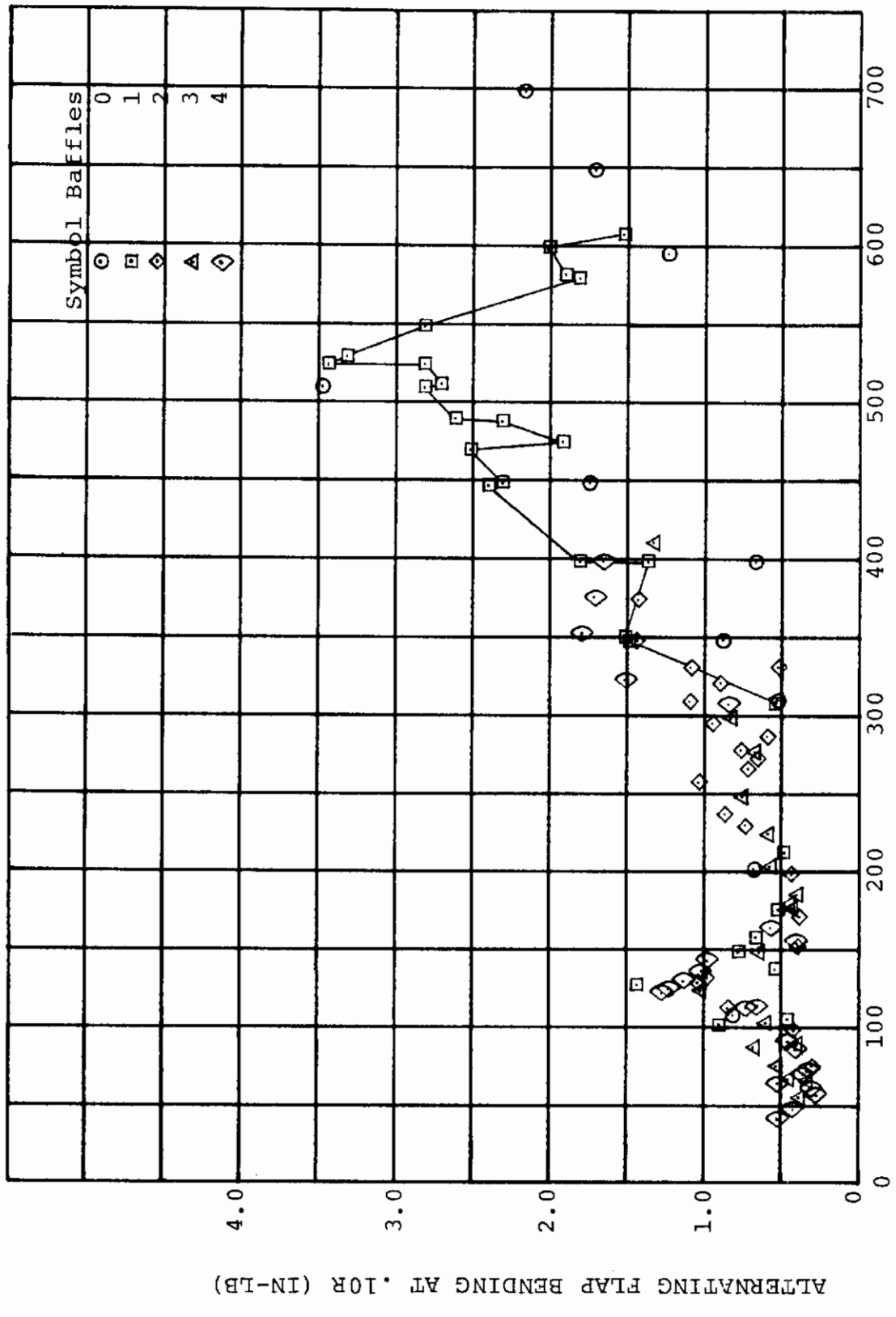


FIGURE 5-4. ALTERNATING FLAP BENDING RESPONSE TO BAFFLES

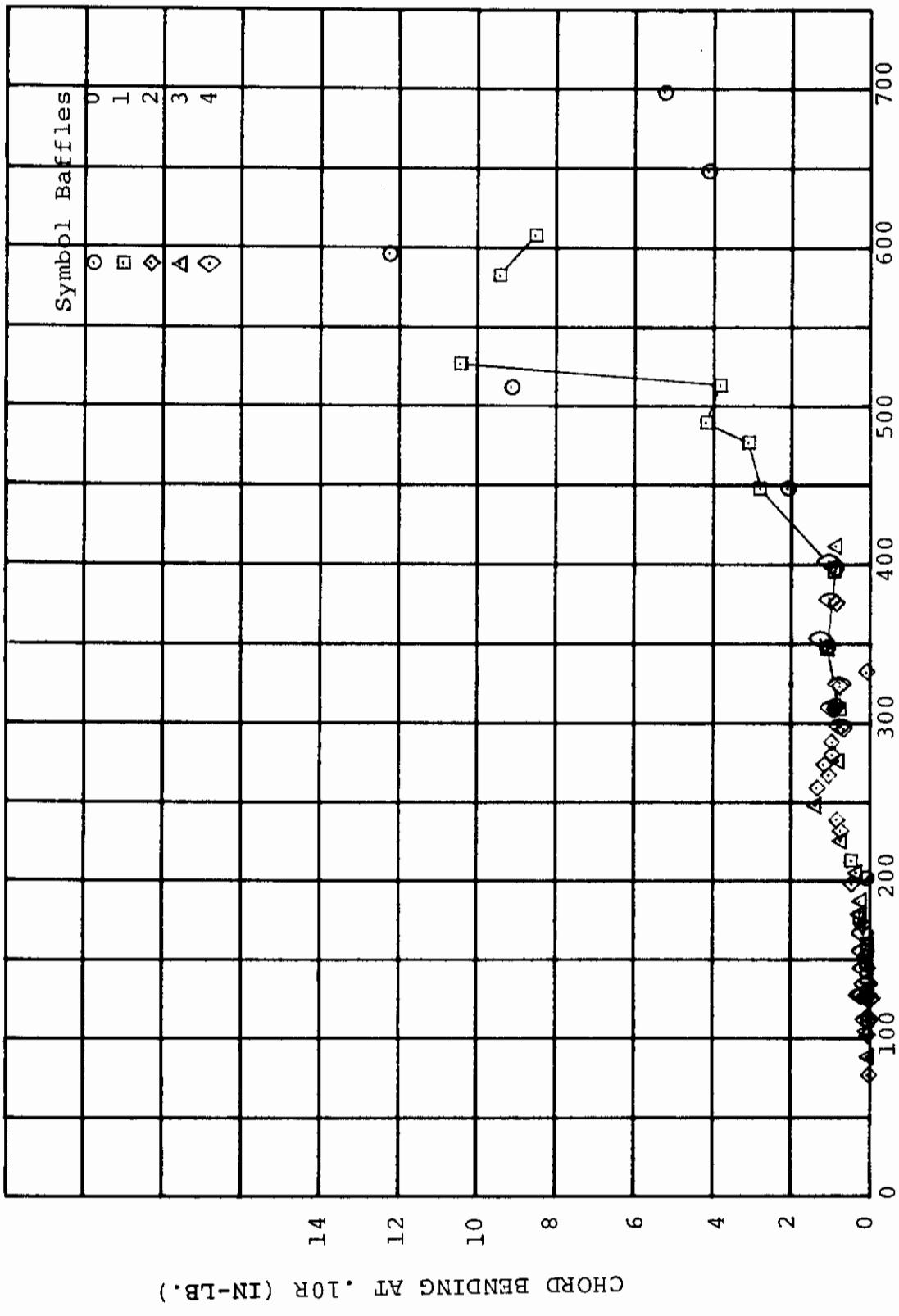


FIGURE 5-5. 1ST HARMONIC CHORD BENDING RESPONSE TO BAFFLES

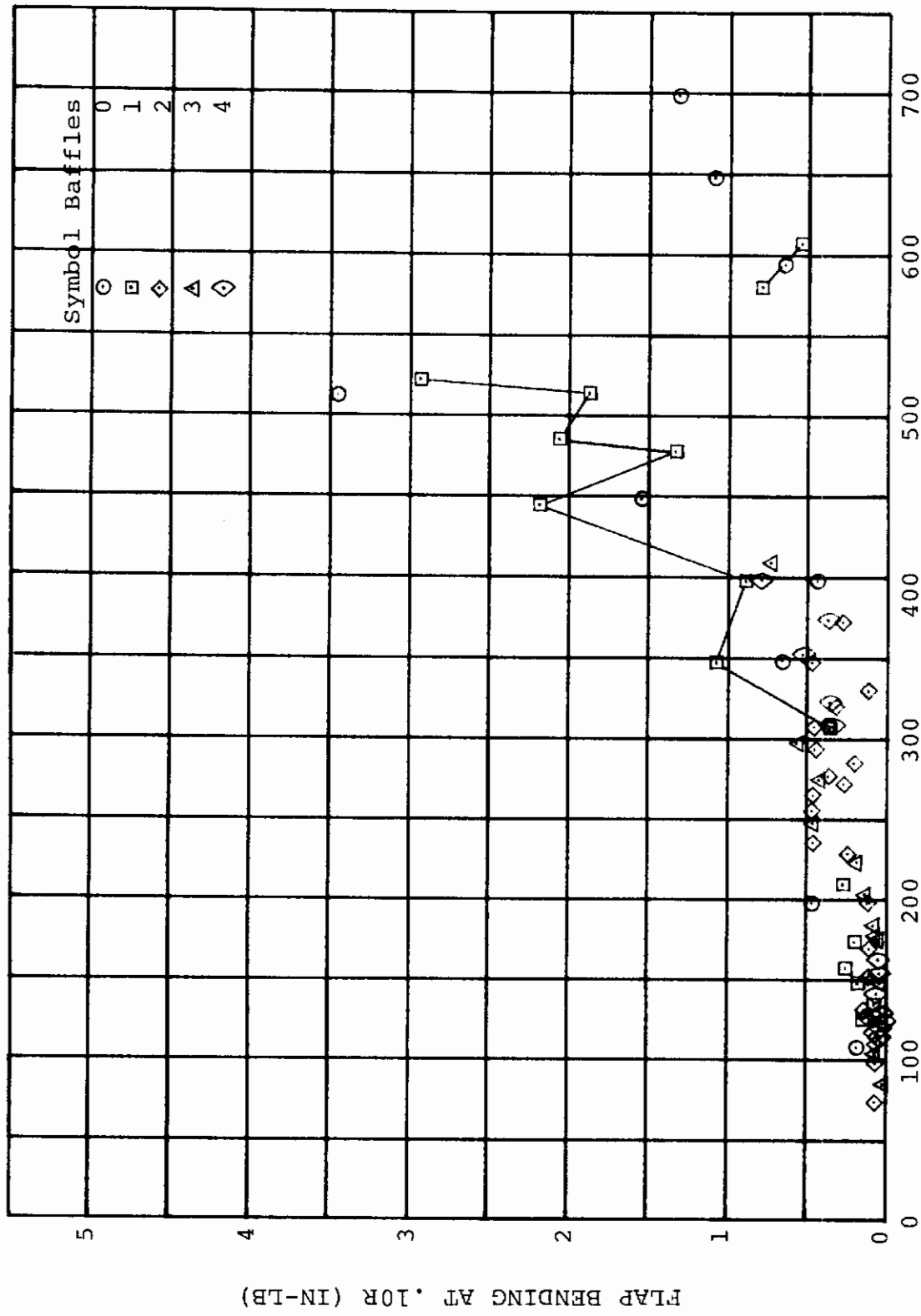


FIGURE 5-6. 1ST HARMONIC FLAP BENDING RESPONSE TO BAFFLES

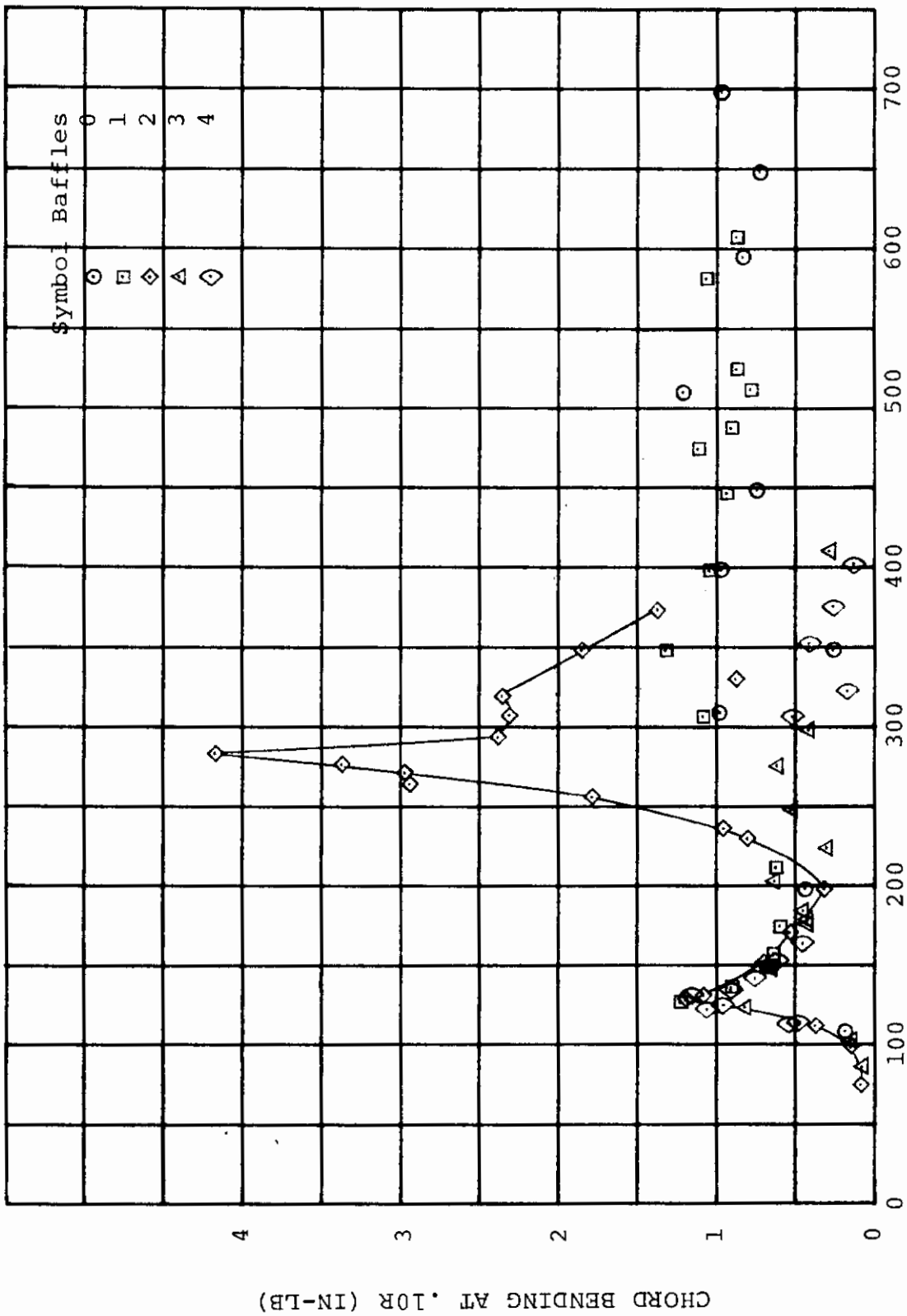


FIGURE 5-7. 2ND HARMONIC CHORD BENDING RESPONSE TO BAFFLES

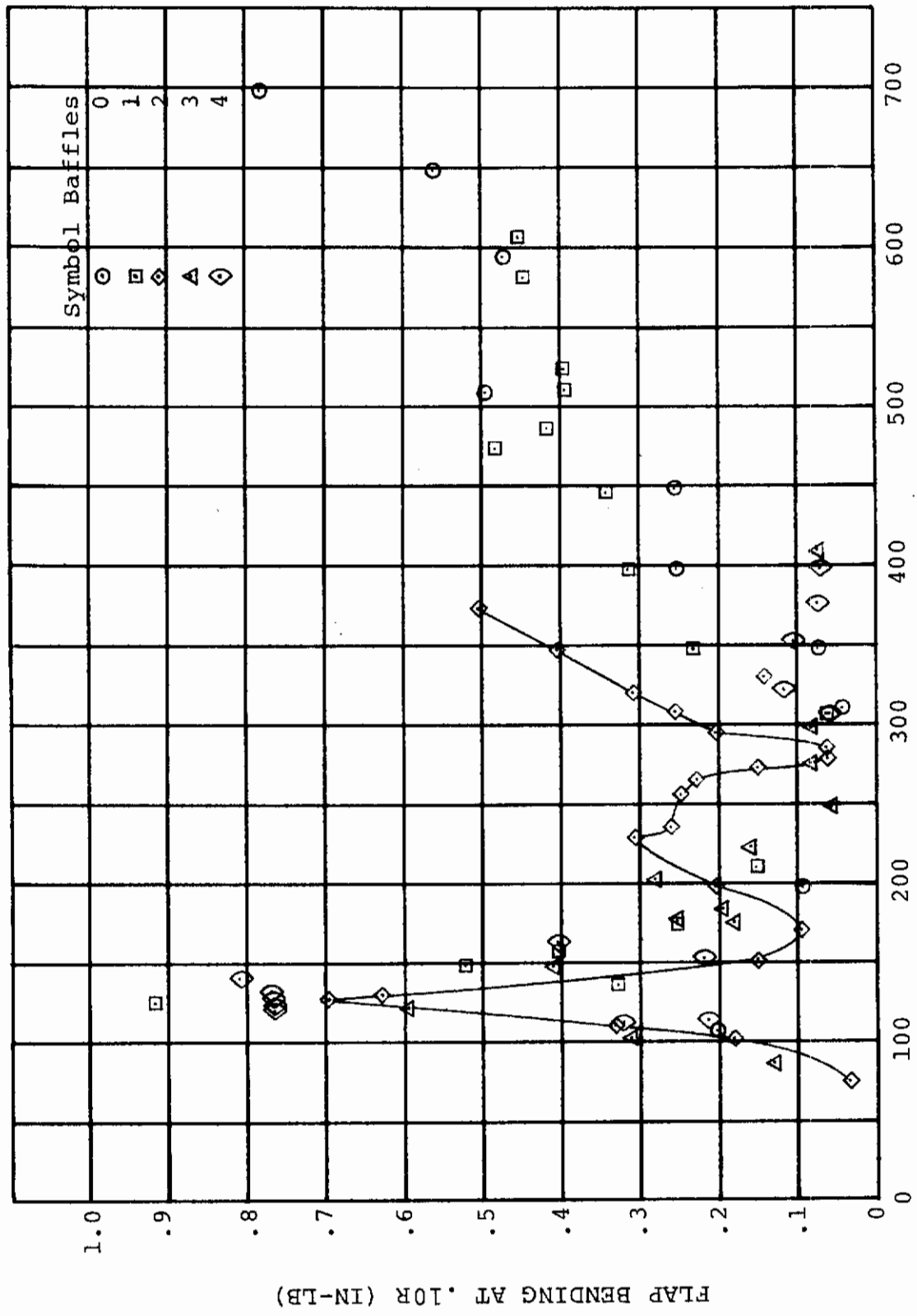


FIGURE 5-8. 2ND HARMONIC FLAP BENDING RESPONSE TO BAFFLES

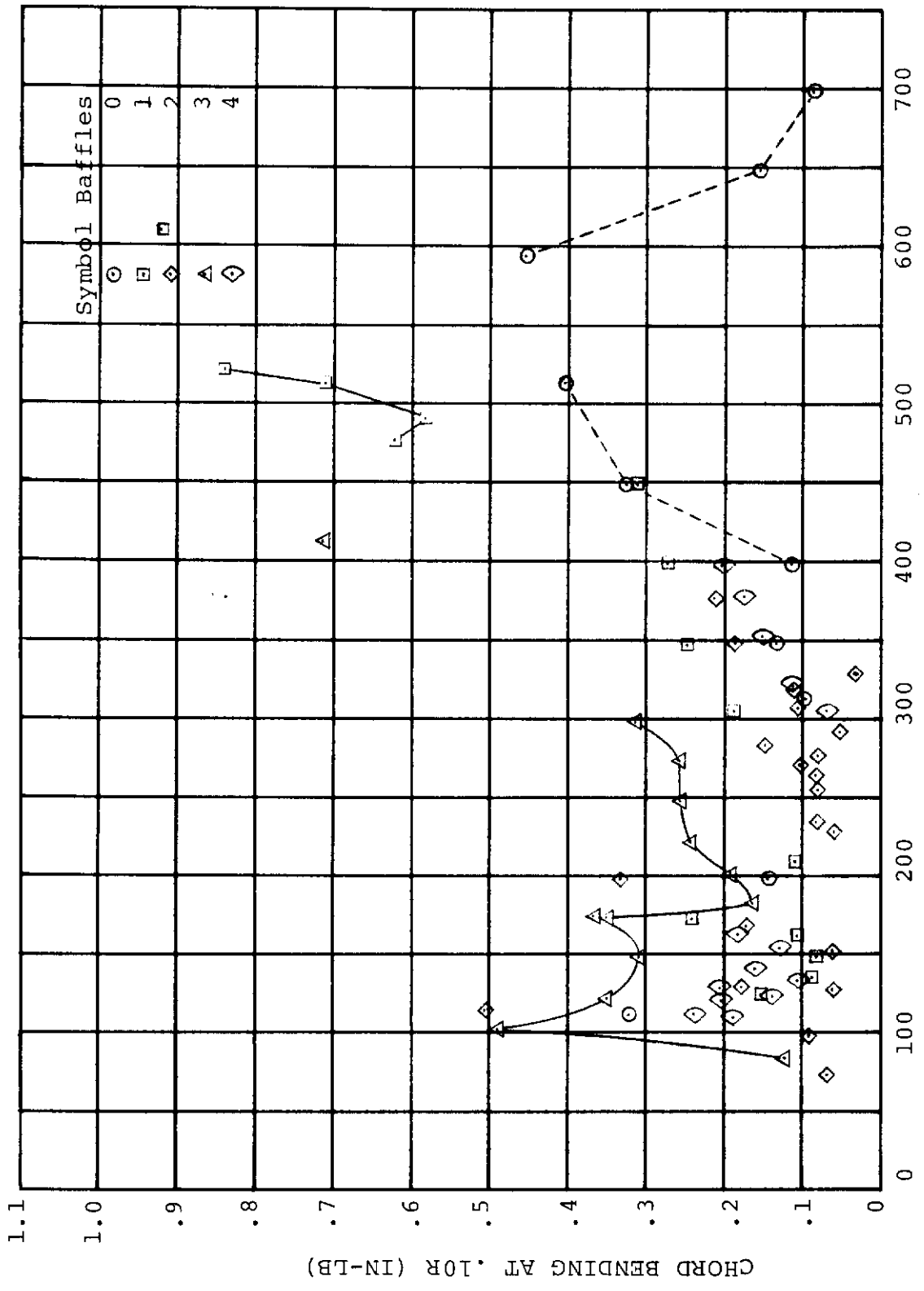


FIGURE 5-9. 3RD HARMONIC CHORD BENDING RESPONSE TO BAFFLES

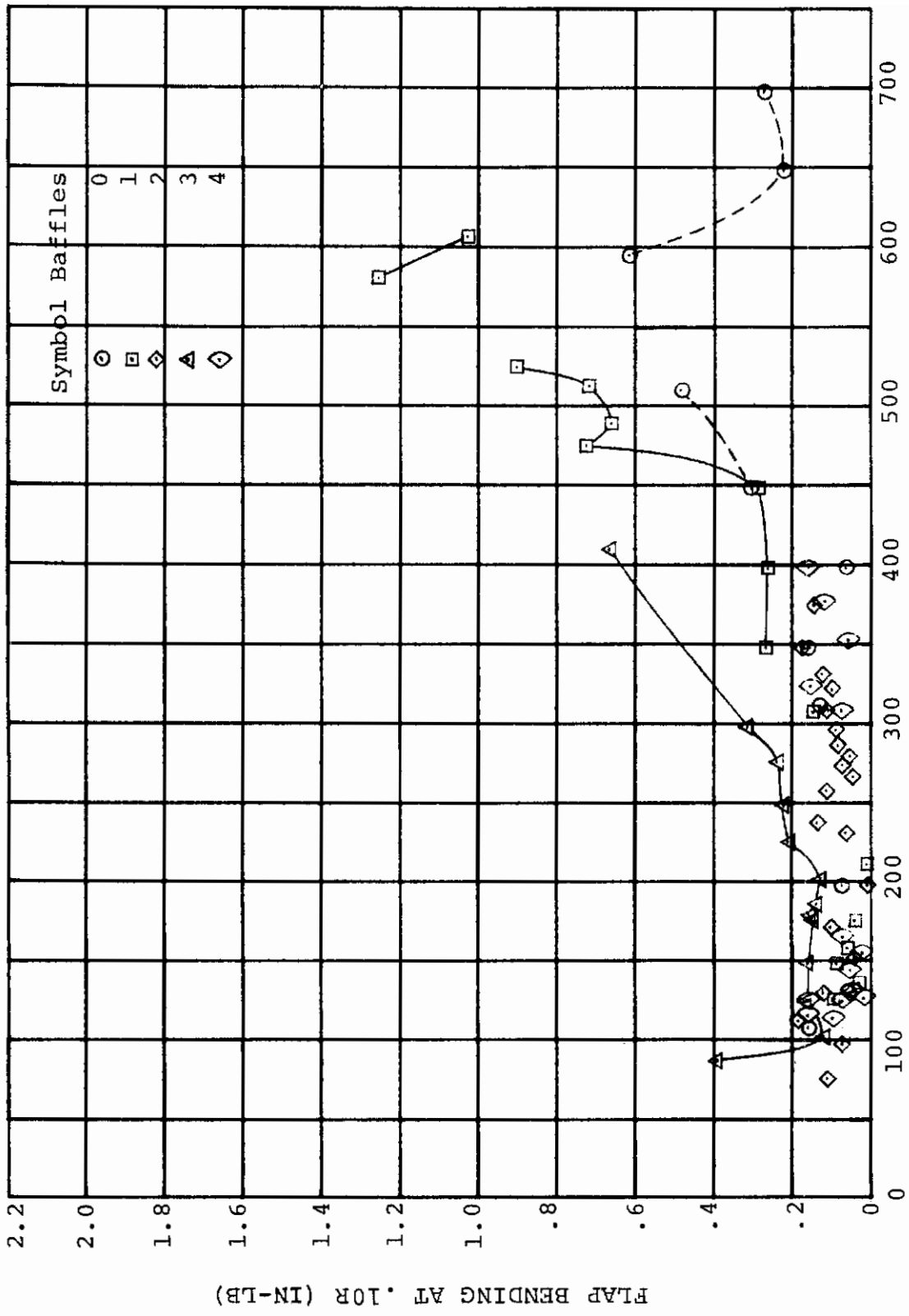


FIGURE 5-10. 3RD HARMONIC FLAP BENDING RESPONSE TO BAFFLES

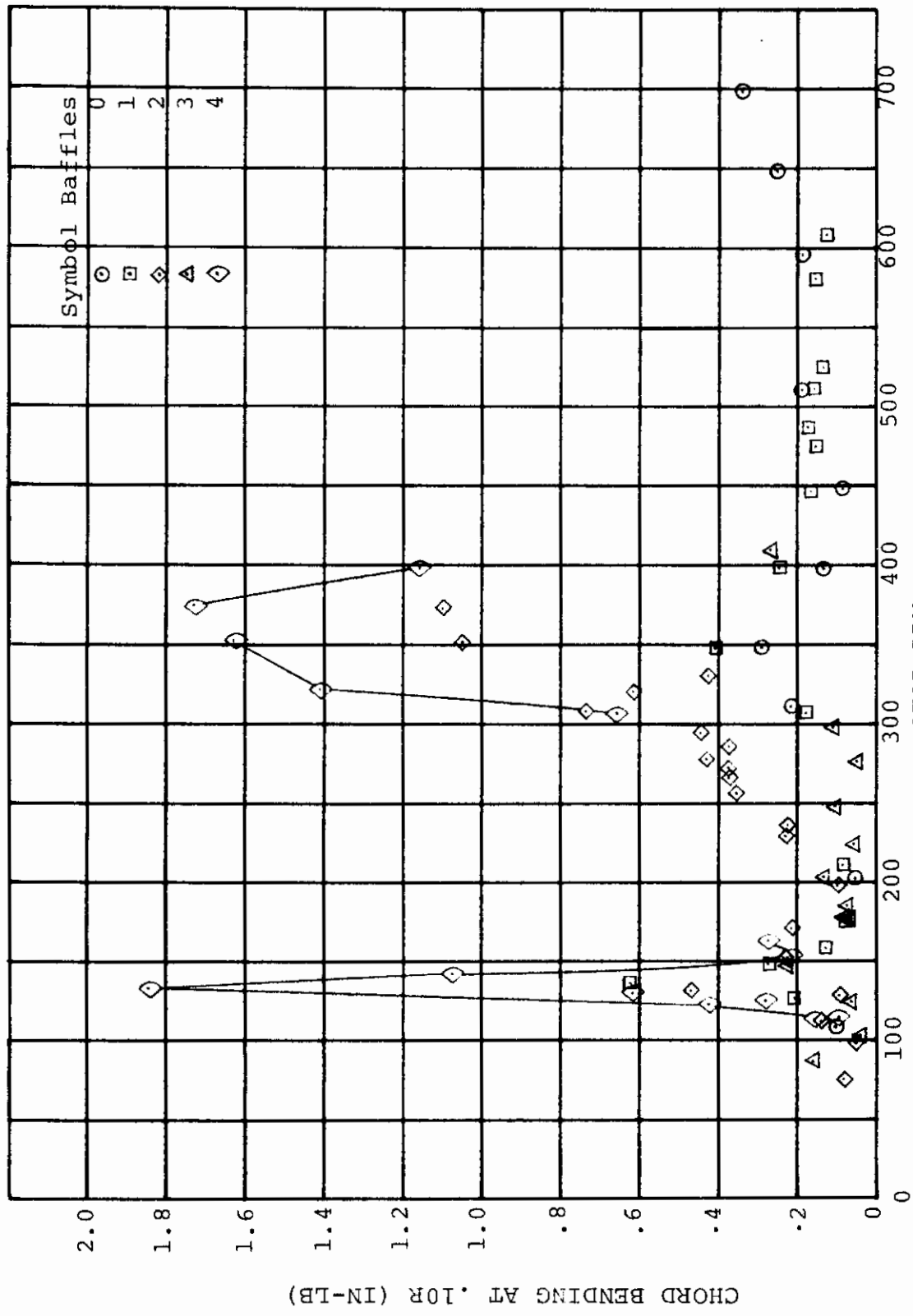


FIGURE 5-11. 4TH HARMONIC CHORD BENDING RESPONSE TO BAFFLES

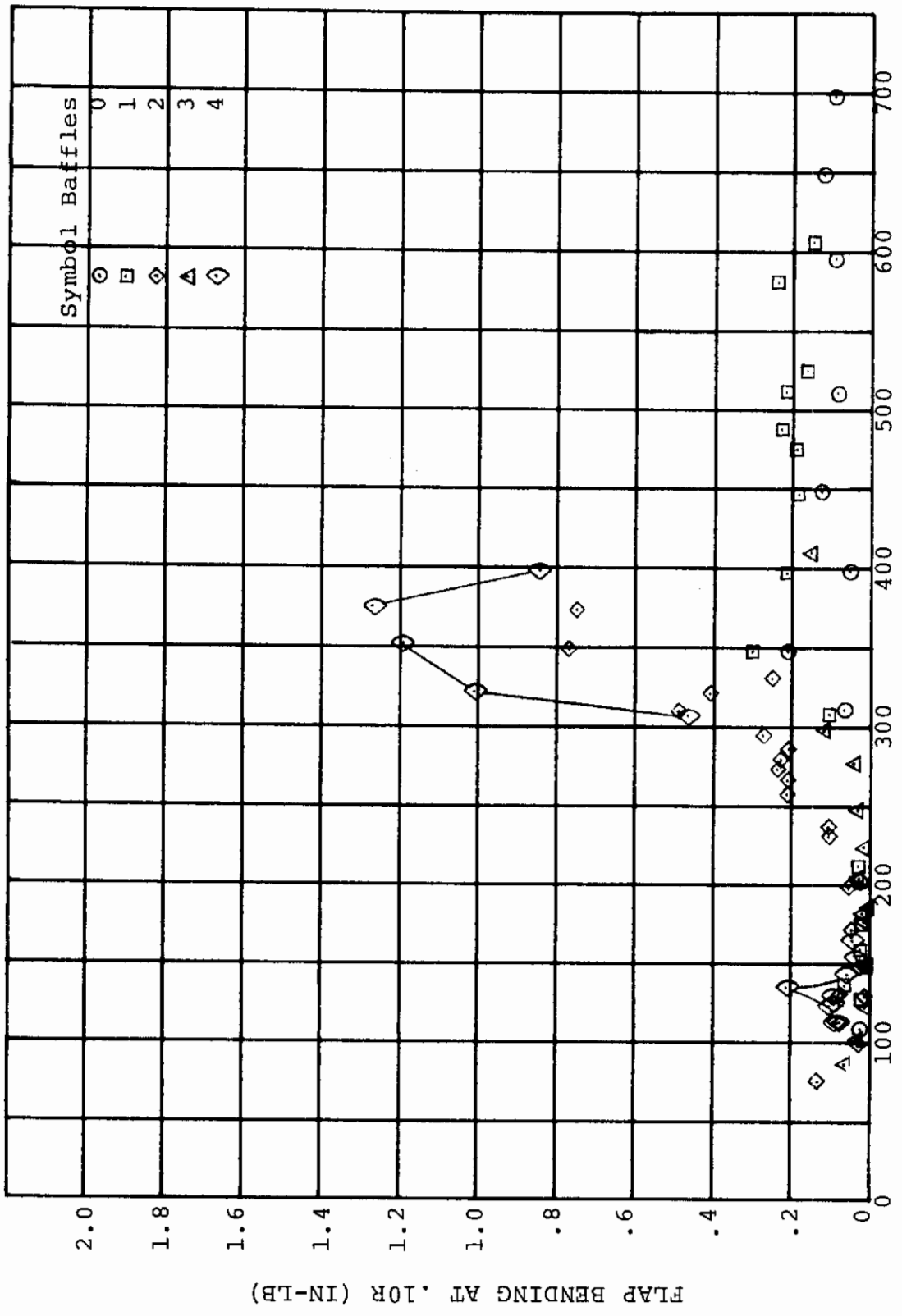


FIGURE 5-12. 4TH HARMONIC FLAP BENDING RESPONSE TO BAFFLES

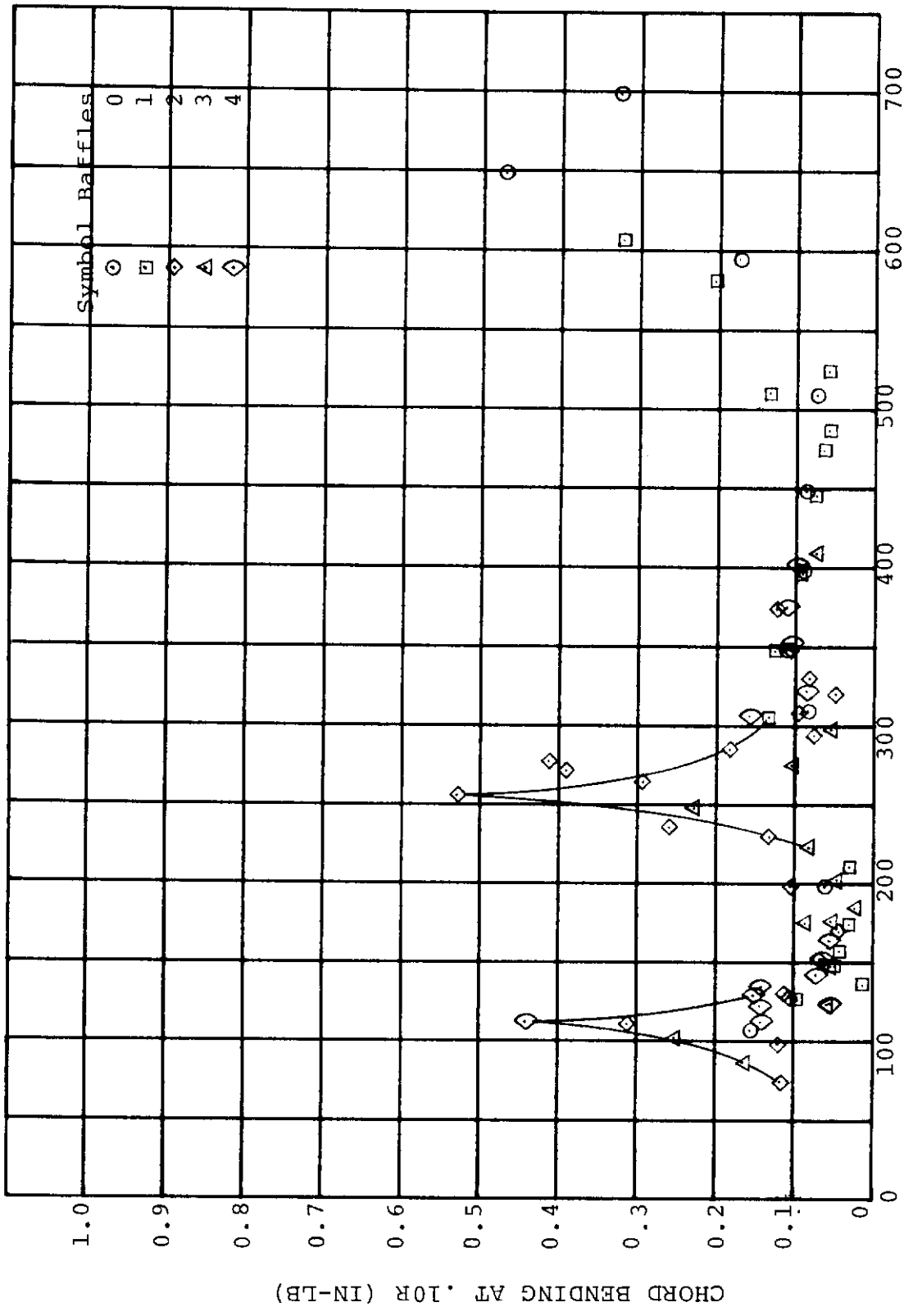


FIGURE 5-13. 5TH HARMONIC CHORD BENDING RESPONSE TO BAFFLES

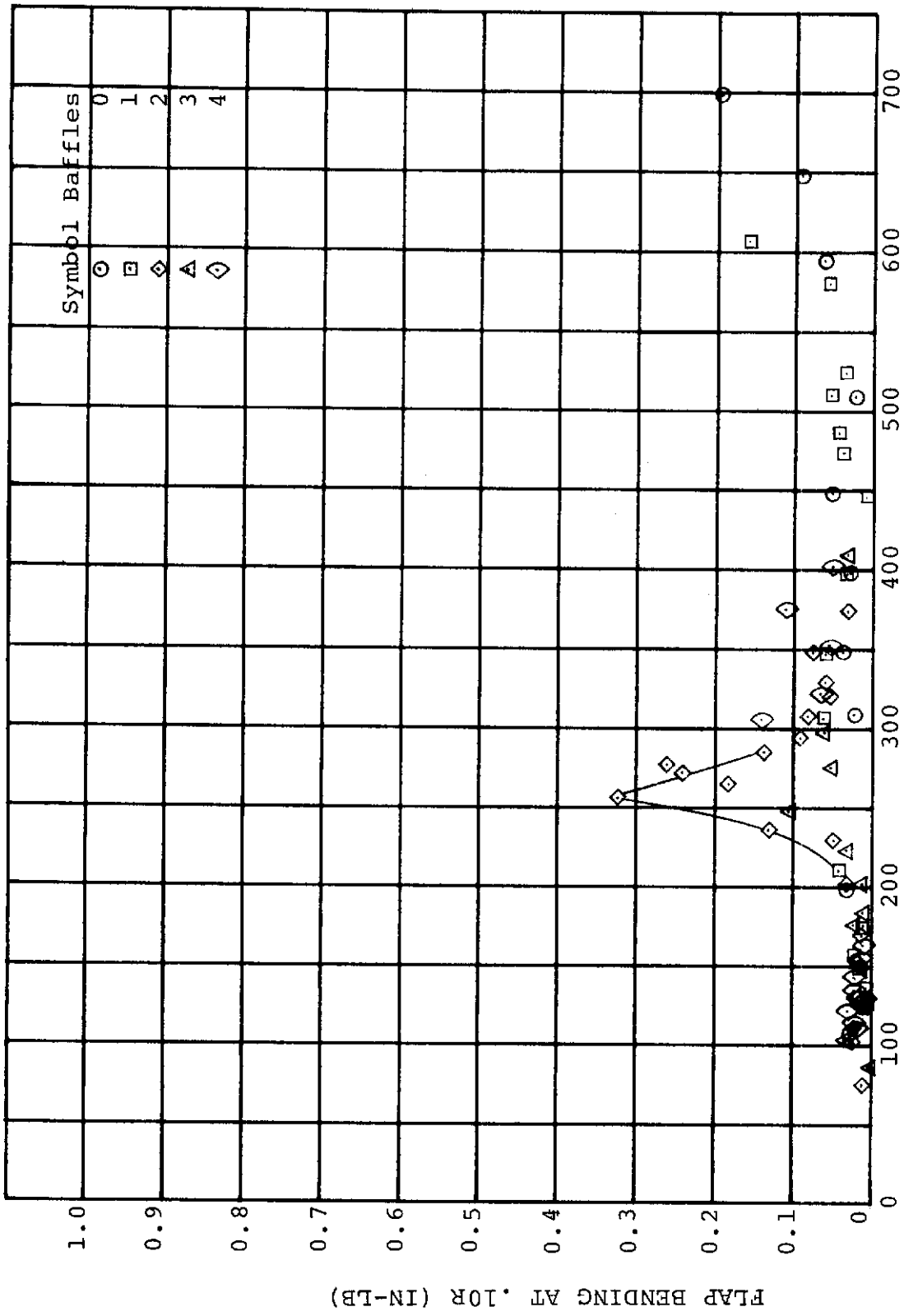


FIGURE 5-14. 5TH HARMONIC FLAP BENDING RESPONSE TO BAFFLES

6.0 BLADE LOADS

A major objective of the test program was to determine blade loads in hover, transition and cruise modes. Parameters varied were collective pitch, cyclic pitch, rotor speed, nacelle tilt, fuselage pitch and yaw, and dynamic pressure. Good results were obtained and the effects of each of the above parameters on blade alternating loads are presented herein.

Since the blade strain gages were bonded to the blade spar, all blade bending moments are with respect to the blade and not with respect to the disc plane. At collective angles ($\theta_{.75}$) as low as 30° the measured root chordwise bending moment is primarily an out of plane deformation due to the high twist of rotor blade. This is discussed in Section 3.3 and illustrated in Figure 3-5.

During the initial portion of the test, one rotor was excessively unbalanced which resulted in higher blade loads and a restricted rotor speed range of operation. Corresponding results are discussed in Section 6.1.1. It was found that excessive unbalance increased not only the minimum values of loads, but also the variation of loads with cyclic. To achieve properly scaled rotor balance would require balancing the model to within 0.0001 in-lb which is not practicable. However, after the initial portion of the test a balance was achieved which gave low minimum loads and is considered to provide good loads data. Because residual unbalance was not zero, the loads data may be conservative.

The wind tunnel wall corrections are less than data scatter for all test conditions because the model operated at very low disc loading (Froude scaling), and the model to tunnel ratios are small. Recirculation is minimized by the slotted test section.

All data presented herein were obtained from the right rotor blade, SN-16 LB at 0.10R for flap and chord bending and 0.15R for torsion.

6.1 HOVER BLADE LOADS

Blade loads data were obtained in hover to show the effect of cyclic pitch, rotor-rotor interference, stall flutter, collective pitch, rotor spin-up and spin-down and low forward speed on blade loads.

Cyclic pitch test results showed that the slope of blade loads per degree of cyclic are not affected by ground effect but IGE

does increase the value of minimum blade loads at zero cyclic. Increased collective pitch increased blade loads sensitivity to cyclic pitch application.

At differential collective pitch values corresponding to that required for roll control power in hover, rotor-rotor interference had essentially no effect on blade alternating loads.

Low amplitude stall flutter inception occurred in hover at a collective pitch angle of approximately 11°; however loads were low.

For a rotor precone of 5 degrees, the test data showed that the steady flap bending at the root is zero at approximately 10.5 degrees collective pitch at 705 RPM.

Rotor start-up and shut-down showed that maximum alternating chord bending lagged the one per rev chordwise frequency crossing by 50 RPM. The maximum chord bending loads during start-up were 1.3 times those during shut-down.

6.1.1 Effect of Cyclic Pitch

One of the primary objectives was to determine the effect of cyclic pitch on blade loads in hover. Additional considerations were the effect of collective pitch and ground interference on blade loads due to cyclic pitch. The results of cyclic pitch effects on blade loads are shown in Figures 6-1 through 6-11 and are discussed below. Samples of blade load wave forms are given in Figure 6-1 for a series of cyclic values. The primary one per rev loads vary with cyclic, as would be expected, and there is also a small amplitude 3 per rev load in the flap bending direction.

Figures 6-2 and 6-3 show alternating chord and flap bending data as cyclic pitch is varied for hovering IGE at 4.7 degrees collective pitch and 825 RPM. These data show a near linear increase with cyclic from a minimum value. The alternating chord bending gradient is 3 inch pounds per degree cyclic and the flap bending gradient is 2 inch pounds per degree cyclic. Figures 6-4 and 6-5 show blade loads data that were recorded early in the test in hover OGE at 4 degrees collective pitch and 750 RPM. The data in these figures were taken when the rotor was out of balance and illustrate the increased sensitivity of cyclic pitch on blade loads with rotor unbalance combined with reduced rotor speed (750 RPM). The design hover speed for the model was 825 RPM but 750 RPM was the optimum rotor speed for minimum blade loads with the unbalanced rotor.

The gradient of these blade loads with cyclic are 8 in lb chord bending per degree cyclic or 2.5 times the gradient for chord bending shown in Figure 6-3. The gradient for flap bending is 1.1 times the gradient for flap bending shown in Figure 6-3. Data for correlation purposes must therefore be chosen to contain minimum rotor unbalance.

Figures 6-6 through 6-9 show the effect of model height on blade loads in hover with cyclic pitch at the design model hover rotor speed. These data show that ground interference does not affect the gradient of blade loads per degree cyclic pitch but does affect the level of blade loads at minimum cyclic. In Figure 6-6, the alternating chord bending at minimum cyclic hovering IGE is 2 times the chord bending hovering OGE in Figure 6-8. Comparing Figures 6-7 and 6-9, model height has no effect on alternating flap bending.

Figures 6-2, 6-3, 6-9 and 6-11 show the effect of collective pitch on blade loads in hover IGE with cyclic pitch. These data show that collective affects the gradient of blade loads per degree of cyclic pitch. In Figure 6-10 at 11 degrees collective the gradient of alternating chord bending per degree cyclic is 2.5 times the gradient of chord bending at 4.7 degrees collective in Figure 6-2. Comparing Figures 6-3 and 6-11, increasing collective from 4.7 to 11 degrees increases the gradient of flap bending loads per degree of cyclic by 25 percent.

6.1.2 Rotor-Rotor Interference

To determine the effect of downwash from one rotor on the blade loads of the other rotor the collective pitch on the left rotor was varied between 5.9^{o} and 13.3^{o} while the collective on the right rotor whose blade loads were being monitored was kept constant. All other parameters were held constant. The test results show that the range of differential collective (10.5 to 13 degrees) required for roll control power when hovering at 1g OGE has essentially no effect on alternating blade loads. The phasing of the harmonic content changes significantly as shown in Figure 6-12. No scales are shown because the phasing and harmonic content are of primary interest. Recirculation was not a factor in these tests because the rotor disc loading was very small and the slotted walls minimized direct recirculation in the test section.

6.1.3 Effect of Collective Pitch

Full scale design analyses show that under 1g conditions five degrees of rotor blade precone produce zero steady

flap bending moment at the blade root with $\theta_{.75}$ = 10° at design hover RPM. This is confirmed by the test results in Figure 6-13 which shows the variation of steady flap bending moment at 0.10R with collective pitch. In this case (Ω = 750 RPM) low collective produces the expected negative steady bending moment at 0.10R (due to centrifugal force acting on the pre-coned blade), and zero steady bending moment results with $\theta_{.75}$ = 10.3°.

6.1.4 Rotor Start-up and Shut-down

Figures 6-14, 6-15 and 6-16 show the effect of rotor start-up and shut-down through the one per rev frequency crossing on blade loads in hover. Figure 6-16 shows that the rate of change of RPM with time through the one per rev crossing were nearly the same. Maximum chord bending during start-up are 1.3 times chord bending during shut-down and are equivalent to the loads obtained by 3.5 degrees of hover cyclic at 825 RPM, as shown in Figure 6-2.

6.1.5 Low Forward Speed

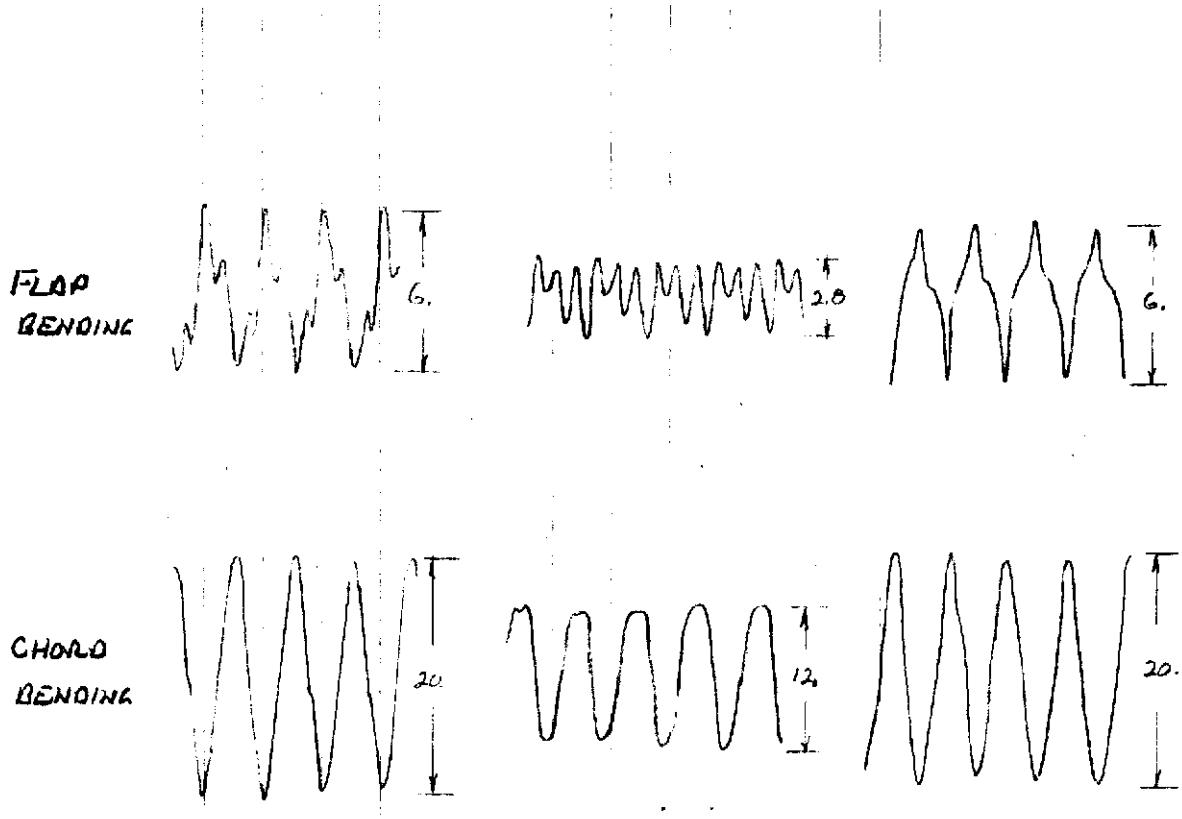
Testing was done OGE at low forward speed to determine whether blade-tip vortex interactions were present and, if so, how the blade loads were affected. Cyclic was varied at a tunnel q corresponding to a full-scale aircraft speed of 23 fps. The results in Figures 6-17 and 6-18 show a large decrease in blade loads as cyclic pitch is increased positively. The sensitivity of these loads to cyclic is in the same order of that in hover and the cyclic pitch value at which minimum load would occur (not reached in this run) had a notable shift to positive as expected.

6.1.6 Stall Flutter

Stall flutter inception is considered to occur at the break in the blade torsion amplitude versus collective pitch. Figure 6-19 shows the alternating torques which resulted from two rotor speeds. Although the load amplitudes are very low, inception is considered to have occurred at $\theta_{.75}$ = 12° for Ω = 750 and $\theta_{.75}$ = 11° for Ω = 830. Blade torsion wave traces are presented in Figure 6-20 for the Ω = 850 RPM case. The peak to peak amplitudes correspond to the circular data points on Figure 6-19.

It is noteworthy that even at the highest collective pitch setting tested (14.2°), the absolute value of alternating torsion moment is very low - about ± 1.1 lb in.

A measurement of rotor thrust in hover was obtained from the steady wing flap bending moment. The maximum thrust values (shown in Figure 6-21) produce a thrust coefficient (C_T--helicopter notation) of 0.01, and a corresponding value of C_T/σ = 0.118.



CEC Data Point	7	10	15
θ_2	$-1.5°$	$-.75°$	$+.5°$

NOTES:
- PEAK-TO-PEAK BENDING MOMENT LOADS ARE INDICATED
- VERTICAL LINES ARE 1/REV MARKER AT $\psi = 90°$

FIGURE 6-1. WAVE-FORM OF ALTERNATING BLADE LOADS FOR A SERIES OF CYCLIC PITCH ANGLES
- RIGHT BLADE .10R
$\theta_{.75} = 9.0°$
$\Omega = 750$ RPM
RUN 31
HOVER

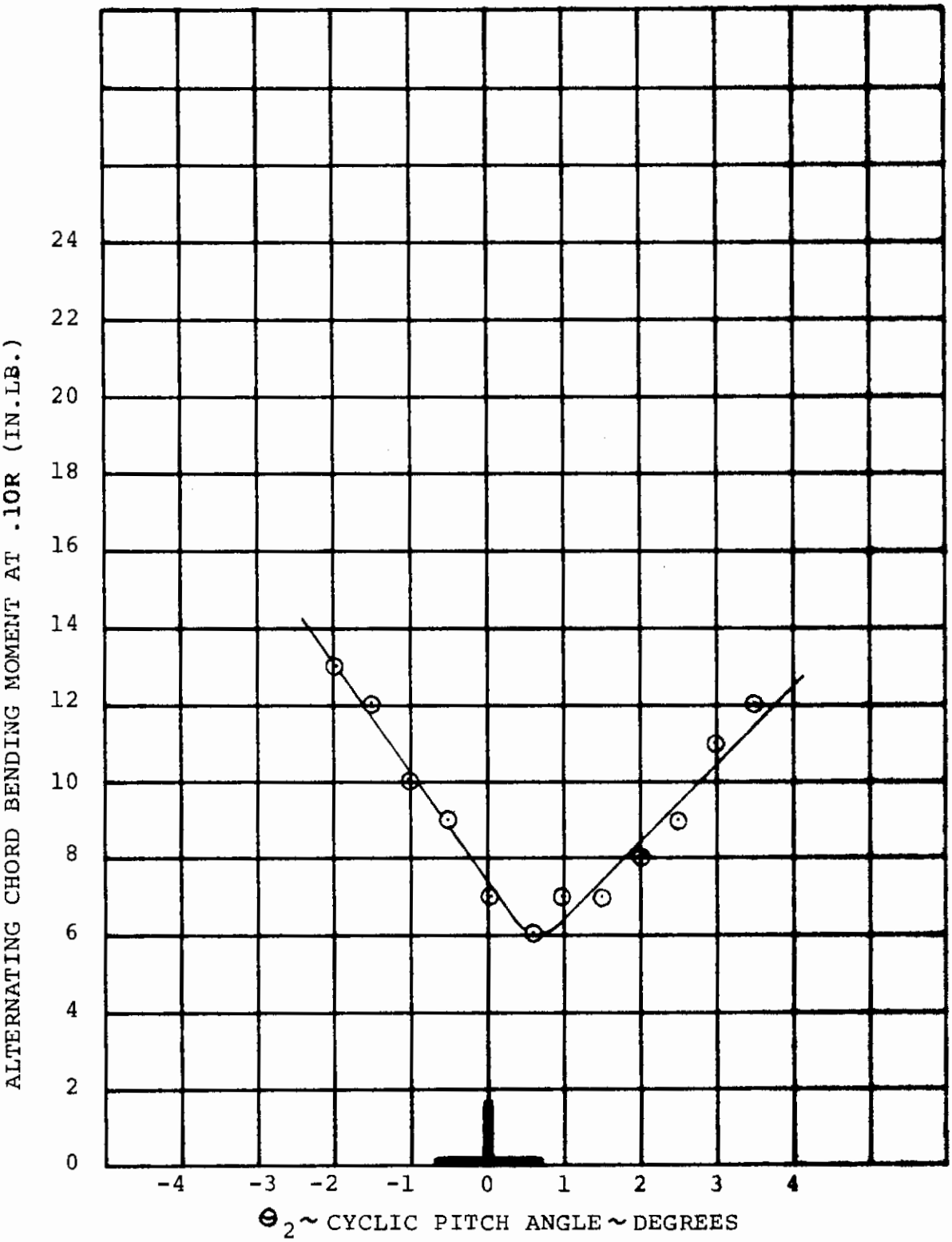


FIGURE 6-2. EFFECT OF CYCLIC PITCH ON RIGHT ROTOR ALTERNATING CHORD BENDING MOMENT IN HOVER IGE FOR i_N = 90 DEG., $\Theta_{.75}$ = 4.7 DEG., AND 825 RPM RUN 61(16)

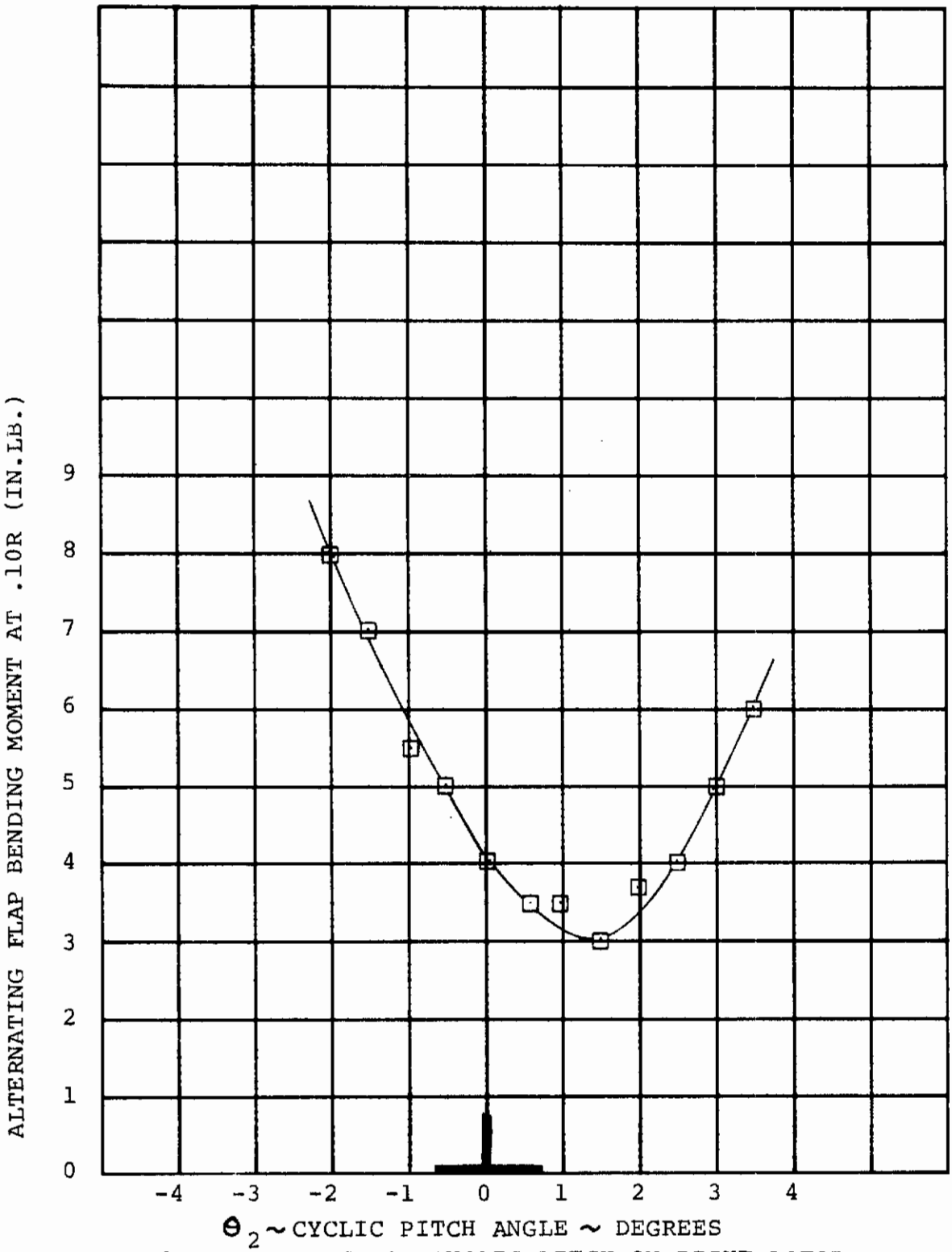


FIGURE 6-3. - EFFECT OF CYCLIC PITCH ON RIGHT ROTOR ALTERNATING FLAP BENDING MOMENT IN HOVER IGE FOR i_N = 90 DEG., $\Theta_{.75}$ = 4.7 DEG., AND 825 RPM. RUN 61(16)

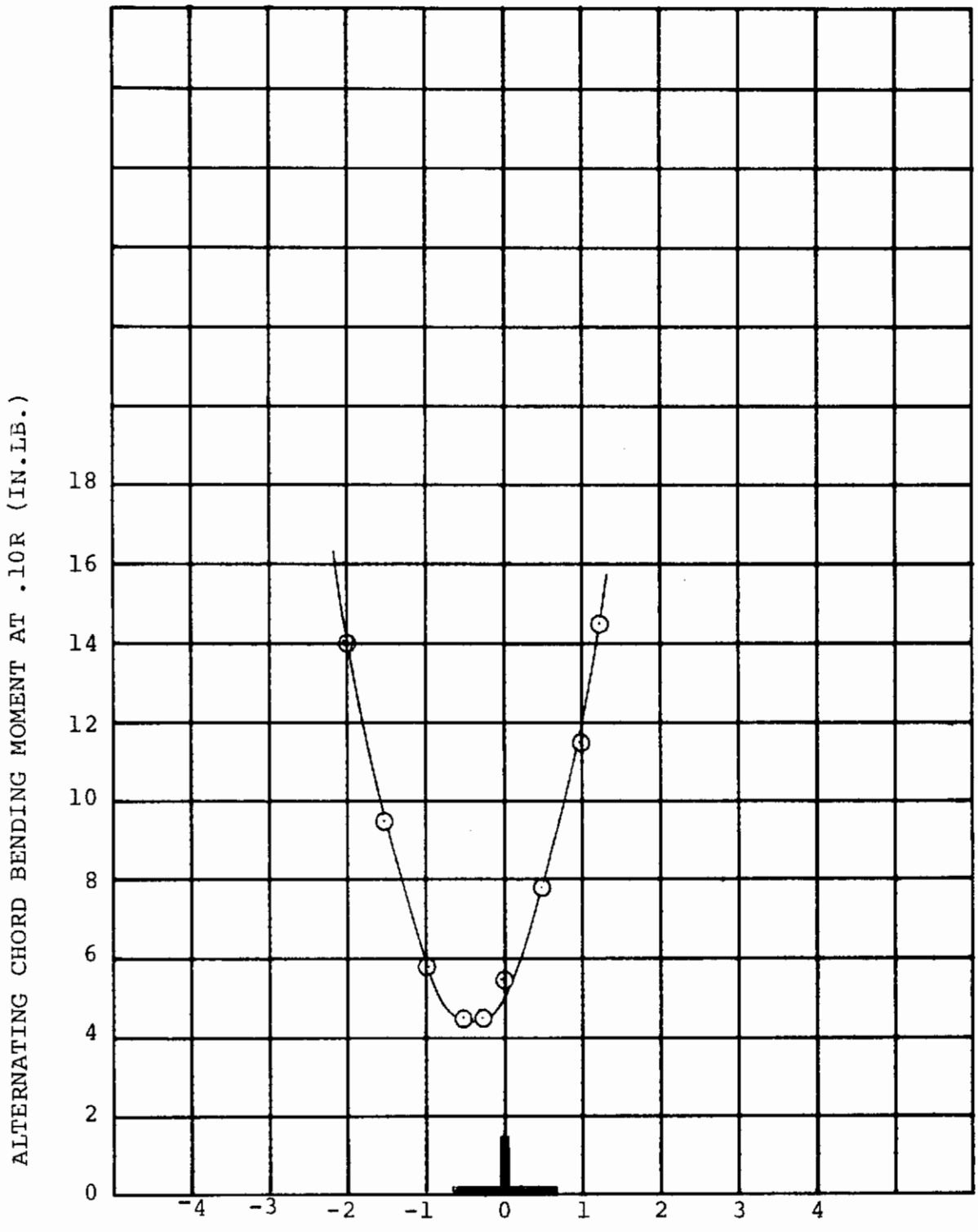


FIGURE 6-4. - EFFECT OF CYCLIC PITCH ON RIGHT ROTOR ALTERNATING CHORD BENDING MOMENT IN HOVER OGE FOR i_N = 90 DEG., $\theta_{.75}$ = 4 DEG, AND 750 RPM
RUN 23(7) - UNBALANCED ROTOR

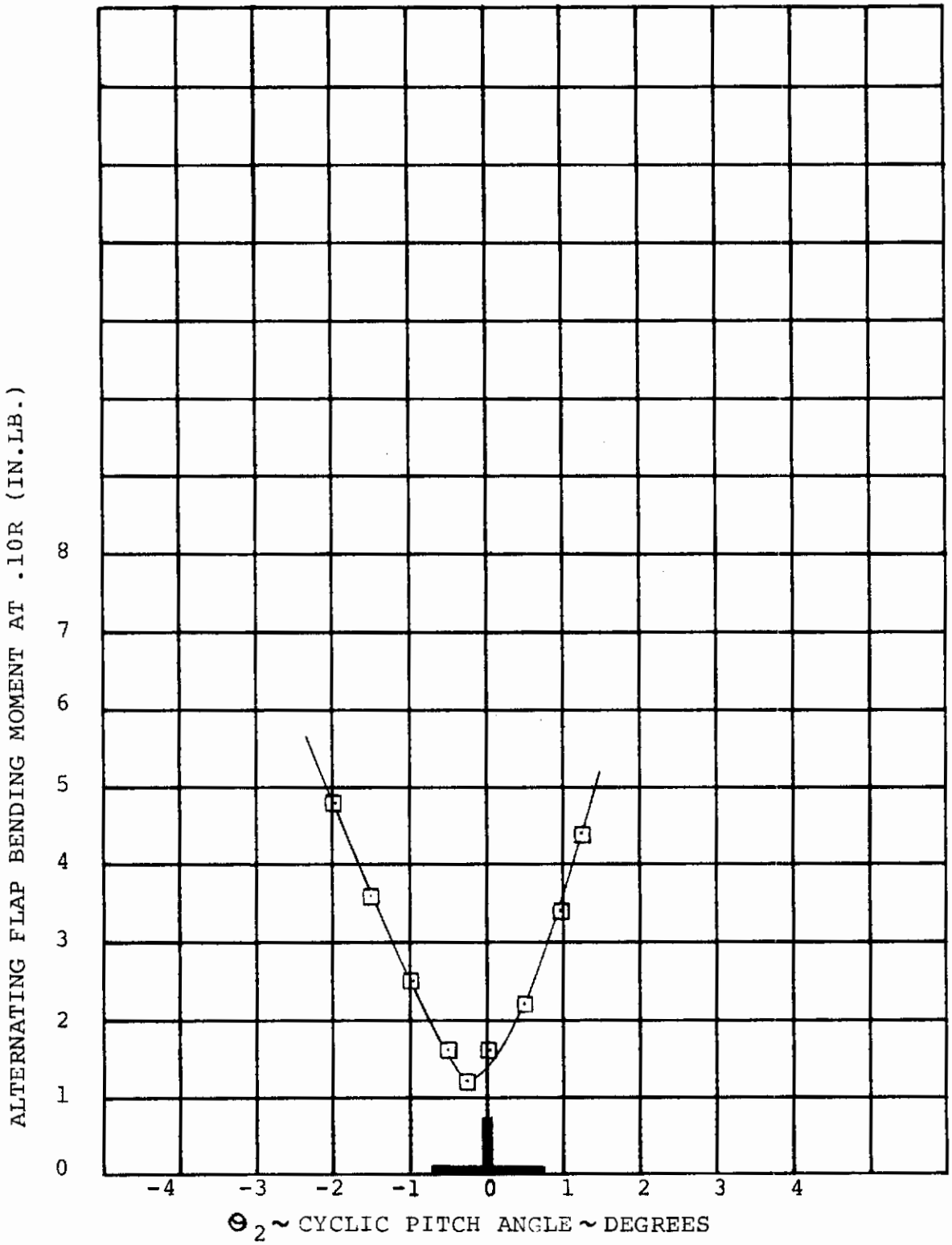


FIGURE 6-5. - EFFECT OF CYCLIC PITCH ON RIGHT ROTOR ALTERNATING FLAP BENDING MOMENT IN HOVER OGE FOR i_N = 90 DEG., $\Theta_{.75}$ = 4 DEG., AND 750 RPM.
RUN 23(7) UNBALANCED ROTOR

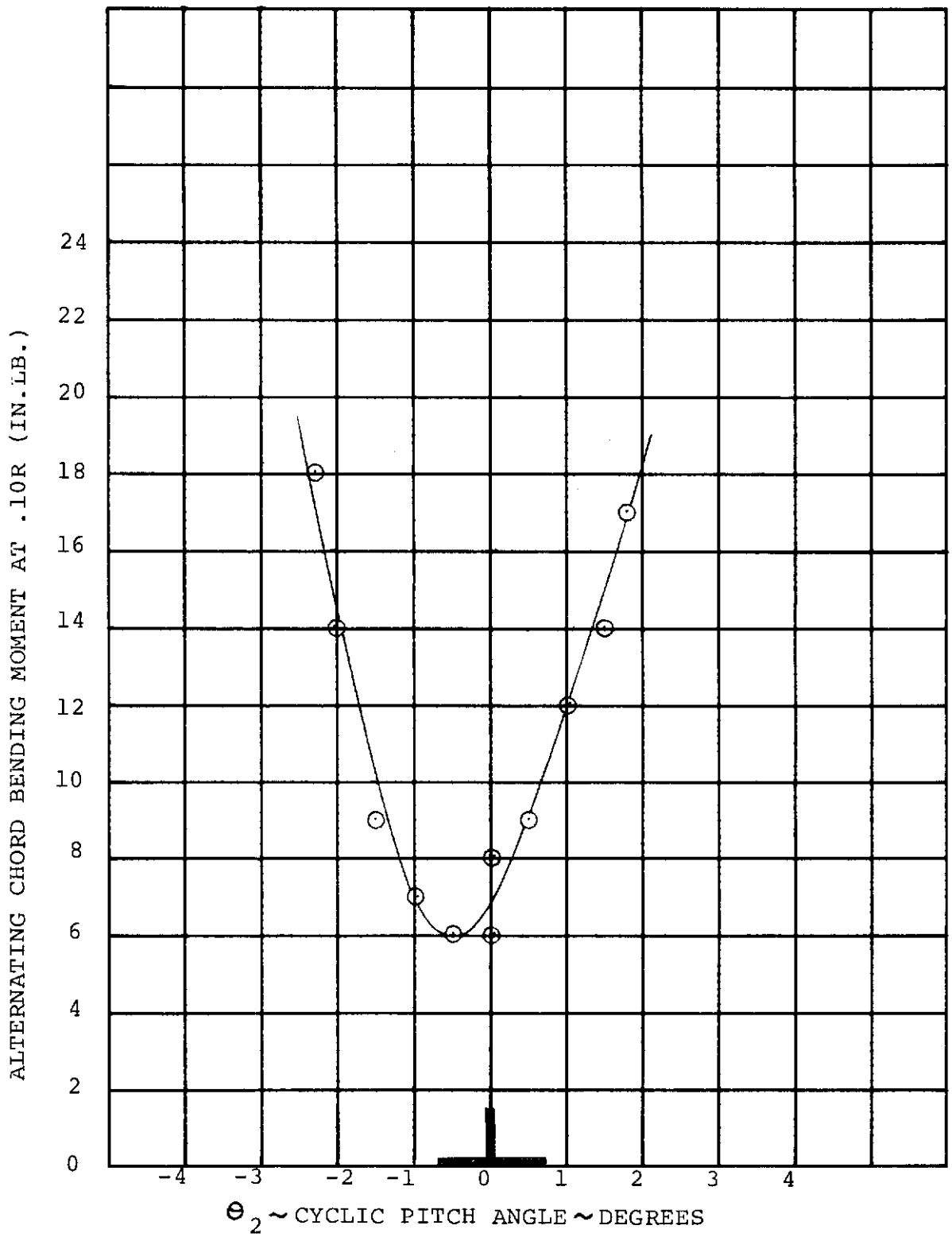


FIGURE 6-6. - EFFECT OF CYCLIC PITCH ON RIGHT ROTOR ALTERNATING CHORD BENDING MOMENT IN HOVER IGE FOR i_N = 90 DEG., $\Theta_{.75}$ = 10.6 DEG., AND 825 RPM.
RUN 61(3)

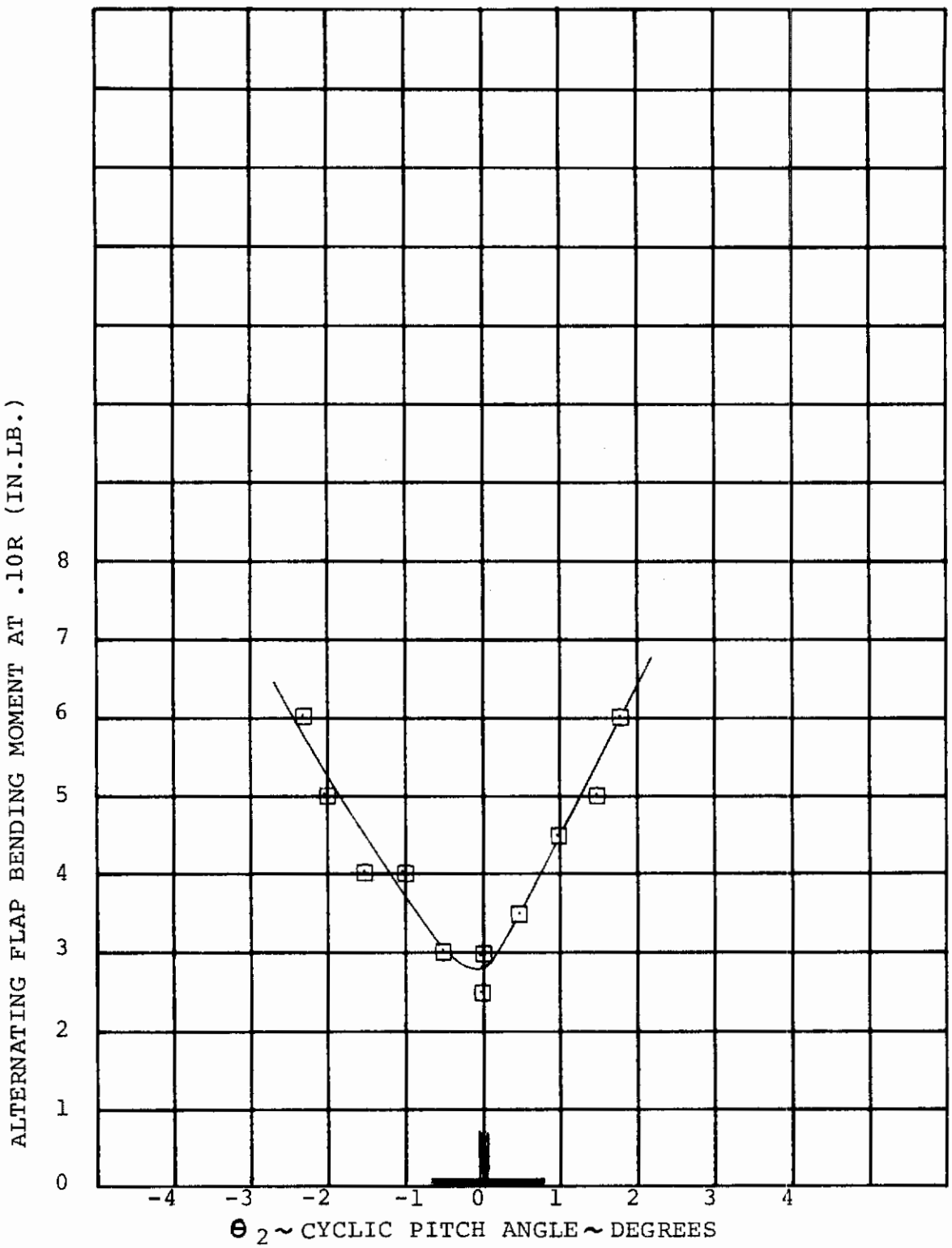


FIGURE 6-7. - EFFECT OF CYCLIC PITCH ON RIGHT ROTOR ALTERNATING FLAP BENDING MOMENT IN HOVER IGE FOR i_N = 90 DEG., $\theta_{.75}$ = 10.6 DEG., AND 825 RPM.
RUN 61(3)

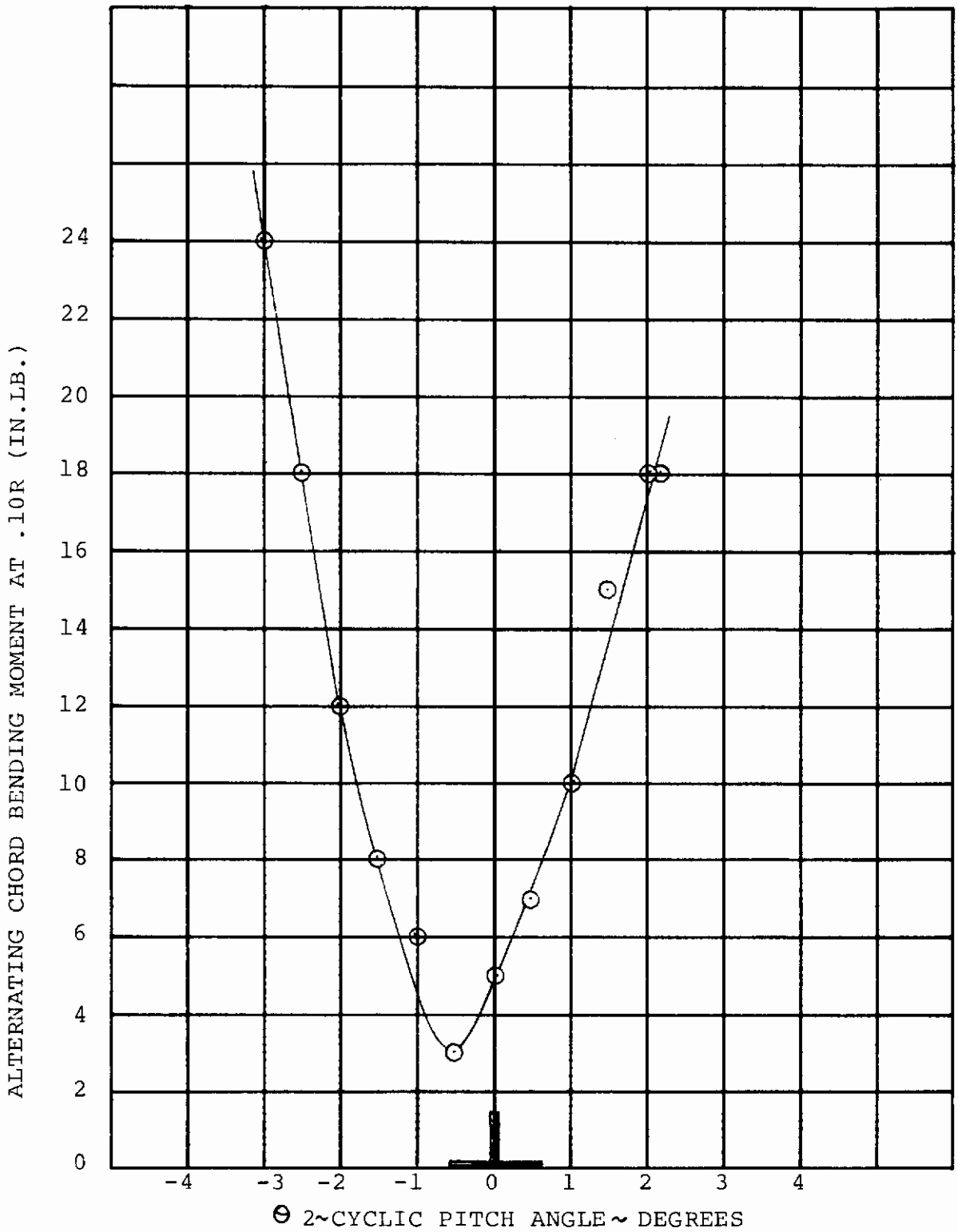


FIGURE 6-8. EFFECT OF CYCLIC PITCH ON RIGHT ROTOR ALTERNATING CHORD BENDING MOMENT IN HOVER OGE FOR i_N=90 DEG., $\Theta_{.75}$= 10.1 DEG. AND 825 RPM.
RUN 65(12)

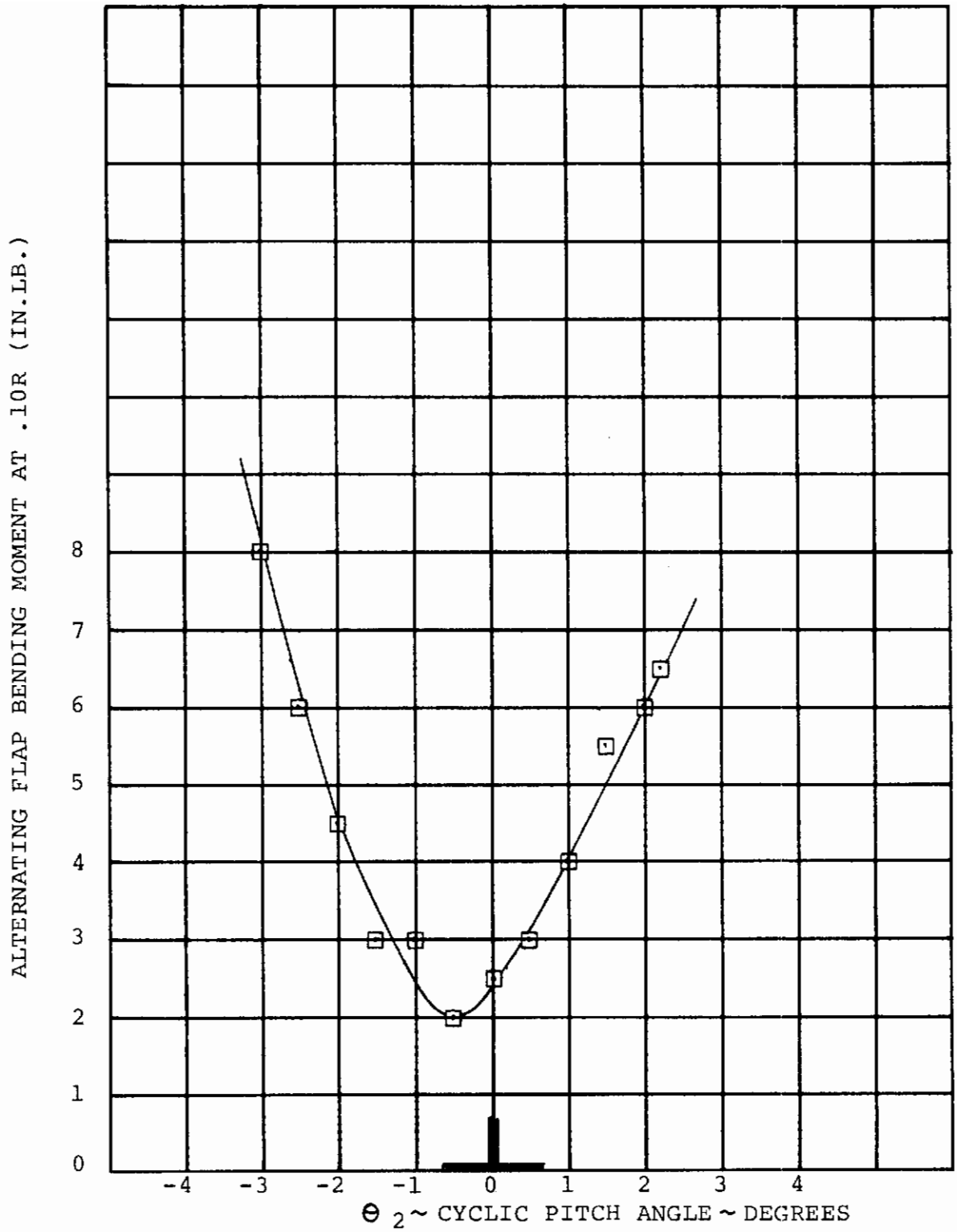


FIGURE 6-9. - EFFECT OF CYCLIC PITCH ON RIGHT ROTOR ALTERNATING FLAP BENDING MOMENT IN HOVER OGE FOR i_N = 90 DEG., $\theta_{.75}$ = 10.1 DEG., AND 825 RPM. RUN 65(12)

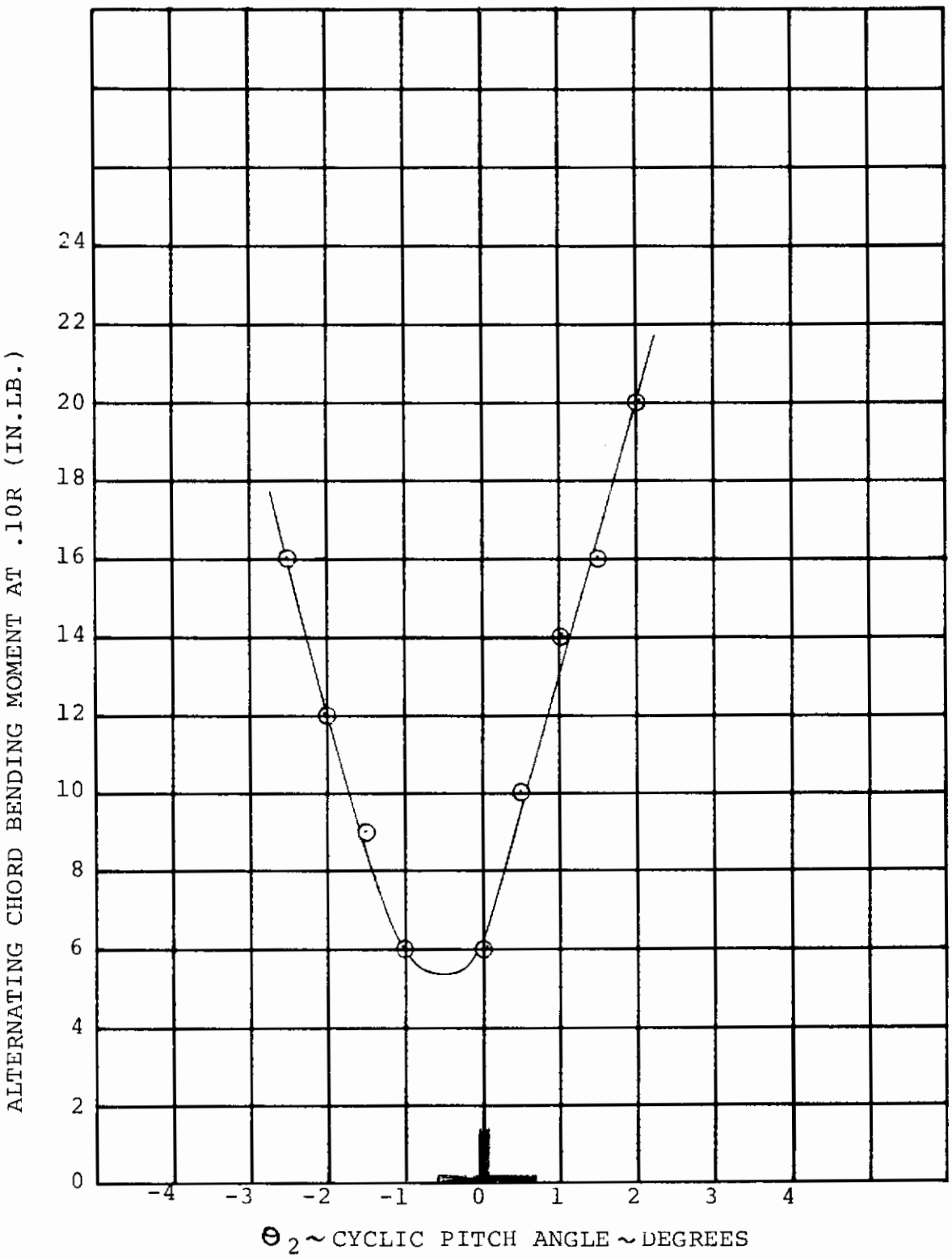


FIGURE 6-10. EFFECT OF CYCLIC PITCH ON RIGHT ROTOR ALTERNATING CHORD BENDING MOMENT IN HOVER IGE FOR i_N = 90 DEG., $\Theta_{.75}$ = 11 DEG., AND 825 RPM.
RUN 62(17)

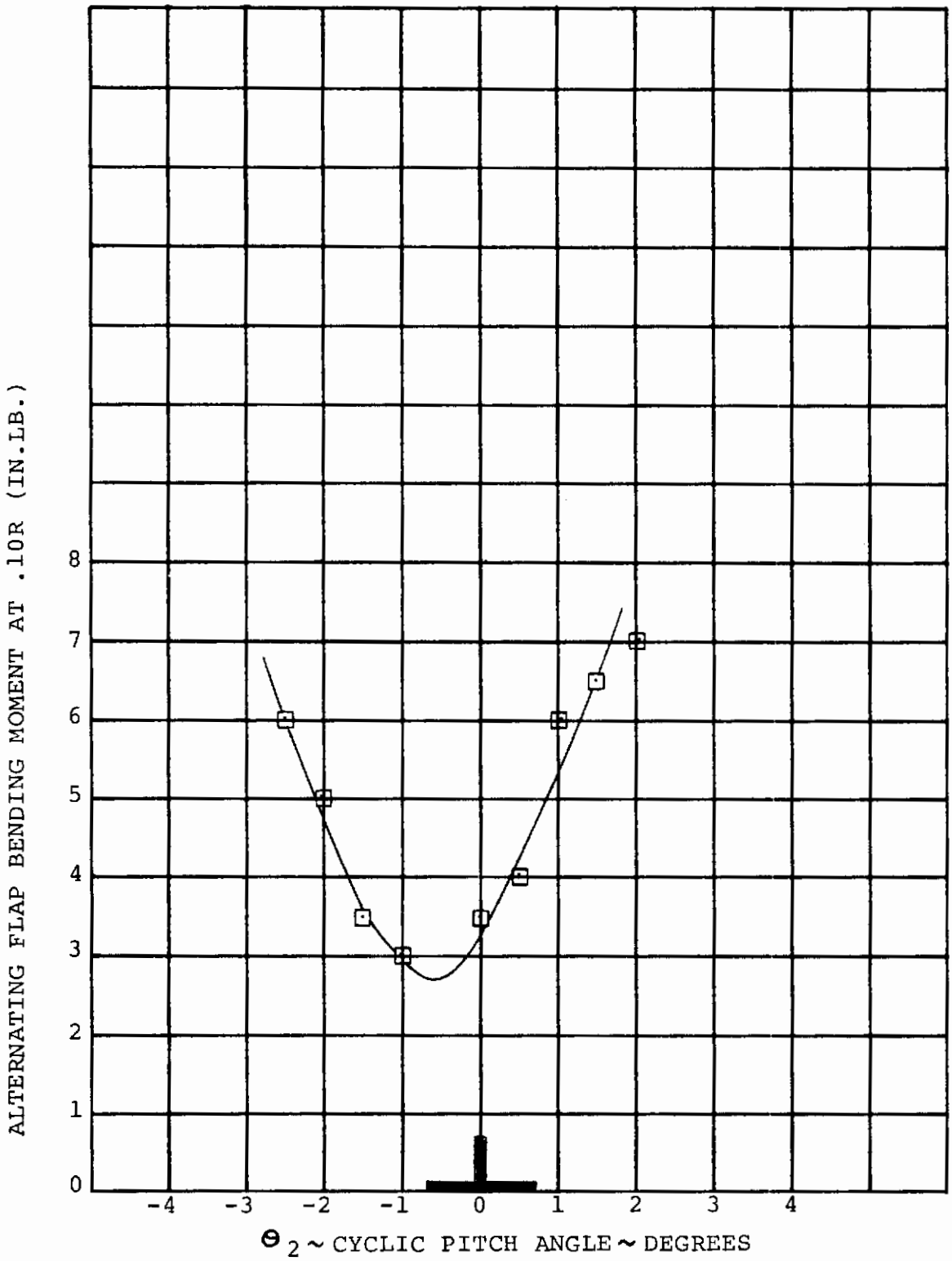


FIGURE 6-11. EFFECT OF CYCLIC PITCH ON RIGHT ROTOR ALTERNATING FLAP BENDING MOMENT IN HOVER IGE FOR i_N = 90 DEG., $\Theta_{.75}$ = 11 DEG., AND 825 RPM.
RUN 62(17)

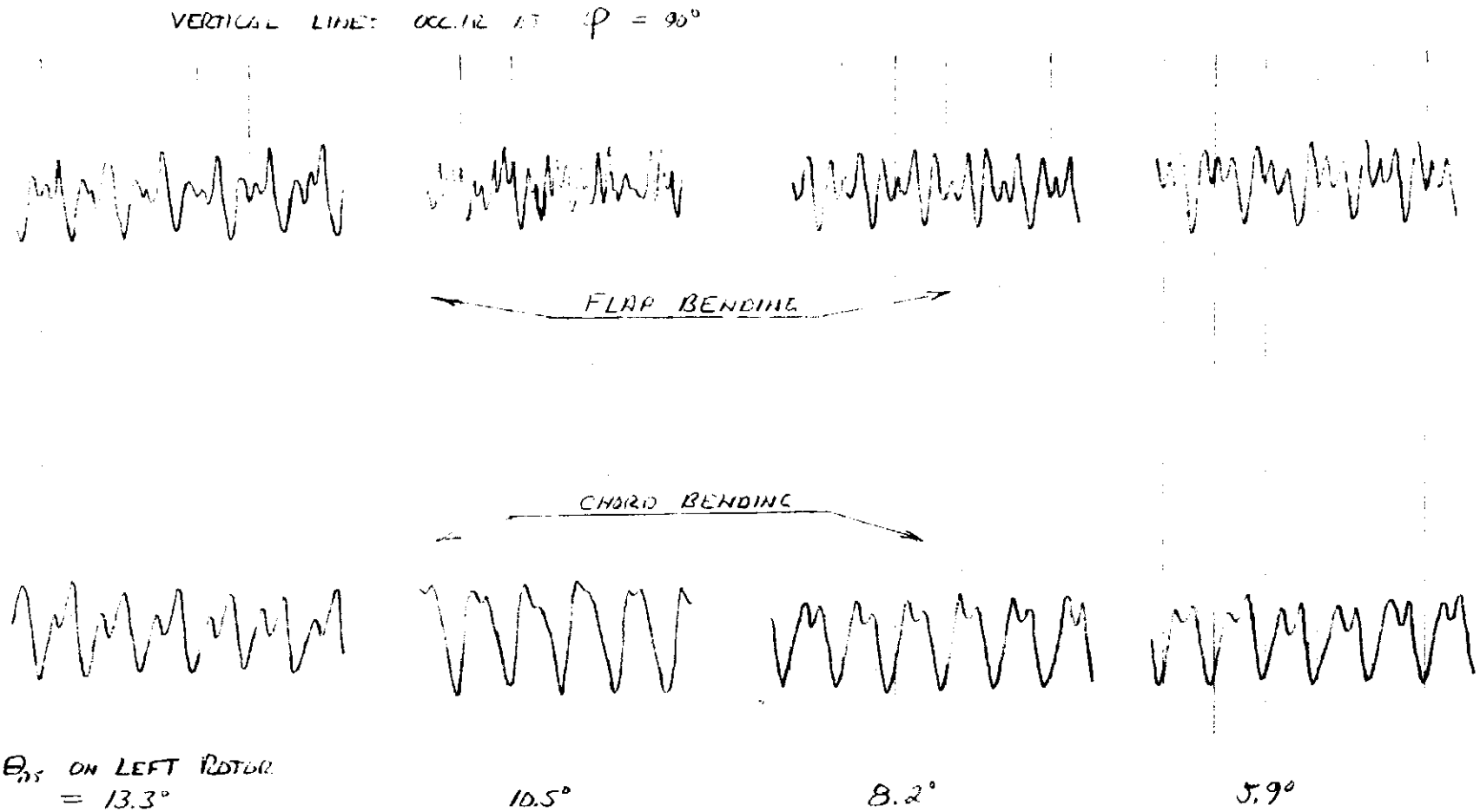


FIGURE 6-12. RIGHT BLADE RESPONSE TO DIFFERENTIAL COLLECTIVE
$\theta_{.75} = 8.1°$ ON RIGHT ROTOR
$\theta_3 = -.5°$ BOTH SIDES
RUN 26 (8-11)
HOVER

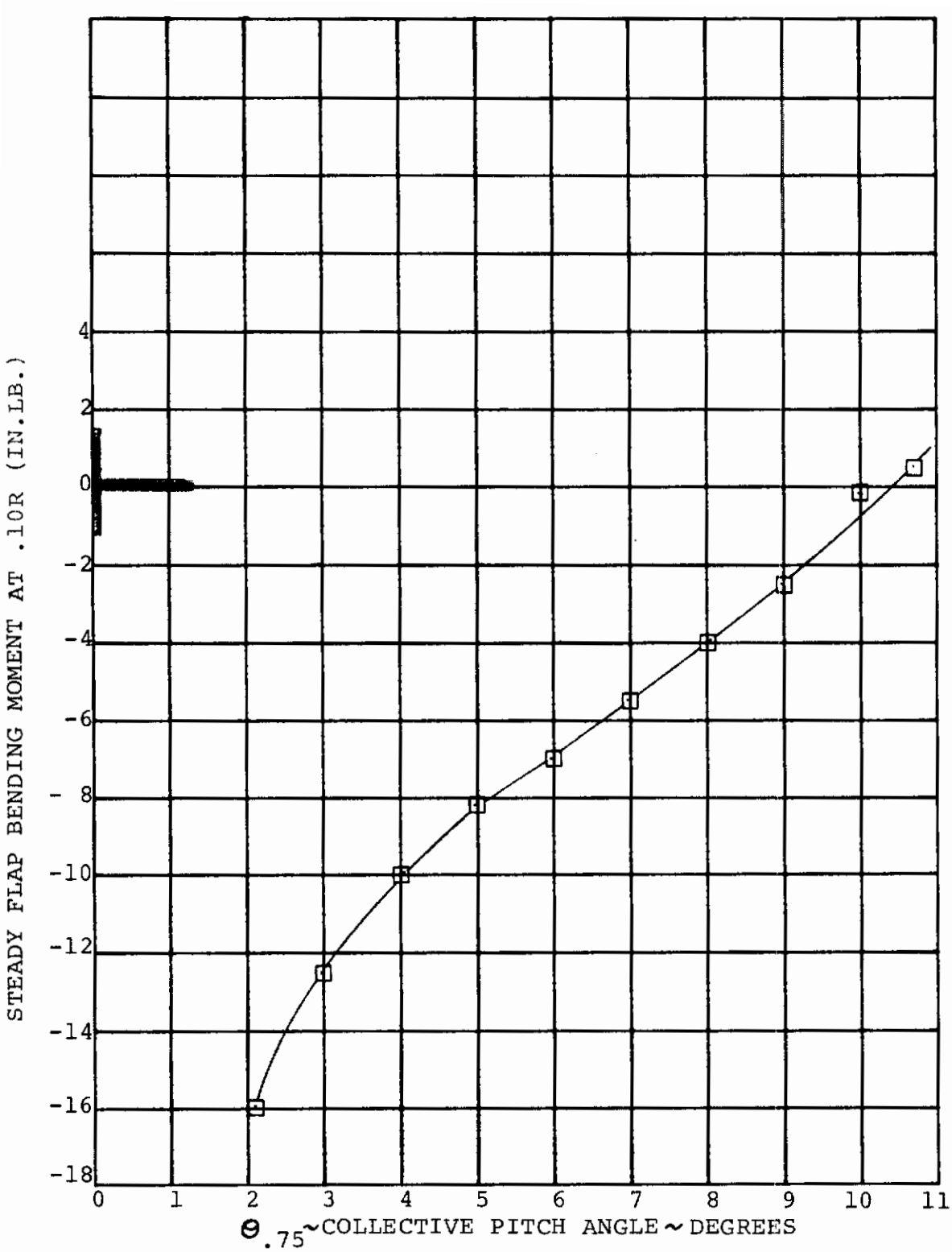


FIGURE 6-13. EFFECT OF COLLECTIVE PITCH ON RIGHT ROTOR STEADY FLAP BENDING MOMENT IN HOVER OGE FOR 90 DEG., $\theta_{2=}$ -.75 DEG., AND 750 RPM.
RUN 32(4)

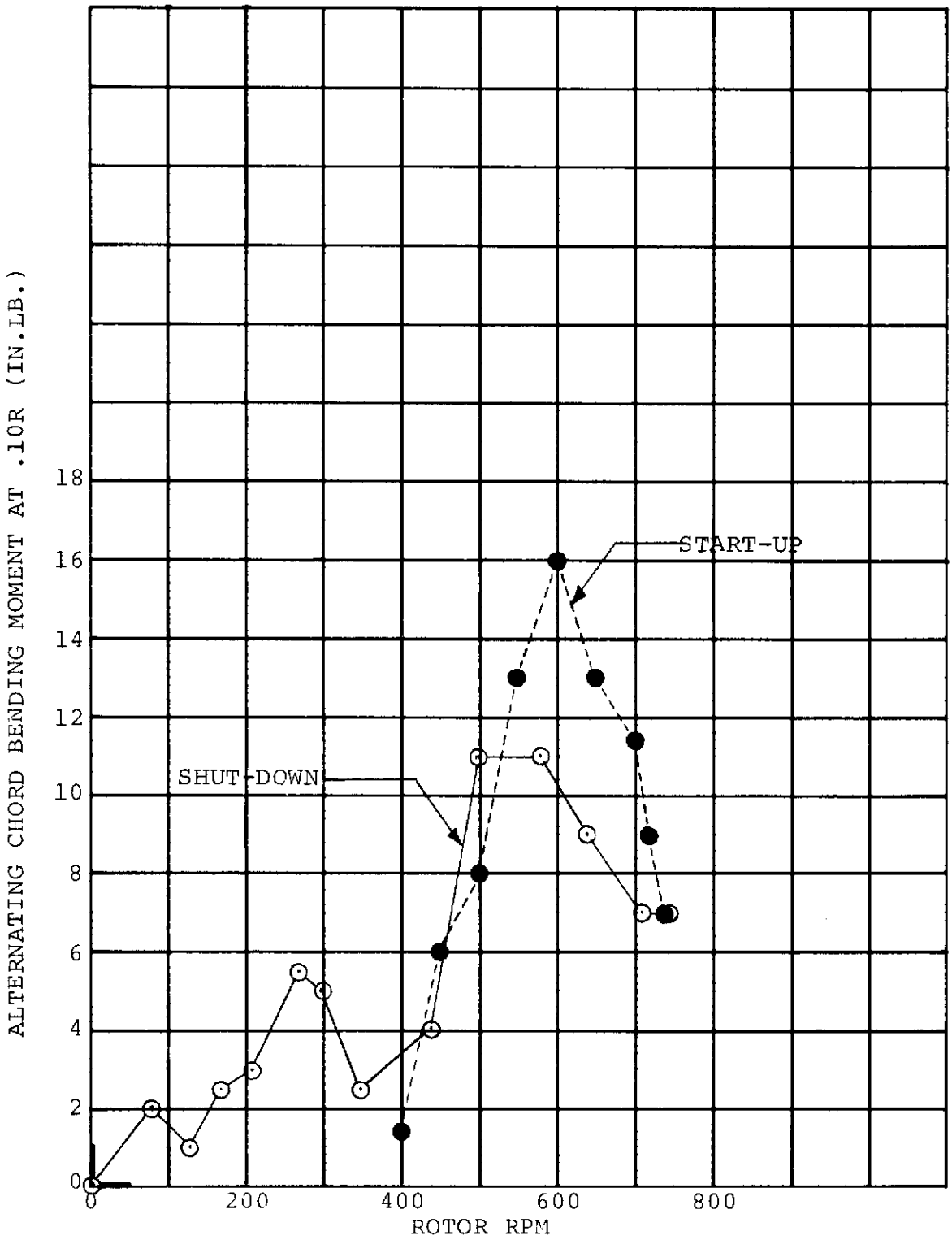


FIGURE 6-14. - EFFECT OF ROTOR START-UP AND SHUT-DOWN ON RIGHT HAND ALTERNATING CHORD BENDING MOMENT IN HOVER OGE FOR i_N= 90 DEG., $\Theta_{.75}$=3.2 DEG., AND Θ_2= 0 DEG. RUN 24

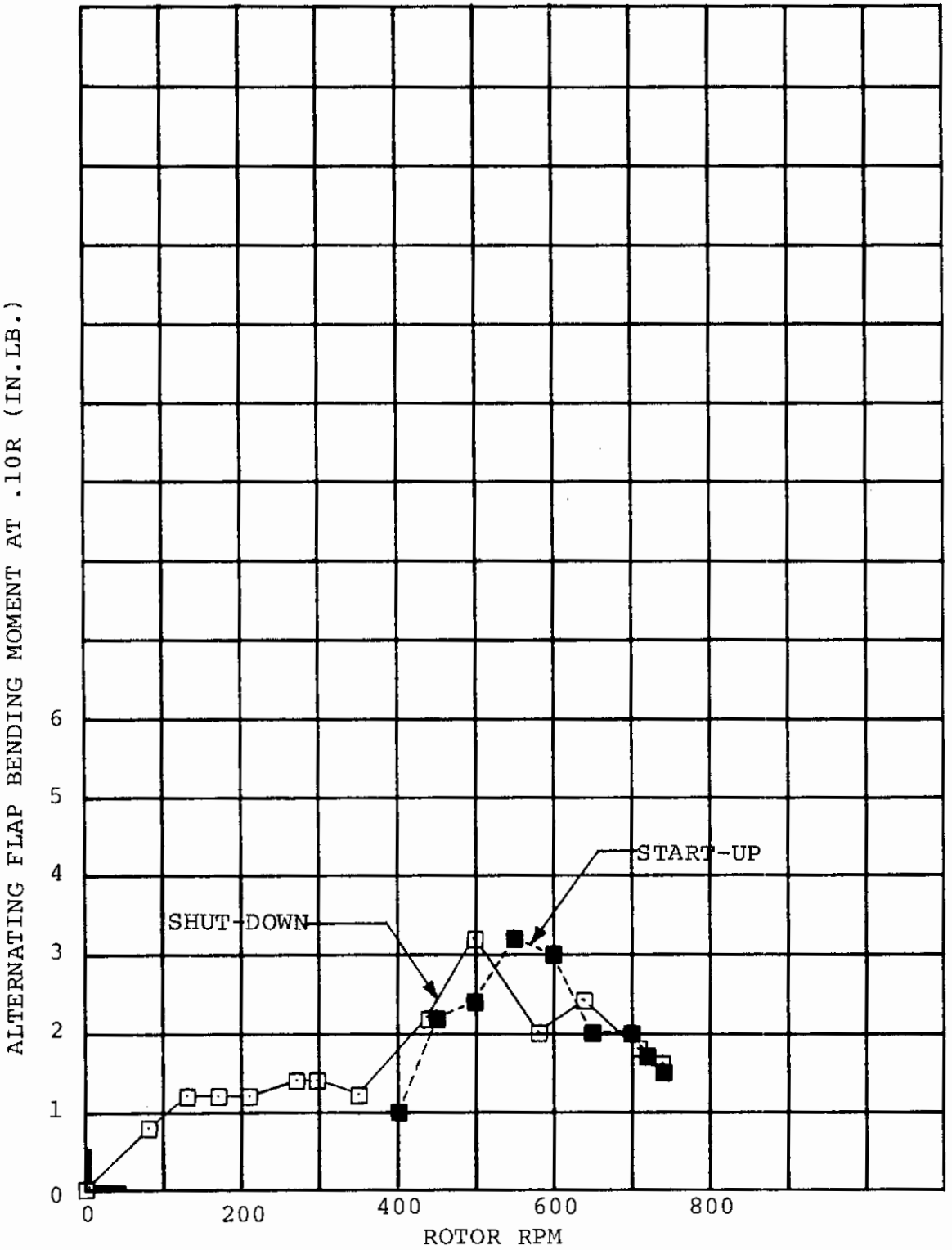


FIGURE 6-15. - EFFECT OF ROTOR START-UP AND SHUT-DOWN ON RIGHT HAND ALTERNATING FLAP BENDING MOMENT IN HOVER OGE FOR i_N= 90 DEG., $\Theta_{.75}$= 3.2 DEG., AND Θ_2= 0 DEG. RUN 24

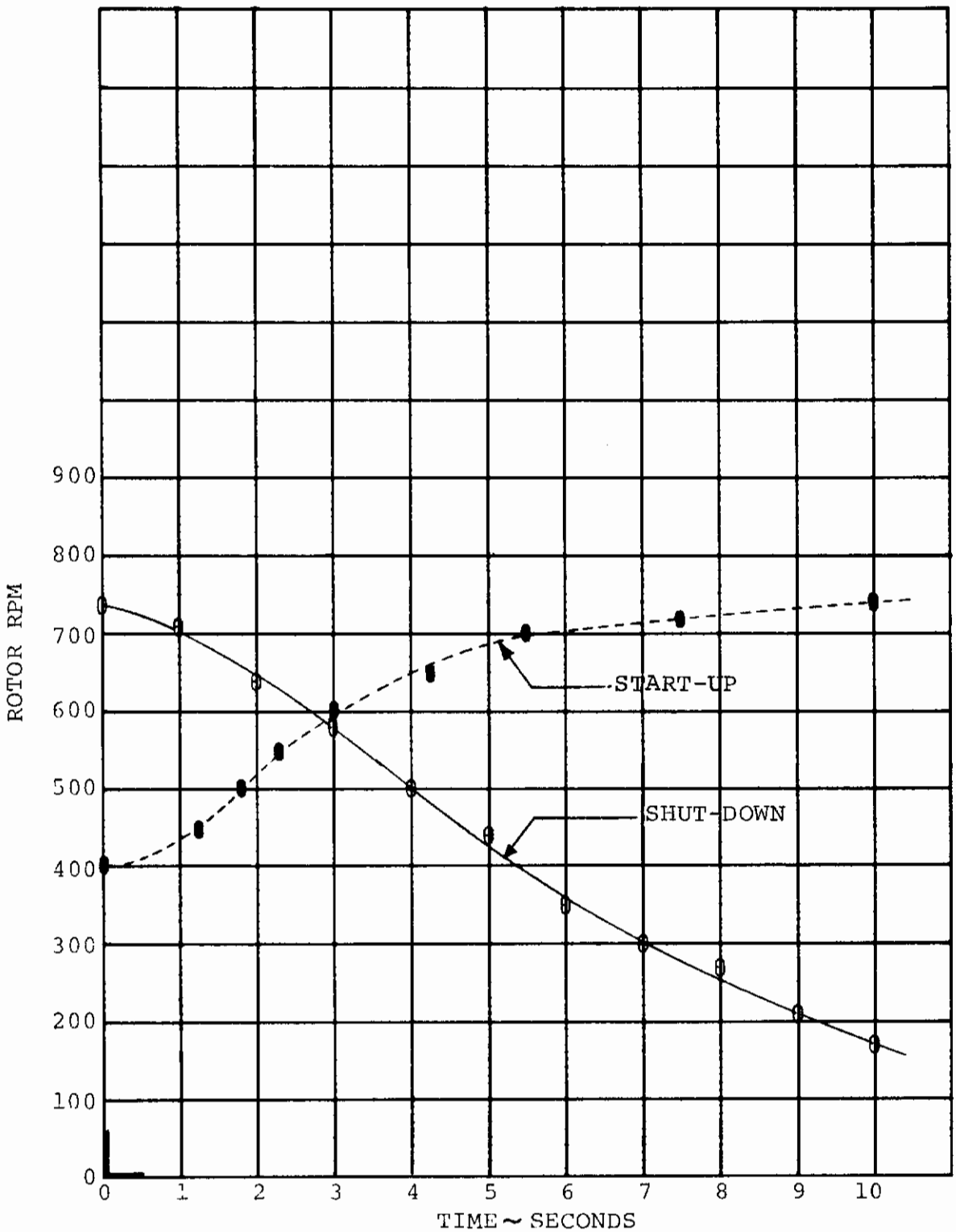


FIGURE 6-16.- ROTOR SPEED DURING START-UP AND SHUT-DOWN IN HOVER OGE FOR i_N= 90 DEG., $\theta_{.75}$ = 3.2 DEG., AND θ_2 = 0 DEG. RUN 24

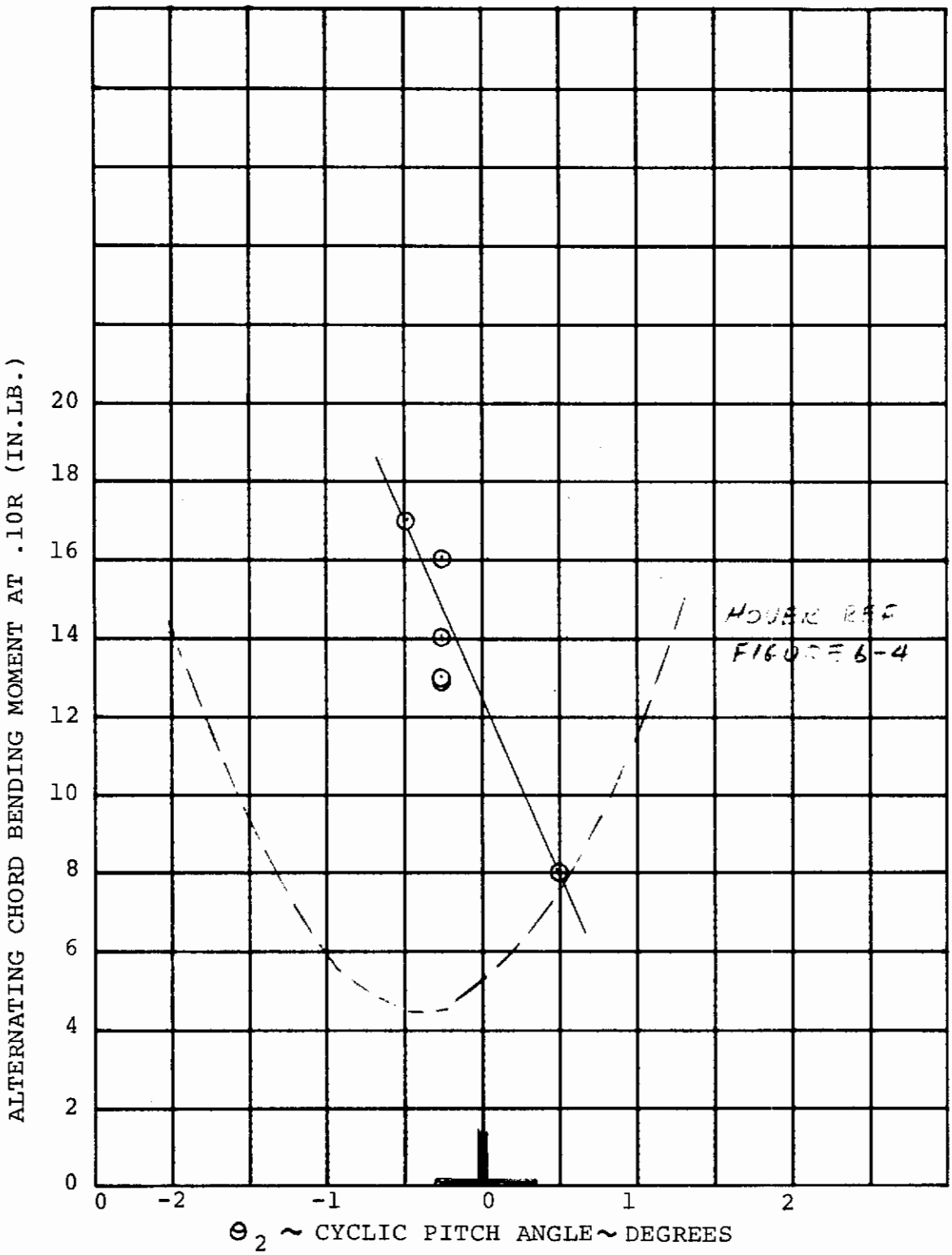


FIGURE 6-17. - EFFECT OF CYCLIC PITCH ON RIGHT ROTOR ALTERNATING CHORD BENDING MOMENT IN HOVER OGE FOR i_N = 90 DEG., $\Theta_{.75}$ = 4 DEG., 750 RPM, AND q = 0.06 PSF.
RUN 33(2-6)

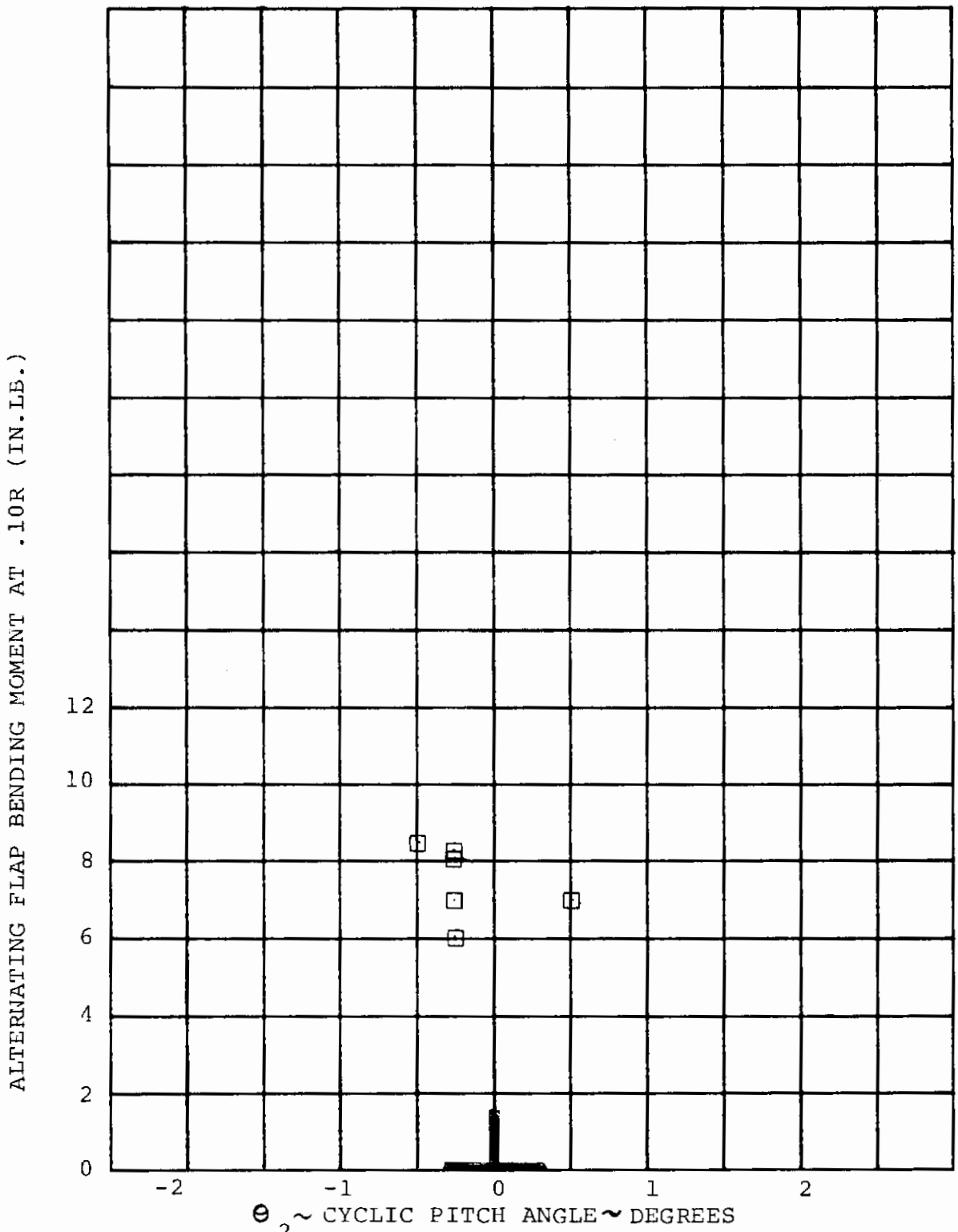


FIGURE 6-18. - EFFECT OF CYCLIC PITCH ON RIGHT ROTOR ALTERNATING FLAP BENDING MOMENT IN HOVER OGE FOR i_N = 90 DEG., $\theta_{.75}$ = 4 DEG., 750 RPM, & q = 0.06 PSF.

RUN 33(2-6)

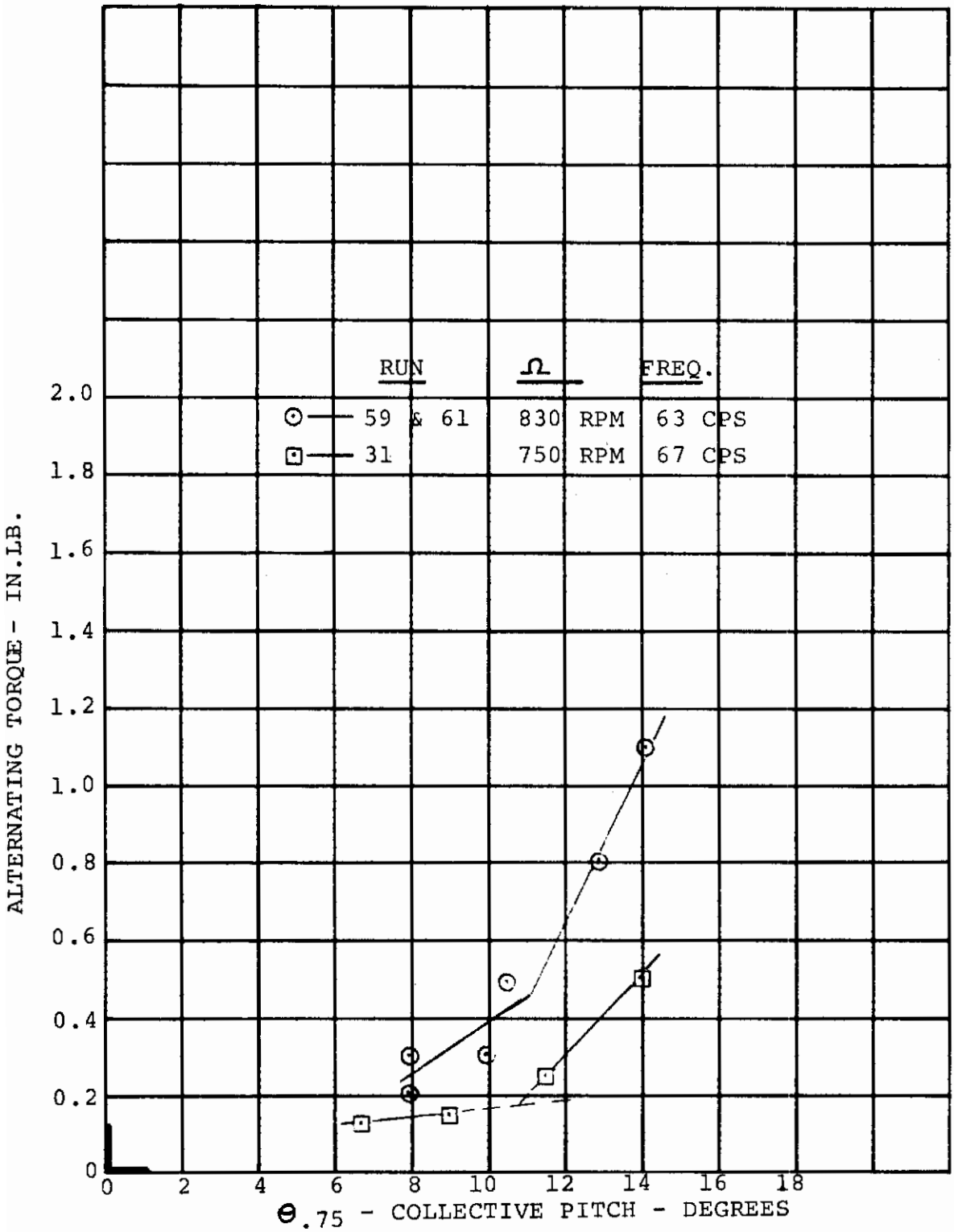


FIGURE 6-19.- EFFECT OF COLLECTIVE PITCH ON BLADE TORSION RESPONSE IN HOVER FOR i_N =90 DEG.,& θ_2=0 DEG.

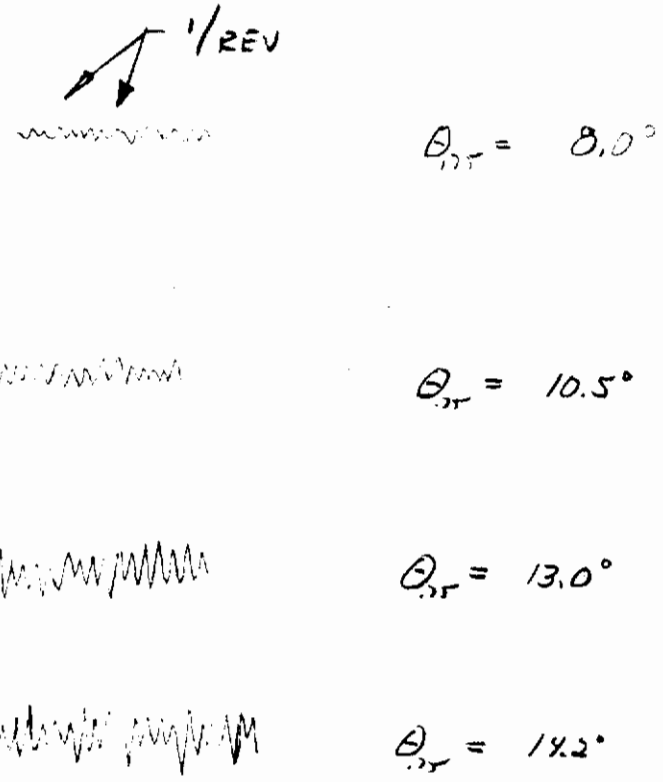


FIGURE 6-20. EFFECT OF COLLECTIVE PITCH ON BLADE TORSION AMPLITUDE
-RIGHT BLADE .15R
Ω = 830 RPM
RUN LD-61
HOVER

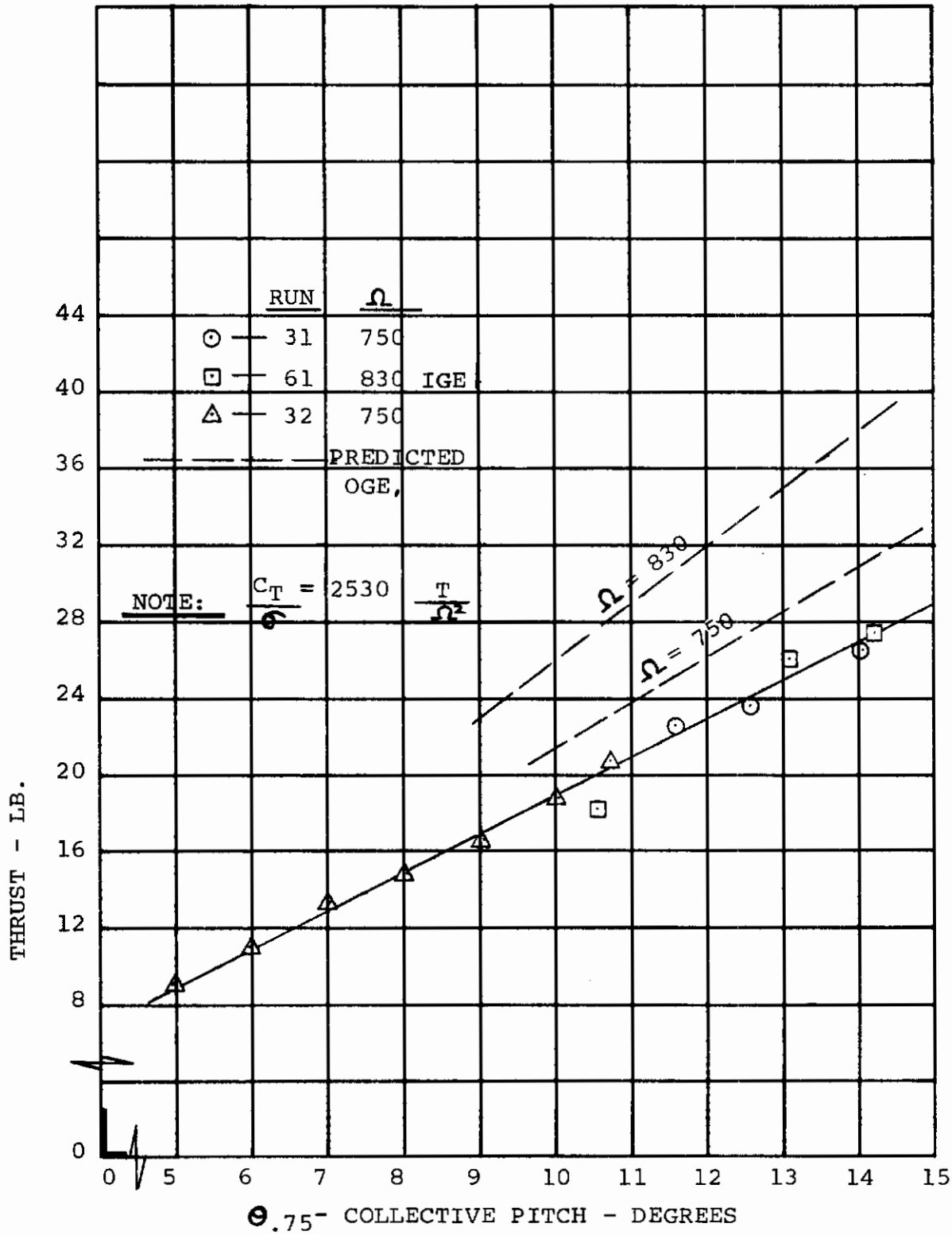


FIGURE 6-21. ROTOR THRUST — — DETERMINED FROM WING STEADY FLAP BENDING MOMENT - RIGHT SIDE - HOVER

6.2 TRANSITION BLADE LOADS

Blade loads data were taken in transition at two nacelle tilt angles, 60 degrees and 40 degrees, to determine the effect of cyclic pitch, dynamic pressure, collective pitch and model pitch and yaw. Test results show that for each nacelle tilt, collective pitch and dynamic pressure, there is a value of positive cyclic pitch at which alternating chord bending loads are minimized. Decreasing the cyclic pitch angle increases the flap bending moments. For a constant nacelle tilt and collective pitch, increased dynamic pressure increased the minimum value of alternating chord bending. Increased collective pitch decreased the minimum value of chord bending. Model pitch and model yaw caused large changes in alternating chord bending with little effect on alternating flap bending.

6.2.1 Effect of Cyclic Pitch

The blade alternating flap and chord bending load data for nacelle tilt angles of 60° and 40° are presented in Figures 6-22 through 6-31 for cyclic pitch variations. Alternating torsion loads remained small throughout this series of tests. The effect of increasing dynamic pressure is shown in Figures 6-22, 6-24 and 6-26 to increase the minimum alternating chord bending load (cyclic bucket) for $i_N = 60^{\circ}$. Note, however, that in Figure 6-22 for .75 = 9.8° (Run 39), higher alternating loads occurred at q = 0.2 psf than at other q values. The loads at the .2q value are suspected to be caused by the recirculation of the tip vortices with the rotor blades. This has also been found in other tests on both tilt rotor and helicopter rotor models. At this point the blade alternating chord load was composed primarily of 1 per rev response whereas 2 per rev was prominent at other points for this run. Figure 6-22 also shows that with increasing dynamic pressure, the value of cyclic pitch at which the minimum blade chord bending loads occur increases. These trends are summarized in Figure 6-28 which also shows that increasing collective pitch decreases the minimum (bucket) value of chord bending load.

The flap bending load is shown in Figure 6-23 to increase with increasing dynamic pressure. An apparent "bucket" also exists for flap bending alternating loads, the trend of which is similar to that for chord bending but it occurs at a higher value of cyclic than that for chord bending and was obtained during the test for q = 0, and 0.5 psf only.

Figures 6-29, 6-30 and 6-31 show the trend of blade alternating loads with cyclic for $i_N = 40°$ which are similar to those for $i_N = 60°$.

6.2.2 Effect of Model Yaw

Figures 6-32 through 6-37 show the effect of model yaw on blade loads for 60 and 40 degrees nacelle tilt. Chord bending loads increased as the model was yawed to the right from zero and decreased as the model was yawed to the left from zero. When the right rotor is yawed to the left, the angle of attack on the advancing side of the disc is decreased, thus decreasing flapping and the resulting blade loads. Flapwise loads showed the same result but the sensitivity of alternating flap bending to changes in model yaw is much less than chord bending sensitivity due to the large amount of flapwise aerodynamic damping.

6.2.3 Effect of Model Pitch

Figures 6-38 through 6-41 show the effect of model pitch on blade loads in transition at 60 and 40 degrees nacelle tilt. At both nacelle tilts, alternating chord bending decreased and flap bending increased as the model was pitched nose up. Increasing model pitch has the same effect as increasing the cyclic pitch angle on the advancing side of the disc and the results are the same as those shown in Section 6.2.1. If model pitch had been increased more, the chord bending loads would have reached a minimum and then increased as they did in Section 6.2.1 when cyclic was varied.

6.2.4 Stall Flutter - Transition

In the transition regime for $i_N = 60°$, blade torsion wave traces showed that the amplitude at its torsional frequency increases with increasing dynamic pressure. This is shown in Figure 6-42. Effects of collective and cyclic pitch are involved, however, and are not isolated from these data. Blade torsion wave shapes for two dynamic pressure conditions in Figure 6-42 are shown in Figure 6-43. Note the input at $\psi = 90^{\circ}$ (advancing side) which induces the torsional frequency! The blade torsional frequency response was also sensitive to the combined influence of fuselage pitch and yaw displacements as shown in Figure 6-44 for $i_N = 40^{\circ}$.

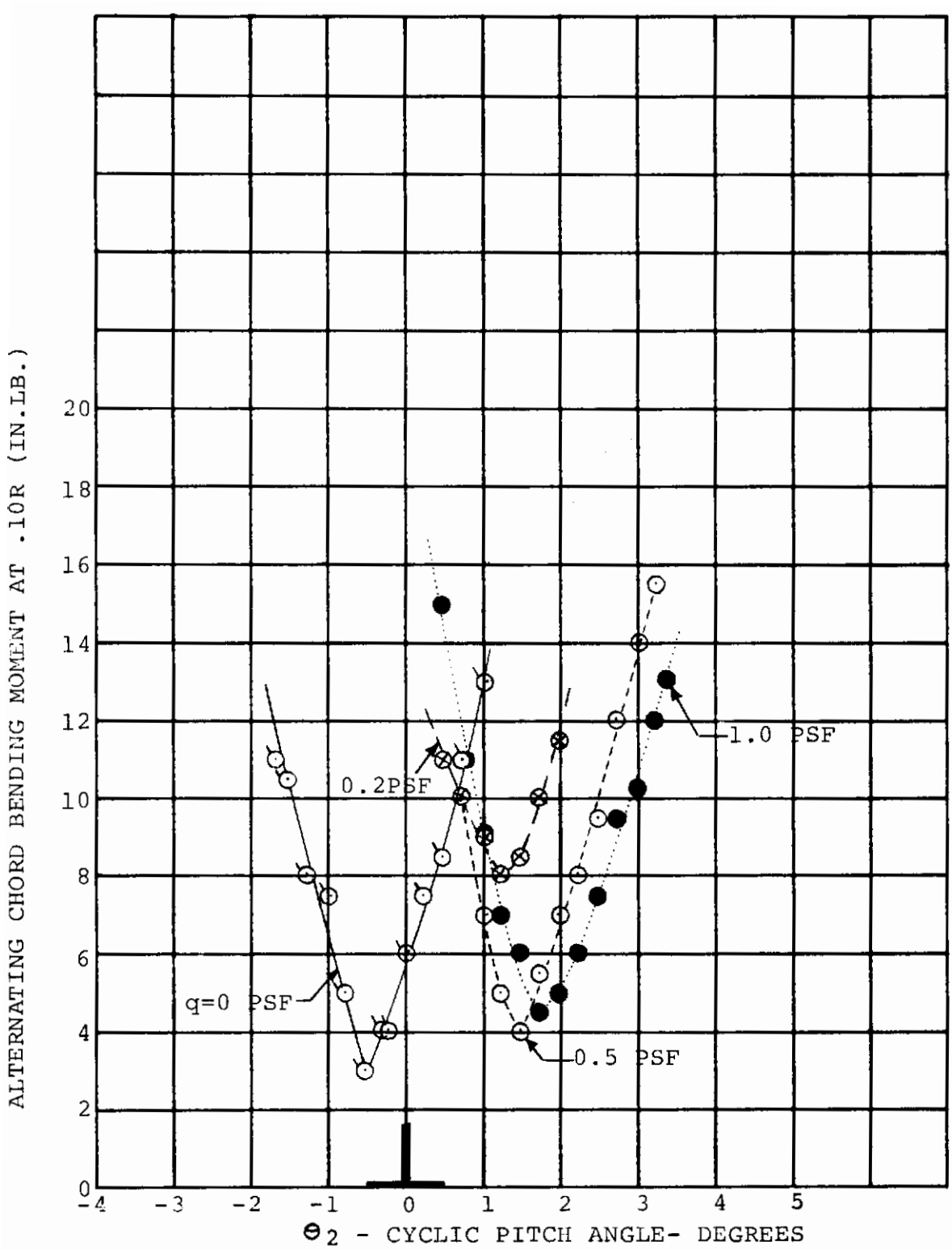


FIGURE 6-22 EFFECT OF DYNAMIC PRESSURE ON RIGHT ROTOR ALTERNATING CHORD BENDING MOMENT IN TRANSITION FOR i_N = 60 DEG., $\Theta_{.75}$ = 9.8 DEG., AND 790 RPM.
RUN 39(6,8,9,12)

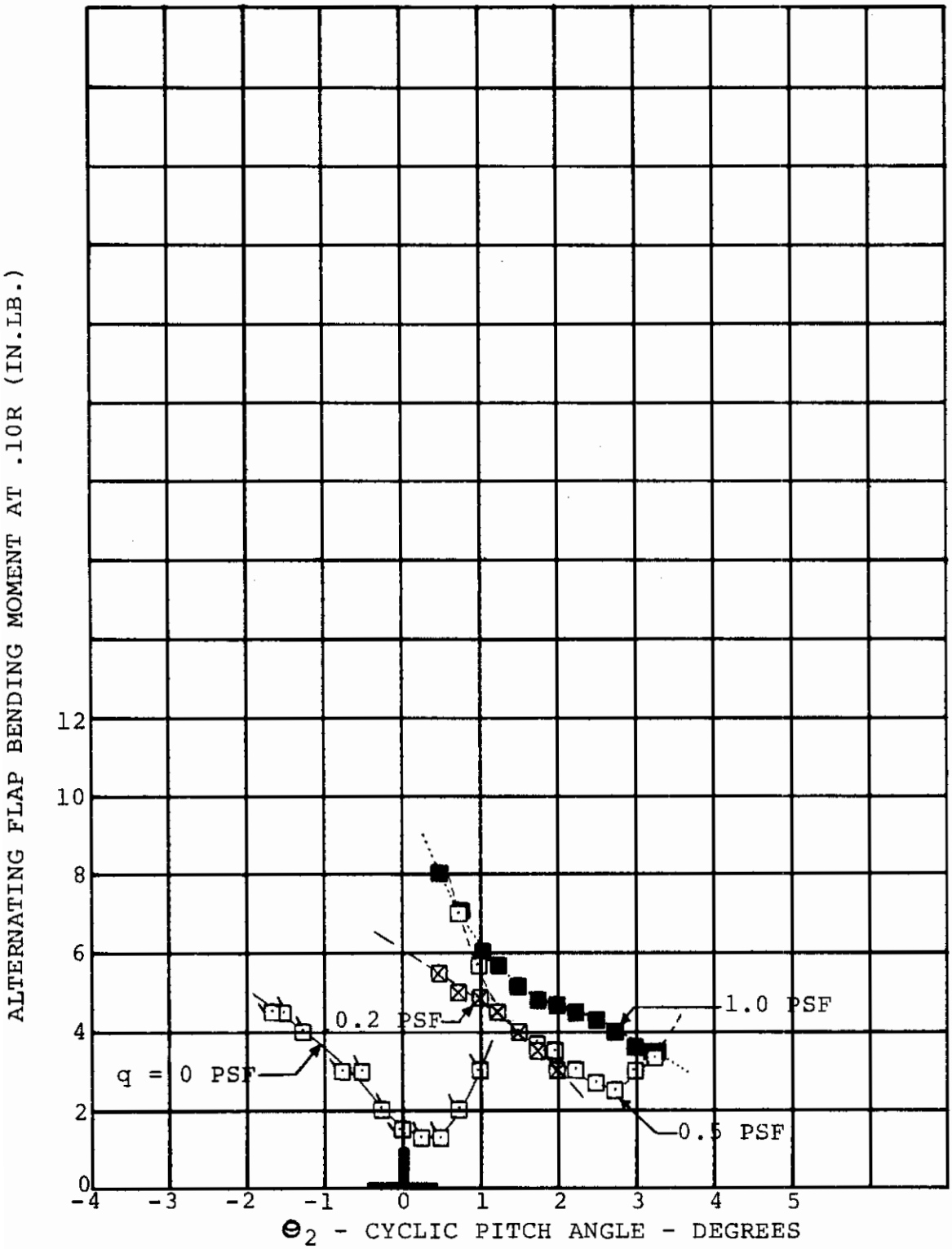


FIGURE 6-23 EFFECT OF DYNAMIC PRESSURE ON RIGHT ROTOR ALTERNATING FLAP BENDING MOMENT IN TRANSITION FOR i_N = 60 DEG., $\Theta_{.75}$ = 9.8 DEG., AND 790 RPM.
RUN 39(6,8,9,12)

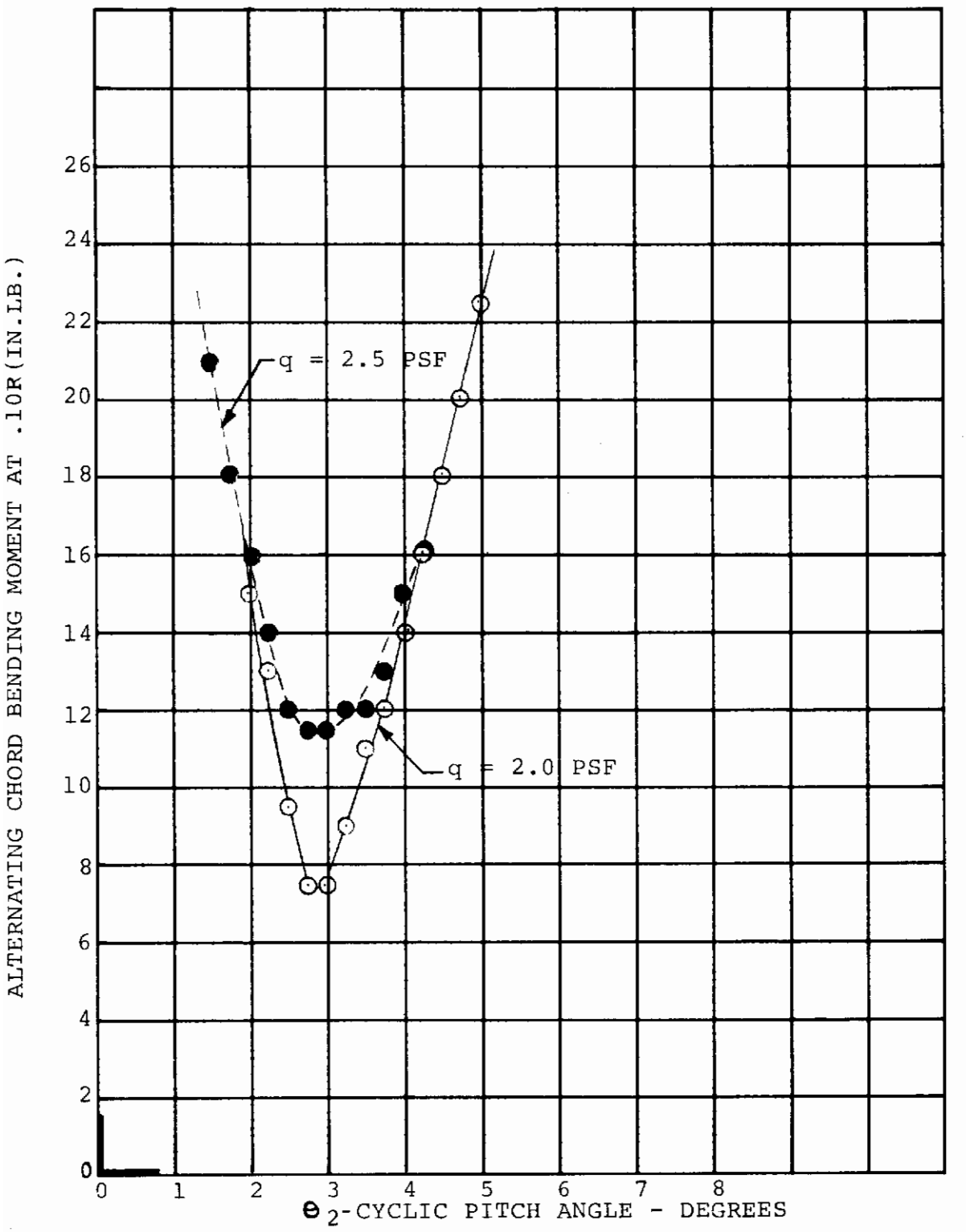


FIGURE 6-24. EFFECT OF DYNAMIC PRESSURE ON RIGHT ROTOR ALTERNATING CHORD BENDING MOMENT IN TRANSITION FOR i_N = 60 DEG., $\theta_{.75}$ = 14.2 DEG., AND 790 RPM.
RUN 41(8,12)

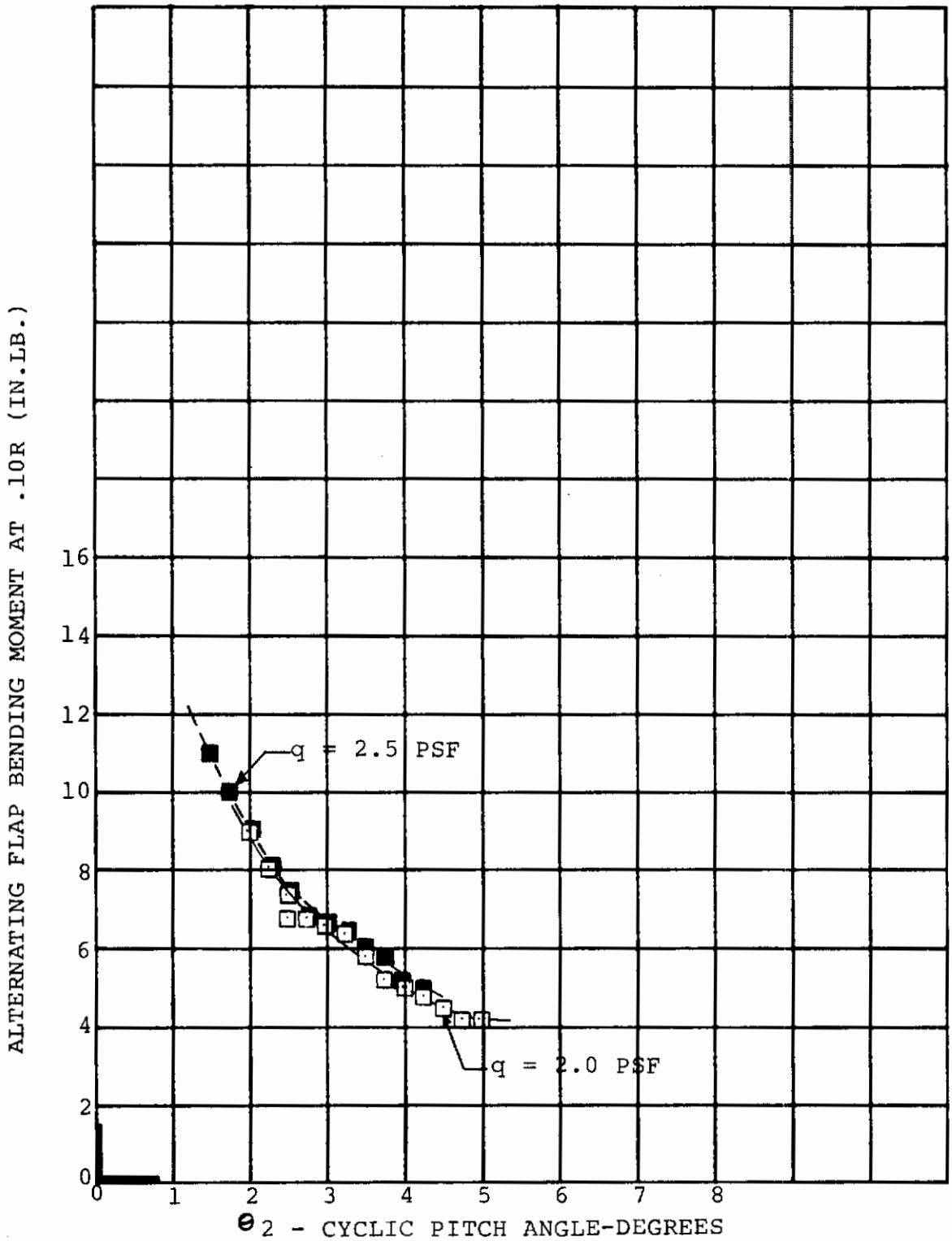


FIGURE 6-25. EFFECT OF DYNAMIC PRESSURE ON RIGHT ROTOR ALTERNATING FLAP BENDING MOMENT IN TRANSITION FOR i_N = 60 DEG., $\theta_{.75}$ = 14.2 DEG., AND 790 RPM.
RUN 41(8,12)

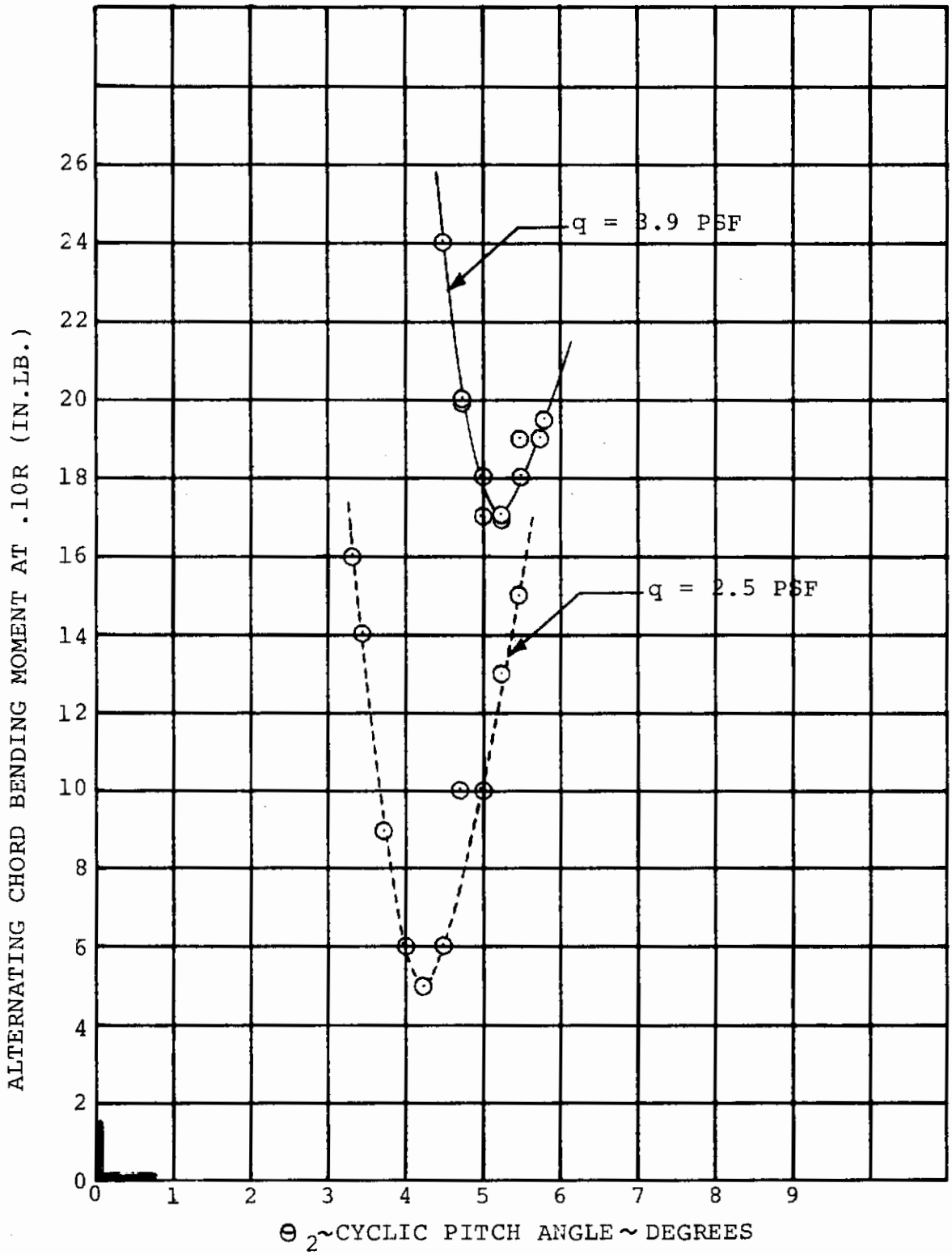


FIGURE 6-26. EFFECT OF DYNAMIC PRESSURE ON RIGHT ROTOR ALTERNATING CHORD BENDING MOMENT IN TRANSITION FOR i_N = 60 DEG., $\Theta_{.75}$ = 16 DEG., AND 790 RPM.

RUN 41(17,24)

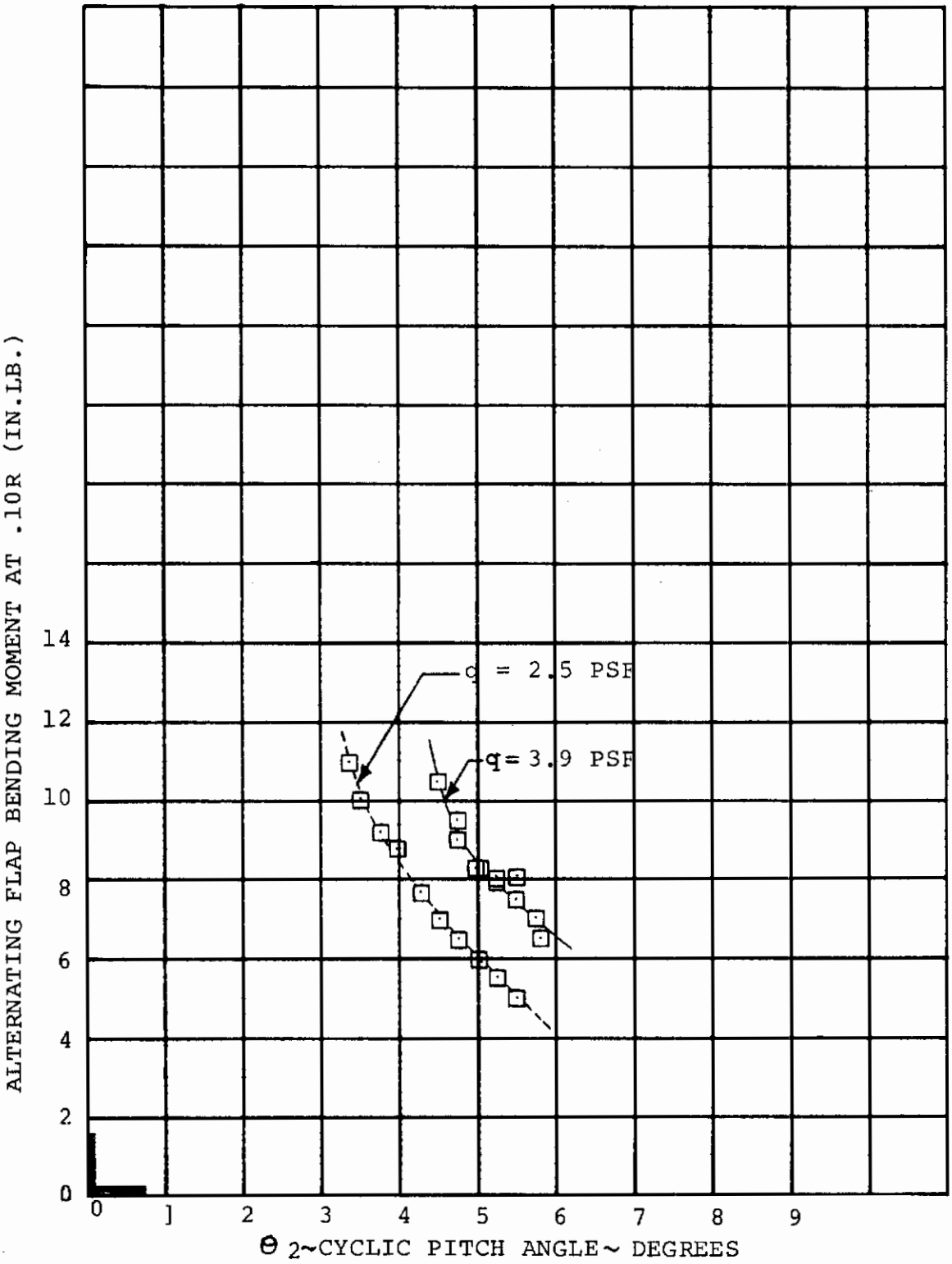


FIGURE 6-27. EFFECT OF DYNAMIC PRESSURE ON RIGHT ROTOR ALTERNATING FLAP BENDING MOMENT IN TRANSITION FOR i_N = 60 DEG., $\theta_{.75}$ = 16 DEG., AND 790 RPM. RUN 41(17,24)

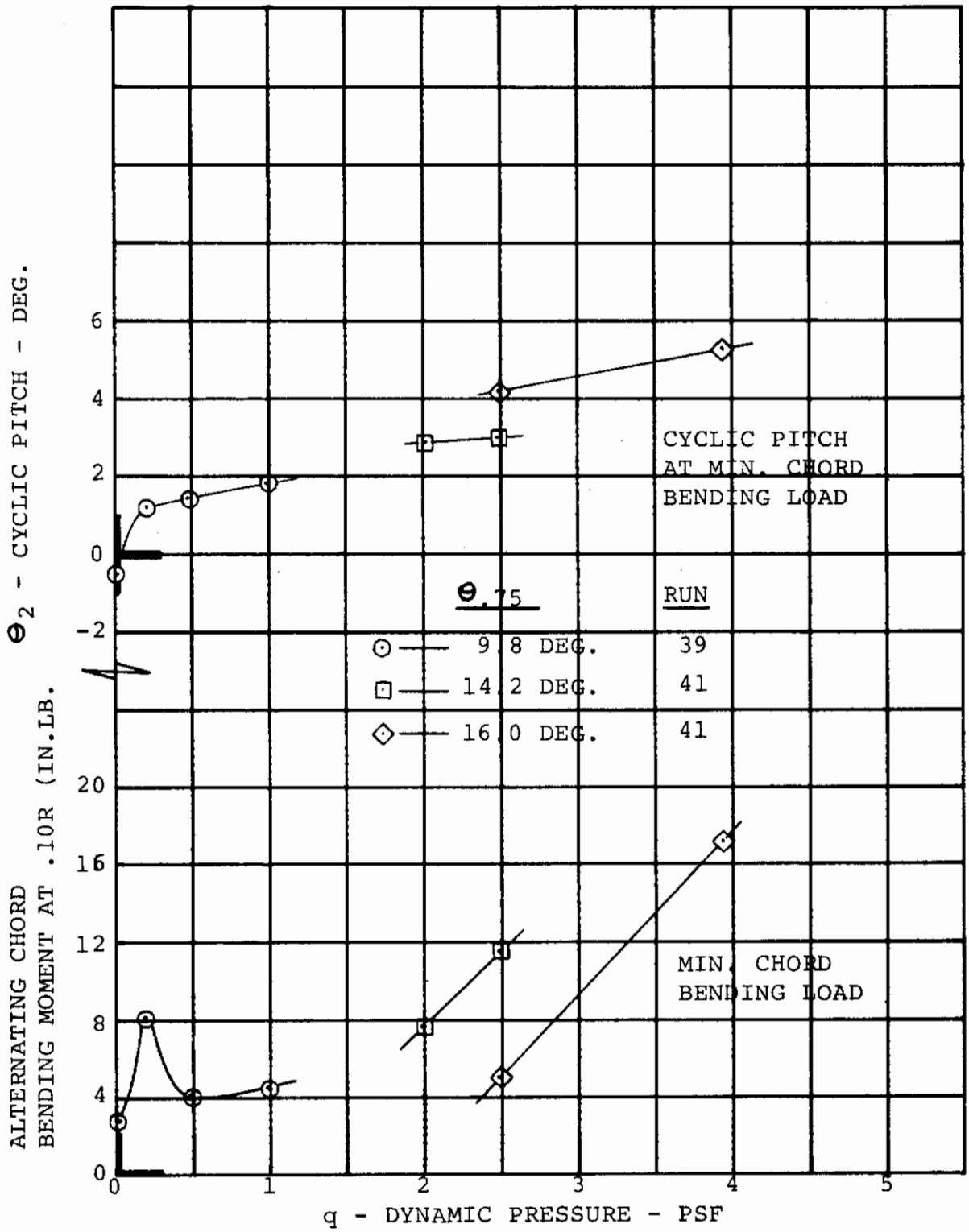


FIGURE 6-28. EFFECT OF DYNAMIC PRESSURE ON BLADE MINIMUM ALTERNATING CHORD BENDING LOAD IN TRANSITION FOR i_N = 60 DEG.

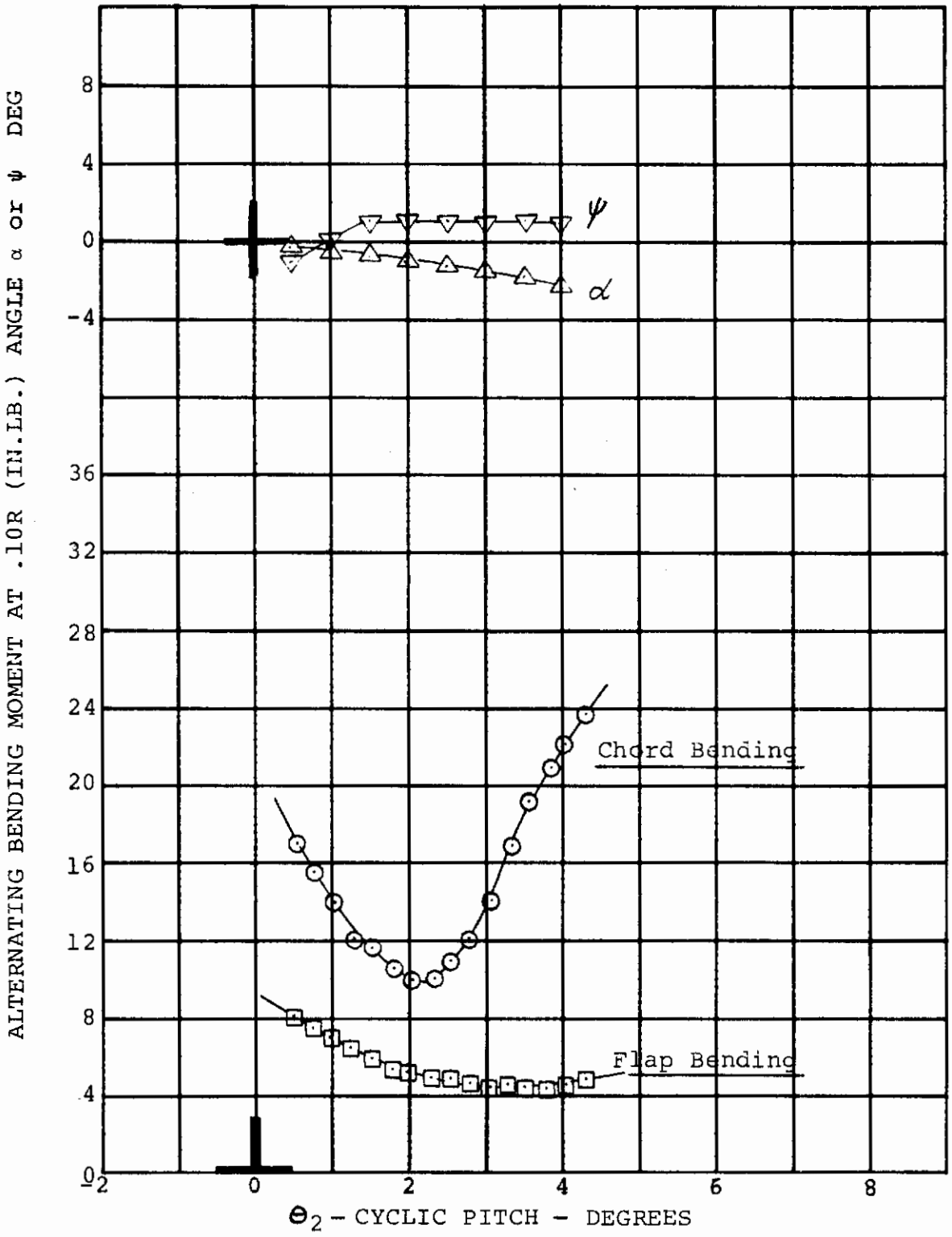


FIGURE 6-29. EFFECT OF CYCLIC PITCH ON RIGHT HAND BLADE ALTERNATING LOADS IN TRANSITION - i_N = 40 DEG., $\Theta_{.75}$ = 16.2 DEG., q = 2.0 PSF, AND Ω = 790 RPM - RUN 43(3)

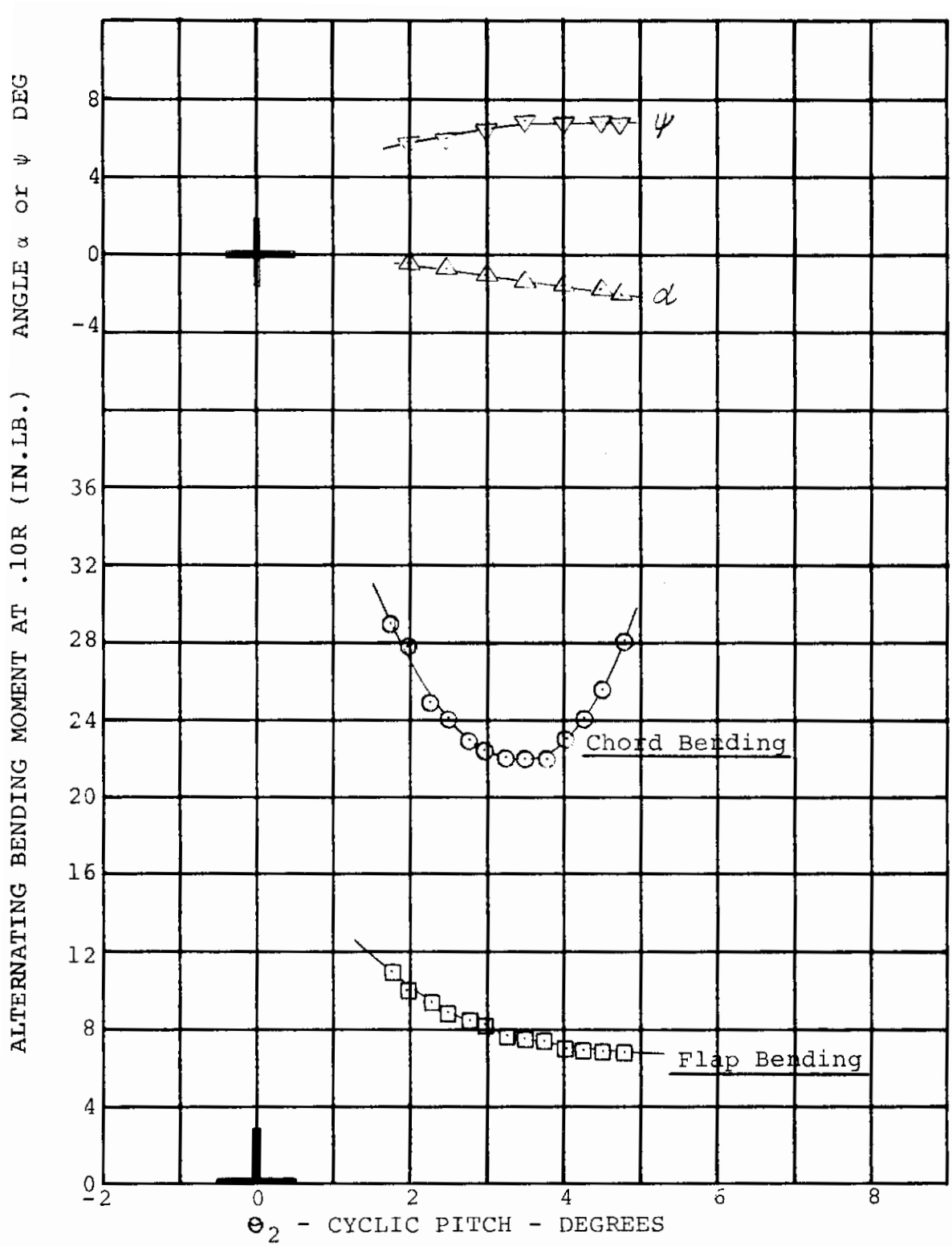


FIGURE 6-30. EFFECT OF CYCLIC PITCH ON RIGHT HAND BLADE ALTERNATING LOADS IN TRANSITION - i_N = 40 DEG., $\theta_{.75}$ = 20.0 DEG., q = 4.0 PSF, AND Ω = 790 RPM - RUN 43(8)

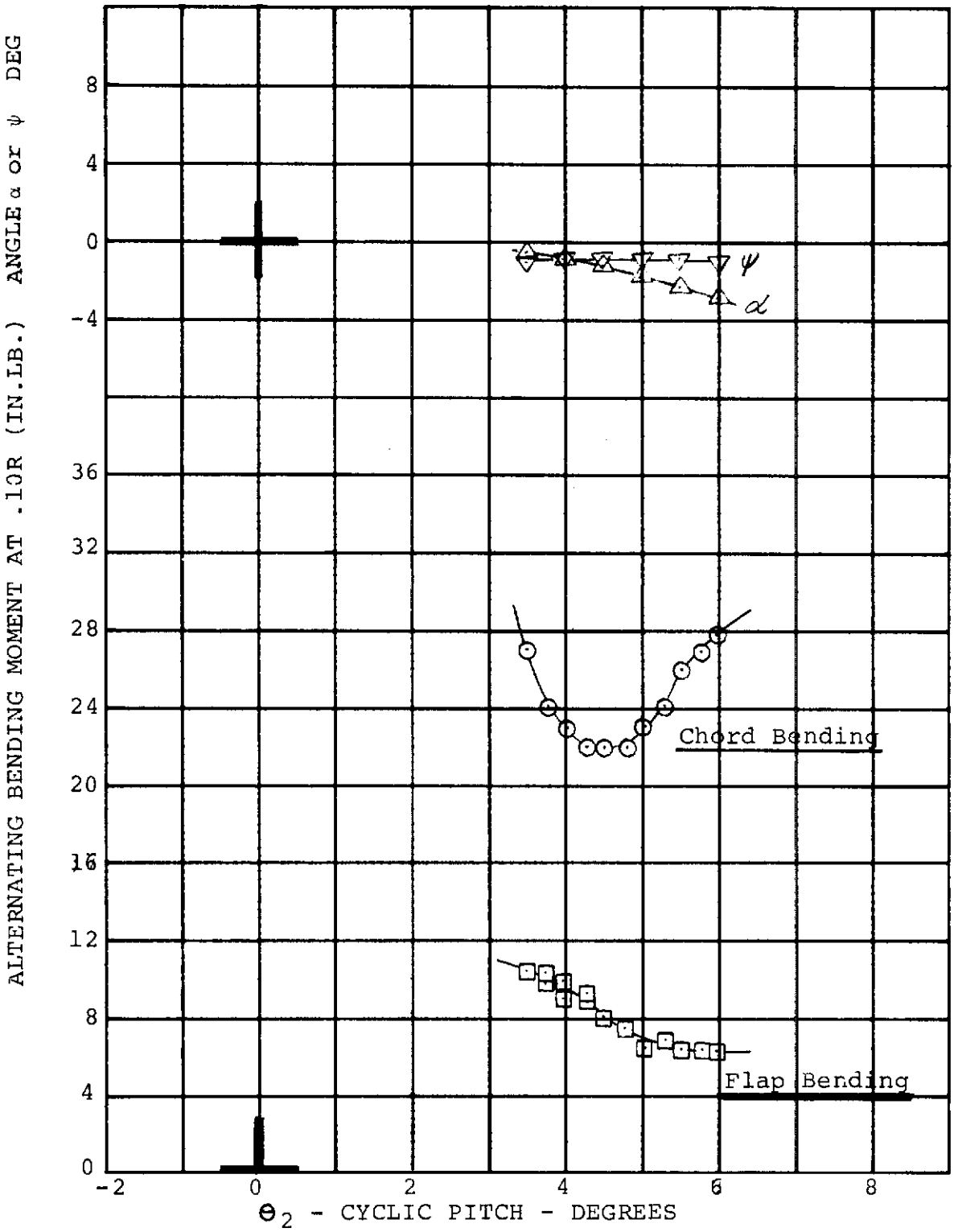


FIGURE 6-31. EFFECT OF CYCLIC PITCH ON RIGHT HAND BLADE ALTERNATING LOADS IN TRANSITION $-i_N$ = 40 DEG., $\theta_{.75}$ = 23 DEG., q = 5.25 PSF, AND Ω =790 RPM – RUN 43 (14)

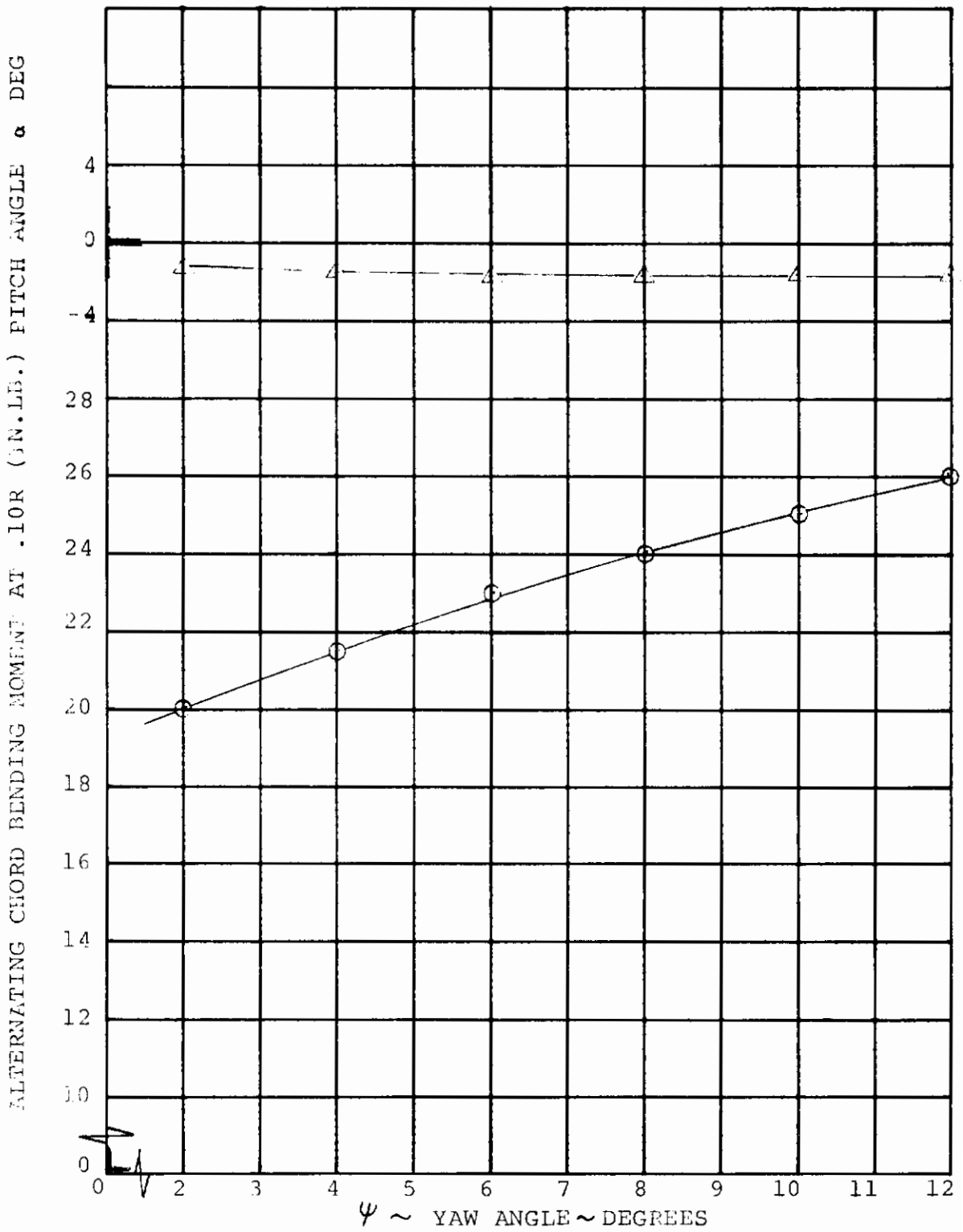


FIGURE 6-32. EFFECT OF MODEL YAW ANGLE ON RIGHT ROTOR ALTERNATING CHORD BENDING MOMENT IN TRANSITION FOR i_N= 60 DEG.,$\theta_{.75}$=16 DEG.,q=4.1 PSF,θ_2=5.8 DEG.,AND 790 RPM. RUN 41(27)

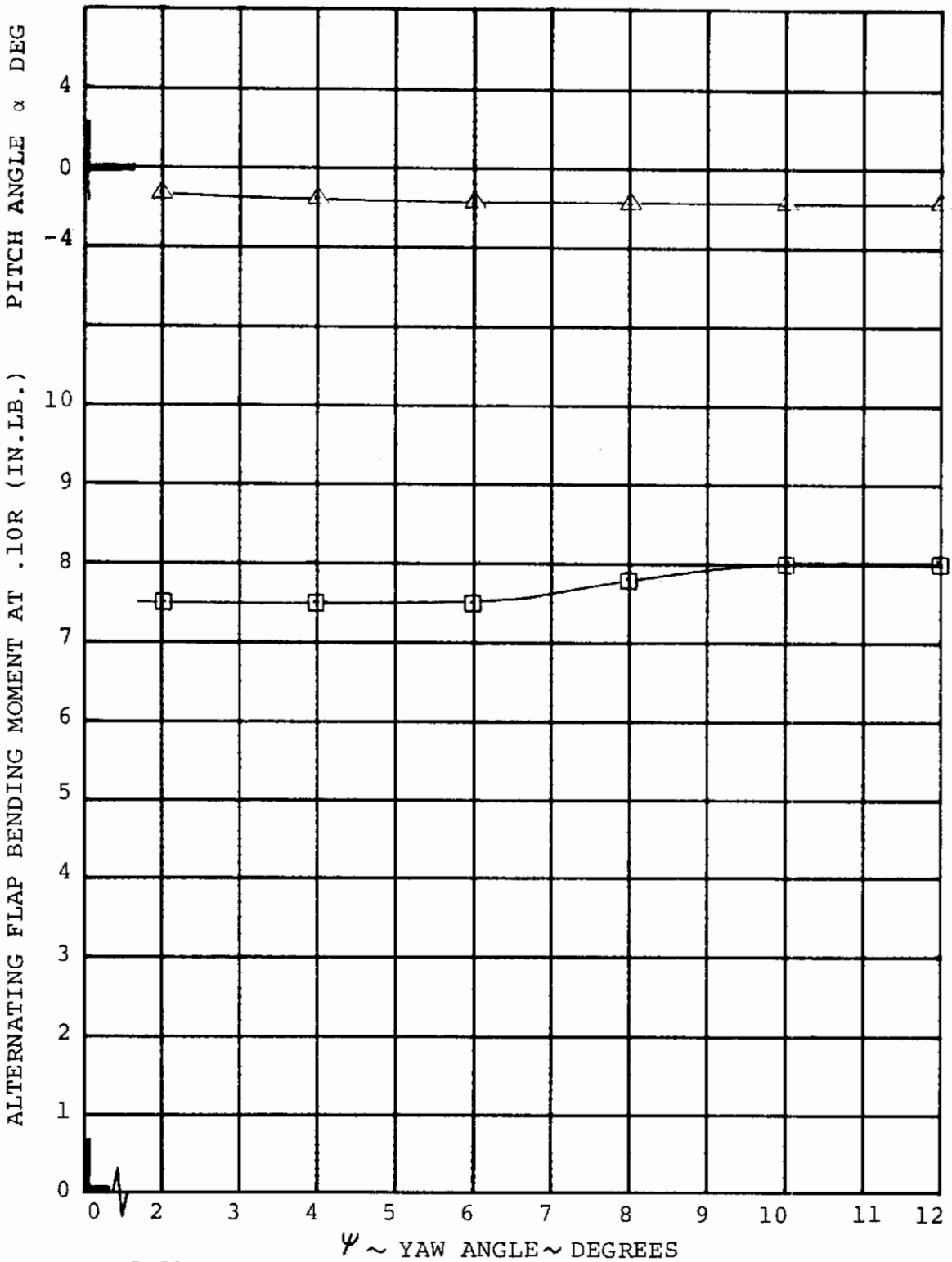


FIGURE 6-33. EFFECT OF MODEL YAW ANGLE ON RIGHT ROTOR ALTERNATING FLAP BENDING MOMENT IN TRANSITION FOR i_N= 60 DEG., $\Theta_{.75}$=16 DEG., q=4.1 PSF, Θ_2=5.8 DEG., AND 790 RPM.

RUN 41(27)

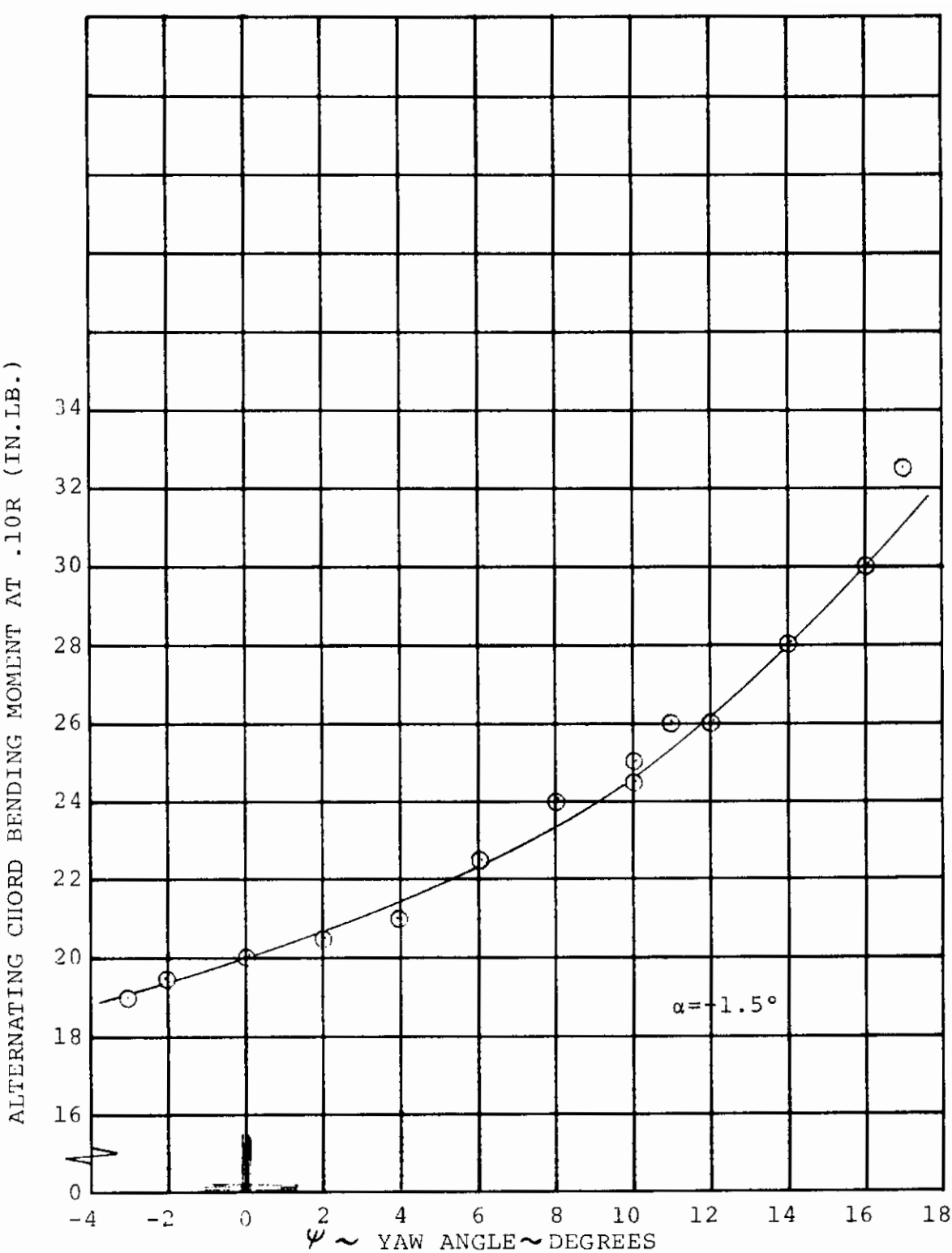


FIGURE 6-34. EFFECT OF MODEL YAW ANGLE ON RIGHT ROTOR ALTERNATING CHORD BENDING MOMENT IN TRANSITION FOR i_N= 40 DEG., $\theta_{.75}$=20 DEG., q=4 PSF, θ_2=3.8 DEG., AND 790 RPM.

RUN 43(10)

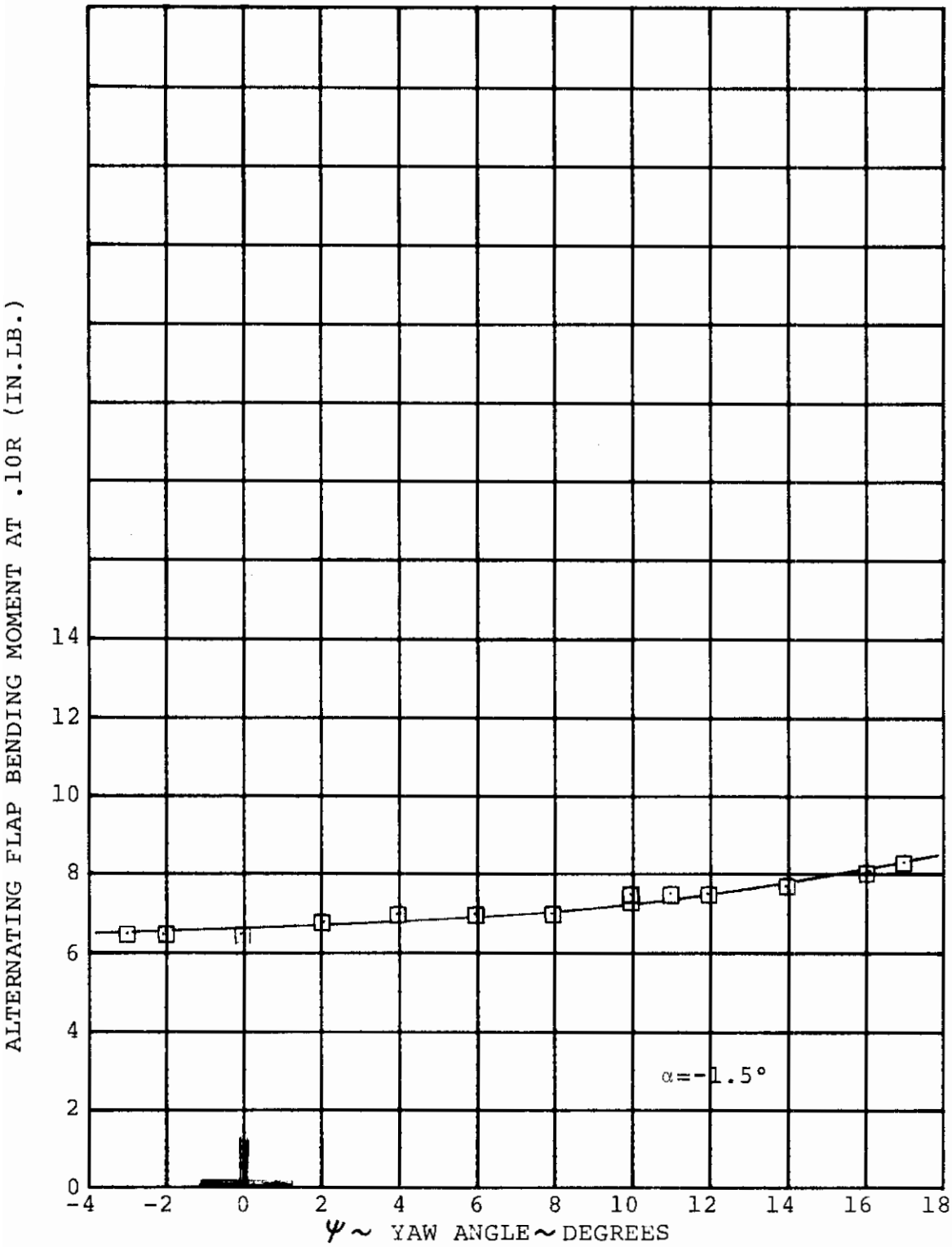


FIGURE 6-35 EFFECT OF MODEL YAW ANGLE ON RIGHT ROTOR ALTERNATING FLAP BENDING MOMENT IN TRANSITION FOR i_N = 40 DEG., $\Theta_{.75}$=20 DEG., q=4 PSF, Θ_2=3.8 DEG., AND 790 RPM. RUN 43(10)

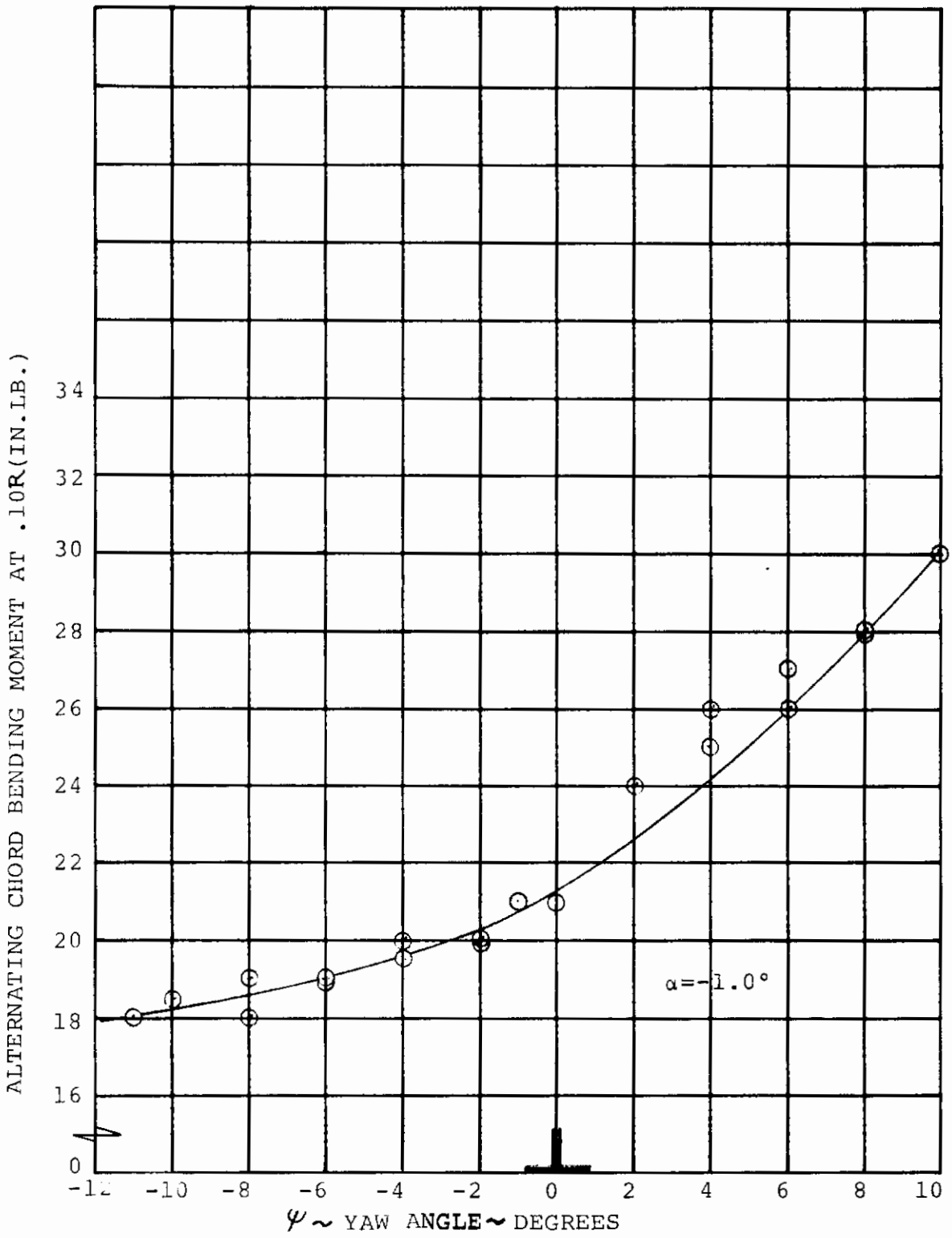


FIGURE 6-36. EFFECT OF MODEL YAW ANGLE ON RIGHT ROTOR ALTERNATING CHORD BENDING MOMENT IN TRANSITION FOR i_N= 40 DEG., $\theta_{.75}$=23 DEG., q=5.25 PSF, θ_2=4.3 DEG., AND 790 RPM.

RUN 43(16)

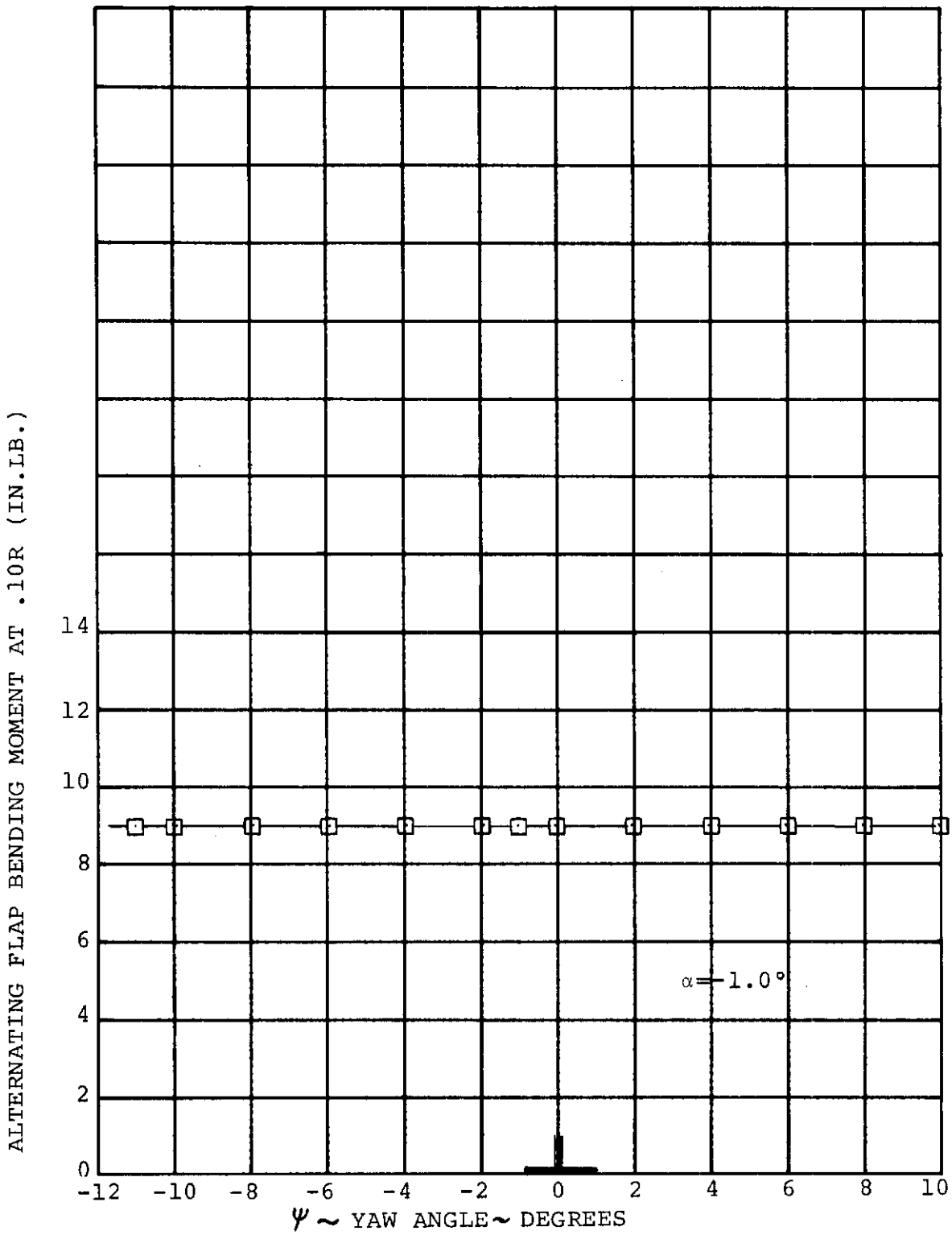


FIGURE 6-37. EFFECT OF MODEL YAW ANGLE ON RIGHT ROTOR ALTERNATING FLAP BENDING MOMENT IN TRANSITION FOR i_N=40 DEG., $\theta_{.75}$=23 DEG., q=5.25 PSF, θ_2=4.3 DEG., AND 790 RPM.

RUN 43(16)

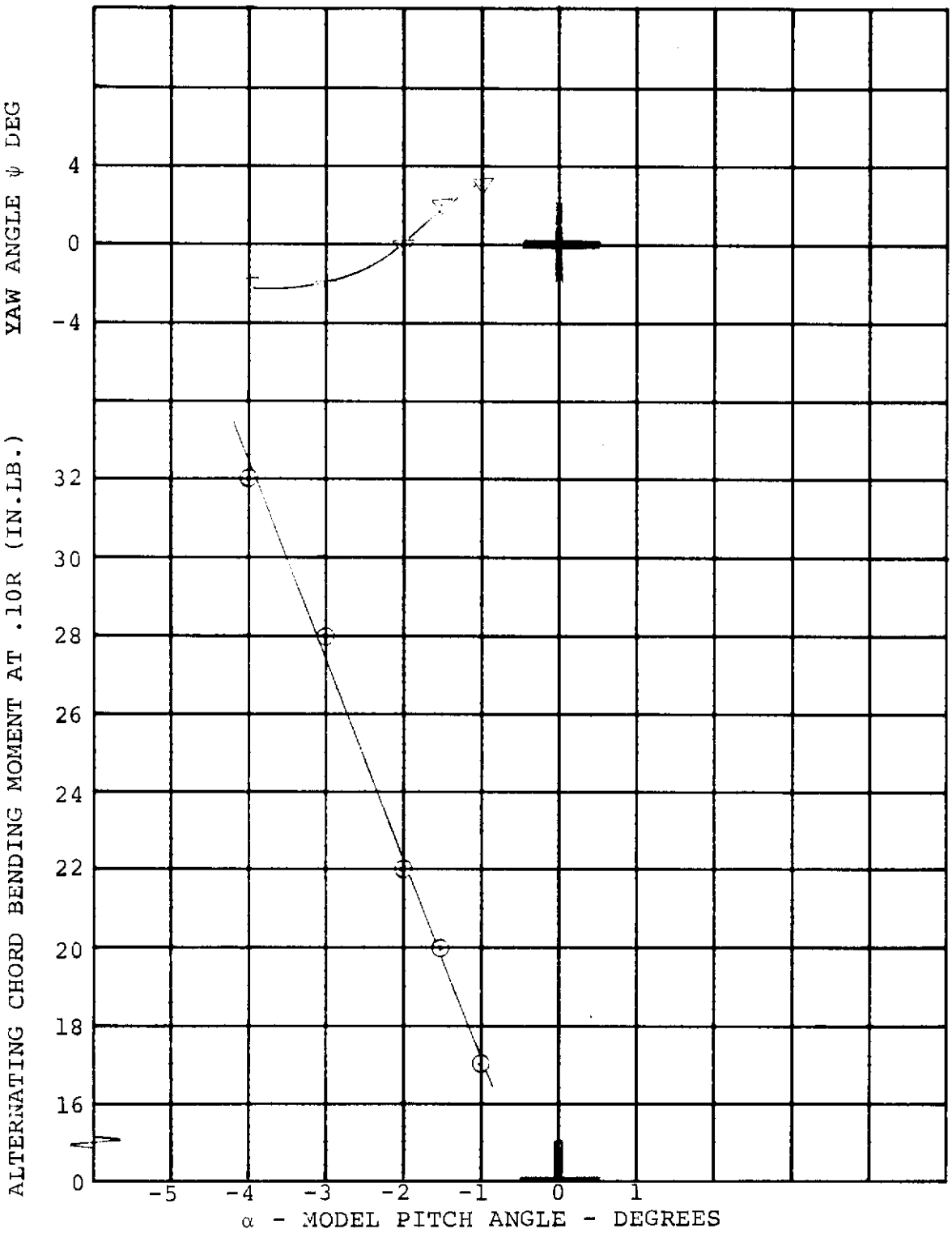


FIGURE 6-38 EFFECT OF MODEL PITCH ANGLE ON RIGHT ROTOR ALTERNATING CHORD BENDING MOMENT IN TRANSITION FOR i_N=60 DEG., $\theta_{.75}$=16 DEG., q=4.1 PSF, θ_2=5.8 DEG., AND 790 RPM

RUN 41(26)

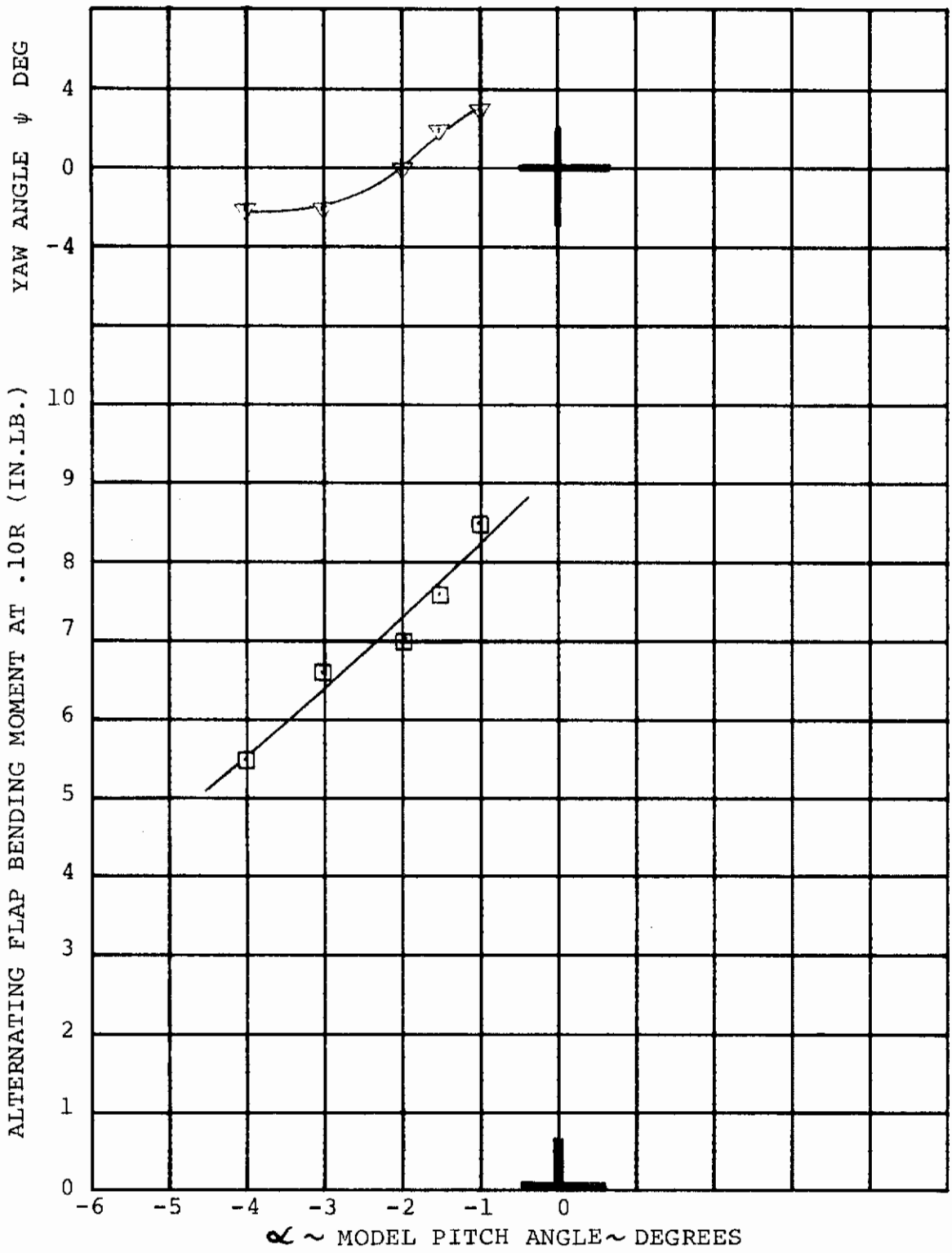


FIGURE 6-39. EFFECT OF MODEL PITCH ANGLE ON RIGHT ROTOR ALTERNATING FLAP BENDING MOMENT IN TRANSITION FOR i_N=60 DEG., $\Theta_{.75}$=16 DEG.,q=4.1 PSF,Θ_2=5.8 DEG., AND 790 RPM.

RUN 41(26)

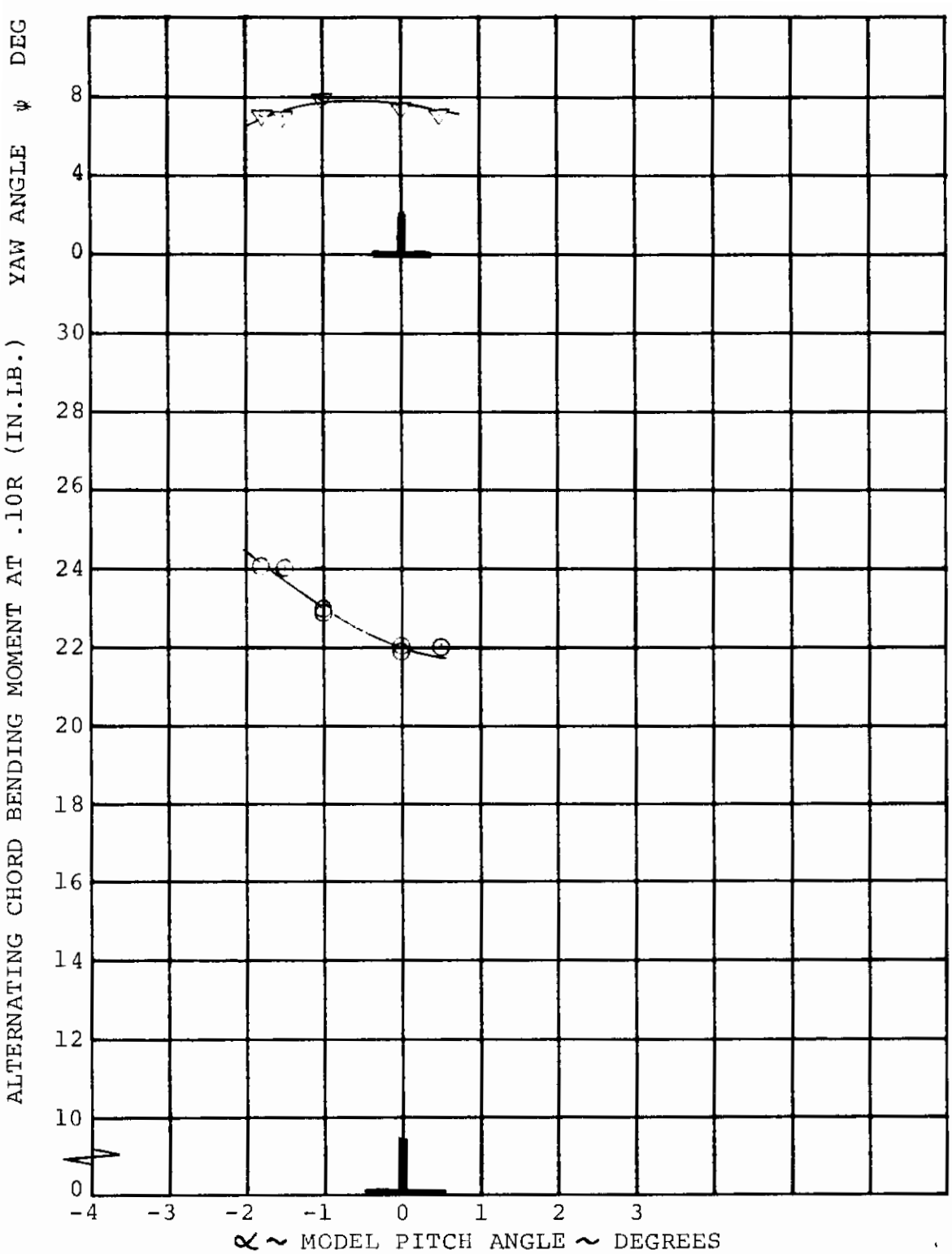


FIGURE 6-40. EFFECT OF MODEL PITCH ANGLE ON RIGHT ROTOR ALTERNATING CHORD BENDING MOMENT IN TRANSITION FOR i_N= 40 DEG., $\Theta_{.75}$=20 DEG., q=4 PSF, Θ_2=3.8 DEG., AND 790 RPM. RUN 43(9)

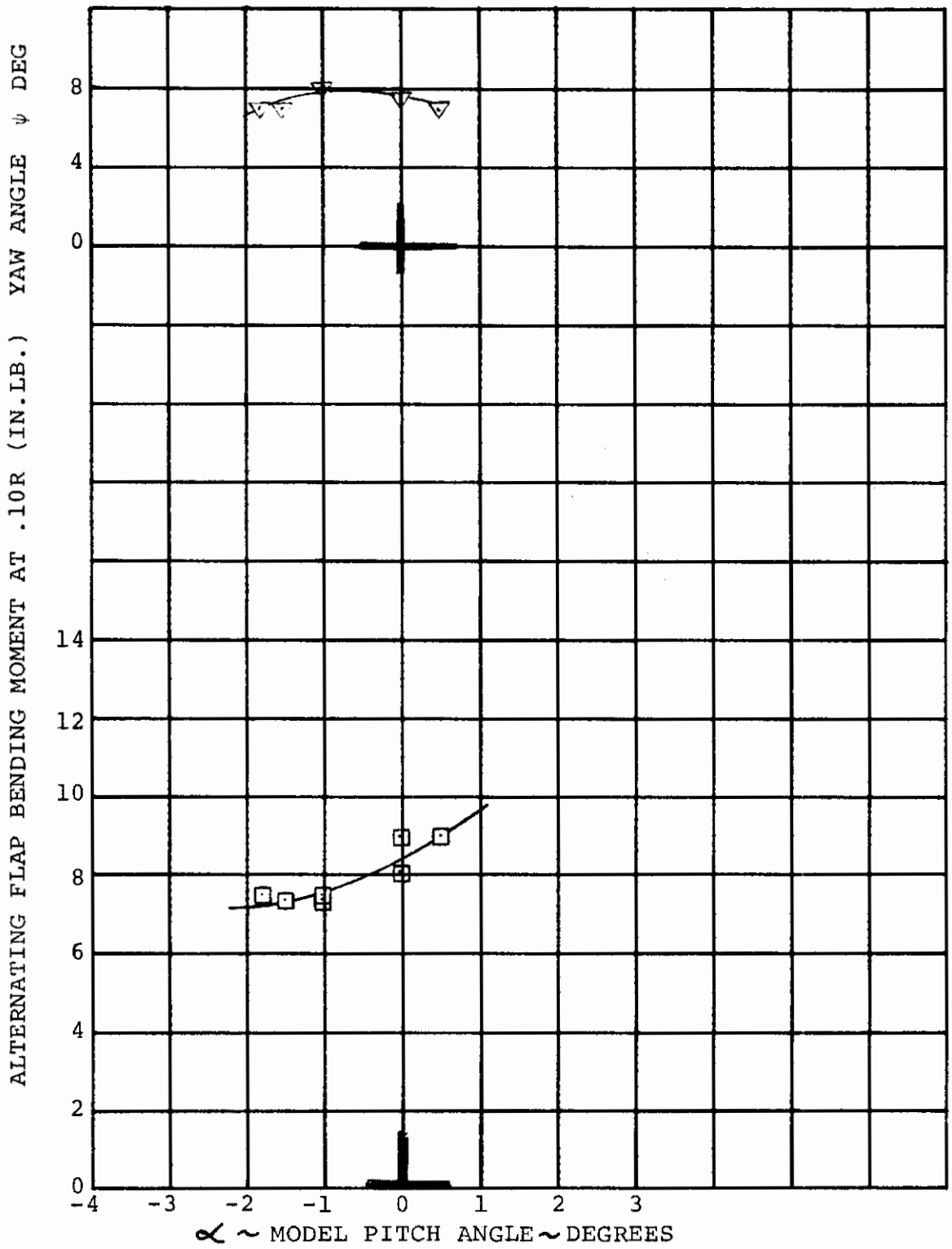


FIGURE 6-41. EFFECT OF MODEL PITCH ANGLE ON RIGHT ROTOR ALTERNATING FLAP BENDING MOMENT IN TRANSITION FOR i_N = 40 DEG., $\theta_{.75}$ = 20 DEG., q=4 PSF, θ_2 = 3.8 DEG., AND 790 RPM.
RUN 43(9)

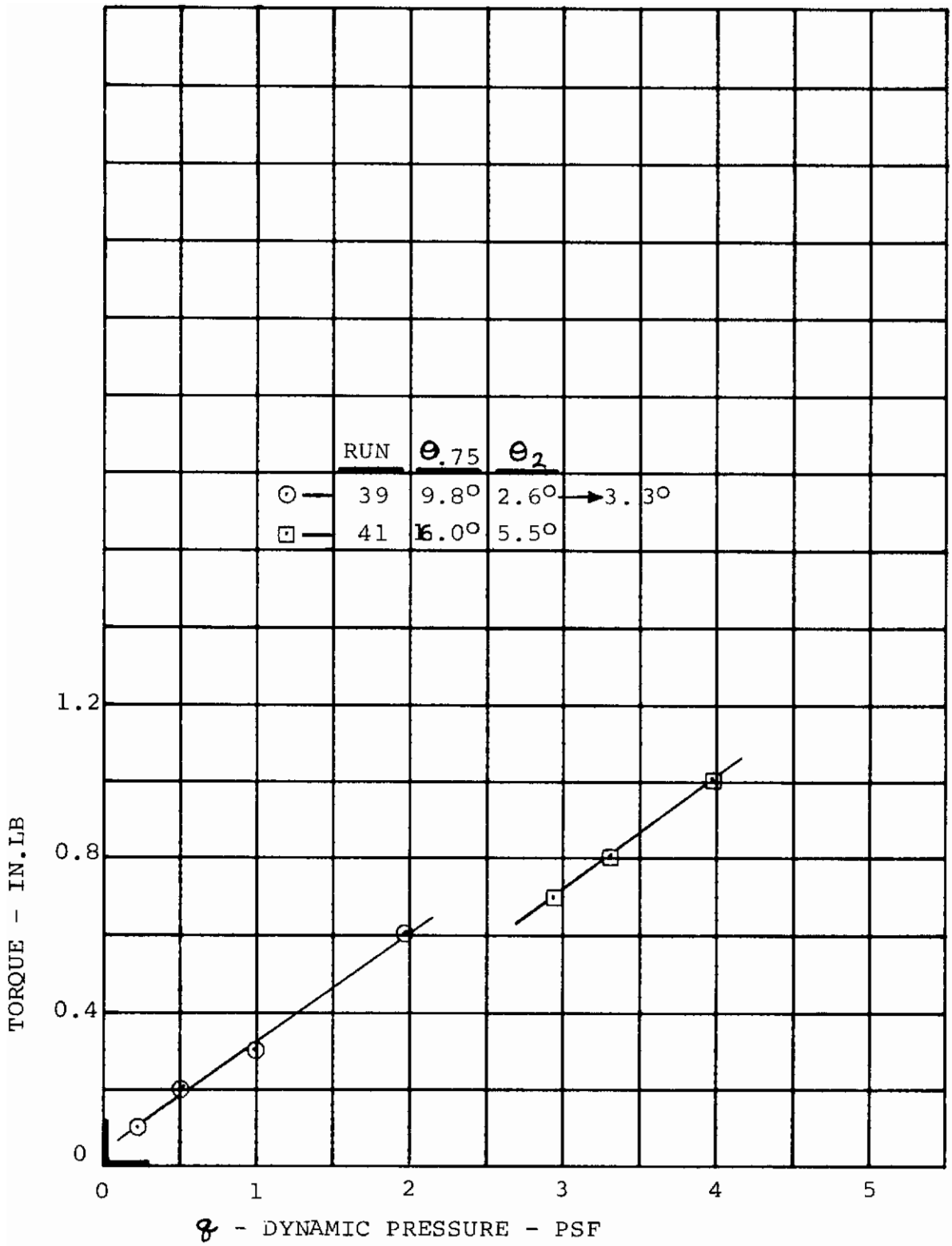
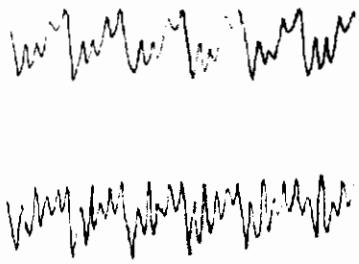


FIGURE 6-42. BLADE RESPONSE AT TORSIONAL FREQUENCY FOR i_N = 60 DEG., f = 65 CPS

VERTICAL LINES AT $\psi = 90°$



RUN	q	$\theta_{.75}$	θ_2	Ω
	PSF	DEG	DEG	RPM
39	1.9	9.8°	3.3°	380
41	3.9	16.0°	5.5°	380

FIGURE 6-43 BLADE TORSION RESPONSE IN TRANSITION
- RIGHT BLADE -.15R $\mu = .40$

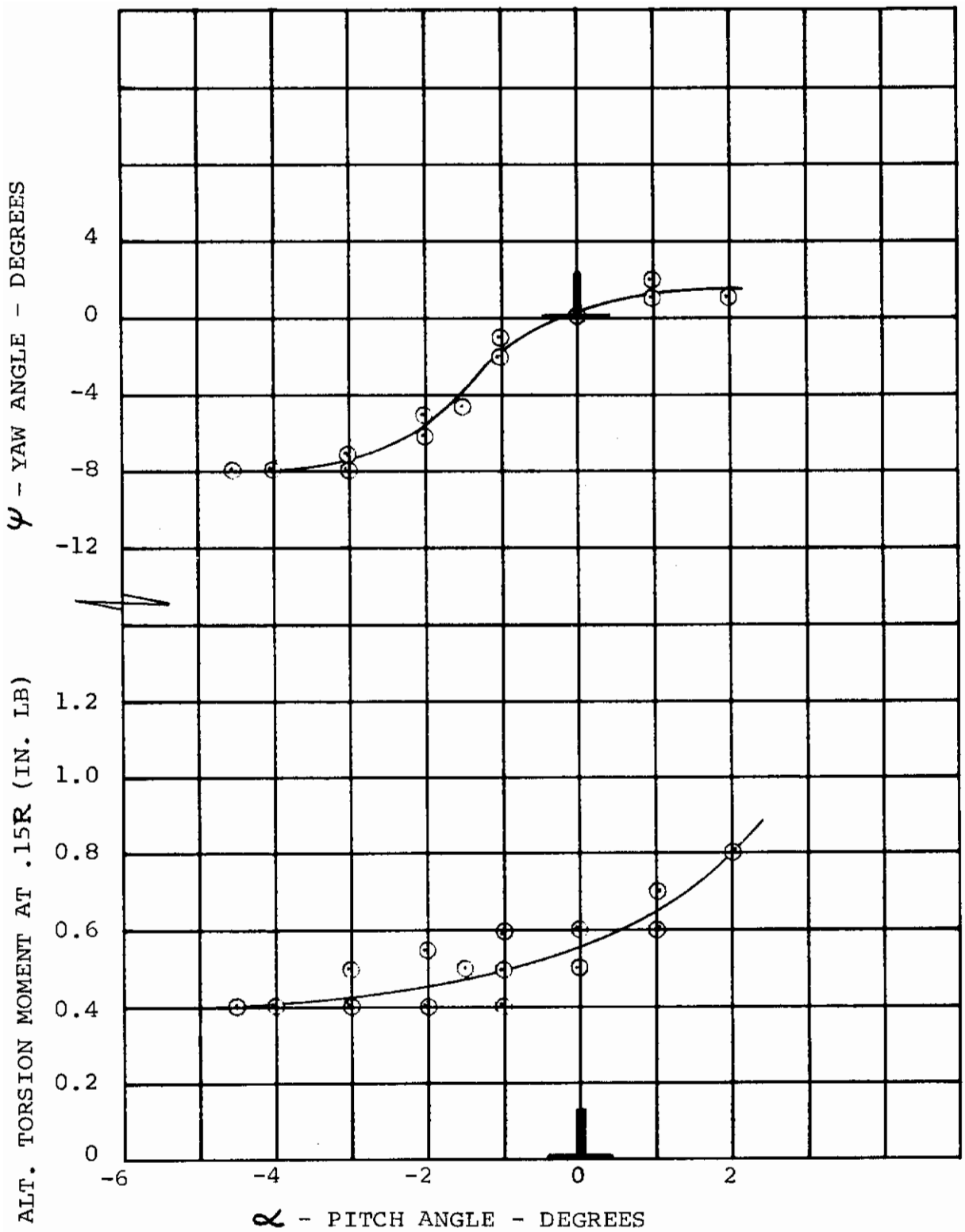


FIGURE 6-44 EFFECT OF FUSELAGE DISPLACEMENT ON RIGHT HAND BLADE ALTERNATING TORSION RESPONSE AT ITS TORSION NATURAL FREQUENCY FOR i_N = 40 DEG., $\Theta_{.75}$ = 23 DEG., Θ_2 = 4.3 DEG., q = 5.25 PSF, AND 790 RPM–RUN 43 (15)

6.3 CRUISE CONFIGURATION BLADE LOADS

Blade loads data were taken in the cruise configuration to show the effect of cyclic pitch, model pitch, and model yaw, collective, and dynamic pressure.

At a tunnel dynamic pressure (q = 4.9 psf) corresponding to a full-scale speed of 120 knots, alternating chord bending sensitivity to cyclic pitch was twice that due to cyclic in hover - OGE. Cyclic pitch was shown to have little effect on alternating flap bending.

At a dynamic pressure (q = 6.65 psf) corresponding to a full-scale speed of 140 knots, 5 degrees of model yaw produced alternating chord bending equivalent to that due to .85 degrees of hover cyclic. Alternating flap bending was insensitive to model yaw.

The isolated effect of model pitch on blade loads could not be determined since the model tended to yaw as the model was pitched. The uncertainty of the physical position of the model in the tunnel after attitude changes through the snubber cables caused the coupling between model pitch and yaw. The yaw was not caused by any aerodynamic or gyroscopic effects.

6.3.1 Effect of Cyclic Pitch

Blade loads data taken in cruise at q = 5 to show the effect of cyclic pitch are shown in Figures 6-45 and 6-46. These data show the alternating flap bending moment to be insensitive to cyclic pitch while alternating chord bending moments increase at the rate of 12 in. lb. per degree of cyclic pitch. Referring to Figure 6-11 the sensitivity of chord bending to cyclic in hover was 6 in. lb. per degree cyclic.

6.3.2 Effect of Model Yaw

Figures 6-47 through 6-50 show the effect of model yaw on blade loads at q = 6 and q = 7 PSF. These data showed alternating chord bending to be sensitive to model yaw whereas alternating flap bending showed little increase with yaw. At a q of 6, 5 degrees of yaw produce alternating chord bending equivalent to .3 degree cyclic in hover. At a q of 7 PSF, 5 degrees of yaw produce alternating chord bending equivalent to .85 degree of hover cyclic.

6.3.3 Effect of Model Pitch

Figures 6-51 through 6-56 show the effect of model pitch on blade loads at q = 6 and q = 7 PSF. These data are inconclusive in determining the isolated effect of model pitch since the model yawed as the model pitch was increased. This was caused by soft model mount and the inability to keep model yaw constant with the snubber cables. However, the blade loads data show the same trend as when the model was yawed. Alternating chord bending is sensitive to changes in pitch whereas there is little change in alternating flap bending.

6.3.4 Effect of Collective Pitch

The effect of collective pitch on steady flap bending in cruise is shown in Figure 6-57. Steady flap bending increased sharply with increased collective and the gradient of steady flap bending with collective increased as dynamic pressure increased. In Figure 6-13 the variation of steady flap bending with collective in hover was 0.7 in. lb per degree collective. As shown in Figure 6-57, at q = 6, the gradient of steady flap bending with collective in cruise is 8 in. lb per degree or 10 times as sensitive as in hover. This is a direct result of the typically increased sensitivity of rotor thrust with collective as advance ratio is increased in cruise, as shown in Figure 6-58.

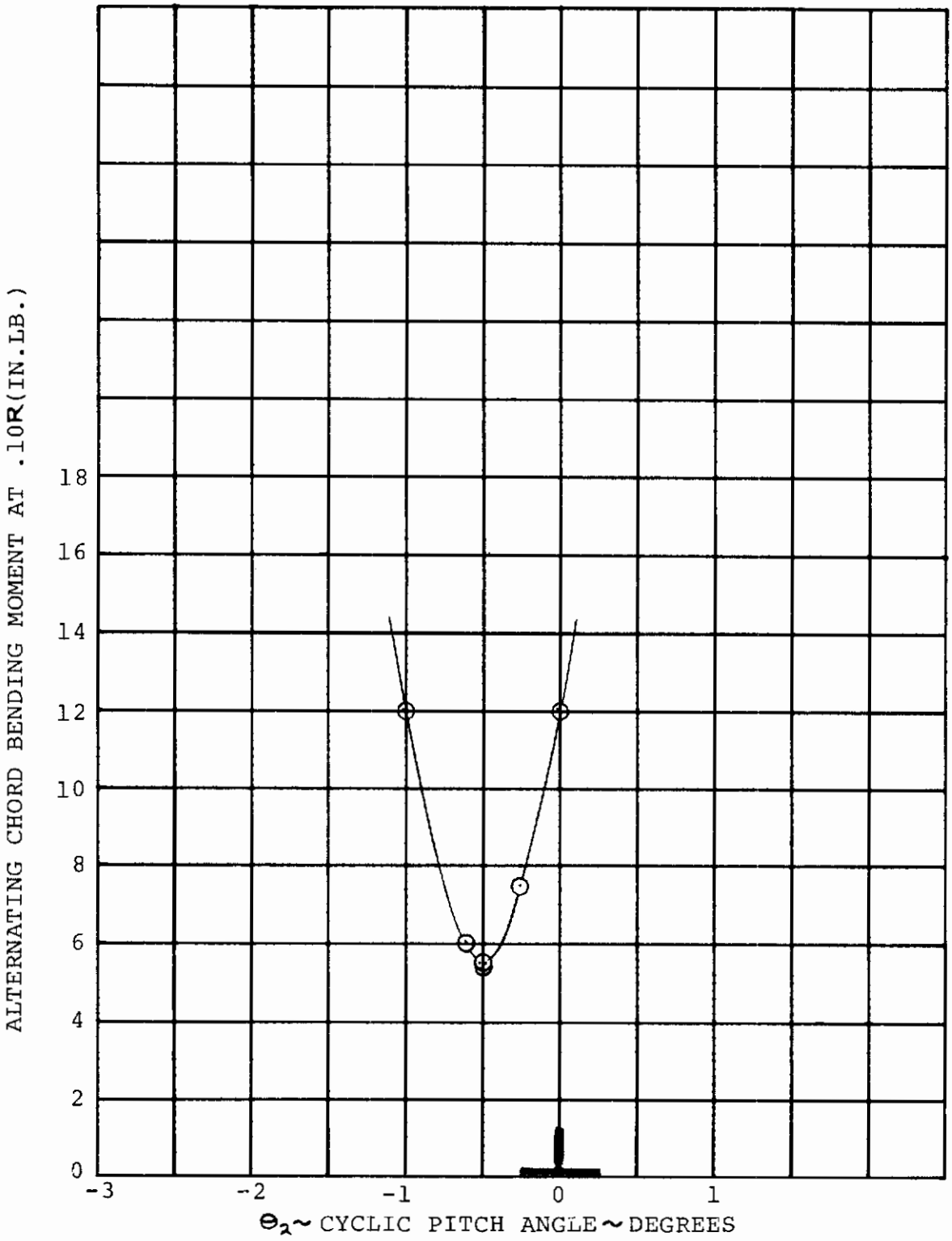


FIGURE 6-45 EFFECT OF CYCLIC PITCH ON RIGHT ROTOR ALTERNATING CHORD BENDING MOMENT IN CRUISE FOR i_N = 0 DEG., $\Theta_{.75}$ = 24.5 DEG., q = 5 PSF, AND 790 RPM. RUN 37(33-37)

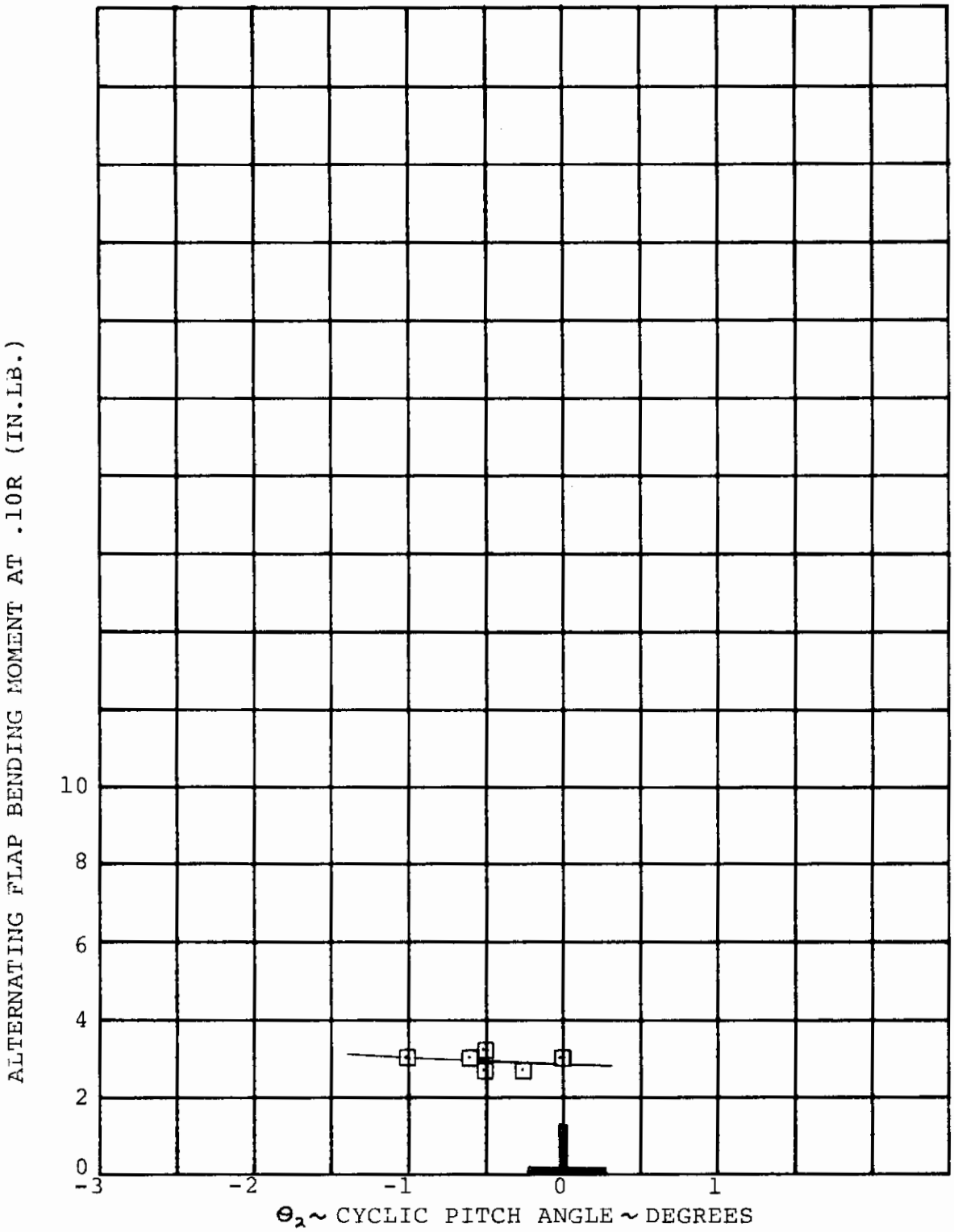


FIGURE 6-46 EFFECT OF CYCLIC PITCH ON RIGHT ROTOR ALTERNATING FLAP BENDING MOMENT IN CRUISE FOR i_N= 0 DEG. $\Theta_{.75}$= 24.5 DEG., q = 5 PSF, AND 790 RPM.
RUN 37(33-37)

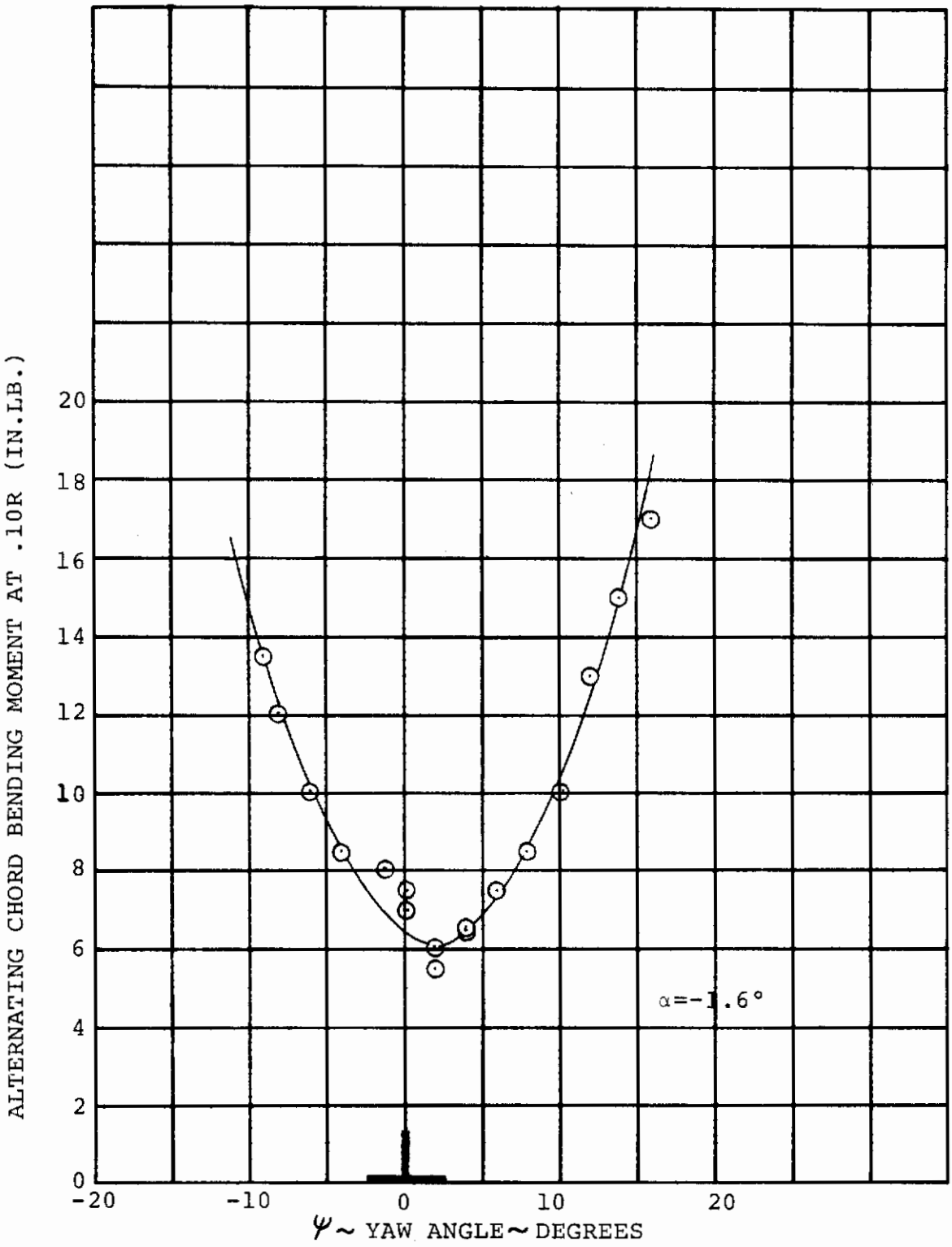


FIGURE 6-47. EFFECT OF MODEL YAW ANGLE ON RIGHT ROTOR ALTERNATING CHORD BENDING MOMENT IN CRUISE FOR i_N =0 DEG. $\Theta_{.75}$=26.7 DEG., q=6 PSF, Θ_2= -0.7 DEG., AND 790 RPM. RUN 38(4)

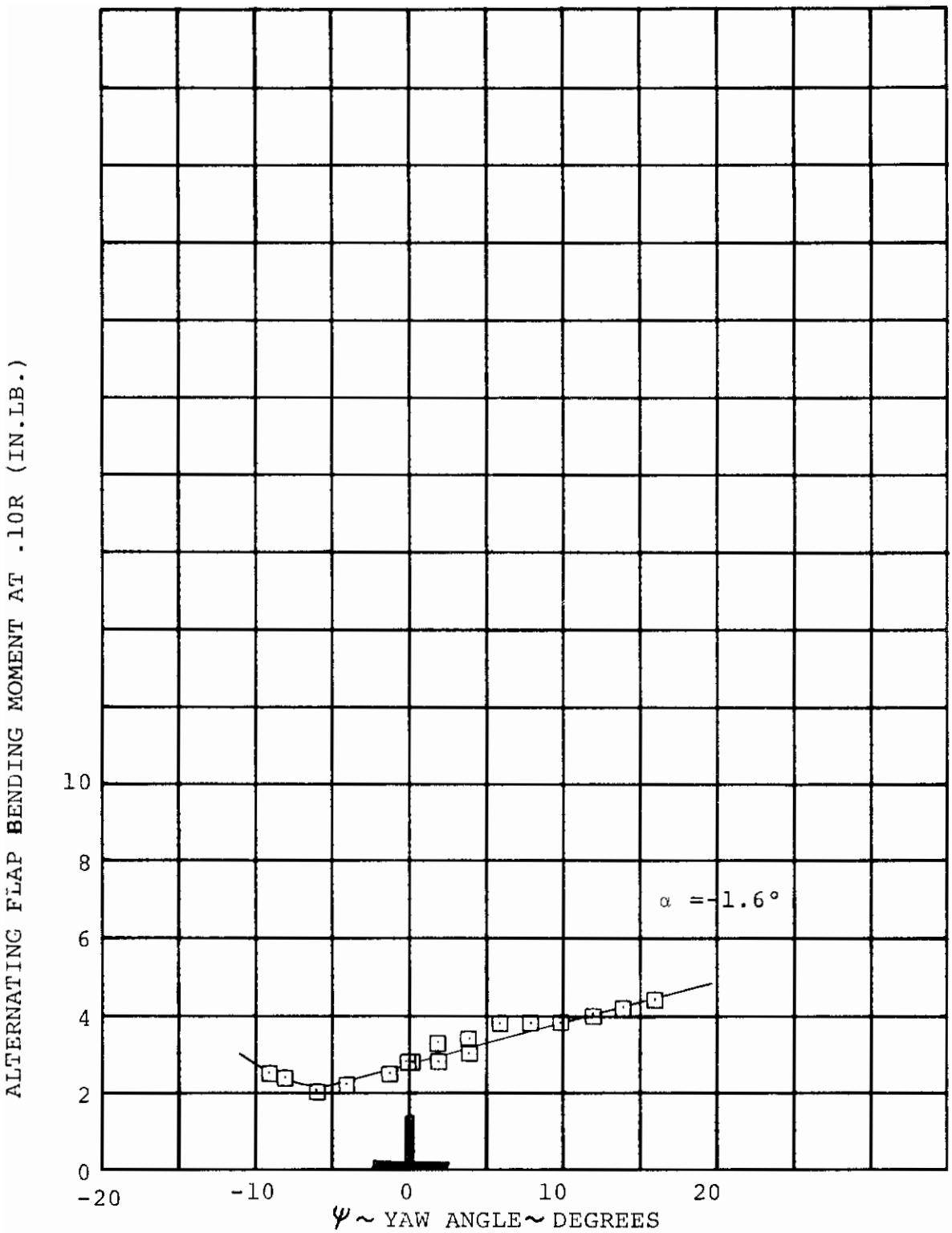


FIGURE 6-48. EFFECT OF MODEL YAW ANGLE ON RIGHT ROTOR ALTERNATING FLAP BENDING MOMENT IN CRUISE FOR i_N= 0 DEG. $\Theta_{.75}$=26.7 DEG., q= 6 PSF, Θ_2= -0.7 DEG., AND 790 RPM. RUN 38(4)

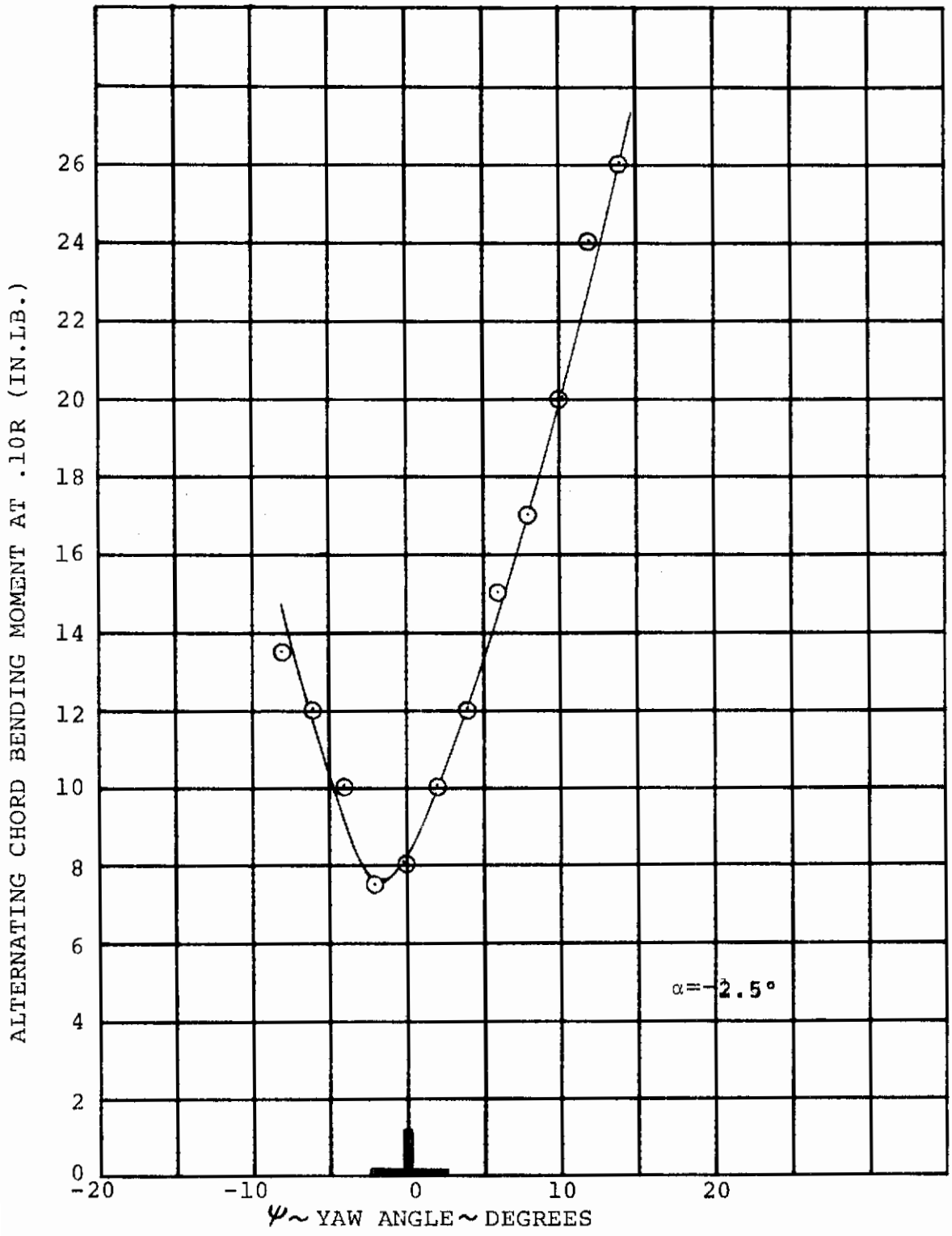


FIGURE 6-49. EFFECT OF MODEL YAW ANGLE ON RIGHT ROTOR ALTERNATING CHORD BENDING MOMENT IN CRUISE FOR i_N= 0 DEG. $\Theta_{.75}$=30.7 DEG., q= 7 PSF, Θ_2=-0.7 DEG., AND 790 RPM. RUN 38(10)

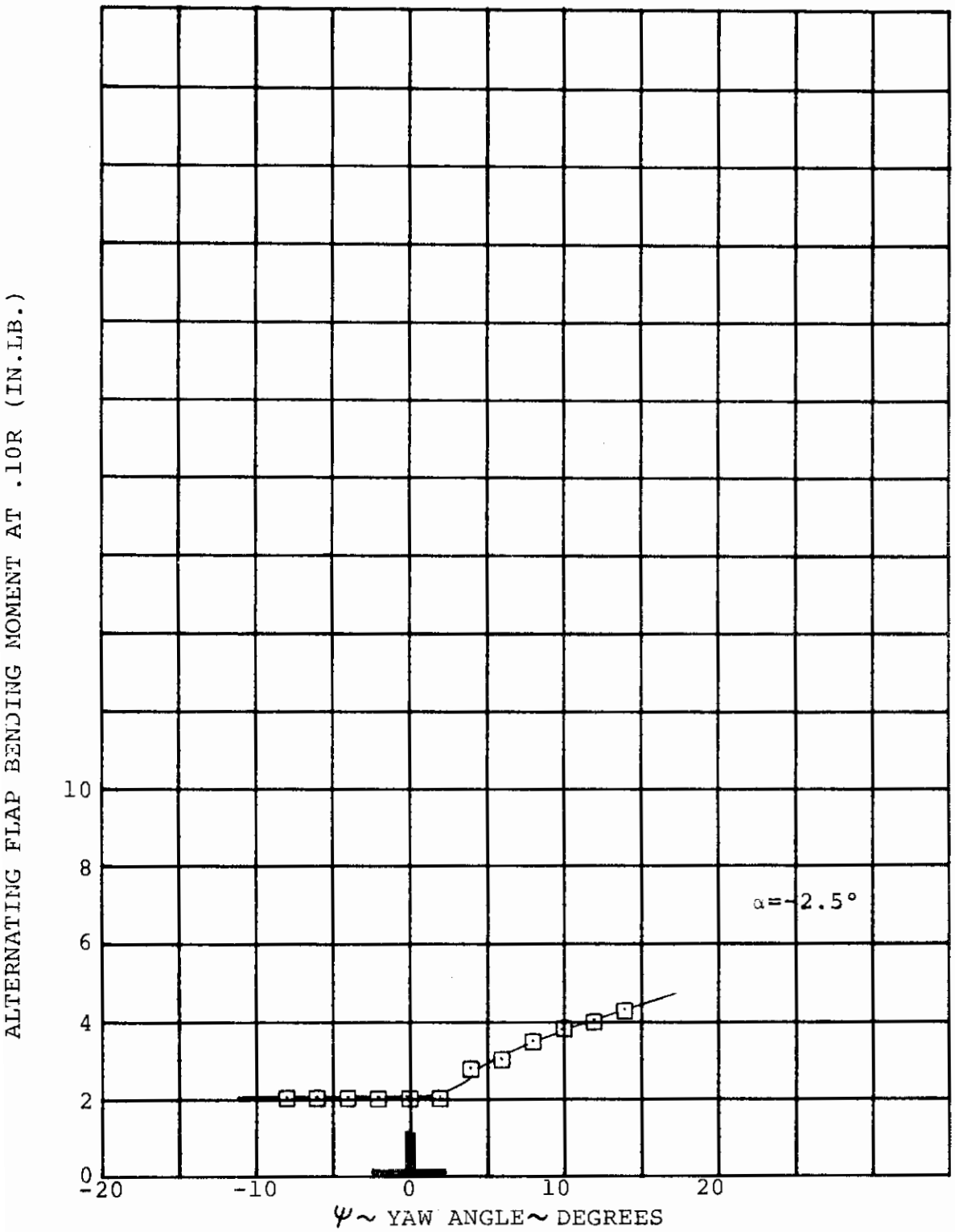


FIGURE 6-50. EFFECT OF MODEL YAW ANGLE ON RIGHT ROTOR ALTERNATING FLAP BENDING MOMENT IN CRUISE FOR i_N=0 DEG. $\Theta_{.75}$ =30.7 DEG., q= 7PSF, Θ_2= -0.7 DEG., AND 790 RPM. RUN 38(10)

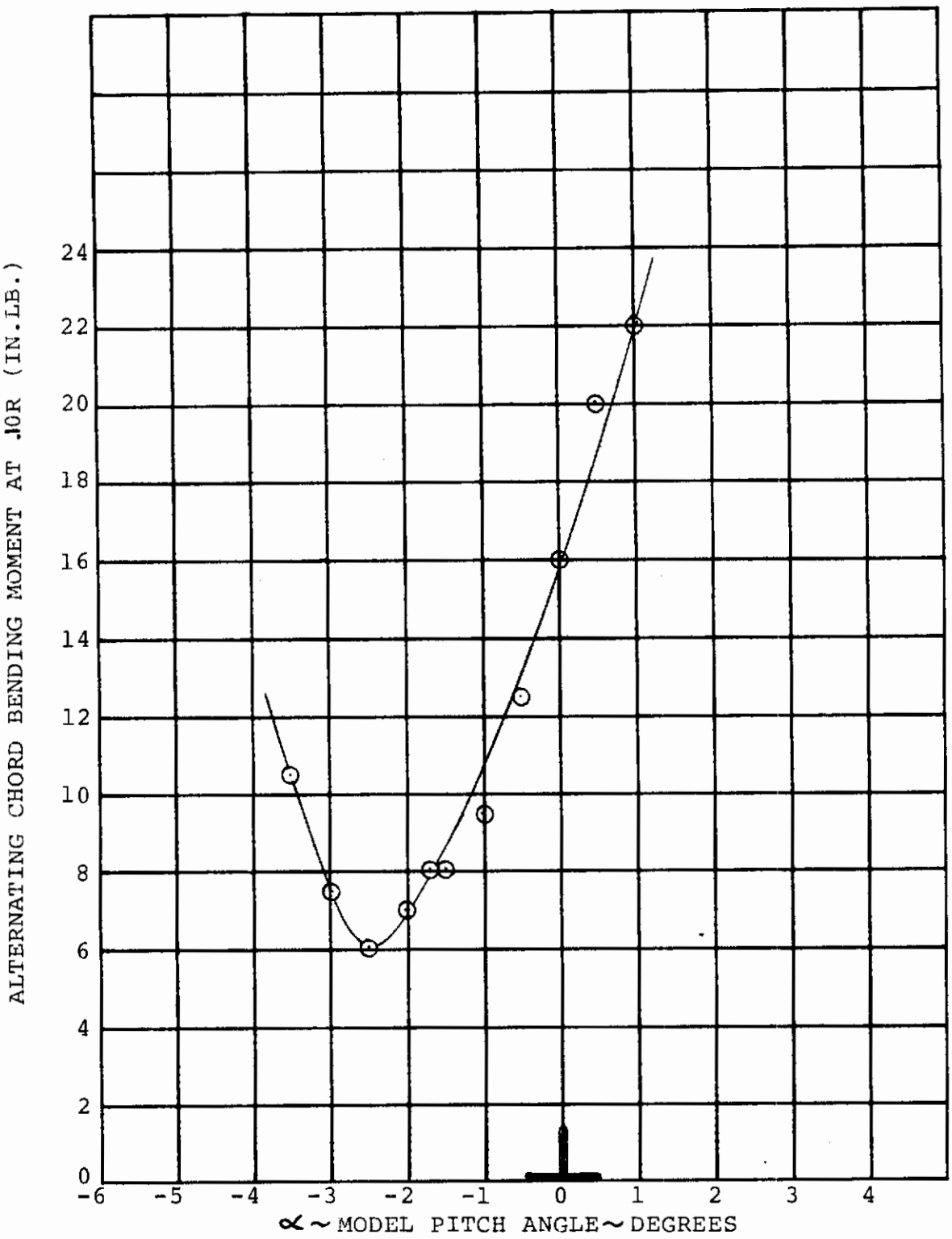


FIGURE 6-51. EFFECT OF MODEL PITCH ANGLE ON RIGHT ROTOR ALTERNATING CHORD BENDING MOMENT IN CRUISE FOR i_N=0 DEG. $\theta_{.75}$= 26.7 DEG.,q=6 PSF, θ_2=-0.7 DEG., AND 790 RPM.
RUN 38(3)

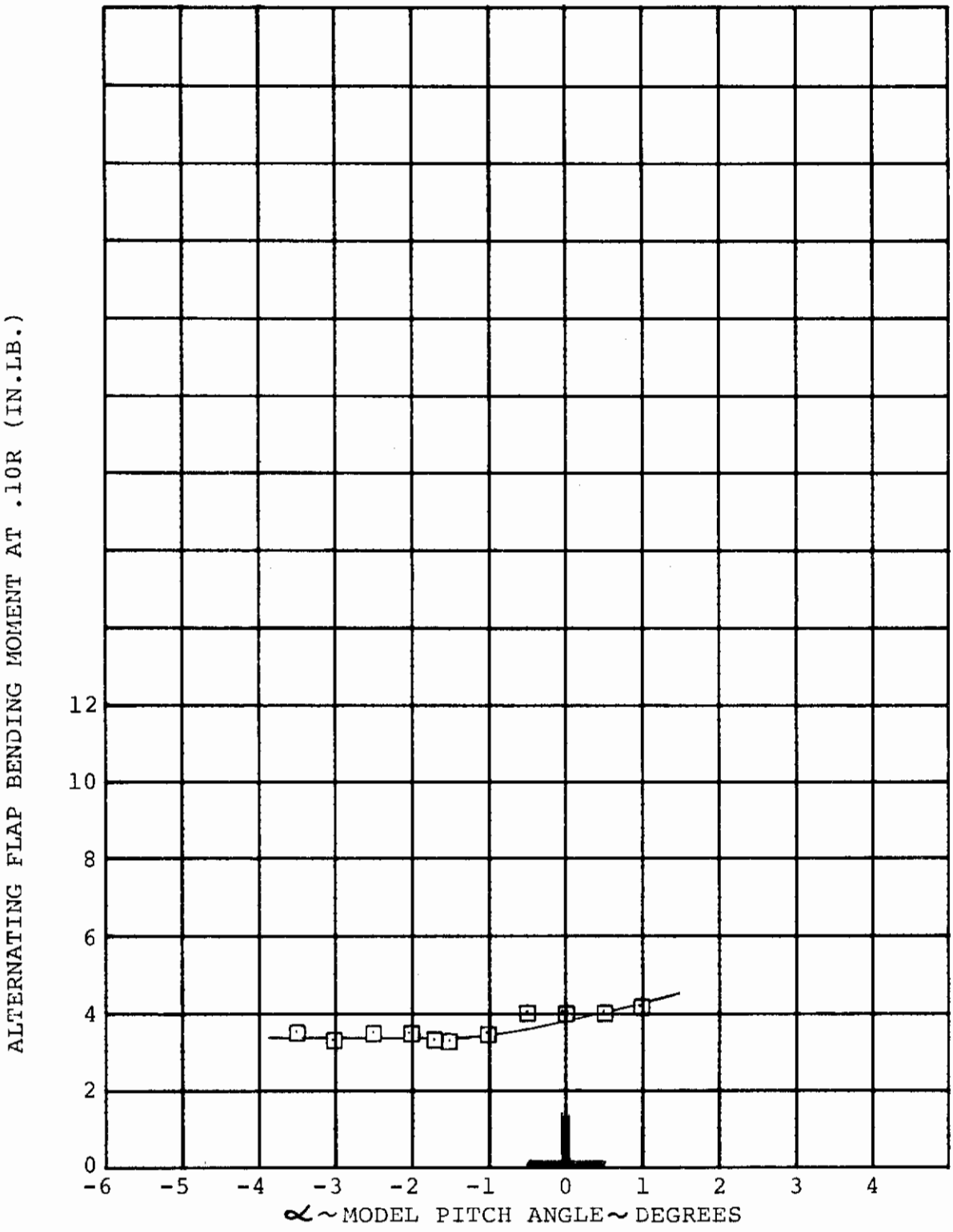


FIGURE 6-52. EFFECT OF MODEL PITCH ANGLE ON RIGHT ROTOR ALTERNATING FLAP BENDING MOMENT IN CRUISE FOR iN= 0 DEG. $\Theta_{.75}$=26.7 DEG., q = 6 PSF, Θ_2 = -0.7 DEG., AND 790 RPM. RUN 38(3)

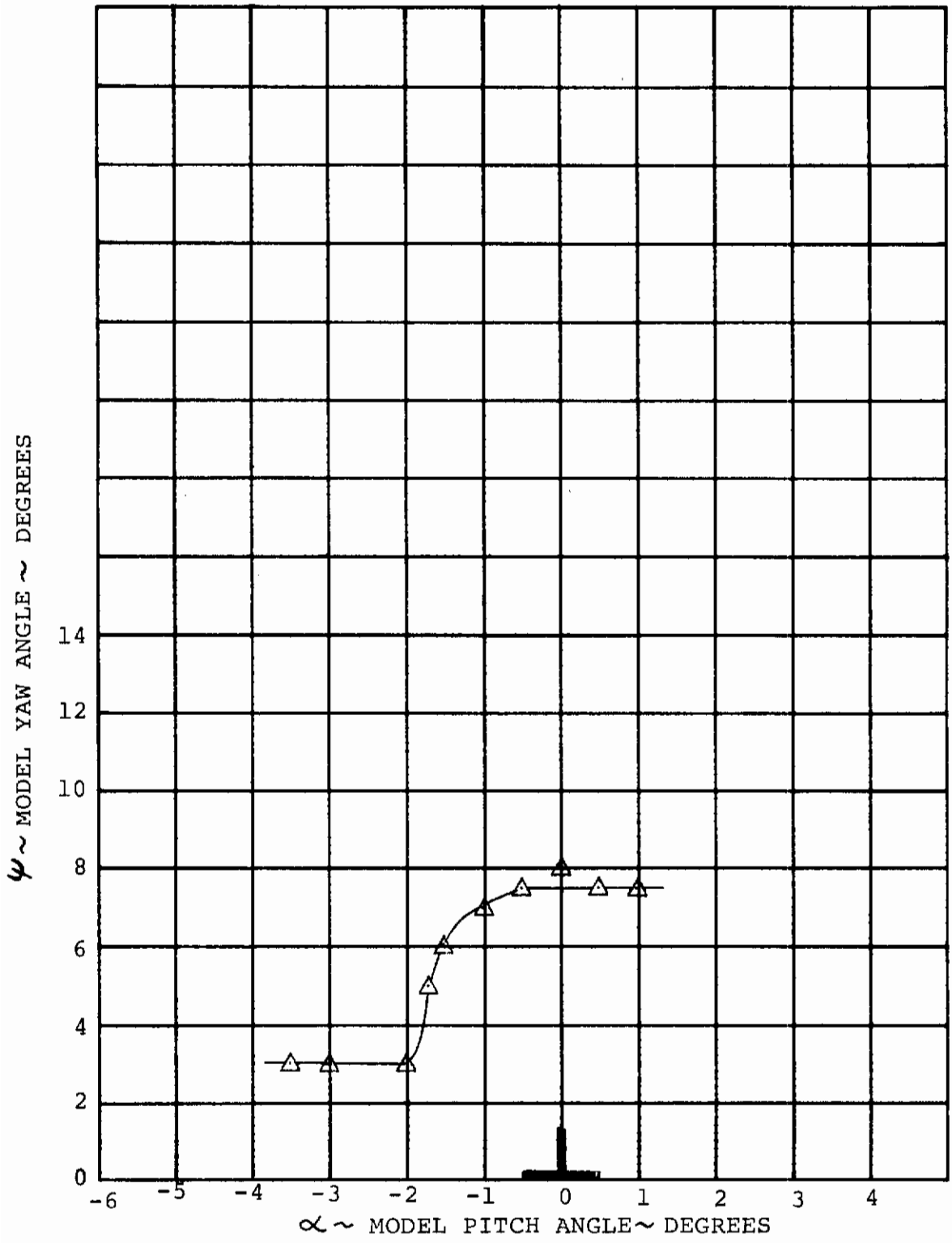


FIGURE 6-53. VARIATION OF MODEL YAW WITH MODEL PITCH ANGLE IN CRUISE FOR i_N = 0 DEG., $\Theta_{.75}$ = 26.7 DEG., q = 6 PSF, Θ_2 = -0.7 DEG., AND 790 RPM.

RUN 38(3)

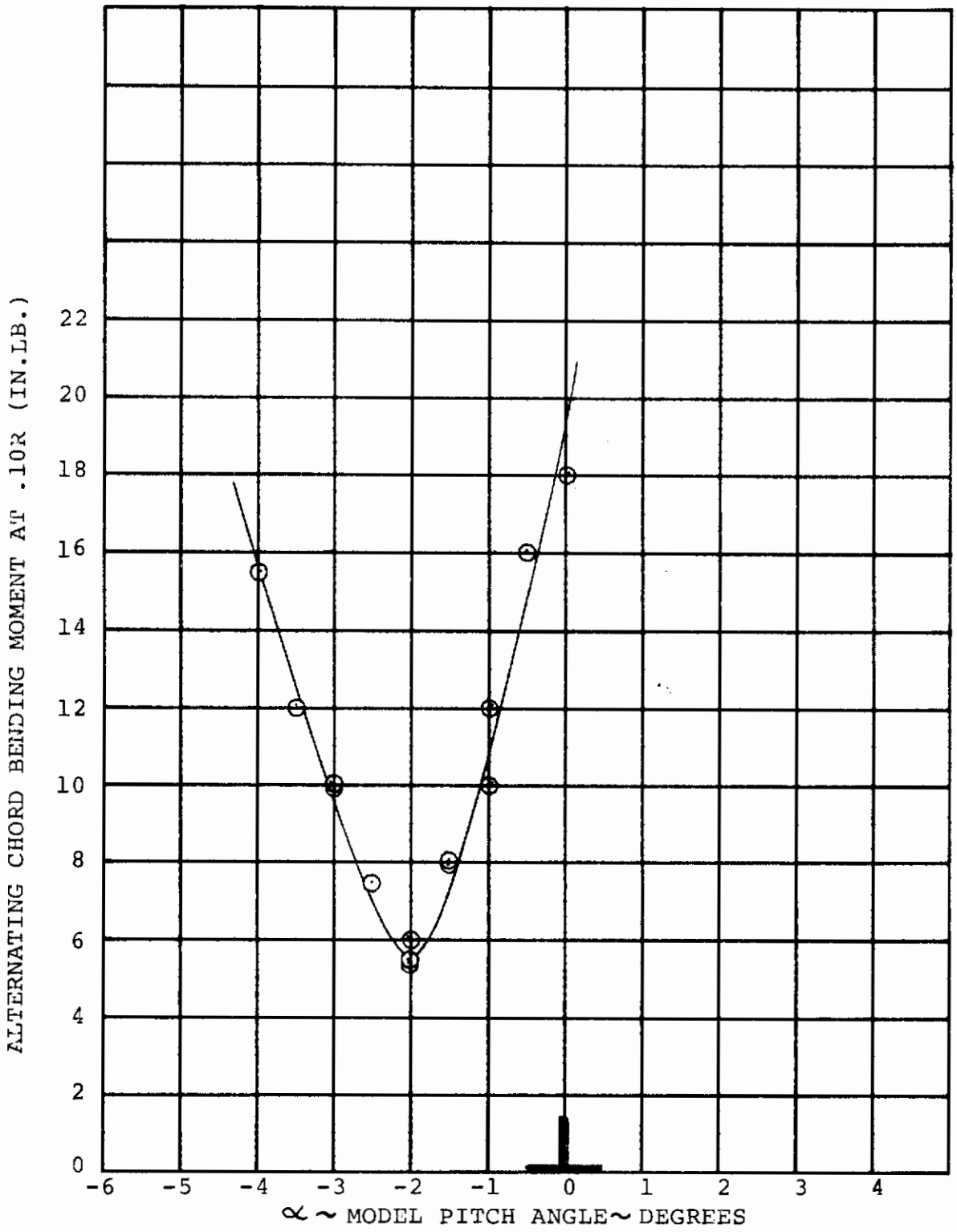


FIGURE 6-54. EFFECT OF MODEL PITCH ANGLE ON RIGHT ROTOR ALTERNATING CHORD BENDING MOMENT IN CRUISE FOR i_N =0 DEG. $\Theta_{.75}$= 30.7 DEG.,q= 7 PSF, Θ_2=-0.7 DEG., AND 790 RPM. RUN 38(9)

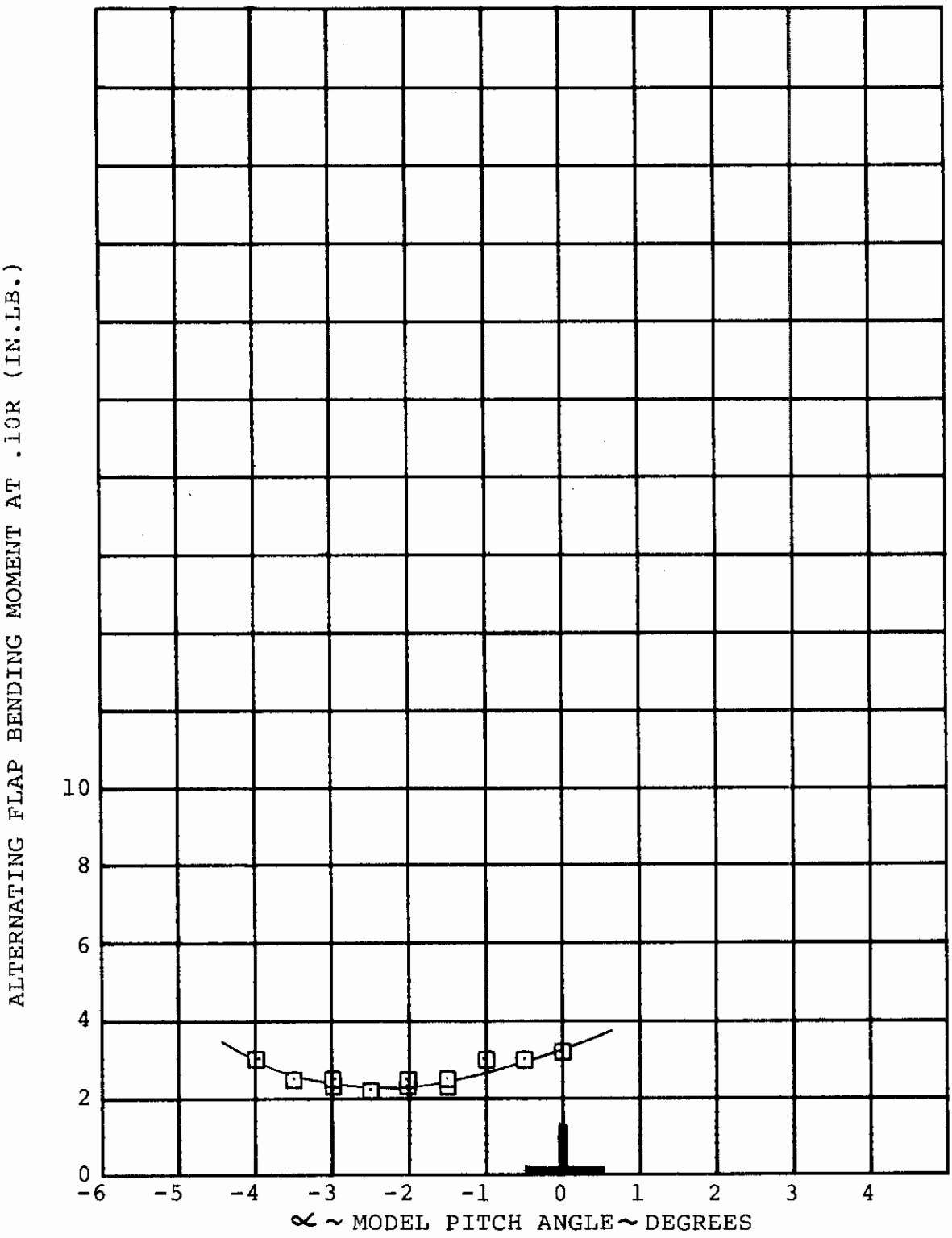


FIGURE 6-55. EFFECT OF MODEL PITCH ANGLE ON RIGHT ROTOR ALTERNATING FLAP BENDING MOMENT IN CRUISE FOR i_N = 0 DEG. $\Theta_{.75}$= 30.7 DEG.,q=7 PSF,Θ_2=-0.7 DEG., AND 790 RPM. RUN 38(9)

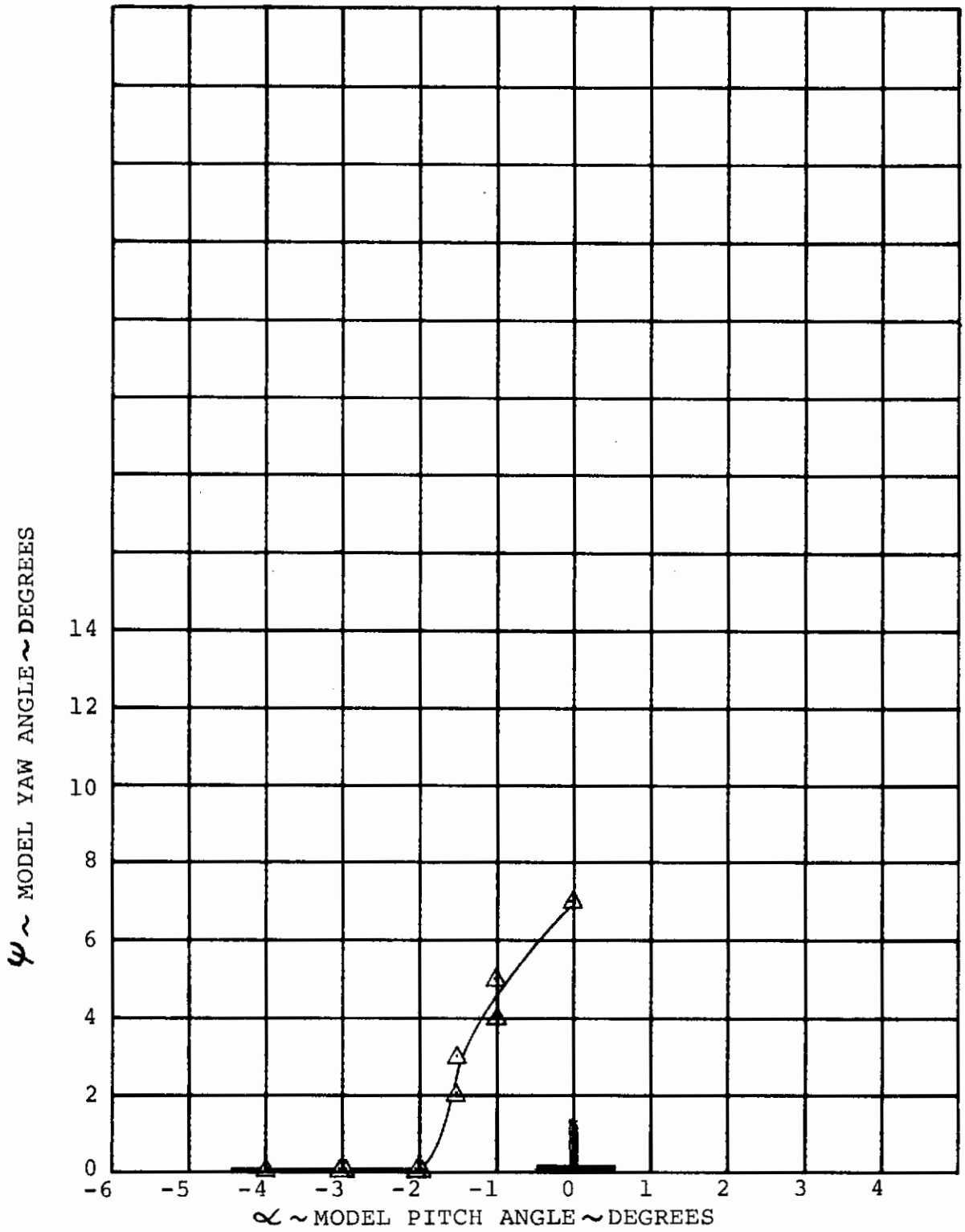


FIGURE 6-56. VARIATION OF MODEL YAW WITH MODEL PITCH ANGLE IN CRUISE FOR i_N= 0 DEG., $\theta_{.75}$= 30.7 DEG., q=7 PSF, θ_2=-0.7 DEG., AND 790 RPM.

RUN 38(9)

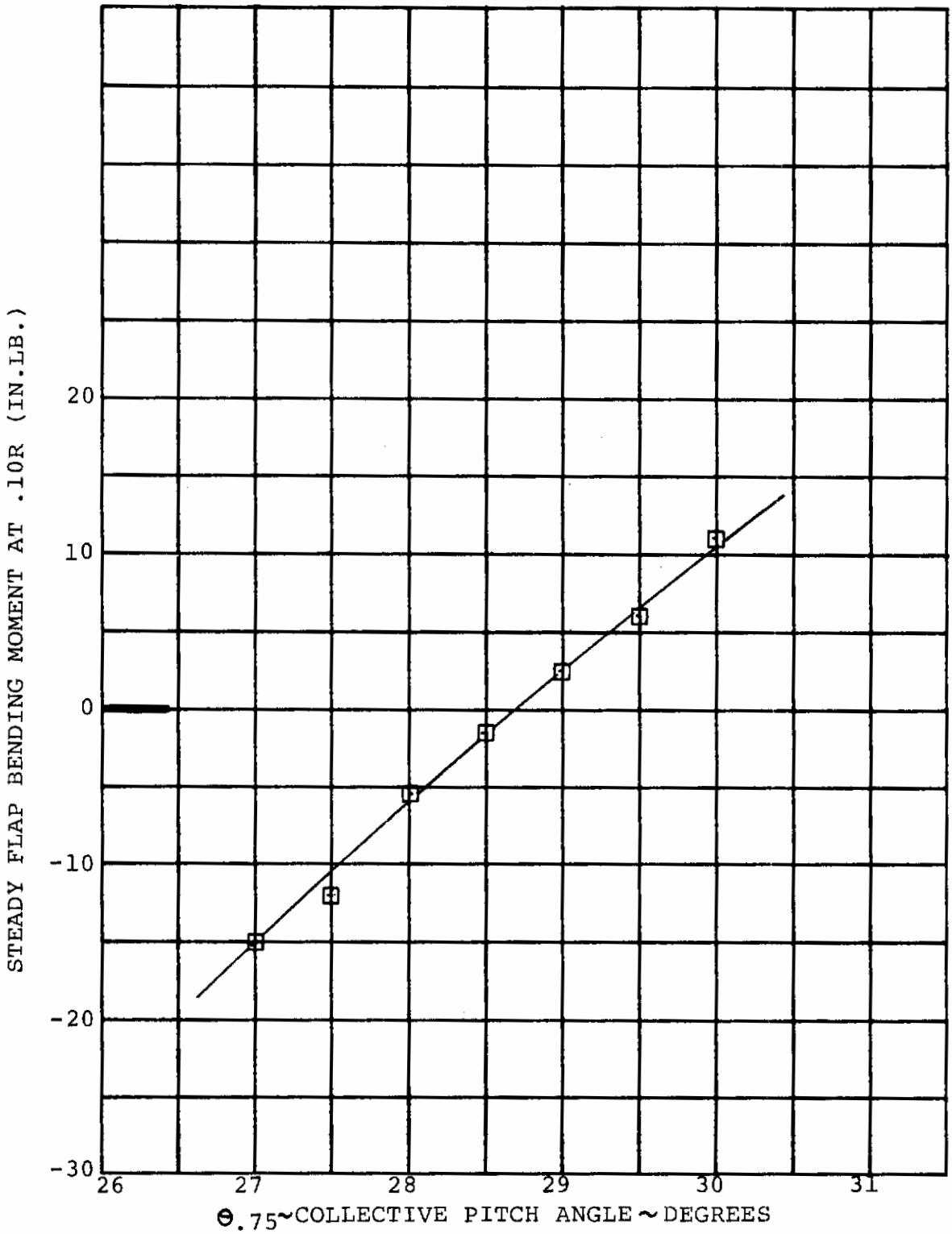


FIGURE 6-57. EFFECT OF COLLECTIVE PITCH ANGLE ON RIGHT ROTOR STEADY FLAP BENDING MOMENT IN CRUISE FOR i_N=0 DEG., q = 6 PSF, θ_2 = -0.5 DEG., AND 790 RPM.

RUN 39(41)

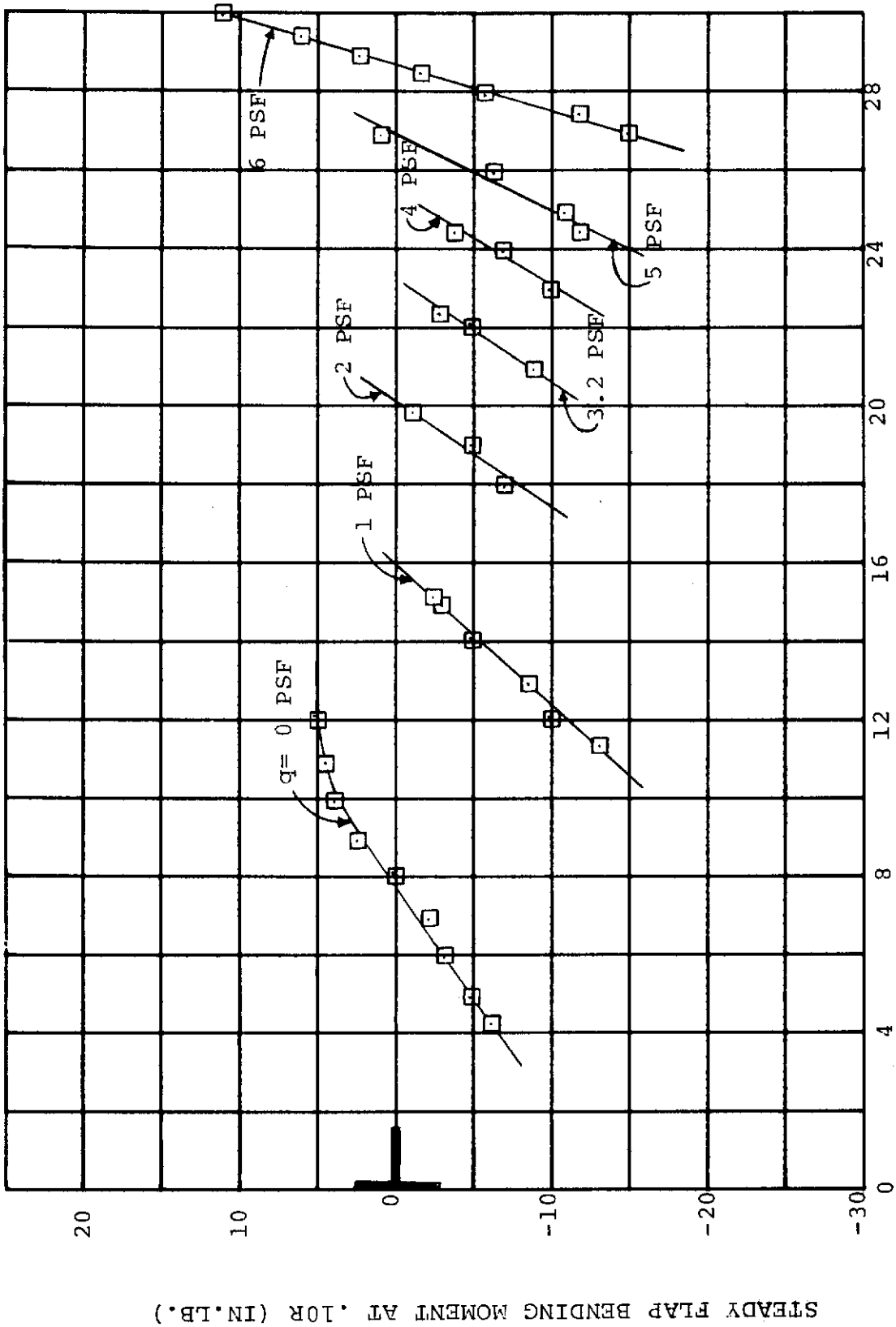


FIGURE 6-58. EFFECT OF COLLECTIVE PITCH ANGLE ON RIGHT ROTOR STEADY FLAP BENDING MOMENT IN CRUISE FOR i_N = 0 DEG., Θ_2 = -0.4 DEG., AND 790 RPM.

6.4 BLADE RESPONSE TO MODEL DISTURBANCE

Pitch disturbances were manually induced to the model under various conditions during the test. Resulting blade load responses are presented in Figure 6-59 and 6-60 for some typical cases representing hover and cruise conditions respectively. In the hover case (Figure 6-59) the disturbance caused the alternating load to momentarily increase with the peak load occurring at the time when the pitching motion was at its maximum rate, approximately 60°/sec.

The blade loads which resulted from a pitch disturbance in the cruise condition (Figure 6-60) were primarily due to angle of attack change and contained essentially no dynamic amplification. The model modes, both rigid body and blade, were heavily damped.

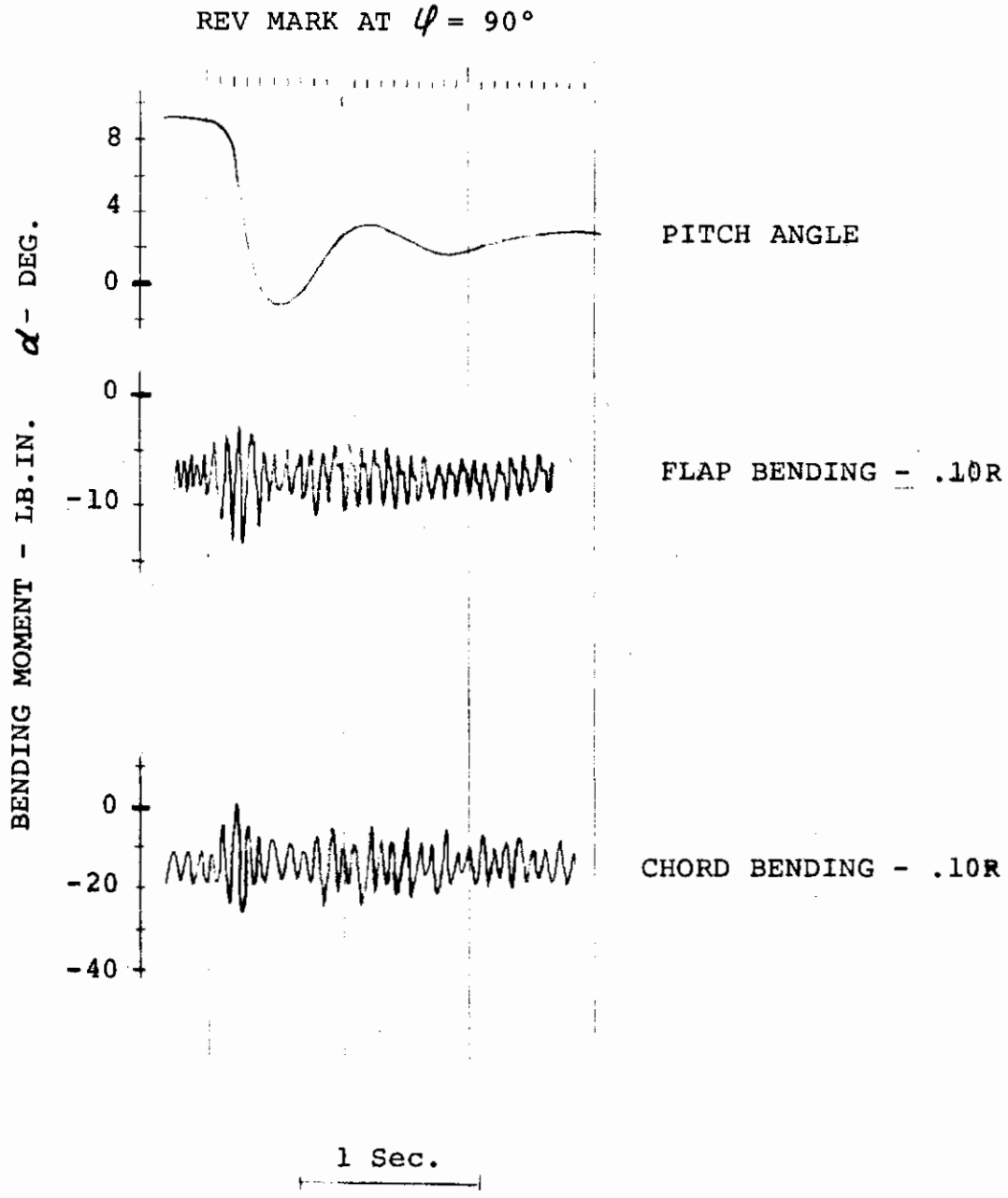


FIGURE 6-59. BLADE RESPONSE TO PITCH DISTURBANCE IN HOVER, $\theta_{.75}$= 6.0DEG., θ_2= -0.5 DEG., Ω = 830 RPM, RUN 65

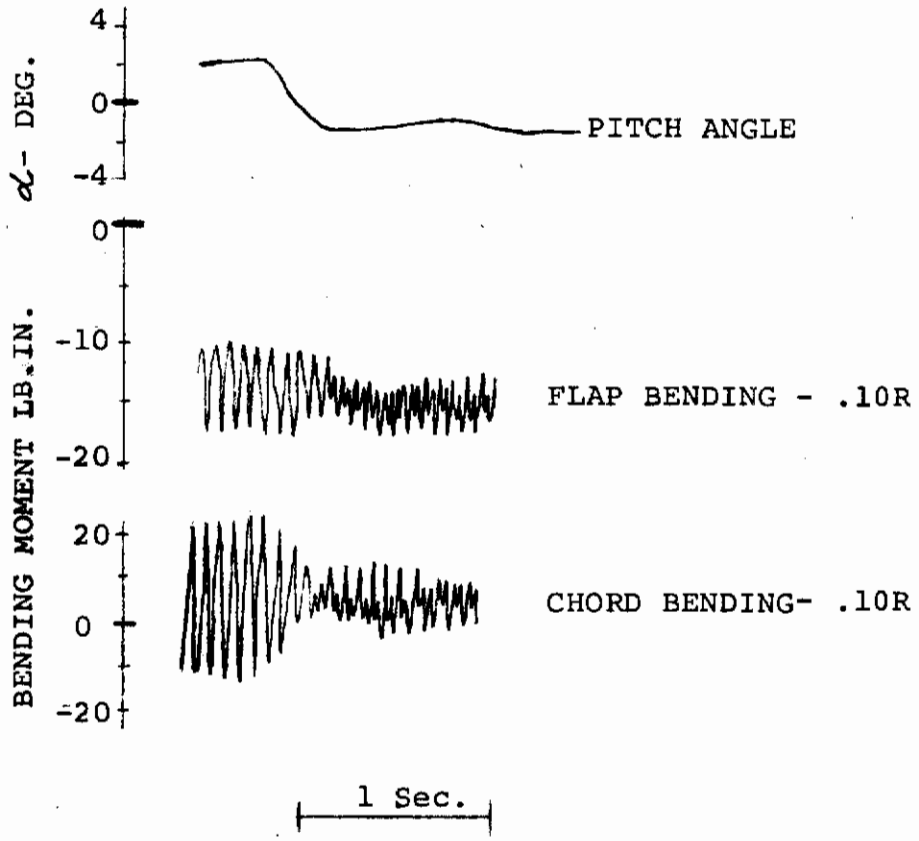


FIGURE 6-60 BLADE RESPONSE TO PITCH DISTURBANCE IN CRUISE ATTITUDE: $\theta_{.75}$ = 28. DEG., Ω = 790 RPM, q = 7.0 PSF, i_w =0 DEG., RUN 50

7.0 VEHICLE DYNAMICS

7.1 PREDICTED FREQUENCY AND DAMPING SPECTRUM

Figures 7-1 through 7-5 present the predicted modal frequencies (f vs Ω) and damping (% critical vs Ω) curves. The circled numbers in the figures serve only to identify the frequency roots without specific order. As noted, the modal characteristics of a given frequency root can change considerably with rotor speed. It can be seen in Figure 7-1 that the root "one" response is predominantly wing torsion at low rotor speeds and changes to predominantly blade response above 600 rpm (damping spectrum plots are associated to the frequency plots by root number). Representative tweak and shake test frequency results are shown at zero rpm on Figures 7-1 and 7-3. Correlation of the observed test frequencies with predictions, in general is good. Comparison of Figures 7-2 and 7-3 shows that the predicted damping values have significant dependence upon collective for this model.

7.2 VEHICLE FREQUENCIES

7.2.1 Rotors Non-Rotating

Modal frequencies and damping for the non-rotating system were obtained from tweak (initial displacement) and shake tests. Test results of the tweak test for the symmetric condition only at various nacelle tilt angles, encompassing cruise through hover, are shown in Table 7-1.

TABLE 7-1

WING FREQUENCIES AND DAMPINGS FROM TWEAK TEST

NACELLE INCIDENCE	SYM.WING FLAP		SYM.WING CHORD		SYM.WING TORSION	
	FREQ. (CPS)	DAMPING ζ	FREQ. (CPS	DAMPING ζ	FREQ. (CPS)	DAMPING ζ
0° (Cruise)	5.2	.022	10.5	.012	16.8	.010
40°	5.1	.041	10.5	.015	17.0	X
60°	5.1	.032	10.3	.017	X	X
90° (Hover)	5.1	.033	10.5	.017	17.5	.012

The effect of tilt angle on frequency was found to be negligible; however, wing flap modal damping varied to some extent.

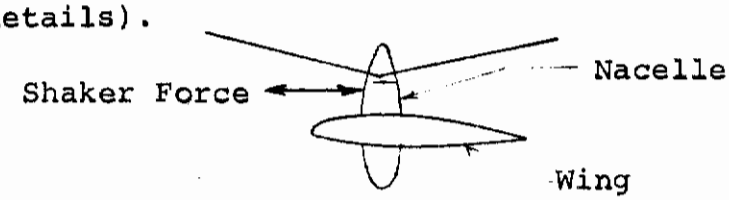
The symmetric and antisymmetric root bending moment frequency response curves from the hover configuration shake tests are shown by Figures 7-6 through 7-11. An index of the data is shown in Table 7-2. A response peak which is predominantly wing flap bending, occurs at a shaker frequency of 6 cps. The response at 9.3 cps is predominantly wing symmetric chord bending with considerable coupling with wing symmetric torsion. The blade chordwise bending frequency at $\Omega=0$ is also in this vicinity. The response near 17.5 cps is wing torsion. No test data is available between shaker frequencies of 10.5 cps and 14 cps for symmetric excitation. Data from tests conducted in this range was incorrect due to a malfunction in the computer analysis system and the test response data was irretrievable.

TABLE 7-2

SHAKE TEST INDEX - ROTORS NON-ROTATING

FIGURE	TYPE OF EXCITATION	FREQUENCY RESPONSE CURVE
7-6	Symmetric	Wing Flap Bending
7-7	Symmetric	Wing Chord Bending
7-8	Symmetric	Wing Torsion
7-9	Antisymmetric	Wing Flap Bending
7-10	Antisymmetric	Wing Chord Bending
7-11	Antisymmetric	Wing Torsion

The shake test was performed employing two air jet shakers, one mounted on each nacelle. These shakers were mounted approximately 4 inches ahead of the wing elastic axis and provided excitation normal to the nacelle as shown below (see Figure 3-6 for details).



The phase between the left and right shakers was adjusted to provide both symmetric and antisymmetric excitation to the wings, with the frequency of excitation varying from zero to 20 cps.

7.2.2 Rotating Rotors

Figures 7-12 through 7-17 show the wing symmetric root bending moment frequency response curves from the cruise and hover configuration shake tests; and an index of the data is shown in Table 7-3. The rotor speed for these shake tests was 790 RPM and 825 RPM respectively.

TABLE 7-3

SHAKE TEST INDEX -- ROTATING ROTORS

FIGURE	NACELLE POSITION	FREQUENCY RESPONSE CURVE
7-12	Cruise	Wing Flap Bending
7-13	Cruise	Wing Chord Bending
7-14	Cruise	Wing Torsion
7-15	Hover	Wing Flap Bending
7-16	Hover	Wing Chord Bending
7-17	Hover	Wing Torsion

An examination of the rotating data indicates that the wing bending frequencies are not significantly changed from the non-rotating ones. The large peak around 13.5 cps is predominantly the 1st harmonic of the rotor excitation caused by an unbalance in the rotor system. A summary of the wing natural frequencies determined from tweak test, shake test and analysis is shown in Table 7-4.

TABLE 7-4

WING FREQUENCY SUMMARY

TWEAK TEST (SYM)	SHAKE TEST				NON-ROTATING CALCULATED		MODE
	SYM.		ANTI-SYM.				
	ROTAT.	NON ROTAT.	ROTAT.	NON ROTAT.	SYM	ANTI-SYM	
5.1CPS	6.9	6.0	*	*	4.6	>20	Flap Bending
10.5	10.0	9.0	*	*	9.1	>20	Chord Bending
17.5	17.2	17.5	-	17.5	18.0	15.4	Torsion

*Not observed during the shake test and predicted to be above the maximum shaker frequency.

7.3 ELASTIC MODE STABILITY

The model was run through an equivalent full scale nacelle tilt/airspeed envelope as shown in Figure 3-8 to check for whirl flutter, divergence, classical flutter or air resonance. There was no evidence of any of these phenomena throughout the test range. The presence of substantial 1/rev loads due to model unbalance and the limited force available from the shaker made it impossible to obtain good quantitative modal damping data, but there was clearly no tendency for any of these modes to be self excited or to persist if excited by abrupt airplane motions or by the shaker unit in the natural frequency tests described in Section 7-1. A model response occurred associated primarily with blade chord bending under conditons near zero thrust in hover and at low velocities in transition. In some cases a limit cycle oscillation developed. This response did not occur at the higher tunnel speeds tested in the cruise mode.

The mode was always sufficiently stable that a substantial volume of test data could be taken at the conditions where it was encountered. These data are presented and the phenomenon is discussed in more detail under Item 7.3.2.

7.3.1 Input Disturbances

The model was disturbed with the snubber cables from its trim position at various conditions throughout the test to determine its rigid body and elastic modal response. The rigid body data were obtained by gradually displacing the model from trim and then releasing the cables quickly. In all cases throughout the test the rigid body modes were very stable. Results are discussed in Section 8.0. Typical pitch and roll responses are shown in Figure 8-1.

7.3.2 Blade Chord Bending Mode Oscillations

As mentioned under 7.3 there were some conditions where a mode, associated primarily with blade chord bending, was lightly damped or developed a limit cycle oscillation. As can be seen from Figure 7-18 the conditions at which the oscillation occurred followed closely the conditions for zero thrust. The data points shown by circles (indicating zero thrust conditions) were obtained from Figure 6-59. A typical oscillograph tape showing the nature of the mode as a limit cycle oscillation is given in Figure 7-19.

The primary characteristic is the blade chord bending trace, which shows a substantial .72 per rev (blade chord bending natural frequency) superimposed on the normal one per rev. This shows up as the .28 per rev beat visible in the trace. The blade flap bending and torsion show the same frequencies though with less amplitude, while wing torsion (fixed system) shows a corresponding 1-.72=.28 per rev superimposed on the one per rev. Aircraft pitching motion also shows a small response at the same .28 per rev frequency. There is very little response at this frequency in wing chord bending or wing torsion.

Correlation of the predicted $\Omega - \omega_1$ mode damping with some limited test data is presented in Figure 7-20. The same type of low frequency beat response was observed over a range of conditions during transition and cruise testing. Boundaries for occurrence of the oscillations are given in Figures 7-21 and 7-22 by showing the regions of non-occurrence, in low damping and limit cycle. Additional data are desirable to define these boundaries more fully.

The beat oscillation could also be found during a rapid shutdown, and the beat frequency used as an additional check on rotating blade chordwise frequencies.

During the shutdown period of Run 40, the rotor was windmilling and beat frequencies occurred which changed with rotor speed (controlled by q). These beat frequencies are shown in the lower portion of Figure 7-29 and when added to the rotor speed ($\Omega + \omega$ beat) below the resonant crossover and subtracted from the rotor speed ($\Omega - \omega$ beat) above the crossover they appear as shown in the frequency spectrum in the upper part of Figure 7-29. Note the correlation with the baffle test and predicted frequencies. Note also that the crossover decreased from the 550 rpm baffle test to 510 rpm. This corresponds almost exactly with the decrease in chord bending tweak frequency which occurred from the start of testing to Run 40 as noted in Figure 4-1 due apparently to monocoat deterioration.

This type of oscillation has been found also on some helicopter models and on one other tilt rotor model (a stiff model with in-plane and out-of-plane frequencies of about 2.2Ω and 1.5Ω respectively). Theory indicates that it is generally associated with large initial out-of-plane deflections of the blade, especially in the negative thrust direction. This is confirmed by the fact that this model, which had 5^{o} precone experienced the instability only near zero thrust, where the blades are bent back from their 5^{o} precone position by centrifugal force. On a full scale aircraft it will be necessary to give careful consideration to this type of instability in selecting the precone angle.

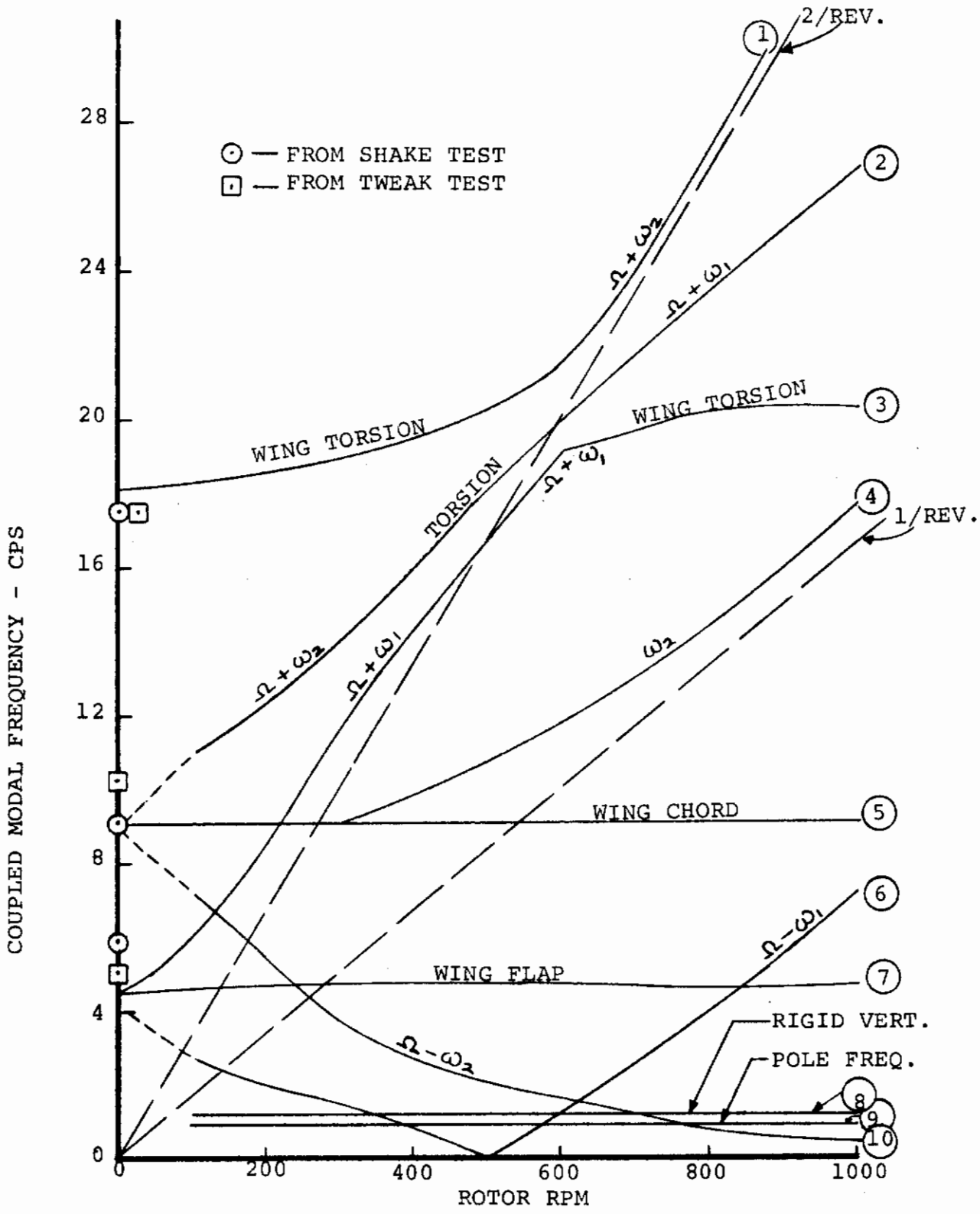


FIGURE 7-1. FREQUENCY SPECTRUM FOR SYMMETRICAL HOVER MODES

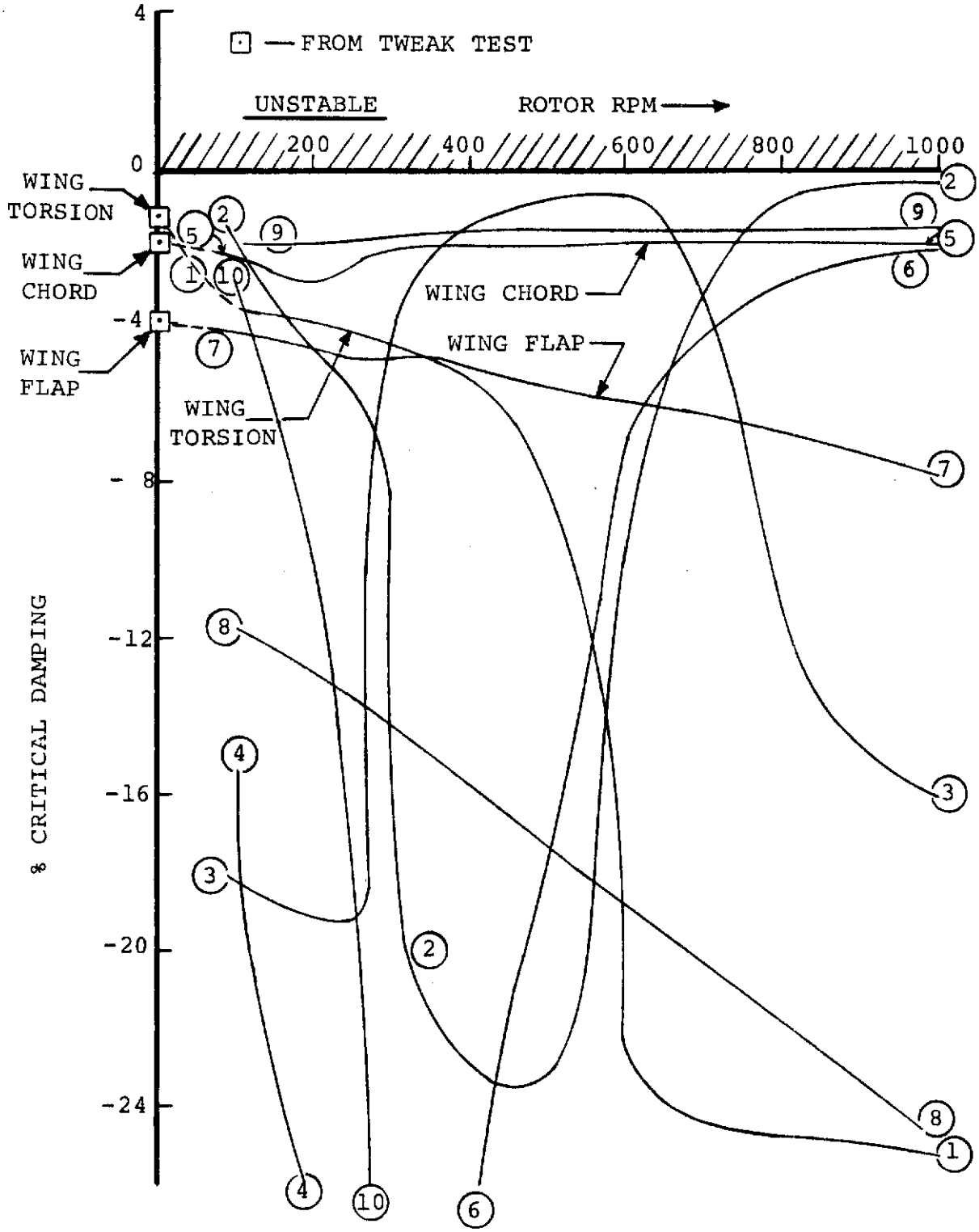


FIGURE 7-2. DAMPING SPECTRUM FOR SYMMETRICAL HOVER MODES, $\Theta_{.75}$=11 DEG.

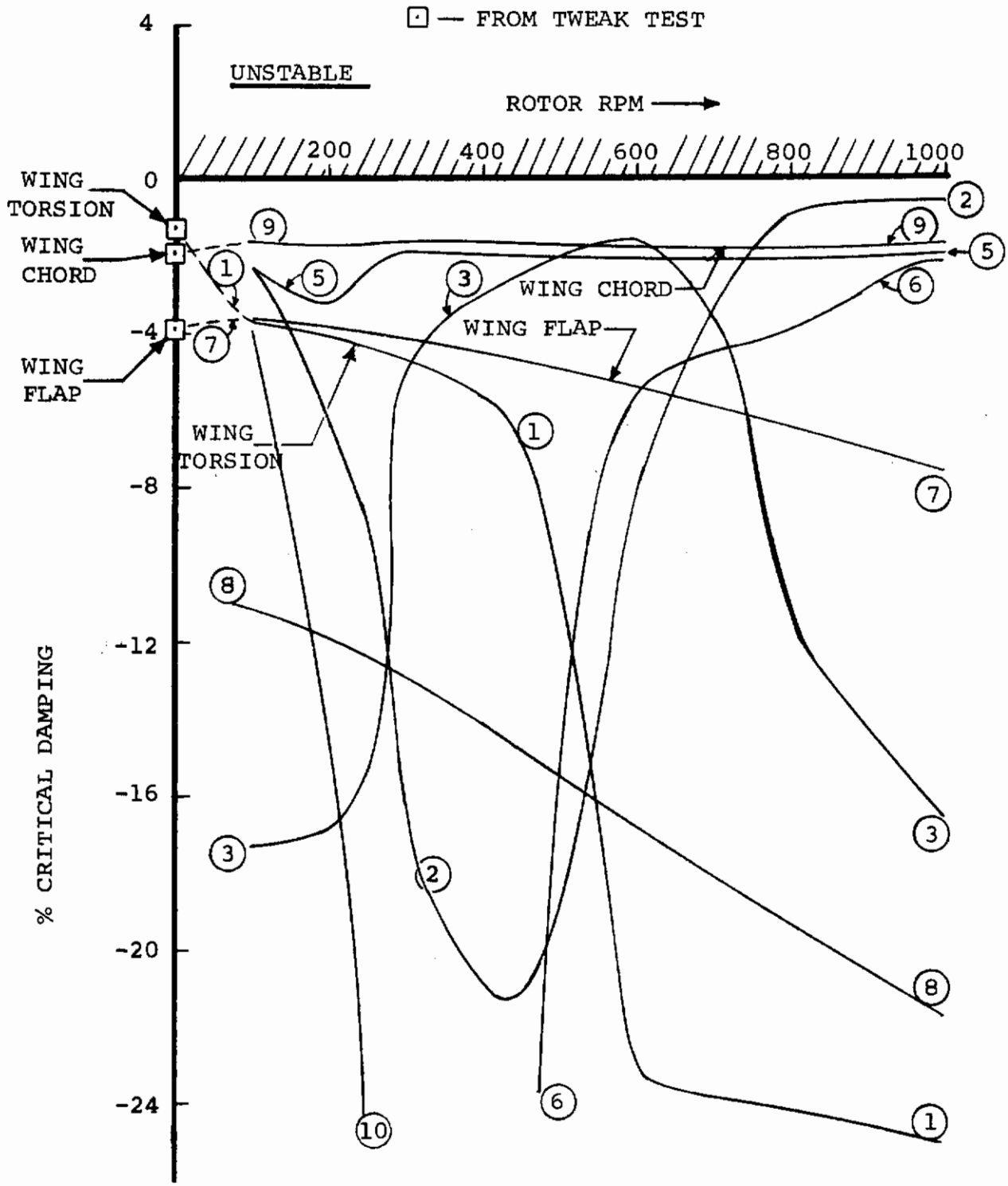


FIGURE 7-3. DAMPING SPECTRUM FOR SYMMETRICAL HOVER MODES ($\theta_{.75} = 0$)

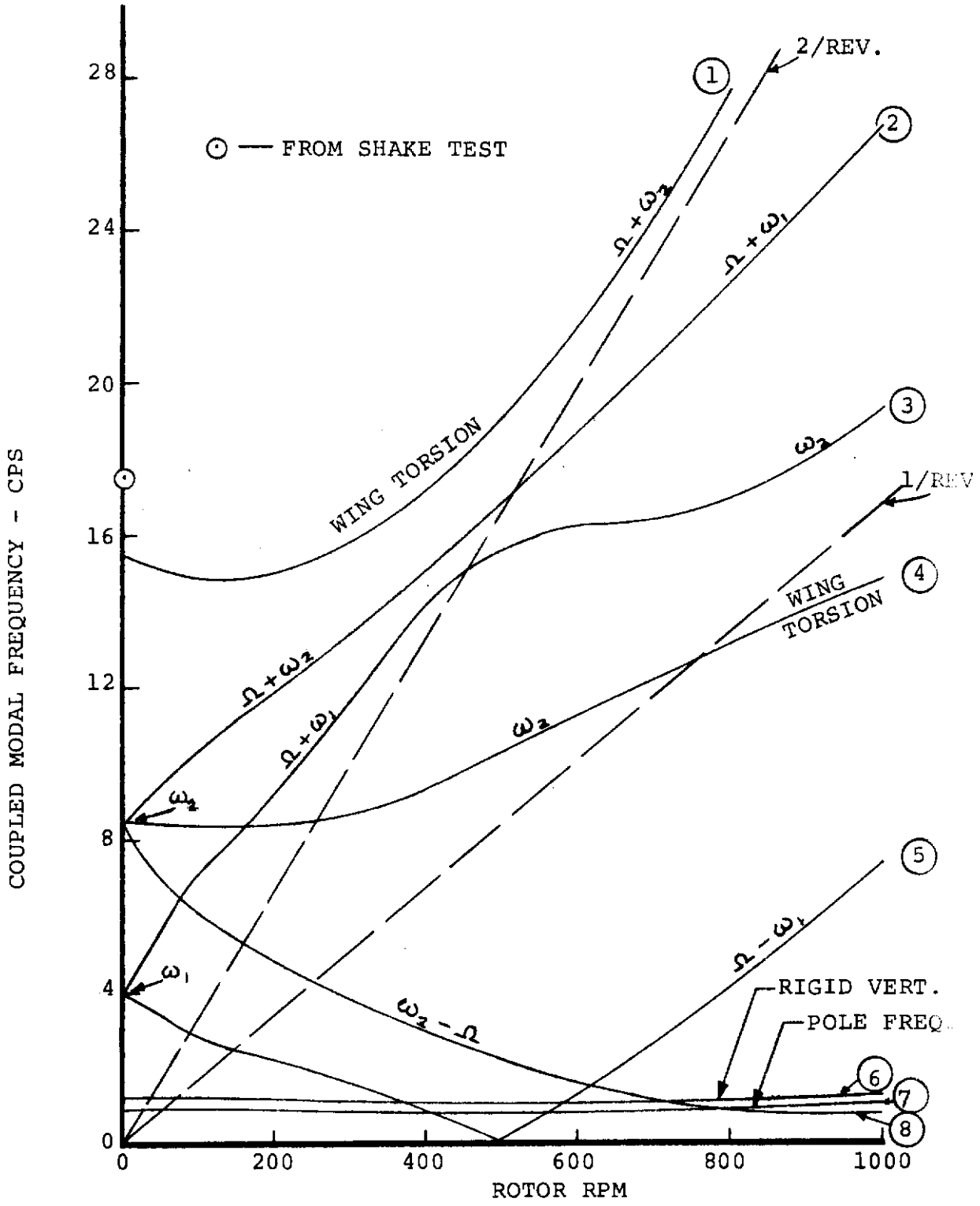


FIGURE 7-4. FREQUENCY SPECTRUM FOR ANTI-SYMMETRICAL HOVER MODES ($\theta_{.75} = 0$)

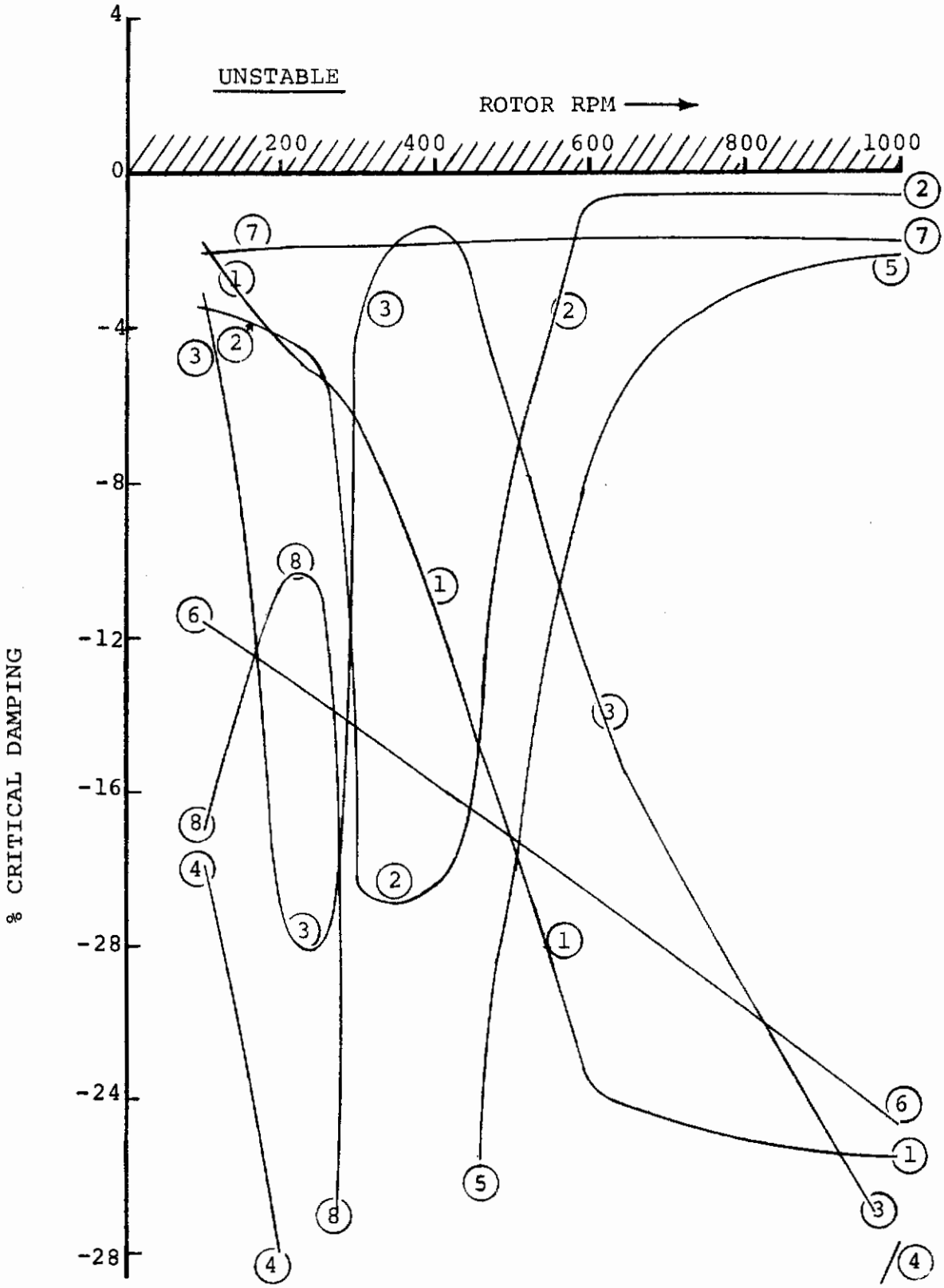


FIGURE 7-5. DAMPING SPECTRUM FOR ANTI-SYMMETRICAL HOVER MODES ($\theta_{.75}$ = 0)

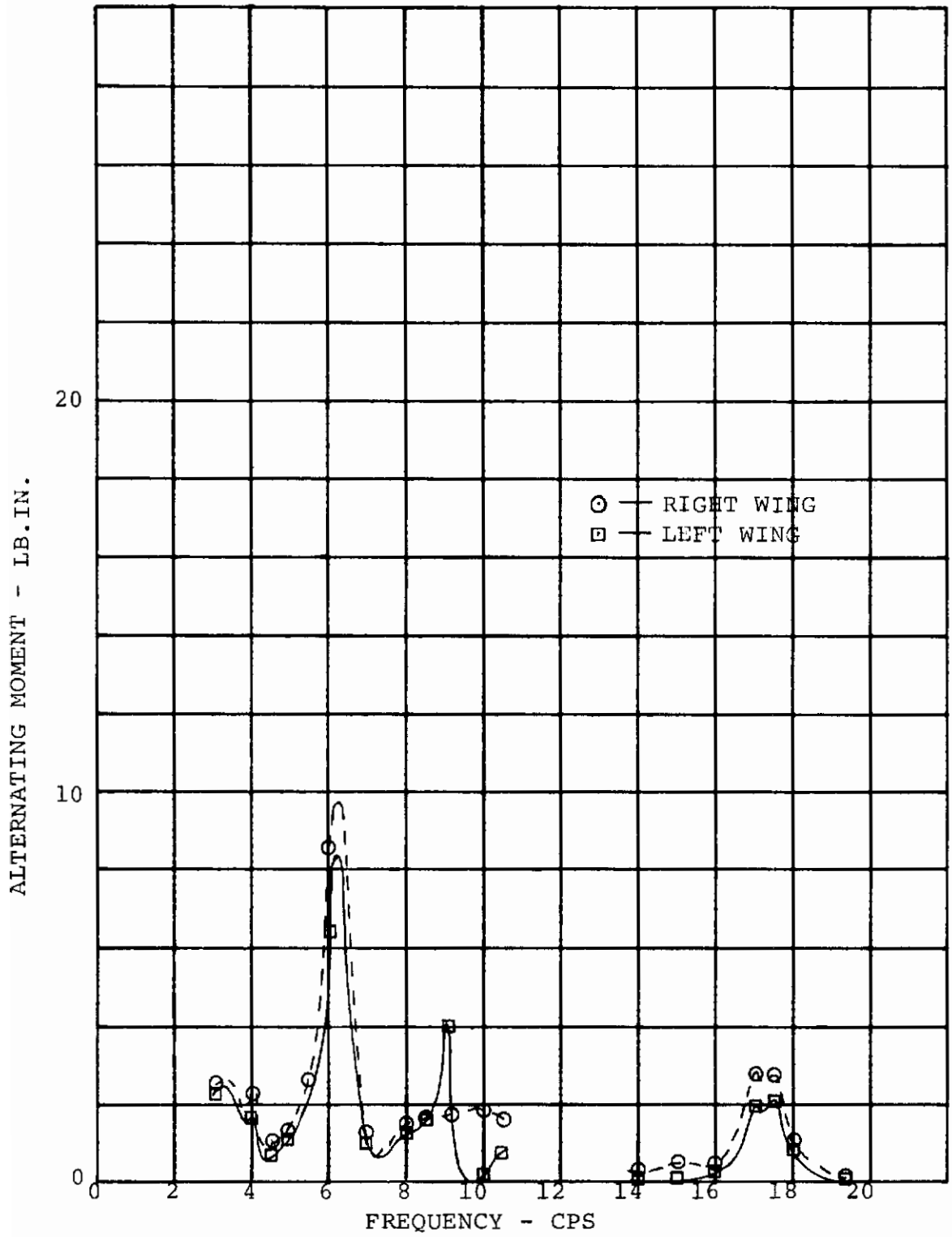


FIGURE 7-6: WING FLAP BENDING RESPONSE TO SYMMETRICAL EXCITATION-NON-ROTATING BLADES, i_N =90DEG.

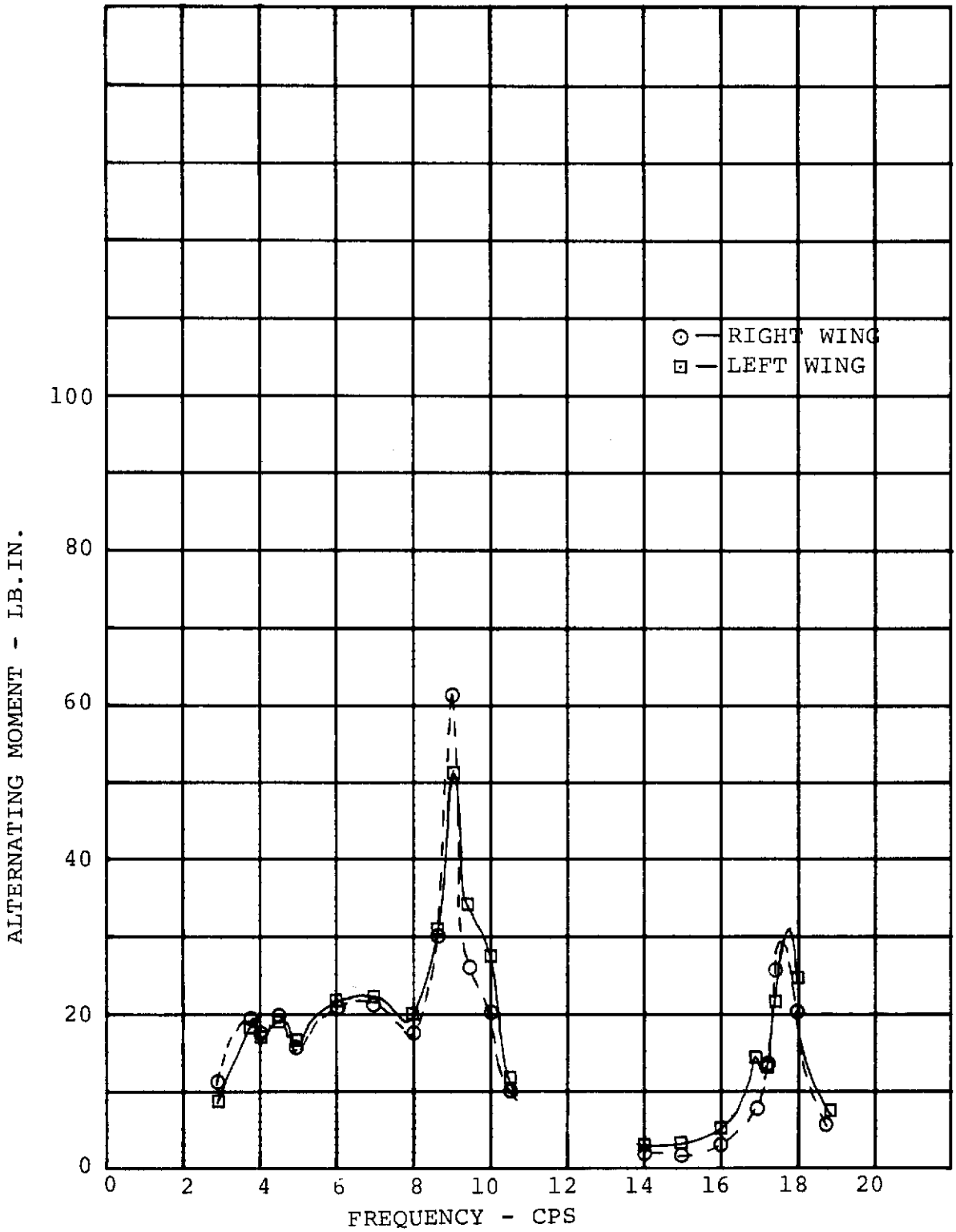


FIGURE 7-7: WING CHORD BENDING RESPONSE TO SYMMETRICAL EXCITATION-NON-ROTATING BLADES, i_N = 90 DEG.

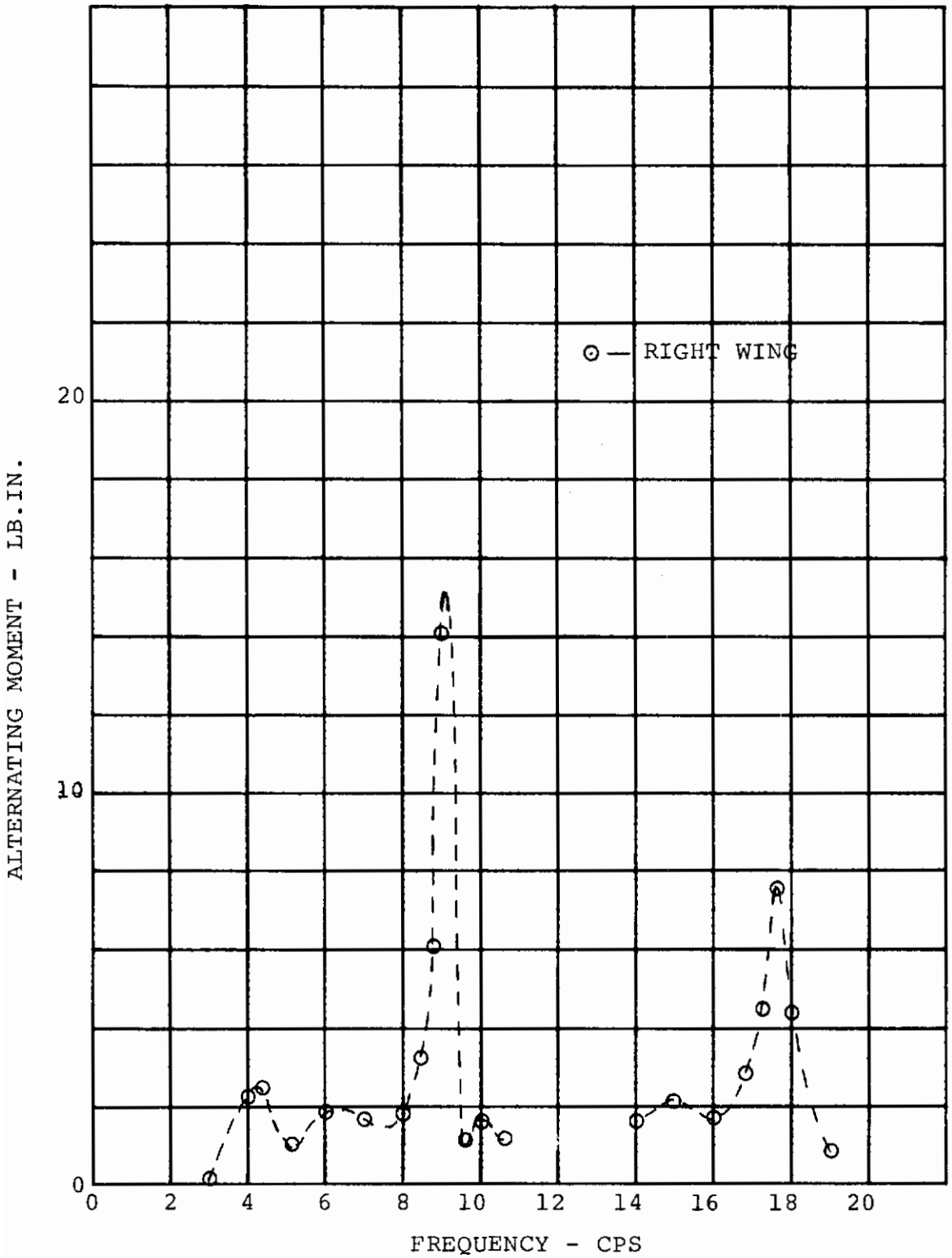


FIGURE 7-8: WING TORSION RESPONSE TO SYMMETRIC EXCITATION NON-ROTATING BLADES, i_N = 90 DEG.

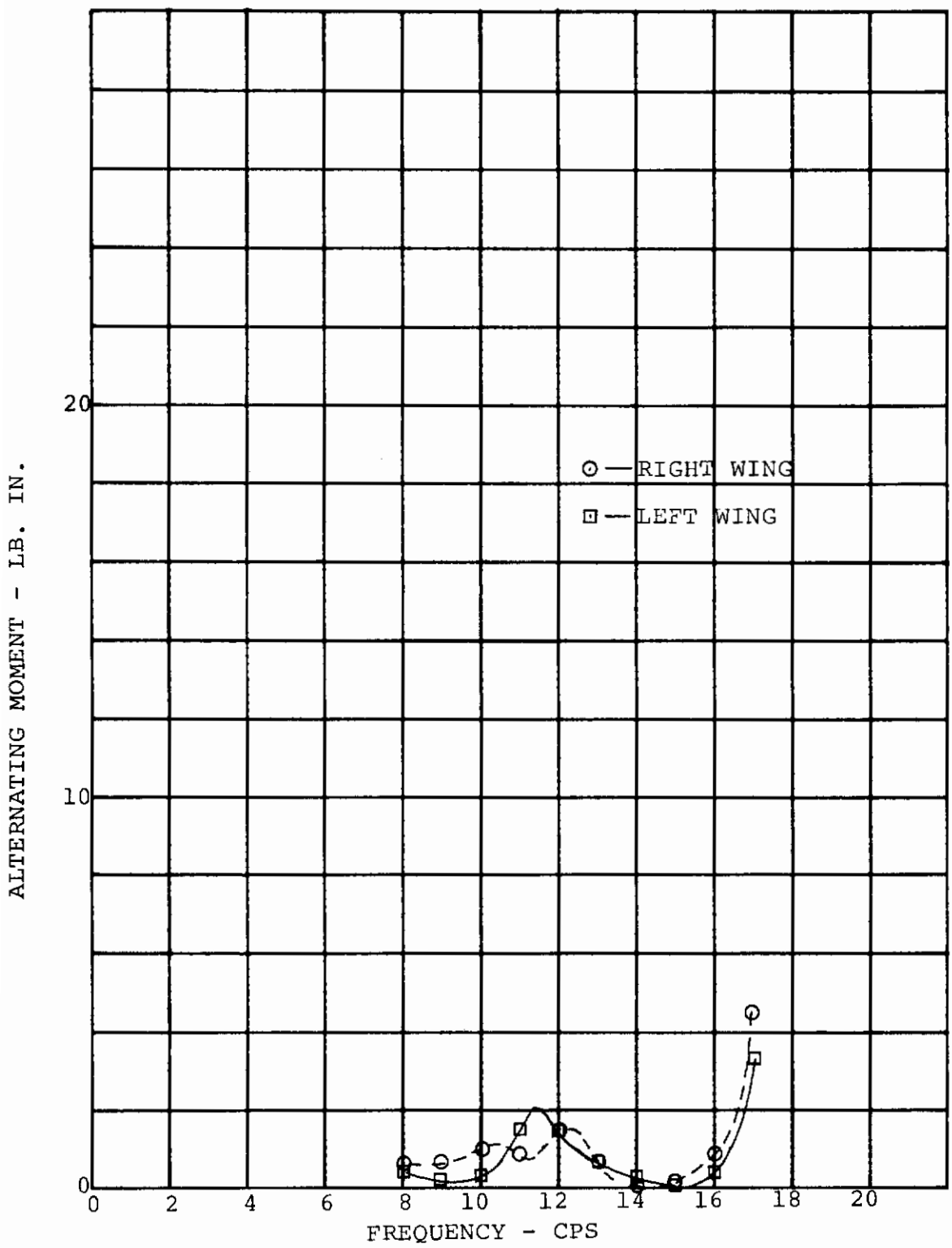


FIGURE 7-9: WING FLAP BENDING RESPONSE TO ANTI-SYMMETRIC EXCITATION-NON-ROTATING BLADES, i_N = 90 DEG.

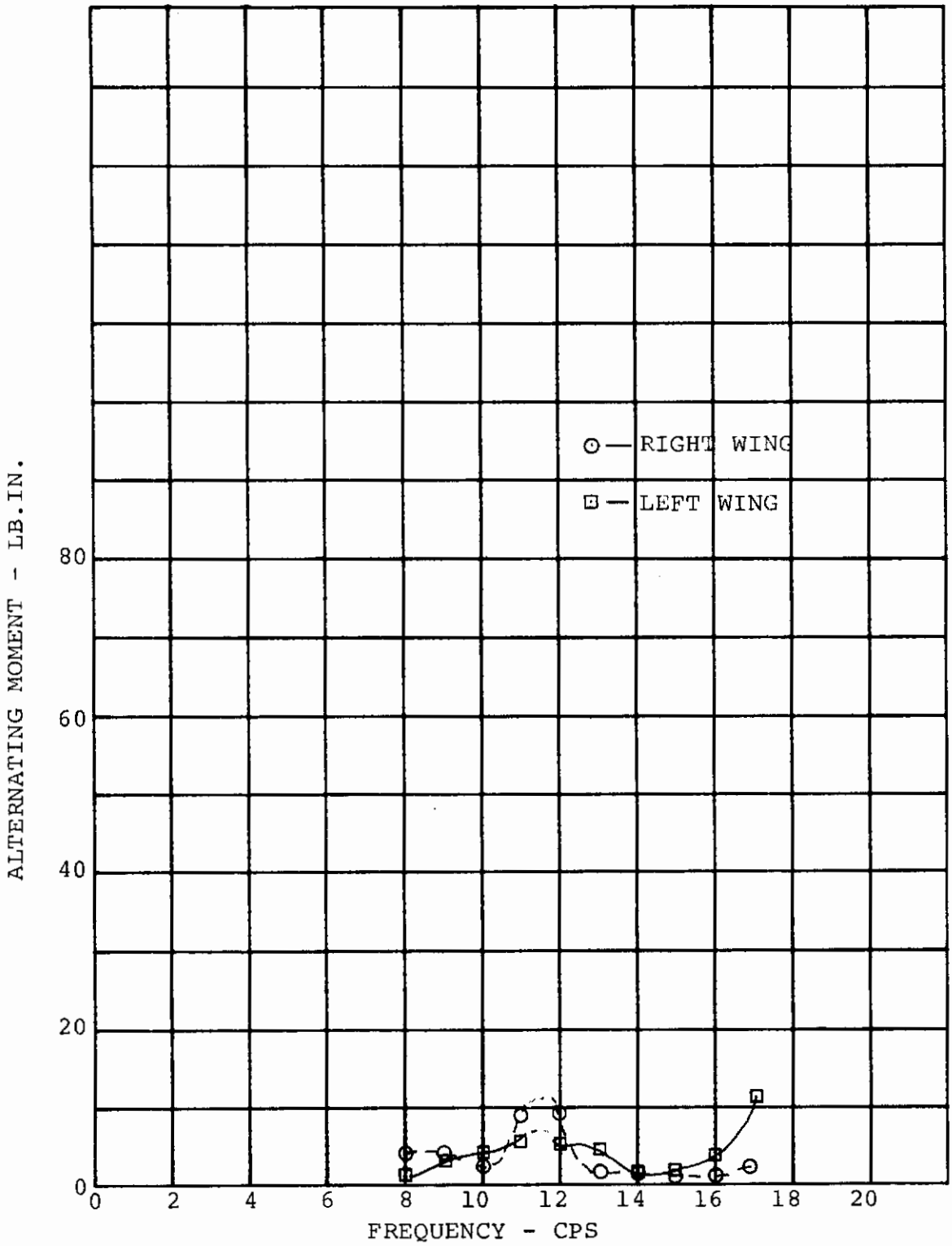


FIGURE 7-10: WING CHORD BENDING RESPONSE TO ANTI-SYMMETRIC EXCITATION-NON-ROTATING BLADES, i_N = 90 DEG.

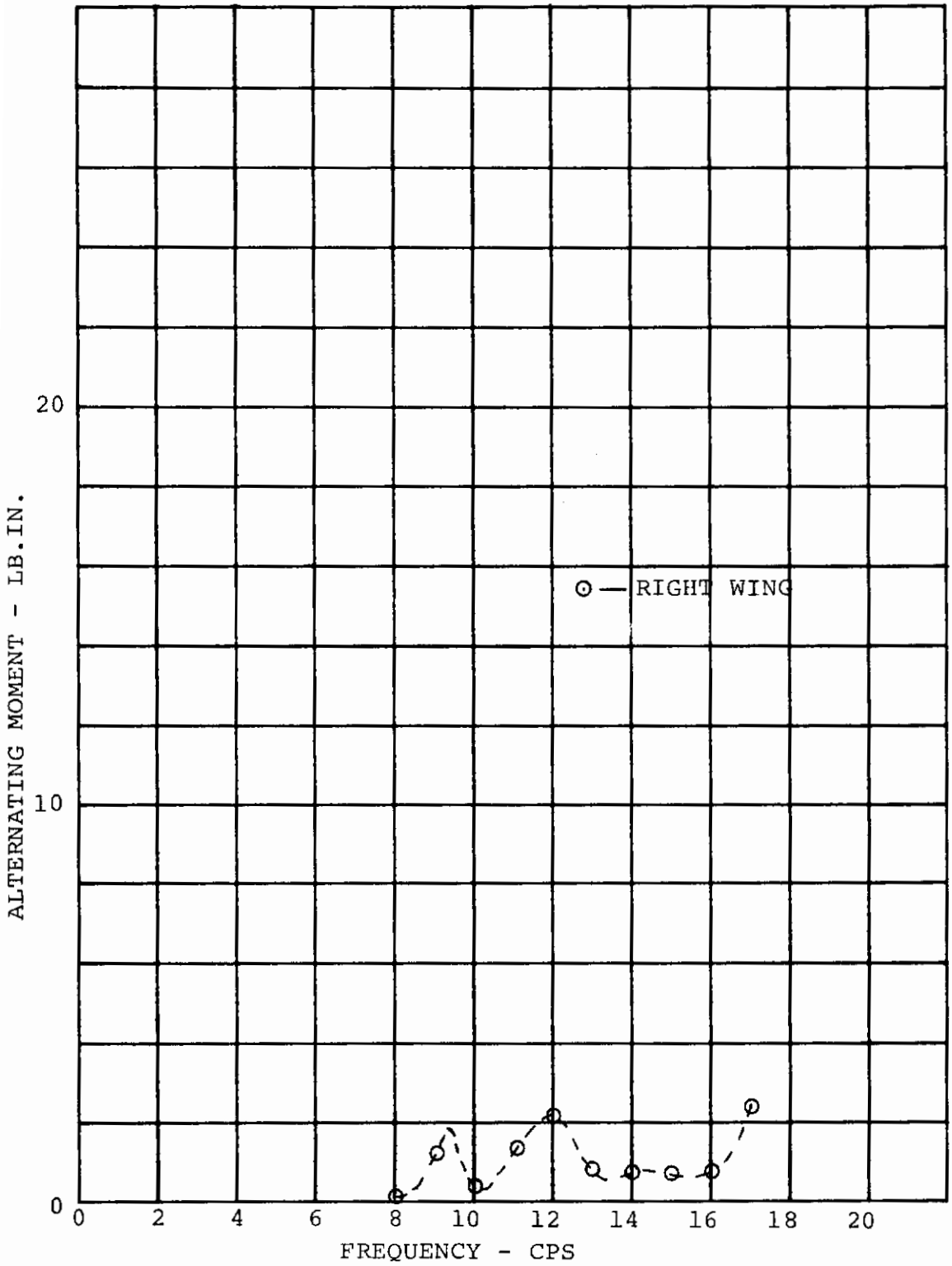


FIGURE 7-11: WING TORSION RESPONSE TO ANTI-SYMMETRIC EXCITATION, NON-ROTATING BLADES, i_N=90 DEG.

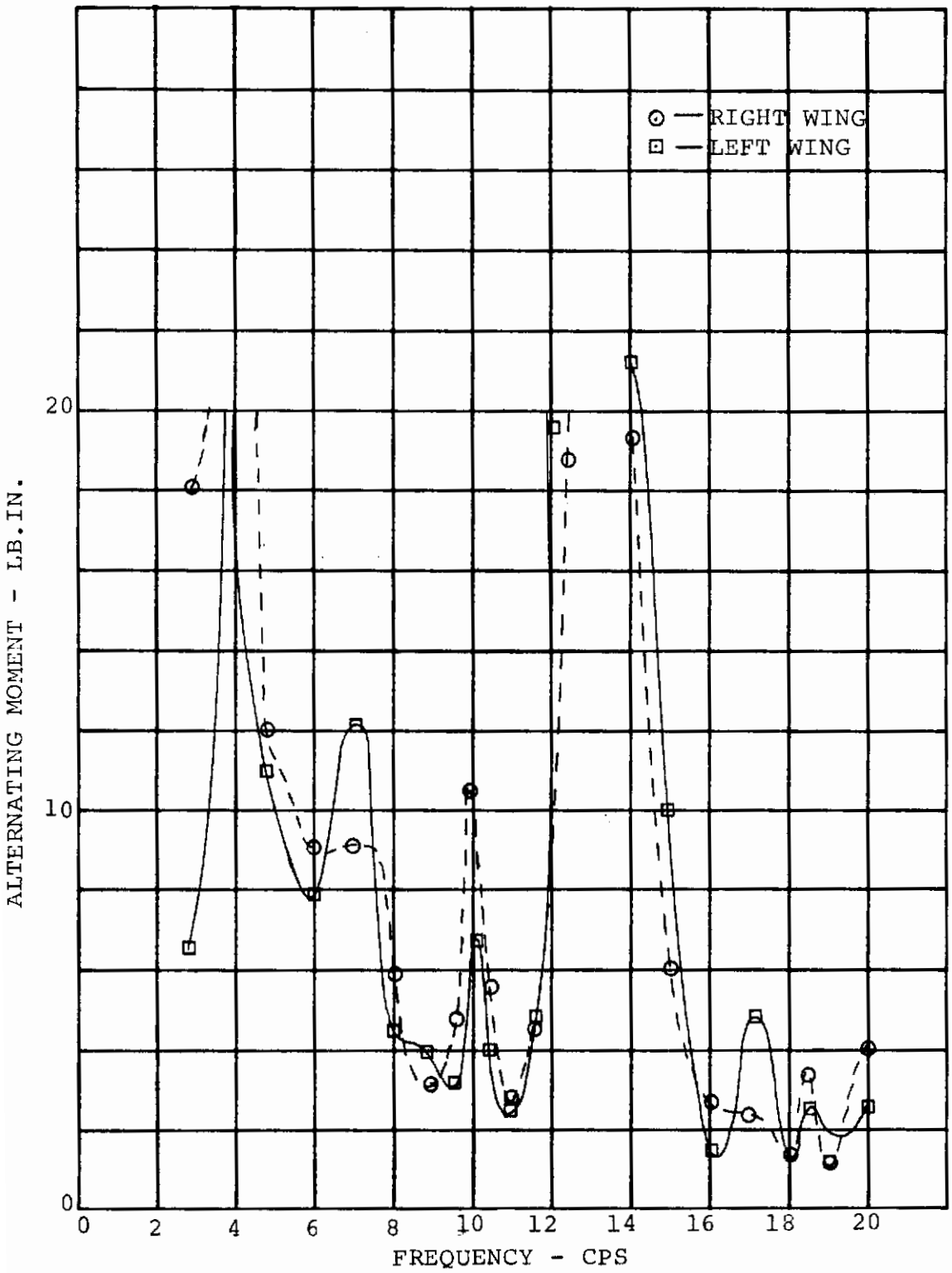


FIGURE 7-12: WING FLAP BENDING RESPONSE TO SYMMETRIC EXCITATION, Ω = 790 RPM, q = 0 PSF., $\theta_{.75}$ = 6 DEG. i_N = 0 DEG., RUN 45

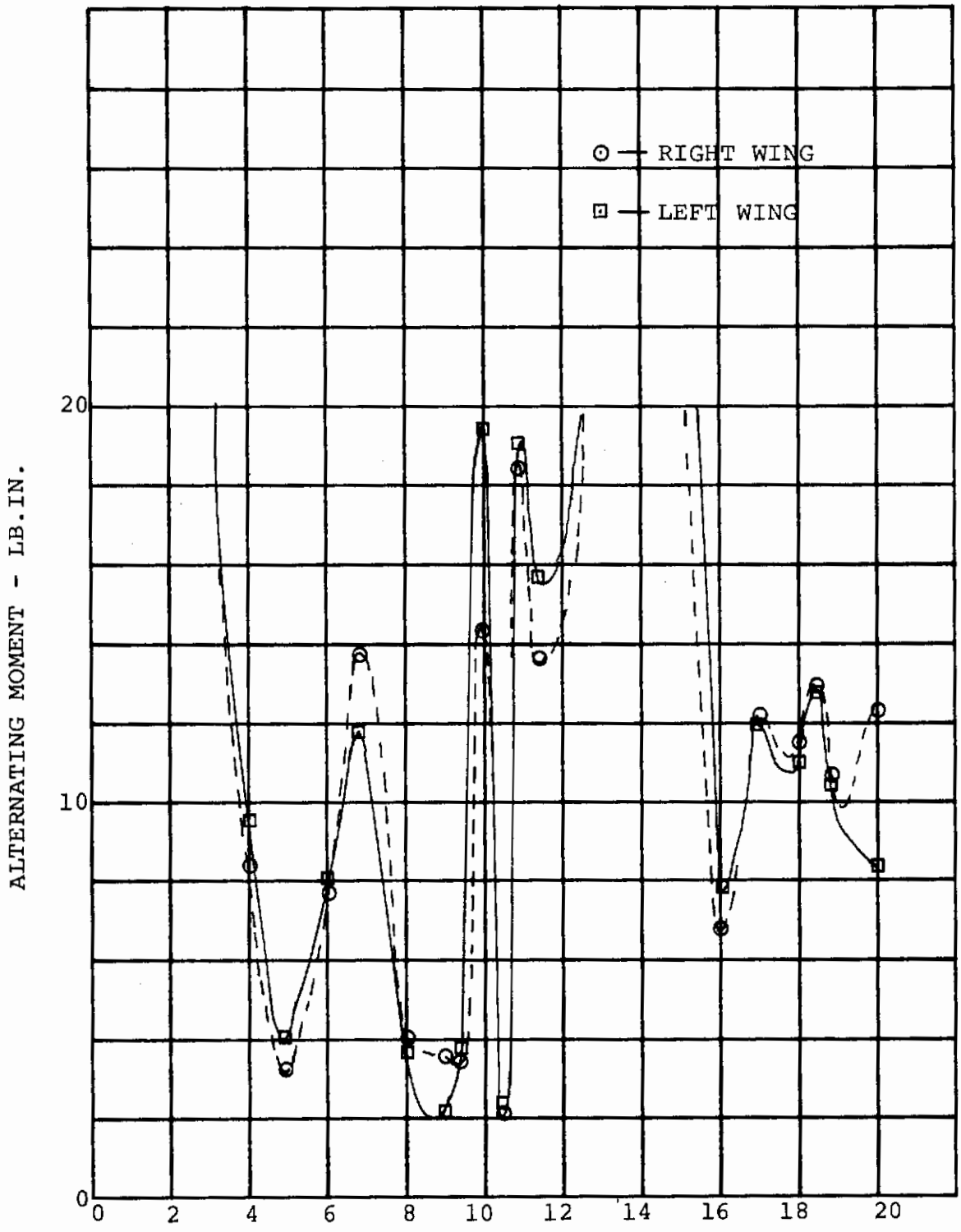


FIGURE 7-13: WING CHORD BENDING RESPONSE TO SYMMETRIC EXCITATION, Ω =790 RPM, q = 0 PSF., $\theta_{.75}$ =6 DEG., i_N = 0 DEG., RUN 45

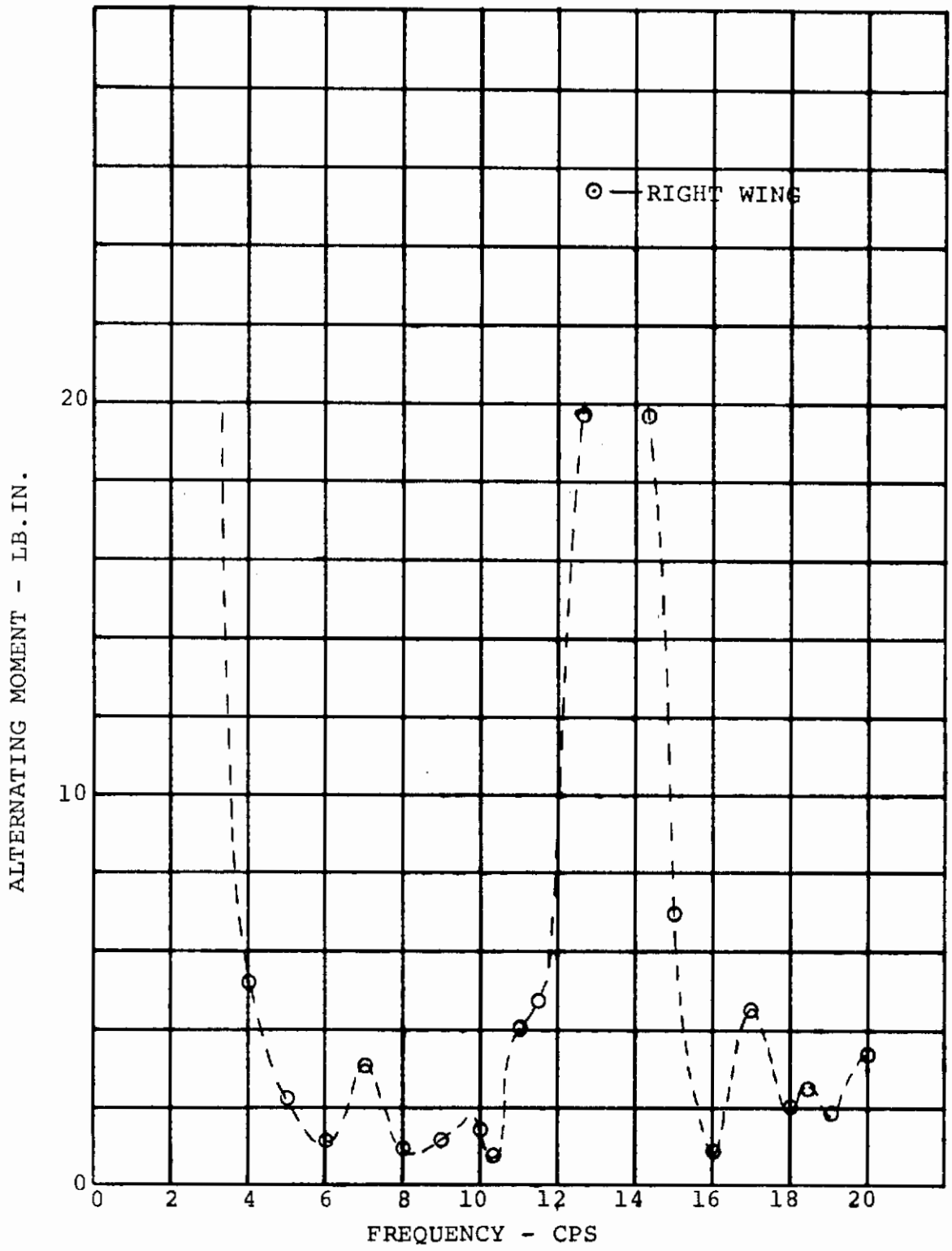


FIGURE 7-14: WING TORSION RESPONSE TO SYMMETRIC EXCITATION - Ω =790 RPM, q = 0 PSF., $\theta_{.75}$= 6 DEG. i_N = 0 DEG., RUN 45

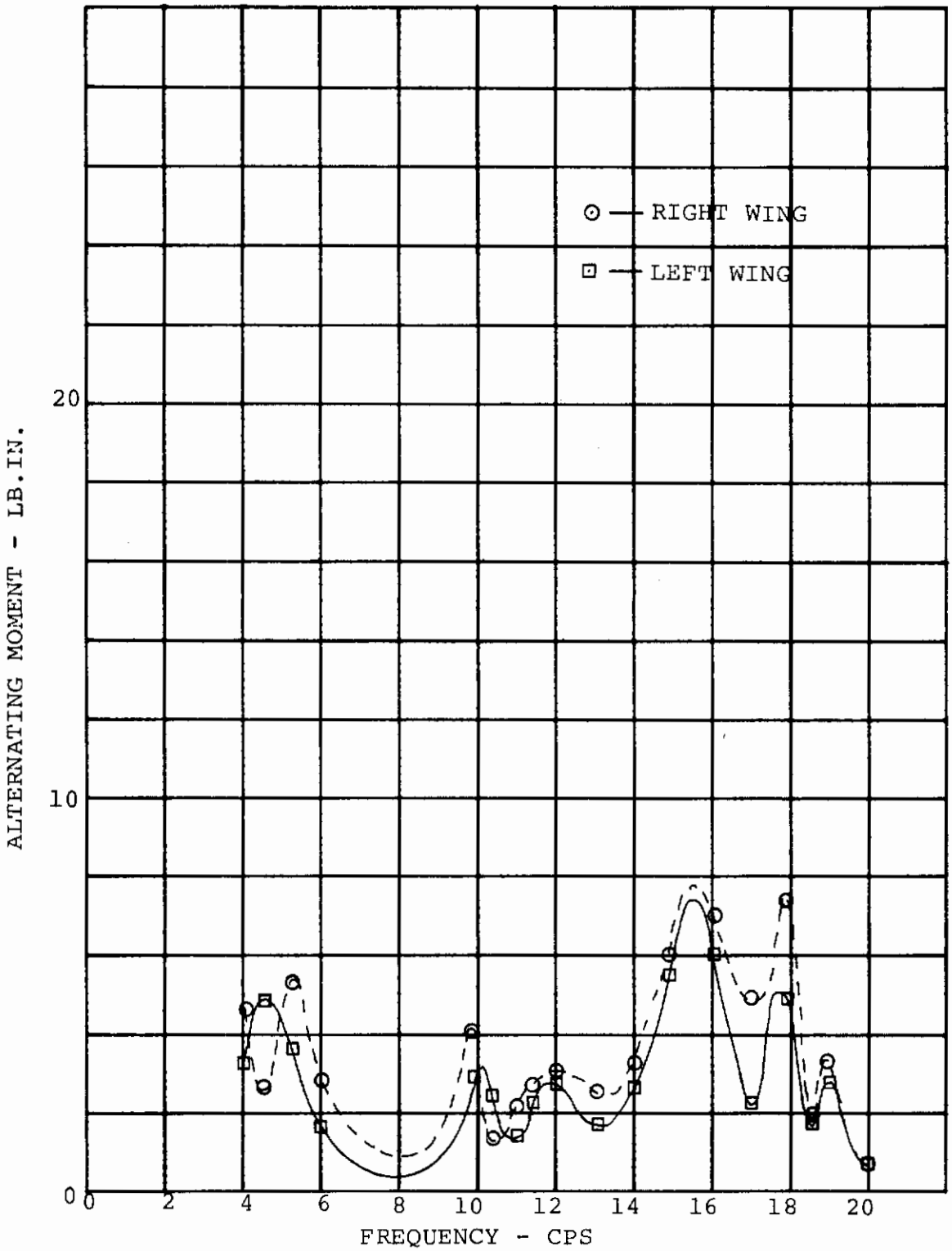


FIGURE 7-15: WING FLAP BENDING RESPONSE TO SYMMETRIC EXCITATION - Ω = 825 RPM, q = 0 PSF., $\theta_{.75}$ = 10 DEG., i_N = 90 DEG.,RUN 52

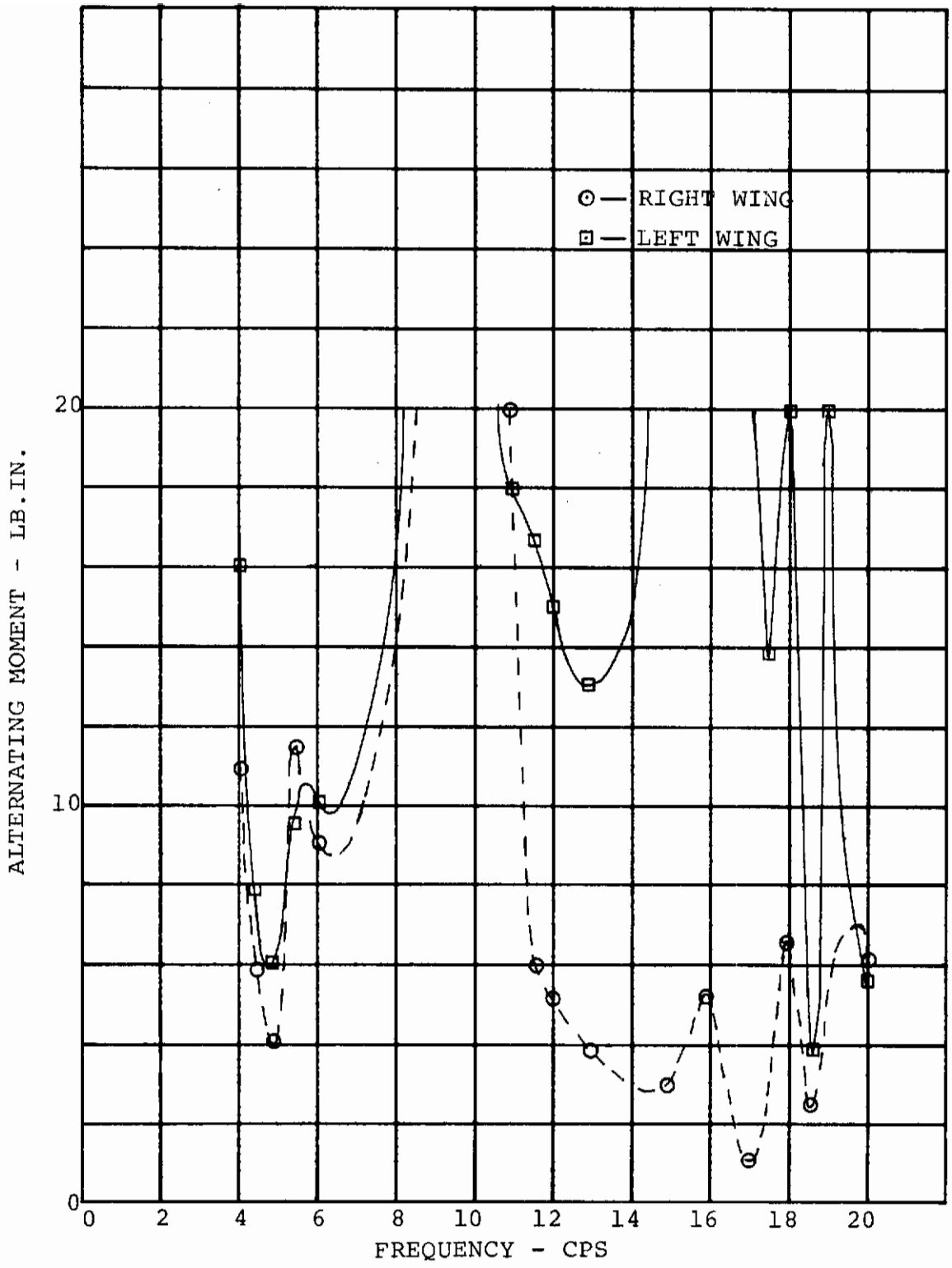


FIGURE 7-16: WING CHORD BENDING RESPONSE TO SYMMETRIC EXCITATION, Ω = 825 RPM, q = 0, $\theta_{.75}$ = 10DEG. i_N = 90 DEG., RUN 52

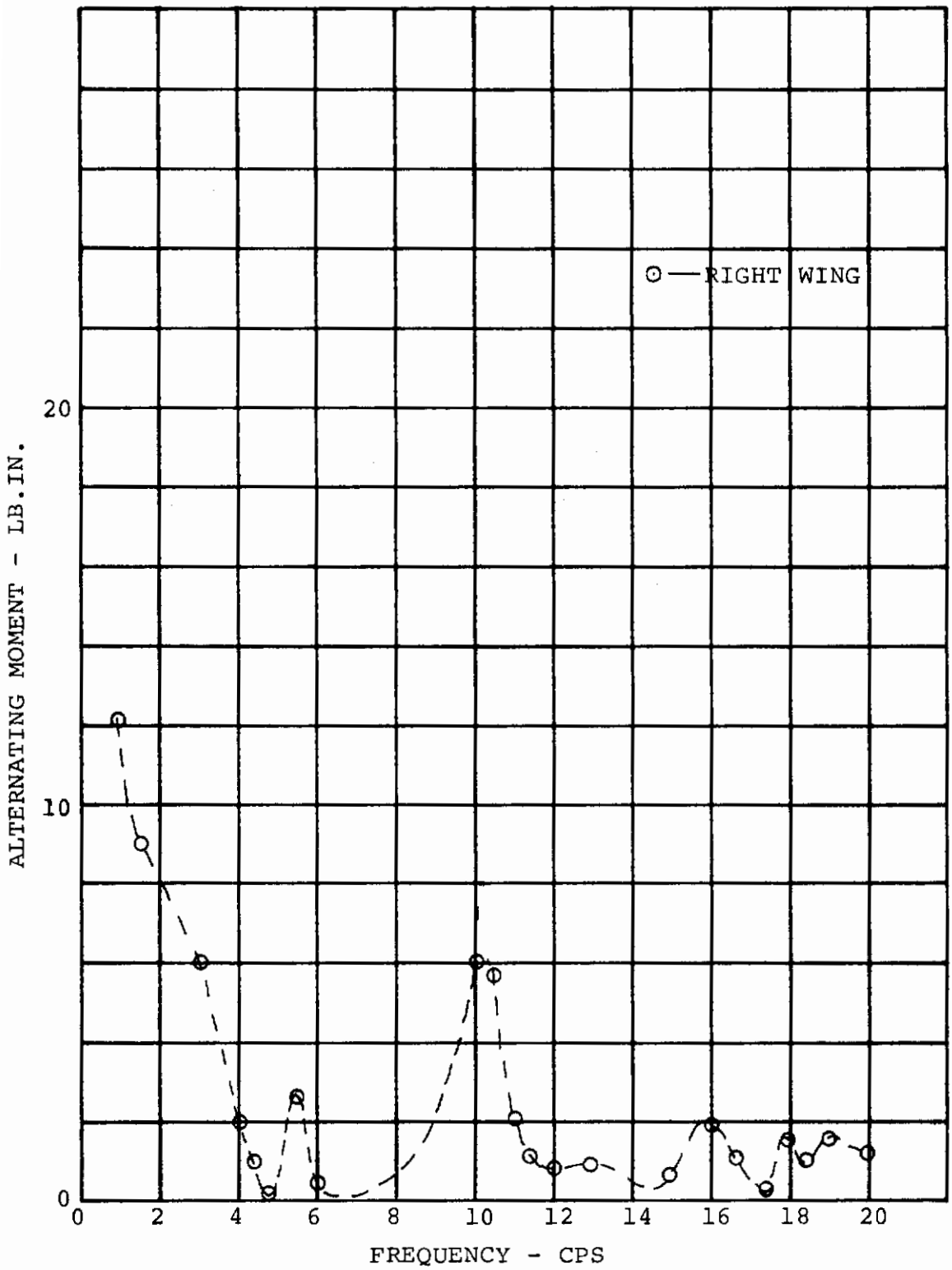


FIGURE 7-17: WING TORSION BENDING RESPONSE TO SYMMETRIC EXCITATION $-\Omega$ = 825 RPM, q = 0, $\theta_{.75}$ = 10 DEG. i_N = 90 DEG., RUN 52

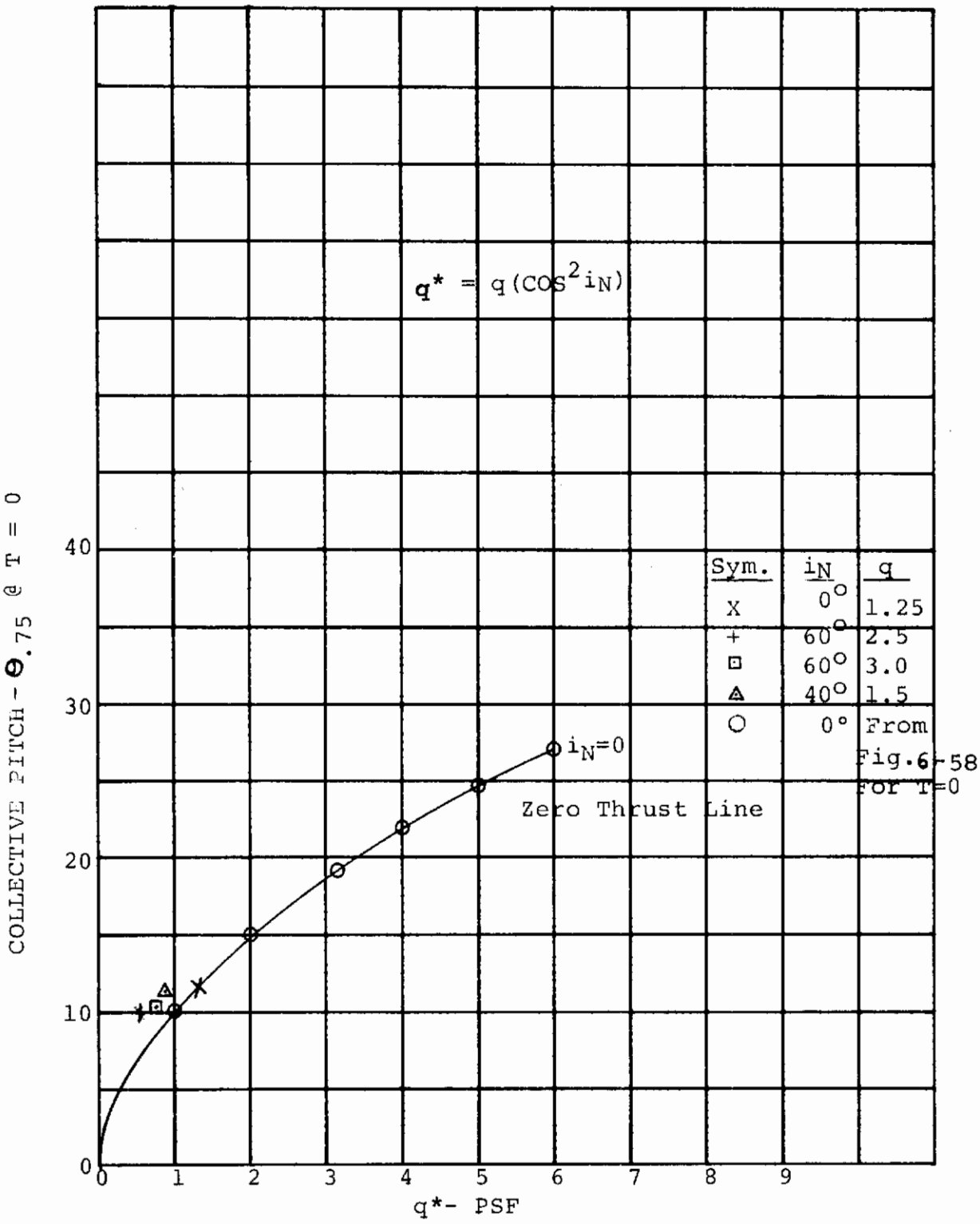
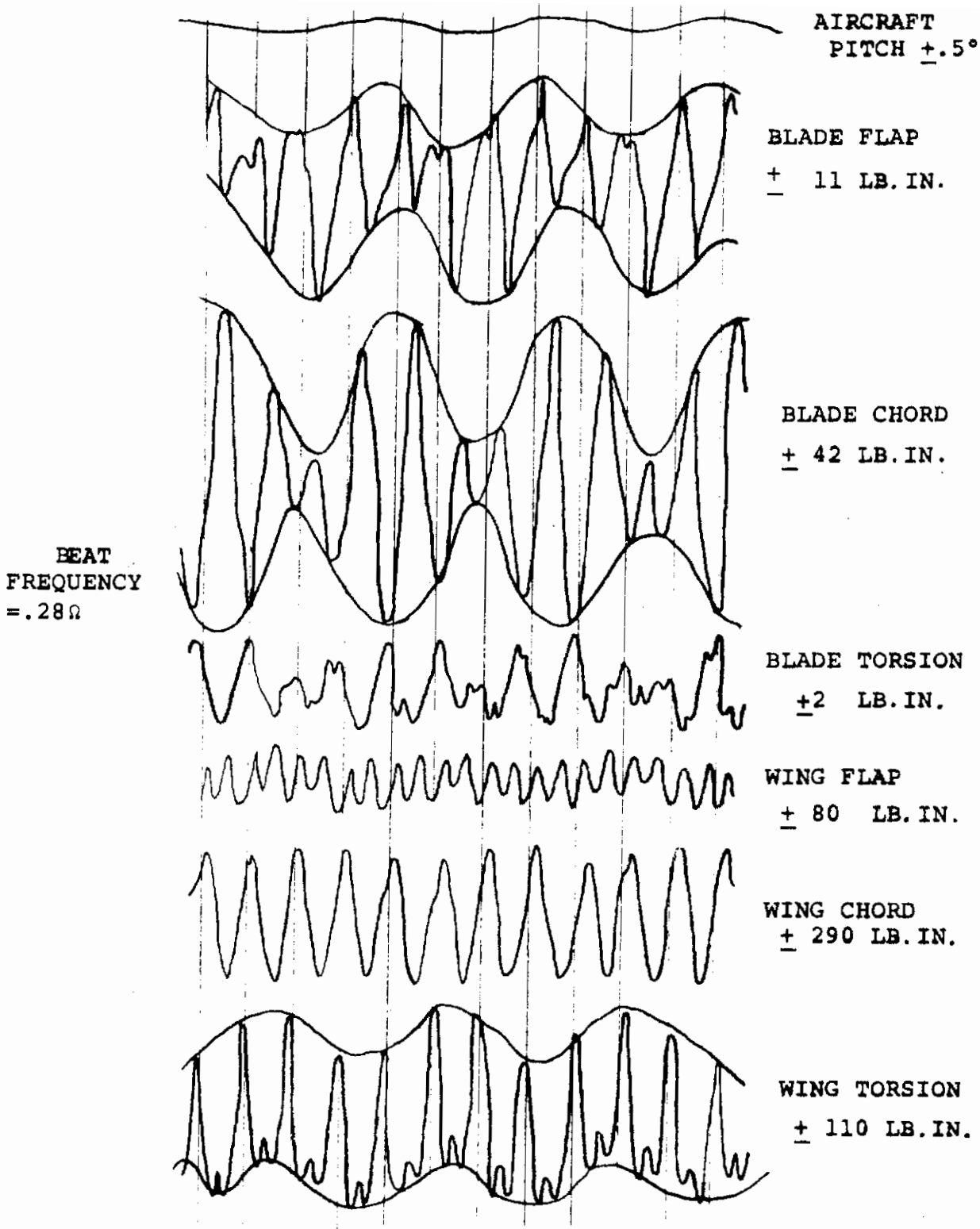


FIGURE 7-18 - LOW FREQUENCY OSCILLATION OCCURRENCES RELATIONSHIP TO ZERO THRUST



NOTE: Oscillation started at q = 2.5 psf

FIGURE 7-19 BEAT FREQUENCY RESPONSE OSCILLOGRAM RIGHT BLADE AND WING, Ω = 790 RPM, $\theta_{.75}$=10.2 DEG., θ_2=3.5 DEG. q= 2.75 PSF, i_N=60 DEG., RUN 40(12).

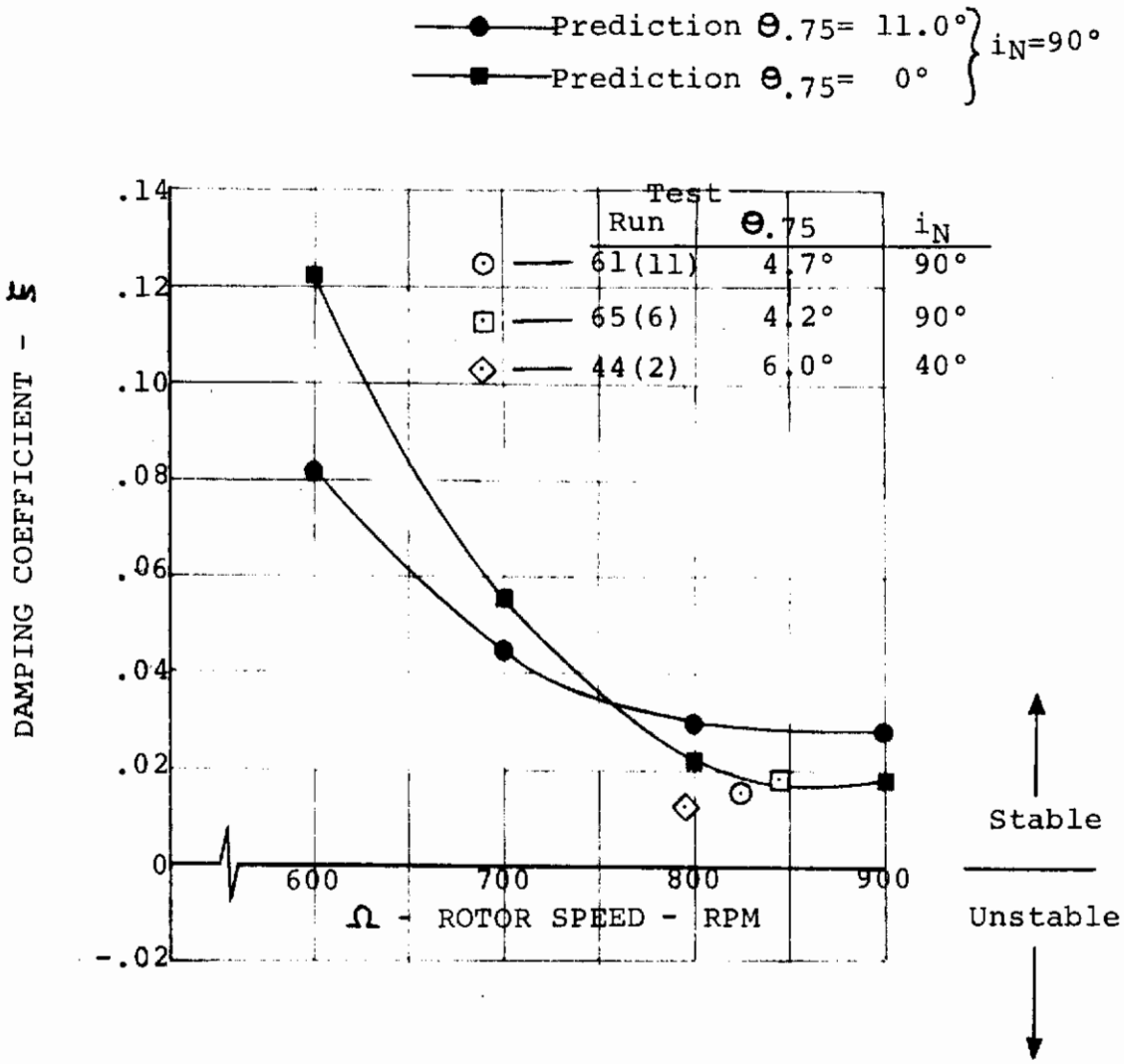


FIGURE 7-20. - CORRELATION OF OBSERVED LOW FREQUENCY MODE DAMPING WITH PREDICTION. q = 0

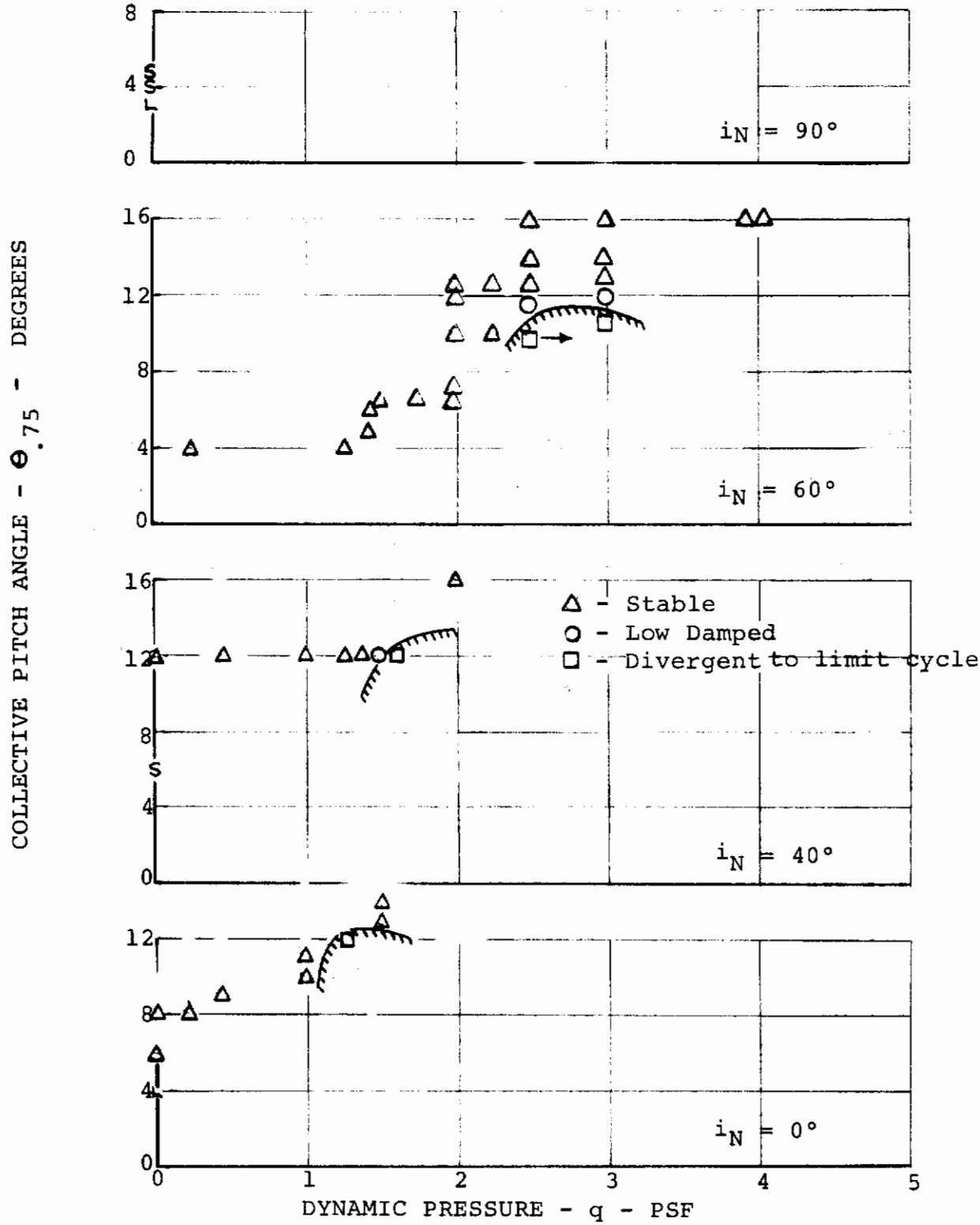


FIGURE 7-21. REGIONS OF LOW FREQUENCY OSCILLATION OCCURRENCES

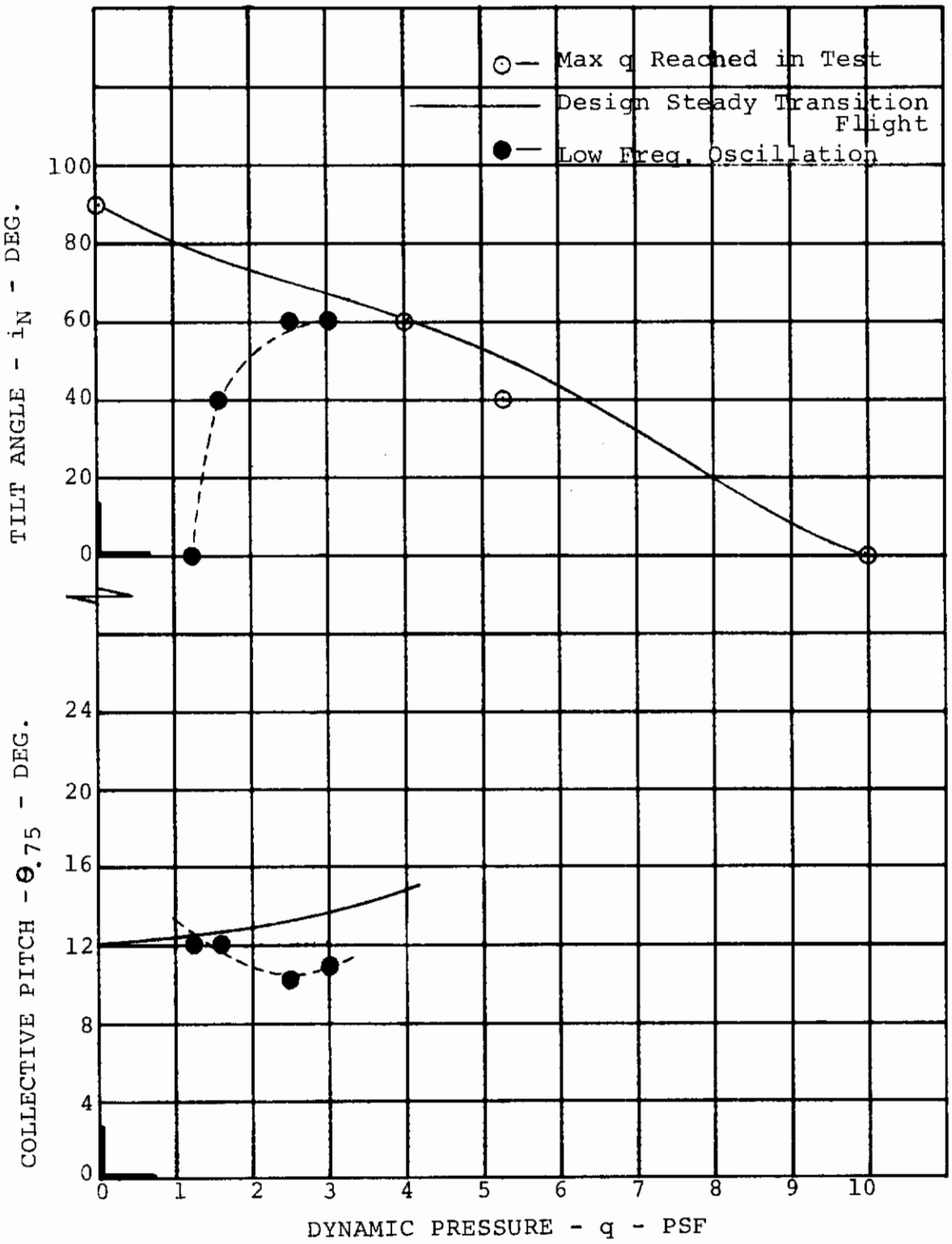


FIGURE 7-22. SUMMARY OF CONDITIONS AT WHICH LOW FREQUENCY OSCILLATION OCCURRED.

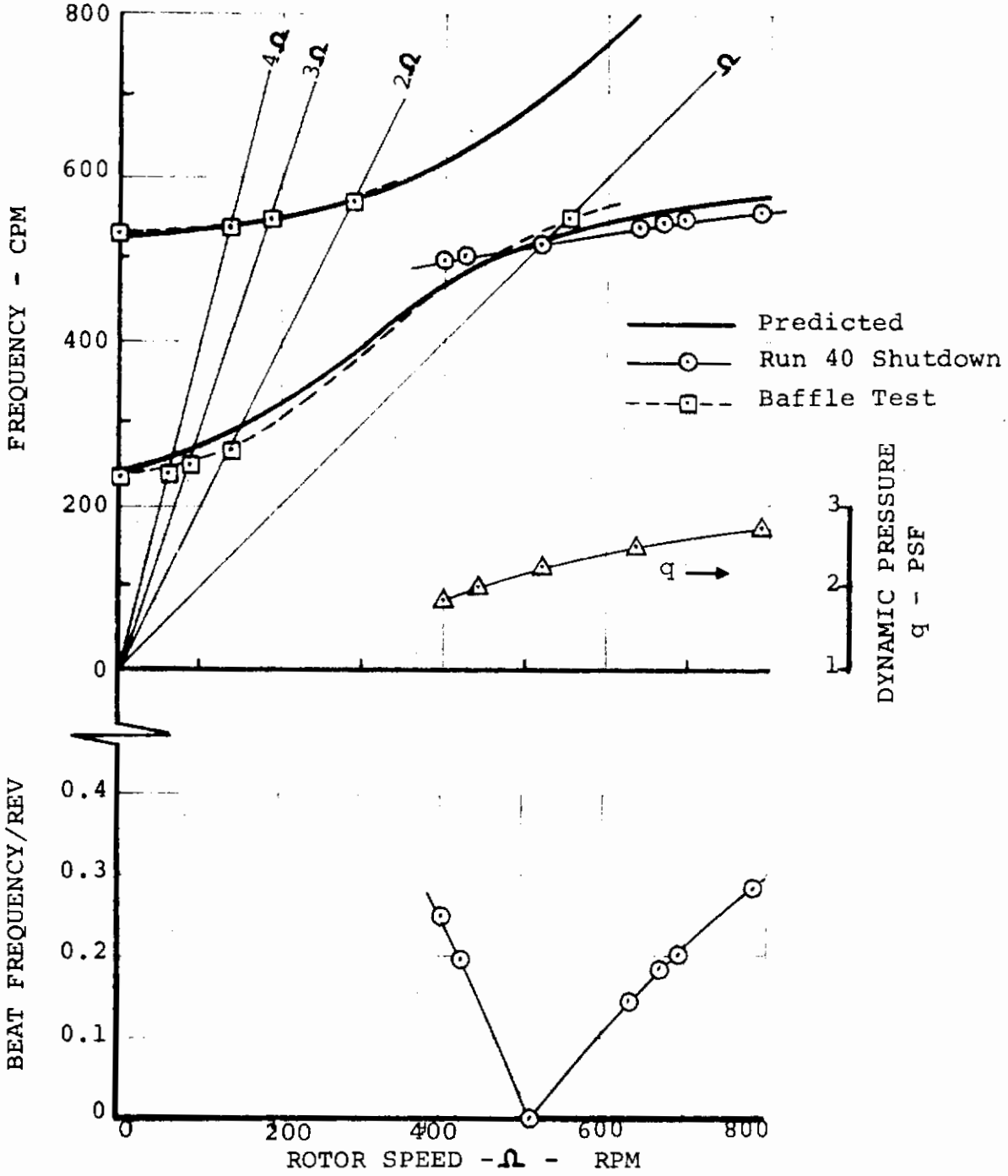


FIGURE 7-23. CORRELATION OF BLADE FREQUENCY OBTAINED FROM BEAT OSCILLATION DURING EMERGENCY SHUTDOWN, $\Theta_{.75}$ = 10.2 DEG., i_N = 60 DEG. RUN 40(12)

8.0 FLYING QUALITIES

8.1 INSTALLED RIGID BODY DYNAMICS

For pure free flight simulation a model must be mounted such that all six rigid body degrees of freedom are unrestrained. Realistic mounting arrangements, however, can only approach this ideal condition with the result that finite built in stiffness and damping exist.

"Tweak" tests were performed prior to the power-on tests specifically to determine the rigid body dynamic properties (frequency and damping). The effects of the umbilical and snubber cables were determined. Prior to the wind tunnel installation an assessment of the umbilical effect was made by conducting simple tests to determine the optimum umbilical arrangement, the results of which showed that the installation used in this test (Figure 3-1) does not change the stiffness and inertia of the rigid body modes. This was confirmed by the data shown in Table 4-2. The model was "tweak" tested periodically throughout the program and the corresponding results are listed in Table 4-1. All damping data were reduced from the motion decrements recorded on the oscillograph in terms of viscous damping coefficients.

8.2 PITCH AND ROLL SPRINGS

The capability of evaluating the effect of a simple attitude feedback control system was provided by installing springs in the pitch and roll axes. These springs added a stiffness of 15.0 in.lb/deg in pitch and 5.0 in.lb/deg in roll, or in terms of control sensitivity in the hover mode, 1.01 rad/sec^2 and 0.061 rad/sec^2, respectively, per degree of fuselage attitude. The power-off, rigid body dynamic effect of the springs is shown by the data in the following table for the hover configuration.

AXIS	SPRINGS REMOVED-RUN 62		SPRINGS ON - RUN 28	
	f	ξ	f	ξ
Pitch	.87	.14	1.5	.027
Roll	.31	.10	.54	.028

8.3 HOVER CHARACTERISTICS

8.3.1 Rigid Body Damping

Rigid body damping data were obtained by gradually displacing the model from the neutral trimmed position with the snubber cables and then releasing the cables quickly. In all cases throughout the test the rigid body modes were very stable. A summary of all data relating to rigid body stability is contained in Table 8-1.

An example of increased stability provided by the rotor in hover is shown by the damping decrements in Figure 8-1 which illustrate a more rapid decay with power on. This is the case for both pitch and roll. Collective has an apparent stabilizing effect in itself as illustrated in the bar chart of Figure 8-2. Here it is shown that the rotors turning with no induced flow ($\theta_{.75} = 0^{o}$) provides no added stability in pitch whereas the high collective substantially increases the damping. Data were not obtained for roll motion at low collective and no ordinate values are given since frequency and damping are of primary interest.

8.3.2 Collective Pitch Effectiveness - Roll Controllability

Differential collective pitch effectiveness for roll control was determined from the wing root flap bending moment data of Figure 8-3. The data reflects the hovering configuration bending moments measured at butt line 6.62 as collective pitch of the prop/rotor is varied. The thrustline is located at butt line 40.68. Except for the non-linearity at the 2.0 degree collective setting, the moment response is essentially linear to 9.0 degrees collective with some increased effectiveness noted between 9.0 and 10.7 degrees.

At the maximum collective value ($\theta_{.75} = 10.7^{o}$) shown in Figure 8-3 the rotor has developed 20.6 lb. of thrust which represents 2/3 of the thrust required to balance the model total weight (61 lb.). Extrapolation of the data indicates that approximately 12^{o} of collective would be required to lift 50 lb. (the equivalent full scale aircraft gross weight).

8.3.3 Cyclic Pitch Effectiveness - Pitch Controllability

Wing root torque arising from the application of longitudinal cyclic pitch on one prop/rotor is shown in Figure 8-4. In the hover configuration ($i_N = 90^{o}$), control effectiveness is

essentially linear. Comparative data are also presented for the cruise configuration ($i_N = 0^o$) at a dynamic pressure (q) of 0.2 PSF. The steady torque measured at zero cyclic is attributed to a residual control input of approximately -1.0 degree cyclic and possibly a distorted wing torsion wave form due to rotor unbalance.

The average torque or pitching moment data per degree of cyclic pitch were measured about zero cyclic and converted to non-dimensional coefficient form. The resulting coefficients are plotted in Figure 8-5 and are compared with predicted levels for the cruise configuration.

8.4 TRANSITION AND CRUISE CHARACTERISTICS

The limited amount of data which were obtained in transition indicates that the pitch stability increases slightly over that in hover. The effect of dynamic pressure on the rigid body pitch frequency and damping is illustrated in Figure 8-6 which shows a general increase in damping with increased dynamic pressure. These data are composed of $i_n = 40^o$, 60^o, and 90^o tilt angles. Additional data are required for each tilt angle to completely define the damping trend.

Data obtained with the rotors off (Run 57) are presented in Figure 8-7 and show that although the damping with wind-on is greater than that with wind-off, a decreasing trend exists with increasing dynamic pressure. Insufficient data exist to establish positive conclusions, but correlations with the data of Figure 8-7 and Run 50(5) in Table 8-1 suggests that the rotor provides added damping in the cruise attitude.

TABLE 8-1

RIGID BODY DAMPING DATA SUMMARY

RUN	i_N	q	Ω	$\theta_{.75R}$	MODE	FREQ. CPS	DAMPING COEFF.
PRE 14	90	0	0	-	PITCH	1.5	.06
					ROLL	.54	.016
21	90	0	750	0	PITCH	1.65	.06
27	90	0	750	8.2	PITCH	1.8	.14
					ROLL	.56	.06
40(9)	60	2.0	790	10.2	PITCH	1.7	.16
42(5)	60	3.0	790	16.	PITCH	1.8	.18
43(9)	40	4.0	790	20.	PITCH	1.7	.22
PRE 44	40	0	0	-	PITCH	1.5	.07
					ROLL	.55	.02
44(4)	40	0	790	6.	PITCH	2.0	.11
50(5)	0	10.0	790	32.	PITCH	1.7	VERY HIGH
PRE 57	0	0	ROTORS OFF		PITCH	1.5	.05
					ROLL	.60	.02
57	0	VARY	ROTORS OFF		PITCH	SEE FIG. 8-7	

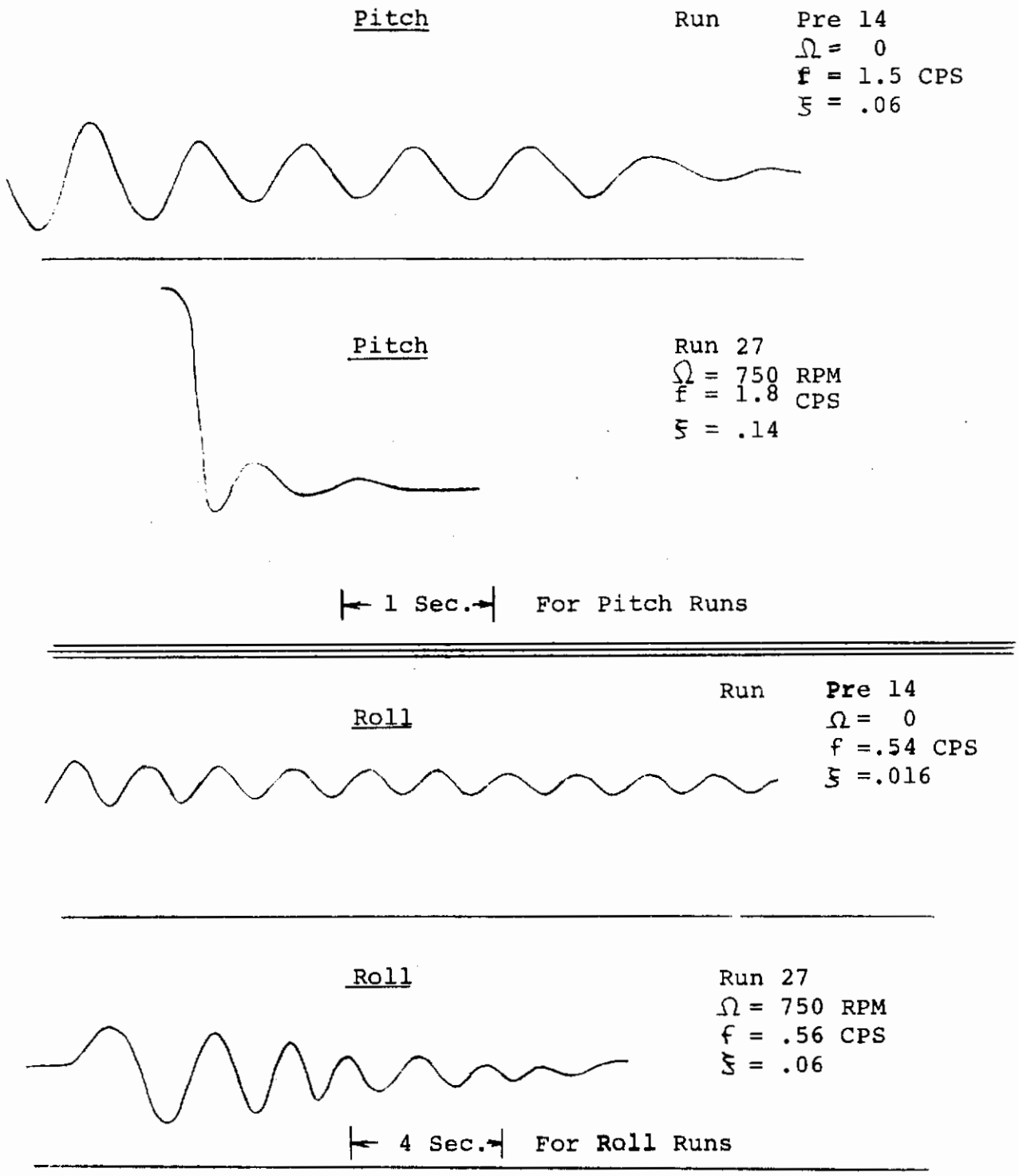


Figure 8-1. Pitch and Roll Damping Decrements - Hover Condition
$\theta_{.75}$ = 10.2°L, 8.2°R for Run 27

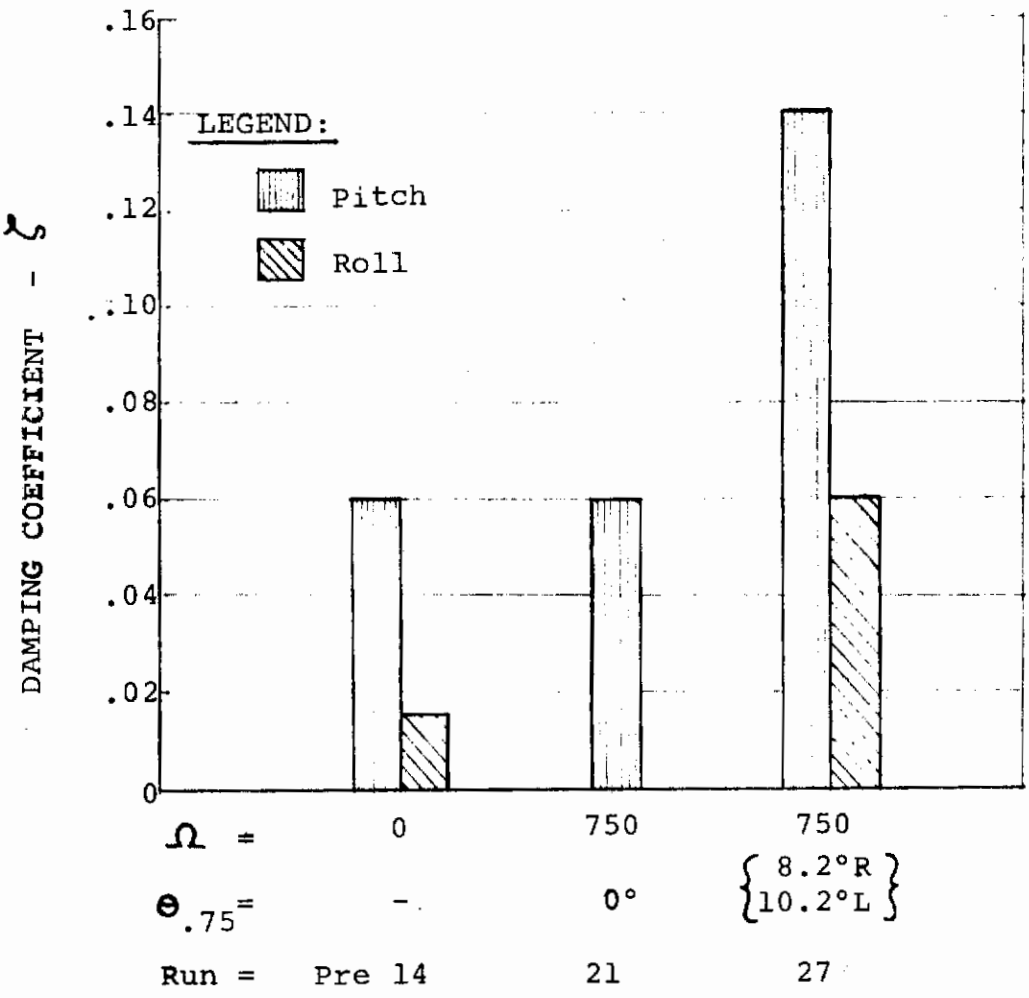


FIGURE 8-2. RIGID BODY DAMPING IN HOVER.

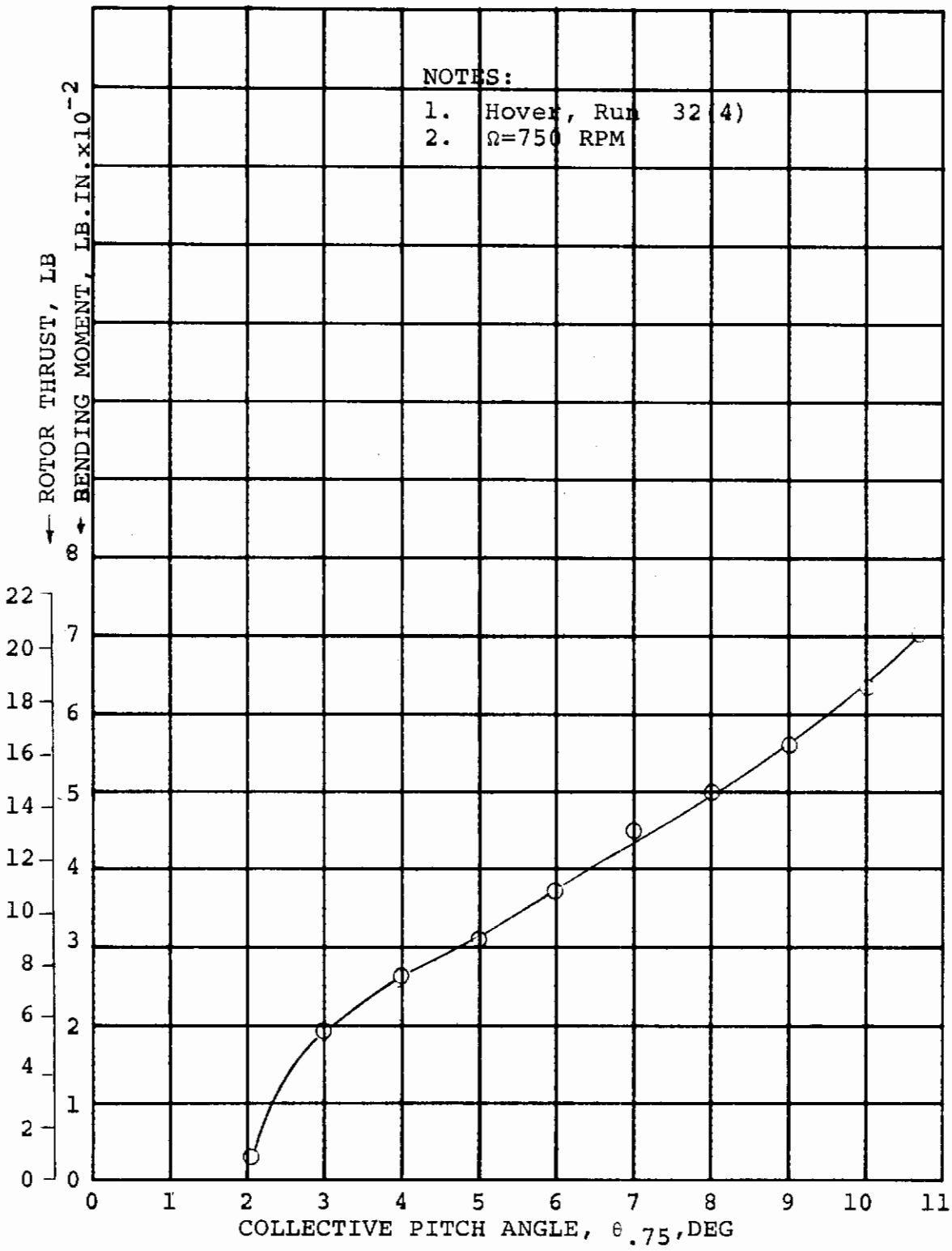


FIGURE 8-3 EFFECT OF COLLECTIVE PITCH ON WING STEADY FLAP BENDING MOMENT

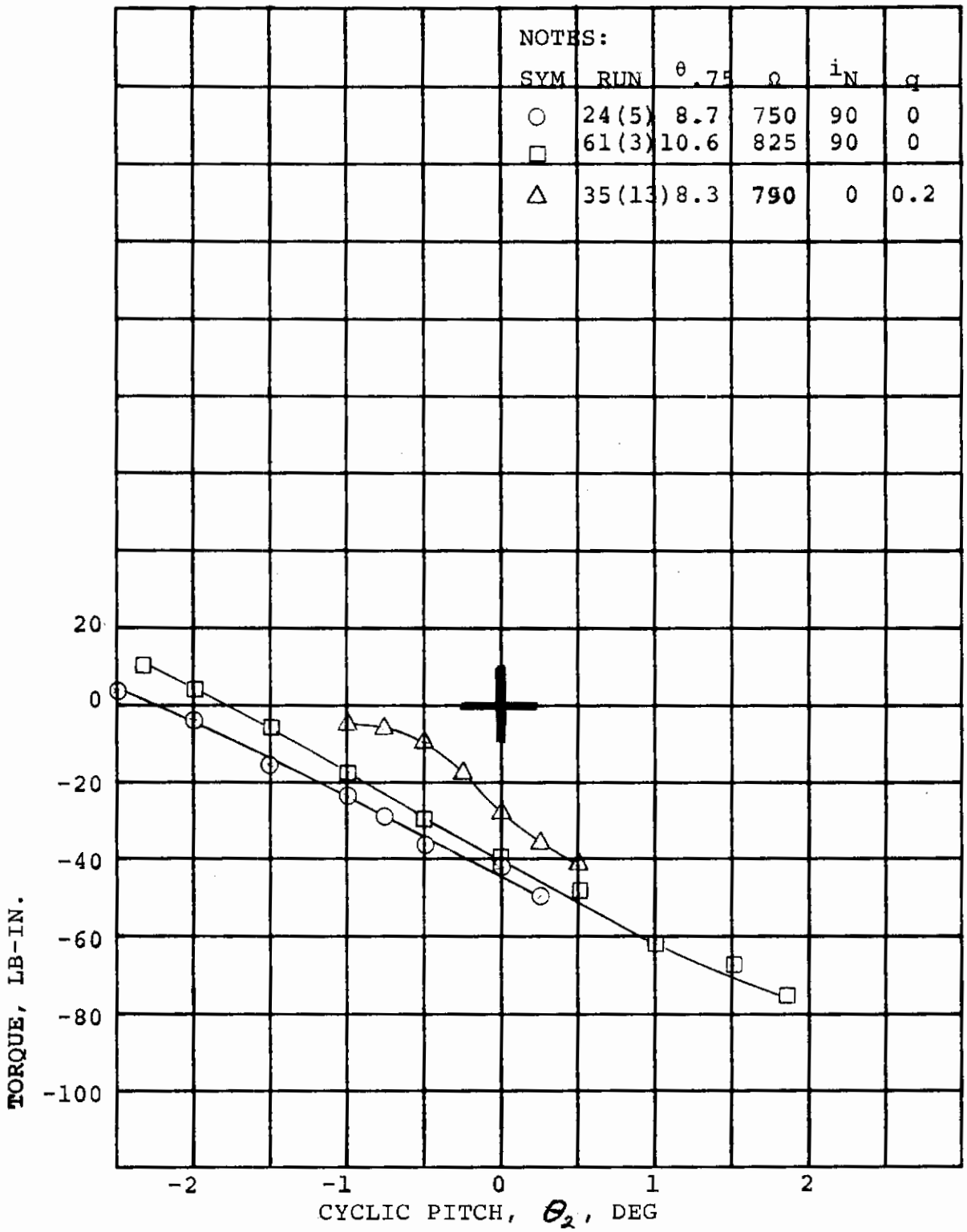


FIGURE 8-4 EFFECT OF CYCLIC PITCH ON WING STEADY TORQUE LOAD ILLUSTRATING CONTROL EFFECTIVENESS

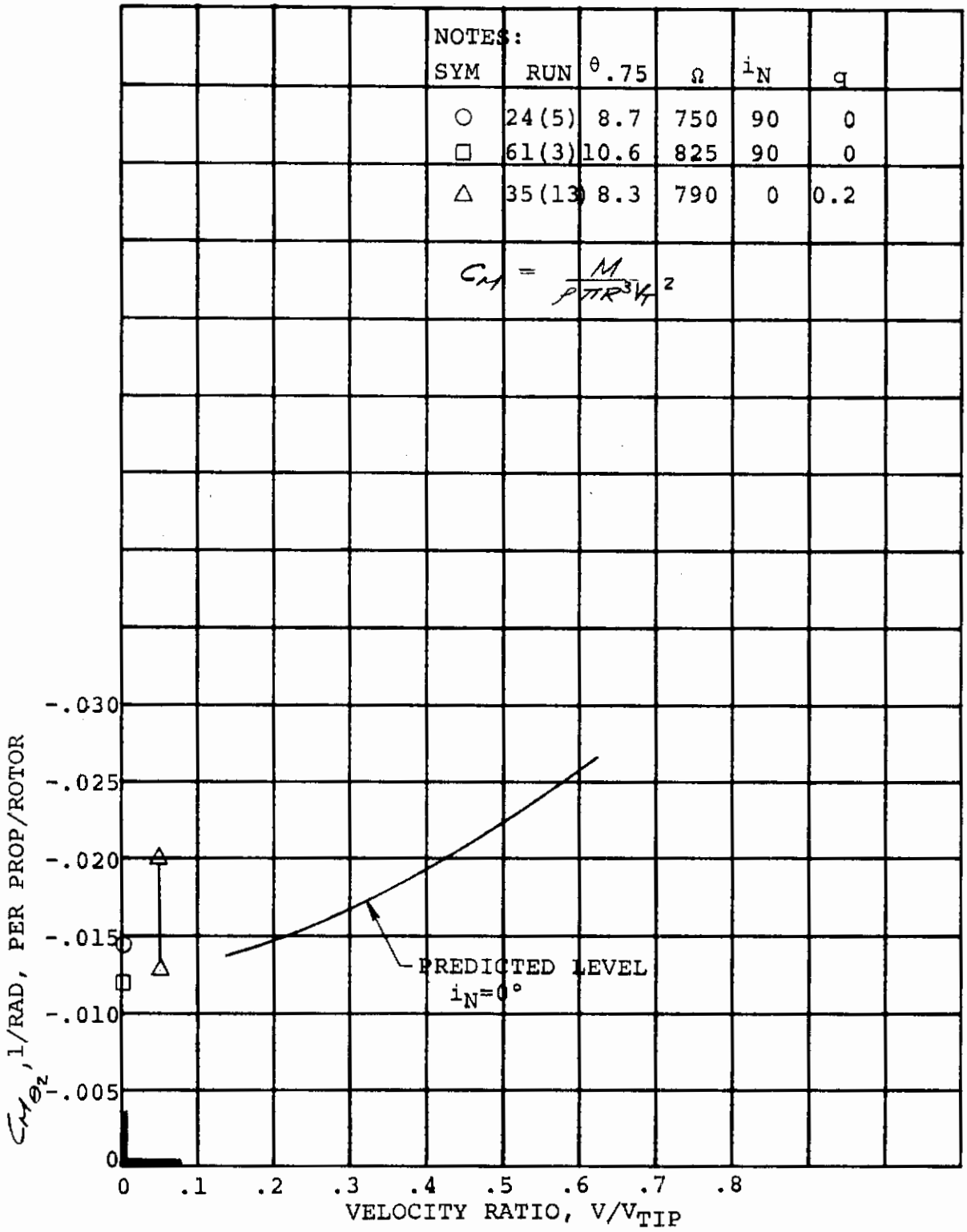


FIGURE 8-5 CYCLIC PITCH CONTROL POWER

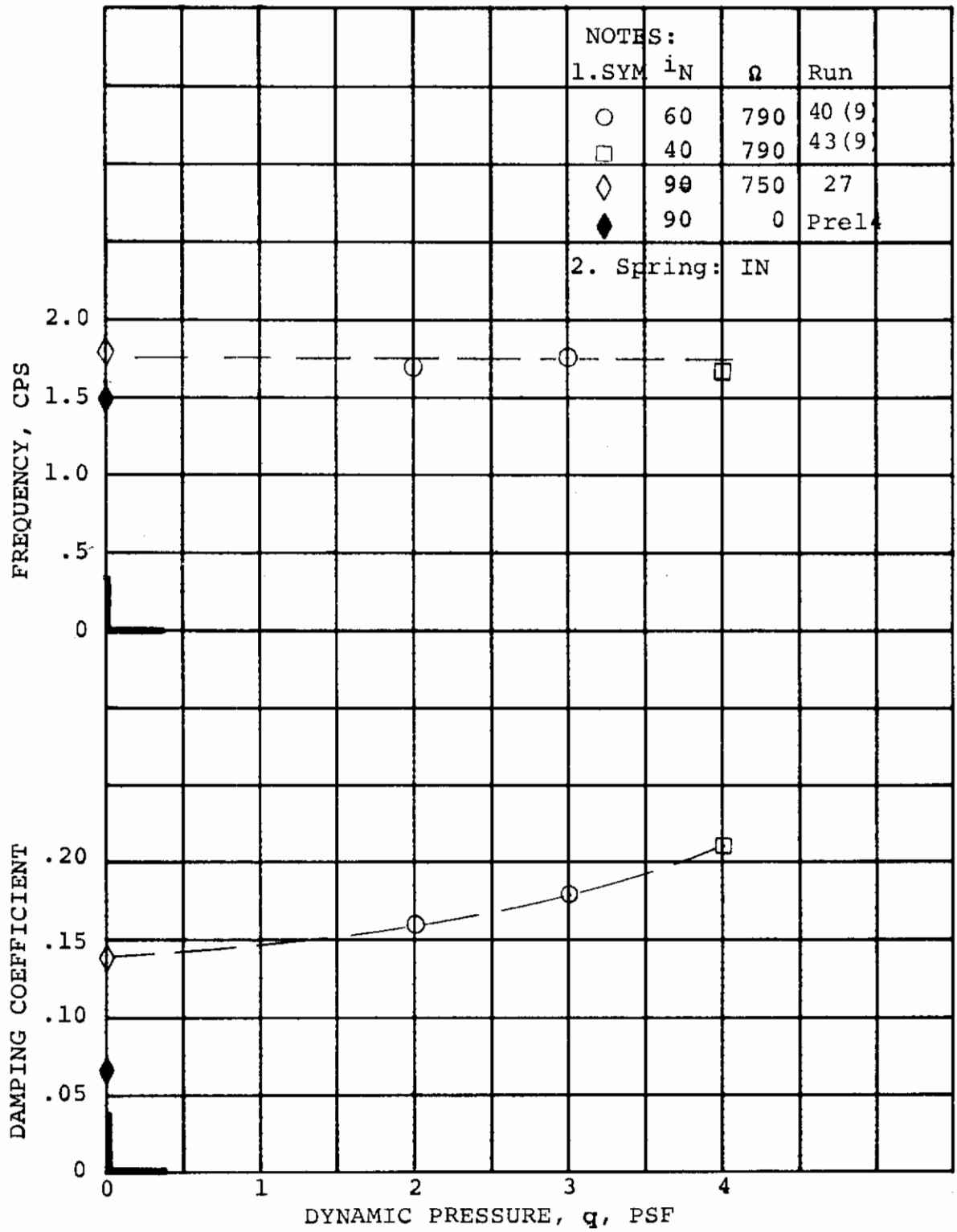


FIGURE 8-6 EFFECT OF DYNAMIC PRESSURE ON RIGID BODY PITCH FREQUENCY AND DAMPING

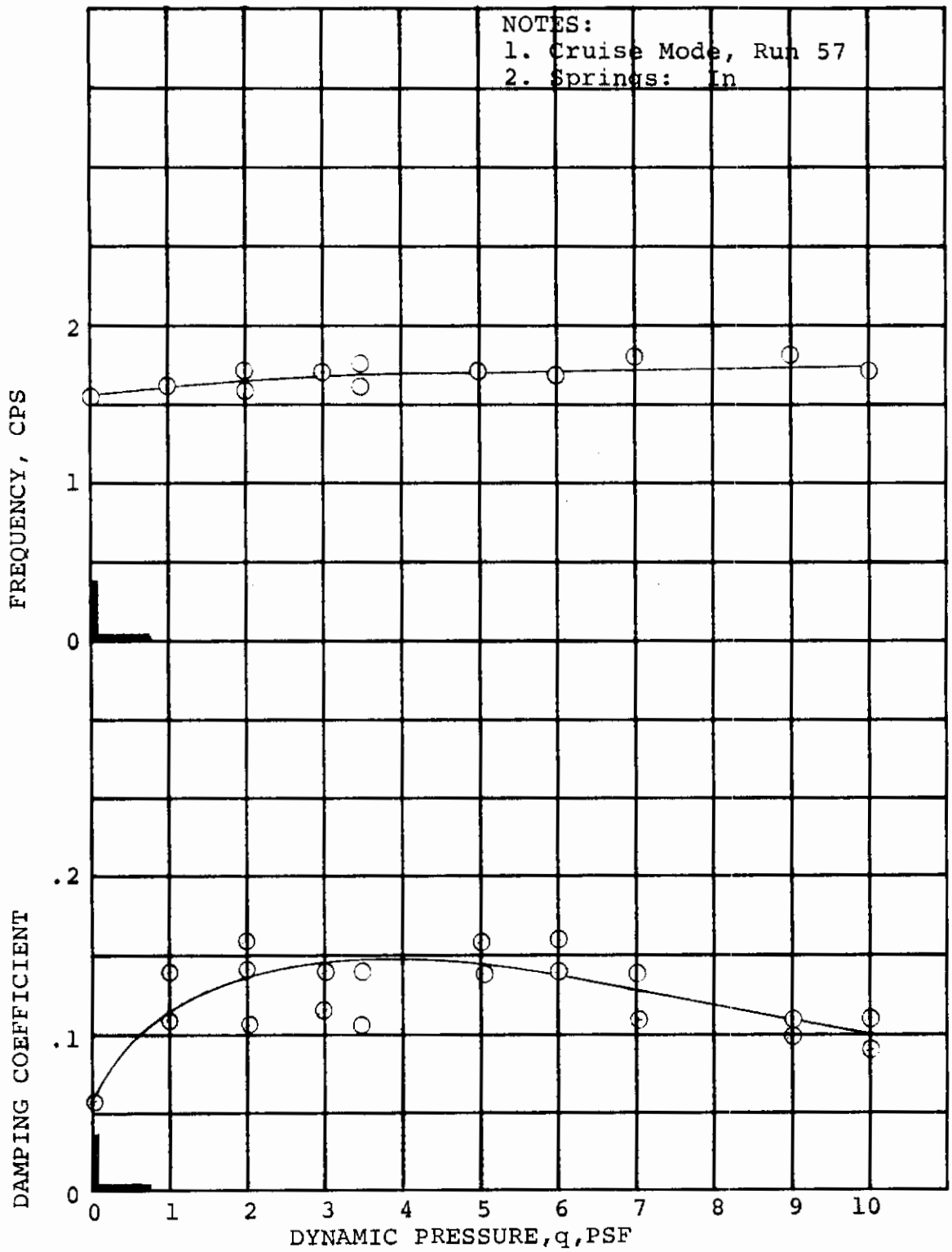


FIGURE 8-7 EFFECT OF DYNAMIC PRESSURE ON THE RIGID BODY PITCH FREQUENCY AND DAMPING WITH ROTORS OFF

9.0 SKITTISHNESS

It is believed that the "skittishness" demonstrated here is caused by unsteady flow around the wings and rotor system and is possibly triggered by flow asymmetries in the vicinity of the fuselage. The lateral oscillation is probably augmented by the lateral restraint of the pole mount. Without the mount, the motion would probably still be oscillatory but of longer period and accompanied by lateral translation of the airplane. Further description would require tests with tufts on the model or smoke for flow visualization to obtain insight as to the actual mechanics of the flow. The motions exhibited are typical of V/STOL aircraft operation in this flight regime and a SAS (Stability Augmentation System) will compensate for the disturbances, as discussed below.

9.1 GROUND EFFECT

In-ground-effect roll attitude motion of the unrestrained model in the hover configuration is shown in Figure 9-1, Run 64, in response to a manual induced disturbance to represent a gust input. The h/D ratio for this data was 0.41. The resulting roll motion is stable and the 0.33 cps oscillations are at least neutrally damped. Long-term self-induced motions which indicate the skittishness of the vehicle are shown in the lower time history as "undisturbed roll motion". The skittish motions are stable and are at least neutrally damped with an average frequency of 0.35 cps. Maximum unperturbed roll displacements are in the order of $\pm 2^{o}$.

An out-of-ground effect response is shown in Figure 9-2 along with the comparable in-ground-effect time history. The OGE response is highly damped and there is no tendency for the model to be excited by any other disturbance other than the initial, intentionally imposed upset.

9.2 EFFECT OF ADDED ROLL STIFFNESS

The neutrally damped oscillations observed in-ground-effect were easily controlled by adding a mounting spring in the roll axis. The mounting spring which provided the restraint characteristic discussed in Section 8-2 represented the attitude stiffness of a simple attitude feedback control system. Figure 9-1, Run 61, shows a damped 0.45 cps oscillation with a damping coefficient of 0.165. Since there is no tendency for the model to respond

to any unintentional disturbances, it is possible to eliminate IGE skittishness by providing attitude stiffness. The equivalent full-scale frequencies based on these test results would be sufficiently low (<0.15 cps) so that pilot control would be sufficient. It should be remarked here that the model was restrained laterally by the cable mount. Roll motion in-ground-effect resulting from a completely free model although coupled with an unrestrained lateral mode are not expected to create problems for the full scale aircraft.

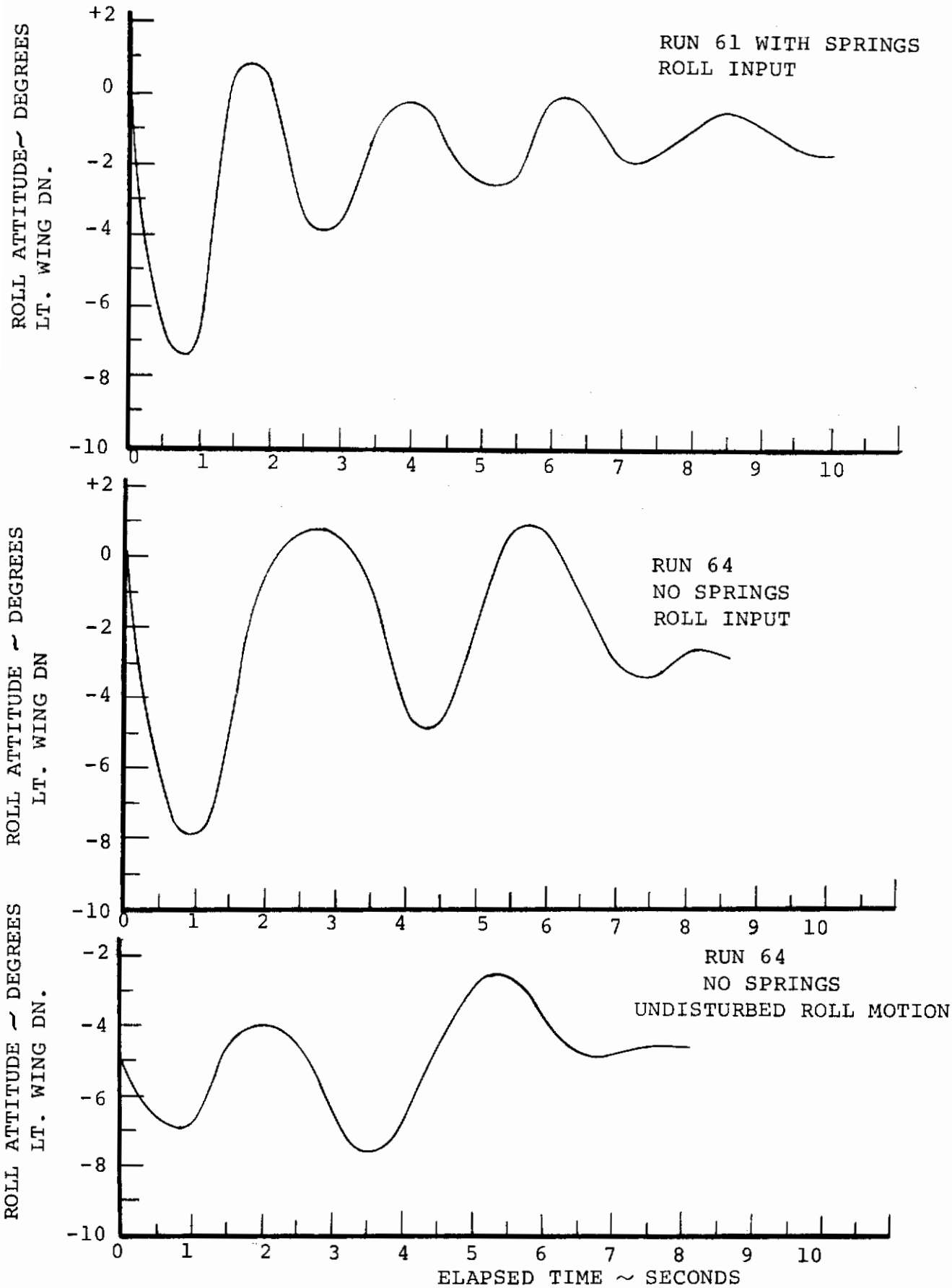


FIGURE 9.1: MODEL ROLL STABILITY IN GROUND EFFECT h/D = .41
HOVER, i_N = 90°, Ω = 825 RPM

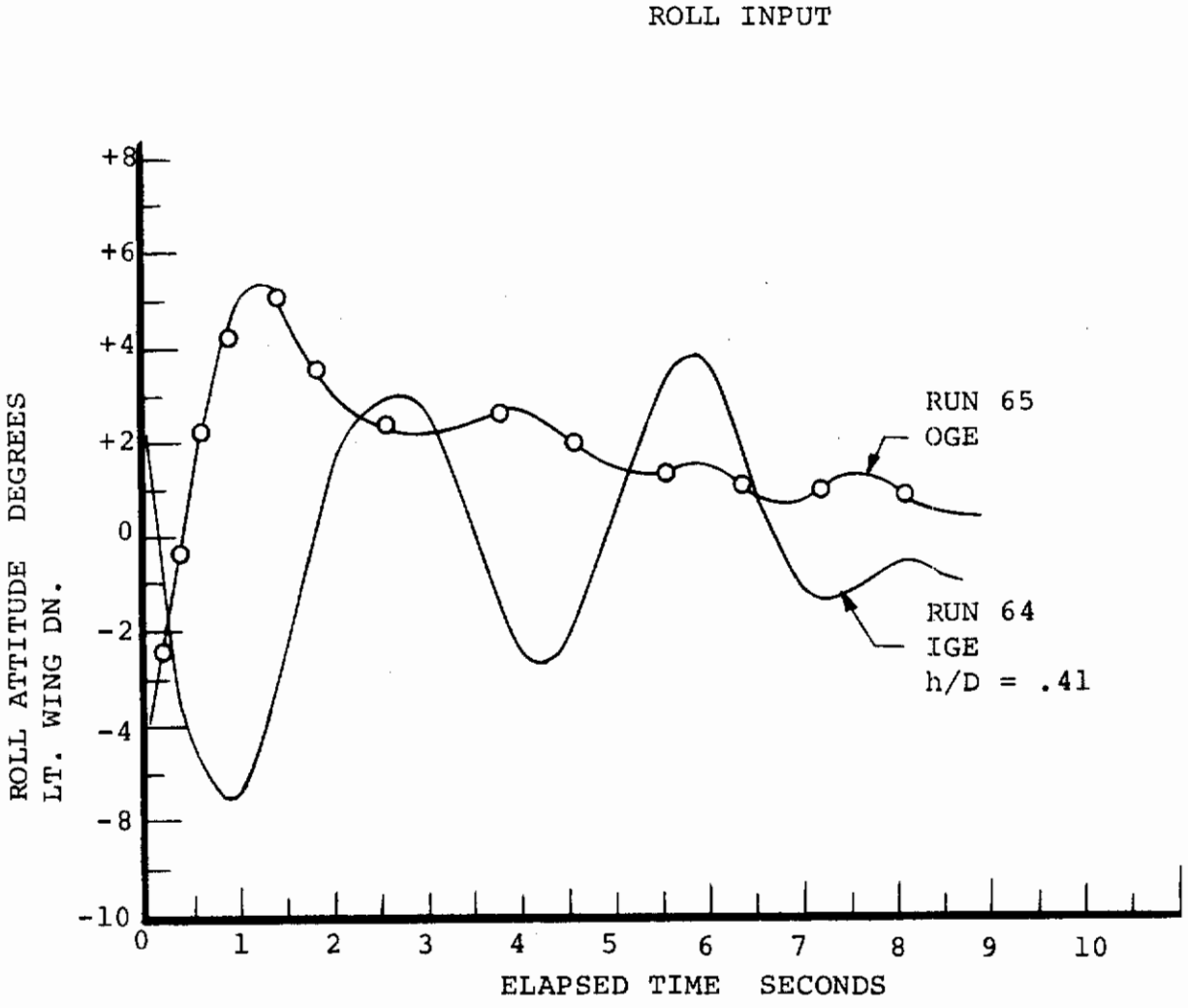


FIGURE 9.2. ROLL STABILITY IN AND OUT OF GROUND EFFECT WITHOUT SPRINGS HOVER, $i_N = 90°, \Omega = 825$ RPM

10.0 CONCLUSIONS

10.1 BLADE LOADS

1) The rotating coupled bending frequencies are accurately predicted by the coupled flap-lag frequency analysis.

2) The sensitivity of blade loads to cyclic pitch in hover are increased by increased collective pitch and are not affected by ground interference.

3) In transition for a specific nacelle tilt and advance ratio, there is a value of cyclic pitch at which alternating chord bending is minimized. For minimum chord bending load the value of cyclic pitch increases positively with increasing dynamic pressure, and the value of the minimum blade load decreases with increasing collective.

4) In all flight modes, alternating chord bending was more sensitive than alternating flap bending to blade section angle changes caused by cyclic pitch, Aq conditions, or model attitude.

10.2 VEHICLE DYNAMICS

1) As predicted, the model did not encounter classical flutter, whirl flutter, air resonance or divergence instabilities in the range tested.

2) Good agreement between measured and predicted wing frequencies was obtained.

3) Wing bending frequencies were found to be independent of nacelle tilt angles.

4) A low frequency limit cycle oscillation consisting predominantly of blade chord bending was identified at conditions near zero thrust at low tunnel speeds.

10.3 STABILITY AND CONTROL

1) Rigid body motions with the prop/rotors removed are well damped over the range of q investigated.

2) Some increase in the damping of rigid body motions is evident with the prop/rotors installed.

10.4 SKITTISHNESS

1) Without artificial damping in-ground effect disturbances are controllable. Oscillations are at least neutrally damped with an average frequency of 0.35 cps (model scale) and maximum amplitude of $\pm 2.0^{o}$.

2) In-ground effect disturbances can be well damped with a simple roll attitude feedback system.

11.0 RECOMMENDATIONS

It is recommended that in the next test the cyclic input axis should be varied in transition tests to determine effect on blade loads and optimum control effectiveness.

12.0 REFERENCES

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R. Hartman
Boeing-Vertol D160-10001-1, August 1969.

5. Wind Tunnel Results for the Model 160 Dynamic Propeller Blade Loads Extended Test Program, November 1968.
J. Zola
Boeing-Vertol D8-2475-1, June 1969.

13.0 APPENDICES

APPENDIX A - MODEL MASS PROPERTIES

1. MODEL COMPONENT WEIGHTS

All items listed were weighed on balance-arm platform scales during model assembly at the Vertol Wind Tunnel. Weights are given in pounds. Component center of gravities (C.G.) were determined either by balancing on knife edge or by suspension. The longitudinal locations of C.G. are given in inches from model nose.

		Weight (LB)
a)	Assembled Model Total Weight	61.0
b)	Major Components	
	(1) Nacelle and contents, including jet shaker, less rotor blades (C.G. on rotor ℄ , 4.55" above drive shaft ℄) Inertia about pivot =1.14 lb.in.sec^2	11.65
	(3) Rotor blades	1.20
	(1) Wing and contents	2.28
	(1) Fuselage including empennage	30.74

c) Sub-component		C.G.	Weight (LB)
Air motor and gearbox		24.45	7.44
Fuselage keel		25.68	5.43
Wing support:	horizontal web, fwd	23.90	0.36
	horizontal web, aft	28.95	5.36
	vertical web, L.H.	26.40	0.44
	vertical web, R.H.	26.40	0.44
Fuselage balsa section,	No.1	5.85	0.61
	2 & 8	13.23	0.57
	3 & 9	19.48	0.62
	4 & 10	24.08	0.62
	5 & 11	31.52	0.80
	6 & 12	37.15	0.61
	7 & 13	43.35	0.53
	14	51.75	0.62

	C.G.	Weight (LB)
Nose Landing Gear, less wheels	7.40	0.54
Axle + 2 tires	-	0.31
Main Landing Gear, less wheels	33.20	1.01
2 Axles + 4 tires	-	0.62
Landing gear damper bar		0.91
Misc. fuselage contents		0.65

d) External Umbilical Line

Air hose (each - 5/8" I.D.)	.13 lbs/ft
Instrumentation wire bundle	0.327 lbs/ft
Air hose fitting	1.18 lbs
Effective umbilical weight (estimated)	.88 lbs

2. MOMENTS OF INERTIA

Moments of inertia of a nacelle and the assembled model were determined by suspending the units on a two-string (bifilar) pendulum. The supports were equidistant from the center of gravity so the inertias given are about the C.G. The model was rotationally displaced and 20 free oscillations counted against a stopwatch to determine the period. The pendulum length and distance between the supports were measured.

	LB-IN-SEC^2
a. Nacelle, complete less rotor blades including shaker	
Pitching about pivot	I = 1.14
b. Model, complete less rotor blades	
Yawing about C.G., i_N=90°	I = 108.7
Rolling about C.G., i_N=0°	I = 82.8
Pitching about C.G., i_N=90°	I = 14.84
Pitching about C.G., i_N= 0°	I = 13.87

APPENDIX B - ROTOR BLADE AND WING PROPERTIES

Table B-1 and Figures B1 through B-3 define the properties of the rotor blade. Figures B-4, B-5, and B-6 define the wing stiffness of the model.

TABLE B-1
BLADE PROPERTIES

STATION r/R	CHORD $c/c_{.75}$	TWIST DEG.	MASS PROPERTIES WEIGHT LB./IN.	MASS PROPERTIES INERTIA $LB.IN^2/IN$	STIFFNESS PROPERTIES EI-FLAP $LB.IN^2 \times 10^4$	STIFFNESS PROPERTIES EI-CHORD $LB.IN^2 \times 10^4$	STIFFNESS PROPERTIES GJ $LB.IN^2 \times 10^4$
1.0					.0217	.2736	
.975	.815	-9.0	.0036	.0012			
.9375							.016
.9	.880	-5.6	.0056	.0014	.0234	.4770	
.85							.018
.8	.961	-1.9	.0075	.0022	.0279	.5364	
.75							.023
.7	1.036	1.5	.0093	.0034	.0358	.7128	
.65							.026
.6	1.114	4.9	.0112	.0050	.0378	.8946	
.55							.028
.5	1.196	8.3	.0130	.0071	.0392	1.0746	
.44							.030
.383					.0504	1.3140	
.38	1.291	12.5	.0150	.0089			
.315							.048
.267					.867	1.6704	
.25	1.394	18.0	.0208	.0110			
.21							.074
.17	1.458	23.6	.0241	.0125			
.15					.1330	2.520	.090
.13	1.480	27.0	.0248	.0076			
.1163							.100
.1025			.0253	.0060			
.096					.1196	1.8000	
.095							.106
.0875			.0200	.0043			
.08					.576	.1220	.132
.0725			.0071	.00007			
.0613							.132
.05			.0071	.00007			
.033					.576	.1220	.132
.033					.102	19.58	
.0175			.0071	.00007			
0					.102	19.58	

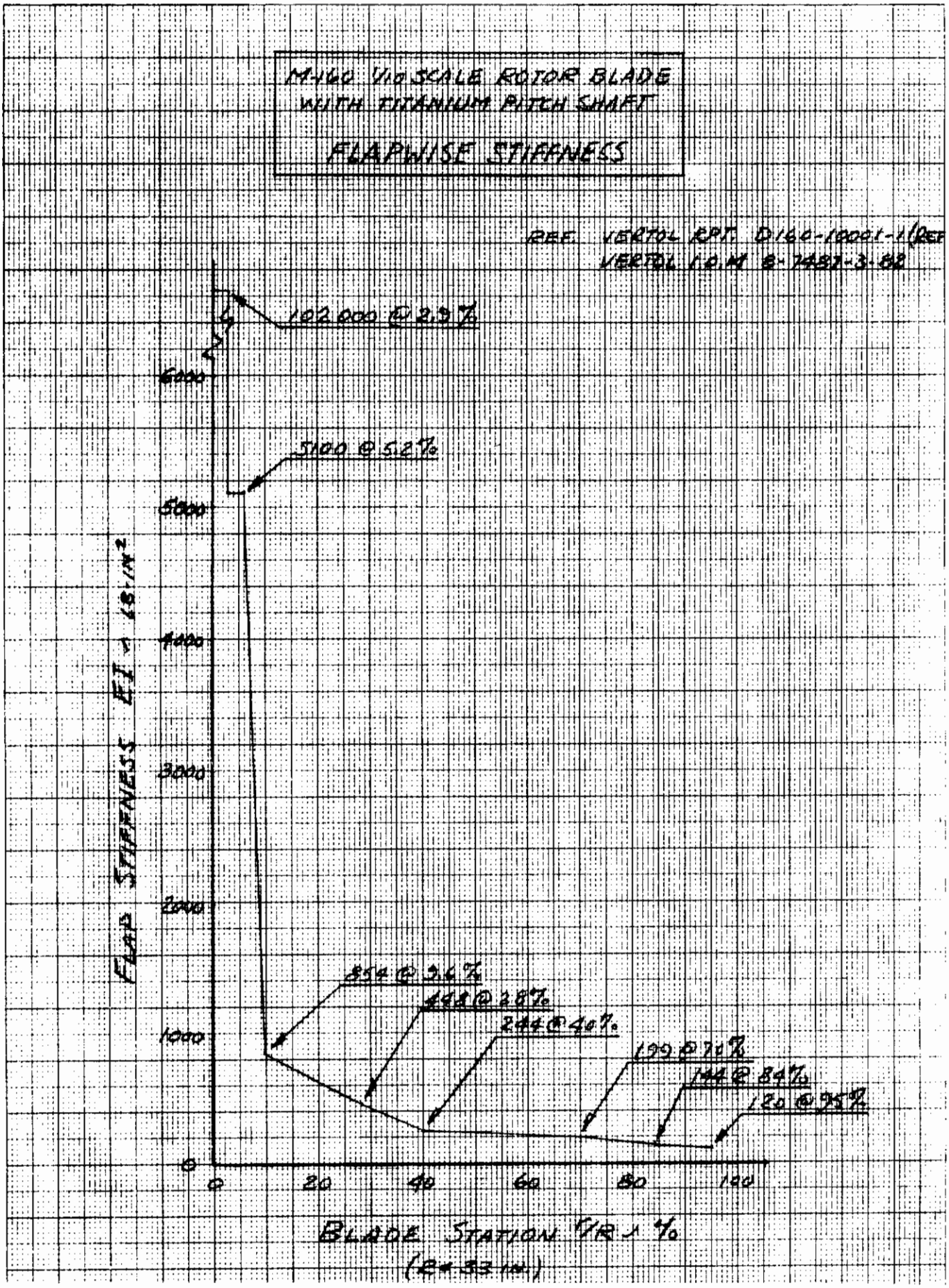

FIGURE B-1 1/10 SCALE M-160 ROTOR BLADE FLAPWISE STIFFNESS

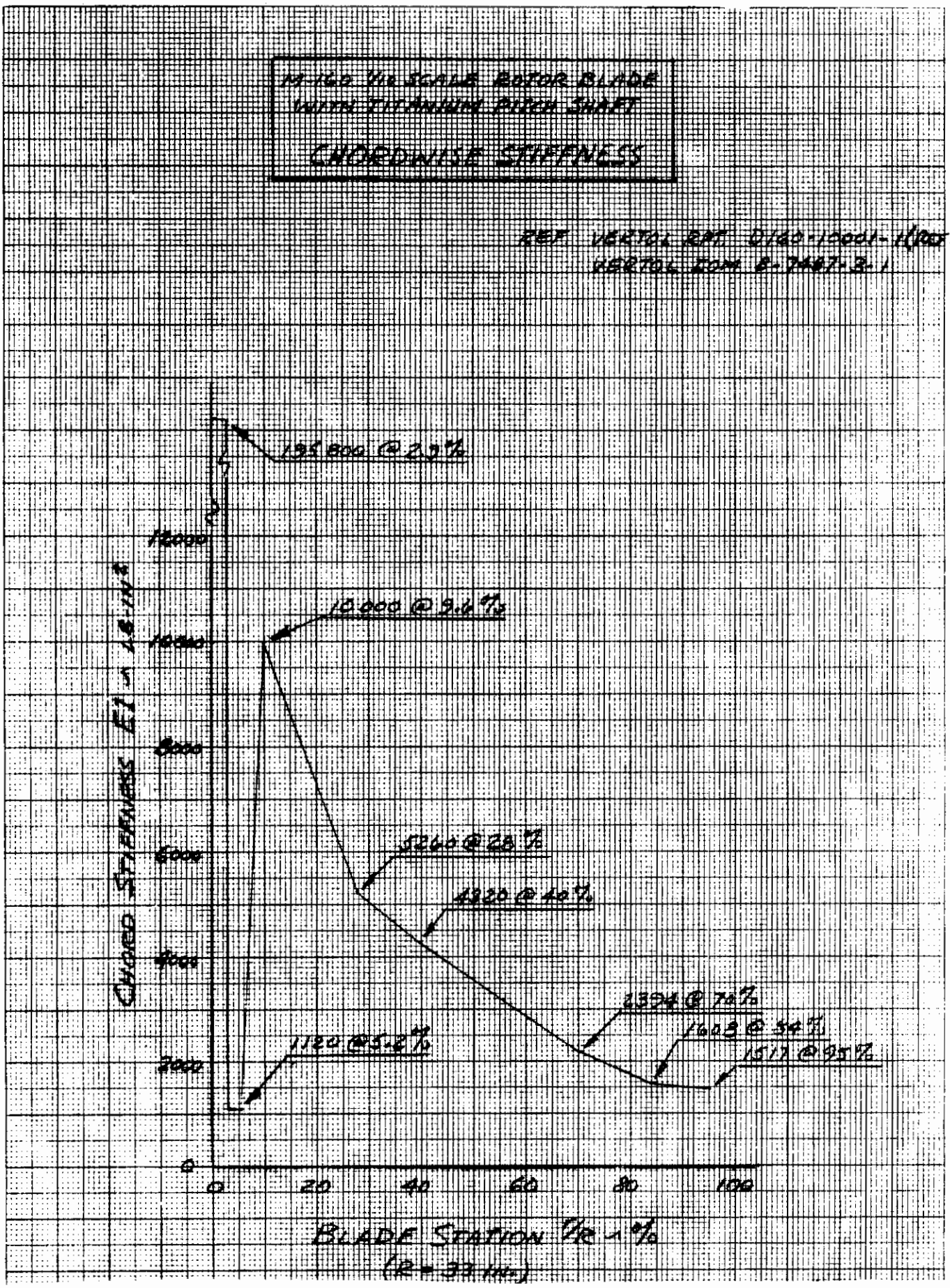

FIGURE B-2 1/10 SCALE M-160 ROTOR BLADE CHORDWISE STIFFNESS

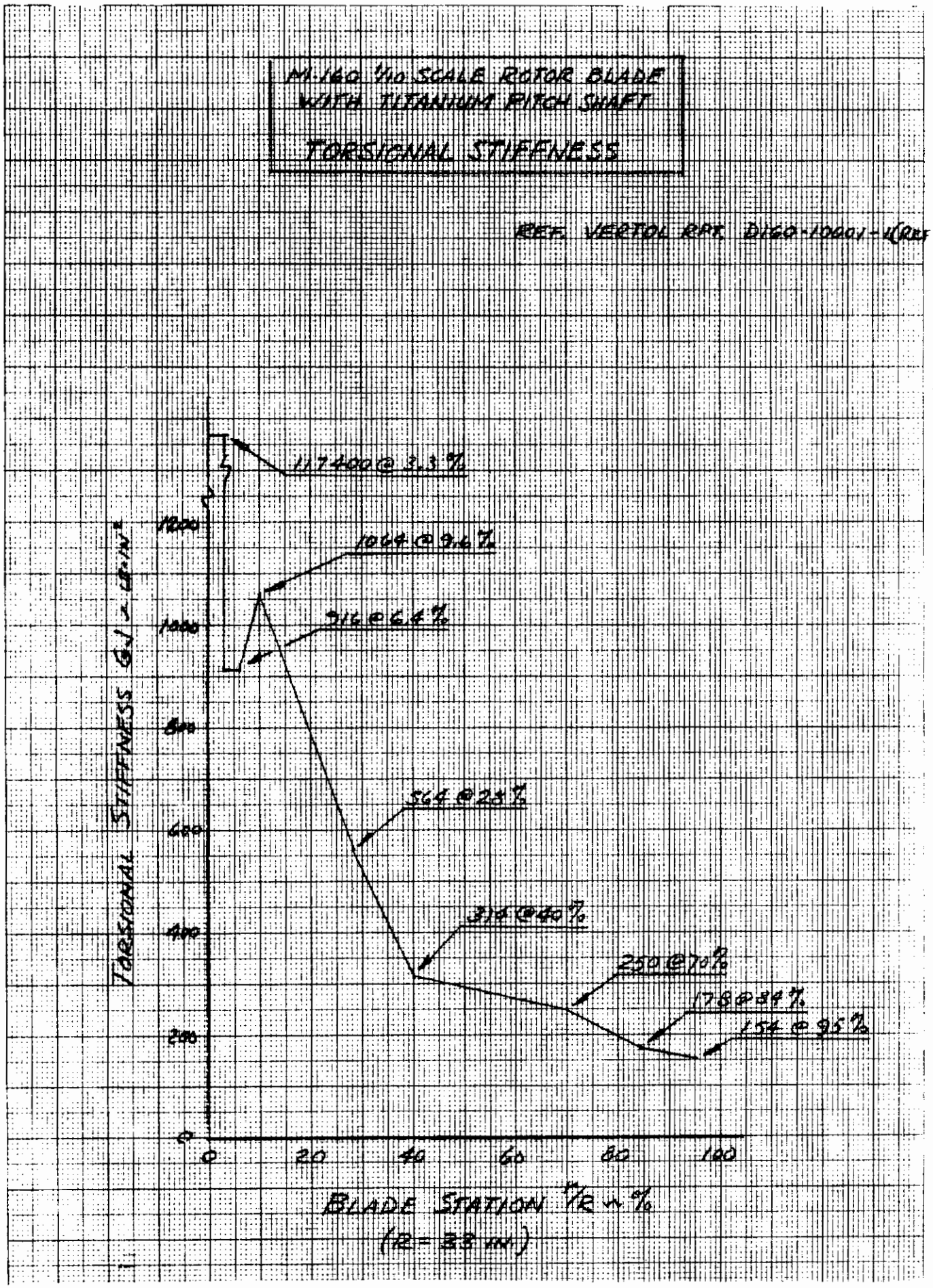

FIGURE B-3 1/10 SCALE M-160 ROTOR BLADE TORSIONAL STIFFNESS

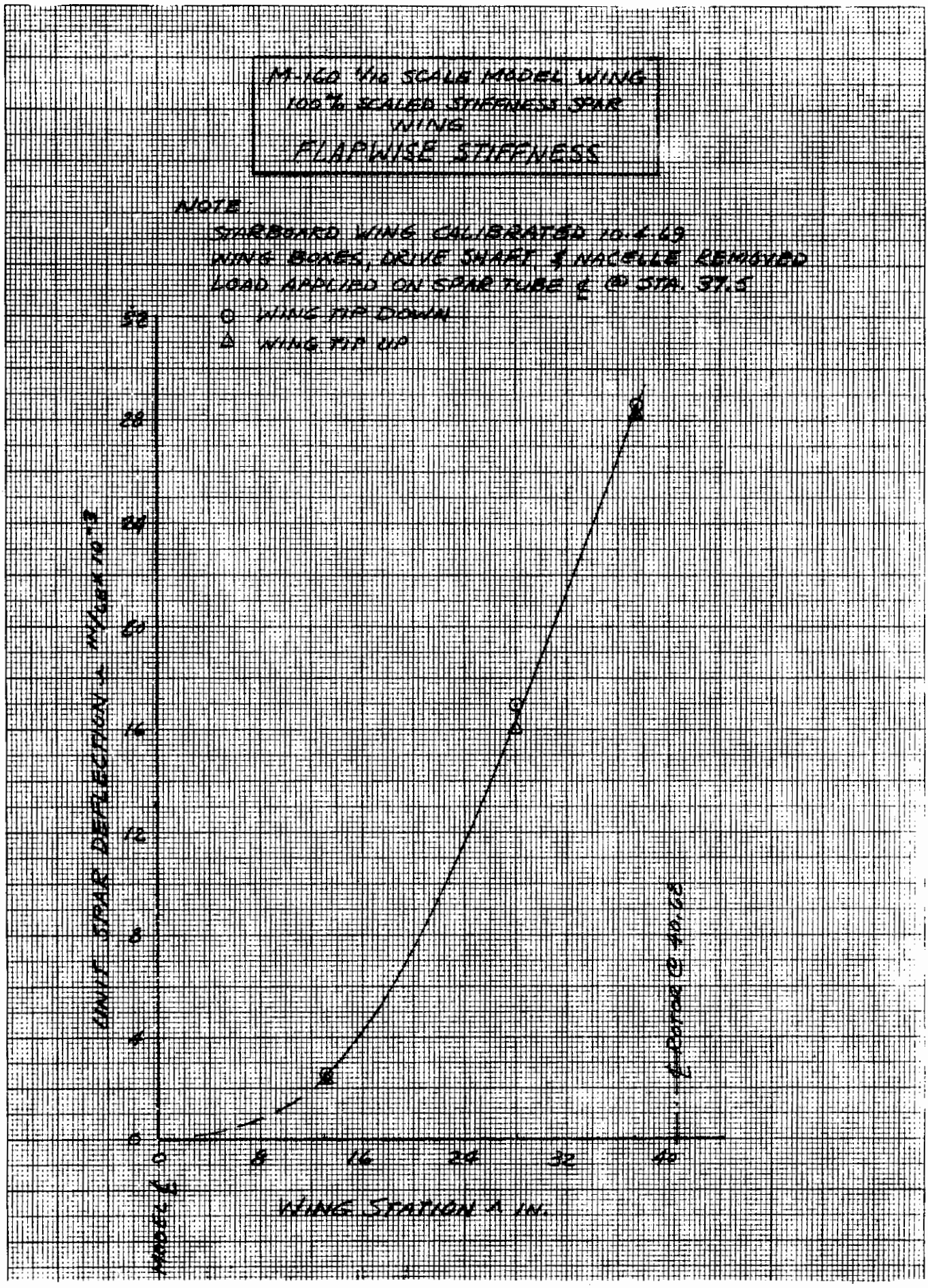

FIGURE B-4 1/10 SCALE M-160 WING FLAPWISE STIFFNESS

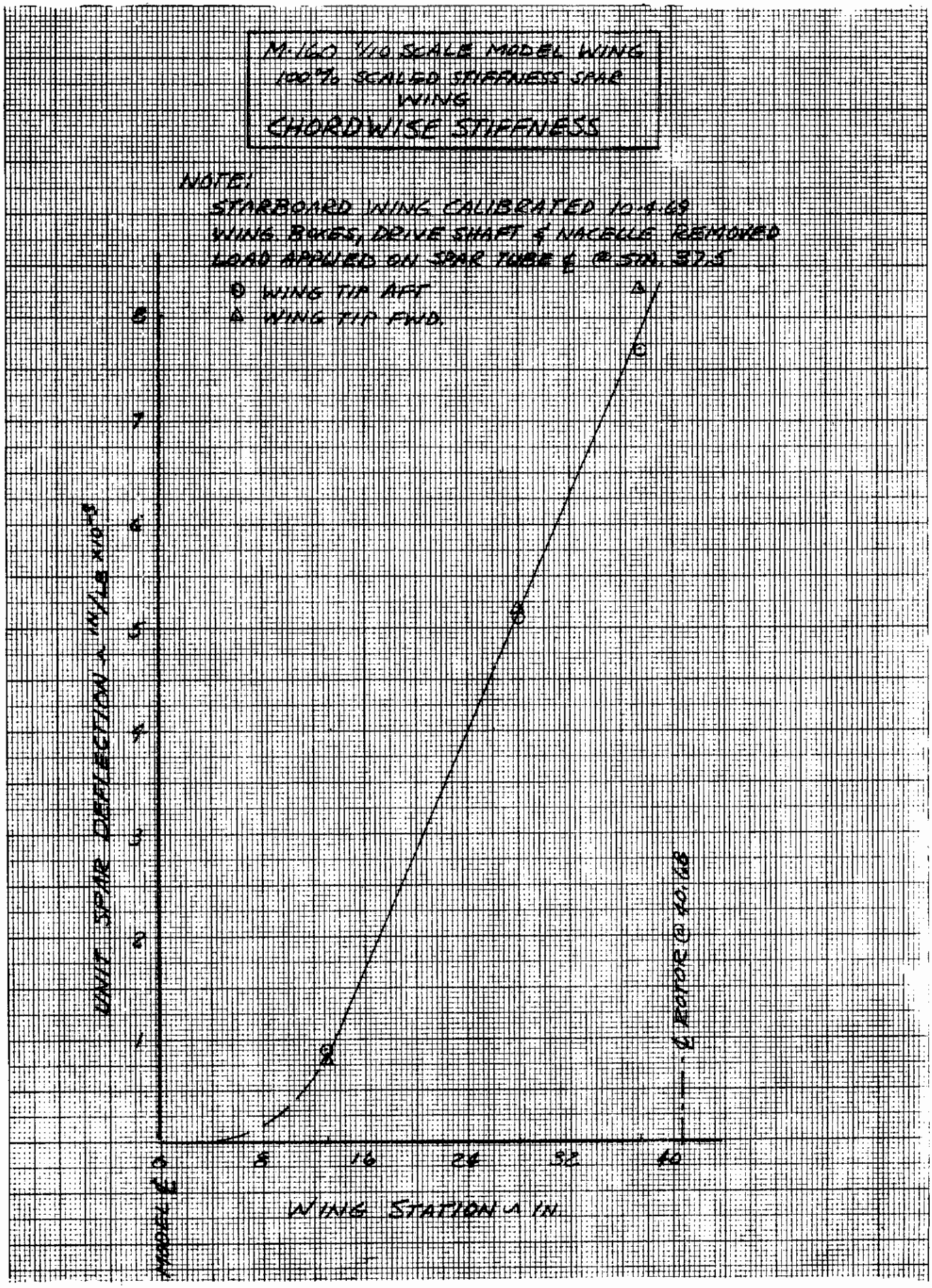

FIGURE B-5 1/10 SCALE M-160 WING CHORDWISE STIFFNESS

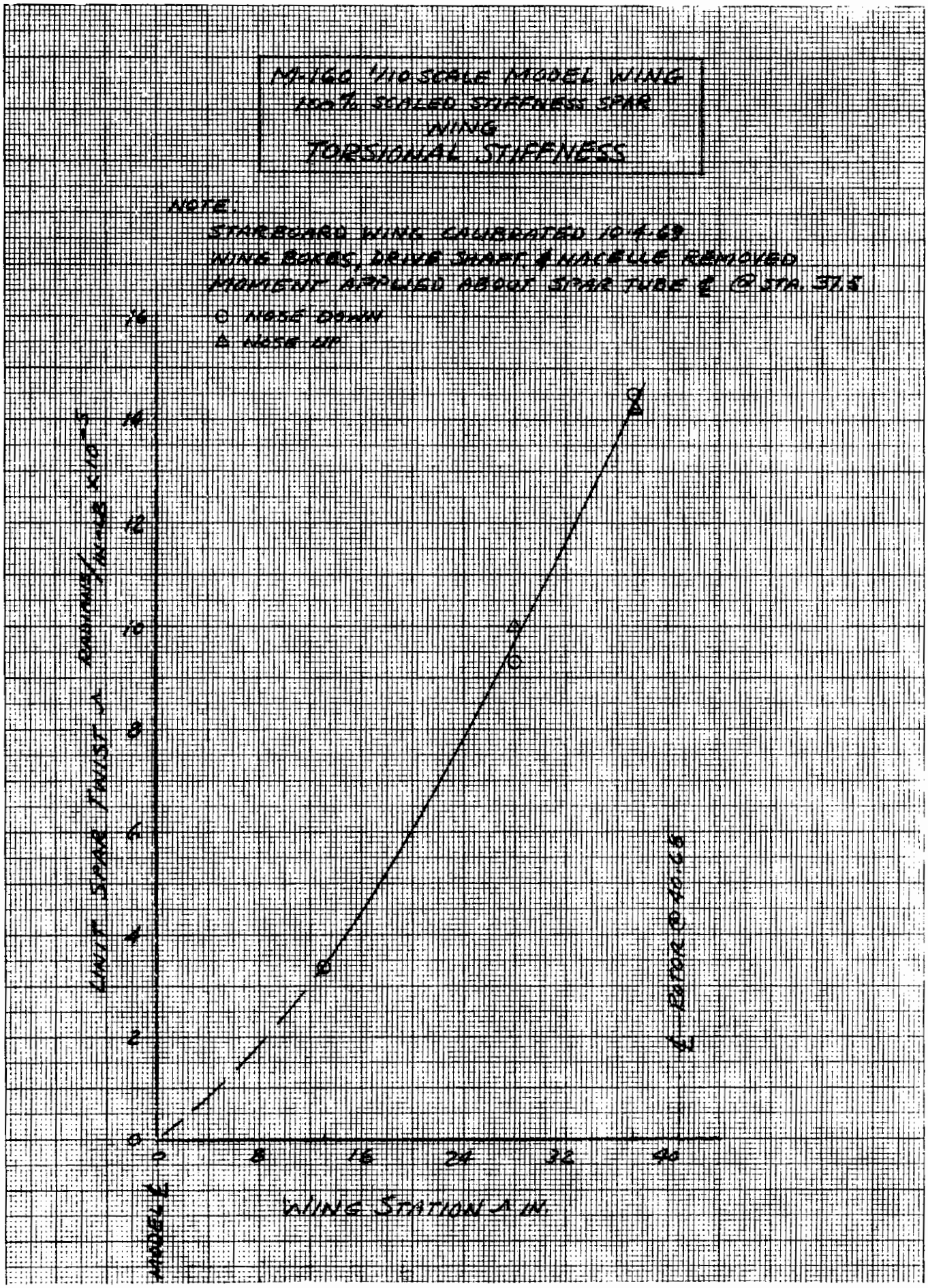

FIGURE B-6 1/10 SCALE M-160 WING TORSIONAL STIFFNESS

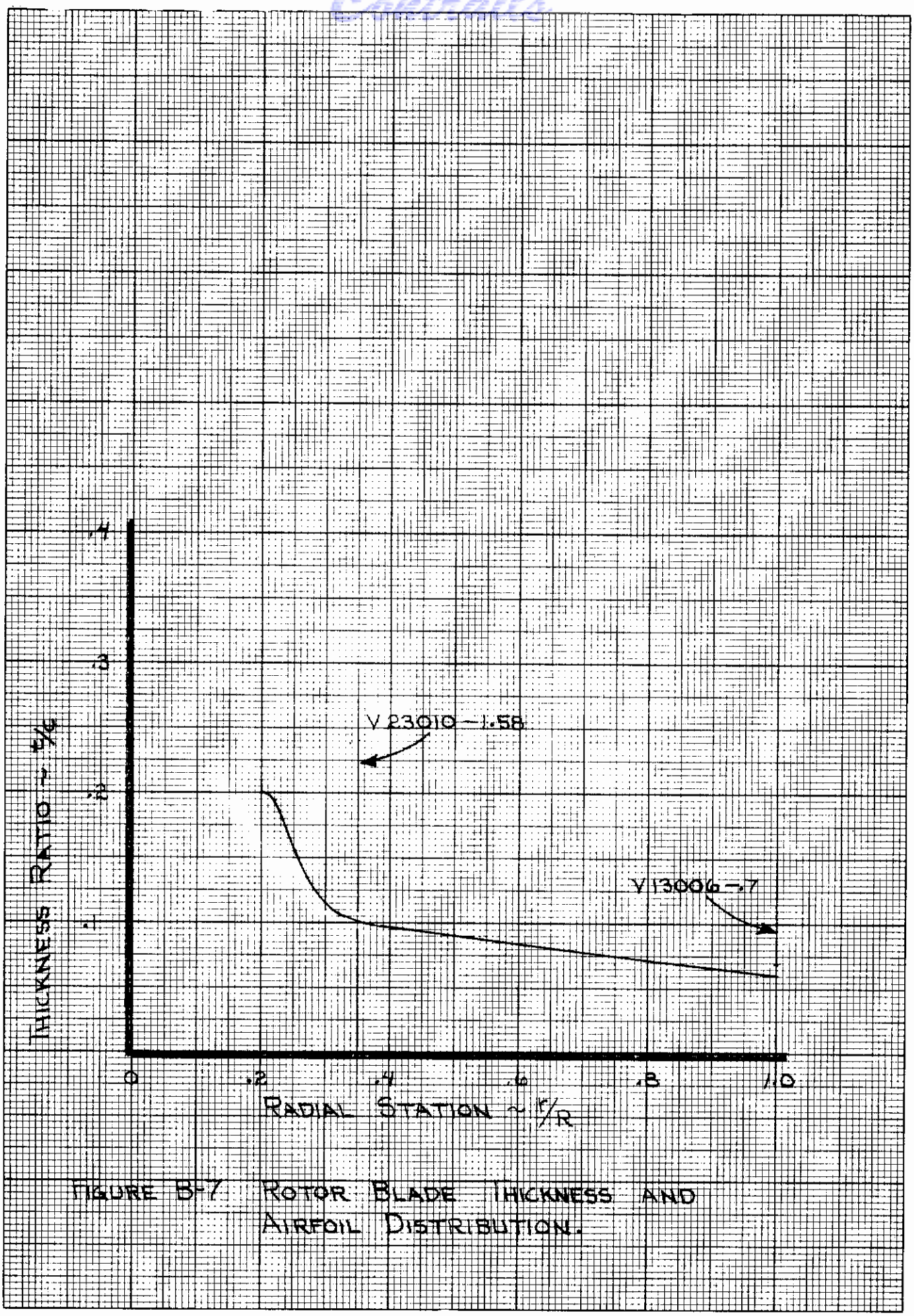

FIGURE B-7 ROTOR BLADE THICKNESS AND AIRFOIL DISTRIBUTION.

APPENDIX C - RUN LOG

A copy of the run log recorded during the test is enclosed in the following pages. This log lists the test conditions by run number and contains added descriptive notes.

FORM 49810 (3/70)

PREP.	CHK.	APPR	REVISED / DATE

RUN NO.	CONFIGURATION	TYPE OF RUN	WT. TARE RUN	R.R. Θ_{75} DEG	R.R B_1 DEG	L.R Θ_{75} DEG	L.R B_1 DEG	RPM	δ_3 DEG	i_N DEG	$\frac{q}{\delta}$ PSF	J		DATE / TIME
1	R.H.R	BAFFLE TEST		3	0	OFF	OFF	VARY	0	90	0	0		1-19-71
2	R.H.R. WITH BAFFLE AT $\psi = 90°$	″		3	″	″	″	″	″	″	″	″		″ ″ ″
3	R.H.R WITH BAFFLES AT $\psi = 90$ & 270	″		3	″	″	″	″	″	″	″	″		″ ″ ″
4	R.H.R. WITH BAFFLES AT $\psi = 90, 210, 330$	″		3	″	″	″	″	″	″	″	″		″ ″ ″
5	R.H.R. WITH BAFFLES AT $\psi = 90, 180, 270$	″		3	″	″	″	″	″	″	″	″		″ ″ ″
6	SEE NOTE BELOW.			—	—	—	—	—	—	90	—	—		1-25-71 [illegible]
7	SEE NOTE BELOW			—	—	—	—	—	—	0	—	—		1-26-71 15:35
8	SEE NOTE BELOW			—	—	—	—	—	—	0	—	—		″ 19:30
9	SEE NOTE BELOW			—	—	—	—	—	—	0	—	—		″ 19:40
10	SEE NOTE BELOW			—	—	—	—	—	—	0	—	—		″ 19:50

1. MODEL LOCATED IN ASSEMBLY AREA FOR BAFFLE TEST NO BAFFLE-ROTOR ℄ 5.5 FT ABOVE FLOOR
2. V-CAL + BLADE BANG TEST ON CEC, RHR BLADE ONLY
3. BLADE BANG TEST ON CEC. RHB ONLY. BAFFLES AT $\psi = 90$ & 270°
4. BAFFLES AT $\psi = 90, 210$ & 330°
5. BAFFLES AT $\psi = 99, 189, 270$ & 0°
6. MODEL ON MONKEY WITHOUT UNBILICAL AND SNUBBERS. RIGID BODY FREQUENCY AND DAMPING CHARACTERISTICS. WERE OBTAINED.
7. SAME AS 6 EXCEPT $i_N = 0°$
8. MODEL ON MONKEY WITH UNBILICAL ATTACHED. AIR HOSES TO MOTOR AND SHAKERS PRESSURISED. EFFECTS OF UNBILICAL AND PRESSURE ON RIGID BODY FREQUENCY AND DAMPING CHARACTERISTICS WERE OBTAINED. SNUBBERS OFF.
9. SAME AS 8 EXCEPT SNUBBERS ARE ON AND LOOSE
10. SAME AS 9 EXCEPT MODEL IS SNUBBED

1/10 SCALE 160 MODEL
VR054D

BVWT 070

RUN NO.	CONFIGURATION	TYPE OF RUN	WT. TARE RUN	R.R. θ_{75} Deg.	R.R. B_1 Deg.	L.R. θ_{75} Deg.	L.R B_1 Deg.	RPM	δ_3 Deg.	i_N Deg.	$\frac{g}{D}$ PSF	J		DATE TIME
11	SEE NOTE BELOW			–	–	–	–	–	–	90	–	–		1-27-71; 22:50
12	SEE NOTE BELOW			–	–	–	–	–	–	90	–	–		1-28-71; 18:00
13	SEE NOTE BELOW			–	–	–	–	–	–	90	–	–		19:00
14	SEE NOTE BELOW			–	–	–	–	–	–	90	–	–		19:25
15	HOVER	TRACK BAL.		4	0	4	0	830	0	90	0	0		23:30
16	〃	〃		0	0	0	–	830	0	90	〃	〃		1-29-71 16:30
17	〃	〃		4	–	4	–	830	0	90	〃	〃		
18	〃	〃		2.5	〃	4.6	〃	830	0	90	〃	〃		
19	〃	〃		2.5	〃	〃	〃	830	0	90	〃	〃		
20		〃		2.5	〃	〃	〃	758	OFF	90	〃	〃		19:15

11. MODEL ON MONKEY WITH UNBICIAL ON AND SNUBBERS OFF. AIR HOSE TO SHAKERS ARE PRESSURISED. AIR HOSE TO MOTOR IS NOT PRESSURISED. MODEL IS IN HOVER CONFIGURATION. SHAKER FREQUENCY FROM 0 TO 22 CPS WITH PRESSURES OF 20 AND 40 CPS. SYMMETRICAL AND ANTI-SYMMETRICAL SHAKE.

12. SAME AS 11 EXCEPT DATA NOW BEING RECORDED ON COMPUTER. SYMMETRICAL SHAKE ONLY AND 40 PSI SHAKER PRESSURE.

13. SAME AS 12 EXCEP ANTI-SYMMETRICAL SHAKE.

14. SAME AS 12. BLADE BANG TEST PRIOR TO THIS RUN

15. TRACK AND BALANCE RUN.

16 TRACK, BALANCE AND HORIZONTIAL TAIL VIBRATION TEST VERTICAL AND HORIZONTIAL TAIL REINFORCED

17 〃 〃 〃 〃 〃 〃 〃 〃 〃 〃

18 VERTICAL TAIL AND HORIZONTIAL TAILS REINFORCED ADDITIONAL REINFORCEMENT ON HORIZONTIAL TAIL AND BALANCE WTS ON L.H. ROTOR

19. BALANCE WTS. OFF- TRACK CHECK.

20. ADDITIONAL STIFFNESS REMOVED FROM HORIZONTIAL TAIL- TRACK ADJUSTMENT TO L.H. ROTOR PRIOR TO THIS RUN. HORIZONTIAL TAIL REMOVED.

APPR. CHK. PREP.

1/10 SCALE 160 MODEL
VR05AD
BYWT 070

RUN NO.	CONFIGURATION	TYPE OF RUN	WT. TARE RUN	R.R. θ_{75} Deg	R.R. B_1 Deg	L.R. θ_{75} Deg	L.R. B_1 Deg	RPM	δ_3 Deg	i_N Deg	$\frac{q}{\delta}$ PSF	J		DATE TIME
21	HOVER	Hover		0	0	0	0	VARY	OFF	90	0	0		1-29-71; 19:40
22	"	"		VARY	0	VARY	0	750	OFF	90	0	0		" 21:40
23	"	"		3.2	VARY (-4 to 2)	4.0	VARY	750	OFF	90	0	0		" 21:53
24	"	"		VARY	VARY	VARY	VARY	750	OFF	90	0	0		" 23:31
25	"	"		3.2	VARY	4.3	VARY	750	OFF	90	0	0		" 23:38
26	"	"		8.1	-1.0	VARY	-1.0	750	OFF	90	0	0		"
27	"	"		8.1	-1.0	10.2	-1.0	750	OFF	90	0	0		" 23:66
28	"	"		3.2	-.5	4.0	-1.0	750	OFF	90	0	0		END 24:45
29	"	"		"	-.5/4.5	"	-1.0/-4.5	"	0	90	"	"		FEB 1, 11:00 16:30

21. HORIZONTIAL TAIL OFF FOR ALL HOVER RUNS. AFTER RUN FOUND CUFF ON RB6 RUBBING AGAINST THE SPINNER. ALSO NOTICED HIGH BLADE TORSION LOADS FOR THIS BLADE DURING RUN.

23. B_1 FROM -4 TO 2 DEG.

24. θ_{75} R.R θ_{75} L.B. B_1 R.R B_1 L.R.

5.8°/8.4° 4.5°/8.7° -1.6°/0° -4.5°/0° RANGE

COULD NOT DECREASE L.R. θ_{75} AFTER THIS RUN DOWN FOR REPAIR

26. DIFFRENTIAL COLLECTIVE

27. RIGID BODY FREQUENCIES

28. 1/REV IN COMPUTER REPLACED WITH OSCILATOR-FREQUENCY VARIED. BLADE FREQ. BANG TEST AFTER THIS RUN.

29 VARIOUS RUN-UPS TO INVESTIGATE HORIZONTIAL TAIL VIBRATION ADDED 45 gr. POINT MASS TO TIP (ONE EACH SIDE) OF HORIZONTIAL TAIL.

1/10 SCALE 160 MODEL VR054D

BVWT 070

FORM 4971 3-70

PREP. | CHK. | APPR.

REVISED LAT.

RUN NO.	CONFIGURATION	TYPE OF RUN	WT. TARE RUN	R.R. θ_{75} Deg.	R.R. B_1 Deg.	L.R. θ_{75} Deg.	L.R. B_1 Deg.	RPM	δ_3 Deg.	i_N Deg.	$\frac{T}{\delta}$ PSF	J		DATE	TIME
30	HOVER			VARY	VARY	VARY	VARY	750	0	90	0	0		2-1-71	19:30
31	HOVER			VARY	VARY	VARY	VARY	750	0	90	0	0		"	20:15 [illegible] 43
32	[redacted] TRANS			10/4	-1.5	10/4	-1.5	750	0	90	-.06				21:47
33	TRANS.			4.0	VARY	4.7	VARY	750	0	90	.06				22:51 23:01
34	CRUISE			VARY	-.5	VARY	-.6	790	0	0	0/4.2				00:52 01:13
35	CRUISE			4/24	-.6	4/25	-.6	790	0/-6.5	0	0/4.5			2-2-71	19:18
36	CRUISE			4.8	-.6	4.8	-.6	VARY	-6.5	0	0				22:55
37	CRUISE			4.8/30	VARY	4.8/30	VARY	790	-6.5	0	0/6				23:15
38	CRUISE			27/32	-1.5	27/32	—	790	-6.5	0	0/7				00:50

1/10 SCALE 160 MODEL
YROSAD

30. ELECTRICAL POWER TO AIR OIL MIST LOST DURING RUN. MODEL WITHOUT AIR-OIL MIST FOR 17 MIN. ALSO A MECHANICAL PROBLEM WITH L.H. B_1 CHANGING WHILE RUNNING. DOWN TO REPAIR B_1 PROBLEM.

32. HIGH BLADE LOADS (100%)

33. PITCHED FUSELAGE TO -10° BLADE LOADS STILL HIGH. 150%.

34. (RPM = 790) COULD NOT RUN AT 750 DUE TO WING SYMETRICAL BENDING AND HIGH C.B. LOADS ON L.H. BLADE.

35. YAW SNUBBERS ADDED

37. LOST AIR PRESSURE TO MODEL. ACTIVATED EMERGENCY TUNNEL SHUT DOWN. ALL LOADS LOW DURING SHUT DOWN — LOOKS LIKE A GOOD WAY TO SHUT DOWN FROM HIGH $\frac{T}{\delta}$ CONDITIONS. [LEFT HAND CYCLIC PITCH READOUT IS INOPERATIVE]

38. PROBLEM WITH DRIVE SYSTEM AT $\theta_{75} = 32°$ AND $\frac{T}{\delta} = 7$. AFTER RUN FOUND TRANSMISSION SHEAR PIN SHEARED.

[redacted]

BVWT 070

Controls

FORM 4[illegible]

APPR	CHK.	PREP.

RUN NO.	CONFIGURATION	TYPE OF RUN	WT. TARE RUN	R.R. $\theta_{.75}$ Deg	R.R. B_1 Deg	L.R. $\theta_{.75}$ Deg.	L.R. B_1 Deg.	RPM	δ_s Deg	i_N * Deg.	$\frac{q}{\delta}$ PSF	I		DATE TIME
39	TRANSITION			VARY	VARY	VARY	VARY	790	0	70	0/2			2-3-71 17:20
40	TRANSITION			10	VARY	9.75	VARY	790	0	70	0/2.75			" 18:30
41	TRANSITION			VARY	VARY	VARY	VARY	790	-10	70	0/4			" 19:30
42	TRANSITION			VARY	VARY	VARY	VARY	790	-10	70	0/3			" 21:51
43	TRANSITION			VARY	VARY	VARY	VARY	790	-3.8	40	0/5.2			" 23:25
44	SEE NOTE BELOW.			5.9	-.63	5.4	-.72	790	-4.2	40	0			" 00.38
45	SHAKE TEST CRUISE MODE			6.0	-.8	5.6	-.5	790	-4.2	0	0			2-4-71 14:37
46	SHAKE TEST CRUISE MODE			"	"	"	"	"	"	"	"			" 15:47
47	CRUISE			10/rest	-.3	10/rest	-.5	0/900	-8.2	0	0/20			" 18:58
48	CRUISE			28.2	0	29.3	0	790	-4	0	0/10			" 21:51

1/10 SCALE 160 MODEL
VR054D

40 YAW, ROLL AND PITCH SWEEPS AT $\frac{q}{\delta}$ = 2 PSF

41 YAW AND PITCH SWEEPS

42 DECREASED THE TENSION ON VERTICAL SUPPORT CABLE

43. YAW AND PITCH SWEEPS AT $\frac{q}{\delta}$ = 4 AND 5.25 PSF

44 YAW, PITCH AND ROLL DISPLACEMENTS TO OBTAIN RIGID BODY STABILITY ALSO SHAKER FREQUENCY SWEEP FROM 4 TO 18 CPS. SYMETRIC SHAKE AT 40 PSI

47 DURING AN ATTEMPT TO WINDMILL UP TO 790 RPM - TWO BLADES ON THE R.H.S (LB 14 & LB 22) CONTACTED THE WING L.E. AND BROKE OF THE BLADE TIPS - SLIGHT OVERSPEED DUE TO R.H. ROTOR COLLECTIVE PITCH MOTOR SHORTING OUT. PRIOR TO RUN 45 THE RIGHT HAND ROTOR WAS STATICALLY BALANCED BY ADDING 32 GMS TO BLADES LB14 & LB. 22. ∎ PRIOR TO RUN 47 ALL LISTED B_1 FOR R.H. ROTOR ARE IN ERROR BY A FACTOR OF 2.0

48. SYMETRICAL SHAKE AT $\frac{q}{\delta}$ = 7.0

BVWT 070

* ALL i_N LISTED AS 70° SHOULD BE 60°.

FORM 4911 (3-67)

APPR	CHK.	PREP.	RUN NO.	CONFIGURATION	TYPE OF RUN	WT. TARE RUN	R.R. θ_{75} Deg.	R.R. B_1 Deg.	L.R. θ_{75} Deg	L.R. B_1 Deg	RPM	δ_3 Deg	z_N Deg	$\frac{q}{p}$ PSF	J		DATE	TIME
			49	CRUISE			28.2	0	29.3	0	790	-4	0	7			2-4-71	22:30
			50	CRUISE			32.9	0	33.6	0	790	-4	0	10			↙	23:00
			51	HOVER			4/15	VARY	4/15	VARY	0/900	0	90	0			↙	00:25
			52	HOVER			10.0	-.35	10.0	1.0	825	0	90	0			2-5	13:43
			53	HOVER			11.8	-.35	10.0	1.0	825	0	90	0			↙	15:10
			54	HOVER			4	-.35	4	-.35	—	0	90	0				18:45
			55	AUTOROTATION			VARY	0	VARY	0	—	0	105	1				23:27
			56	HOVER			VARY	0	VARY	0	825	0	90	0			2-8	15:00
			57	CRUISE- ROTORS OFF			—	—	—	—	—	VARY	0	0/10			↙	23:09
			58	HOVER- IN GROUND EFFECT			4.0	0	4.0	0	825	0	90	0			2-9	0030

1/10 SCALE 160 MODEL
VRO54D

49. ANTISYMMETRICAL SHAKE AT $\frac{q}{p}$ = 7
RIGID BODY PITCH [redacted] DISPLACEMENT

50. RIGID BODY PITCH DISPLACEMENT

51 RIGID BODY PITCH DISPLACEMENT AT VARIOUS θ_{75} & RPM.

52 SYMMETRICAL SHAKE. STATICALLY BALANCED L.H. ROTOR (23 gm on RB1 & 28 on RB5)

53. ANTI SYMMETRICAL SHAKE- TRANSMISSION SHEAR PIN SHEARED ON THIS RUN.

54. THE SHEAR PIN WAS NOT SHEARED IN RUN 53 - THIS WAS A TRANSMISSION PROBLEM. SPROCKET HAD SLIPPED OFF SHAFT.

55 MOTOR & TRANSMISSION HAVE BOTH BEEN REMOVED FROM THE MODEL ATTEMPED TO AUTOROTATE HOWEVER WAS UNSUCCESSFUL IN GETTING THE ROTORS UP TO SPEED.

56 DURING HOVER RUN-UP DIFFRENTIAL COLLECTIVE PITCH WAS INADVERTENTLY APPLIED CAUSING THE ROLL SNUBBER TO BREAK RESULTING IN LOSS OF THE L.H. ROTOR AND THE CROSS SHAFT TO BREAK.

57. RIGID BODY STABILITY ROTORS OFF.

58. TRACK RUN.

BVWT 070

RUN NO.	CONFIGURATION	TYPE OF RUN	WT. TARE RUN	R.R. $\theta_{.75}$ Deg.	R.R. B_1 Deg.	L.R. $\theta_{.75}$ Deg.	L.R. B_1 Deg.	RPM	δ_3 Deg.	i_N Deg.	$\frac{g}{g}$ PSF	J		DATE TIME
59	HOVER- IN GROUND EFFECT			8.8	0	4.4	0	825	0	90	-	-		2-10-71/0935
60	" " " "			8.8/10.6	0	4.4/9.5	0	825	0	90	-	-		10:23
61	" " " "			4.8/14	VARY	-.8/12	VARY	825	0	90	-			11:22
62	" " " "			4/11	VARY	4/9.6	-.7	825	0	90	-	-		13:46
63	" " " "			11.3	-.65	9.6	-.7	825	0	90	-	-		14:33
64	" " " "			VARY	VARY	VARY	-.7	825	0	90	-	-		15:14
65	HOVER			VARY	VARY	VARY	-.7	825	0	90	-	-		17:51
66	~~HOVER~~ TRANSITION			11.9	VARY	9.2	VARY	825	18	60	0/3	-		19:01
67	TRANSITION			11.9	VARY	9.2	VARY	825	18	40	0/2			

1/10 SCALE 160 MODEL
VR054D

59 TRACK RUN

60 RIGID BODY STABILITY DATA IN ROLL AND PITCH

61 RIGID BODY STABILITY DATA IN ROLL AND PITCH - WITH & WITHOUT TENTION IN YAW SNUBBERS.

62 REMOVED PITCH AND ROLL SPRINGS
RIGID BODY PITCH AND ROLL STABILITY DATA.

63 REPEATE OF RUN 62 - NO DATA.
BECAUSE LEFT WING WAS AGAINST ROLL STOPS.

64. REPEATE OF RUN 62

65. RIGID BODY PITCH AND ROLL STABILITY.

66 YAW DISPLACEMENTS

67. RIGID BODY PITCH OSCILLATION NEAR 1.7 g RESULTING IN HIGH BLADE LOADS. VISSUALY IT SEEMS TO BE A LIMIT OSCILLATION AT ABOUT 1 CPS WITH THE MODEL SNUBBED.

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13. ABSTRACT

This report presents the results of a wind tunnel test on a powered dynamic model of the Boeing M-160 tilt rotor aircraft with 5.5 foot diameter rotors. The model was tested in the Boeing V/STOL 20 x 20 foot wind tunnel during January-February 1971 and was supported to simulate free flight conditions with mount frequencies much lower than the dynamic aircraft frequencies. Blade loads, wing loads, flying qualities and skittishness in ground effect data were obtained.

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14. KEY WORDS	LINK A		LINK B		LINK C	
	ROLE	WT	ROLE	WT	ROLE	WT
Stowed Rotor Hover Cruise Blade Frequency Stall Flutter Tilt Rotor Transition Rotor Loads Skittishness Flying qualities						–

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