

**AFFDL-TR-72-42  
VOLUME II**

**MAGIC III: AN AUTOMATED GENERAL PURPOSE  
SYSTEM FOR STRUCTURAL ANALYSIS**

**VOLUME II: USER'S MANUAL**

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## FOREWORD

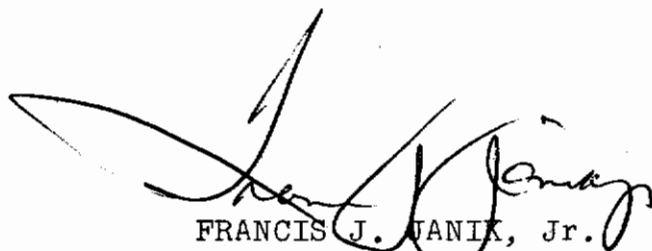
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This report, "MAGIC III: An Automated General Purpose System for Structural Analysis" is published in three volumes, "Volume I: Engineer's Manual", "Volume II: User's Manual", and "Volume III: Programmer's Manual". The manuscript for Volume II was released by the authors in July 1972 for publication as an AFFDL Technical Report.

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This technical report has been reviewed and is approved.



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## ABSTRACT

An automated general purpose system for analysis is presented. This system, identified by the acronym, 'MAGIC III' for Matrix Analysis via Generative and Interpretive Computations, is an extension of the structural analysis capability available in the initial MAGIC System. MAGIC III provides a powerful framework for implementation of the finite element analysis technology and provides diversified capability for displacement, stress, vibration, and stability analyses.

Additional elements have been added to the MAGIC element library in this phase of MAGIC development. These are the solid elements; rectangular prism, tetrahedron, triangular prism, symmetric triangular prism, and triangular ring (asymmetrical loading). Also included are the symmetric shear web element and a revised quadrilateral thin shell element. The finite elements listed include matrices for stiffness, mass, prestrain load, thermal load, distributed mechanical load, pressure and stress.

The MAGIC III System for structural analysis is presented as an integral part of the overall design cycle. Considerations in this regard include, among other things, preprinted input data forms, automated data generation, data confirmation features, re-start options, automated output data reduction and readable output displays.

Documentation of the MAGIC III System is presented in three parts; namely, Volume I: Engineer's Manual, Volume II: User's Manual and Volume III: Programmer's Manual. The subject document Volume II (User's Manual) is an extension of the primary technical document and contains instructions for the preparation of input data and for interpretation of output data.

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## SECTION I

### INTRODUCTION

#### A. General Considerations

The MAGIC III System for structural analysis is an extension of the MAGIC I and MAGIC II Systems reported in References 1 to 6. All capabilities available in the original systems have been retained and improved upon. Extension of the MAGIC System has been in the following areas:

- (a) Incorporation of four (4) solid finite element representations
  - (1) Rectangular Prism
  - (2) Tetrahedron
  - (3) Triangular Prism
  - (4) Symmetric Triangular Prism
- (b) Incorporation of a triangular cross-section ring finite element which accommodates asymmetric loading.
- (c) Incorporation of a symmetric quadrilateral shear web finite element.
- (d) Incorporation of a quadrilateral thin shell finite element which reflects high aspect ratio usage.
- (e) The addition of miscellaneous arithmetic modules to the System to support the existing computational procedures.
- (f) Incorporation of an additional out-of-core variable bandwidth equation solver based on the modified square-root Cholesky method.
- (g) The addition to the System of a module designated as ANALIC (Analysis In Core) which can be used to perform a complete linearly elastic stress analysis, selected portions of a linear elastic analysis, or as a general purpose equation solver.

## B. Applicable MAGIC Documentation

The work reported herein is a discussion (from the User's point of view) of the extensions listed in Section A. This volume, User's Manual (Volume II) is an extension of the MAGIC II User's Manual (Reference 5) and as such is to be used in conjunction with that manual to effectively utilize the MAGIC III System. It is emphasized that all information contained in Reference 5 is directly applicable to MAGIC III without exception and the subject volume can be thought of as a supplement to Reference 5.

In order to avoid any confusion and to save the reader from frequent consultation of the Reference Section at the end of this document, the manuals applicable to the usage and understanding of the MAGIC III System are listed as follows:

### Theoretical Documents

- (a) Mallett, R.H. and Jordan, S., "MAGIC: An Automated General Purpose System for Structural Analysis: Volume I. Engineer's Manual", AFFDL-TR-68-56, Volume I, Air Force Flight Dynamics Laboratory, Wright-Patterson AFB, Ohio, January 1969.
- (b) Jordan, S., "MAGIC II: An Automated General Purpose System for Structural Analysis: Volume I. Engineer's Manual (Addendum)", AFFDL-TR-71-1, Volume I, Air Force Dynamics Laboratory, Wright-Patterson AFB, Ohio, May 1971.
- (c) Batt, J.R., and Jordan, S., "MAGIC III: An Automated General Purpose System for Structural Analysis: Volume I. Engineer's Manual", AFFDL-TR-72-42, Volume I, Air Force Flight Dynamics Laboratory, Wright-Patterson AFB, Ohio, April 1972.

## User Documents

- (a) Jordan, S., and Gallo, A.M., "MAGIC II: An Automated General Purpose System for Structural Analysis, Volume II. User's Manual", AFFDL-TR-71-1, Volume II, Air Force Flight Dynamics Laboratory, Wright-Patterson AFB, Ohio, May 1971.
- (b) Jordan, S., and Batt, J.R., "MAGIC III: An Automated General Purpose System for Structural Analysis, Volume II. User's Manual", AFFDL-TR-72-42, Volume II, Air Force Flight Dynamics Laboratory, Wright-Patterson AFB, Ohio, April 1972.

## Programming Document

- (a) Gallo, A.M., "MAGIC III: An Automated General Purpose System for Structural Analysis, Volume III. Programmer's Manual", AFFDL-TR-72-42, Volume III, Air Force Flight Dynamics Laboratory, Wright-Patterson AFB, Ohio, April 1972.

## C. Summary of Manual Contents

Section II presents additions to the abstraction instruction library of the MAGIC System, descriptions of new agendums for the ANALIC module, and detailed abstraction instructions required for the analysis of structures using the asymmetric triangular ring. Additional preprinted input data sheets have been designed and are explained. Newly implemented finite elements and instructions for their use are discussed in detail.

Section III is devoted to interpretation of the input to and output from the MAGIC III System. Preprinted input data forms are presented for specific example problems which utilize each of the MAGIC III finite element representations. Output from these problems is also displayed and discussed in detail.

# *Contrails*

Appendix A is included which delineates corrections and updates to the MAGIC II User's Manual (Reference 5). Appendix B is a compilation of all preprinted input data forms required to perform an analysis using MAGIC III.

## SECTION II

### INPUT TO THE MAGIC III SYSTEM

#### A. Introduction

The MAGIC III System presents two input data interfaces to the Structural Analyst. The first encountered is referred to as the System Input Data interface. The System data instructs the program as to what operations should be performed during any execution. These operations may be viewed as the interpretive portion of the MAGIC System. For example, the matrix abstraction instructions which are required to perform a structural analysis are System Input Data. All abstraction instructions available to the System prior to MAGIC III are delineated in detail in Reference 5. Instructions added during the MAGIC III development are discussed in detail in the next section.

The second input data interface with the User concerns the Structural Input Data. For example, grid point coordinates and boundary condition information are viewed as Structural Input Data. This problem oriented data accounts for nearly all the effort expended in conducting structural analyses.

As with the matrix abstraction instructions, the bulk of Structural Input Data parameters have been fully documented in Reference 5. Additional preprinted input data forms and specific finite element data for newly implemented elements evolved during the MAGIC III development are included and explained in this Section.

#### B. System Input Data

##### 1. General Description

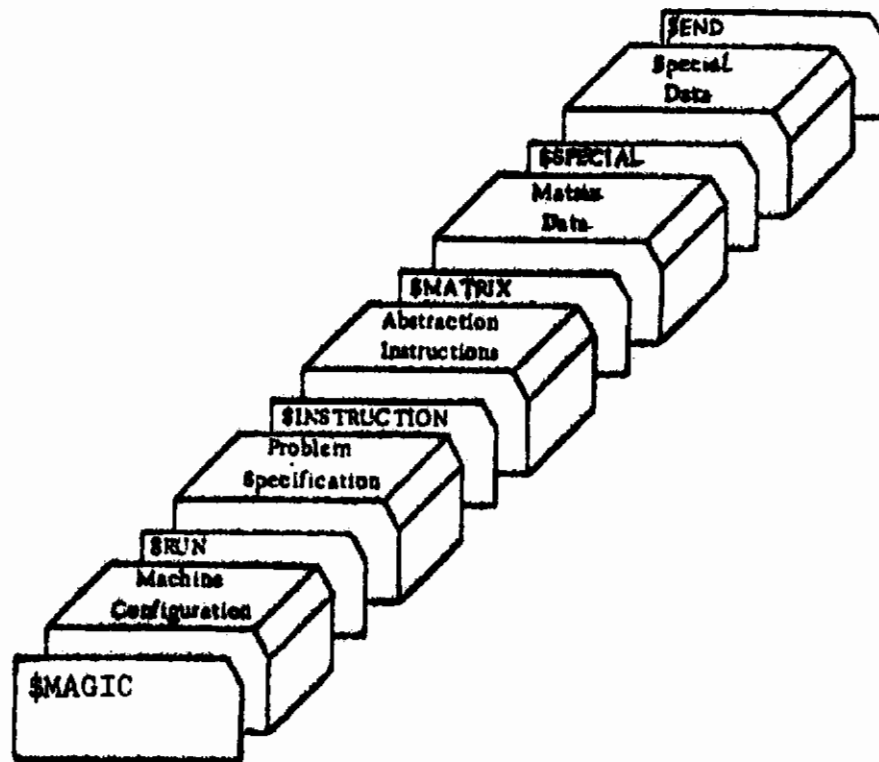
The input data for a general MAGIC execution consists of control and specification data, the abstraction instruction sequence, and problem data. Control, machine configuration and

# Contrails

problem specification data constitute the control and specification data. Matrix and special (non-matrix) data constitute the problem data. These data must be sequenced as follows:

- (1) Machine Configuration Data
- (2) Problem Specification Data
- (3) Abstraction Instruction Sequence Data
- (4) Matrix Data
- (5) Special Data

where each section is preceded by a control card which indicates the beginning of and the options chosen for that section. The last section is followed by a control card indicating the end of all input to a MAGIC case. A sketch of a typical MAGIC deck set up is shown below.





All MAGIC III data-deck set-ups have the form typified in the sketch. It is noted that all abstraction instruction data is placed behind the \$INSTRUCTION card. All input matrix data is placed behind the \$MATRIX card and all special data is placed behind the \$SPECIAL card. All structural input data such as gridpoint coordinates, element descriptions, temperature and pressure data are classified as special data and as such appear behind the \$SPECIAL card in a data-deck set up.

These data and their relationship to a MAGIC execution are discussed in detail in Reference 5, the companion manual to this document. Of interest here is the additional abstraction instruction capability (Item 3 above) which has been added to MAGIC III.

## 2. Additional Arithmetic Abstraction Instructions Added to MAGIC III

The basic form for arithmetic statements is:

$$c = \pm a .Op. \pm b$$

where a and b are known matrix names, c is the name of the matrix to be computed, Op is the operation to be performed in computing c from a and b and the positive signs of a and b may be omitted.

Variations of this basic form are required for certain operations. These variations are described with the corresponding operational definitions when they occur in the following arithmetic statements.

a. Solution of Equations by Cholesky Triangularization

Statements are of the form:

$$T, x = A, \beta. \text{CHTRIA.}$$

$Ax = \beta$  is the system of equations considered

Output matrix T = Triangularized matrix

Output matrix x = Solution vector

Input matrix A must be symmetric.

b. Computation of Triangularized Matrix

Statements are of the form:

$$T = A. \text{TRIA.}$$

Input matrix A must be symmetric

Output matrix T = Triangularized form of matrix A

c. Computation of Linear Equation Solution when Triangularized Matrix is Known (Back Substitution)

Statements are of the form:

$$x = T. \text{CHOL. } \beta$$

Input matrix T is in triangularized matrix form

Input matrix  $\beta$  is the known set of constants

Output matrix x = solution of back substitution system

This instruction is especially useful for linear equation solution of  $Ax = \beta$  when the matrix A has already been triangularized into T. Note also that T matrix must have been generated from a symmetric matrix A.

## d. REPEAT

### (1) General

The operation REPEAT has been added to the matrix abstraction capability of the system. The new operation provides a looping capability analogous to the FORTRAN "DO" statement where a certain sequence of instructions is to be repeated a specified number of times.

The sequence of instructions to be repeated is expanded into the range of the REPEAT loop during preprocessing, and unique matrix names attained by appending subscripts which are automatically incremented each time the sequence is repeated. Manipulation of subscripted matrices in the instruction sequence prior and subsequent to the REPEAT loop is entirely general and provision is made for the card input of such matrices. In the absence of a REPEAT statement in the sequence of instructions, use of subscripts on matrix names is optional.

Potential applications include synthesis of fully-stressed structural designs, analysis of structural nonlinearity due to large deflections, creep, short-time plasticity and combinations thereof, 3-dimensional matrix algebra and solutions of systems of nonlinear equations. REPEAT provides for a more expedient mode of abstraction instruction input in such applications.

### (2) Abstraction Instruction

"Repeat" statements are of the form

```
REPEAT (n,m)
```

where the arguments are

- n - the number of abstraction instructions in the sequence immediately following the REPEAT statement which are to be repeated
- m - the number of times the sequence of n instructions is to be repeated

The instruction can be literally interpreted as "repeat the following series of n instructions m times".

The sequence of instructions is expanded into the range of the REPEAT loop during preprocessing. Matrix names which are initially subscripted automatically have their subscripts incremented by one each time the sequence is repeated; unsubscripted matrix names remain the same.

### (3) Subscripted Matrix Names\*

Subscripted matrix names are specified in an abstraction instruction as one to six alphameric characters, the first of which must be alphabetic (as previously). The subscript of the matrix name, if any, must be a decimal integer between 1 and 9999 enclosed in slashes. If a matrix name is not subscripted, integer one is assumed. Negative or zero subscripts are not allowed.

The matrix name has the form:

NAMEA/k/ or NAMEA

where k is a one to four digit decimal integer.

Subscripted matrix names are specified in card input matrix data by the entry of the matrix name in card columns 67 through 72 (as previously) and the subscript in card columns 73 through 76 (right justified). A modified version of the card input matrix data standard form is shown in Figure II-1, page 20.

### (4) Restrictions

Restrictions on the use of the REPEAT loop are as follows:

A statement number may not appear on a statement which lies in the range of a REPEAT loop. This implies there can be no transfer into or within the range of the loop.

---

\* The matrix name is stored in memory as one character per word. The seventh word of the matrix name contains a positive or negative integer. The absolute value of this integer is the subscript of the matrix name. The sign of this integer is the sign of the matrix name.

# Contrails

Nesting of REPEAT loops is not permitted.

The total number of statements generated by the REPEAT loop is restricted by the amount of working storage (NWORK) available for the instruction analyzing module and the allocation module. (Typically with NWORK = 10000, approximately 100 instructions are permitted.)

Matrix names on the left side of an equals sign must be subscripted.

## (5) Error Messages

Additional control error messages which pertain to the REPEAT module and which emanate from the instruction processor module are listed below.

INST10     STATEMENT NUMBER SPECIFIED WITHIN RANGE OF LOOP.  
            STATEMENT NUMBER IGNORED

INST11     SYNTAX ERROR IN -REPEAT- INSTRUCTION

INST12     MATRIX NAME LEFT OF EQUALS SIGN NOT SUBSCRIPTED  
            WITHIN RANGE OF LOOP

INST13     INVALID NESTED LOOPS

INST14     SYNTAX ERROR IN SUBSCRIPTED MATRIX NAME

INST15     INSUFFICIENT CORE STORAGE FOR PROCESSING LOOP

INST16     RANGE OF REPEAT LOOP IS UNSATISFIED

INST\*\*     THIS INSTRUCTION NOT AVAILABLE

Other control error messages which include matrix names as additional descriptive information have been modified to accommodate subscripted matrix names.

## (6) Application

As an example of the use of REPEAT consider a nonlinear matrix equation of the form

$$A_0 + A_1x + A_2x^2 = 0$$

which may be processed iteratively to approximate  $x$  as follows:

$$x_{i+1} = -A_1^{-1} (A_0 + A_2x_i^2)$$

# Contrails

The appropriate abstraction instruction sequence using REPEAT is as follows.

```
1      7  
$INSTRUCTION  
  A1INV = -A1 .INVERS.  
  X /1/ = A1INV .MULT.  A0  
  PRINT(,,,)X/1/  
  REPEAT (6, 7)  
  XR /1/ = X /1/.RENAME.  
  X2 /1/ = XR /1/.EMULT.  X /1/  
  AX2/1/ = A2 .MULT.  X2 /1/  
  AAX/1/ = A0 .ADD.  AAX/1/  
  X /2/ = A1INV .MULT.  AAX/1/  
  PRINT(,,,)X/2/
```

where matrices A0, A1, and A2 are either card input or are available on an input matrix data set.

The effective expanded instruction sequence which would result is as follows.

```
$INSTRUCTION  
  A1INV = -A1 .INVERS.  
  X /1/ = A1INV .MULT.  A0  
  PRINT(,,,)X/1/  
  XR /1/ = X /1/.RENAME.  
  X2 /1/ = XR /1/.EMULT.  X /1/  
  AX2/1/ = A2 .MULT.  X2 /1/  
  AAX/1/ = A0 .ADD.  AAX/1/  
  X /2/ = A1INV .MULT.  AAX/1/  
  PRINT(,,,)X/2/  
  XR /2/ = X /2/.RENAME.  
  X2 /2/ = XR /2/.EMULT.  X /2/  
  AX2/2/ = A2 .MULT.  X2 /2/  
  AAX/2/ = A0 .ADD.  AAX/2/  
  X /3/ = A1INV .MULT.  AAX/2/  
  PRINT(,,,)X/3/  
  .  
  .  
  .  
  XR /7/ = X /7/.RENAME.  
  X2 /7/ = XR /7/.EMULT.  X /7/  
  AX2/7/ = A2 .MULT.  X2 /7/  
  AAX/7/ = A0 .ADD.  AAX/7/  
  X /8/ = A1INV .MULT.  AAX/7/  
  PRINT(,,,)X/8/
```

e. EIGEN2

Large Order Eigensolution statements are of the form:

$$c_1, c_2, c_3, c_4, c_5 = a.EIGEN2. b(d,e,f,g,h,i)$$

where d eigenvalues and the corresponding eigenvectors are extracted from the matrix a, the real parts of the eigenvalues and eigenvectors are named matrix c<sub>1</sub> and matrix c<sub>2</sub> respectively, the imaginary parts are named matrix c<sub>4</sub> and c<sub>5</sub> respectively and the residual error is named matrix c<sub>3</sub>. The following auxiliary definitions apply with matrix a of order (n x n)

- c<sub>1</sub> - is the matrix of real eigenvalues (d x 1)
- c<sub>2</sub> - is the matrix of real eigenvectors (n x d)
- c<sub>3</sub> - is the matrix (currently null) of residuals (n x 1)
- c<sub>4</sub> - is the matrix of imaginary eigenvalues (d x 1)
- c<sub>5</sub> - is the matrix of imaginary eigenvectors (n x d)
- a - is the name of the input eigenmatrix (n x n)
- b - is the name of an input starting vector (n x 1); this represents an approximation of the dominant eigenvector. If b is blank a unit vector is assumed.
- d - is the number of eigenvalues requested (an unsigned integer, preferably  $\leq 5$ ).
- e - is the number of calculation vectors (an unsigned integer  $> 3$  and  $\geq d$  but as small as possible).
- f - is the maximum number of iterations (an unsigned integer  $< 40$ ).
- g - is the starting vector recalculation exponent (a signed integer, nominally -2).
- h - is the eigenvalue-eigenvector accuracy criterion (an unsigned floating point number with or without exponent, e.g., 1.OE-4)
- i - is the eigenvalue uniqueness criterion (an unsigned floating point number with or without exponent, e.g., 1.OE-8).



f. MATRIX PARTITIONING (DJOIN)

(1) General

The base capability for matrix abstraction has been extended by the incorporation of the matrix operation DJOIN (opposite of ADJOIN). This provides for column partitioning of a matrix into two user-named sub-matrices, a capability hitherto effected by post multiplication of the subject matrix by two card extractor matrices.

(2) Abstraction Instruction

Matrix Dejoin statements are of the form:

$$c_1, c_2 = a.DJOIN.(j,o)$$

where the matrix a is column partitioned immediately before its jth column and the resulting dejoined matrices are named c<sub>1</sub> and c<sub>2</sub> (i.e.,  $\begin{bmatrix} c_1 \\ \vdots \\ c_2 \end{bmatrix} = a$ ). Matrices c<sub>1</sub> and c<sub>2</sub> are of order (m x j-1) and (m x n-j+1) respectively with matrix a of order (m x n). Note that  $1 \leq j \leq n$ .

The "0" argument in the statement indicates column dejoining of matrix a. Row dejoining may be effected by initially transposing matrix a. Provisions have been made to accept a "1" in place of the "0" to indicate row dejoin; currently the module will branch to a nonexistent subroutine.

(3) Error Messages

MATRIX COLUMN DIMENSIONS IS TOO SMALL IN .DJOIN.

This error results when the column number, j, is greater than the matrix column dimension, n.



## (4) Application

Two example applications are given below.

(1)  $X, Y = Z.DJOIN. (40,0)$

If Z is order 300 x 100 then  
X will be of order 300 x 39 and  
Y will be of order 300 x 61

(2)  $G, P = H.DJOIN. (1,0)$

G will be a null column with  
the row dimensions of H, and  
P will be a copy of H.

## g. Matrix REPLAC

Matrix Replace statements are of the form:

$$a = \pm b.REPLAC.c$$

where the input matrix b may be of order n x m  
the input matrix c may be of order n x m  
the output matrix a may be of order n x m

Wherever the elements of matrix b are equal to the corresponding elements of matrix c or wherever elements of matrix c are equal to 0.0, the output matrix a contains a direct mapping of matrix b. However, when the elements of b are not equal to the corresponding elements of matrix c, (excluding  $c = 0.0$ ), the output elements of the resulting matrix a are equal to those elements of matrix c. This instruction is useful whenever it is desired to form a new matrix a, such that its corresponding elements will be the same as those of matrix b except where modified by elements of matrix c which are not equal to 0.0.

## h. STRUCTURE CUTTER (STRCUT)

### (1) General

The matrix abstraction capability of the MAGIC III System includes the "Structure Cutter" module which generates a solution of "n" linear simultaneous equations in "m" unknowns by Jordanian elimination (where  $n \leq m$ ). This module takes advantage of sparsity of the coefficient matrix and utilizes a more effective mode of pivot selection.

The user may optionally control the pivotal acceptance levels used by the module and a list of the column numbers of the unreduced (non-pivotal) columns of the coefficient matrix is now included in the unconditional printed output for a successful execution. If execution is terminated for reason of unacceptable pivots the row numbers of the remaining (dependent) equations in which acceptable pivots cannot be found are listed.

The revised module also includes a restart capability which may be deployed should execution be terminated during the pivot selection phase for abnormal reasons; e.g., system malfunction. The four scratch data sets used during execution must be saved if a restart is to be made. Detailed information relating to this module is contained in Reference 7.

### (2) Abstraction Instruction

"Structure-Cutter" statements are of the form:

$$c_1, c_2 = \pm a.STRCUT. \pm b, (d,e,f,g,h)$$

where the solution, Y, of the system of "n" linear simultaneous equations in "m" unknowns,  $\pm AY \pm B = 0$ , where  $n \leq m$ , is formed by Jordanian elimination and the two parts of the solution are named matrix  $c_1$  and matrix  $c_2$ . The following auxiliary definitions apply:

# Contrails

- a - is the transpose of the coefficient matrix, A.
- b - is the transpose of the matrix of constants, B.
- $c_1$  - is the homogeneous solution.
- $c_2$  - is the particular solution.
- d - is an unsigned floating point number, with or without exponent, bounding matrix element values of matrices  $c_1$  and  $c_2$  which are trivial and to be suppressed. That is the matrix element  $c_{ij}$  is suppressed if  $|c_{ij}| \leq d$ . If  $d$  is blank, zero valued elements are suppressed.
- e - is either of the two literal constants. STOP or CONT. When  $e$  is STOP, execution is terminated if the available pivot elements do not satisfy the accuracy requirement. When  $e$  is CONT, termination of execution for reason of unacceptable pivot elements is delayed until the STRCUT instruction has been completely executed, including printing. If  $e$  is blank, the STOP option applies.
- f - is the name of the matrix of weighting factors. If  $f$  is blank, a unit matrix of weighting factors automatically applies.
- g - is the first pivotal acceptance level. If  $g$  is blank,  $10^{-3}$  is used.
- h - is the second pivotal acceptance level. If  $h$  is blank,  $10^{-5}$  is used.

Matrices  $c_1$  and  $c_2$  are normal output data for this process. For the case  $m = n$ ,  $c_1$  does not theoretically exist. In this case, a null matrix of order  $(n \times 1)$  is generated as  $c_1$ .

This subroutine unconditionally prints both a list of pivot element values with the corresponding column numbers of matrix A and a list of the column numbers of the unreduced (non-pivotal) columns of matrix A as special output data.

# Contrails

If execution is terminated for reason of unacceptable pivot elements the best remaining pivot is printed together with the row number of matrix A from which it emanates. Row numbers for all remaining rows which contain unacceptable pivots are also listed.

### (3) Error Message

Error messages emanating from the Structure Cutter are listed below in alphabetical order on the first word.

CANNOT LOCATE MATRIX FOR STRUCTURE CUTTER.

ERROR IN STRUCTURE CUTTER INPUT - IMAX = \*\*\*\* AND JMAX = \*\*\*\*

ERROR IN STRUCTURE CUTTER INPUT - NULL COLUMNS  
MATRIX \*\*\*\*\* NULL COLUMN = \*\*\*\* (ETC.)

ERROR IN STRUCTURE CUTTER INPUT - NULL ROWS  
NULL ROW = \*\*\*\* (ETC.)

INSUFFICIENT STORAGE FOR STRUCTURE CUTTER

INSUFFICIENT TAPES FOR STRUCTURE CUTTER

MATRIX IS SINGULAR. BEST UNACCEPTABLE PIVOT = ±0.XXXXXXE-XX  
EQUATION \*\*\*\*

FOLLOWING EQUATIONS CONTAIN UNACCEPTABLE PIVOTS  
\*\*\*\* (ETC.)

### 3. Matrix Data

Card Input matrix data are specified on the Standard Form shown on the following page. (Figure II-1)

A matrix header card having an H in card column 1, and containing the matrix name and its row and column dimensions, is required for each matrix.

It is noted that columns 73 thru 76 are set aside for subscript information. A blank in these locations indicates that the subscript associated with the matrix in question is equal to the integer one (1). Note that this subscripting option is extremely useful when used in conjunction with the REPEAT abstraction instruction discussed previously.

It is also noted that this revised form is identical to the original form provided for card input matrix data (Pg. 27, Reference 5), with the exception of Columns 73 thru 76. The heading PROG. NO. associated with these columns now reads SUBSCRIPT.

The last card after all \$MATRIX data must contain an E in card column 1 with the rest of the card blank.

Each matrix may contain up to 6000 randomly ordered elements. Machine sortability requires that the sequence number (first three digits) for each matrix is unique and identical in both header and element cards.

### 4. USERO4

#### a. Introduction

The fourth user coded module of the program is the structural generator of the MAGIC System.

This .USERO4. instruction plays the most important role in MAGIC and it is explained in detail on Pages 28 thru 35 of Reference 5, the companion document to this volume.

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H	I	J	I MAX	J MAX
1	2	3	4	5
6	7	8	9	
H				

**MAGIC  
MATRIX/INPUT DATA FORMAT**

MATRIX NAME		SUB SCRIPT		SEQ. NO.	
67	68	70	71	72	73
74	75	76	77	78	79
80					

I	J	VALUE	EXP.	I	J	VALUE	EXP.	I	J	VALUE	EXP.	I	J	VALUE	EXP.	I	J	VALUE	EXP.						
																				46	47	48	49	50	51
55	56	57	58	59	60	61	62	63	64	65	66	67	68	69	70	71	72	73	74	75	76	77	78	79	80

Figure II-1 MAGIC Matrix/Input Data Format



# Contrails

The Structural Generative System may have as many as fifteen actual output matrices and require as many as four actual input matrices. The basic form of the .USER04. instruction may be represented as follows:

```
OMP1, OMP2, OMP3, OMP4, OMP5, OMP6, OMP7, OMP8,  
OMP9, OMP10, OMP11, OMP12, OMP13, OMP14, OMP15 =  
IMP1, IMP2, IMP3, IMP4, .USER04. ;
```

where OMP is read as output matrix position and IMP as input matrix position. All matrix positions, whether input or output, must be present. They may contain matrix names or be blank, but there must be nineteen matrix positions represented by the appropriate number of commas. Blank matrix positions are discussed in the next section. The output matrix positions, if nonblank, will contain the following matrices upon exit from the Structural Generative System:

```
OMP1 - copy of input structure data deck  
OMP2 - revised material library  
OMP3 - interpreted input (structure input data  
as stored after being read and interpreted)  
OMP4 - external system grid point loads and  
load scalar matrix  
OMP5 - transformation matrix for application of  
boundary conditions  
OMP6 - transformation matrix for assembly of  
element matrices  
OMP7 - element stiffness matrices stored as one  
matrix  
OMP8 - element generated load matrices stored  
as one matrix  
OMP9 - element stress matrices stored as one matrix  
OMP10 - element thermal stress matrices stored as  
one matrix  
OMP11 - element incremental stiffness matrices  
stored as one matrix
```

# Contrails

- OMP12 - element mass matrices stored as one matrix
- OMP13 - structural system constants stored as one matrix
- OMP14 - element matrices in compressed format stored as one matrix
- OMP15 - prescribed displacement matrix

The input matrix positions, if nonblank must contain the following matrices:

- IMP1 - structure data deck (this would be a previously generated matrix saved in OMP1)
- IMP2 - interpreted input (this would be a previously generated matrix saved in OMP3 used for restart)
- IMP3 - existing material library (this would be a previously generated matrix saved in OMP2)
- IMP4 - displacement or stress matrix to be used for stability analyses (the stress matrix must have been generated by the structural abstraction instruction .STRESS.)

In the explanation of the .ANALIC. module and in the explanation of the agendum to use the triangular ring for asymmetric loading, the preceding general discussion of the form of the .USERO4. instruction will prove valuable.

## b. .ANALIC. (Analysis in Core)

### (1) Introduction

.ANALIC. is a MAGIC III abstraction instruction which can be used in conjunction with the .USERO4. abstraction instruction to perform a complete statics analysis using in-core routines exclusively. This module may also be used to perform selected portions of a static analysis or as a general purpose equation



solver. The ANALIC module is capable of solving problems of approximately 200 reduced degrees of freedom with 18,000 words of working storage. For problems of this size, ANALIC is significantly faster than the STATICS agendum. This abstraction also features 'dynamic' storage which allows the maximum size problem to fit in core, a choice of four different equation solvers and engineering printout of output matrices.

ANALIC reduces the amount of time required for solution of the statics problem mainly by reducing the overhead involved in many MAGIC abstraction instructions. In the STATICS agendum, each abstraction instruction must be analyzed, devices must be allocated for all input and output matrices, and finally the abstractions must be executed. The execution of these instructions consists of reading input matrices from intermediate devices, computing and then writing output matrices on other intermediate devices. These output matrices will, in general, become input matrices for subsequent abstractions and hence the above process is repeated. ANALIC eliminates the amount of I/O time required above by creating and operating on intermediate matrices in core. The need to write and then re-read information in successive abstraction instructions is eliminated.

The flexibility that is lost by having one abstraction instruction instead of several is made up in part by the suppression feature of the ANALIC instruction. This suppression feature is similar to the corresponding feature in the .USER04. instruction. If an output matrix is not desired, simply leave the name blank and code only the comma to denote the position of this missing matrix. Certain input matrices and scalars may also be left blank to indicate they are not present. For example, it is possible to use ANALIC to (1) generate only element stresses, or (2) calculate element forces and reactions for a prescribed displacement problem, or (3) compute stresses for a substructure analysis.

# Contrails

ANALIC is also flexible in allowing the user to solve the largest possible problem based on the particular elements he is using and the amount of working storage, NWORK, available to the MAGIC III System. The User indicates the maximum number of grid points, NNOM, and the maximum number of rows in the stress matrix, NRSELM, for any element used in the analysis. The values of NNOM and NRSELM for all the elements in MAGIC III can be found in a table below. Storage is allocated dynamically based on NNOM, NRSELM and other input parameters found in the SC matrix. ANALIC determines the amount of storage required and if there is insufficient storage available, it tries to reduce the number of load conditions to make the problem fit. If the problem still does not fit with one load condition, ANALIC returns control to MAGIC III indicating insufficient storage to solve the problem.

The User has the option of selecting from four different equation solvers in ANALIC. The reader is referred to Section III of Reference 9 for a detailed theoretical discussion of each equation solver. All four of these methods are designed and coded to operate on symmetric matrices. The first technique generates displacements by computing the inverse of the symmetric stiffness matrix and then multiplying by the loads. The method of bordering is used to calculate the symmetric inverse. Cholesky triangularization is the third method presently available in ANALIC. This method is probably the most effective method of solving system of equations. The fourth method available is the Gauss Wavefront method of Reference 10. This method was designed specifically for problems arising in linearly elastic stress analyses.

Engineering printout of many intermediate results as well as final output matrices is provided by the ANALIC module. The assembled/reduced stiffness matrix and element applied load

# Contrails

column are printed with reference to the original grid points and degrees of freedom. The total load column is also printed as well as the inverse of the stiffness matrix if it is generated. The displacements and reactions are printed corresponding to the system grid points and degrees of freedom. Element stresses and forces are printed with appropriate labels indicating the stress point and degree of freedom or the grid point.

(2) .ANALIC. With .USER04.

A complete linearly elastic stress analysis which generates displacements, stresses, forces, and reactions can be obtained by using a .USER04. instruction followed by an .ANALIC. instruction in the \$INSTRUCTION section of a MAGIC data deck. It is noted that two instructions provide essentially the same output as a standard STATICS agendum. (Note Page 41 and 42, Reference 5 for a comparison.)

An example of the use of the .ANALIC. instruction in conjunction with the .USER04. instruction follows:

```
CC      CC      CC
1       7       16
$MAGIC
$INSTRUCTION      SOURCE
                  ,MLIB,,XLD,TR,,KEL,FTEL,SEL,STEL,,,SC,EM,=,,, .USER04.
                  DISPL,STR,FORCE,REACT=TR,SC,EM,XLD,,,,.ANALIC.(KALC,NNOM,
                  NRSELM)
```

where for the .USER04. instruction

```
MLIB      -      updated material library
XLD       -      external load columns with element applied load
              scalar as first row
TR        -      transformation matrix from unordered to
              ordered system
```

# Contrails

KEL - element stiffness matrices generation control  
FTEL - element applied load columns generation control  
SEL - element stress matrices generation control  
STEL - element thermal stress columns generation control  
SC - system constants  
EM - all generated element matrices stored as columns

and for the .ANALIC. instruction

DISPL - system displacements  
STR - element stresses  
FORCE - element forces  
REACT - system reactions  
KALC - equation solver calculation control  
(See Table I )  
NNOM - maximum number of nodes in any element employed in the analysis (See Table II )  
NRSELM - maximum number of rows in the stress matrix of any element employed in the analysis  
(See Table II)

The three scalar values associated with the .ANALIC. module are KALC, NNOM and NRSELM. These scalars may be entered or suppressed. If the scalar is suppressed, the default values defined below will apply. Commas must be entered in any case to show the position of suppressed scalars. Note that scalars 2 and 3 are used in dynamic storage allocation. Selecting values which correspond to the specific problem are better than taking the default values, since larger problems may be run than with the default values.

# Contrails

SCALAR 1 (KALC) - This scalar indicates the method of solving for displacements based on the following Table (Table I).

KALC	METHOD
1	SYMMETRIC INVERSE
2	GAUSS ELIMINATION
3	CHOLESKY TRIANGULARIZATION
4	GAUSS WAVEFRONT
Anything Else (Default)	CHOLESKY TRIANGULARIZATION

Table I - KALC Scalar Control for .ANALIC.

SCALAR 2 (NNOM) - This scalar indicates the maximum number of grid points used for any element in the analysis. The default value is 8. Table II displays the number of grid points associated with each element in MAGIC III.

SCALAR 3 (NRSELM) - This scalar is the maximum number of rows in the stress matrix for any element used in the analysis. The default value is 40. Table II can be used to choose the largest value of NRSELM for any element used in the analysis.

# Contrails

ELEMENT	IDENT. NUMBER	NNOM	NRSELM
Frame	11	3	12
Incremental Frame	13	3	12
Triangular Thin Shell	20	6	30
Quadrilateral Thin Shell	21	8	40
Quadrilateral Shear Panel	25	4	1
Triangular Plate	27	3	8
Quadrilateral Plate	28	4	12
Symmetric Shear Web	29	2	1
Toroidal Ring (Shell Cap)	30	2	15
High Aspect Ratio Quadrilateral Thin Shell	38	8	40
Triangular Cross-Section Ring	40	3	4
Trapezoidal Ring (Core)	41	4	20
Tetrahedron	50	4	6
Triangular Prism (Symmetric Triangular Prism)	51	6	6
Rectangular Prism	52	8	6

Table II - Element Classification for .ANALIC.

### (3) .ANALIC. As An Equation Solver

In addition to using the .ANALIC. instruction with the .USERO4. instruction, .ANALIC. can be utilized as an equation solver as follows:

The equation solvers in .ANALIC. are available to use on any system of equations with symmetric coefficient matrices.

$$\begin{matrix} [A] & [X] & = & [B] \\ (NxN) & (NxM) & & (NxM) \end{matrix}$$

The form of the abstraction instruction used in this context is:

```
X,,, = ,,,,A,B,,.ANALIC.(KALC,N,M)
```

where

- OUTPUT MATRIX 1(X) - is the solution vector of order (NxM)
- INPUT MATRIX 6(A) - is the matrix of coefficients of order (NxN) in full form. Note that matrix A is symmetric.
- INPUT MATRIX 7(B) - is the right hand side vector of order (NxM)

SCALAR 1 (KALC) - This scalar indicates the method of solution based on the following Table:

KALC	METHOD
1	SYMMETRIC INVERSE
2	GAUSS ELIMINATION
3	CHOLESKY TRIANGULARIZATION
4	GAUSS WAVEFRONT
Anything Else (Default)	CHOLESKY TRIANGULARIZATION



SCALAR 2 (N) - is the order of the system of equations.

SCALAR 3 (M) - is the number of right hand columns.

All matrices and scalars must be entered with the exception of Scalar 1 (KALC).

#### (4) Miscellaneous Uses of .ANALIC.

The .ANALIC. module offers considerable flexibility to a User and its generality is examined in detail in this section.

The most general format of an .ANALIC. instruction is as follows:

```
DISPL,STRESS,FORCE,REACT = TR,SC,EM,XLD,PD,SUBK,  
SUBF,SUBL,GDIS.ANALIC.  
(KALC,NNOM,NRSELM)
```

##### (a) Output Matrices

.ANALIC. will generate any combination of the four output matrices DISPL, STRESS, FORCE, REACT based on the following conventions. To generate the output matrix, enter a name in the appropriate position in the instruction. To suppress the matrix generation, do not enter a name; code only the comma which indicates the position of the matrix which is not generated; i.e., if only stresses are desired, and TR, SC, EM, and XLD are the appropriate matrices output by a prior .USER04. instruction, write:

```
,STRESS,, = TR,SC,EM,XLD,,,,.ANALIC.(3,,)
```

The format of the output matrices generated by .ANALIC. are as shown on the following pages.



# Contrails

## Output Matrix One (DISPL)

- Contents - Displacements in unordered form
- Number of Rows - Number of degrees of freedom in total system
- Number of Columns - Number of load conditions
- Column Records - NDIR\*NDEG displacements for each system grid point

## Output Matrix Two (STRESS)

- Contents - Element stress matrices
- Number of Rows - Sum of number of rows in each element stress matrix
- Number of Columns - Number of load conditions
- Column Records - Element stress matrix repeated for each element

## Output Matrix Three (FORCE)

- Contents - Element force matrices
- Number of Rows - Sum of number of degrees of freedom in each element force matrix repeated for each element
- Number of Columns - Number of load conditions
- Column Records - Element force matrix repeated for each element

## Output Matrix Four (REACTIONS)

- Contents - Reactions
- Number of Rows - Number of degrees of freedom in total system
- Number of Columns - Number of load conditions
- Column Records - NDIR\*NDEG reactions for each system grid point

## FORMAT OF OUTPUT MATRIX 1 (DISPL)

Displacements from .ANALIC. module (Unordered format)

	Load #1	Load #2	.....	Load #NL
Node Pt. #1	NDIR * NDEG		.....	
Node Pt. #2	NDIR * NDEG		.....	
Node Pt. #3			.....	
Node Pt.			.....	
.			.....	
.			.....	
.			.....	
.			.....	
.			.....	
Node Pt. #NREF			.....	

Matrix is of the order  
(NSYS x NL)

where

NSYS = NREF\*NDIR\*NDEG

NL - number of load conditions

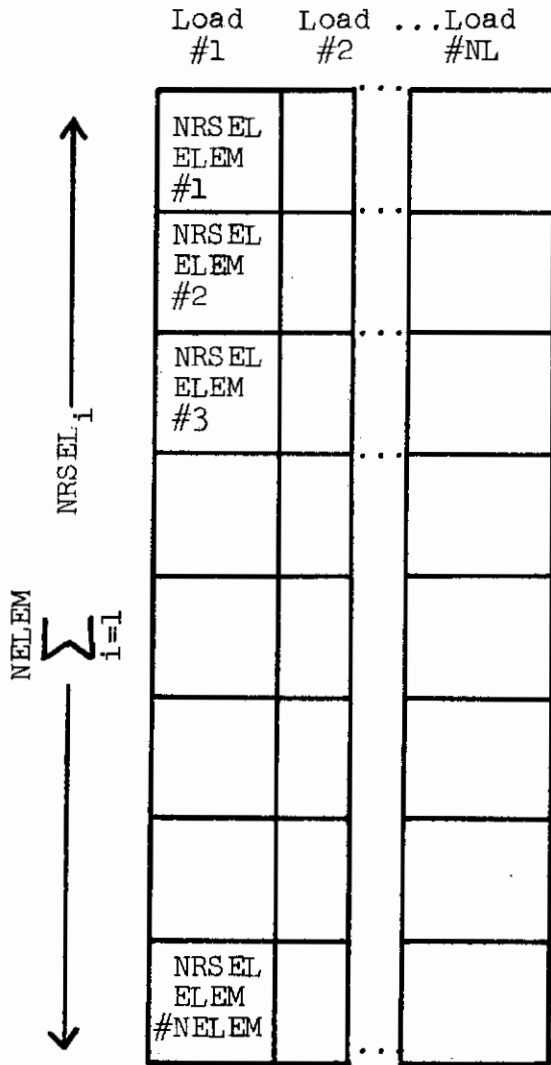
Column records are of the form

LOAD,ZERO,NSYS,(W(I),I=1,NSYS)

NREF = number of reference  
node points

## FORMAT OF OUTPUT MATRIX 2 (STRESS)

Element Stress matrix from .ANALIC.



Matrix is of the order  
(NTRSEL x NL.)

where

$$NTRSEL = \sum_{i=1}^{NELEM} NRSEL_i$$

NL = number of load conditions

Column records are of the form  
LOAD,ZERO,NTRSEL,(W(I),I=1,NTRSEL)

# Contrails

## FORMAT OF OUTPUT MATRIX 4 (REACT)

REACTIONS from .ANALIC. (Unordered format)

	Load #1	Load #2	...	Load #NL
Node Pt. #1	NDIR *	NDEG		
Node Pt. #2	NDIR *	NDEG		
Node Pt. #3	NDIR *	NDEG		
.				
.				
.				
.				
.			...	
.				
Node Pt. #NREF			...	

Matrix is of the order  
(NSYS x NL)

where

NSYS = NREF \* NDIR \* NDEG

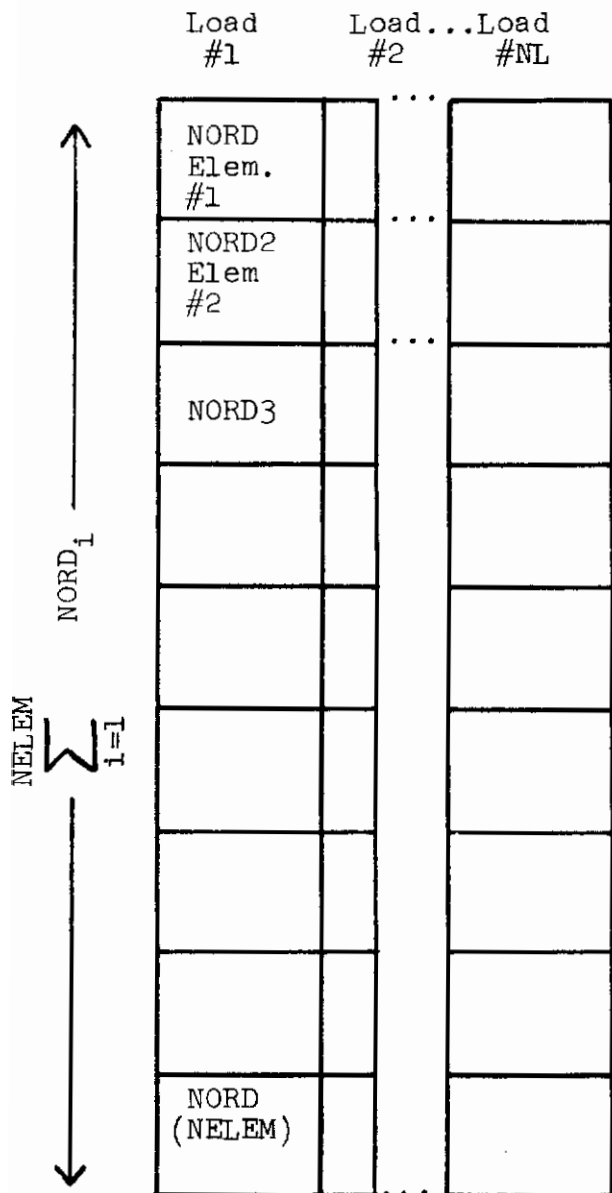
NL - number of load conditions

Column records are of the form  
LOAD, ZERO, NSYS, (W(I), I=1, NSYS)

NREF = number of reference node  
points

## FORMAT OF OUTPUT MATRIX 3 (FORCE)

Element Forces from .ANALIC.



Matrix is of the order  
(NTELEM x NL)

where

$$NTELEM = \sum_{i=1}^{NTELEM} NORD_i$$

NL = number of load conditions

Column records are of the form

LOAD,ZERO,NTELEM(W(I),I=1,NTELEM)

## (b) Input Matrices

There are nine possible input matrices for the .ANALIC. instruction. The first three matrices reflect the element generation information obtained from a .USER04. instruction and are required for a complete statics analysis (Section III.C.2). If .ANALIC. is used as an equation solver only, they are not input (Section III.C.3).

Input Matrix 1 (TR) - Is a transformation matrix which orders the system into the 0-1-2 order used by the .ANALIC. instruction. This matrix is usually obtained from output matrix position six of the .USER04. instruction.

Input Matrix 2 (SC) - Is a vector of system constants generated as output matrix 13 of the .USER04. instruction.

Input Matrix 3 (EM) - Is a matrix containing the element matrices generated by the .USER04. instruction. This matrix is output matrix 14 of the .USER04. instruction.

Input Matrix 4 (XLD) (Optional) - Is a matrix containing the external loading columns generated by the .USER04. instruction as output matrix 4. This matrix is unordered with load scalar as first row.

Input Matrix 5 (PD) (Optional) - Is a matrix containing prescribed displacements generated as output matrix 15 of a .USER04. instruction. This matrix is unordered.

Input Matrix 6 (SUBK) (Optional) - Is a matrix which contains a stiffness matrix in one of the following forms:

- (a) If input matrix 8 is not present, SUBK is a stiffness matrix of order  $M \times M$ ,  $M \leq N$  where  $N$  is the order of the reduced stiffness matrix generated in the .ANALIC. module. This matrix must be ordered the same way that the stiffness matrix in .ANALIC. is ordered. This matrix is added to the stiffness matrix assembled inside .ANALIC..

# Contrails

- (b) If input matrix 8 is present, SUBK is a stiffness matrix of order  $M \leq N$  where  $N$  is the maximum order of the reduced stiffness generated in the .ANALIC. module. This matrix does not have to be ordered the same way that the reduced stiffness matrix is. Input matrix 8 is the transformation matrix which will map the degrees of freedom into the assembled system degrees of freedom. SUBK can then be added to the stiffness matrix generated in .ANALIC..

Input Matrix 7 (SUBF) (Optional) - Is a matrix which contains applied loads in one of the following forms:

- (a) If input matrix 8 is not present, SUBF is an applied load matrix of order  $(M \times 1)$   $M \leq N$  where  $N$  is the order of the assembled/reduced applied load column. This matrix must be ordered the same way as the assembled/reduced applied load column in .ANALIC..
- (b) If input matrix 8 is present, SUBF is an applied load column of order  $M \leq N$  where  $N$  is the order of the applied load column used in .ANALIC.. This matrix may be unordered as long as input matrix 8 is present to order this matrix the same way as the assembled applied load column in .ANALIC.. SUBF can then be added correctly to the applied load column inside .ANALIC..

Input Matrix 8 (SUBL) (Optional) - Is a matrix which maps input matrices six and/or seven into the assembled system used inside .ANALIC.. This matrix is of order  $M \times 1$  where  $M$  is the order of input matrices seven and eight.

Input Matrix 9 (GDIS) (Optional) - Is a matrix which is reserved for future system use. It has no meaning presently for .ANALIC.. See Reference 8, Programmer's Manual for the procedure required to add a new equation solver to the .ANALIC. module.

KALC, NNOM and NRSELM were explained in detail in Section III.C.2.

c. Additional Abstraction Instructions Available in MAGIC III to Perform Structural Analyses

(1) Introduction

In the MAGIC II System for Structural Analysis, the procedure for performance of the following types of analyses was described in detail on Pages 39 thru 92 of Reference 5, the companion volume to the subject document.

1. Statics
2. Statics With Condensation
3. Statics With Prescribed Displacements
4. Stability
5. Dynamics (Modes and Frequencies)
6. Dynamics With Condensation

In the subject document, the procedure to perform a linearly elastic stress analysis using the triangular ring which accommodates asymmetric load will be discussed and explained in detail. In addition, additional Agendums for conventional linear elastic static analysis, statics analysis with condensation, static analysis with prescribed displacements, elastic instability analysis, dynamic analysis (with and without condensation) and free-free dynamic analysis (with and without condensation) are listed.



The analyses listed above may be performed in two different ways. In the first the User can elect to place the proper set of abstraction instructions in front of his structural input data deck for any given analyses. The second option, utilizes the Agendum Level abstraction capability which has been incorporated into the MAGIC II and MAGIC III Systems. Using this option, the abstraction instructions for the type of analyses desired are automatically generated by the System when the User specifies the corresponding option on the \$INSTRUCTION Card. This Agendum level capability will be discussed in detail after the presentation and explanation of the newly available abstraction instructions.

(2) Statics Instruction Sequence Using Triangular Ring With Asymmetric Loading (STATICS ASYM)

Figure II-2 presents the suggested set of abstraction instructions for use in performing a linearly elastic displacement and stress analysis using the triangular cross-section ring which accommodates asymmetric loading. This finite element and instruction for its proper use will be explained in detail in the following section. In addition, a sample problem showing the type of input required and the output obtained for this element is presented in Section III.

Before explaining Figure II-2, two key abstraction instructions used in this Agendum are defined.

.HDECO. - Extracts the harmonic dependent element stiffness matrix and updates the harmonic loop control matrix

Instructions are of the form:

HLC,HM1 = CF,EM,SC.HDECO.

# Contrails

There are two output matrices and three input matrices for this instruction.

Output matrix HLC - is the harmonic loop control matrix

Output matrix HMI - is the harmonic dependent element stiffness matrix

Input matrix CF - is the input harmonic loop matrix to be updated by this instruction. This matrix is used to form the output matrix HLC

Input matrix EM - is generated by the .USER04. instruction and contains all the element stiffness matrices for all desired harmonics

Input matrix SC - is generated by the .USER04. instruction and is a matrix of system constants which contains such items as the total number of elements and the harmonic number

.HSUM. - Computes the sums of harmonic stress, the sum of harmonic displacements and the sum of reactions.

Instructions are of the form:

SUMS,SUMD,SUMR = SC,INPS,INPD,INPR.HSUM.

There are three output matrices and four input matrices for this instruction.

Output matrix SUMS - is the harmonic sum stress matrix

Output matrix SUMD - is the harmonic sum displacement matrix

Output matrix SUMR - is the harmonic sum reaction matrix

# Contrails

```

$STATICASYM
C   ASSYMMETRIC CROSS SECTION RING ELEMENT AGENDUM                00100010
C---- GENERATE MASTER STIFFNESS MATRIX                            00100020
      ,MAT,,,TR,,,KEL,FTEL,SEL,STEL,,,SC,EM, = ,,,, .USER04.    00100030
C---- NEXT THREE INSTRUCTIONS GENERATE HARMONIC LOOP CONTROL CF MATRIX 00100040
      CA,CB = SC .DEJOIN. (4,1)                                   00100050
      CC,CD = CA .DEJOIN. (3,1)                                   00100060
      CE, HLC /1/ = CD .DEJOIN. (1,0)                             00100070
      TR1,TR12 = TR .DEJOIN. (SC(5,1),1)                          00100080
      REPEAT ( 13,8)                                             00100090
      HLC /2/, EM1 /2/ = HLC /1/, EM, SC .MDECO.                 00100100
      SAI /1/ = EM1 /2/ .ASSEM. SC, (10)                          00100110
      LA1 /1/ = EM1 /2/ .ASSEM. SC, (4)                           00100120
      RE1 /1/, RE2 /1/ = SAI /1/ .DEJOIN. (SC(5,1),1)            00100130
      LE1 /1/, LE2 /1/ = RE2 /1/ .DEJOIN. (SC(5,1),0)            00100140
      B1 /1/, X1 /1/ = LE2 /1/ .CHTRIA. LA1 /1/                   00100150
      XX1 /1/ = TR12 .TMULT. X1 /1/                                00100160
      XO1 /1/ = TR .MULT. XX1 /1/                                  00100170
      ST1 /1/ = EM1 /2/, XO1 /1/ .STRESS. (4,)                    00100180
      ATT1 /1/ = SAI /1/ .MULT. XO1 /1/                            00100190
      LBI /1/ = EM1 /2/ .ASSEM. SC, (40)                           00100200
      ACT1 /1/ = ATT1 /1/ .SUBT. LBI /1/                           00100210
      IF ( HLC /2/ .NULL. ) GO TO 200                             00100220
200  IF ( HLC /2/ .NULL. ) GO TO 2000                             00100230
      IF ( HLC /3/ .NULL. ) GO TO 3000                             00100240
2000 SUM1, SUMD1, SUMR1 = SC, ST1 /1/, XX1 /1/, ACT1 /1/ .HSUM. 00100250
      IF ( HLC /2/ .NULL. ) GO TO 1000                             00100260
3000 ST12 = ST1 /1/ .ADJOIN. ST1 /2/                               00100270
      XO12 = XX1 /1/ .ADJOIN. XX1 /2/                               00100280
      ACT12 = ACT1 /1/ .ADJOIN. ACT1 /2/                           00100290
      IF ( HLC /3/ .NULL. ) GO TO 1020                             00100300
      ST313 = ST12 .ADJOIN. ST1 /3/                                00100310
      XO313 = XO12 .ADJOIN. XX1 / 3/                               00100320
      ACT313 = ACT12 .ADJOIN. ACT1 /3/                              00100330
      IF ( HLC /4/ .NULL. ) GO TO 1030                             00100340
      ST414 = ST313 .ADJOIN. ST1 /4/                               00100350
      XO414 = XO313 .ADJOIN. XX1 /4/                               00100360
      ACT414 = ACT313 .ADJOIN. ACT1 /4/                            00100370
      IF ( HLC /5/ .NULL. ) GO TO 1040                             00100380
      ST515 = ST414 .ADJOIN. ST1 /5/                               00100390
      XO515 = XO414 .ADJOIN. XX1 /5/                               00100400
      ACT515 = ACT414 .ADJOIN. ACT1 /5/                            00100410
      IF ( HLC /6/ .NULL. ) GO TO 1050                             00100420
      ST616 = ST515 .ADJOIN. ST1 /6/                               00100430
      XO616 = XO515 .ADJOIN. XX1 /6/                               00100440
      ACT616 = ACT515 .ADJOIN. ACT1 /6/                            00100450
      IF ( HLC /7/ .NULL. ) GO TO 1060                             00100460
      ST717 = ST616 .ADJOIN. ST1 /7/                               00100470
      XO717 = XO616 .ADJOIN. XX1 /7/                               00100480
      ACT717 = ACT616 .ADJOIN. ACT1 /7/                           00100490
      IF ( HLC /8/ .NULL. ) GO TO 1070                             00100500

```

Figure II-2 STATICS Agendum for Triangular Ring with Asymmetric Loading

# Contrails

ST818 = ST717 .ADJOIN. ST1 /8/	00100510
X0818 = X0717 .ADJOIN. XX1 /8/	00100520
ACT818 = ACT717 .ADJOIN. ACT1 /8/	00100530
IF ( HLC /9/ .NULL. ) GO TO 1080	00100540
1020 SUMS12, SUMD12, SUMR12 = SC, ST12 , X012 , ACT12 .HSUM.	00100550
IF ( HLC /3/ .NULL. ) GO TO 1000	00100560
1030 SUMS13, SUMD13, SUMR31 = SC, ST313, X0313, ACT313 .HSUM.	00100570
IF ( HLC /4/ .NULL. ) GO TO 1000	00100580
1040 SUMS14, SUMD14, SUMR41 = SC, ST414, X0414, ACT414 .HSUM.	00100590
IF ( HLC /5/ .NULL. ) GO TO 1000	00100600
1050 SUMS15, SUMD15, SUMR51 = SC, ST515, X0515, ACT515 .HSUM.	00100610
IF ( HLC /6/ .NULL. ) GO TO 1000	00100620
1060 SUMS16, SUMD16, SUMR61 = SC, ST616, X0616, ACT616 .HSUM.	00100630
IF ( HLC /7/ .NULL. ) GO TO 1000	00100640
1070 SUMS17, SUMD17, SUMR71 = SC, ST717, X0717, ACT717 .HSUM.	00100650
IF ( HLC /8/ .NULL. ) GO TO 1000	00100660
1080 SUMS18, SUMD18, SUMR81 = SC, ST818, X0818, ACT818 .HSUM.	00100670
1000 CAA = CA .RENAME.	00100680

Figure II-2 (Concluded)

# Contrails

Input matrix SC - is a matrix of system constants generated by the .USERO4. instruction.

Input matrix INPS - is the input stress matrix to be summed. This matrix contains element stresses for each element and for each harmonic.

Input matrix INPD - is the input displacement matrix to be summed. This matrix contains displacements for each harmonic.

Input matrix INPR - is the input reaction matrix to be summed. This matrix contains reactions for each harmonic.

Table III is provided to give explicit definition to the STATICS Agendum for the Triangular Ring with Asymmetric Load illustrated in Figure II-2. Engineering definition for each abstraction instruction which is executed by the System is set forth in detail.

TABLE III  
 STATICS INSTRUCTION SEQUENCE - TRIANGULAR RING

STATEMENT NUMBER	(ASYMMETRIC LOADING) INSTRUCTION AND EXPLANATION
1	<p>,MAT,,,TR,,KEL,FTEL,SEL,STEL,,,SC,EM=,,,USERO4.</p> <p>Generates harmonic numbers, harmonic coefficients, and element matrices for each harmonic number. The controls KEL, FTEL, SEL, STEL must be present to cause these matrices to be generated in EM. MAT is an optional material library maintained by the user. TR and SC matrices are transformation and system control matrices respectively. Statement numbers ②, ③, and ④ generate the harmonic loop control CP for</p>
5	CA,CB = SC.DEJOIN. (4,1)
6	CC,CD = CA.DEJOIN. (3,1)
7	<p>CE,HLC/1/ = CD.DEJOIN. (1,0)</p> <p>These statements are needed to generate the harmonic loop control matrix HLC/1/ which has the dimension of 1 x 1. The control number in this matrix should be greater than zero and less than 12.</p>
8	<p>TR1,TR12 = TR.DEJOIN.(SC(5,1),1)</p> $\begin{bmatrix} \text{TR1} \\ \text{TR12} \end{bmatrix} = [\text{TR}]$ <p>Forms matrix TR12 which when transposed will regenerate the reduced displacement into the non-reduced displacement.</p>
9	<p>Repeat (13, 8)</p> <p>Generate the following 13 statements 8 times. The index of each matrix will be increased by one for each REPEAT.</p>
10	<p>HLC/2/,EML/2/ = HLC/1/,EM,SC.HDECO.</p> <p>Updates the harmonic loop control matrix HLC/1/ by decreasing its control value by one. If the control value is equal to zero, then a null matrix will be output as HLC, otherwise a matrix HLC with the dimension of 1 x 1 will be output.</p> <p>Extract the element stiffness matrix EML from the total set stiffness EM. The extraction is dependent on the harmonic loop control value.</p>



# Contrails

TABLE III (Continued)

STATEMENT NUMBER	INSTRUCTION AND EXPLANATION
11	<p>SA1/1/ = EML/2/.ASSEM.SC,(10)</p> <p>Generates the assembled stiffness matrix SA1/1/ in form of 0-1 ordered system for harmonic number one. SC contains system constants which are required by the .ASSEM. modules</p>
12	<p>LA1/1/ = EML/2/.ASSEM.SC,(4)</p> <p>Generates the assembled element applied load column in the 0-1 ordered system from the harmonic one element stiffness matrix EML. SC contains system constraints which are required by the .ASSEM. modules.</p>
13	<p>RE1/1/,RE2/1/ = SA1/1/.DEJOIN. (SC(5,1),1)</p> $\begin{bmatrix} RE1 \\ RE2 \end{bmatrix} = [SA1]$ <p>The NMDB rows of SA1 which correspond to the 1's are forwarded in the RE2 matrix.</p>
14	<p>LE1/1/,LE2/1/ = RE2/1/.DEJOIN. (SC(5,1),0)</p> $[LE1/1/, LE2/1/] = [RE2/1/]$ <p>The (NMDB x NMDB) reduced harmonic one element stiffness matrix LE2 is forwarded.</p>
15	<p>BI/1/,X1/1/ = LE2/1/.CHTRIA.LA1/1/</p> <p>Solves for the harmonic one displacements in the reduced system X1 by using Cholesky method to solve the system of simultaneous equation.</p>
16	<p>XX1/1/ = TR12.TMULT.X1/1/</p> $XX1/1/ = [TR12]^T [X1/1/]$ <p>Forms unordered system of displacements.</p>
17	<p>XO1/1/ = TR.MULT.XX1/1/</p> $[XO1/1/] = [TR] [XX1/1/]$ <p>Forms 0-1 ordered displacement columns in XO1,</p>
18	<p>ST1/1/ = EML/2/,XO1/1/.STRESS. (4.)</p> <p>Calculates net element stresses for each element.</p>
19	<p>ATT1/1/ = SA1/1/.MULT.XO1/1/</p> $ATT1/1/ = [SA1/1/] [XO1/1/]$ <p>To form system displacement vector ATT1 by multiplying the 0-1 ordered displacement vector with the 0-1 ordered stiffness matrix SA1.</p>

TABLE III (Continued)

STATEMENT NUMBER	INSTRUCTION AND EXPLANATION
20	<p><math>LB1/1/ = EM1/2/.ASSEM.SC,(40)</math></p> <p>Generates the system applied load vector LB1/1/ from the element stiffness matrix EM1 and the system matrix SC.</p>
21	<p><math>ACT1/1/ = ATT1/1/.SUBT.LB1/1/</math></p> <p><math>ACT1/1/ = [ATT1/1/] - [LB1/1/]</math></p> <p>Generates the reaction vector ACT1 by subtracting the system applied load vector from the system displacement.</p>
22	<p>If (HLC/2/.NULL.) go to 200.</p> <p>Test the harmonic control matrix HLC/2/ for number of harmonic loops.</p>
23	<p>200 If (HLC/2/.NULL.) go to 2000</p> <p>If the harmonic number is equal to one, then go to statement 25.</p>
24	<p>If (HLC/3/.NULL.) go to 3000</p> <p>If the harmonic number is greater than one, then go to statement 25.</p>
25	<p>2000 <math>SUM1,SUMD1,SUMR1 = SC,ST1/1/,XO1/1/,ACT1/1/.HSUM.</math></p> <p>Compute the sum of element stress, the sum of displacements and the sum reactions, and output the sum of element stresses, the sum of displacements and sum of reactions. This statement is used when the harmonic number is equal to one.</p>
26	<p>If (HLC/2/.NULL.) go to 1000</p> <p>Branch to statement 1000 to terminate the analysis.</p>
27	<p>3000 <math>ST12 = ST1/1/.ADJOIN.ST1/2/</math></p> <p>Adjoin the element stress ST1/1/ matrix for harmonic one with the element stress ST1/1/ matrix for harmonic two.</p>
28	<p><math>XO12 = XX1/1/.ADJOIN.XX1/2/</math></p> <p>Adjoin the system displacement XX1/1/ for harmonic one with the system displacement XX1/1/ matrix for harmonic two.</p>
29	<p><math>ACT12 = ACT1/1/.ADJOIN.ACT1/2/</math></p> <p>Adjoin the system reaction ACT1/1/ matrix with the system reaction ACT1/1/ matrix for harmonic two.</p>



# Contrails

TABLE III (Continued)

STATEMENT NUMBER	INSTRUCTION AND EXPLANATION
30	If (HLC/3/.NULL.) go to 1020. Test the harmonic control value in the harmonic control matrix HLC/3/ for the element stress matrices, the system displacement matrices and the system reaction to be adjoined.
31-34	Adjoin the element stress matrices, system displacement matrices, system reaction matrices for harmonic one, two, three and test the harmonic value in harmonic control matrix HLC/4/.
35-38	Adjoin the element stress matrices, system displacement matrices, system reaction matrices for harmonic one, two, three, four and test the harmonic value in harmonic control matrix HLC/5/.
39-42	Adjoin the element stress matrices, system displacement matrices, system reaction matrices for harmonic one, two, three, four, five and test the harmonic value in harmonic control matrix HLC/6/.
43-46	Adjoin the element stress matrices, system displacement matrices, system reaction matrices for harmonic one, two, three, four, five, six and test the harmonic value in harmonic control matrix HLC/7/.
47-50	Adjoin the element stress matrices, system displacement matrices, system reaction matrices for harmonic one, two, three, four, five, six, seven and test the harmonic value in harmonic control matrix HLC/8/.
51-54	Adjoin the element stress matrices, system displacement matrices, system reaction matrices for harmonic one, two, three, four, five, six, seven, eight and test the harmonic value in harmonic control matrix HLC/9/.
55	1020 SUMS12,SUMD12,SUMR12=SC,ST12,XO12,ACT12.HSUM. Compute the sum of element stress, the sum of displacements and the sum reactions, and output the sum of element stresses, the sum of displacements and sum of reactions. This statement is used when the harmonic number is equal to two.

# Contrails

TABLE III (Continued)

STATEMENT NUMBER	INSTRUCTION AND EXPLANATION
56	<p>If (HLC/3/.NULL.) go to 1000            Branch to statement 1000 to terminate the analysis.</p>
57	<p>1030 SUMS13,SUMD13,SUMR31=SC,ST313,X0313,ACT313.HSUM.            Compute the sum of element stress, the sum of displacements and the sum reactions, and output the sum of element stresses, the sum of displacements and sum of reactions. This statement is used when the harmonic number is equal to three.</p>
58	<p>If (HLC/4/.NULL.) go to 1000            Branch to statement 1000 to terminate the analysis.</p>
59	<p>1040 SUMS14,SUMD14,SUMR41=SC,ST414,X0414,ACT414.HSUM.            Compute the sum of element stress, the sum of displacements and the sum reactions, and output the sum of element stresses, the sum of displacements and sum of reactions. This statement is used when the harmonic number is equal to four.</p>
60	<p>If (HLC/5/.NULL.) go to 1000            Branch to statement 1000 to terminate the analysis.</p>
61	<p>1050 SUMS15,SUMD15,SUMR51=SC,ST515,X0515,ACT515.HSUM.            Compute the sum of element stress, the sum of displacements and the sum reactions, and output the sum of element stresses, the sum of displacements and sum of reactions. This statement is used when the harmonic number is equal to five.</p>
62	<p>If (HLC/6/.NULL.) go to 1000            Branch to statement 1000 to terminate the analysis.</p>
63	<p>1060 SUMS16,SUMD16,SUMR61=SC,ST616,X0616,ACT616.HSUM.            Compute the sum of element stress, the sum of displacements and the sum reactions, and output the sum of element stresses, the sum of displacements and sum of reactions. This statement is used when the harmonic number is equal to six.</p>
64	<p>If (HLC/7/.NULL.) go to 1000            Branch to statement 1000 to terminate the analysis.</p>
65	<p>1070 SUMS17,SUMD17,SUMR71=SC,ST717,X0717,ACT717.HSUM.            Compute the sum of element stress, the sum of displacements and the sum reactions, and output the sum of element stresses, the sum of displacements and sum of reactions. This statement is used when the harmonic number is equal to seven.</p>

# Contrails

TABLE III (Concluded)

STATEMENT NUMBER	INSTRUCTION AND EXPLANATION
66	If (HLC/8/.NULL.) go to 1000 Branch to statement 1000 to terminate the analysis.
67	1080 SUMS18,SUMD18,SUMR81=SC,ST818,XO818,ACT818.HSUM. Compute the sum of element stress, the sum of displacements and the sum reactions, and output the sum of element stresses, the sum of displacements and sum of reactions. This statement is used when the harmonic number is equal to eight.
68	CAA = CA.RENAME.  This instruction terminates the analysis.

### (3) Alternate Statics Instruction Sequence Using Triangular Ring with Asymmetric Loading

Figure II-3 presents an alternate set of abstraction instructions for use in performing a linearly elastic displacement and stress analysis using the triangular cross-section ring which accommodates asymmetric loading. It is noted that the suggested set of instructions presented in Figure II-2 made frequent use of the REPEAT option available to MAGIC III. (Note Statement 9) This REPEAT option was explained in detail on pp. 9 thru 12 of this report.

The instructions presented in Figure II-3 do not make use of the REPEAT option. Consequently, sets of instructions are written separately for each harmonic considered in the analysis. It is noted from Figure II-3 that thirteen statements are required for each harmonic which is considered. (Statements 28 thru 41 for Harmonic Number 2 for instance.) With this in mind, it is suggested that the standard set of instructions from Figure II-2 be utilized for most problems. However, the User has the option (if he prefers) of using the instructions as outlined in Figure II-3 which are given explicit engineering definition in Table IV.

# Contrails

```
C      ASSYMETRIC CROSS SECTION RING ELEMENT                                00100010
C----- GENERATE MASTER STIFFNESS MATRIX                                00100020
      ,MAT,,,TR,,KEL,FTEL,SEL,STEL,,,SC,EM, = ,,, .USER04.                00100030
C----- NEXT THREE INSTRUCTIONS GENERATE HARMONIC LOOP CONTROL CF MATRIX 00100040
      CA,CB = SC .DEJOIN. (4,1)                                           00100050
      CC,CD = CA .DEJOIN. (3,1)                                           00100060
      CE,CF = CD .DEJOIN. (1,0)                                           00100070
C----- EXTRACT STIFFNESS MATRIX FROM MASTER STIFFNESS MATRIX          00100080
C----- DECREASED LOOP CONTROL MATRIX BY ONE                           00100090
      HLC, EM1 = CF, EM, SC .HDECO.                                        00100100
C----- ASSEMBLE STIFF MATRIX FOR HARMONIC ONE                          00100110
      SA1 = EM1 .ASSEM. SC, (10)                                           00100120
C----- GENERATE SYSTEM APPLIED LOAD FOR HARMONIC ONE                  00100130
      LA1 = EM1 .ASSEM. SC, (4)                                           00100140
      RE1,RE2 = SA1 .DEJOIN. (SC(5,1),1)                                  00100150
      LE1,LE2 = RE2 .DEJOIN. (SC(5,1),0)                                  00100160
      B1, X1 = LE2 .CHTRIA. LA1                                           00100170
      TR1,TR12 = TR .DEJOIN. (SC(5,1),1)                                  00100180
      XX1 = TR12 .TMULT. X1                                               00100190
      XO1 = TR .MULT. XX1                                                 00100200
      ST1 = EM1, XO1 .STRESS. (4,)                                        00100210
      ATT1 = SA1 .MULT. XO1                                               00100220
      LB1 = EM1 .ASSEM. SC , (40)                                         00100230
      ACT1 = ATT1 .SUBT. LB1                                              00100240
C----- TEST HARMONIC LOOP CONTROL MATRIX                               00100250
      IF ( HLC .NULL. ) GO TO 100                                          00100260
C----- IF MORE THAN ONE HARMONIC FOLLOWING THIS PATH                  00100270
      HLC1, EM2 = HLC , EM, SC .HDECO.                                    00100280
      SA2 = EM2 .ASSEM. SC, (10)                                           00100290
      LA2 = EM2 .ASSEM. SC, (4)                                           00100300
      RE3,RE4 = SA2 .DEJOIN. (SC(5,1),1)                                  00100310
      LE3,LE4 = RE4 .DEJOIN. (SC(5,1),0)                                  00100320
      B2, X2 = LE4 .CHTRIA. LA2                                           00100330
      XX2 = TR12 .TMULT. X2                                               00100340
      XO2 = TR .MULT. XX2                                                 00100350
      ST2 = EM2,XO2 .STRESS. (4,)                                        00100360
      ATT2 = SA2 .MULT. XO2                                               00100370
      LB2 = EM2 .ASSEM. SC , (40)                                         00100380
      ACT2 = ATT2 .SUBT. LB2                                              00100390
C----- TEST HARMONIC LOOP CONTROL MATRIX                               00100400
      IF ( HLC1 .NULL. ) GO TO 200                                          00100410
C----- IF MORE THAN 3 HARMONIC FOLLOWING THIS PATH                    00100420
      HLC2, EM3 = HLC1, EM, SC .HDECO.                                    00100430
      SA3 = EM3 .ASSEM. SC, (10)                                           00100440
      LA3 = EM3 .ASSEM. SC, (4)                                           00100450
      RE5,RE6 = SA3 .DEJOIN. (SC(5,1),1)                                  00100460
      LE5,LE6 = RE6 .DEJOIN. (SC(5,1),0)                                  00100470
      B3, X3 = LE6 .CHTRIA. LA3                                           00100480
      XX3 = TR12 .TMULT. X3                                               00100490
      XO3 = TR .MULT. XX3                                                 00100500
      ST3 = EM3,XO3 .STRESS. (4,)                                        00100510
      ATT3 = SA3 .MULT. XO3                                               00100520
      LB3 = EM3 .ASSEM. SC , (40)                                         00100530
      ACT3 = ATT3 .SUBT. LB3                                              00100540
      IF ( HLC2 .NULL. ) GO TO 200                                          00100550
      HLC3, EM4 = HLC2, EM, SC .HDECO.                                    00100560
      SA4 = EM4 .ASSEM. SC, (10)                                           00100570
      LA4 = EM4 .ASSEM. SC, (4)                                           00100580
      RE7,RE8 = SA4 .DEJOIN. (SC(5,1),1)                                  00100590
      LE7,LE8 = RE8 .DEJOIN. (SC(5,1),0)                                  00100600
```

Figure II-3 Alternate STATICS Agendum for Triangular Ring



# Contrails

B4, X4 = LEB .CHTRIA. LA4	00100610
XX4 = TR12 .TMULT. X4	00100620
X04 = TR .MULT. XX4	00100630
ST4 = EM4,X04 .STRESS. (4,)	00100640
ATT4 = SA4 .MULT. X04	00100650
LB4 = EM4 .ASSEM. SC, (40)	00100660
ACT4 = ATT4 .SUBT. LB4	00100670
IF ( HLC3 .NULL. ) GO TO 200	00100680
HLC4, EM5 = HLC3, EM, SC .HDECO.	00100690
SA5 = EM5 .ASSEM. SC, (10)	00100700
LA5 = EM5 .ASSEM. SC, (4)	00100710
RE9,RE10 = SA5 .DEJOIN. (SC(5,1),1)	00100720
LE9,LE10 = RE8 .DEJOIN. (SC(5,1),0)	00100730
B5, X5 = LE10.CHTRIA. LA5	00100740
XX5 = TR12 .TMULT. X5	00100750
X05 = TR .MULT. XX5	00100760
ST5 = EM5,X05 .STRESS. (4,)	00100770
ATT5 = SA5 .MULT. X05	00100780
LB5 = EM5 .ASSEM. SC, (40)	00100790
ACT5 = ATT5 .SUBT. LB5	00100800
IF ( HLC4 .NULL. ) GO TO 200	00100810
HLC5, EM6 = HLC4, EM, SC .HDECO.	00100820
SA6 = EM6 .ASSEM. SC,(10)	00100830
LA6 = EM6 .ASSEM. SC, (4)	00100840
RE11,RE12 = SA6 .DEJOIN. (SC(5,1),1)	00100850
LE11,LE12 = RE12 . DEJOIN. (SC(5,1),0)	00100860
B6, X6 = LE12 .CHTRIA. LA6	00100870
XX6 = TR12 .TMULT. X6	00100880
X06 = TR .MULT. XX6	00100890
ST6 = EM6, X06 .STRESS. (4,)	00100900
ATT6 = SA6 .MULT. X06	00100910
LB6 = EM6 .ASSEM. SC, (40)	00100920
ACT6 = ATT6 .SUBT. LB6	00100930
IF ( HLC5 .NULL. ) GO TO 200	00100940
HLC6, EM7 = HLC5, EM, SC .HDECO.	00100950
SA7 = EM7 .ASSEM. SC,(10)	00100960
LA7 = EM7 .ASSEM. SC, (4)	00100970
RE13,RE14 = SA7 .DEJOIN. (SC(5,1),1)	00100980
LE13,LE14 = RE14 . DEJOIN. (SC(5,1),0)	00100990
B7, X7 = LE14 .CHTRIA. LA7	00101000
XX7 = TR12 .TMULT. X7	00101010
X07 = TR .MULT. XX7	00101020
ST7 = EM7, X07 .STRESS. (4,)	00101030
ATT7 = SA7 .MULT. X07	00101040
LB7 = EM7 .ASSEM. SC, (40)	00101050
ACT7 = ATT7 .SUBT. LB7	00101060
IF ( HLC6 .NULL. ) GO TO 200	00101070
HLC7, EM8 = HLC6, EM, SC .HDECO.	00101080
SA8 = EM8 .ASSEM. SC,(10)	00101090
LA8 = EM8 .ASSEM. SC, (4)	00101100
RE15,RE16 = SA8 .DEJOIN. (SC(5,1),1)	00101110
LE15,LE16 = RE16 . DEJOIN. (SC(5,1),0)	00101120
B8, X8 = LE16 .CHTRIA. LA8	00101130
XX8 = TR12 .TMULT. X8	00101140
X08 = TR .MULT. XX8	00101150
ST8 = EM8, X08 .STRESS. (4,)	00101160
ATT8 = SA8 .MULT. X08	00101170
LB8 = EM8 .ASSEM. SC, (40)	00101180
ACT8 = ATT8 .SUBT. LB8	00101190
IF ( HLC7 .NULL. ) GO TO 200	00101200

# Contrails

200	ST12 = ST1 .ADJOIN. ST2	00101210
	X012 = XX1 .ADJOIN. XX2	00101220
	ACT12 = ACT1 .ADJOIN. ACT2	00101230
	IF ( HLC1 .NULL. )GO TO 1020	00101240
	ST313 = ST12 .ADJOIN. ST3	00101250
	X0313 = X012 .ADJOIN. XX3	00101260
	ACT313 = ACT12 .ADJOIN. ACT3	00101270
	IF ( HLC2 .NULL. )GO TO 1030	00101280
	ST414 = ST313.ADJOIN. ST4	00101290
	X0414 = X0313.ADJOIN. XX4	00101300
	ACT414 = ACT313 .ADJOIN. ACT4	00101310
	IF ( HLC3 .NULL. )GO TO 1040	00101320
	ST515 = ST414.ADJOIN. ST5	00101330
	X0515 = X0414.ADJOIN. XX5	00101340
	ACT515 = ACT414 .ADJOIN. ACT5	00101350
	IF ( HLC4 .NULL. )GO TO 1050	00101360
	ST616 = ST515 .ADJOIN. ST6	00101370
	X0616 = X0515 .ADJOIN. XX6	00101380
	ACT616 = ACT515 .ADJOIN. ACT6	00101390
	IF ( HLC5 .NULL. ) GO TO 1060	00101400
	ST717 = ST616 .ADJOIN. ST7	00101410
	X0717 = X0616 .ADJOIN. XX7	00101420
	ACT717 = ACT616 .ADJOIN. ACT7	00101430
	IF ( HLC6 .NULL. ) GO TO 1070	00101440
	ST818 = ST717 .ADJOIN. ST8	00101450
	X0818 = X0717 .ADJOIN. XX8	00101460
	ACT818 = ACT717 .ADJOIN. ACT8	00101470
	IF ( HLC7 .NULL. ) GO TO 1080	00101480
1020	SUMS12, SUMD12, SUMR12 = SC, ST12 , X012 , ACT12 .HSUM.	00101490
	IF ( HLC1 .NULL. ) GO TO 1000	00101500
1030	SUMS13, SUMD13, SUMR31 = SC, ST313, X0313, ACT313 .HSUM.	00101510
	IF ( HLC2 .NULL. ) GO TO 1000	00101520
1040	SUMS14, SUMD14, SUMR41 = SC, ST414, X0414, ACT414 .HSUM.	00101530
	IF ( HLC3 .NULL. ) GO TO 1000	00101540
1050	SUMS15, SUMD15, SUMR51 = SC, ST515, X0515, ACT515 .HSUM.	00101550
	IF ( HLC4 .NULL. ) GO TO 1000	00101560
1060	SUMS16, SUMD16, SUMR61 = SC, ST616, X0616, ACT616 .HSUM.	00101570
	IF ( HLC5 .NULL. ) GO TO 1000	00101580
1070	SUMS76, SUMD17, SUMR71 = SC, ST717, X0717, ACT717 .HSUM.	00101590
	IF ( HLC6 .NULL. ) GO TO 1000	00101600
1080	SUMS86, SUMD18, SUMR81 = SC, ST818, X0818, ACT818 .HSUM.	00101610
	IF ( HLC7 .NULL. ) GO TO 1000	00101620
100	SUM1,SUMD1,SUMR1 = SC, ST1, XX1, ACT1 .HSUM.	00101630
1000	CAA = CA .RENAME.	00101640

Figure II-3 (Concluded)



TABLE IV  
ALTERNATE STATICS INSTRUCTION SEQUENCE  
TRIANGULAR RING - ASYMMETRIC LOADING

STATEMENT NUMBER	INSTRUCTION AND EXPLANATION
3	<p>,MAT,,,TR,,KEL,,FTEL,,SEL,,STEL,,,SC,EM ....DETERO<sup>4</sup>.</p> <p>Generates harmonic numbers, harmonic coefficients, and element matrices for each harmonic number. The controls KEL, FTEL, SEL, STEL must be present to cause these matrices to be generated in EM. MAT is an optional material library maintained by the user. TR and SC matrices are transformation and system control matrices respectively.</p> <p>Statement numbers ②, ③, and ④ generate the harmonic loop control CF for</p>
5	CA,CB=SC.DEJOIN. (4,1)
6	CC,CD=CA.DEJOIN. (3,1)
7	CE,CF=CD.DEJOIN. (1,0)
10	<p>HLC,EM1=CF,EM,SC.HDECO</p> <p>Updates the harmonic loop control matrix CF by decreasing its control value by one. If the control value is equal to zero, then a null matrix will be output as HLC; otherwise a matrix HLC with the dimension of 1 x 1 will be output.</p> <p>Extract the element stiffness matrix EM1 from the total set stiffness EM. The extraction is dependent on the harmonic loop control value.</p>
12	<p>SA1=EM1.ASSEM.SC, (10)</p> <p>Generate the assembled stiffness matrix SA1 in form of 0-1 ordered system for harmonic number one. SC contains system constraints which are required by the .ASSEM. modules.</p>
14	<p>LA1= EM1.ASSEM.SC, (4)</p> <p>Generates the assembled element applied load column in the 0-1 ordered system from the harmonic one element stiffness matrix EM1. SC contains system constants which are required by the .ASSEM. modules.</p>

# Contrails

TABLE IV - (CONTINUED)

STATEMENT NUMBER	INSTRUCTION AND EXPLANATION
15	<p>RE1,RE2=SA1.DEJOIN. (SC(5,1),1)</p> $\begin{bmatrix} RE1 \\ RE2 \end{bmatrix} = [SA1]$ <p>The NMDB rows of SA1 which correspond to the 1's are forwarded in the RE2 matrix.</p>
16	<p>LE1,LE2=RE2.DEJOIN. (SC(5,1),0)</p> $[LE1, LE2] = [RE2]$ <p>The (NMDB x NMDB) reduced harmonic one element stiffness matrix LE2 is formed.</p>
17	<p>BLX1=LE2.CHTRIA.LA1</p> <p>Solves for the harmonic one displacements in the reduced system X1 by using Cholesky method to solve the system of simultaneous equations.</p>
18	<p>TR1,TR12=TR.DEJOIN. (SC(5,1),1)</p> $\begin{bmatrix} TR1 \\ TR12 \end{bmatrix} = [TR]$ <p>Forms matrix TR12 which when transposed will regenerate the reduced displacement X1 into the non-reduced displacement XX1.</p>
19	<p>XX1 = TR12.TMULT.X1</p> $XX1 = [TR12]^T [XX]$ <p>Forms unordered system of displacements.</p>
20	<p>XO1 = TR.MULT.XX1</p> $[XO1] = [TR] [XX1]$ <p>Forms 0-1 ordered displacement columns in XO1.</p>
21	<p>ST1 = EML, XO1.STRESS. (4,)</p> <p>Calculates net element stresses for each element</p>
22	<p>ATT1 = SA1.MULT.XO1</p> $ATT1 = [SA1] [XO1]$ <p>Forms system displacement vector ATT1 by multiplying the 0-1 ordered displacement vector with the 0-1 ordered stiffness matrix SA1.</p>

# Contrails

TABLE IV - (CONTINUED)

STATEMENT NUMBER	INSTRUCTION AND EXPLANATION
23	<p><math>LB1 = EM1.ASSEM.SC, (40)</math></p> <p>Generates the system applied load vector LB1 from the element stiffness matrix EM1 and the system matrix SC.</p>
24	<p><math>ACT1 = ATT1.SUBT.LB1</math></p> <p><math>ACT1 = [ATT1] - [LB1]</math></p> <p>Generates the reaction vector ACT1 by subtracting the system applied load vector from the system displacement.</p>
26	<p>If (HLC.NULL.) go to 100.</p> <p>Test the harmonic control matrix HLC for number of harmonic loops.</p>
28-41	<p>Statements 28 thru 41 are used for computation of the second harmonic, when the harmonic control value is dependent on the harmonic control matrix HLC. The explanations for the Statements 28 thru 41 are the same as Statements 10 thru 26.</p>
43-55	<p>Statements 43 thru 55 are used for computation of the third harmonic dependent on the harmonic control matrix HLC1. The explanations for the Statements 43 thru 55 are the same as Statements 10 thru 26.</p>
56-68	<p>Statements 56 thru 68 are used for computation of the fourth harmonic dependent on the harmonic control matrix HLC2. The explanations for the Statements 56 thru 68 are the same as Statements 10 thru 26.</p>
69-81	<p>Statements 69 thru 81 are used for computation of the harmonic 5 when the harmonic control value in the harmonic control matrix HLC3 is greater than zero. The explanations for Statements 69 thru 81 are the same as Statements 10 thru 26.</p>
82-94	<p>Statements 82 thru 94 are used for computation of harmonic 6 when the harmonic control value in the harmonic control matrix HLC4 is greater than zero. The explanations for the Statements 82 thru 94 are the same as the Statements 10 thru 26.</p>
95-107	<p>Statements 95 thru 107 are used for computation of harmonic 7 when the harmonic control value in the harmonic control matrix HLC5 is greater than zero. The explanations for Statements 95 thru 107 are the same as for Statements 10 thru 26.</p>

# Contrails

TABLE IV (CONTINUED)

STATEMENT NUMBER	INSTRUCTION AND EXPLANATION
108-120	Statements 108 thru 120 are used for computation of harmonic 8 when the harmonic control value in the harmonic control matrix HLC6 is greater than zero. The explanations for Statements 108 thru 120 are the same as for Statements 10 thru 26.
121	200 ST12 = ST1.ADJJOIN.ST2 Adjoin the element stress ST1 matrix for harmonic one with the element stress ST2 matrix in harmonic two.
122	X012 = XX1.ADJJOIN.XX2 Adjoin the system displacement XX1 for harmonic one with the system displacement XX2 matrix for harmonic two.
123	ACT12 = ACT1.ADJJOIN.ACT2 Adjoin the system reaction ACT1 matrix with the system reaction ACT2 matrix for harmonic two.
124	If (HLC1.NULL.) to go 1020. Test the harmonic control value in the harmonic control matrix HLC1 for the element stress matrix, the system displacement matrices and the system reaction matrix to be adjointed.
125-128	Adjoin the element stress matrices, system displacement matrices, system reaction matrices for harmonic one, two, three and test the harmonic value in harmonic control matrix HLC2.
129-132	Adjoin the element stress matrices, system displacement matrices, system reaction matrices for one, two, three, four and test the harmonic value in harmonic control matrix HLC3.
133-136	Adjoin the element stress matrices, system displacement matrices, system reaction matrices for harmonic one, two, three, four, five and test the harmonic value in harmonic control Matrix HLC4.
137-140	Adjoin the element stress matrices, system displacement matrices, system reaction matrices for harmonic one, two, three, four, five, six and test the harmonic value in harmonic control matrix HLC5.

# Contrails

TABLE IV -(CONTINUED)

STATEMENT NUMBER	INSTRUCTION AND EXPLANATION
141-144	Adjoin the element stress matrices, system displacement matrices, system reaction matrices for harmonic one, two, three, four, five, six, seven and test the harmonic value in harmonic control matrix HLC6.
145-148	Adjoin the element stress matrices, system displacement matrices, system reaction matrices for harmonic one, two, three, four, five, six, seven, eight and test the harmonic value in harmonic control matrix HLC7.
149	1020 SUMS12,SUMD12,SUMR12=SC,ST12,X12,ACT12.HSUM. Compute the sum of element stress, the sum of displacements and the sum of reactions and output the sum of element stress, the sum of displacements and sum of reactions. This statement is used when the harmonic number is equal to two.
150	If (HLC1.NULL.) go to 1000 Branch to statement 164 to terminate the analysis.
151-152	These statements are used when the harmonic number is equal to three. For explanation of these statements, see Statements 149 and 150.
153-154	These statements are used when the harmonic number is equal to four. For explanation of these statements, see Statements 149 and 150.
155-156	These statements are used when the harmonic number is equal to five. For explanation of these statements, see statements 149 and 150.
157-158	These statements are used when the harmonic number is equal to six. For explanation of these statements, see statements 149 and 150.
159-160	These statements are used when the harmonic number is equal to seven. For explanation of these statements, see statements 149 and 150.
161-162	These statements are used when the harmonic number is equal to eight. For explanation of these statements, see statements 149 and 150.

# Contrails

TABLE IV - (CONCLUDED)

STATEMENT NUMBER	INSTRUCTION AND EXPLANATION
163	100 SUM1,SUMD1,SUMR1=SC,ST1,X01,ACT1.HSUM. This statement is used when harmonic number is equal to one. For explanation of this statement see statement 149.
164	1000 CAA = CA.RENAME. Terminates the analysis.



## (4) Statics Instruction Sequence Using Cholesky Triangularization (STATICS)

Figure II-4 presents the suggested set of abstraction instructions for use in performing a linearly elastic displacement and stress analysis. This set of instructions differs from those reported on pp. 40 thru 52 of Reference 5 in the following respect.

The set of simultaneous linear equations which arise in the analysis are solved by Cholesky triangularization. The use of these instructions are explained in detail on page 8 of this report.

Statement 47 of Figure II-4 has the following form:

```
TRIA,XX = STIFF,TLOADR.CHTRIA.
```

where in Reference 5, the equation solution had the following form:

```
XX = STIFF.SEQEL.TLOADR
```

Note Statement 47 of Figure II-C (Page 41) of Reference 5.

It is again emphasized that the User is not restricted to this particular set of instructions. The flexibility of the System allows the use of additional or alternate instructions to accommodate special needs and requirements of the User. All instructions available from MAGIC II (Reference 5) are available in MAGIC III.





```
C          00000600
C          ELEMENTS HAVE 1 OR 2 DEGREES OF FREEDOM          00000610
C          00000620
GPRINT(4,,,FX,FY,FZ,MX,MY,MZ,SC,TR )FTELA          00000630
GPRINT(4,,,FX,FY,FZ,MX,MY,MZ,SC, )LOADS          00000640
GPRINT(2,,,U,V,W,THETAX,THETAY,THETAZ,SC, )X          00000650
GPRINT(1,,,FX,FY,FZ,MX,MY,MZ,SC,TR )REACTP          00000660
IF (I3.NULL.) GO TO 600          00000670
C          00000680
C          ELEMENTS HAVE 3 DEGREES OF FREEDOM          00000690
C          00000700
10 GPRINT(4,,,FR.O.FZ.O.MBETA.O.F1.O.F3,SC,TR )FTELA          00000710
GPRINT(4,,,FR.O.FZ.O.MBETA.O.F1.O.F3,SC, )LOADS          00000720
GPRINT(2,,,U.O.W.O.THETAY.O.W*.O.W**,SC, )X          00000730
GPRINT(1,,,FR.O.FZ.O.MBETA.O.F1.O.F3,SC,TR )REACTP          00000740
C          00000750
C          GENERATE STRESSES AND FORCES          00000760
C          00000770
600 STRESP=EM,XO .STRESS.(4, )          00000780
    FORCEP=EM,XO .FORCE.(4, )          00000790
```

FIGURE II-4 - (CONCLUDED)

## (5) Statics Instruction Sequence With Condensation Using Cholesky Triangularization (STATICSC)

Figure II-5 presents the suggested set of abstraction instructions for use in performing a linearly elastic displacement and stress analysis with condensation. The condensation (reduction) technique is that of Guyan (Reference 11). With the use of this option, the User is provided the flexibility to perform a static analysis utilizing a rational condensation procedure. The only basic difference in abstraction instructions between using the statics with condensation option and the standard statics option is the additional instructions required to form the condensed stiffness matrix, i.e.,

$$[K]_R = \begin{bmatrix} K_{11} & -K_{12} & & \\ & K_{22}^{-1} & & \\ & & K_{21} & \\ & & & \end{bmatrix}$$

This set of instructions differs from those reported on pp. 53 thru 55 of Reference 5 in the following respects.

In the Agendum presented in this document, the reduced stiffness matrix,  $[K]_R$  and the deflections, D1 are found using Cholesky Triangularization. Sequence Numbers 213 thru 217 of the present agendum are as follows:

```
K22I,KR1 = K22,K12T.CHTRIA.  
KR2 = K12.MULT.-KR1  
KR = K11.ADD.KR2
```

And upon solving for displacements, D1, we have:  
TRIA,D1 = KR,P1.CHTRIA.

It is noted that in Reference 5, the reduced stiffness matrix,  $[K]_R$ , and the displacements D1 were obtained as follows. (Note Sequence Numbers 393 thru 398, p. 54, Reference 5.)

# Contrails

```
K22I = -K22 .INVERS.  
KR1 = K22I.MULT.K12T  
KR2 = K12.MULT.KR1  
KR = K11.ADD.KR2  
D1 = KR.SEQEL.P1
```

The suggested set of instructions presented herein avoids the use of inversion to form the reduced stiffness matrix. Additionally, the instruction using .SEQEL. has been replaced with .CHTRIA.



# Contrails

```

      P1,P12 = TLOAD .DEJOIN. (SC(5,1),1)
C-----CONDENSE EXTERNAL LOAD COLUMNS
      P1,P2 = P12 .DEJOIN. (SC(6,1),1)
C
C-----FCRM (K11 - K12*K22(INVS)*K12T)
C
      K22I,KR1 = K22,K12T .CHTRIA.
      KR2 = K12 .MULT. -KR1
      KP = K11 .ADD. KR2
C-----SOLVE FOR DISPLACEMENTS D1
      TRIA,D1 = KR,P1 .CHTRIA.
C-----SOLVE FOR DISPLACEMENTS D2
      D2 = KR1 .MULT. D1
C-----FCRM TOTAL DISPLACEMENT VECTOR
      D1T = C1 .TRANSP.
      D2T = C2 .TRANSP.
      D12 = D1T .ADJOIN. D2T
      XX = D12 .TRANSP.
C-----EXPAND DISPLACEMENTS TO TOTAL SYSTEM DEGREES OF
C-----FREEDOM AND REARRANGE TO 0-1-2 SYSTEM
      TR0,TR12 = TR .DEJOIN.(SC(5,1),1)
      X = TR12.TMULT.XX
      XC = TR.MULT.X
C
C      CALCULATE REACTIONS AND INVERSE CHECK
C
      REACTS = KELA.MULT.XD
      REACTP= REACTS.SUBT.TLOAD
      IF (DIFF.NULL.) GO TO 10
C
C
C      PRINT ELEMENT APPLIED LOADS, EXTERNAL LOADS, DISPLACEMENTS,
C      REACTIONS AND INVERSE CHECK IN ENGINEERING FORMAT
C
C      ELEMENTS HAVE 1 OR 2 DEGREES OF FREEDOM
C
      GPRINT (4,,,FX.FY.FZ.MX.MY.MZ,SC,TR )FTELA
      GPRINT (4,,,FX.FY.FZ.MX.MY.MZ,SC, )LOADS
      GPRINT (2,,,U.V.W.THETAX.THETAY.THETA.Z,SC, )X
      GPRINT (1,,,FX.FY.FZ.MX.MY.MZ,SC,TR )REACTP
      IF (I3.NULL.) GO TO 600
C
C      ELEMENTS HAVE 3 DEGREES OF FREEDOM
C
C
C      10 GPRINT (4,,,FR.0.FZ.0.MBETA.0.F1.0.F3,SC,TR )FTELA
      GPRINT (4,,,FR.0.FZ.0.MBETA.0.F1.0.F3,SC, )LOADS
      GPRINT (2,,,U.0.W.0.THETAY.0.W*.0.W**,SC, )X
      GPRINT (1,,,FR.0.FZ.0.MBETA.0.F1.0.F3,SC,TR )REACTP
C
C      GENERATE STRESSES AND FORCES
C
C
C      600 STRESP = EM,XD .STRESS. (4, )
      FCRCEP = EM,XD .FORCE. (4, )

```

Figure II-5 (Concluded)

(6) Statics Instruction Sequence With Prescribed Displacements Using Cholesky Triangularization (STATICS2)

Figure II-6 presents the suggested set of abstraction instructions for use in performing a linearly elastic displacement and stress analysis with prescribed displacements. With the use of this option, applied loading may be prescribed in terms of non-zero displacement values. The number of prescribed displaced grid points is the number of grid points that are assigned known values of displacement other than zero.

This set of instructions differs from those reported on pp. 56 thru 68 of Reference 5 in the following respect. The set of simultaneous linear equations which arise in the analysis are solved by Cholesky triangularization. Statement 127 of Figure II-6 has the following form:

$$\text{TRIA},X1 = K11,K4.\text{CHTRIA}.$$

where in Reference 5, the equation solution had the following form:

$$X1 = K11.\text{SEQEL}.K4$$

Note Statement 127 of Figure II-e (Page 58) of Reference 5.



# Contrails

```

8STATICS2                                00000800
C                                          00000810
C-----STATICS AGENDUM WITH PRESCRIBED DISPLACEMENTS 00000820
C                                          00000830
C          STATICS INSTRUCTION SEQUENCE 00000840
C                                          00000850
C          GENERATE ELEMENT MATRICES    00000860
C          ,MLIB,,XLD,TR, ,KEL,FTEL,SEL,STEL,,,SC,EM,PD=,, ,.USER04. 00000870
C                                          00000880
C          FORM (1 X 1) UNIT AND (1 X 1) NULL MATRICES 00000890
C          DETERMINE PRINT FORMAT FOR TYPE OF ELEMENTS USED 00000900
C                                          00000910
C          I1 = SC.IDENTC.                00000920
C          I3 = I1.NULL.SC                00000930
C          DIFF = I1 .SMULT. SC(9,1)      00000940
C                                          00000950
C          ASSEMBLE STIFFNESS MATRIX AND ELEMENT APPLIED LOADS 00000960
C                                          00000970
C          KELA = EM .ASSEM. SC.(10)      00000980
C          FTELA = EM .ASSEM .SC.(40)     00000990
C          LSCALE,LOADS = XLD .DEJOIN.(1,1) 0001000
C                                          0001010
C          REDUCE STIFFNESS MATRIX AND PRINT 0001020
C                                          0001030
C          KO,KNO = KELA .DEJOIN.( SC(5,1),1) 0001040
C          KCO,STIFF = KNO.DEJDIN.( SC(5,1),0) 0001050
C          PRINT(FORCE,DISP,,) STIFF     0001060
C                                          0001070
C          MULTIPLY ELEMENT APPLIED LOADS BY LOAD SCALAR 0001080
C          FTELS = FTELA.MULT.LSCALE     0001090
C          TRANSFORM EXTERNAL LOADS TO 0-1-2 ASSEMBLED SYSTEM 0001100
C          LOADU = TR.MULT.LOADS          0001110
C          FORM TOTAL LOAD COLUMNS       0001120
C          TLJAD = FTELS.ADD.LOADU        0001130
C          TL,TLJADR = TLJAD.DEJOIN.(SC(5,1),1) 0001140
C                                          0001150
C          SOLVE FOR DISPLACEMENTS       0001160
C          PRESCRIBED DISPLACEMENTS ARE PRESENT 0001170
C                                          0001180

C                                          0001190
C          K1,K2 = STIFF.DEJOIN.(SC(6,1),1) 0001200
C          K11,K12 = K1.DEJOIN.(SC(6,1),0) 0001210
C          PD3 = TR.MULT.PD                0001220
C          PR,D2 = PD3 .DEJOIN.( SC(8,1),1) 0001230
C          K3 = K12.MULT.D2                0001240
C          P1,P2 = TLJADR.DEJOIN.(SC(6,1),1) 0001250
C          K4 = P1.SUBT.K3                 0001260
C          TRIA,X1 = K11,K4 .CHTRIA.       0001270
C          X1T = X1.TRANSP.                0001280
C          X2T = D2.TRANSP.                0001290
C          X12T = X1T.ADJJOIN.X2T          0001300
C          XOT = X1T.NULL.KCO              0001310
C          XT = XOT.ADJJOIN.X12T           0001320
C          XO = XT.TRANSP.                  0001330
C          X = TR.TMULT.XO                  0001340
C          CALCULATE AND PRINT REACTIONS 0001350
C                                          0001360

```

Figure II-6: Statics With Prescribed Displacements - Cholesky Triangularization

# Contrails

```

REACTT = KELA.MULT.XO
REACT = REACTT.SUBT.TLOAD
C
C     ELEMENTS HAVE 1 OR 2 DEGREES OF FREEDOM
C
C     PRINT ELEMENT APPLIED LOADS AND EXTERNAL LOADS
C     PRINT ASSEMBLED DISPLACEMENT COLUMN
C
IF (DIFF.NULL.) GO TO 10
GPRINT(4,,,FX.FY.FZ.MX.MY.MZ,SC,TR )FTELA
GPRINT(4,,,FX.FY.FZ.MX.MY.MZ,SC,  )LOADS
GPRINT(2,,,U.V.W.THETAX.THETAY.THETAZ,SC, )X
GPRINT(1,,,FX.FY.FZ.MX.MY.MZ,SC,TR )REACT
IF (I3.NULL.) GO TO 60
C
C     ELEMENTS HAVE 3 DEGREES OF FREEDOM
C
C
10  GPRINT(4,,,FR.O.FZ.O.MBETA.O.F1.O.F3,SC,TR )FTELA
GPRINT(4,,,FR.O.FZ.O.MBETA.O.F1.O.F3,SC,  )LOADS
GPRINT(2,,,U.O.W.O.THETAY.O.W*.O.W**,SC, )X
GPRINT(1,,,FR.O.FZ.O.MBETA.O.F1.O.F3,SC,TR )REACT
C
C     GENERATE STRESSES AND FORCES
C
60  STRESS = EM,XO .STRESS. (4, )
FORCE = EM,XO .FORCE. (4, )
```

```

00001370
00001380
00001390
00001400
00001410
00001420
00001430
00001440
00001450
00001460
00001470
00001480
00001490
00001500
00001510
00001520
00001530
00001540
00001550
00001560
00001570
00001580
00001590
00001600
00001610
00001620
```

Figure II-6 - (Concluded)

## (7) Stability Analysis Instruction Sequence (STABILITY)

Figure II-7 presents the suggested set of abstraction instructions for use in performing elastic instability analyses.

The structural stability analysis is a two-phase process, the first step of which is a linear elastic stress analysis for which the initial stress state is zero. The second phase of the analysis procedure, begins with the formation of element incremental stiffness matrices which are derived from the mid-plane stress resultants determined in the linear stress analysis. After assembly of element incremental stiffness matrices, a linear eigenvalue solution is obtained for the critical buckling load. Using this approach, the assumption is made that all mid-plane forces remain in a fixed ratio to one another at all levels of applied load, from the onset of loading to the achievement of instability. A detailed derivation of the algebraic expressions used for the Stability Analyses is given in Section III of Reference 4.

It is to be noted that in the MAGIC III System, incremental stiffness matrices are provided for the following finite element representations:

- a. Quadrilateral Plate (Ident. No. 28)
- b. Triangular Plate (Ident. No. 27)
- c. Incremental Frame (Ident. No. 13)

The derivations of these elements are presented in detail in Reference 4.

The stability analysis instruction sequence of Reference 5 is presented for comparison purposes in Figure II-8. It is included in this document without change. Detailed matrix operations concerning the use of these operations are presented on pp. 69-80 of Reference 5.

# Contrails

The suggested form of solving the elastic instability analysis, shown in Figure II-7, uses the Cholesky triangularization method. Differences in instructions between Figures II-7 and II-8 are as follows:

Statement No. 346 of Figure II-7 has the form:

FLEX,XR = STIFF,TLOADR.CHTRIA.

where

FLEX = The Triangularized Stiffness Matrix  
XR = The Reduced Displacement Solution Vector  
STIFF = The Reduced Stiffness Matrix  
TLOADR = The Reduced Total Applied Load Vector

(Note p. 8 , Section II.B.2 of this report.)

Once the triangularized stiffness matrix, FLEX, has been determined, and after the assembly of the element incremental stiffness matrices, INCR, Statement 386 is utilized as follows:

EIG = FLEX.CHOL.INCR

where EIG = The solution of the back substitution system.

Statement Number 386 of Figure II-7 is equivalent to:

EIG = FLEX.MULT.INCR which is Statement Number 239 of Figure II-8.

The use of the instructions as outlined in Figure II-7 avoids the inversion of the stiffness matrix which for large order systems may prove inefficient and computationally prohibitive.

# Contrails

```

$STABILITY                                00003110
C                                          00003120
C-----STABILITY AGENDUM ANALYSIS        00003130
C                                          00003140
C      STABILITY ANALYSIS INSTRUCTION SEQUENCE 00003150
C                                          00003160
C      GENERATE ELEMENT MATRICES           00003170
C                                          00003180
C      ,MLIB,INTP,XLD ,TR, ,KEL,FTEL,SEL,STEL,,SC,EM,, ,.USER04. 00003190
C                                          00003200
C      FORM (1 X 1) UNIT AND (1 X 1) NULL MATRICES 00003210
C      DETERMINE PRINT FORMAT FOR TYPE OF ELEMENTS USED 00003220
C                                          00003230
C      I1 = SC.IDENTC.                      00003240
C      I3 = I1.NULL.SC                      00003250
C      DIFF = I1 .SMULT. SC(9,1)           00003260
C                                          00003270
C      ASSEMBLE STIFFNESS MATRIX AND ELEMENT APPLIED LOADS 00003280
C                                          00003290
C      STIFF= EM .ASSEM. SC,(1)             00003300
C      FTELA = EM .ASSEM .SC,(40)          00003310
C      LSCALE,LOADS = XLD .DEJOIN.(1,1)    00003320
C      PRINT(FORCE,DISP,,) STIFF           00003330
C                                          00003340
C      MULTIPLY ELEMENT APPLIED LOADS BY LOAD SCALAR 00003350
C      FTELS = FTELA.MULT.LSCALE           00003360
C      TRANSFORM EXTERNAL LOADS TO 0-1-2 ASSEMBLED SYSTEM 00003370
C      LJCADD = TR.MULT.LLOADS             00003380
C      FORM TOTAL LOAD COLUMNS            00003390
C      TLOAD = FTELS.ADD.LJCADD            00003400
C      FORM REDUCED TOTAL LOAD COLUMN      00003410
C      TL,TLOADR = TLOAD.DEJOIN.( SC(5,1),1) 00003420
C                                          00003430
C      PRINT FLEXIBILITY MATRIX            00003440
C                                          00003450
C      FLEX,XR = STIFF,TLOADR .CHTRIA.     00003460
C      PRINT (DISP,FORCE,,) FLEX          00003470
C                                          00003480
C      SOLVE FOR DISPLACEMENTS            00003490
C                                          00003500
C      TRO,TR12 = TR.DEJOIN.( SC(5,1),1)  00003510
C      X = TR12.TMULT.XR                   00003520
C      XC = TR.MULT.X                      00003530
C      IF (DIFF.NULL.) GO TO 10            00003540

```

Figure II-7 - Stability Instruction Sequence -  
Cholesky Triangularization

# Contrails

```
C          00003550
C          PRINT ELEMENT APPLIED LOADS AND EXTERNAL LOADS          00003560
C          00003570
C          ELEMENTS HAVE 1 OR 2 DEGREES OF FREEDOM                  00003580
C          00003590
C          GPRINT (4,,,FX,FY,FZ,MX,MY,MZ,SC,TR )FTELA                00003600
C          GPRINT (4,,,FX,FY,FZ,MX,MY,MZ,SC, )LOADS                 00003610
C          GPRINT (2,,,U,V,W,THETAX,THETAY,THETAZ,SC, ) X           00003620
C          IF (I3,NULL,) GO TO 60                                     00003630
C          00003640
C          ELEMENTS HAVE 3 DEGREES OF FREEDOM                       00003650
C          00003660
C          10 GPRINT (4,,,FR,0,FZ,0,MBETA,0,F1,0,F3,SC,TR )FTELA     00003670
C          GPRINT (4,,,FR,0,FZ,0,MBETA,0,F1,0,F3,SC, )LOADS         00003680
C          GPRINT (2,,,U,0,W,0,THETAY,0,W*,0,W**,SC, ) X            00003690
C          00003700
C          GENERATE STRESSES                                         00003710
C          00003720
C          60 STRESS = EM,XO .STRESS. (4,)                            00003730
C          00003740
C          GENERATE ELEMENT INCREMENTAL STIFFNESS MATRIX           00003750
C          00003760
C          ,,,,,,,,,,NEL,, ,EL,=,INTP, ,STRESS.USER04.             00003770
C          00003780
C          ASSEMBLE AND REDUCE INCREMENTAL MATRIX                   00003790
C          00003800
C          INCR = EL .ASSEM. SC,(3)                                  00003810
C          PRINT(,,, ) INCR                                         00003820
C          00003830
C          CREATE INPUT EIGENVALUE MATRIX                            00003840
C          00003850
C          EIG = FLEX .CHOL. INCR                                    00003860
C          PRINT (,,, ) EIG                                         00003870
C          00003880
C          CALCULATE AND PRINT E-VALUES,E-VECTORS,FREQUENCIES     00003890
C          00003900
C          EVALUE,EVECTR,, = EIG, .EIGEN1. SC                       00003910
C          GPRINT (3,,,,,SC,TR12) EVECTR,EVALUE                    00003920
```

Figure II-7 - Concluded







```

ELEMENTS HAVE 3 DEGREES OF FREEDOM
10  GPRINT(4,,,FR.O.FZ.O.MBETA.O.F1.O.F3,SC,TR )FTELA
    GPRINT(4,,,FR.O.FZ.O.MBETA.O.F1.O.F3,SC, )LOADS
    GPRINT(2,,,U.O.W.O.THETAY.O.W*.O.W**,SC, ) X
    GENERATE STRESSES
50  STRESS = EM,XO .STRESS. (4,)
    GENERATE ELEMENT INCREMENTAL STIFFNESS MATRIX
    ,,,,,,NEL,, ,EL,=,INTP, ,STRESS.USER04.
    ASSEMBLE AND REDUCE INCREMENTAL MATRIX
    INCR = EL .ASSEM. SC,(3)
    PRINT(,,, ) INCR
    CREATE INPUT EIGENVALUE MATRIX
    EIG = FLEX.MULT.INCR
    PRINT (,,, ) EIG
    CALCULATE AND PRINT E-VALUES,E-VECTORS,FREQUENCIES
    EVALUE,EVECTR,, = EIG, .EIGEN1. SC
    GPRINT(3,,,SC,TR12) EVECTR,EVALUE
```

Figure II-8 (Concluded)

## (8) Dynamics Analysis Instruction Sequence (DYNAMICS)

Figure II-9 presents the suggested set of abstraction instructions for use in performance of a vibration analysis. This particular set of instructions provides modes and frequencies for a structural system in which the rigid body modes have been suppressed (i.e. the assembled stiffness matrix has been rendered non-singular by the appropriate application of physical boundary conditions. As seen from Figure II-9 the .EIGEN 1. abstraction instruction is used in this sequence. This instruction is based on the "power method" of extracting eigenvalues and eigenvectors. The desired number of modes and frequencies are supplied as input by the User in the Structural Analysis Input Section. This information is contained on a specialized preprinted input data form entitled DYNAM. This form was described in detail in the Structural Input Data Section of Reference 5.

The Dynamics Analysis Instruction Sequence has been written to accommodate non-structural lumped masses to augment the structural mass matrix generated by the MAGIC III System. A specialized preprinted input data form entitled Lumped Masses has been provided for input of the lumped mass values and is displayed in Figure II-16 of Section II.C.5. It is noted that this data form can also be utilized to input lumped structural mass values at the option of the User.

If this were the case, the User would specify a mass density value of zero (0.0) on the Material Tape Input Section data form which is described in detail on pp. 97 - 101 of Reference 5. In addition, output matrix position twelve (OMP 12) of the USER04. instruction. Statement No. 401 of Figure II-9 would be left blank so that the MAGIC III System would not generate element mass matrices (MEL) for the application in question.

# *Contrails*

Additional output data from this set of instructions include generalized mass and generalized stiffness values for each mode requested.

Table V is provided as a supplement to Figure II-9. This Table provides engineering and matrix definition for each abstraction instruction listed in Figure II-9.

```
DYNAMICS                                00003930
-----DYNAMICS AGENDUM ANALYSIS        00003940
                                         00003950
      DYNAMICS ANALYSIS INSTRUCTION SEQUENCE 00003960
                                         00003970
      GENERATE ELEMENT MATRICES          00003980
                                         00003990
      ,MLIB,MLD,TR, ,KEL, , , ,MEL,SC,EM, = , , ,USER04. 00004000
                                         00004010
      ASSEMBLE STIFFNESS MATRIX AND MASS MATRIX 00004020
                                         00004030
      STIFF= EM .ASSEM. SC,(1)           00004040
      MASSM = EM .ASSEM. SC,(2)         00004050
                                         00004060
      DEFINE LUMP MASS AND TOTAL MASS MATRIX 00004070
                                         00004080
      MSCAL,LMASS = MLD .DEJOIN. (1,1)  00004090
      LUMPC = TR .MULT. LMASS           00004100
      LL,LUMP = LUMPC .DEJOIN. (SC(5,1),1) 00004110
      DLUMP = LUMP .DIAGON.             00004120
                                         00004130

      MASS = MASSM .ADD. DLUMP          00004140
                                         00004150
      PRINT STIFFNESS MATRIX AND MASS MATRIX 00004160
      PRINT(FORCE,DISP,,) STIFF         00004170
      PRINT(FORCE,ACCEL,,) MASS         00004180
                                         00004190
      GENERATE DYNAMICS MATRIX          00004200
                                         00004210
      KINV,DYNAM = STIFF,MASS .CHTRIA.  00004220
                                         00004230
      FIND E-VALUES, E-VECTORS, NORMAL MODES, 00004240
      FREQUENCIES AND PRINT            00004250
                                         00004260
      EVALUE,EVECT,, = DYNAM, .EIGEN1. SC 00004270
                                         00004280
      TRO,TR12 = TR .DEJOIN. (SC(5,1),1) 00004290
      GPRINT(3, , , ,SC,TR12) EVECT,EVALUE 00004300
                                         00004310
      GENERATE STIFFNESS AND GENERALIZED MASS 00004320
      MATRICES AND PRINT                00004330
                                         00004340
      KGEN1 = EVECT.TMULT.STIFF         00004350
      KGEN = KGEN1.MULT.EVECT           00004360
      MGEN1 = EVECT.TMULT.MASS          00004370
      MGEN = MGEN1.MULT.EVECT          00004380
      PRINT( , , ,) MGEN,KGEN,KINV,DYNAM 00004390
                                         00004400
                                         00004410
```

Figure II-9 - Dynamics Analysis Instruction Sequence

TABLE V  
DYNAMICS INSTRUCTION SEQUENCE  
(STEP BY STEP DESCRIPTION)

STATEMENT SEQUENCE NUMBER	INSTRUCTION AND EXPLANATION
401	<p>,MLIB,,MLD,TR,,KEL,,,,,MEL,SC,EM,=,,,USERO4.</p> <p>Generates the element stiffness matrices KEL, lumped mass matrix column MLD, and element mass matrices MEL, required for the dynamics problem.</p>
405	<p>STIFF=EM.ASSEM.SC,(1)</p> <p>Forms the assembled reduced stiffness matrix, STIFF from the element stiffness matrices stored in EM. SC contains system constants required by the .ASSEM. routine.</p>
406	<p>MASSM=EM.ASSEM.SC,(2)</p> <p>Forms the assembled reduced mass matrix, MASS from the element mass matrices stored in EM. System information required by .ASSEM. is input in SC.</p>
410	<p>MSCAL,LMASS=MLD.DEJOIN.(1,1)</p> <p><math>\begin{bmatrix} \text{MSCAL} \\ \text{LMASS} \end{bmatrix} = \text{MLD}</math></p> <p>The mass scalar, MSCAL and the lumped mass column LMASS are dejoined in the MLD matrix. It is noted that MSCAL is the first row of MLD.</p>
411	<p>LUMPO=TR.MULT.LMASS</p> <p><math>\begin{bmatrix} \text{LUMPO} \end{bmatrix} = \begin{bmatrix} \text{TR} \end{bmatrix} \begin{bmatrix} \text{LMASS} \end{bmatrix}</math></p> <p>Transforms the unordered total lumped mass column, LMASS, to the 0-1-2 ordered assembled column, LUMPO.</p>
412	<p>LL,LUMP=LUMPO.DEJOIN.(SC(5,1),1)</p> <p><math>\begin{bmatrix} \text{LL} \\ \text{LUMP} \end{bmatrix} = \begin{bmatrix} \text{LUMPO} \end{bmatrix}</math></p> <p>Forms the reduced total lumped mass column, LUMP, which reflects 1's and 2's.</p>
413	<p>DLUMP=LUMP.DIAGON.</p> <p>Diagonalizes the vector, LUMP, to form a square diagonal matrix, DLUMP.</p>

TABLE V  
(CONTINUED)

STATEMENT SEQUENCE NUMBER	INSTRUCTION AND EXPLANATION
414	<p>MASS=MASSM.ADD.DLUMP  <math>[MASS] = [MASSM] + [DLUMP]</math>                      Augments the assembled structural mass matrix, MASSM with the additional (non-structural) contribution DLUMP to form the total mass matrix, MASS.</p>
418	<p>PRINT(FORCE,DISP,,) STIFF                      Prints the reduced stiffness matrix.</p>
420	<p>PRINT(FORCE,ACCEL,,) MASS                      Prints the reduced mass matrix.</p>
424	<p>KINV,DYNAM=STIFF,MASS.CHTRIA.                      Solves the following set of equations:  <math>[STIFF][DYNAM] = [MASS]</math>  <math>[KINV] =</math> Triangularized stiffness matrix  <math>[DYNAM]</math> is the dynamic matrix and is equivalent to the inverse of the stiffness matrix times the mass matrix, i.e., <math>[K]^{-1} [M]</math>.</p>
429	<p>EVALUE, ETECT, ,=DYNAM,.EIGEN1.SC                      solve <math>[[DYNAM] - [EVALUE][I]] [ETECT] = [0]</math>                      Computes the required eigenvalues and corresponding eigenvectors of the dynamics matrix using the power method. The eigenvalues are stored in the column matrix EVALUE and the corresponding eigenvectors are stored as columns in ETECT. The frequencies and mode shapes are also printed out.</p>
431	<p>TRO,TR12 = TR.DEJOIN.(SC(5,1),1)  <math>\begin{bmatrix} TRO \\ TR12 \end{bmatrix} = [TR]</math>                      Forms the matrix TR12 which will be used by the .GPRINT. instruction.</p>
432	<p>.GPRINT.(3,,,SC,TR12)ETECT,EVALUE                      Prints the eigenvalue column and the eigenvector in engineering format.</p>

TABLE V  
(CONTINUED)

STATEMENT SEQUENCE NUMBER	INSTRUCTION AND EXPLANATION
437	<p> <math display="block">\text{KGEN1} = \text{EVECT.TMULT.STIFF}</math> <math display="block">[\text{KGEN1}] = [\text{EVECT}]^T [\text{STIFF}]</math> </p> <p>Forms the product of the transpose of the eigenvector matrix and the reduced stiffness matrix.</p>
438	<p> <math display="block">\text{KGEN} = \text{KGEN1.MULT.EVECT}</math> <math display="block">[\text{KGEN}] = [\text{EVECT}]^T [\text{STIFF}] [\text{EVECT}]</math> </p> <p>Forms the generalized stiffness matrix in KGEN by forming the product of KGEN1 and EVECT.</p>
439	<p> <math display="block">\text{MGEN1} = \text{EVECT.TMULT.MASS}</math> <math display="block">[\text{MGEN1}] = [\text{EVECT}]^T [\text{MASS}]</math> </p> <p>Forms the product of the transpose of the eigenvalue matrix and the reduced mass matrix.</p>
440	<p> <math display="block">\text{MGEN} = \text{MGEN1.MULT.EVECT}</math> <math display="block">[\text{MGEN}] = [\text{EVECT}]^T [\text{MASS}] [\text{EVECT}]</math> </p> <p>Forms the generalized mass matrix in MGEN by forming the product of MGEN1 and EVECT.</p>
441	<p> <math display="block">\text{PRINT}(,,,) \text{MGEN, KGEN, KINV, DYNAM}</math> </p> <p>Prints the generalized stiffness matrix, the generalized mass matrix, the triangularized stiffness matrix and the dynamic matrix.</p>



## (9) Free-Free Dynamics Analysis Instruction Sequence (DYNAMICSF)

Figure II-10 presents the suggested set of abstraction instructions for use in performance of a free-free vibration analysis. This particular set of instructions provides modes and frequencies for a structural system in which the rigid body modes are present. Provision for lumped non-structural mass is provided as well as the provision for lumped structural mass. The values are input, if required, via the lumped mass preprinted input data form shown in Figure II-16. It is noted from the lumped mass form that provision is made to input a mass scalar value. This value is utilized in the performance of a free-free vibration analysis as follows.

Given the equations of motion of a free-free system:

$$[M] \{\ddot{\delta}\} + [K_f] \{\delta\} = \{0\} \quad (1)$$

where  $[K_f]$  is a singular stiffness matrix. The natural frequencies and corresponding mode shapes can be determined from lowest to highest by solution of the following eigenvalue problem.

$$\left[ a_0 [M] + [K_f] \right]^{-1} [M] \{\phi_i\} = \lambda_i \{\phi_i\} \quad (2)$$

from which the natural frequencies may be recovered as follows:

$$f_{n_i} = \frac{1}{2\pi} \sqrt{\left( \frac{1}{\lambda_i} \right) - a_0} \quad (3)$$

where  $a_0$  is the mass scalar value input on the lumped mass input data form. Detailed discussion of the above procedure can be found in References 12 and 13.

It is noted that when the above technique is utilized, caution must be exercised in choosing the value of the scalar  $a_0$ . Problems arise in some cases when diagonal mass matrices are employed whose terms are on the order of  $10^{-2}$  compared to terms on

```

SDYNAMICSF                                00004420
C                                           00004430
C * * * * *                               00004440
C-----DYNAMICS (FREE-FREE) AGENDUM ANALYSIS. 00004450
C      DYNAMICS ANALYSIS INSTRUCTION SEQUENCE 00004460
C      GENERATE ELEMENT MATRICES              00004470
C      ,MLTB,,M, TR,, KEL,,,,,MEL,SC,EM, = ,,,, .USER04. 00004480
C                                           00004490
C      ASSEMBLE STIFFNESS AND CONSISTENT MASS MATRICES 00004500
C                                           00004510
C      STIFF = EM .ASSEM. SC, (1)              00004520
C      MASSM = EM .ASSEM. SC, (2)             00004530
C      DEFINE LUMP MASS                       00004540
C      MSCAL, LMASS = M .DEJOIN.(1,1)         00004550
C      LUMPO = TR .MULT. LMASS                00004560
C      LL, LUMP = LUMPO .DEJOIN. ( SC(5,1),1) 00004570
C      DLUMP = LUMP .DIAGON.                  00004580
C                                           00004590
C      DEFINE TOTAL MASS MATRIX               00004600
C      MASS = MASSM .ADD. DLUMP               00004610
C      MASS1 = MASS .SMULT. MSCAL(1,1)        00004620
C                                           00004630
C      PRINT STIFFNESS AND MASS MATRICES     00004640
C      PRINT (FORCE, DISP,,) STIFF           00004650
C      PRINT ( FORCE, ACCEL,,) MASS           00004660
C                                           00004670
C      COMPUTE DYNAMIC MATRIX                 00004680
C                                           00004690
C      STIFFM = MASS1 .ADD. STIFF             00004700
C      FLEX, DYNAM = STIFFM, MASS .CHTRIA.    00004710
C                                           00004720
C      GENERATE E-VALUES AND FREQUENCIES AND PRINT 00004730
C                                           00004740
C      EVALUE, ETECT,, = DYNAM, .EIGEN1. SC   00004750
C      TRO,TR12 = TR .DEJOIN.(SC(5,1),1)     00004760
C      GPRINT ( 3,,,, SC, TR12) ETECT,EVALUE 00004770
C                                           00004780
C      GENERATE GENERALIZED STIFF AND MASS , PRINT 00004790
C                                           00004800
C      KGEN1 = ETECT .TMULT. STIFF            00004810
C      KGEN = KGEN1 .MULT. ETECT.            00004820
C      MGEN1 = ETECT .TMULT. MASS            00004830
C      MGEN = MGEN1 .MULT. ETECT            00004840
C      PRINT (,,, ) MGEN, KGEN, FLEX, DYNAM 00004850

```

Figure II-10 - Free-Free Dynamics Analysis Instruction Sequence

the order of  $10^6$  in the stiffness matrix. This requires the analyst to adjust the value of  $a_0$ , so that the matrix product  $a_0 [M]$  when added to the stiffness matrix will render it non-singular. A large value of  $a_0$  can cause problems when the elastic frequencies of interest are low (say below 10 to 15 cps) since the frequencies being calculated are a function of:

$$\sqrt{\left(\frac{1}{\lambda_i}\right) - a_0}$$

It has been found, in general, that when consistent mass matrices are employed in the vibration analysis, a value of  $a_0 = 1.0$  will usually suffice as the scalar value of the mass matrix multiplier.

Table VI is provided as a supplement to Figure II-10. This Table provides engineering and matrix definition for each abstraction instruction listed in Figure II-10.

TABLE VI  
 FREE-FREE DYNAMICS INSTRUCTION SEQUENCE  
 (STEP BY STEP DESCRIPTION)

STATEMENT SEQUENCE NUMBER	INSTRUCTION AND EXPLANATION
448	<p>,MLIB,,M,TR,,KEL,,,,,MEL,SC,EM,=,,,,.USERO4.</p> <p>Generates the element stiffness matrices, KEL, lumped mass matrix column, M and element mass matrices MEL, required for the dynamics problem.</p>
452	<p>STIFF=EM.ASSEM.SC,(1)</p> <p>Forms the assembled reduced stiffness matrix, STIFF from the element stiffness matrices stored in EM. SC contains system constants required by the .ASSEM. routine.</p>
453	<p>MASSM=EM.ASSEM.SC,(2)</p> <p>Forms the assembled reduced mass matrix, MASS from the element mass matrices stored in EM. System information required by .ASSEM. is input in SC.</p>
455	<p>MSCAL,LMASS=M.DEJOIN.(1,1)</p> $\begin{bmatrix} \text{MSCAL} \\ \text{LMASS} \end{bmatrix} = [M]$ <p>The mass scalar, MSCAL and the lumped mass column LMASS are dejoined in the M matrix. It is noted that MSCAL is the first row of M.</p>
456	<p>LUMPO = TR.MULT.LMASS</p> $[LUMPO] = [TR][LMASS]$ <p>Transforms the unordered total lumped mass column, LMASS, to the 0-1-2 ordered assembled column, LUMPO.</p>
457	<p>LL,LUMP=LUMPO.DEJOIN.(SC(5,1),1)</p> $\begin{bmatrix} \text{LL} \\ \text{---} \\ \text{LUMP} \end{bmatrix} = [LUMPO]$ <p>Forms the reduced total lumped mass column, LUMP, which reflects 1's and 2's.</p>

TABLE VI  
(CONTINUED)

STATEMENT SEQUENCE NUMBER	INSTRUCTION AND EXPLANATION
458	DLUMP=LUMP.DIAGON. Diagonalizes the vector, LUMP, to form a square diagonal matrix, DLUMP.
461	MASS=MASSM.ADD.DLUMP $[MASS] = [MASSM] + [DLUMP]$ Augments the assembled structural mass matrix, MASSM, with the additional (non-structural) contribution DLUMP to form the total mass matrix, MASS.
462	MASS1=MASS.SMULT.MSCAL(1,1) $[MASS1] = MSCAL [MASS]$ Performs the scalar multiplication of MSCAL times MASS. This is equivalent to $a_o [M]$ detailed in the writeup.
465	PRINT(FORCE,DISP,,) STIFF Prints the reduced stiffness matrix.
466	PRINT(FORCE,ACCEL,,) MASS Prints the reduced mass matrix.
470	STIFFM=MASS1.ADD.STIFF $[STIFFM] = [MASS1] + [STIFF]$ Adds $[MASS1]$ to $[STIFF]$ to form $[STIFFM]$ . This is equivalent to $[a_o [M] + [K]]$ as described in the writeup.
471	FLEX,DYNAM=STIFFM,MASS.CHTRIA. Solves the following set of equations $[STIFFM] [DYNAM] = [MASS]$ $[FLEX] = \text{Triangularized Stiffness Matrix}$ $[DYNAM]$ is the dynamic matrix and is equivalent to the inverse of the stiffness matrix times the mass matrix, i.e., $[K]^{-1} [M]$ .

TABLE VI  
(CONTINUED)

STATEMENT SEQUENCE NUMBER	INSTRUCTION AND EXPLANATION
475	<p>EVALUE,EVECT,,=DYNAM,.EIGEN1.SC            solve <math>\left[ \begin{matrix} \text{[DYNAM]} &amp; - &amp; \text{[EVALUE]}\text{[I]} \end{matrix} \right] \text{[EVECT]} = \text{[0]}</math>            Computes the required eigenvalues and corresponding eigenvectors of the dynamics matrix using the power method. The eigenvalues are stored in the column matrix EVALUE and the corresponding eigenvectors are stored as columns in EVECT. The frequencies and mode shapes are also printed out.</p>
476	<p>TRO,TR12=TR.DEJOIN.(SC(5,1),1)  <math>\begin{bmatrix} \text{TRO} \\ \text{TR12} \end{bmatrix} = \text{[TR]}</math>            Forms the matrix TR12 which is used by the .GPRINT. instruction.</p>
477	<p>.GPRINT.(3,,,SC,TR12)EVECT,EVALUE            Prints the eigenvalue column and the eigenvector matrix in engineering format.</p>
481	<p>KGEN1=EVECT.TMULT.STIFF  <math>\text{[KGEN1]} = \text{[EVECT]}^T \text{[STIFF]}</math>            Forms the product of the transpose of the eigenvector matrix and the reduced stiffness matrix.</p>
482	<p>KGEN=KGEN1.MULT.EVECT  <math>\text{[KGEN]} = \text{[EVECT]}^T \text{[STIFF]}\text{[EVECT]}</math>            Forms the generalized stiffness matrix in KGEN by forming the product of KGEN1 and EVECT.</p>
483	<p>MGEN1=EVECT.TMULT.MASS  <math>\text{[MGEN1]} = \text{[EVECT]}^T \text{[MASS]}</math>            Forms the product of the transpose of the eigenvalue matrix and the reduced mass matrix.</p>
484	<p>MGEN=MGEN1.MULT.EVECT  <math>\text{[MGEN]} = \text{[EVECT]}^T \text{[MASS]}\text{[EVECT]}</math>            Forms the generalized mass matrix in MGEN by forming the product of MGEN1 and EVECT.</p>

TABLE VI  
(CONTINUED)

STATEMENT SEQUENCE NUMBER	INSTRUCTION AND EXPLANATION
485	PRINT(,,,)MGEN,KGEN,FLEX,DYNAM Prints the generalized stiffness matrix, the generalized mass matrix, the triangularized stiffness matrix, and the dynamic matrix.



(10) Dynamics Analysis Instruction Sequence With  
Condensation (DYNAMICSC)

Figure II-11 presents the suggested set of abstraction instructions for use in performance of a vibration analysis utilizing condensation. The condensation technique used is that of Guyan (Reference 11).

The use of this technique allows degrees of freedom considered to be superfluous to be eliminated through the use of a condensation transformation. The technique is analogous to that of Statics with Condensation (STATICSC) with the additional step of applying the condensation transformation to the mass matrix as well as the stiffness matrix. This technique yields an eigenvalue problem which is much reduced in size.

As with the standard dynamics agendum of Figure II-9 (DYNAMICS), lumped structural and non-structural masses are accommodated. The specialized preprinted input data form entitled Lumped Masses (Figure II-16) is utilized, if required.

Degrees of freedom that are considered superfluous and are to be condensed (eliminated) in a particular analysis are designated by the number '2' in the Boundary Condition Section which was discussed in detail on pp. 129-133 of Reference 5.

A detailed algebraic statement of the condensation procedure which is performed using the instructions of Figure II-11 is given on pp. 87-89 of Reference 5.

```
DYNAMICS C                                00004860
----- DYNAMICS AGENDUM, WITH CONDENSATION 00004870
-----DYNAMICS AGENDUM ANALYSIS          00004880
                                           00004890
                                           00004900
                                           00004910
                                           00004920
                                           00004930
                                           00004940
                                           00004950
                                           00004960
                                           00004970
                                           00004980
                                           00004990
                                           0005000
                                           0005010
                                           0005020
                                           0005030
                                           0005040
                                           0005050
                                           0005060
                                           0005070
                                           0005080
                                           0005090
                                           0005100
                                           0005110
                                           0005120
                                           0005130
                                           0005140
                                           0005150
                                           0005160
                                           0005170
                                           0005180
                                           0005190
                                           0005200
                                           0005210
                                           0005220
                                           0005230
                                           0005240
                                           0005250
                                           0005260
                                           0005270
                                           0005280
                                           0005290
                                           0005300
                                           0005310

DYNAMICS ANALYSIS INSTRUCTION SEQUENCE

GENERATE ELEMENT MATRICES

,MLIB,MLD,TR,,KEL,,,,MEL,SC,EM, = ,,,,USER04.

ASSEMBLE STIFFNESS MATRIX AND MASS MATRIX

STIFF = EM .ASSEM. SC,(1)
MASSM = EM .ASSEM. SC,(2)

DEFINE LUMP MASS AND TOTAL MASS MATRIX

MSCAL,LMASS = MLD .DEJOIN. (1,1)
LUMPO = TR .MULT. LMASS
LL,LUMP = LUMPO .DEJOIN. (SC(5,1),1)
DLUMP = LUMP .DIAGON.
MASS = MASSM .ADD. DLUMP

PRINT STIFFNESS MATRIX AND MASS MATRIX
PRINT(FORCE,DISP,,) STIFF

PRINT(FORCE,ACCEL,,) MASS

GENERATE DYNAMICS MATRIX

TOP,BOT = STIFF .DEJOIN. (SC(6,1),1)
K11,K12 = TOP .DEJOIN. (SC(6,1),0)
K12T,K22 = BOT .DEJOIN. (SC(6,1),0)
K22I,KR1 = K22,K12T .CHTRIA.
KR2 = K12 .MULT. -KR1
KR = K11 .ADD. KR2
IDENT = K11 .IDENTR.
KRIT = -KR1 .TRANSP.
GAMT = IDENT .ADJOIN. KRIT
GAM = GAMT .TRANSP.
MR1 = GAMT .MULT. MASS
MR = MR1 .MULT. GAM
KRI,DYNAM = KR,MR .CHTRIA.
```

Figure II-11 - Dynamics Analysis Instruction Sequence with Condensation

```
FIND E-VALUES, E-VECTORS, NORMAL MODES,          00005320
FREQUENCIES AND PRINT                             00005330
EVALUE,EVECT,, = DYNAM, .EIGEN1. SC              00005340
T R01, TR2 = TR .DEJOIN. (SC(8,1),1)             00005350
T R0,TR1 = T R01 .DEJOIN. (SC(5,1),1)           00005360
GPRINT (3,,,SC,TR1) EVECT,EVALUE                00005370
GENERATE STIFFNESS AND GENERALIZED MASS          00005380
MATRICES AND PRINT                               00005390
KGEN1 = EVECT.TMULT.KR                           00005400
KGEN = KGEN1.MULT.EVECT                          00005410
MGEN1 = EVECT.TMULT.MR                           00005420
MGEN = MGEN1.MULT.EVECT                          00005430
PRINT(...,MGEN,KGEN,DYNAM,KR,MR)                00005440
                                                00005450
                                                00005460
                                                00005470
                                                00005480
                                                00005490
```

Figure II-11 -(Concluded)

(11) Free-Free Dynamics Analysis Instruction Sequence  
with Condensation (DYNAMICSCF)

Figure II-12 presents the suggested set of abstraction instructions for use in performance of a free-free vibration analysis with condensation. This particular set of instructions provides modes and frequencies for a structural system in which the rigid body modes are present and for which the technique of condensation is employed. Provision for lumped non-structural mass is provided as well as the provision for lumped structural mass. The Mass Scalar value,  $a_0$ , described in the Free-Free Dynamics Analysis Instruction Sequence previously is available to this set of instructions and is used in exactly the same manner as in DYNAMICSCF.

Degrees-of-freedom that are considered superfluous and are to be condensed (eliminated) in a particular analysis are designated by the number '2' in the Boundary Condition Section which was discussed in detail on pp. 129-133 of Reference 5. It is noted that User judgement is required in deciding which degrees-of-freedom in a particular analysis are superfluous and which are essential. An objective approach to this decision making process is presented in Reference 14.

The procedure utilized in Figure II-12 is very similar to that employed in dynamic substructuring. A detailed algebraic statement of the dynamic substructuring process is given on pp. 146-165 of Reference 4.

DYNAMICS CF	0005500
----	0005510
---- DYNAMICS AGENDUM, WITH CONDENSATION	0005520
----	0005530
----DYNAMICS AGENDUM ANALYSIS	0005540
----	0005550
DYNAMICS ANALYSIS INSTRUCTION SEQUENCE	0005560
----	0005570
GENERATE ELEMENT MATRICES	0005580
----	0005590
, MLIB, M, TR,, KEL,,,,,MEL,SC,EM, = ,,, .USER04.	0005600
----	0005610
ASSEMBLE STIFFNESS AND CONSISTENT MASS MATRICES	0005620
----	0005630
STIFF = EM, ASSEM, SC, (1)	0005640
MASSM = EM, ASSEM, SC,(2)	0005650
DEFINE LUMP MASS	0005660
MSCAL, LMASS = M, DEJOIN.(1,1)	0005670
LUMPO = TR .MULT. LMASS	0005680
LL, LUMP = LUMPO .DEJOIN. (SC(5,1),1)	0005690
DLUMP = LUMP .DIAGON.	0005700
----	0005710
DEFINE TOTAL MASS MATRIX	0005720
MASS = MASSM .ADD. DLUMP	0005730
----	0005740
PRINT STIFFNESS MATRIX AND MASS MATRIX	0005750
PRINT(FORCE, DISP,, ) STIFF	0005760
----	0005770
PRINT(FORCE, ACCEL,, ) MASS	0005780
----	0005790
GENERATE DYNAMICS MATRIX	0005800
----	0005810
TCP,BOT = STIFF .DEJOIN. (SC(6,1),1)	0005820
K11,K12 = TOP .DEJOIN. (SC(6,1),0)	0005830
K12T,K22 = BOT .DEJOIN. (SC(6,1),0)	0005840
K22I,KR1 = K22,K12T .CHTRIA.	0005850
KR2 = K12 .MULT. -KR1	0005860
KR = K11 .ADD. KR2	0005870
IDENT = K11 .IDENTR.	0005880
KR1T = -KR1 .TRANSP.	0005890
GAMT = IDENT .ADJOIN. KR1T	0005900

Figure II-12 - Free-Free Dynamics Analysis Instruction Sequence with Condensation

```

GAM = GAMT .TRANSP.                                00005910
MR1 = GAMT .MULT. MASS                             00005920
MR = MR1 .MULT. GAM                               00005930
MM = MR .SMULT. MSCAL(1,1)                        00005940
KM = MM . ADD. KR                                 00005950
KRI,DYNAM = KM,MR .CHTR IA.                       00005960
                                                    00005970
    FIND E-VALUES, E-VECTORS, NORMAL MODES,
    FREQUENCIES AND PRINT                          00005980
                                                    00005990
EVALUE,EVECT,, = DYNAM, .EIGEN1. SC               00006000
                                                    00006010
                                                    00006020
TR01, TR2 = TR .DEJIN. (SC(8,1),1)                00006030
TR0,TR1 = TR01 .DEJIN. (SC(5,1),1)                00006040
GPRINT (3,,,SC,TR1) EVECT,EVALUE                 00006050
                                                    00006060
    GENERATE STIFFNESS AND GENERALIZED MASS
    MATRICES AND PRINT                              00006070
                                                    00006080
KGEN1 = EVECT.TMULT.KR                             00006090
KGEN = KGEN1.MULT.EVECT                            00006100
MGEN1 = EVECT.TMULT.MR                             00006110
MGEN = MGEN1.MULT.EVECT                            00006120
                                                    00006130
PRINT(,,, ) MGEN,KGEN,DYNAM,KR,MP                 00006140
                                                    00006150

```

Figure II-12 - (Concluded)

## d. Agendum Level Abstraction Instructions

The Agendum level abstraction capability incorporated into the MAGIC II System has been retained and expanded in the MAGIC III System. The abstraction instructions for specified analyses will be automatically generated for the User when he specifies the corresponding option on the \$INSTRUCTION card. The Agendum library is expandable and the addition of more abstraction instruction sequences (Agendum) only requires the updating of subroutine AGENDM, and of course the Agendum library itself. The use of an Agendum in no way restricts the User because he can include in his input deck his own abstractions to be merged with the selected agendum.

Subroutine AGENDM controls the selection from the Agendum library of the abstraction instruction sequence requested on the \$INSTRUCTION card. At present, this subroutine has the capability to select the following Agendums.

1. STATICSASYM (Linear Elastic Displacement and Stress Analysis, Triangular Ring -Asymmetric Loading)
2. STATICS (Linear Elastic Displacement and Stress Analysis)
3. STATICSC (Linear Elastic Displacement and Stress Analysis With Condensation)
4. STATICS2 (Linear Elastic Displacement and Stress Analysis With Prescribed Displacements)
5. STABILITY (Linear Elastic Instability Analysis Using Cholesky Triangularization)
6. STABILITYA (Linear Elastic Instability Analysis Using Matrix Inversion)
7. DYNAMICS (Vibration Frequencies, Mode Shapes, Generalized Mass and Stiffness for Supported Structures)



# Contrails

8. DYNAMICSF (Free-Free Vibration Frequencies, Mode Shapes, Generalized Mass and Generalized Stiffness for Unsupported Structures)
9. DYNAMICSC (Vibration Frequencies, Mode Shapes, Generalized Mass and Generalized Stiffness with Condensation for Supported Structures)
10. DYNAMICSCF (Free-Free Vibration Frequencies, Mode Shapes, Generalized Mass and Generalized Stiffness with Condensation for Unsupported Structures)

The present AGENDUM Library is designed to be updated as new Agendums become available. The programming procedure utilized to add additional options to the library is discussed in Appendix IX of Reference 8.

It is emphasized that the User is not restricted to the use of the above Agendums. They are included as a convenience feature to automatically generate the required instructions for a given standard analysis.

An example of non-agendum usage is as follows

```
CC
1      7      16
$MAGIC
$RUN      GO
$INSTRUCTION  SOURCE
      [User Input Abstraction Instructions]
$SPECIAL
      [Report From Input Deck for .USER04. Instruction]
$END
```

## C. STRUCTURAL INPUT DATA

### 1. General Description

Significant portions of the labor and computer costs of structural analysis are occasioned by incomplete or improper specification of structural input data. In recognition of this, a number of features have been incorporated into the MAGIC System to assist in the confirmation of problem data prior to execution. The most important of these are the prelabeled input data forms which are an integral part of the MAGIC System.

All features which were incorporated into MAGIC I and II are retained and expanded in MAGIC III. Additional prelabeled input data forms have been added to MAGIC III to support the expanded capability of the System. These input data forms contain a number of special features, e.g.,

- (1) "MODAL" Options are provided which preset a table to a given set of values. This MODAL option may be used where indicated.
- (2) "REPEAT" Options are provided which minimize the input data specified by the User. This REPEAT option may be used where indicated.
- (3) The User exercises control options simply by placing an 'X' in a given location on a prelabeled input data form.
- (4) The prelabeled input data forms have permanent label cards which automatically precede subsets of data thereby allowing flexibility in the arrangement of input decks.
- (5) Zeros must be indicated where pertinent. Blanks are never zeros except where specifically indicated.
- (6) Only prelabeled input forms associated with options that are exercised in any particular problem are needed. Data associated with options not exercised are simply omitted.

# Contrails

Prelabeled input data forms new to the MAGIC III System are as follows:

- (1) Element Temperature Input Section
- (2) Element Pressure Input Section
- (3) Element Pre-Strain and Pre-Stress Input Section
- (4) Lumped Mass and Free-Free Input Data Section

Additional prelabeled input data forms peculiar to the triangular ring element which accommodates asymmetric loading have also been added to MAGIC III. These data forms will be described in detail in the Element Input Section which appears later in this document.

The numerical input pertinent to the above data is presented in floating point and fixed point notations. In floating point notation, the decimal point is always shown on the input data and in fixed point notation the decimal is never shown. The floating point notation is applicable, for example, to measurable quantities such as loads, coordinates, etc. The fixed point notation is limited to whole numbers or integers such as grid point numbers.

In floating point notation, a number may be written in either the conventional manner or as a factor of  $10^n$ ; for example, the number 30 000 000 =  $30 \times 10^6$  can be written as either 30 000 000 or 30.0 E6. For numerical input data (both fixed and floating point) plus signs are not normally used. Negative numbers and negative exponents, however, must be preceded by a minus sign.

It is to be noted that the prelabeled input data forms discussed in this section are to be used in conjunction (when necessary) with the existing MAGIC System prelabeled data forms. The description for proper usage of existing forms is delineated in detail on pp. 93 - 213 of Reference 5.

The procedure used in the preparation of the additional prelabeled data forms will now be explained in detail. It is important to note that slashes (/) which appear on the prelabeled input data forms, instruct the Key punch Operator to proceed to the next entry position on the input data form, or if all entries have been punched, to the next data section.

## 2. Element Temperature Input Section (Figure II-13)

Loading which arises from elevated temperature is considered as element applied loading and is transformed into consistent energy equivalent grid point loads according to element type. For convenience to the User, temperature values (or temperature gradients) can either be input at each grid point, or as element related data.

To provide for grid point temperature input, the Grid Point Temperature labeled data form was provided in the MAGIC II System and is detailed on pp. 114-117 of Reference 5.

An additional option is provided in MAGIC III for element related temperature data. In this section, the User may employ two time saving devices:

- (1) MODAL - The MODAL option automates the specification of recurring values within a subset of input data. This feature enables data-prescribed initialization of tables. Explicit data requirements are thereby limited to the specification of exceptions to the MODAL initialization.
- (2) REPEAT - A REPEAT option is available which allows the User to retain data from a previous point for the indicated point.

The pre-labeled input data form provided for the Element Temperature Input Section is shown in Figure II-13. The first entry on the form is pre-labeled ELTEMP and requires no information from the User.

The second entry on the form is the MODAL entry which allows the User to input element temperature data which the System assumes to apply to every element unless otherwise indicated in the Element Number Entries which follow the MODAL entry. MODAL is pre-labeled in Columns 1 through 5. Columns 6 through 12 are left blank. The number of temperatures to be entered as MODAL values is entered as

# Contrails

a right justified fixed point number in Columns 13 and 14. The next sixty columns of this card (Columns 15 through 74) and the same sixty columns of the next card combine to form twelve ten column fields. Up to twelve temperatures per element may be entered as MODAL values. If six or less temperatures are entered, only one card is used for the MODAL values. The number and sequence of temperatures which are entered in these locations are functions of the type of element being employed in the analysis. This input is element related and will be explained in detail for each element in the sections which delineate the element descriptions.

The third and following entries in the section contain information pertaining to the Element Numbers, Repeat Option, Number of Temperatures, and Element Temperature Input, e.g.,

## Element Number - (Col. 7 - 11)

- (1) Element numbers are entered as fixed point numbers.
- (2) Element numbers must be entered consistent with the order in which they were entered in the Element Control Data Section.

## Repeat - (Col. 12)

The repeat option provides the User with the opportunity to repeat Temperature Input from element to element. This is accomplished in the following manner. If the Temperature Input for a number of elements is identical, the User enters the element number and associated input for the first element. For the following elements having the same input, only the Element Number (Col. 7 - 11) and an 'X' in the Repeat column need be entered. If the Repeat option is used, do not make any further entries on this card. (Be sure to leave Cols. 13 and 14 blank.)



## Number of Temperatures (Cols. 13 - 14)

The number of temperatures to be entered for the element is entered as a fixed point number in Cols. 13 and 14. This field must be left blank for subsequent entries if they are being repeated from previous entries.

## Temperatures (Cols. 15 - 72)

Up to twelve temperatures are entered in fields of ten starting in Column 15 and continuing to 74 for the first six, and again in Cols. 15 through 74 of a second card if necessary. The number of temperatures needed depends upon the element being described. This information is delineated in detail in the section on Element Descriptions.

### REMEMBER:

- (1) For a problem with identical input for every element only the MODAL entry is required.
- (2) The repeat option can be used effectively for sets of elements that have the same input. However, element numbers must be entered consistent with the order in which they were entered in the Element Control Data Section.
- (3) If the repeat option is used, leave the field for the number of temperatures blank (Cols. 13 and 14) for subsequent entries if they are being repeated from previous entries.
- (4) If six or less temperatures are entered, only one card is used for that particular element number. Do not put in an extra blank card.
- (5) The type of temperature input required for an element is a function of element type.





### 3. Element Pressure Input Section (Figure II-14)

Loading which arises from distributed pressure is considered as element applied loading and is transformed into consistent energy equivalent grid point loads according to element type. For convenience to the User, pressure values can either be input at each grid point, or as element related data.

To provide for grid point pressure input, the Grid Point Pressure labeled data form was provided in the MAGIC II System and is detailed on pp. 110-113 of Reference 5.

An additional option is provided in MAGIC III for element related pressure data. In this section, the User may employ the same two time saving devices as previously described in the Element Temperature Section, e.g., The MODAL and Repeat Options.

The pre-labeled input data form provided for the Element Pressure Input Section is shown in Figure II-14. The first entry on the form is pre-labeled ELPRESS and requires no information from the User.

The second entry on the form is the MODAL entry which allows the User to input element pressure data which the System assumes to apply to every element unless otherwise indicated in the Element Number Entries which follow the MODAL entry. MODAL is pre-labeled in Cols. 1 through 5. Columns 6 through 12 are left blank. The number of pressures to be entered as MODAL values is entered as a right justified fixed point number in Columns 13 and 14. The next sixty columns of this card (Cols. 15 through 74) and the same sixty columns of the next card combine to form twelve ten column fields. Up to twelve pressures may be entered as MODAL values. If six or less pressures are entered, only one card is used for the MODAL values. The number and sequence of pressures which are entered in these locations are functions of the type of element being employed in the analysis. This input is element related and will be explained in detail for each element in the sections which delineate the element descriptions.

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The third and following entries in the section contain information pertaining to the Element Numbers, Repeat Option, Number of Pressures, and Element Pressure Input, e.g.,

## Element Number - (Col. 7 - 11)

- (1) Element numbers are entered as fixed point numbers.
- (2) Element numbers must be entered consistent with the order in which they were entered in the Element Control Data Section.

## Repeat - (Col. 12)

The repeat option provides the User with the opportunity to repeat Pressure Input from element to element. This is accomplished in the following manner. If the Pressure Input for a number of elements is identical, the User enters the element number and associated input for the first element. For the following elements having the same input, only the Element Number (Col. 7 - 11) and an 'X' in the Repeat column need be entered. If the Repeat option is used, do not make any further entries on this card. (Be sure to leave Cols. 13 and 14 blank.)

## Number of Pressures - (Col. 13 - 14)

The number of pressures to be entered for the element is entered as a fixed point number in Cols. 13 and 14. This field must be left blank for subsequent entries if they are being repeated from previous entries.

## Pressures (Col. 15 - 74)

Up to twelve pressures are entered in fields of ten starting in Column 15 and continuing to 74 for the first six, and again in Cols. 15 through 74 of a second card if necessary. The number of pressures needed depends upon the element being described. This information is delineated in detail in the section on element description.

REMEMBER:

- (1) For a problem with identical input for every element only the MODAL entry is required.
  - (2) The repeat option can be used effectively for sets of elements that have the same input. However, element numbers must be entered consistent with the order in which they were entered in the Element Control Data section.
  - (3) If the repeat option is used, leave the field for the number of pressures blank (Cols. 13 and 14) for subsequent entries if they are being repeated from previous entries.
  - (4) If six or less pressures are entered, only one card is used for that particular element number. Do not put in an extra blank card.
  - (5) The type of pressure input required for an element is a function of element type.
4. Element Pre-Strain and Pre-Stress Input Section (Figure II-15)

A pre-labeled input data form is provided for element pre-strain and pre-stress input. This form is used for elements which accommodate pre-strain and/or pre-stress input (Figure II-15).

The first entry on the input data form is pre-labeled STST and requires no information from the user.

The second entry on the form identifies all the following information as pertaining only to strain, only to stress, or both strain and stress. Columns seven and eight are the only columns that contain information on the second card. An 'X' in Column 7 and Column 8 left blank identifies that only pre-strain data will follow. A blank in Column 7 and an 'X' in Column 8 means that only pre-stress data will follow. If both Columns 7 and 8 contain an 'X', both pre-strain and pre-stress data will follow. Note that this card must be present in an 'STST' input section, and an 'X'



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must appear in either Column 7 and/or Column 8. No default has been allowed for this card, and its omission is an error.

The third entry on the input data form is the MODAL entry. This entry allows the user to input pre-strain and/or pre-stress data (depending on what was indicated on card number two) which the System assumes to apply to every element unless otherwise indicated in the Element Number entries which follow the MODAL entry.

MODAL is pre-labeled in Cols. 1 through 5. Column 6 through 12 are left blank. The next sixty columns (Cols. 13 - 72) are divided into six ten column fields. If only pre-strain input is indicated on card two, six values of pre-strain are placed on this card. If only pre-stress input is indicated, six values of pre-stress are placed on this card. If both pre-strain and pre-stress are indicated, six values of pre-strain are placed on this card and six values of pre-stress are placed on the next card in the corresponding fields. The MODAL entry is optional and should be employed only when the User wishes to input pre-strain and/or pre-stress data for every element.

The following entries in this section contain information pertaining to Element Numbers, Repeat Option and Pre-Strain and/or Pre-Stress Input, e.g.,

## Element Number - (Cols. 7 - 11)

- (1) Element numbers are entered as fixed point numbers.
- (2) Element numbers must be entered consistent with the order in which they were entered in the Element Control Data Section.

## Repeat - (Col. 12)

The repeat option provides the User with the opportunity to repeat pre-strain and/or pre-stress input from element to element. This is accomplished in the following manner. If the input for a number of elements is identical, the User enters the element number and associated element input for the first element. For the following elements



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having the same input, only the Element Number (Col. 7 - 11) and an 'X' in the Repeat column need be entered.

## Pre-Strain or Pre-Stress Data (Col. 13 - 72)

The format of this data is analogous to that of the MODAL entry. One or two cards are used depending upon whether only pre-strain, only pre-stress, or both pre-strain and pre-stress are indicated on card number two.

The information describing the sequence of pre-strain or pre-stress data is element dependent and is presented for each of the applicable element types in the section on element description.

### REMEMBER:

- (1) For a problem with identical input for every element only the MODAL entry is required.
- (2) The repeat option can be used effectively for sets of elements that have the same input. However, element numbers must be entered consistent with the order in which they were entered in the Element Control Data Section.
- (3) The type of pre-strain and/or pre-stress input required for an element is a function of element type.





## 5. Lumped Mass and Free-Free Input Section (Figure II-16)

Lumped structural and non-structural masses are specified by component against grid point number. The axes of reference are specified with reference to the Global System.

The labeled input data format provided for the Lumped Mass Section is shown in Figure II-16. A total of nine possible mass values are provided for in this section. These are as follows:

- (1) Three Direct Inertias ( $M_x, M_y, M_z$ )
- (2) Three Rotational Inertias ( $M_{\theta x}, M_{\theta y}, M_{\theta z}$ ) and
- (3) Three Generalized Inertias ( $M_1, M_2, M_3$ ).

The total number of degrees of freedom entries per grid point is dependent on the element type being employed in the analysis. Three types appear in the MAGIC III System, i.e.,

- (1) Triangular Cross-Section Ring, Trapezoidal Cross-Section Ring (Core) - Three Degree-of-Freedom entries per point: Possible Inertia Values ( $M_x, M_y, M_z$ ).
- (2) Frame Element, Incremental Frame, Quadrilateral Shear Panel, Quadrilateral and Triangular Thin Shell Elements, Quadrilateral and Triangular Plate Elements, Symmetric Shear Web, High Aspect Ratio Quadrilateral Thin Shell, Tetrahedron, Triangular Prism, Rectangular Prism - Six Degree-of-Freedom entries per point: Possible Inertia Values ( $M_x, M_y, M_z, M_{\theta x}, M_{\theta y}, M_{\theta z}$ ).
- (3) Toroidal Thin Shell Ring - Nine Degree-of-Freedom entries per point: Possible Inertia Values ( $M_x, 0, M_z, 0, M_{\theta y}, 0, M_1, 0, M_3$ ). The  $M_1, 0$  and  $M_3$  are a set of generalized masses which correspond to non-physical derivative degrees-of-freedom for the toroidal ring. In general, these values are set equal to zero.



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The applicable concentrated masses are entered as floating point numbers. It is important to note that Key punch Personnel have been instructed to ignore entries that are not filled in. Blank entries are not considered as zeros. Zeros must be entered in an entry when applicable.

The first entry on the Lumped Mass input data form is pre-labeled MASS and requires no information from the User. The second entry is pre-labeled SCALE in Columns 1-5 and the integer 1 in Column 11. The User supplies one item of information for this entry as follows:

## Mass Scalar - (Cols. 13-22)

The Mass Scalar value is entered as a floating point number and is used when performing a free-free vibration analysis with or without condensation.

The value of the mass scalar corresponds to the value of the constant,  $a_0$ , which multiplies the assembled mass matrix. (Note the descriptions of free-free dynamics analysis (DYNAMICSF) and free-free dynamics analysis with condensation (DYNAMICSCF) which appear on pp. 82-84 and 92 of this report.)

It is noted that if a free-free analysis is not being performed, the mass scalar is not utilized. Furthermore, this input data form need only be utilized for the following:

- (1) Free-free vibration analyses with or without condensation and with or without lumped structural or non-structural masses.
- (2) Vibration analysis (rigid body modes suppressed) with or without condensation and with lumped structural or non-structural masses.

The next entry on the form is the MODAL entry. This entry allows the User to input a set of mass values which the program assumes to apply to every grid point unless otherwise indicated by a separate grid point entry on the grid point cards. MODAL is pre-labeled on this card and the only information required by the User are the lumped mass values which have been discussed previously.

The third and following entries contain information pertaining to the Grid Point Numbers, Repeat Option and Lumped Masses, as follows:

Grid Point Number - (Cols. 7-11)

- (1) Grid Point Numbers are entered as fixed point numbers.
- (2) Grid Point Numbers can be entered in any sequence desired.

Repeat - (Col. 12)

The repeat option allows the User to repeat values of lumped mass from grid point to grid point. This is accomplished in the following manner. If the lumped mass values at a number of grid points are identical, the User enters the grid point number and associated lumped mass values for the first grid point. For the following points bearing identical lumped masses only, the grid point number (Col. 7-11) and an "X" in the repeat (Col. 12) need be entered. If the repeat option is employed, only one card per grid point is required for the repeated entry irregardless of whether the degree-of-freedom entries per grid point are three, six or nine.

Remember:

- (1) The Lumped Mass input data section is utilized for the following:
  - a. Free-Free vibration analysis with or without condensation and with or without lumped structural or non-structural masses. Note that for free-free analysis a mass scalar value not equal to zero is required to properly perform the analyses as defined by the DYNAMICSF and DYNAMICSCF Agendums.
  - b. Vibration analyses with or without condensation (in which the rigid body modes have been suppressed) with lumped structural or non-structural masses. For this case the mass scalar value is set equal to 0.0 or it is not entered. If there are no lumped masses present, the form is omitted.
- (2) The Repeat option can be used effectively for sets of grid points having identical lumped masses.
- (3) Lumped masses are not element related and should not be confused with element generated mass matrices.
- (4) Zeros must be entered when applicable. Blanks are not zeros.
- (5) If the number of degree-of-freedom entries per grid point is equal to three (3) then only the inertia values ( $M_x$ ,  $M_y$ ,  $M_z$ ) are applicable. The other two entries (Rotational and Generalized Masses) are ignored by the User.
- (6) If the number of degrees-of-freedom entries per grid point is equal to six (6) then the Translational and Rotational Inertia values must be considered. If, for instance, at a certain grid point there are translational inertias but no rotational inertias,



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zeros must be entered for the rotational inertia values or this entry will be ignored by the Key punch Operator. This would cause premature termination of the run since six degree-of-freedom elements require two lumped mass cards per grid point.

- (7) If the number of degree-of-freedom entries per grid point is equal to (9), then Translational, Rotational and Generalized Masses must be entered. If some of these entries are equal to zero, these zero values must still be entered; otherwise, the entries will be ignored by the Key punch Operator causing premature termination of the run.
- (8) Repeated grid points require only one card.

## 6. Element Control Data Section (Figure II-17)

The Element Control Data Section establishes control on the types and number of elements which are to be used in a specific analysis. A pre-labeled input data form is provided for the Element Control Data Section and is shown in Figure II-17. This form is applicable to all finite elements which are contained in the MAGIC Library. Upon examination of the form, it is seen that certain data are applicable to all of the elements in the library while other data are element dependent.

The first entry on the form is pre-labeled ELEM and requires no information from the User. The second and following entries contain the following information.

### Element Number - (Cols. 7-10)

- (1) The element number which defines the element being considered is entered in this location.
- (2) Elements can be entered in any sequence desired.
- (3) The element number is entered as a fixed point number.

### Plug Number - (Cols. 11-12)

- (1) Each additional finite element in the Element Library has an identification number as follows:
  - (a) Number 52 - (Rectangular Prism)
  - (b) Number 50 - (Tetrahedron)
  - (c) Number 51 - (Triangular Prism and Symmetric Triangular Prism)
  - (d) Number 29 - (Symmetric Shear Web)
  - (e) Number 38 - (High Aspect Ratio Quadrilateral Thin Shell)
  - (f) Number 31 - (Triangular Cross-Section Ring, Asymmetric Loading)
- (2) Identification Numbers are entered as fixed point numbers.





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## Material Number - (Cols. 13-18)

The material number is the number of the material associated with the element in question. This number is referenced to the material tape. For instance, if the User were using material number 138, this material would have had to be on the tape at the time of the run or be a material that the User was adding to the tape for this particular run. The material number must appear exactly as it was in Cols. 10-15 of the MATER section.

## Temperature Interpolate Option - (Col. 19)

The Temperature Interpolate Option is exercised in the following manner:

- (1) If an entry is not made in Column 19, the program will average the node point temperatures of the element in question and use this average temperature when establishing material properties from the material tape.
- (2) If a '1' is entered in Column 19, the program will use the Material Temperature entered in Columns 20-27 when establishing material properties from the material tape.
- (3) If a number  $n$  ( $n > 1$ ) is entered in Column 19, then this number is equal to the number of node points which will participate in the averaging process. The first  $n$  node points entered in Columns 36-71 (Node Point Section), of the Element Control Data Section will then be used in the averaging process.

## Material Temperature - (Cols. 20-27)

If the User exercises the Temperature Interpolate Option by placing a '1' in Column 19, then a temperature associated with the element in question should be entered in Columns 20-27 in a thermal stress analysis. The program will then use this temperature when establishing material properties from the Material Tape.

## Repeat Element Matrices - (Col. 28)

Element matrices generated for assembly against a particular finite element specification can also be used for the next element in the calculation sequence. This avoids repeated calculation of identical element matrices. Experience indicates a high frequency of opportunities for exploiting this feature. Input data requirements and execution times can be significantly reduced with use of this feature. The option is exercised by the User by placing an 'X' in Col. 28 opposite the Element Number for which element matrices are to be repeated.

## Element Input - (Col. 29)

Certain of the additional elements contained in the MAGIC III System Element Library require element input. The rectangular prism, symmetric shear web, high aspect ratio quadrilateral thin shell, and triangular cross-section ring elements always require element input. An 'X' is placed in Column 29 for these elements.

A pre-labeled input data form is provided especially for element input. This form will be discussed in detail immediately following the discussion of the Element Control Data input form.

## Interpolated Input Print - (Col. 30)

If the User places an 'X' in Column 30, the following information is obtained:

- (1) Material Number
- (2) Material Identification
- (3) Type of Material; i.e., Isotropic or Orthotropic

- (4) Interpolated Material Properties, which include
  - (a) Temperature
  - (b) Young's Modulus
  - (c) Poisson's Ratio
  - (d) Thermal Expansion Coefficients
  - (e) Rigidity Moduli

### Element Matrix Print - (Col. 31)

If the User places an 'X' in Column 31, a print of element matrices associated with the element in question is obtained.

### Full Print (Col. 32)

If the User places an 'X' in Column 32 a total print of all element matrices and intermediate computations is obtained for the element in question. In general, this option is exercised when debugging a problem.

### Number of Input Nodes - (Cols. 33-34)

The number of input nodes is the number of node points which define an element. The following number of code points are applicable to the additional elements in the MAGIC Library.

- |                                       |               |
|---------------------------------------|---------------|
| (1) Rectangular Prism                 | 8 Node Points |
| (2) Tetrahedron                       | 4 Node Points |
| (3) Triangular Prism                  | 6 Node Points |
| (4) Symmetric Prism                   | 3 Node Points |
| (5) Symmetric Shear Web               | 2 Node Points |
| (6) High Aspect Ratio Quadrilateral   | 8 Node Points |
| (7) Triangular Ring (Asymmetric Load) | 3 Node Points |

## Pressure Suppression Option - (Col. 35)

Pressure Load Matrices are generated at the element level in the MAGIC System. The User has the option of placing an "X" in Column 35, if it is desired to suppress the generation of the pressure Load Vector for any particular element.

## Node Points - (Cols. 36-71)

These locations are reserved for the node points which describe the element in question. The User should note that three column fields are set aside for each node point. There are 12 locations set aside for node points.

## 7. Element Input Section - (Figure II-18)

A labeled input data form is provided for the Element Input Section. This form is used for elements which require Element Input: (Column 29 of the Element Control Data Section).

The first entry on the form is pre-labeled EXTERN and requires no information from the User. The second entry on the input data form is the MODAL entry which allows the User to input element input which the program assumes to apply to every element unless otherwise indicated in the Element Number entries which follow the MODAL card. It can be seen from the input data form that the Element Input is labeled A, B, C, D, E, F with each item contained in a ten column field. These are the locations where the element input is entered, if the element being used requires element input. The entries made in Locations A through F are entered as floating point numbers. The values which are entered in these locations are functions of the type of element being employed in the analysis. This input, therefore, is element related and will be explained in detail for each element in the following section.

The third and following entries in the section contain information pertaining to the Element Numbers, Repeat Option and Element Input, i.e.:

## Element Number - (Cols. 7-11)

- (1) Element Numbers are entered as fixed point numbers.
- (2) Element Numbers must be entered consistent with the order in which they were entered in the Element Control Data Section.

## Repeat - (Col. 12)

The repeat option provides the User with the opportunity to repeat Element Input from element to element. This is accomplished in the following manner. If the element input for a number of elements is identical, the User enters the element number and associated element input for the first element. For the following elements having the same element input, only the Element Number (Col. 7-11) and an 'X' in the Repeat column need be entered.

## REMEMBER:

- (1) For a problem with identical Element Input for every element only the MODAL entry is required.
- (2) The repeat option can be used effectively for sets of elements that have the same Element Input.
- (3) The type of element input required for an element is a function of element type. This element input will be completely described in the following sections.







## 8. Element Input Description

### a. Rectangular Prism (Ident. No. 52)

The rectangular prism element, Figure II-19, is a powerful tool for the analysis of solid structures, thick plates and beams. It can be used in conjunction with the triangular prism and tetrahedral discrete elements for the analysis of arbitrary solid geometries, or with plate elements for the analysis of built-up regions. The shape of the element is defined by the coordinates of the eight corner points.

Trilinear Lagrangian interpolation formulas were used as assumed displacement functions in the development of the subject element. Due to the assumption of linear interpolation formulas, the edges of the prism remain linear in deformation. A direct consequence is that, although a single element may warp under a force-couple, it may not bend under any conditions. The foregoing assumed displacement functions lead to three translational displacement degrees of freedom at each of the eight corner grid points; thus, the complete element deformation is described by twenty-four (24) displacement degrees of freedom.

The element is written to accommodate three dimensional orthotropic material. Element stresses are given at the centroid of the element and include stresses due to displacements of the element (apparent stress), stresses due to the pre-strain state within the element and stresses due to temperature within the element. Two specific cases are denoted with respect to the pre-strain and thermal stress (and associated loads) states. These are called out under "Strain Control" below and represent a constant strain (or temperature) state throughout the element and a non-constant strain (or temperature) state throughout the element.

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The following element matrices are provided for the rectangular prism in the MAGIC System:

STIFFNESS

STRESS

APPLIED LOAD (includes thermal, pressure and initial strain contributions)

APPLIED STRESS (includes thermal and initial strain contributions)

CONSISTENT MASS

Element referenced temperatures are provided by listing eight grid point temperatures on the Element Temperature Data Form (Figure II-13). The User has the option of calling out a constant temperature state or a temperature state which is of the same functional form as the assumed displacement mode shapes (i.e., trilinear Lagrangian interpolation formulas). The option is specified on the Element Input form as described below. Temperatures must be listed consistent with element numbering system.

The rectangular prism is provided with uniform pressures acting on the 6 faces of the element. The normal pressure is considered positive when acting away from the face in question (See Figure II-19). The pressures are input on the element level according to the Element Pressure Data Form (See Figure II-14). in the following manner:

Number of pressures = 6

Col. 15 - 24	is the pressure acting on face	1234
25 - 34	is the pressure acting on face	5678
35 - 44	is the pressure acting on face	1458
45 - 54	is the pressure acting on face	2367
55 - 64	is the pressure acting on face	1256
65 - 74	is the pressure acting on face	3478

# Contrails

Initial strains are input on the element level according to the Element Strain-Stress Input Data Form (See Figure II-15) in the following manner:

Col. 13 - 22 is  $\epsilon_{xx}$   
23 - 32 is  $\epsilon_{yy}$   
33 - 42 is  $\epsilon_{zz}$   
43 - 52 is  $\epsilon_{xy}$   
53 - 62 is  $\epsilon_{yz}$   
63 - 72 is  $\epsilon_{zx}$  .

The element formulation does not use the initial stress data so blank cards must be inserted.

The element control data which is required for the Rectangular Prism Element is as follows (See Figure II-17).

Element Number - Cols. 7-10

Refer to Element Control Section.

Plug Number - Cols. 11-12

The Rectangular Prism Element is identified as Number 52.

Material Number - Cols. 13-18

Refer to Element Control Section.

Temperature Interpolate Option - Col. 19

If the User exercises this option by not making an entry in Col. 19, the program will average the 8 node point temperatures when establishing material properties from the material tape. If the user wishes to employ a specific number of node points,  $n$ , in the average process ( $1 < n < 8$ ), then this number is entered in Column 19 and the first  $n$  node points entered in Cols. 36-71 will be used for the averaging process. If a "1" is entered

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in this location, the program will use the material temperature entered in Cols. 20-27 when establishing material properties from the material tape.

## Material Temperature - Cols. 20-27

Refer to Element Control Section.

## Repeat Element Matrices - Col. 28

Refer to Element Control Section.

## Element Input - Col. 29

The rectangular prism element always requires Element Input; therefore, an 'X' is always placed in Column 29 when a rectangular prism element is being used. The Element Input (Figure II-18) required for the Rectangular Prism consists of the following information:

### Location A - Cols. 13-22

Strain Control, SC

if SC = 0.0, the element is under a constant strain (temperature).

if SC = 1.0, the element is not at a constant strain (temperature).

Returning to the Element Control Data Section, the list of data items continues as follows:

Interpolated Input Print - Col. 30

Refer to Element Control Section.

Element Matrix Print - Col. 31

Refer to Element Control Section

Full Print - Col. 32

Refer to Element Control Section.

Number of Input Nodes - Cols. 33-34

The Rectangular Prism Element is always defined by eight input nodes.

Pressure Suppression Option - Col. 35

Refer to Element Control Section.

Node Points (Cols. 36-71)

The Rectangular Prism Element is defined by 8 grid points.

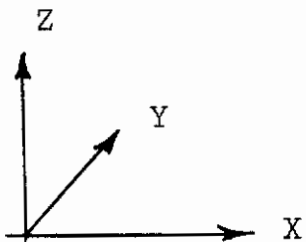
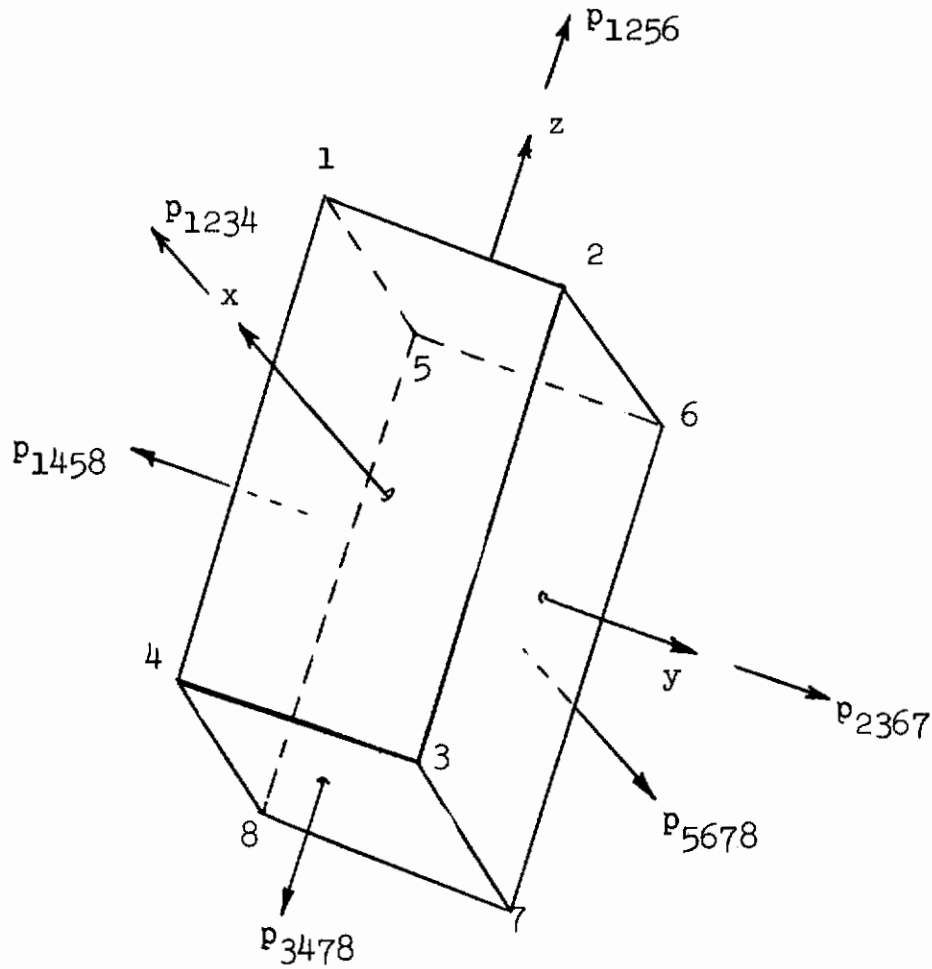


Figure II-19 - Rectangular Prism Element

b. Tetrahedron (Ident. No. 50)

The tetrahedron discrete element, Figure II-20 , can be used to analyze solid structures such as beams and plates. It can also be used in conjunction with the rectangular prism and triangular prism solid elements and in fact is used to generate the triangular prism element. The shape of the element is defined by the coordinates of the four corner points.

A linear polynomial is used for each of the three displacement modes. These mode shapes lead to a total of twelve (12) undetermined coefficients for the element which are chosen to correspond to three translational displacement degrees of freedom at each of the four vertices of the element. The nature of the assumed displacement modes is such that the strains throughout the element are constant.

The element is written to accommodate three dimensional orthotropic material. Element stresses include stresses due to displacement (apparent stress), stresses due to the prestrain state within the element and stresses due to temperature within the element.

The following element matrices are provided for the tetrahedron in the MAGIC System:

STIFFNESS

STRESS

APPLIED LOAD (includes thermal, pressure and initial strain contributions)

APPLIED STRESS (includes thermal and initial strain contributions)

CONSISTENT MASS

Element referenced temperatures are provided by listing four grid point temperatures on the Element Temperature Data Form (Figure II-13). These temperatures are then averaged in the MAGIC System to provide a weighted element input temperature. Temperatures must be listed consistent with element numbering system.



# Contrails

The tetrahedron is provided with uniform pressures acting on the 4 faces of the element. The normal pressure is considered positive when acting away from the face in question (see Figure II-20). The pressures are input on the element level according to the Element Pressure Data form (See Figure II-14) in the following manner:

Number of pressures = 4

Col. 15-24 is the pressure acting on face 134  
25-34 is the pressure acting on face 234  
35-44 is the pressure acting on face 124  
45-54 is the pressure acting on face 123

Initial strains are input on the element level according to the Element Strain-Stress Input Data Form (see Figure II-15) in the following manner:

Col. 13-22 is  $\epsilon_{xx}$   
23-32 is  $\epsilon_{yy}$   
33-42 is  $\epsilon_{zz}$   
43-52 is  $\epsilon_{xy}$   
53-62 is  $\epsilon_{yz}$   
63-72 is  $\epsilon_{zx}$

The element formulation does not use the initial stress data so blank cards must be inserted.

The element Control Data which is required for the Tetrahedron Element is as follows (see Figure II-17).

Element Number - Cols. 7-10

Refer to Element Control Section.

Plug Number - Cols. 11-12

The Tetrahedron Element is identified as Number 50.

Material Number - Cols. 13-18

Refer to Element Control Section.

# Contrails

## Temperature Interpolate Option (Col. 19)

If the user exercises this option by not making an entry in Column 19, the program will average the 4 node point temperatures when establishing material properties from the material tape. If the user wishes to employ a specific number of node points,  $n$ , in the average process ( $1 < n < 4$ ), then this number is entered in Column 19 and the first  $n$  node points entered in Columns 36-71 will be used for the averaging process. If a "1" is entered in this location, the program will use the Material Temperature entered in Columns 20-27 when establishing material properties from the material tape.

## Material Temperature - Cols. 20-27

Refer to Element Control Section.

## Repeat Element Matrices - Col. 28

Refer to Element Control Section.

## Element Input - Col. 29

The tetrahedron element requires no element input.

## Interpolated Input Print - Col. 30

Refer to Element Control Section.

## Element Matrix Print - Col. 31

Refer to Element Control Section.

## Full Print - Col. 32

Refer to Element Control Section.

## Number of Input Nodes - Cols. 33-34

The tetrahedron element is always defined by 4 input nodes.

## Pressure Suppression Option - Col. 35

Refer to Element Control Section.

## Node Points - Col. 36-71

The tetrahedron element is defined by 4 grid points.

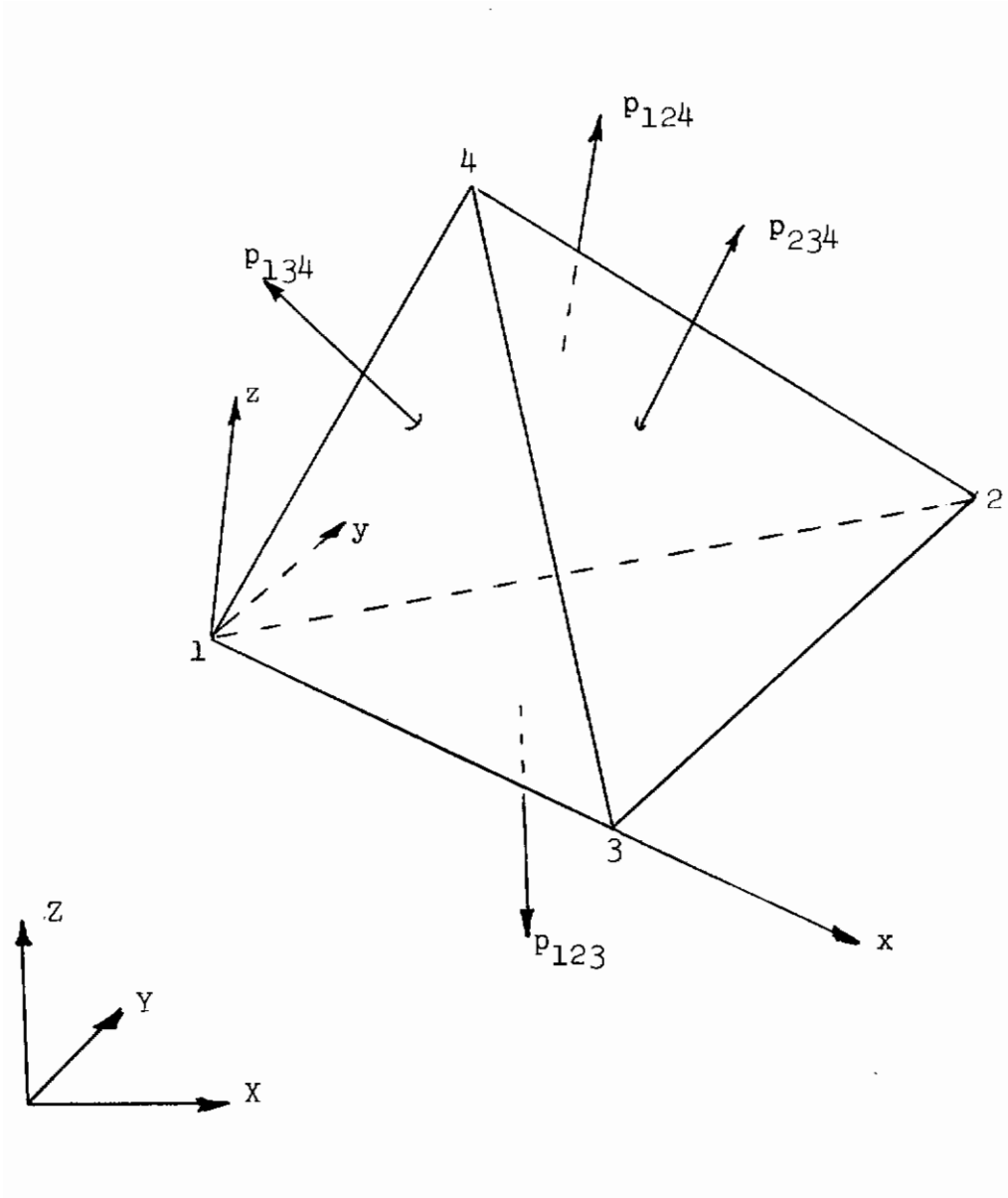


FIGURE II-20 - TETRAHEDRON ELEMENT

c. Triangular Prism (Ident. No. 51)

Three tetrahedrons are assembled as shown in Figure II-21 to form a triangular prism. Using this approach element matrices for three tetrahedrons are computed and assembled automatically within the MAGIC III System. A considerable reduction in input is realized which leads to a corresponding reduction in the possibility of input error when large scale analyses are performed. The input for one triangular prism element is identical to that for one tetrahedron except that six grid points define the prism instead of four which define the tetrahedron.

Element stresses are output for each tetrahedron which comprise the triangular prism. These include stresses due to displacement (apparent stress), stresses due to the prestrain state within the element and stresses due to temperature within the element.

The symmetric triangular prism finite element shown in Figure II-22 is a special case of the full, triangular prism element. This element was developed to eliminate conditioning problems inherent in the analysis of thin symmetric sections. As an example, in the analysis of aircraft wing or tail sections, the element can be used very effectively to model full-depth honeycomb core constructions which are used for shear transfer between the top and bottom skins. The use of this element allows the analysis to be performed using either the top or bottom symmetric half of the structure.

Appropriate boundary conditions are applied at the element level which specialize the full-depth prism into the symmetric element. The procedure employed in the reduction is as follows. Six tetrahedron elements are automatically assembled within the program with the three on the lower side of the axis of

symmetry being the mirror images of the corresponding three tetrahedrons on the upper side. This approach assures that symmetric and antisymmetric modes will uncouple when the element is specialized to a symmetric representation. Appropriate symmetric and antisymmetric boundary conditions are imposed on the centerline of symmetry at the element level. Based on these conditions, the degrees of freedom associated with the bottom symmetric half of the structure are expressed in terms of the remaining degrees of freedom. Thus, a transformation between deformations on the full prism and symmetric prism is derived which is used in a simple fashion to generate the desired matrices.

The following element matrices are provided for the triangular prism in the MAGIC system.

STIFFNESS

STRESS

APPLIED LOAD (includes thermal, pressure and initial strain contributions)

APPLIED STRESS (includes thermal and initial strain contributions)

CONSISTENT MASS

Element referenced temperatures are provided by listing six grid point temperatures on the Element Temperature Data Form (Figure II-13). Temperatures must be listed consistent with element numbering system.

The triangular prism is provided with uniform pressures acting on the 5 faces of the element. The normal pressure is considered positive when acting away from the face in question (see Figure II-21). The pressures are input on the element level according to the Element Pressure Data Form (See Figure II-14) in the following manner:

# Contrails

Number of pressures = 5

Col. 15-24 is the pressure acting on face 123  
25-34 is the pressure acting on face 456  
35-44 is the pressure acting on face 2365  
45-54 is the pressure acting on face 1364  
55-64 is the pressure acting on face 2541

Initial strains are input on the element level according to the Element Strain-Stress Input Data Form (See Figure II-15 ) in the following manner:

Col. 13-22 is  $\epsilon_{xx}$   
23-32 is  $\epsilon_{yy}$   
33-42 is  $\epsilon_{zz}$   
43-52 is  $\epsilon_{xy}$   
53-62 is  $\epsilon_{yz}$   
63-72 is  $\epsilon_{zx}$

The element formulation does not use the initial stress data so blank cards must be inserted.

The element Control Data which is required for the Triangular Prism Element is as follows (See Figure II-17)

Element Number - Cols. 7-10

Refer to Element Control Section.

Plug Number - Cols. 11-12

The triangular prism element is identified as number 51.

Material Number - Cols. 13-18

Refer to Element Control Section.

Temperature Interpolate Option - Col. 19

If the user exercises this option by not making an entry in Col. 19, the program will average the 6 node point temperatures when establishing material properties from the material tape. If the user wishes to employ a specific number of node points, n, in the average process

( $1 < n < 6$ ), then this number is entered in Col. 19 and the first  $n$  node points entered in Cols. 36-71 will be used for the averaging process. If a "1" is entered in this location, the program will use the material temperature entered in Cols. 20-27 when establishing material properties from the material tape.

Material Temperature - Cols. 20-27

Refer to Element Control Section.

Repeat Element Matrices - Col. 28

Refer to Element Control Section.

Element Input - Col. 29

The triangular prism element requires no element input.

Interpolated Input Print - Col. 30

Refer to Element Control Section.

Element Matrix Print - Col. 31

Refer to Element Control Section.

Full Print - Col. 32

Refer to Element Control Section.

Number of Input Nodes - Cols. 33-34

The triangular and symmetric triangular prism elements are always defined by 6 input nodes.

Pressure Suppression Option - Col. 35

Refer to Element Control Section.

Node Points - Cols. 36-71

The triangular prism element is defined by 6 grid points.

If node points 4, 5, and 6 do not exist (that is, are not input), the element then becomes a symmetrical triangular prism with the plane of symmetry being midway between node points 1, 2, 3 and node points 4, 5 and 6 (namely the XY plane of the structure - See Figure II-22 ).



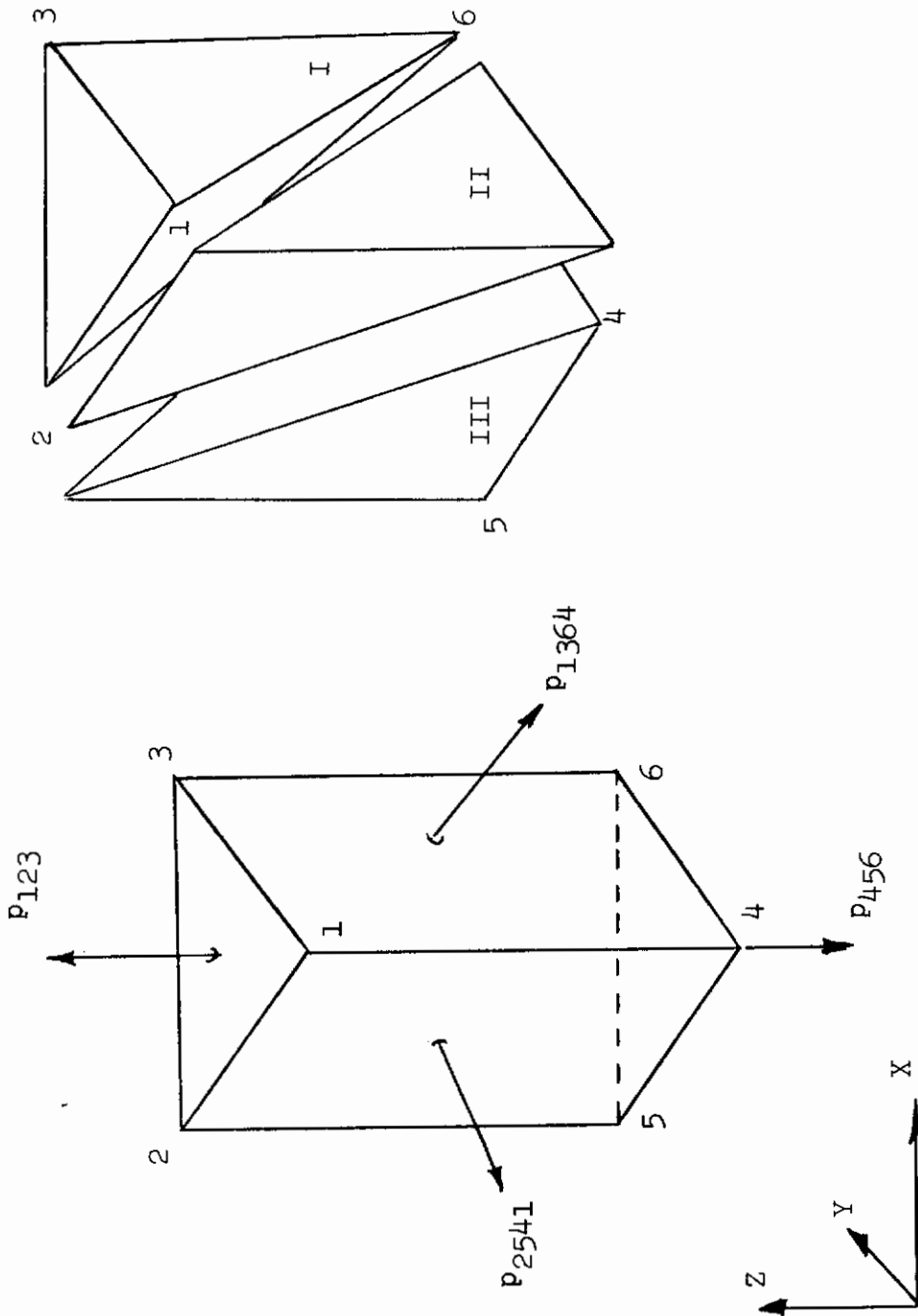


FIGURE II-21 TRIANGULAR PRISM ELEMENT

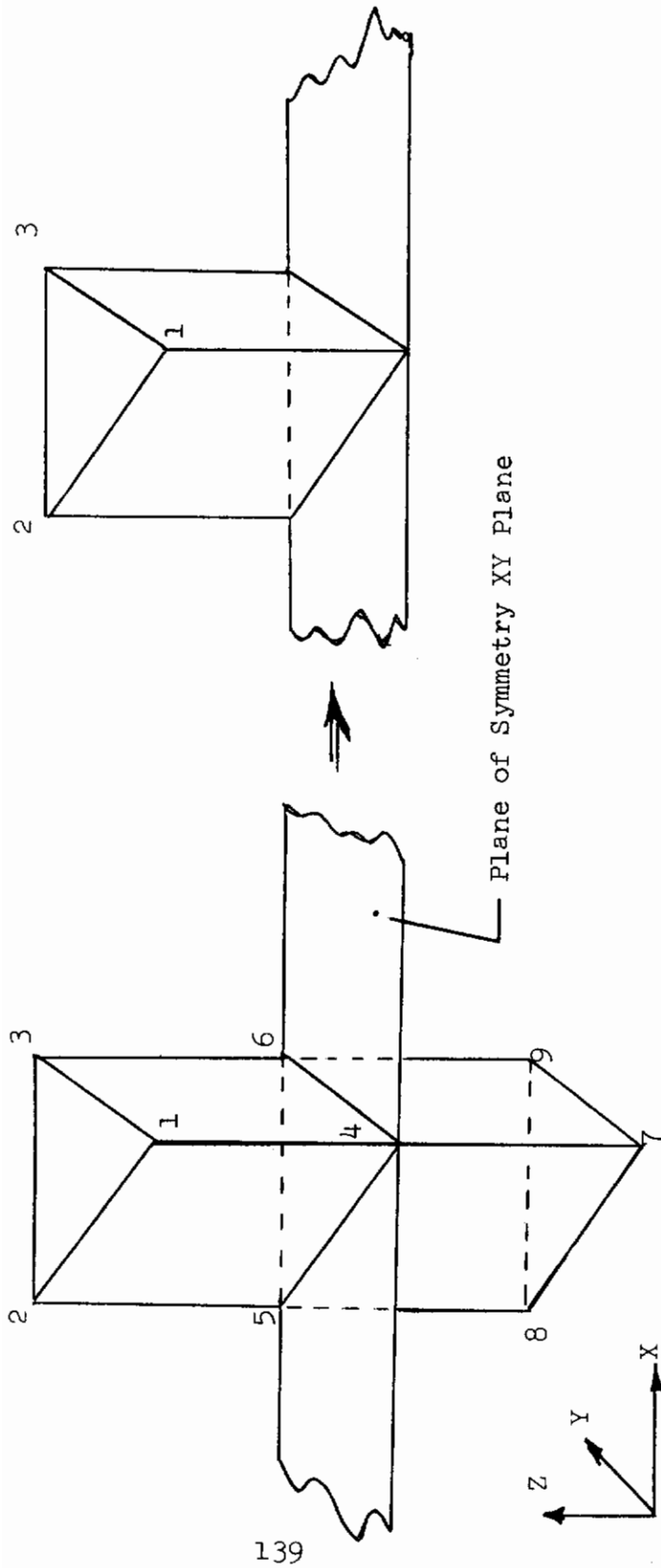


FIGURE II-22 SYMMETRIC TRIANGULAR PRISM ELEMENT

## d. Symmetrical Shear Web (Ident. No. 29)

The symmetric shear web element as shown in Figure II-23 was developed to conduct analyses of the type discussed in the previous section, Section C.8.b, Triangular Prism. Appropriate symmetric and antisymmetric boundary conditions are imposed on the centerline of symmetry at the element level. Based on these conditions, element matrices can be readily derived using only the two upper grid points as reference points.

The assumed displacement method is utilized to derive the stiffness and stress matrices. These displacement functions in the local coordinate system are:

$$u(x, z) = (a_1 + a_2 x)z$$
$$w(x) = b_1 + b_2 x + b_3 x^2 + b_4 x^3$$

These functions yield six translational deformations, three translations at each of two grid points. Element stresses are evaluated at the midpoint of the element's length and yield the shearing stress at that point.

The following element matrices are provided for the symmetric shear web in the MAGIC System:

STIFFNESS

STRESS

The element Control Data which is required for the Symmetrical Shear Web is as follows (See Figure II-17):

Element Number - Cols. 7-10

Refer to Element Control Section.

Plug Number - Cols. 11-12

The Symmetrical shear web element is identified as Number 29.

Material Number - Cols. 13-18

Refer to Element Control Section.

## Temperature Interpolate Option - Col. 19

If the user exercises this option by not making an entry in Col. 19, the program will average the 2 node point temperatures when establishing material properties from the material tape. If the user wishes to employ a specific number of node points,  $n$ , in the average process ( $1 \leq n \leq 2$ ), then this number is entered in Col. 19 and the first  $n$  node points entered in Cols. 36-71 will be used for the averaging process. If a "1" is entered in this location, the program will use the Material Temperature entered in Cols. 20-27 when establishing material properties from the material tape.

## Material Temperature - Cols. 20-27

Refer to Element Control Section.

## Repeat Element Matrices - Col. 28

Refer to Element Control Section.

## Element Input - Col. 29

The symmetrical shear web element always requires Element Input. Therefore, an 'X' is always placed in Col. 29 when the symmetrical shear web is being employed.

The Element Input (Figure II-18) required for the symmetrical shear web consists of the following information:

### Location A - Cols. 13-22

THICKNESS, (t)

The above is the only Element Input which is required for the shear web.

Returning to the Element Control Data Section, the list of data items continues as follows:

# Contrails

## Interpolated Input Print - Col. 30

Refer to Element Control Section.

## Element Matrix Print - Col. 31

Refer to Element Control Section

## Full Print - Col. 32

Refer to Element Control Section.

## Number of Input Nodes - Cols. 33-34

The symmetrical shear web element is always defined by 2 input nodes.

## Pressure Suppression Option - Col. 35

Refer to Element Control Section.

## Node Points - Cols. 36-71

The symmetrical shear web element is defined by 2 grid points.

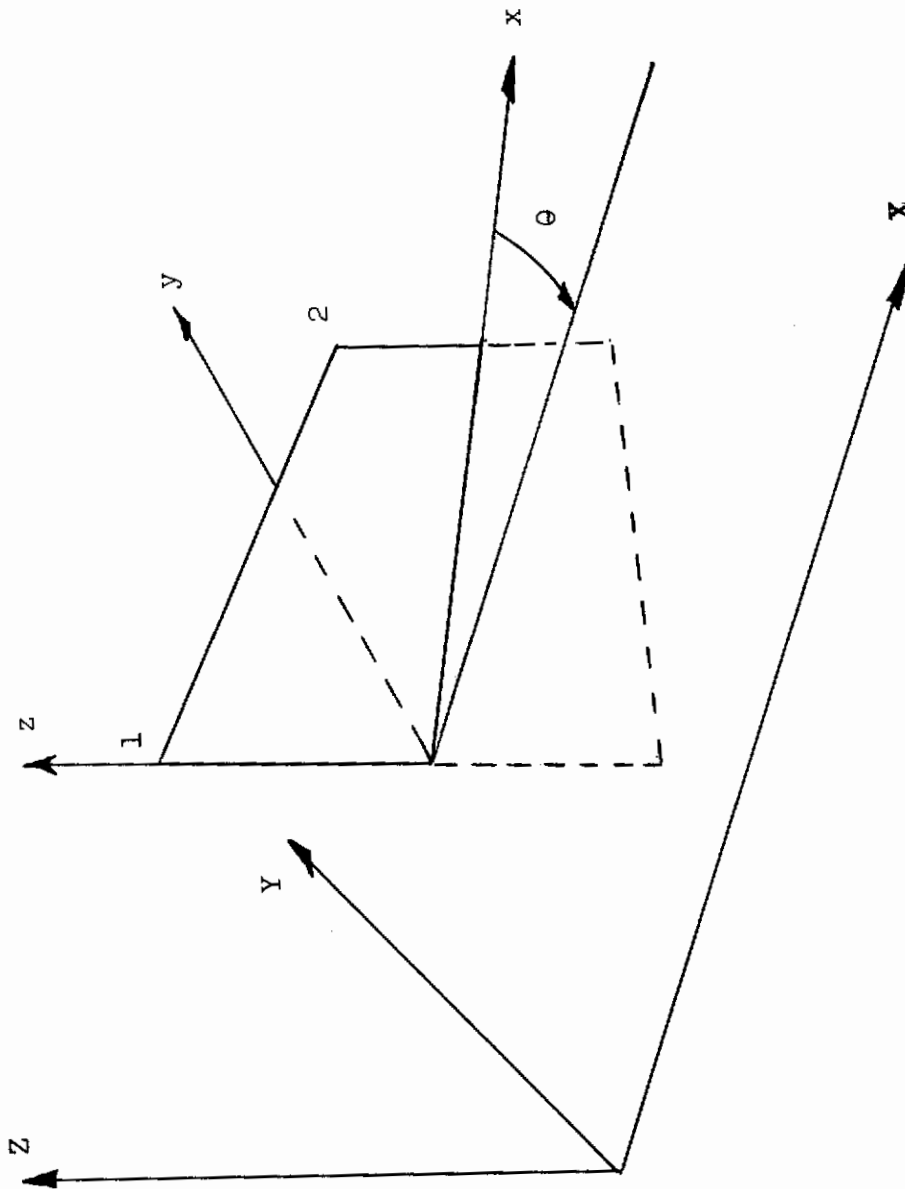


FIGURE II-23 - SYMMETRIC SHEAR WEB ELEMENT

e. High Aspect Ratio Quadrilateral Thin Shell (Ident. No. 38)

This finite element differs from the present MAGIC II quadrilateral thin shell element (Ident. No. 21) only in the approximation of in-plane behavior. No difference other than the identification number is evident to the User.

This additional finite element representation is included in the MAGIC III System for use in the idealization of membranes and plane-strain sections that require elongated finite element shapes. This circumstance is frequently encountered. One important class of applications requiring high aspect ratio finite elements is the stress analysis of structural joints. A rule of thumb that may be applied to guide the choice of element type for such applications is to use the modified quadrilateral thin shell element for those elements whose aspect ratio exceeds six.

All element matrices available to Element Ident. No. 21 are available to this element as well, i.e., stiffness, stress, distributed loading, thermal loading and consistent mass.

All input data required for this element is identical to that required for the original Quadrilateral Thin Shell (Ident. No. 21). Therefore, in the interest of conciseness, the reader is referred to Pages 175 thru 184 of Reference 5 for detailed element input description.

An example application utilizing this finite element is presented in Section II - C.8.e of this report.



f. Triangular Ring (Asymmetrical Load) - (Ident. No. 31)

The triangular ring (asymmetrical loading), hereafter called the asymmetric triangular ring, is a new tool which can be used for the analysis of thick-walled and solid axisymmetric structures of finite length. It may be used to idealize any axisymmetric structure taking into account

- 1) arbitrary axial variations in geometry,
- 2) axial variation in orientation of material axes of orthotropy,
- 3) radial and axial variations in material properties,
- 4) any asymmetric loading system including distributed mechanical and thermal loads.

The asymmetric triangular ring element and its accompanying applied mechanical loadings are pictured in Figure II-24. These mechanical loads are assumed evenly distributed over the loaded face, possessed of circumferential variation of magnitude and acting (or directed) parallel to the axial and radial direction of the ring (see Figure II-24). Positive directions of loading are illustrated in this figure. The complete theoretical development of this element is presented in the Engineer's Manual. A brief review of this development is given below.

The load and displacement fields for the asymmetric triangular ring element are assumed expressed in a Fourier series form in terms of the circumferential coordinate  $\theta$ . Utilizing these expressions to write the total potential energy, the energy (and consequently the analysis) can be shown to decompose into an uncoupled form. Thus the three dimensional problem represented by an asymmetrically loaded solid of revolution can be solved by the carrying out of a sequence of two dimensional analyses. The resulting economy and accuracy introduced is obvious.

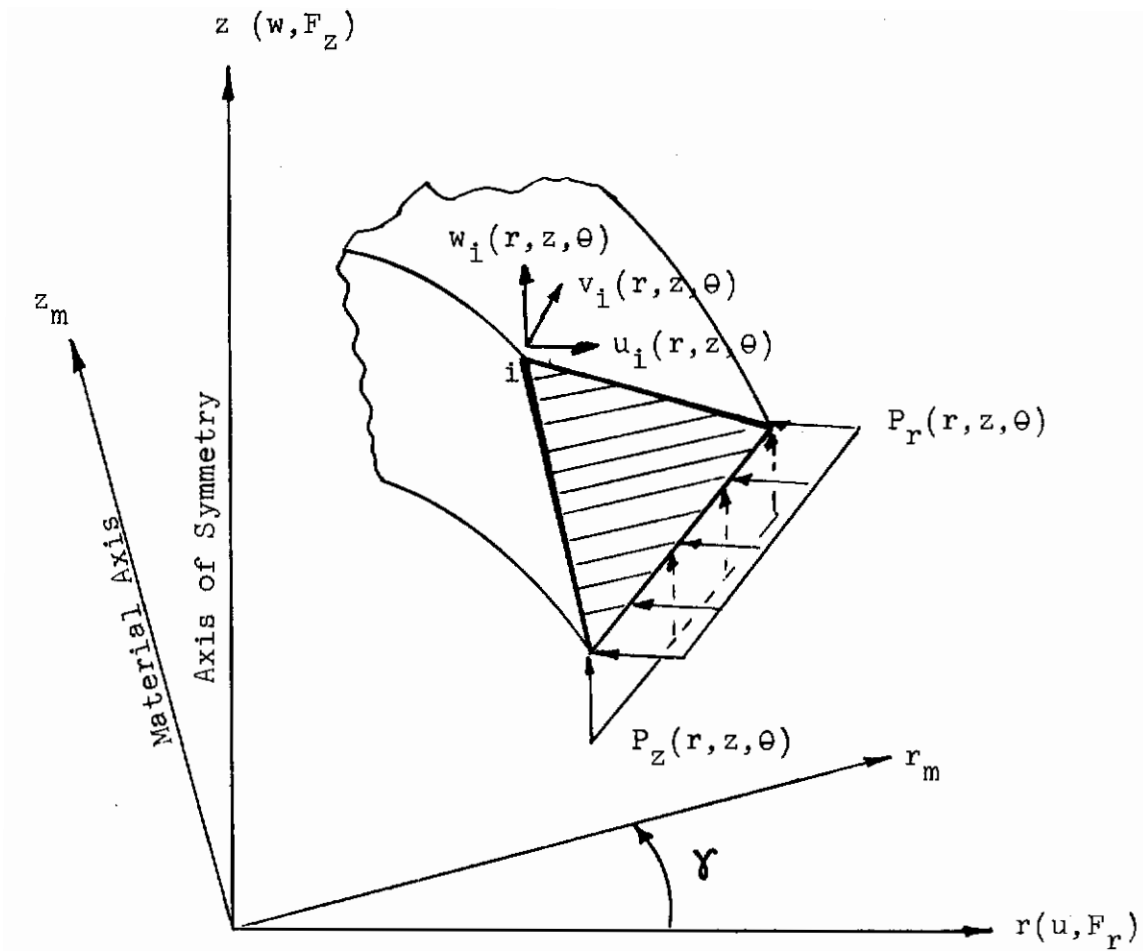


Figure II-24 - Triangular Ring Element (Asymmetric Loading)

The steps inherent in the analysis procedure can be listed as follows:

- 1) Utilizing input values of the applied load at regular circumferential stations around the structure, the Magic III program automatically generates a Fourier series representation of the loading system,

$$\{P\} = \{P_0\} + \sum_{n=1}^m \{P_n\} [C_n] + \sum_{n=1}^m \{\bar{P}_n\} [\bar{C}_n] \quad (1)$$

where  $\{P_0\}$ ,  $\{P_n\}$  and  $\{\bar{P}_n\}$  can be interpreted as harmonic load vectors and the diagonal matrices  $[C_n]$   $[\bar{C}_n]$  are composed of appropriate combinations of trigonometric elements  $\cos n\theta$  and  $\sin n\theta$ .

- 2) The User specifies a maximum number of harmonics ( $m$ ) to be considered in the analysis. In response to this definition, the Magic III program automatically selects the ( $m$ ) most significant harmonics. The harmonics selected by the program are a function of the applied loading system.
- 3) ( $m$ ) individual two dimensional analyses are then carried out. Harmonic displacements and stresses are obtained and combined to obtain the gross stresses and displacements of the structure.

An example of this analysis procedure is given below (with Reference to the theoretical development in the Engineer's Manual, Reference 9.)

Assume that a limit on the harmonic analyses has been set at three ( $m=3$ ) and that the most significant description of the load system has been selected by the Magic III program as

$$\{P\} = \{P_0\} + \{P_1\} [C_1] + \{P_2\} [C_2] \quad (2)$$

The following individual harmonic analyses are then carried out

$$\begin{aligned}
 \{P_0\} &= [K_0] \{\delta_0\} \\
 \{P_1\} &= [K_1] \{\delta_1\} \\
 \{\bar{P}_2\} &= [\bar{K}_2] \{\bar{\delta}_2\}
 \end{aligned}
 \tag{3}$$

to determine the harmonic displacements  $\{\delta_0\}$ ,  $\{\delta_1\}$  and  $\{\bar{\delta}_2\}$  for the structure. The gross (actual) displacements of the structure  $\{\delta\}$  can now be determined by combining  $\{\delta_0\}$ ,  $\{\delta_1\}$  and  $\{\bar{\delta}_2\}$  in the following series.

$$\{\delta\} = \{\delta_0\} + [C_1] \{\delta_1\} + [\bar{C}_2] \{\bar{\delta}_2\}.
 \tag{4}$$

Harmonic  $\{\sigma_m\}$  and actual  $\{\sigma\}$  stresses can be obtained in a similar manner.

The ring element geometry is defined with respect to cylindrical coordinate axes. The configuration of the element, as pictured in Figure II-24, is completely defined by specifying the radial and axial coordinates of the corner points.

The orthotropy (cylindrical anisotropy) is provided for in the mechanical and physical material properties of the ring element. The orientation of this orthotropy is assumed oriented in the  $\gamma_m$ ,  $z_m$  and  $\theta$  directions (see Figure II-24). Transformation to the geometrical or structural system is accomplished utilizing the material angle  $\gamma$ .

The development of the asymmetric triangular ring element is an expansion of that utilized in deriving the axisymmetric triangular ring element (Ident. No. 40). Similar linear polynomial functions are employed in both elements and are employed for displacement mode shapes leading to constant element strain and stress states.

# Contrails

Due to the asymmetric deformations which the asymmetric ring can accommodate, 9 degrees of freedom (as opposed to six for the axisymmetric ring) are required to define the deformational behavior of this element. The predicted element stress behavior is constant over the triangular cross-section. Radial, circumferential and axial stresses are predicted. As in the axisymmetric ring element (Ident. No. 40) the asymmetric ring is numbered in the following manner. The element is numbered in the counter-clockwise direction.

A major difference between the two elements (asymmetric and axisymmetric ring), other than the accommodation of asymmetric loads in the former, is the interpretation of the applied loads themselves. Loads applied to the axisymmetric ring (Ident. No. 40) are assumed applied at grid points while loads applied to the asymmetric ring (Ident. No. 31) are assumed applied to the element.

In order to account for this difference as well as the circumferential variation of the magnitude of the loads an alternate set of load and data input cards must be provided to accommodate the asymmetric ring. These are provided and discussed in the discussion which follows. For solids of revolution subjected to axisymmetric loadings, it is suggested that the axisymmetric element be used.

The Element Control Data which is required for the Asymmetric Triangular Ring Element is as follows: (see Figure II-17)

Element Number - (Cols. 7-10)

Refer to Element Control Section

Plug Number - (Cols. 11-12)

The Triangular Cross-section Ring Element is identified as Number 31.

# Contrails

## Material Number - (Cols. 13-18)

Refer to Element Control Section

## Temperature Interpolate Option - (Col. 19)

Not available for this element.

## Material Temperature - (Cols. 20-27)

Refer to Element Control Section.

## Repeat Element Matrices - (Col. 28)

Refer to Element Control Section.

## Element Input - (Col. 29)

To utilize this option, place an X in Col. 29.

Note: The Asymmetric Triangular Cross-Section Ring Element only requires Element Input under certain special conditions as follows: Referring to Figure II-24, it is seen that there is a possibility that in some cases the material axis, and element geometric axis of the element will not coincide. If this is the case the Element Input (Figure II-18) required for the Triangular Cross-Section Ring consists of the following:

### Location A - (Cols. 13-22)

Material Axes Angle (Gamma -  $\gamma_{mg}$ )

Since the Triangular Cross-Section Ring Element is written to accommodate anisotropy of mechanical and physical properties, provision is made in the program for differences in orientation of material and element geometric axes for an element. The User inputs the angle between the element material axis ( $X_m$ ) and the element geometric axis ( $X_g$ ).

The angle gamma ( $\gamma_{mg}$ ) is input in degrees and is



# Contrails

considered positive when measured from the material axes to the element geometric axes, in a counter-clock-wise direction (Figure II-24).

## Remember

Element Input is not required for the Triangular Ring if the material and geometric axes coincide,

i.e.,  $\gamma_{mg} = 0$ .

Interpolated Input Print - (Col. 30)

Element Matrix Print - (Col. 31)

Full Print (Col. 32)

} Refer to Element  
Control Section

Number of Input Nodes (Col. 33-34)

The Asymmetric Triangular Cross-Section Ring Element is always defined by 3 input nodes.

Pressure Suppression Option (Col. 35)

Not available for this element.

Node Points - (Cols. 36-71)

The three node points which define each Triangular Ring are entered in the first three entries provided in the Node Point Section of the Element Control Data Form.

As previously mentioned an alternate set of load and data input cards are provided in the MAGIC III system to accommodate this particular element. These input cards replace the element pressure and temperature data cards shown in Figures II-14 and II-13 and are explained in detail below.



## Stress and Displacement Output Section

The first entry on the input data form Figure II-25 is a pre-labeled HSDC and requires no other information from the User.. The second entry contains the reference, incremental and final circumferential angular values at which output stress and displacement data is desired. These entries are described below:

### Reference Value Col. (7-11)

The entry in these columns is a fixed point right adjusted number representing the reference angle in degrees. The entry must not be less than zero nor greater than 359°.

### Increment Value Col. (12-16)

The entry in these columns is a fixed point right adjusted number representing the increment value in degrees. The entry must not be less than 1° nor greater than 360°.

### Final Circumferential Value Col. (17-21)

The entry in these columns is a fixed point right adjusted number representing the final circumferential value in degrees. This entry must not be greater than 360°.

Defining

RV = Reference Value

IV = Increment Value,

and FV = Final Value.

The following inequalities must hold

$$IV < FV - RV$$

$$0 < FV - RV .$$

The values defined above are utilized to define the region and quantity of information (output) desired for a given structure.

**MAGIC STRUCTURAL ANALYSIS SYSTEM  
INPUT DATA FORMAT**

TRIANGULAR RING ELEMENT (ASYMMETRIC LOADING)  
HARMONIC INCREMENTS

H	S	D	C		
1	2	3	4	5	6

( / )

REF. VALUE	INC. VALUE	FINC. VALUE
1		
7	2	2
8	3	7
9	4	8
0	5	9
1	6	0
		1

( / )

Figure II-25 - Harmonic Stress and Displacement Output Control

## Harmonic Pressure Loading Section

A pre-labeled input data form entitled HARM is provided for the entry of pressure load data and is shown in Figure II-26. The first entry on the form is labeled HARM and requires no other information from the User. The second entry pertains to the number of loaded elements, the maximum number of harmonics to be used per element, and the maximum number of output harmonics for the system. The third set of input data is concerned with element number, an element loading repeat option, the number of loading points and the harmonic pressure values. The last two sets of data must be input by the User and the instructions for doing so are described below. Entries on the second input data card, Figure II-26 are:

### Number of Loaded Elements (Cols. 7-9)

The entry in three columns is a fixed point right adjusted number which represents the elements which have imposed pressure loads. Only the quantity of such elements is entered.

### Number of Harmonics per Element (Col. 10)

The maximum number of harmonics to be used to represent the pressure loading for each element is entered as a fixed point number in column 10. This entry must be greater than zero and less than nine in value.

### Number of Harmonics Output (Col. 11)

The maximum number of harmonics to be used in the calculation of output data for the entire element structure is entered as a fixed point number in column 11. This entry must be less than or equal to the number of harmonics per element.

Entries on the third and following input data cards Figure II-26, is described below:

## Element Number (Cols. 7-9)

The number of the element on which the pressure load is to be applied is entered in Cols 7-9 using a fixed point right adjusted format. The element numbers are to be entered in ascending order and a maximum of 500 elements may be entered loaded.

## Element Loading Repeat Option (Col. 10)

This piece of input data determines whether or not the element pressure loadings are to be repeated for succeeding elements. If these loadings are to be repeated the User enters an "X" in column 10 and omits remaining load data pertaining to this element. The pressure data from the proceeding element is automatically applied to this element. If these loadings are not to be repeated the User leaves column 10 blank.

## Number of Radial Loading Points (Cols. 11-13)

The entry in these columns is a fixed point right adjusted number representing the number of points at which radial pressures will be defined. These points are spaced at equal intervals about the circumference of the element. The value of this entry must be greater than zero and less than 60 if a radial pressure is to be applied. If no radial pressure is present, a zero is entered and radial pressure values are omitted.

## Number of Axial Loading Points (Cols. 14-16)

The entry in these columns is a fixed point right adjusted number representing the number of points at which axial pressures will be defined. These points are spaced at equal intervals about the circumference of the element. The value of this entry must be greater than zero and less than 60 if an axial load is to be applied on this element. Note that this entry does not have to be the same as the previous entry. If no axial pressure is present a zero is entered and axial pressure values are omitted.

## Pressure Loading Values (Cols. 17-76)

The User enters pressure load values which are equal in quantity to the sum of the number of radial and axial loading points (Cols. 11-16). The radial values are entered first followed by the axial values. These values are entered in columns 17-76 in a floating point right adjusted format six to a card as shown in Figure II-26. A maximum of 20 such cards are allowed permitting a maximum entry of 120 pressure values per element. Note that the 2nd to 20th cards do not contain entries in columns 7 to 16. Pressures are applied on face number one (between nodes 1 and 2) and have same sense as the global coordinate system.

## Harmonic Thermal Loading Section

A pre-labeled input data form entitled HTEM is provided for the entry of thermal load data and is shown in Figure II-27. The first entry on the form is labeled HTEM, and requires no other information from the User. The second entry pertains to the number of loaded elements, the maximum number of harmonics to be used per element, and the maximum number of output harmonics for the system. The third set of input data is concerned with element number, an element loading repeat option, the number of temperature loading points and the harmonic thermal values. The last two sets of data must be input by the User and the instructions for doing so are described below. Entries on the second input data card, Figure II-27 are:

## Number of Loaded Elements (Cols. 7-9)

The entry in these columns is a fixed point right adjusted number which represents the elements which have imposed thermal loads. Only the quantity of such elements is entered.







## Number of Harmonics per Element (Col. 10)

The maximum number of harmonics to be used to represent the thermal loading for each element is entered as a fixed point number in column 10. This entry must be greater than zero and less than nine in value.

## Number of Harmonics Output (Col. 11)

The maximum number of harmonics to be used in the calculation of output data for the entire structure is entered as a fixed point number in column 11. This entry must be less than or equal to the number of harmonics per element.

Entries on the third and following input data cards, Figure II-27, is described below:

## Element Number (Cols. 7-9)

The number of the element on which the thermal load is to be applied is entered in Cols. 7-9 using a fixed point right adjusted format. The element numbers are to be entered in ascending order and a maximum of 500 elements may be entered (loaded).

## Element Loading Repeat Option (Cols. 10)

This piece of input data determines whether or not the element thermal loadings are to be repeated for succeeding elements. If these loadings are to be repeated the User enters an "X" in column 10 and omits remaining load data pertaining to this element. The thermal data from the proceeding element is automatically applied to this element. If these loadings are not to be repeated the User leaves column 10 blank.





## Number of Thermal Loading Points (Cols. 11-13)

The entry in these columns is a fixed point right adjusted number representing the number of points at which thermal loads will be defined. These points are spaced at equal intervals about the circumference of the element. The value of this entry must be greater than zero and less than 60.

## Thermal Loading Values (Cols. 17-76)

The User enters thermal load values which are equal in quantity to the number of thermal loading points (Cols. 11-13). These values are entered in Columns 17-76 in a floating point right adjusted format six to a card as shown in Figure II-27. A maximum of 10 such cards are allowed permitting a maximum entry of 60 pressure values per element. Note that the 2nd to 10th cards do not contain entries in Columns 7 to 13. Temperatures which are input are assumed applied to the element as a whole and must be interpreted as temperature changes (either increase (+) or decrease (-) from a thermal stress free state) to which the element is subjected.

## SECTION III

### INPUT AND OUTPUT OF MAGIC III SYSTEM

#### A. GENERAL DESCRIPTION

In this section, the proper interpretation of the input supplied to the MAGIC III system and the output supplied by the MAGIC III system is provided by reference to specific example problems. These examples will use the finite elements added to the MAGIC system; namely,

- 1) Rectangular Prism
- 2) Tetrahedron
- 3) Triangular Prism
- 4) Symmetric Triangular Prism
- 5) Symmetric Shear Web
- 6) Revised Quadrilateral Thin Shell
- 7) Triangular Cross-Section Ring

#### B. RECTANGULAR PRISM ELEMENT

A three-element cantilever beam subjected to an end moment is shown in Figure III-B.1 as the first example. This figure shows the loading, idealization, dimensions and material properties. The preprinted input data forms associated with this example are given in Figures III-B.2 to III-B.10.

Figure III-B.6, Boundary Condition Section, shows the use of the MODAL and REPEAT options. There are 4 exceptions to the MODAL card (Grid points 1, 5, 9 and 13). Grid points 5, 9 and 13 have exactly the same boundary conditions as grid point 1, therefore the REPEAT option is employed by placing an 'X' in column 12 opposite the entry for grid points 5, 9, and 13. Note that the four exceptions to the MODAL card are called out on the System Control Information Data Form, Figure III-B.4.

The following load data is evident by inspection of Figure III-B.7, External Loads Section.

- 1) One load condition is input.
- 2) The external applied load scalar equals zero.

- 3) Grid point 4 is loaded with a force in the -Y direction equal to 66.66667 pounds. The REPEAT option is used for grid point 12 which is subjected to the same load. Grid point 8 is loaded with a force in the +Y direction equal to 66.66667 pounds. Again the REPEAT option is used for grid point 16 which is subjected to the same load. Note that no entries corresponding to External Moments are made since the rectangular prism element only admits translational displacements.

In Figure III-B.9, Element Input, it is noted that only the MODAL entry is used. This means that every element in this example problem is subjected to a constant pre-strain state. Reference to the Engineers Manual (Reference 7) shows that the User has the option of calling out a constant element pre-strain or temperature state or an element pre-strain or temperature state which is the same functional form as the assumed displacement mode shapes (i.e., trilinear Lagrangian interpolation formulas). It was decided to use the former in this problem, hence the entry 0.0 was made. The User must be aware of his choice and be consistent throughout the analysis. Actually in this problem no element pre-strain or temperatures were considered so that either of the above options could have been chosen.

The output supplied by the MAGIC III system for this particular example is described below and shown in Figures III-B.11 to III-B.26.

Figure III-B.11 shows the matrix abstraction instructions associated with this example. A complete description of these instructions is provided in Reference 5. Figures III-B.12 to III-B.15 display the output from the Structural Systems Monitor. These figures record the input data pertinent to the problem being solved.

Figure III-B.12 displays the problem title and material data output. The gridpoint coordinates, temperatures and pressures are given in Figure III-B.13. Boundary condition information and finite element description is shown on Figure III-B.14. In the boundary condition portion of the figure, zeros ('0') represent degrees of



# Contrails

freedom that are fixed (i.e., no motion), ones ('1') represent degrees of freedom that are free or have unknown values of displacement, and twos ('2') represent degrees of freedom that are eliminated in the analysis procedure through the condensation technique. The second last column represents the cumulative number of degrees of freedom which actively participate in the equation solving process for displacements. The last column accumulates the number of two which participate in the calculation of the reduced stiffness matrix. The second portion of Figure III-B.14 shows the finite element description. Each of the three elements is called out in turn with grid points, print options and material number. Note that no extra grid points are listed nor needed for this element. The same comment also holds for section properties since all pertinent data are calculated within the program.

Figure III-B.15 displays the external load condition and the transformed external assembled load column. This 48 x 1 vector is the total unreduced load which is read row-wise. The ordering of this vector is consistent with that of the boundary condition table given in Figure III-B.14. Note that a load of 66.66667 pounds is applied at node point 4 in the negative global Y direction. This is position (11,1) in the load vector which corresponds to the eleventh entry in the boundary condition table which is the global V displacement for node point 4. The other loads shown follow the same pattern.

MAGIC III system output of final results are displayed in Figures III-B.16 to III-B.26. Figure III-B.17 shows the stiffness matrix for this problem. It is noted that only the non-zero terms are displayed. The stiffness matrix is presented row-wise and its ordering is consistent with that of the boundary condition table previously discussed. In this problem the ordering is

$$\{\Delta\}^T = \left[ V_2, W_2, V_3, W_3, \dots, V_{16}, W_{16} \right]$$

The externally applied load vector (GPRINT OF MATRIX LOADS)



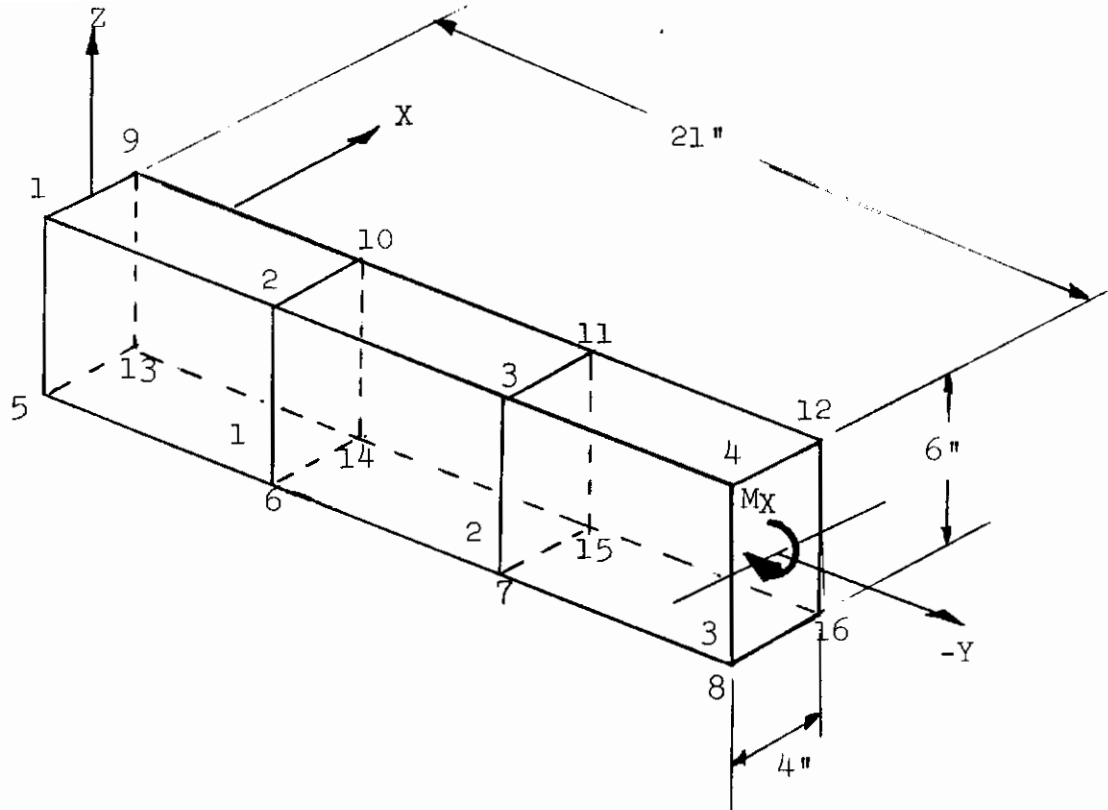
# Contrails

is presented in Figure III-B.18. This figure shows that forces ( $F_Y$ ) are applied in the negative and positive global Y directions at node points 4, 8, 12 and 16. These forces are numerically equal to  $\pm 66.66667$  pounds and are directed to form a moment of  $M_X = 800$  in. pounds applied at the tip of the cantilever.

The displacements of the cantilever beam resulting from the above loads are given in Figure III-B.19. It is noted that the displacements (U, V, W) are output corresponding to node point number and are referenced to the global axes unless otherwise specified. Figure III-B.20 shows the reactions ( $F_X$ ,  $F_Y$ ,  $F_Z$ ). These are output corresponding to node point number and are referenced to the global axes system unless otherwise specified.

The stresses arising in the structure are displayed in tabular form in Figures III-B.21 to III-B.23. Stresses are referenced to the local coordinate system and for this element are defined at the centroid. Six stress values are printed for the apparent element stress, element applied stress and net element stress categories. The apparent stress arises from element deformations and the applied stress arises from pre-strain and thermal effects. The net stress is the difference between the apparent and applied stress values. These stresses are given mathematical symbolism description in Reference 7.

The last set of output is given in Figures III-B.24 to III-B.26 and consist of the global oriented element forces. Three sets of forces are given and are categorized as above. The force points 1 through 8, in this example, correspond to element grid point numbers. For element number one, for example, force points 1, 2, 3, 4, 5, 6, 7, 8 correspond to element grid points 1, 2, 6, 5, 9, 10, 14, and 13 respectively.



$$E_X = E_Y = E_Z = E = 30.0 \times 10^6 \text{ psi}$$

$$\nu_{XY} = \nu_{YX} = \nu_{YZ} = \nu_{ZY} = \nu_{ZX} = \nu_{XZ} = \nu = .333$$

$$\rho = .00073395 \text{ #sec}^2/\text{in}^4$$

$$\bar{\epsilon}_X = \bar{\epsilon}_Y = \bar{\epsilon}_Z = 0, T_1 = T_2 = \dots T_{16} = 0.0$$

$$M_X = 800 \text{ in.lbs.}$$

FIG. III-B.1 RECTANGULAR PRISM ELEMENT -  
CANTILEVER BEAM WITH END MOMENT





## MAGIC STRUCTURAL ANALYSIS SYSTEM INPUT DATA FORMAT

### SYSTEM CONTROL INFORMATION

ENTER APPROPRIATE NUMBER, RIGHT ADJUSTED, IN BOX OPPOSITE APPLICABLE REQUESTS

	<table border="1" style="margin: auto;"> <tr><td>S</td><td>Y</td><td>S</td><td>T</td><td>E</td><td>M</td></tr> <tr><td>1</td><td>2</td><td>3</td><td>4</td><td>5</td><td>6</td></tr> </table>	S	Y	S	T	E	M	1	2	3	4	5	6	( / )			
S	Y	S	T	E	M												
1	2	3	4	5	6												
1. Number of System Grid Points	<table border="1" style="margin: auto;"> <tr><td></td><td></td><td></td><td></td><td>1</td><td>6</td></tr> <tr><td>1</td><td>2</td><td>3</td><td>4</td><td>5</td><td>6</td></tr> </table>					1	6	1	2	3	4	5	6				
				1	6												
1	2	3	4	5	6												
2. Number of Input Grid Points	<table border="1" style="margin: auto;"> <tr><td></td><td></td><td></td><td></td><td>1</td><td>6</td></tr> <tr><td>7</td><td>8</td><td>9</td><td>10</td><td>11</td><td>12</td></tr> </table>					1	6	7	8	9	10	11	12				
				1	6												
7	8	9	10	11	12												
3. Number of Degrees of Freedom/Grid Point	<table border="1" style="margin: auto;"> <tr><td></td><td>3</td></tr> <tr><td>13</td><td>14</td></tr> </table>		3	13	14												
	3																
13	14																
4. Number of Load Conditions	<table border="1" style="margin: auto;"> <tr><td></td><td>1</td></tr> <tr><td>15</td><td>16</td></tr> </table>		1	15	16												
	1																
15	16																
5. Number of Initially Displaced Grid Points	<table border="1" style="margin: auto;"> <tr><td></td><td></td><td></td><td></td><td></td><td>0</td></tr> <tr><td>17</td><td>18</td><td>19</td><td>20</td><td>21</td><td>22</td></tr> </table>						0	17	18	19	20	21	22				
					0												
17	18	19	20	21	22												
6. Number of Prescribed Displaced Grid Points	<table border="1" style="margin: auto;"> <tr><td></td><td></td><td></td><td></td><td></td><td>0</td></tr> <tr><td>23</td><td>24</td><td>25</td><td>26</td><td>27</td><td>28</td></tr> </table>						0	23	24	25	26	27	28				
					0												
23	24	25	26	27	28												
7. Number of Grid Point Axes Transformation Systems	<table border="1" style="margin: auto;"> <tr><td></td><td>0</td></tr> <tr><td>29</td><td>30</td></tr> </table>		0	29	30												
	0																
29	30																
8. Number of Elements	<table border="1" style="margin: auto;"> <tr><td></td><td></td><td></td><td></td><td></td><td>3</td></tr> <tr><td>31</td><td>32</td><td>33</td><td>34</td><td>35</td><td>36</td></tr> </table>						3	31	32	33	34	35	36				
					3												
31	32	33	34	35	36												
9. Number of Requests and/or Revisions of Material Tape.	<table border="1" style="margin: auto;"> <tr><td></td><td>1</td></tr> <tr><td>37</td><td>38</td></tr> </table>		1	37	38												
	1																
37	38																
10. Number of Input Boundary Condition Points	<table border="1" style="margin: auto;"> <tr><td></td><td></td><td></td><td></td><td></td><td>4</td></tr> <tr><td>39</td><td>40</td><td>41</td><td>42</td><td>43</td><td>44</td></tr> </table>						4	39	40	41	42	43	44				
					4												
39	40	41	42	43	44												
11. $T_0$ For Structure (With Decimal Point)	<table border="1" style="margin: auto;"> <tr><td></td><td></td><td></td><td></td><td>0</td><td>.</td><td>0</td></tr> <tr><td>45</td><td>46</td><td>47</td><td>48</td><td>49</td><td>50</td><td>51</td><td>52</td></tr> </table>					0	.	0	45	46	47	48	49	50	51	52	( / )
				0	.	0											
45	46	47	48	49	50	51	52										

FIGURE III-B.4 SYSTEM CONTROL INFORMATION - RECTANGULAR PRISM ELEMENT, CANTILEVER BEAM













MAGIC STRUCTURAL ANALYSIS SYSTEM  
INPUT DATA FORMAT

CHECK OR END CARD

C	H	E	C	K
1	2	3	4	5

 (/)

E	N	D
1	2	3

 (/)

FIGURE III-B.10 END CARD - RECTANGULAR PRISM ELEMENT,  
CANTILEVER BEAM



```

21 KR2 = K12 .MULT. KR1
22 KR = K11 .ADD. KR2
C----- SOLVE FOR DISPLACEMENTS D1
23 TRIA, D1 = KR, P1 .CHTRIA.
C----- SOLVE FOR DISPLACEMENTS D2
24 D2 = K11 .MULT. D1
C----- FORM TOTAL DISPLACEMENT VECTOR
25 DIT = D1 .TRANSP.
26 D2T = D2 .TRANSP.
27 D12 = D1T .ADJOIN. D2T
28 XX = D12 .TRANSP.
C----- EXPAND DISPLACEMENTS TO TOTAL SYSTEM DEGREES OF
C----- FREEDOM AND REARRANGE TO G-1-2 SYSTEM
29 TRO, TR12 = TR .DEJOIN. (SC(5,11),1)
30 X = TR12 .MULT. XX
31 XO = TR .MULT. X
C
C CALCULATE REACTIONS AND INVERSE CHECK
C
32 REACTS = KELA .MULT. XO
33 REACTP = REACTS .SUBT. TLOAD
34 IF (DIFF .NULL.) GO TO 10
C
C PRINT ELEMENT APPLIED LOADS, EXTERNAL LOADS, DISPLACEMENTS,
C REACTIONS AND INVERSE CHECK IN ENGINEERING FORMAT
C
C ELEMENTS HAVE 1 OR 2 DEGREES OF FREEDOM
35 GPRINT(4,.,.,FX,FY,FZ,MX,MY,MZ,SC,TR )FTELA
36 GPRINT(4,.,.,FX,FY,FZ,MX,MY,MZ,SC, )LOADS
37 GPRINT(2,.,.,U,V,W,THE TAY, THE TAZ, SC, )X
38 GPRINT(1,.,.,FX,FY,FZ,MX,MY,MZ,SC,TR )REACTP
39 IF (13 .NULL.) GO TO 600
C
C ELEMENTS HAVE 3 DEGREES OF FREEDOM
C
40 GPRINT(4,.,.,FR,0,FZ,0,MBETA,0,F1,0,F3,SC,TR )FTELA
41 GPRINT(4,.,.,FR,0,FZ,0,MBETA,0,F1,0,F3,SC, )LOADS
42 GPRINT(2,.,.,U,V,W,THE TAY,0,THE TAZ,0,MB,0,MB,SC, )X
43 GPRINT(1,.,.,FR,0,FZ,0,MBETA,0,F1,0,F3,SC,TR )REACTP
C
C GENERATE STRESSES AND FORCES
C
44 600 STRESP = EM,XO .STRESS. (4,.)
45 FORCEP = EM,XO .FORCE. (4,.)

```

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FIGURE III-B.11 MAGIC ABSTRACTION INSTRUCTION LISTING - RECTANGULAR PRISM ELEMENT, CANTILEVER BEAM (CONCLUDED)



THREE ELEMENT CANTILEVERED BEAM SUBJECTED TO AN END  
MOMENT, RECTANGULAR PRISM ELEMENT IDEN. NO.52  
STATICS ANALYSIS CONCENTRATED LOADS EQUIVALENT TO MOMENT  
REVISIONS OF MATERIAL TAPE

ASTERISK (\*) PRECEEDING MATERIAL  
IDENTIFICATION INDICATES THAT INPUT  
ERROR RETURNS WILL NOT RESULT IN  
TERMINATION OF EXECUTION

\*\*\*\*\*

REVISION

MATERIAL NUMBER 1 INPUT CODE I  
MATERIAL IDENTIFICATION STEEL  
NUMBER OF MATERIAL PROPERTY POINTS . . . . 1  
NUMBER OF PLASTIC PROPERTY POINTS . . . . 0  
MASS DENSITY . . . . . 0.73394994E-03

MATERIAL PROPERTIES

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YOUNG'S MODULI

DIRECTIONS

TEMPERATURE 0.0 0.30000E 08 0.30000E 08 0.30000E 08 0.30000E 08  
TH. EXP. COEF. XX YY ZZ XY XZ YZ

POISSON'S RATIOS

DIRECTIONS

0.33300E 00 0.33300E 00 0.33300E 00 0.33300E 00  
RIGIDITY MODULI XX YY XZ YZ

DIRECTIONS

TEMPERATURE 0.0 0.65000E-05 0.65000E-05 0.65000E-05 0.65000E-05  
RIGIDITY MODULI XX YY XZ YZ

\*\*\*\*\*

FIGURE III-B.12 TITLE AND MATERIAL DATA OUTPUT - RECTANGULAR PRISM ELEMENT, CANTILEVER BEAM



16 REF. POINTS

NO. DIRECTIONS = 3 NO. DEGREES OF FREEDOM = 1

POINT	GRIDPOINT DATA (IN RECTANGULAR COORDINATES)			TEMPERATURES			PRESSURES		
	X	Y	Z						
1	-0.2000000E 01	0.0	0.300000000E 01	0.0	0.0	0.0	0.0	0.0	0.0
2	-0.2000000E 01	-0.7000000E 01	0.300000000E 01	0.0	0.0	0.0	0.0	0.0	0.0
3	-0.2000000E 01	-0.1400000E 02	0.300000000E 01	0.0	0.0	0.0	0.0	0.0	0.0
4	-0.2000000E 01	-0.2100000E 02	0.300000000E 01	0.0	0.0	0.0	0.0	0.0	0.0
5	-0.2000000E 01	0.0	-0.300000000E 01	0.0	0.0	0.0	0.0	0.0	0.0
6	-0.2000000E 01	-0.7000000E 01	-0.300000000E 01	0.0	0.0	0.0	0.0	0.0	0.0
7	-0.2000000E 01	-0.1400000E 02	-0.300000000E 01	0.0	0.0	0.0	0.0	0.0	0.0
8	-0.2000000E 01	-0.2100000E 02	-0.300000000E 01	0.0	0.0	0.0	0.0	0.0	0.0
9	0.2000000E 01	0.0	0.300000000E 01	0.0	0.0	0.0	0.0	0.0	0.0
10	0.2000000E 01	-0.7000000E 01	0.300000000E 01	0.0	0.0	0.0	0.0	0.0	0.0
11	0.2000000E 01	-0.1400000E 02	0.300000000E 01	0.0	0.0	0.0	0.0	0.0	0.0
12	0.2000000E 01	-0.2100000E 02	0.300000000E 01	0.0	0.0	0.0	0.0	0.0	0.0
13	0.2000000E 01	0.0	-0.300000000E 01	0.0	0.0	0.0	0.0	0.0	0.0
14	0.2000000E 01	-0.7000000E 01	-0.300000000E 01	0.0	0.0	0.0	0.0	0.0	0.0
15	0.2000000E 01	-0.1400000E 02	-0.300000000E 01	0.0	0.0	0.0	0.0	0.0	0.0
16	0.2000000E 01	-0.2100000E 02	-0.300000000E 01	0.0	0.0	0.0	0.0	0.0	0.0

FIGURE III-B-13 GRID POINT DATA OUTPUT - RECTANGULAR PRISM ELEMENT, CANTILEVER BEAM

BOUNDARY CONDITION INFORMATION			
NODES	DEGREES OF FREEDOM	NO. OF ONES	NO. OF TWOS
1	0	0	0
2	1	2	1
3	1	4	2
4	1	6	3
5	0	6	3
6	1	8	4
7	1	10	5
8	1	12	6
9	0	12	6
10	1	14	7
11	1	16	8
12	1	18	9
13	0	18	9
14	1	20	10
15	1	22	11
16	1	24	12

TOTAL NO. ELEMENTS = 3

ELEM TYPE	MAT. NO.	CJDE	TEMP.	PRNT	NC.	-----GRID POINTS-----	EXTRA GRID PTS	-----SECTION PROPERTIES-----
1 52	1	8	0.0	0	8	1 2 6 5 9 10 14 13		0.0 0.0
2 52	1	8	0.0	0	8	3 2 10 11 7 6 14 15		0.0 0.0
2 52	1	8	0.0	0	8	4 12 16 8 3 11 15 7		0.0 0.0

FIGURE III-B.14 BOUNDARY CONDITION AND FINITE ELEMENT DESCRIPTION OUTPUT -  
RECTANGULAR PRISM ELEMENT, CANTILEVER BEAM

EXTERNAL LOAD CONDITIONS 1

LOAD NO.	1	NUMBER OF LOADED NODES	16	ELEMENT LOAD SCALAR =	0.0
4	0.0	-0.66667E 02	0.0		
12	0.0	-0.66667E 02	0.0		
8	0.0	0.66667E 02	0.0		
16	0.0	0.66667E 02	0.0		
1	0.0	0.0	0.0		
2	0.0	0.0	0.0		
3	0.0	0.0	0.0		
5	0.0	0.0	0.0		
6	0.0	0.0	0.0		
7	0.0	0.0	0.0		
9	0.0	0.0	0.0		
10	0.0	0.0	0.0		
11	0.0	0.0	0.0		
13	0.0	0.0	0.0		
14	0.0	0.0	0.0		
15	0.0	0.0	0.0		

LOAD NO.	1	TRANSFORMED EXTERNAL ASSEMBLED LOAD COLUMN	48 X 1	TRANSFORMED EXTERNAL ASSEMBLED LOAD COLUMN	48 X 1
4	0.0	0.0	0.0	0.0	0.0
12	0.0	0.0	0.0	0.0	0.0
8	0.0	0.0	0.0	-0.66666992E 02	0.0
16	0.0	0.0	0.0	0.0	0.0
1	0.0	0.0	0.0	0.666666992E 02	0.0
2	0.0	0.0	0.0	0.0	0.0
3	0.0	0.0	0.0	0.0	0.0
5	0.0	0.0	0.0	-0.666666992E 02	0.0
6	0.0	0.0	0.0	0.0	0.0
7	0.0	0.0	0.0	0.0	0.0
9	0.0	0.0	0.0	0.0	0.0
10	0.0	0.0	0.0	0.0	0.0
11	0.0	0.0	0.0	0.0	0.0
13	0.0	0.0	0.0	0.0	0.0
14	0.0	0.0	0.0	0.0	0.0
15	0.0	0.0	0.0	0.666666992E 02	0.0

FIGURE III-B.15 EXTERNAL LOAD CONDITIONS - RECTANGULAR PRISM ELEMENT, CANTILEVER BEAM

T-ZERO FOR STRUCTURE = 0.0

FIGURE III-B.16 TRANSFORMED EXTERNAL ASSEMBLED LOAD OUTPUT - RECTANGULAR PRISM ELEMENT, CANTILEVER BEAM

MATRIX STIFF / 1/

CUTOFF = 0.0

DISP	FORCE	FORCE	FORCE	FORCE	SIZE	FORCE	SIZE	FORCE	FORCE	FORCE	FORCE	
1	1	0.721690E 08	3	-0.763991E 07	4	0.372846E 07	7	0.185801E 08	9	0.185801E 08	9	-0.819605E 07
10	10	-0.112303E 08	13	-0.330032E 07	15	-0.136662E 08	16	0.186423E 07	19	0.186423E 07	19	-0.104023E 08
21	21	-0.502113E 07	22	-0.561516E 07	26	-0.559269E 07	29	-0.279634E 07	32	-0.279634E 07	32	0.168455E 08
35	35	0.842275E 07										
DISP	2	2	0.814384E 08	3	-0.372846E 07	4	0.139294E 08	8	-0.291933E 08	9	-0.291933E 08	-0.112303E 08
	10	10	-0.105134E 08	14	0.133441E 07	15	-0.186423E 07	16	-0.268149E 07	20	-0.268149E 07	-0.342890E 08
	21	21	-0.561516E 07	22	-0.101798E 08	25	-0.393061E 08	26	-0.982654E 07	28	-0.982654E 07	-0.130496E 08
	29	29	-0.326240E 07	31	0.130496E 08	32	0.326240E 07	34	0.393061E 08	35	0.393061E 08	0.582654E 07
DISP	3	1	-0.763991E 07	2	-0.372846E 07	3	0.721690E 08	5	-0.763991E 07	6	-0.763991E 07	0.372846E 07
	7	7	-0.819605E 07	8	0.112303E 08	9	0.185801E 08	11	-0.819605E 07	12	-0.819605E 07	-0.112303E 08
	13	13	-0.136662E 08	14	-0.186423E 07	15	-0.330032E 07	17	-0.136662E 08	18	-0.136662E 08	0.186423E 07
	19	19	-0.902113E 07	20	0.561516E 07	21	-0.104023E 08	23	-0.902113E 07	24	-0.902113E 07	-0.561516E 07
	25	25	0.559269E 07	27	-0.559269E 07	28	0.279634E 07	30	-0.559269E 07	31	-0.559269E 07	-0.561516E 07
	33	33	0.168455E 08	34	-0.842275E 07	36	0.842275E 07					-0.168455E 08
DISP	4	1	0.372846E 07	2	0.139294E 08	4	0.814384E 08	5	-0.372846E 07	6	-0.372846E 07	0.139294E 08
	7	7	0.112303E 08	8	-0.105134E 08	10	-0.291933E 08	11	-0.112303E 08	12	-0.112303E 08	-0.105134E 08
	13	13	0.186423E 07	14	-0.288149E 07	16	0.133441E 07	17	-0.186423E 07	18	-0.186423E 07	-0.288149E 07
	19	19	0.561516E 07	20	-0.101798E 08	22	-0.342890E 08	23	-0.561516E 07	24	-0.561516E 07	-0.101798E 08
	25	25	-0.982654E 07	26	-0.393061E 08	27	-0.982654E 07	28	-0.982654E 07	29	-0.982654E 07	-0.130496E 08
	30	30	-0.326240E 07	31	0.326240E 07	32	0.130496E 08	33	0.326240E 07	34	0.326240E 07	0.982654E 07
	35	35	0.393061E 08									
DISP	5	3	-0.763991E 07	4	-0.372846E 07	5	0.360845E 08	6	-0.112303E 08	9	-0.112303E 08	-0.819605E 07
	10	10	0.112303E 08	11	0.929007E 07	12	0.372846E 07	15	-0.136662E 08	16	-0.136662E 08	-0.186423E 07
	17	17	-0.165010E 07	18	-0.561516E 07	21	-0.902113E 07	22	0.561516E 07	23	0.561516E 07	-0.520117E 07
	24	24	0.186423E 07	26	0.559269E 07	27	0.168455E 08	29	0.279634E 07	30	0.279634E 07	0.842275E 07
	32	32	-0.168455E 08	33	-0.559269E 07	35	-0.842275E 07	36	-0.279634E 07			
DISP	6	3	0.372846E 07	4	0.139294E 08	5	-0.112303E 08	6	0.407192E 08	9	0.407192E 08	0.112303E 08
	10	10	-0.105134E 08	11	-0.372846E 07	12	-0.145967E 08	15	0.186423E 07	16	0.186423E 07	-0.288149E 07
	17	17	-0.561516E 07	18	0.667204E 06	21	0.561516E 07	22	-0.101798E 08	23	-0.101798E 08	-0.186423E 07
	24	24	-0.171449E 08	26	-0.982654E 07	27	-0.196531E 08	29	-0.326240E 07	30	-0.326240E 07	-0.652481E 07
	32	32	0.326240E 07	33	0.652481E 07	35	0.982654E 07	36	0.196531E 08			
DISP	7	1	0.185801E 08	3	-0.819605E 07	4	0.112303E 08	7	0.721690E 08	9	0.721690E 08	-0.763991E 07
	10	10	-0.372846E 07	13	-0.104023E 08	15	-0.902113E 07	16	0.561516E 07	19	0.561516E 07	-0.330032E 07
	21	21	-0.136662E 08	22	-0.186423E 07	26	-0.279634E 07	29	-0.559269E 07	32	-0.559269E 07	0.842275E 07
	35	35	0.168455E 08									
DISP	8	2	-0.291933E 08	3	0.112303E 08	4	-0.105134E 08	8	0.814384E 08	9	0.814384E 08	0.372846E 07
	10	10	0.139294E 08	14	-0.342890E 08	15	0.561516E 07	16	-0.101798E 08	20	-0.101798E 08	0.133441E 07
	21	21	0.186423E 07	22	-0.288149E 07	25	0.130496E 08	26	0.326240E 07	28	0.326240E 07	0.393061E 08
	29	29	0.582654E 07	31	-0.393061E 08	32	-0.982654E 07	34	-0.982654E 07	35	-0.982654E 07	-0.326240E 07

FIGURE III-B.17 STIFFNESS MATRIX - RECTANGULAR PRISM ELEMENT, CANTILEVER BEAM

CUTOFF = 0.0

DISP	FORCE			FORCE			FORCE			FORCE		
	9	10	11	12	13	14	15	16	17	18	19	20
	1	2	3	4	5	6	7	8	9	10	11	12
	7	8	9	10	11	12	13	14	15	16	17	18
	13	14	15	16	17	18	19	20	21	22	23	24
	19	20	21	22	23	24	25	26	27	28	29	30
	25	26	27	28	29	30	31	32	33	34	35	36
	33	34	35	36	37	38	39	40	41	42	43	44
	5	6	7	8	9	10	11	12	13	14	15	16
	17	18	19	20	21	22	23	24	25	26	27	28
	29	30	31	32	33	34	35	36	37	38	39	40
	47	48	49	50	51	52	53	54	55	56	57	58
	65	66	67	68	69	70	71	72	73	74	75	76
	83	84	85	86	87	88	89	90	91	92	93	94
	101	102	103	104	105	106	107	108	109	110	111	112
	120	121	122	123	124	125	126	127	128	129	130	131
	149	150	151	152	153	154	155	156	157	158	159	160
	178	179	180	181	182	183	184	185	186	187	188	189
	207	208	209	210	211	212	213	214	215	216	217	218
	237	238	239	240	241	242	243	244	245	246	247	248
	267	268	269	270	271	272	273	274	275	276	277	278
	297	298	299	300	301	302	303	304	305	306	307	308
	327	328	329	330	331	332	333	334	335	336	337	338
	357	358	359	360	361	362	363	364	365	366	367	368
	387	388	389	390	391	392	393	394	395	396	397	398
	407	408	409	410	411	412	413	414	415	416	417	418
	437	438	439	440	441	442	443	444	445	446	447	448
	467	468	469	470	471	472	473	474	475	476	477	478
	487	488	489	490	491	492	493	494	495	496	497	498
	507	508	509	510	511	512	513	514	515	516	517	518
	537	538	539	540	541	542	543	544	545	546	547	548
	567	568	569	570	571	572	573	574	575	576	577	578
	587	588	589	590	591	592	593	594	595	596	597	598
	607	608	609	610	611	612	613	614	615	616	617	618
	637	638	639	640	641	642	643	644	645	646	647	648
	667	668	669	670	671	672	673	674	675	676	677	678
	687	688	689	690	691	692	693	694	695	696	697	698
	707	708	709	710	711	712	713	714	715	716	717	718
	737	738	739	740	741	742	743	744	745	746	747	748
	767	768	769	770	771	772	773	774	775	776	777	778
	787	788	789	790	791	792	793	794	795	796	797	798
	807	808	809	810	811	812	813	814	815	816	817	818
	837	838	839	840	841	842	843	844	845	846	847	848
	867	868	869	870	871	872	873	874	875	876	877	878
	887	888	889	890	891	892	893	894	895	896	897	898
	907	908	909	910	911	912	913	914	915	916	917	918
	937	938	939	940	941	942	943	944	945	946	947	948
	967	968	969	970	971	972	973	974	975	976	977	978
	987	988	989	990	991	992	993	994	995	996	997	998
	1007	1008	1009	1010	1011	1012	1013	1014	1015	1016	1017	1018

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FIGURE III-B.17 CONTINUED

CUTOFF = 0.0

DISP	FORCE	FORCE		FORCE		FORCE		FORCE		SIZE	FORCE		FORCE	PAGE							
		16	17	18	19	20	21	22	23		24	25			26	27	28	29	30	31	32
16	13	0.372846E 07	14	0.139294E 08	16	0.814384E 08	17	-0.372846E 07	18	0.139294E 08	17	-0.372846E 07	18	0.139294E 08							
19	19	0.112303E 08	20	-0.105134E 08	22	-0.291933E 08	23	-0.112303E 08	24	-0.105134E 08	23	-0.112303E 08	24	-0.105134E 08							
25	25	-0.326240E 07	26	-0.130496E 08	27	-0.326240E 07	28	-0.130496E 08	29	-0.326240E 07	28	-0.130496E 08	29	-0.326240E 07							
30	30	-0.982654E 07	31	0.982654E 07	32	0.393061E 08	33	0.982654E 07	34	0.982654E 07	33	0.982654E 07	34	0.982654E 07							
35	35	0.130496E 08	36	0.326240E 07																	
DISP	17	3	-0.136662E 08	4	-0.186423E 07	5	-0.165016E 07	6	-0.561516E 07	9	-0.561516E 07	9	-0.902113E 07								
	10	0.561516E 07	11	0.520117E 07	12	0.186423E 07	15	-0.763991E 07	16	-0.763991E 07	15	-0.763991E 07	16	-0.929007E 07							
	17	0.360845E 08	18	-0.112303E 08	21	-0.819605E 07	22	0.112303E 08	23	0.929007E 07	22	0.112303E 08	23	0.929007E 07							
	24	0.372846E 07	26	0.168455E 08	27	0.559269E 07	29	0.842279E 07	30	0.279634E 07	29	0.842279E 07	30	0.279634E 07							
	32	-0.559269E 07	33	-0.168455E 08	35	-0.279634E 07	36	-0.842279E 07			36	-0.842279E 07									
DISP	18	3	0.186423E 07	4	-0.288148E 07	5	-0.561516E 07	6	0.667204E 06	9	0.667204E 06	9	0.561516E 07								
	10	-0.101798E 08	11	-0.186423E 07	12	-0.171445E 08	15	0.372846E 07	16	0.139294E 08	15	0.372846E 07	16	0.139294E 08							
	17	-0.112303E 08	18	0.407192E 08	21	0.112303E 08	22	-0.105134E 08	23	-0.372846E 07	22	-0.105134E 08	23	-0.372846E 07							
	24	-0.145967E 08	26	-0.326240E 07	27	-0.652481E 07	29	-0.982654E 07	30	-0.196531E 08	29	-0.982654E 07	30	-0.196531E 08							
	32	0.582654E 07	33	0.196531E 08	35	0.326240E 07	36	0.652481E 07			36	0.652481E 07									
DISP	19	1	-0.104023E 08	3	-0.902113E 07	4	0.561516E 07	7	-0.330032E 07	9	-0.330032E 07	9	-0.136662E 08								
	10	-0.186423E 07	13	0.185801E 08	15	-0.819605E 07	16	0.112303E 08	19	0.721690E 08	16	0.112303E 08	19	0.721690E 08							
	21	-0.763991E 07	22	-0.372846E 07	26	-0.842279E 07	29	-0.168455E 08	32	0.279634E 07	29	-0.168455E 08	32	0.279634E 07							
	35	0.559269E 07																			
DISP	20	2	-0.342890E 08	3	0.561516E 07	4	-0.101798E 08	8	0.133441E 07	9	0.133441E 07	9	0.186423E 07								
	10	-0.288149E 07	14	-0.291933E 08	15	0.112303E 08	16	-0.105134E 08	20	0.814384E 08	16	-0.105134E 08	20	0.814384E 08							
	21	0.372846E 07	22	0.139294E 08	25	0.393061E 08	26	0.982654E 07	28	0.130496E 08	26	0.982654E 07	28	0.130496E 08							
	29	0.326240E 07	31	-0.130496E 08	32	-0.326240E 07	34	-0.393061E 08	35	-0.982654E 07	34	-0.393061E 08	35	-0.982654E 07							
DISP	21	1	-0.902113E 07	2	-0.561516E 07	3	-0.104023E 08	5	-0.902113E 07	6	-0.902113E 07	6	0.561516E 07								
	7	-0.136662E 08	8	0.186423E 07	9	-0.330032E 07	11	-0.136662E 08	12	-0.186423E 07	11	-0.136662E 08	12	-0.186423E 07							
	13	-0.819605E 07	14	-0.112303E 08	15	0.165801E 08	17	-0.819605E 07	18	0.112303E 08	17	-0.819605E 07	18	0.112303E 08							
	19	-0.763991E 07	20	0.372846E 07	21	0.721690E 08	23	-0.763991E 07	24	-0.372846E 07	23	-0.763991E 07	24	-0.372846E 07							
	25	0.842279E 07	27	-0.842279E 07	28	0.168455E 08	30	0.842279E 07	31	-0.279634E 07	30	0.842279E 07	31	-0.279634E 07							
	33	0.279634E 07	34	-0.559269E 07	36	0.559269E 07					36	0.559269E 07									
DISP	22	1	-0.561516E 07	2	-0.342890E 08	4	-0.342890E 08	5	0.561516E 07	6	0.561516E 07	6	-0.101798E 08								
	7	-0.186423E 07	8	-0.288149E 07	10	0.133441E 07	11	0.186423E 07	12	0.288149E 07	11	0.186423E 07	12	0.288149E 07							
	13	-0.112303E 08	14	-0.105134E 08	16	-0.291933E 08	17	0.112303E 08	18	-0.105134E 08	17	0.112303E 08	18	-0.105134E 08							
	19	-0.372846E 07	20	0.139294E 08	22	0.814384E 08	23	-0.372846E 07	24	0.139294E 08	23	-0.372846E 07	24	0.139294E 08							
	25	0.982654E 07	26	0.393061E 08	27	0.982654E 07	28	0.326240E 07	29	0.130496E 08	28	0.326240E 07	29	0.130496E 08							
	30	0.326240E 07	31	-0.326240E 07	32	-0.130496E 08	33	-0.326240E 07	34	-0.982654E 07	33	-0.326240E 07	34	-0.982654E 07							
	35	-0.393061E 08	36	-0.982654E 07																	
DISP	23	3	-0.502113E 07	4	-0.561516E 07	5	-0.520117E 07	6	-0.186423E 07	9	-0.186423E 07	9	-0.136662E 08								
	10	0.186423E 07	11	-0.165016E 07	12	0.561516E 07	15	-0.819605E 07	16	-0.112303E 08	15	-0.819605E 07	16	-0.112303E 08							
	17	0.929007E 07	18	-0.372846E 07	21	-0.763991E 07	22	-0.372846E 07	23	0.372846E 07	22	-0.372846E 07	23	0.372846E 07							

FIGURE III-B.17 CONTINUED



MATRIX STIFF / I/

CUTOFF = 0.0

DISP	FORCE	FORCE	FORCE	FORCE	FORCE	FORCE	SIZE	36 BY	36	FORCE	PAGE
23	0.112303E 08	26	0.842275E 07	27	0.279634E 07	29	0.168455E 08	30	0.559269E 07	07	
32	-0.279634E 07	33	-0.842275E 07	35	-0.559269E 07	36	-0.168455E 08				
24	3	-0.561516E 07	4	-0.101798E 08	5	0.1866423E 07	6	-0.171445E 08	9	-0.186423E 07	
10	-0.288148E 07	11	0.561516E 07	12	0.667204E 06	15	-0.112303E 08	16	-0.105134E 08		
17	0.372846E 07	18	-0.145967E 08	21	-0.372846E 07	22	0.139294E 08	23	0.112303E 08		
24	0.407192E 08	26	0.982654E 07	27	0.196531E 08	29	0.326240E 07	30	0.652481E 07		
32	-0.326240E 07	33	-0.652481E 07	35	-0.982654E 07	36	-0.196531E 08				
25	2	-0.393061E 08	3	0.559269E 07	4	-0.982654E 07	8	0.130496E 08	9	0.279634E 07	
10	0.326240E 07	14	-0.130496E 08	15	0.168455E 08	16	-0.326240E 07	20	0.393061E 08		
21	0.842275E 07	22	0.982654E 07	25	0.125112E 09	26	0.248478E 08	28	0.450516E 08		
29	0.804782E 07	31	-0.947472E 08	32	-0.269019E 08	34	-0.561258E 08	35	-0.156390E 08		
26	1	-0.559269E 07	2	-0.982654E 07	4	-0.393061E 08	5	0.559269E 07	6	-0.982654E 07	
7	-0.279634E 07	8	0.326240E 07	10	0.130496E 08	11	0.279634E 07	12	0.326240E 07		
13	-0.168455E 08	14	-0.326240E 07	16	-0.130496E 08	17	0.168455E 08	18	-0.326240E 07		
19	-0.842275E 07	20	0.982654E 07	22	0.393061E 08	23	0.842275E 07	24	0.582654E 07		
25	0.248478E 08	26	0.125112E 09	27	0.248478E 08	28	0.450516E 08	29	0.450516E 08		
30	0.804782E 07	31	-0.269019E 08	32	-0.947472E 08	33	-0.561258E 08	34	-0.156390E 08		
35	-0.561258E 08	36	-0.156390E 08								
27	3	-0.559269E 07	4	-0.982654E 07	5	0.168455E 08	6	-0.196531E 08	9	-0.279634E 07	
10	0.326240E 07	11	0.652481E 07	12	0.652481E 07	15	-0.559269E 07	16	-0.326240E 07		
17	0.559269E 07	18	-0.652481E 07	21	-0.842275E 07	22	0.982654E 07	23	0.279634E 07		
24	0.196531E 08	26	0.248478E 08	27	0.625360E 08	29	0.804782E 07	30	0.225258E 08		
32	-0.269019E 08	33	-0.473736E 08	35	-0.156390E 08	36	-0.280629E 08				
28	2	-0.130496E 08	3	0.279634E 07	4	-0.326240E 07	8	0.393061E 08	9	0.559269E 07	
10	0.982654E 07	14	-0.393061E 08	15	0.842275E 07	16	-0.982654E 07	20	0.130496E 08		
21	0.168455E 08	22	0.326240E 07	25	0.450516E 08	26	0.804782E 07	28	0.125112E 09		
29	0.248478E 08	31	-0.561258E 08	32	-0.156390E 08	34	-0.947472E 08	35	-0.269019E 08		
29	1	-0.279634E 07	2	-0.326240E 07	4	-0.130496E 08	5	0.279634E 07	6	-0.326240E 07	
7	-0.559269E 07	8	0.982654E 07	10	0.393061E 08	11	0.559269E 07	12	0.982654E 07		
13	-0.842275E 07	14	-0.982654E 07	16	-0.393061E 08	17	0.842275E 07	18	-0.982654E 07		
19	-0.168455E 08	20	0.326240E 07	22	0.130496E 08	23	0.168455E 08	24	0.326240E 07		
25	0.804782E 07	26	0.450516E 08	27	0.804782E 07	28	0.248478E 08	29	0.125112E 09		
30	0.248478E 08	31	-0.156390E 08	32	-0.561258E 08	33	-0.947472E 08	34	-0.269019E 08		
35	-0.947472E 08	36	-0.269019E 08								
30	3	-0.279634E 07	4	-0.326240E 07	5	0.842275E 07	6	-0.652481E 07	9	-0.559269E 07	
10	0.582654E 07	11	0.168455E 08	12	0.196531E 08	15	-0.842275E 07	16	-0.982654E 07		
17	0.279634E 07	18	-0.196531E 08	21	-0.168455E 08	22	0.326240E 07	23	0.559269E 07		
24	0.652481E 07	26	0.804782E 07	27	0.225258E 08	29	0.248478E 08	30	0.625560E 08		

FIGURE III-B 17 CONTINUED





GPRINT CF MATRIX LOADS (SET 1)

RDM	FX	FY	FZ
1	0.0	0.0	0.0
2	0.0	0.0	0.0
3	0.0	0.0	0.0
4	0.0	-0.6666692E 02	0.0
5	0.0	0.0	0.0
6	0.0	0.0	0.0
7	0.0	0.0	0.0
8	0.0	0.6666692E 02	0.0
9	0.0	0.0	0.0
10	0.0	0.0	0.0
11	0.0	0.0	0.0
12	0.0	-0.6666692E 02	0.0
13	0.0	0.0	0.0
14	0.0	0.0	0.0
15	0.0	0.0	0.0
16	0.0	0.6666692E 02	0.0

FIGURE III-B.18 GPRINT OF MATRIX LOADS - RECTANGULAR PRISM ELEMENT, CANTILEVER BEAM

DISPLACEMENT MATRIX FOR LOAD CONDITION 1

48 X 1

ROW	U	V	W
1	0.0	0.0	0.0
2	0.7212636E-06	-0.43426453E-05	-0.50667686E-05
3	0.57006582E-06	-0.90564117E-05	-0.20699066E-04
4	0.60850289E-06	-0.13695938E-04	-0.47243157E-04
5	0.0	0.0	0.0
6	-0.72112090E-06	0.43425607E-05	-0.50667450E-05
7	-0.56904446E-06	0.90560370E-05	-0.20699066E-04
8	-0.60586717E-06	0.13695391E-04	-0.47243229E-04
9	0.0	0.0	0.0
10	-0.72141148E-06	-0.43426089E-05	-0.50665412E-05
11	-0.56976023E-06	-0.90560670E-05	-0.20698557E-04
12	-0.60687034E-06	-0.13695461E-04	-0.47242473E-04
13	0.0	0.0	0.0
14	0.72154944E-06	0.43426799E-05	-0.50665967E-05
15	0.57072793E-06	0.90564722E-05	-0.20698542E-04
16	0.60934690E-06	0.13695978E-04	-0.47242473E-04

FIGURE III-B.19 DISPLACEMENT MATRIX - RECTANGULAR PRISM ELEMENT, CANTILEVER BEAM

REACTIONS AND INVERSE CHECK FOR LOAD CONDITION 1

ROW	FX	FY	FZ
1	-0.28477386E-02	0.66678650E-02	0.32339096E-02
2	0.47016144E-03	0.78678131E-03	-0.41007996E-04
3	0.70276260E-02	0.55789948E-03	-0.27370453E-02
4	0.74005127E-02	0.38146973E-03	-0.18739700E-02
5	0.28476883E-02	-0.66676590E-02	0.27313232E-02
6	0.84018707E-03	0.32854080E-02	-0.14524460E-02
7	0.63781738E-02	-0.36373138E-02	-0.41780472E-02
8	0.26702881E-02	-0.40283203E-02	-0.41713715E-02
9	0.28478012E-02	0.66677841E-02	0.24719238E-02
10	-0.57220459E-03	-0.57306290E-02	-0.85163116E-03
11	-0.3526876E-02	0.29077530E-02	-0.14371872E-02
12	-0.57754517E-02	0.32806396E-02	-0.40054321E-04
13	-0.28477570E-02	-0.66678787E-02	0.17709732E-02
14	-0.62462503E-03	-0.24547577E-02	-0.23851395E-02
15	-0.79050044E-02	0.22563934E-02	0.51784515E-02
16	-0.59204102E-02	0.22430420E-02	0.30946732E-02

FIGURE III-B.20 REACTION MATRIX - RECTANGULAR PRISM ELEMENT, CANTILEVER BEAM

S T R E S S E S F O R T H E R E C T A N G U L A R P R I S M E L E M E N T  
(STRESSES EVALUATED AT THE ELEMENT CENTROID)

LOAD CONDITION NUMBER	ELEMENT NUMBER	ELEMENT TYPE	ELEMENT GRID POINTS										
	1	52	1	2	6	5	5	10	14	13			
APPARENT ELEMENT STRESSES													
STRESS POINT 1	SIGMA-X 0.28610229E-04	SIGMA-Y 0.44822653E-04	SIGMA-Z 0.63896179E-04	SIGMA-XY -0.28610229E-05	SIGMA-YZ -0.42515344E-03	SIGMA-ZX 0.66757202E-05							
ELEMENT APPLIED STRESSES													
STRESS POINT 1	SIGMA-X 0.0	SIGMA-Y 0.0	SIGMA-Z 0.0	SIGMA-XY 0.0	SIGMA-YZ 0.0	SIGMA-ZX 0.0							
NET ELEMENT STRESSES													
STRESS POINT 1	SIGMA-X 0.28610229E-04	SIGMA-Y 0.44822653E-04	SIGMA-Z 0.63896179E-04	SIGMA-XY -0.28610229E-05	SIGMA-YZ -0.42515344E-03	SIGMA-ZX 0.66757202E-05							

FIGURE III-B.21 STRESS OUTPUT, ELEMENT NO. 1 - RECTANGULAR PRISM ELEMENT, CANTILEVER BEAM

S T R E S S E S F O R T H E R E C T A N G U L A R P R I S M E L E M E N T  
(STRESSES EVALUATED AT THE ELEMENT CENTROID)

LOAD CONDITION NUMBER	ELEMENT NUMBER	ELEMENT TYPE	ELEMENT GRID POINTS												
1	2	52	3	2	10	11	7	6	14	15					
APPARENT ELEMENT STRESSES															
STRESS POINT 1	SIGMA-X 0.30517578E-04	SIGMA-Y -0.30517578E-04	SIGMA-Z -0.10681152E-03	MEMBRANE STRESSES			SIGMA-XY 0.20313263E-03	SIGMA-YZ 0.24795532E-04	SIGMA-ZX -0.46730042E-04						
ELEMENT APPLIED STRESSES															
STRESS POINT 1	SIGMA-X 0.0	SIGMA-Y 0.0	SIGMA-Z 0.0	MEMBRANE STRESSES			SIGMA-XY 0.0	SIGMA-YZ 0.0	SIGMA-ZX 0.0						
NET ELEMENT STRESSES															
STRESS POINT 1	SIGMA-X 0.30517578E-04	SIGMA-Y -0.30517578E-04	SIGMA-Z -0.10681152E-03	MEMBRANE STRESSES			SIGMA-XY 0.20313263E-03	SIGMA-YZ 0.24795532E-04	SIGMA-ZX -0.46730042E-04						

FIGURE III-B.22 STRESS OUTPUT, ELEMENT NO. 2 - RECTANGULAR PRISM ELEMENT, CANTILEVER BEAM

STRESSES FOR THE RECTANGULAR PRISM ELEMENT  
(STRESSES EVALUATED AT THE ELEMENT CENTROID)

LOAD CONDITION NUMBER	ELEMENT ALMPER	ELEMENT TYPE	ELEMENT GRID POINTS							
1	3	52	4	12	16	8	3	11	15	7
APPARENT ELEMENT STRESSES										
STRESS POINT 1	SIGMA-X -0.61035156E-04	SIGMA-Y -0.39672852E-03	SIGMA-Z -0.30517578E-04	MEMBRANE STRESSES			SIGMA-XY -0.40054321E-04	SIGMA-YZ 0.20217896E-03	SIGMA-ZX -0.49591064E-04	
ELEMENT APPLIED STRESSES										
STRESS POINT 1	SIGMA-X 0.0	SIGMA-Y 0.0	SIGMA-Z 0.0	MEMBRANE STRESSES			SIGMA-XY 0.0	SIGMA-YZ 0.0	SIGMA-ZX 0.0	
NET ELEMENT STRESSES										
STRESS POINT 1	SIGMA-X -0.61035156E-04	SIGMA-Y -0.39672852E-03	SIGMA-Z -0.30517578E-04	MEMBRANE STRESSES			SIGMA-XY -0.40054321E-04	SIGMA-YZ 0.20217896E-03	SIGMA-ZX -0.49591064E-04	

FIGURE III-B.23 STRESS OUTPUT, ELEMENT NO. 3 - RECTANGULAR PRISM ELEMENT, CANTILEVER BEAM



F O R C E S F O R T H E R E C T A N G U L A R P R I S M E L E M E N T

LOAD CONDITION NUMBER	ELEMENT NUMBER	ELEMENT TYPE	ELEMENT GRID POINTS
1	1	52	1 2 6 5 10 14 13

APPARENT ELEMENT FORCES	FORCES
POINT	FX FY FZ
1	-0.28477417E 02 0.66678665E 02 0.32653809E-02
2	-0.59162750E 01 -0.66672806E 02 -0.27008057E-02
3	0.59156494E 01 0.66670883E 02 -0.29954910E-02
4	0.28476898E 02 -0.66676605E 02 0.27618408E-02
5	0.28478012E 02 0.66677841E 02 0.24566650E-02
6	0.59155121E 01 -0.66671478E 02 -0.12664795E-02
7	-0.59147644E 01 0.66672318E 02 -0.33569336E-02
8	-0.28477585E 02 -0.66678818E 02 0.17547607E-02

E L E M E N T A P P L I E D F O R C E S

ELEMENT POINT	FORCES
POINT	FX FY FZ
1	0.0 0.0 0.0
2	0.0 0.0 0.0
3	0.0 0.0 0.0
4	0.0 0.0 0.0
5	0.0 0.0 0.0
6	0.0 0.0 0.0
7	0.0 0.0 0.0
8	0.0 0.0 0.0

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N E T E L E M E N T F O R C E S

NET ELEMENT FORCES	FORCES
POINT	FX FY FZ
1	-0.28477417E 02 0.66678665E 02 0.32653809E-02
2	-0.59162750E 01 -0.66672806E 02 -0.27008057E-02
3	0.59156494E 01 0.66670883E 02 -0.29954910E-02
4	0.28476898E 02 -0.66676605E 02 0.27618408E-02
5	0.28478012E 02 0.66677841E 02 0.24566650E-02
6	0.59155121E 01 -0.66671478E 02 -0.12664795E-02
7	-0.59147644E 01 0.66672318E 02 -0.33569336E-02
8	-0.28477585E 02 -0.66678818E 02 0.17547607E-02

FIGURE III-B.24 FORCE OUTPUT, ELEMENT NO. 1 - RECTANGULAR PRISM ELEMENT, CANTILEVER BEAM

FORCES FOR THE RECTANGULAR PRISM ELEMENT

LOAD CONDITION NUMBER 1 ELEMENT NUMBER 2 ELEMENT TYPE 52 ELEMENT GRID POINTS 3 2 10 11 7 6 14 15

APPARENT ELEMENT POINT	FORCES	FORCES	FORCES
	FX	FY	FZ
1	0.11796875E 01	-0.66669357E 02	-0.12207031E-02
2	0.59157715E 01	0.66673553E 02	0.33721924E-02
3	-0.59153137E 01	0.66665924E 02	0.10223389E-02
4	-0.11799622E 01	-0.66663589E 02	-0.48828125E-03
5	-0.11765747E 01	0.66663559E 02	-0.40235519E-02
6	-0.59152527E 01	-0.66667511E 02	0.71716309E-03
7	0.59148560E 01	-0.66673813E 02	0.24414063E-03
8	0.11774902E 01	0.66671112E 02	0.73242188E-03

ELEMENT APPLIED POINT	FORCES	FORCES	FORCES
	FX	FY	FZ
1	0.0	0.0	0.0
2	0.0	0.0	0.0
3	0.0	0.0	0.0
4	0.0	0.0	0.0
5	0.0	0.0	0.0
6	0.0	0.0	0.0
7	0.0	0.0	0.0
8	0.0	0.0	0.0

NET ELEMENT POINT	FORCES	FORCES	FORCES
	FX	FY	FZ
1	0.11796875E 01	-0.66669357E 02	-0.12207031E-02
2	0.59157715E 01	0.66673553E 02	0.33721924E-02
3	-0.59153137E 01	0.66665924E 02	0.10223389E-02
4	-0.11799622E 01	-0.66663589E 02	-0.48828125E-03
5	-0.11765747E 01	0.66663559E 02	-0.40235519E-02
6	-0.59152527E 01	-0.66667511E 02	0.71716309E-03
7	0.59148560E 01	-0.66673813E 02	0.24414063E-03
8	0.11774902E 01	0.66671112E 02	0.73242188E-03

FIGURE III-B.25 FORCE OUTPUT, ELEMENT NO. 2 - RECTANGULAR PRISM ELEMENT, CANTILEVER BEAM

F O R C E S F O R T H E R E C T A N G U L A R P R I S M E L E M E N T

LOAD CONDITION NUMBER	ELEMENT NUMBER	ELEMENT TYPE	ELEMENT GRID POINTS
1	3	52	4 12 16 8 3 11 15 7

APPARENT ELEMENT POINT	FORCES	FORCES
	FX	FZ
1	0.75988770E-02	-0.12512207E-02
2	-0.39520264E-02	0.16021729E-02
3	-0.57678223E-02	0.41198730E-02
4	0.18768311E-02	-0.58593750E-02
5	-0.11749878E-01	-0.24414063E-03
6	0.11782227E-01	0.24414063E-03
7	-0.11833191E-01	0.31633377E-02
8	0.11813965E-01	-0.26855469E-02

ELEMENT APPLIED POINT	FORCES	FORCES
	FY	FZ
1	0.0	0.0
2	0.0	0.0
3	0.0	0.0
4	0.0	0.0
5	0.0	0.0
6	0.0	0.0
7	0.0	0.0
8	0.0	0.0

NET ELEMENT POINT	FORCES	FORCES
	FX	FZ
1	0.75988770E-02	-0.12512207E-02
2	-0.39520264E-02	0.16021729E-02
3	-0.57678223E-02	0.41198730E-02
4	0.18768311E-02	-0.58593750E-02
5	-0.11749878E-01	-0.24414063E-03
6	0.11782227E-01	0.24414063E-03
7	-0.11833191E-01	0.31633377E-02
8	0.11813965E-01	-0.26855469E-02

FIGURE III-B.26 FORCE OUTPUT, ELEMENT NO. 3 - RECTANGULAR PRISM ELEMENT, CANTILEVER BEAM

## C. TETRAHEDRON ELEMENT

An eighteen element cantilever beam subjected to a constant pressure load is shown in Figure III-C.1 as the second example. This figure shows the loading, idealization, dimensions and material properties. The preprinted input data forms for this example problem is given in Figures III-C.2 to III-C.10.

Inspection of the figures shows that the input data is very similar to that given in the preceding example with the exception of the element pressure input data form, Figure III-C.9. On that form element related pressure data is recorded for each of the eighteen elements. The MODAL and REPEAT options are used to efficiently enter these data. The MODAL data indicates that a zero pressure is input for each of the four faces of each tetrahedron. The exceptions to this are given by the data cards following the MODAL inputs. In this particular example face 134 of elements, 1, 7 and 13 is pressurized and face 123 of elements 2, 8 and 14 is pressurized. It must be noted that the face numbers given above correspond to tetrahedron local numbering system.

The output supplied by the MAGIC III System for this example is described below and shown in Figures III-C.11 to III-C.27.

Figure III-C.11 displays the matrix abstraction instructions associated with this example. A complete description of these instructions is provided in Reference 5. Figures III-C.12 to III-C.16 show the output data obtained from the Structural Systems Monitor. These figures record the input data pertinent to the problem being solved.

An alternative means of obtaining the output shown in Figure III-C.12 to Figure III-C.27 is to use the .ANALIC. instruction sequence, Figure III-C.11A, in place of the standard STATICS AGENDUM shown on Figure III-C.11. Comparison of these two sets of abstraction instructions shows that the .ANALIC. sequence requires only two statements whereas the STATICS AGENDUM requires forty-five such statements. A considerable difference is evident.

# Contrails

Reference to Section II.B.4 of this report allows the reader to interpret the .ANALIC. instruction listing. To make use of .ANALIC. the User must input the following three cards; \$MAGIC, \$RUN-GO and \$INSTRUCTION-SOURCE. The next two cards contain the .ANALIC. instructions. The first card is identical to the first card in the standard STATICS AGENDUM of Figure III-C.11. The second card pertains to the .ANALIC. instruction and each entry (DISPL, STR, etc) is defined on pp 25 thru 27 of this report. In this example problem, the three scalar values KALC, NNOM and NRSLEM were suppressed and the default values were used. Table I Page 27 shows that the default for KALC results in the use of the Cholesky triangularization method for solution of the governing equations. The default value for NNOM is eight which means that a maximum number of eight grid points can be used to define the element. The default value is forty for NRSLEM. This entry indicates the maximum number of rows in the element stress matrix. Consultation of Table II page 28 shows that NNOM equals 4 and NRSLEM equals 6 for the tetrahedron element used in this example problem. These values could have been used in place of the default values.

It is emphasized that .ANALIC. should be utilized for problems which are of the size that can be executed entirely in core. Depending on the type of finite elements being employed, the upper limit in the MAGIC III System for .ANALIC. is approximately two-hundred reduced degrees-of-freedom.

Figure III-C.12 displays the problem title and material data output. The gridpoint coordinates, temperatures and pressures are given in Figure III-C.13. Boundary condition information and finite element description is shown on Figure III-C.14. In the boundary condition portion of the figure, zeros ('0') represent degrees of freedom that are fixed, (i.e. no motion) and ones ('1') represent

# Contrails

degrees of freedom that are free (have unknown values of displacement). Note that no condensation procedure is used in this problem hence twos ('2') are not used. The second last column accumulates the number of active degrees of freedom which in this problem is 36. The second portion of Figure III-C.14 shows the finite element description. Each of the eighteen elements is called out in turn with grid points, print options and material number. Note that neither grid points nor section properties are presented since these are not required for the tetrahedron element.

Element input pressures are given on the Element Pressure Table in Figure III-C.15 for those elements subjected to such pressures. Four columns of pressure data are presented and reflect the input pressure on tetrahedra faces 134, 234, 124 and 123 respectively. Note again that these face numbers refer to local coordinate systems.

Figure III-C.16 displays the external load condition and transformed external assembled load column. Note that all loads are of zero magnitude since the only loading present in this example is the pressure which is considered an element applied load and not an external load as such.

MAGIC III System output of final results are displayed in Figures III-C.17 to III-C.27. The stiffness matrix is shown in Figure III-C.17. Only the non-zero terms are displayed and it is presented row-wise. It's ordering is consistent with that of the boundary condition table.

In this problem the ordering is

$$\{\bar{U}\}^T = \left[ U_2, V_2, W_2, U_3, V_3, W_3, \dots, U_{16}, V_{16}, W_{16} \right]$$

with degrees of freedom  $U_1, V_1, W_1, U_5, V_5, W_5, U_9, V_9, W_9$  and  $U_{13}, V_{13}, W_{13}$  fixed.



# Contrails

The matrix of element applied loads (GPRINT OF MATRIX FTELA) is shown in Figure III-C.18. This represents the work equivalent loads due to element applied pressure. It is this force vector, defined at each grid point, which loads the structure. This figure shows that loads of varying magnitude are applied in the negative global Z direction. The next figure, Figure III-C.19, shows the externally applied load vector (GPRINT OF MATRIX LOADS) which as discussed in the previous paragraph, are of zero magnitude.

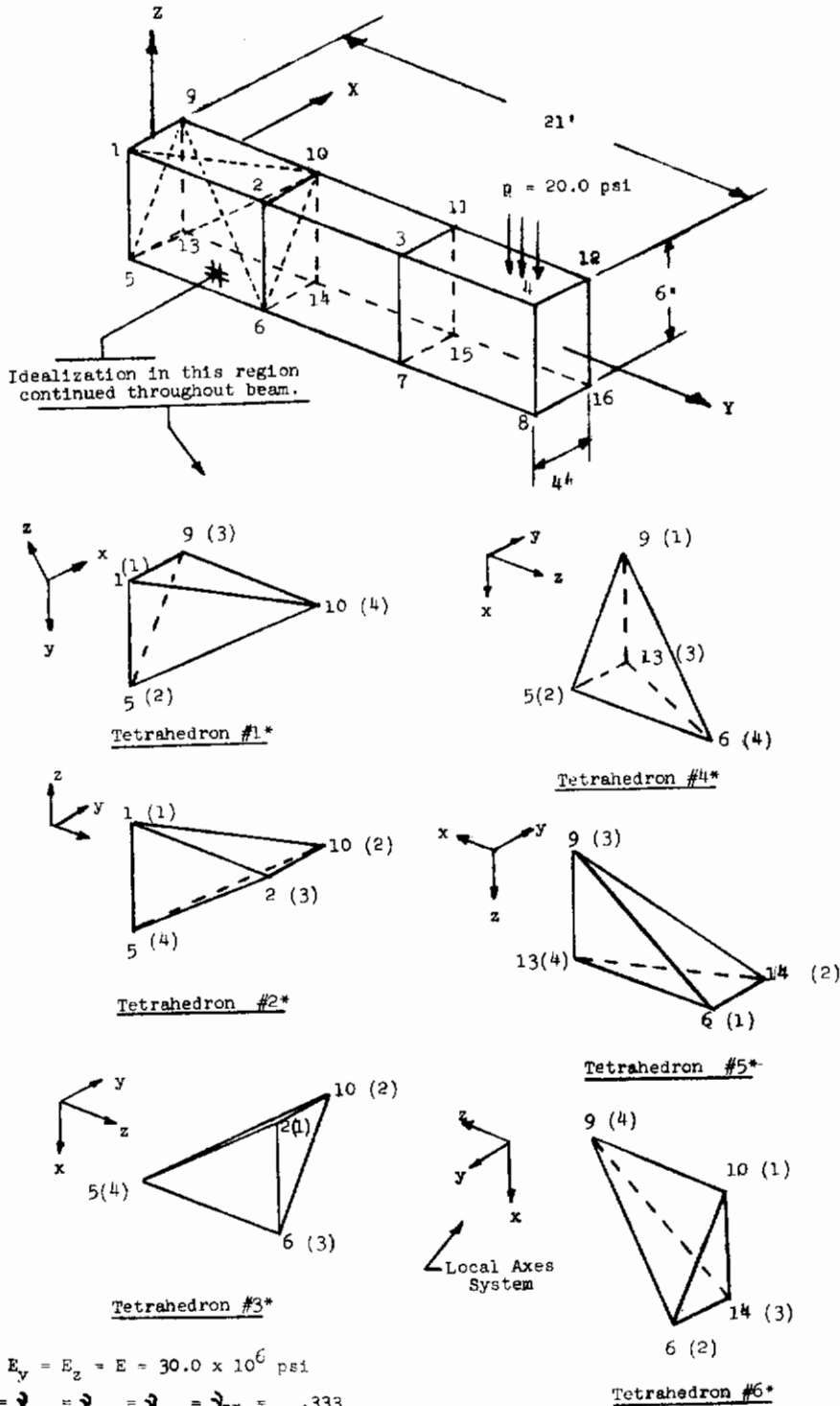
The displacements of the cantilever beam resulting from the above loads are presented in Figure III-C.20. It is noted that the displacements (U, V, W) are output corresponding to node point numbers and are referenced to the global axes. Figure III-C.21 shows the reactions ( $F_x$ ,  $F_y$ ,  $F_z$ ). These are also output corresponding to node point number and are referenced to the global axes unless otherwise specified.

The stresses arising in the structure are displayed in tabular form for each element. Typical results are presented in Figures III-C.22 to III-C.24 for elements 1, 7 and 18 respectively. Stresses are referenced to the global axes system and are defined for any point in the tetrahedron element since this element is a constant strain element. (See Reference 7). Normally the user will consider the stresses to be defined at the element's centroid, and the labeling (STRESSES EVALUATED AT ELEMENT CENTROID) reflects this consideration. Since no pre-strain or temperatures were considered in this problem the element applied stresses are zero and only the apparent element stresses are of significance. Thus the net element stresses and apparent element stresses are equal.

The last set of output is given in Figures III-C.25 to III-C.27 and consist of the global oriented element forces. Output labeling is analogous to the stress output labeling. The apparent element forces arise from the cantilever deformation and the element applied forces exist due to the element applied pressure. The force point 1, 2, 3,4, in this example correspond to element grid point numbers. For element number one, for example, force points 1,2,3,4 correspond to element grid points 1, 5, 9, 10 respectively.



# Contrails



$$E_x = E_y = E_z = E = 30.0 \times 10^6 \text{ psi}$$

$$\nu_{xy} = \nu_{yx} = \nu_{yz} = \nu_{zy} = \nu_{zx} = \nu_{xz} = .333$$

$$E_x = E_y = E_z = 0, T_1 = T_2 = T_3 = \dots T_{16} = 0.0$$

$p = 20.0 \text{ psi}$

FIGURE III-C.1 TETRAHEDRON ELEMENT - CANTILEVER BEAM WITH PRESSURE LOAD, EIGHTEEN ELEMENTS





## MAGIC STRUCTURAL ANALYSIS SYSTEM INPUT DATA FORMAT

### SYSTEM CONTROL INFORMATION

ENTER APPROPRIATE NUMBER, RIGHT ADJUSTED, IN BOX OPPOSITE APPLICABLE REQUESTS

	<table border="1" style="border-collapse: collapse; width: 100%;"> <tr> <td style="width: 12.5%;">S</td> <td style="width: 12.5%;">Y</td> <td style="width: 12.5%;">S</td> <td style="width: 12.5%;">T</td> <td style="width: 12.5%;">E</td> <td style="width: 12.5%;">M</td> </tr> <tr> <td style="text-align: center;">1</td> <td style="text-align: center;">2</td> <td style="text-align: center;">3</td> <td style="text-align: center;">4</td> <td style="text-align: center;">5</td> <td style="text-align: center;">6</td> </tr> </table>	S	Y	S	T	E	M	1	2	3	4	5	6	( / )		
S	Y	S	T	E	M											
1	2	3	4	5	6											
1. Number of System Grid Points	<table border="1" style="border-collapse: collapse; width: 100%;"> <tr> <td style="width: 12.5%;"></td> <td style="width: 12.5%;"></td> <td style="width: 12.5%;"></td> <td style="width: 12.5%;"></td> <td style="width: 12.5%; text-align: center;">16</td> <td style="width: 12.5%;"></td> </tr> <tr> <td style="text-align: center;">1</td> <td style="text-align: center;">2</td> <td style="text-align: center;">3</td> <td style="text-align: center;">4</td> <td style="text-align: center;">5</td> <td style="text-align: center;">6</td> </tr> </table>					16		1	2	3	4	5	6			
				16												
1	2	3	4	5	6											
2. Number of Input Grid Points	<table border="1" style="border-collapse: collapse; width: 100%;"> <tr> <td style="width: 12.5%;"></td> <td style="width: 12.5%;"></td> <td style="width: 12.5%;"></td> <td style="width: 12.5%;"></td> <td style="width: 12.5%; text-align: center;">16</td> <td style="width: 12.5%;"></td> </tr> <tr> <td style="text-align: center;">7</td> <td style="text-align: center;">8</td> <td style="text-align: center;">9</td> <td style="text-align: center;">10</td> <td style="text-align: center;">11</td> <td style="text-align: center;">12</td> </tr> </table>					16		7	8	9	10	11	12			
				16												
7	8	9	10	11	12											
3. Number of Degrees of Freedom/Grid Point	<table border="1" style="border-collapse: collapse; width: 100%;"> <tr> <td style="width: 12.5%;"></td> <td style="width: 12.5%; text-align: center;">3</td> </tr> <tr> <td style="text-align: center;">13</td> <td style="text-align: center;">14</td> </tr> </table>		3	13	14											
	3															
13	14															
4. Number of Load Conditions	<table border="1" style="border-collapse: collapse; width: 100%;"> <tr> <td style="width: 12.5%;"></td> <td style="width: 12.5%; text-align: center;">1</td> </tr> <tr> <td style="text-align: center;">15</td> <td style="text-align: center;">16</td> </tr> </table>		1	15	16											
	1															
15	16															
5. Number of Initially Displaced Grid Points	<table border="1" style="border-collapse: collapse; width: 100%;"> <tr> <td style="width: 12.5%;"></td> <td style="width: 12.5%;"></td> <td style="width: 12.5%;"></td> <td style="width: 12.5%;"></td> <td style="width: 12.5%; text-align: center;">0</td> <td style="width: 12.5%;"></td> </tr> <tr> <td style="text-align: center;">17</td> <td style="text-align: center;">18</td> <td style="text-align: center;">19</td> <td style="text-align: center;">20</td> <td style="text-align: center;">21</td> <td style="text-align: center;">22</td> </tr> </table>					0		17	18	19	20	21	22			
				0												
17	18	19	20	21	22											
6. Number of Prescribed Displaced Grid Points	<table border="1" style="border-collapse: collapse; width: 100%;"> <tr> <td style="width: 12.5%;"></td> <td style="width: 12.5%;"></td> <td style="width: 12.5%;"></td> <td style="width: 12.5%;"></td> <td style="width: 12.5%; text-align: center;">0</td> <td style="width: 12.5%;"></td> </tr> <tr> <td style="text-align: center;">23</td> <td style="text-align: center;">24</td> <td style="text-align: center;">25</td> <td style="text-align: center;">26</td> <td style="text-align: center;">27</td> <td style="text-align: center;">28</td> </tr> </table>					0		23	24	25	26	27	28			
				0												
23	24	25	26	27	28											
7. Number of Grid Point Axes Transformation Systems	<table border="1" style="border-collapse: collapse; width: 100%;"> <tr> <td style="width: 12.5%;"></td> <td style="width: 12.5%; text-align: center;">0</td> </tr> <tr> <td style="text-align: center;">29</td> <td style="text-align: center;">30</td> </tr> </table>		0	29	30											
	0															
29	30															
8. Number of Elements	<table border="1" style="border-collapse: collapse; width: 100%;"> <tr> <td style="width: 12.5%;"></td> <td style="width: 12.5%;"></td> <td style="width: 12.5%;"></td> <td style="width: 12.5%;"></td> <td style="width: 12.5%; text-align: center;">18</td> <td style="width: 12.5%;"></td> </tr> <tr> <td style="text-align: center;">31</td> <td style="text-align: center;">32</td> <td style="text-align: center;">33</td> <td style="text-align: center;">34</td> <td style="text-align: center;">35</td> <td style="text-align: center;">36</td> </tr> </table>					18		31	32	33	34	35	36			
				18												
31	32	33	34	35	36											
9. Number of Requests and/or Revisions of Material Tape.	<table border="1" style="border-collapse: collapse; width: 100%;"> <tr> <td style="width: 12.5%;"></td> <td style="width: 12.5%; text-align: center;">1</td> </tr> <tr> <td style="text-align: center;">37</td> <td style="text-align: center;">38</td> </tr> </table>		1	37	38											
	1															
37	38															
10. Number of Input Boundary Condition Points	<table border="1" style="border-collapse: collapse; width: 100%;"> <tr> <td style="width: 12.5%;"></td> <td style="width: 12.5%;"></td> <td style="width: 12.5%;"></td> <td style="width: 12.5%;"></td> <td style="width: 12.5%; text-align: center;">4</td> <td style="width: 12.5%;"></td> </tr> <tr> <td style="text-align: center;">39</td> <td style="text-align: center;">40</td> <td style="text-align: center;">41</td> <td style="text-align: center;">42</td> <td style="text-align: center;">43</td> <td style="text-align: center;">44</td> </tr> </table>					4		39	40	41	42	43	44			
				4												
39	40	41	42	43	44											
11. T <sub>0</sub> For Structure (With Decimal Point)	<table border="1" style="border-collapse: collapse; width: 100%;"> <tr> <td style="width: 12.5%;"></td> <td style="width: 12.5%;"></td> <td style="width: 12.5%;"></td> <td style="width: 12.5%;"></td> <td style="width: 12.5%; text-align: center;">0</td> <td style="width: 12.5%; text-align: center;">.</td> <td style="width: 12.5%; text-align: center;">0</td> </tr> <tr> <td style="text-align: center;">45</td> <td style="text-align: center;">46</td> <td style="text-align: center;">47</td> <td style="text-align: center;">48</td> <td style="text-align: center;">49</td> <td style="text-align: center;">50</td> <td style="text-align: center;">51</td> </tr> </table>					0	.	0	45	46	47	48	49	50	51	( / )
				0	.	0										
45	46	47	48	49	50	51										

FIGURE III-C.4 SYSTEM CONTROL INFORMATION - TETRAHEDRON ELEMENT, CANTILEVER BEAM















MAGIC STRUCTURAL ANALYSIS SYSTEM

INPUT DATA FORMAT

CHECK OR END CARD

C	H	E	C	K
1	2	3	4	5

 (/)

E	N	D
1	2	3

 (/)

FIGURE III-C.10 END CARD - TETRAHEDRON ELEMENT,  
CANTILEVER BEAM



MAGIC ABSTRACTION INSTRUCTION LISTING

TEST MAGIC

```

19 C          REACTS = KELA.MLLT.XO
20          REACTP= REACTS.SUBT.TLOAD
21          IF (DIFF.NULL.) GO TO 10

22 C          PRINT ELEMENT APPLIED LCADS, EXTERNAL LGADS, DISPLACEMENTS,
23          REACTIONS AND INVERSE CHECK IN ENGINEERING FORMAT
24 C          ELEMENTS HAVE 1 OR 2 DEGREES OF FREEDOM
25 C          GPR INTI 4, , , FX, FY, FZ, MX, MY, MZ, SC, TR IFTELA
26          GPR INTI 4, , , FX, FY, FZ, MX, MY, MZ, SC, ILCADS
27          GPR INTI 2, , , U, V, W, THETA, X, THEIAY, THETAZ, SC, X
28          GPR INTI 1, , , FX, FY, FZ, MZ, MY, MZ, SC, TR IREACTP
29          IF (I3=NULL.) GO TO 600

30 C          ELEMENTS HAVE 3 DEGREES OF FREEDOM
31 C          GPR INTI 4, , , FR, O, FZ, O, BETA, O, F1, O, F3, SC, TR IFTELA
32          GPR INTI 4, , , FR, O, FZ, O, BETA, O, F1, O, F3, SC, ILCADS
33          GPR INTI 2, , , U, O, b, O, THETA, O, b, O, b, , SC, X
34          GPR INTI 1, , , FR, O, FZ, O, BETA, O, F1, O, F3, SC, TR IREACTP

35 C          GENERATE STRESSES AND FORCES
36 C          STRESS=EM,XO .STRESS.(4,)
37          FORCEP=EM,XO .FORCE.(4,)

```

FIGURE III-C.11 CONCLUDED

MAGIC - MATRIX ANALYSIS VIA GENERATIVE AND INTERPRETIVE COMPUTATIONS

MAGIC

MAGIC PROBLEM SPECIFICATION DATA

\$RUN GO

MAGIC ABSTRACTION INSTRUCTION LISTING PAGE 1

\$INSTRUCTION	SOURCE
1	,MLIB,,XLD,TR,,KEL,FTEL,SEL,STEL,,,SC,EM,= ,,,.USER04.
2	DISPL,STR,FDR,REACT = TR,SC,EM,XLD,,,, .ANALIC.(,,)

FIGURE III-C.11A .ANALIC. ABSTRACTION INSTRUCTION LISTING

EIGHTEEN ELEMENT CANTILEVERED BEAM SUBJECTED TO A PRESSURE  
LOAD. TETRAHEDRON ELEMENT IDEN. NO.50 STATICS ANALYSIS.

REVISIONS OF MATERIAL TAPE

ASTERISK (\*) PRECEDING MATERIAL  
IDENTIFICATION INDICATES THAT INPUT  
ERROR RETURNS WILL NOT RESULT IN  
TERMINATION OF EXECUTION

```

*****
REVISION
MATERIAL NUMBER 1
MATERIAL IDENTIFICATION STEEL
NUMBER OF MATERIAL PROPERTY PCINTS. . . . 1
NUMBER OF PLASTIC PROPERTY PCINTS . . . . 0
MASS DENSITY. . . . . 0.73394994E-03
MATERIAL PROPERTIES
214
*****
YOUNG'S MODULI
DIRECTIONS
TEMPERATURE XX 0.30000E 08 YY 0.30000E 08 ZZ 0.30000E 08 XY 0.333000E 00 XZ 0.333000E 00
0.0 0.30000E 08 0.30000E 08 0.30000E 08 0.333000E 00 0.333000E 00
TM. EXP.COEF.
RIGIDITY MODULI
DIRECTIONS
TEMPERATURE XX 0.65000E-05 YY 0.65000E-05 ZZ 0.65000E-05 XY 0.112528E 08 XZ 0.112528E 08
0.0 0.65000E-05 0.65000E-05 0.65000E-05 0.112528E 08 0.112528E 08
*****
    
```

FIGURE III-C-12 TITLE AND MATERIAL DATA OUTPUT - TETRAHEDRON ELEMENT, CANTILEVER BEAM



16 REF. POINTS

NO. DIRECTIONS = 3 NC. DEGREES OF FREEDOM = 1

GRIDPOINT DATA  
(IN RECTANGULAR COORDINATES)

POINT	X	Y	Z	TEMPERATURES	PRESSURES
1	-0.2000000E 01	0.0	0.3000000E 01	0.0	0.0
2	-0.2000000E 01	-0.7000000E 01	0.3000000E 01	0.0	0.0
3	-0.2000000E 01	-0.1400000E 02	0.3000000E 01	0.0	0.0
4	-0.2000000E 01	-0.2100000E 02	0.3000000E 01	0.0	0.0
5	-0.2000000E 01	0.0	-0.3000000E 01	0.0	0.0
6	-0.2000000E 01	-0.7000000E 01	-0.3000000E 01	0.0	0.0
7	-0.2000000E 01	-0.1400000E 02	-0.3000000E 01	0.0	0.0
8	-0.2000000E 01	-0.2100000E 02	-0.3000000E 01	0.0	0.0
9	0.2000000E 01	0.0	0.3000000E 01	0.0	0.0
10	0.2000000E 01	-0.7000000E 01	0.3000000E 01	0.0	0.0
11	0.2000000E 01	-0.1400000E 02	0.3000000E 01	0.0	0.0
12	0.2000000E 01	-0.2100000E 02	0.3000000E 01	0.0	0.0
13	0.2000000E 01	0.0	-0.3000000E 01	0.0	0.0
14	0.2000000E 01	-0.7000000E 01	-0.3000000E 01	0.0	0.0
15	0.2000000E 01	-0.1400000E 02	-0.3000000E 01	0.0	0.0
16	0.2000000E 01	-0.2100000E 02	-0.3000000E 01	0.0	0.0

FIGURE III-C.13 GRIDPOINT DATA OUTPUT  
TETRAHEDRON ELEMENT,  
CANTILEVER BEAM

BOUNDARY CONDITION INFORMATION

MODES	DEGREES OF FREEDOM	NO. OF ONES	NO. OF TWOS
1	0	0	0
2	1	3	0
3	1	6	0
4	1	9	0
5	0	9	0
6	1	12	0
7	1	15	0
8	1	16	0
9	1	18	0
10	0	21	0
11	1	24	0
12	1	27	0
13	0	27	0
14	1	30	0
15	1	33	0
16	1	36	0

TOTAL NO. ELEMENTS = 18

ELEM TYPE	MAT. NO.	CJSE	TEMP.	PRNT	NO.	GRID POINTS	EXTRA GRID PTS	SECTION PROPERTIES
1	50	1	4	0	4	1 5 9 10	0	0.0
7	50	1	4	0	4	2 6 10 11	0	0.0
13	50	1	4	0	4	3 7 11 12	0	0.0
2	50	1	4	0	4	1 10 2 5	0	0.0
8	50	1	4	0	4	2 11 3 6	0	0.0
14	50	1	4	0	4	3 12 4 7	0	0.0
3	50	1	4	0	4	2 10 6 5	0	0.0
4	50	1	4	0	4	9 5 13 6	0	0.0
5	50	1	4	0	4	6 14 19 13	0	0.0
6	50	1	4	0	4	10 6 14 9	0	0.0
9	50	1	4	0	4	3 11 7 6	0	0.0
10	50	1	4	0	4	10 6 14 7	0	0.0
11	50	1	4	0	4	7 15 10 14	0	0.0
12	50	1	4	0	4	11 7 15 10	0	0.0
15	50	1	4	0	4	4 12 8 7	0	0.0
16	50	1	4	0	4	11 7 15 8	0	0.0
17	50	1	4	0	4	8 16 11 15	0	0.0
18	50	1	4	0	4	12 8 16 11	0	0.0

FIGURE III-C.14 BOUNDARY CONDITION & FINITE ELEMENT DESCRIPTION - TETRAHEDRON ELEMENT, CANTILEVER BEAM

ELEMENT PRESSURE TABLE

ELEM NO. OF PRESS. LIST OF PRESSURES

ELEM	NO. OF PRESS.	LIST OF PRESSURES					
1	1	-0.200000E 02					
7	1	-0.200000E 02					
13	1	-0.200000E 02					
2	4	0.0	0.0	0.0	0.0	0.0	-0.200000E 02
8	4	0.0	0.0	0.0	0.0	0.0	-0.200000E 02
14	4	0.0	0.0	0.0	0.0	0.0	-0.200000E 02
3	4	0.0	0.0	0.0	0.0	0.0	0.0
4	4	0.0	0.0	0.0	0.0	0.0	0.0
5	4	0.0	0.0	0.0	0.0	0.0	0.0
6	4	0.0	0.0	0.0	0.0	0.0	0.0
9	4	0.0	0.0	0.0	0.0	0.0	0.0
10	4	0.0	0.0	0.0	0.0	0.0	0.0
11	4	0.0	0.0	0.0	0.0	0.0	0.0
12	4	0.0	0.0	0.0	0.0	0.0	0.0
15	4	0.0	0.0	0.0	0.0	0.0	0.0
16	4	0.0	0.0	0.0	0.0	0.0	0.0
17	4	0.0	0.0	0.0	0.0	0.0	0.0
18	4	0.0	0.0	0.0	0.0	0.0	0.0

FIGURE III-C.15 ELEMENT PRESSURE TABLE OUTPUT - TETRAHEDRON ELEMENT, CANTILEVER BEAM



		CUTOFF = 0.0		SIZE 36 BY 36		PAGE 1				
DISP	FORCE	FORCE	FORCE	FORCE	FORCE	FORCE	FORCE			
1	1	0.279072E 09	2	0.336910E 08	3	-0.786124E 08	4	-0.643016E 07	5	-0.112528E 08
6	6	-0.131283E 08	10	-0.262565E 08	11	0.224382E 08	12	0.523558E 08	19	-0.235955E 09
20	20	-0.336910E 08	21	0.262566E 08	22	-0.203010E 02	23	0.336910E 08	24	0.131283E 08
2	1	0.336910E 08	2	0.136698E 09	3	0.224606E 08	4	-0.224382E 08	5	-0.256821E 08
6	6	-0.750187E 07	10	0.112528E 08	11	-0.262565E 08	12	-0.224607E 08	19	-0.336910E 08
20	20	-0.590772E 08	21	0.750187E 07	22	0.336910E 08	23	-0.853910E 01	24	-0.750187E 07
3	1	-0.786124E 08	2	0.224606E 08	3	0.176806E 09	4	-0.261779E 08	5	-0.149588E 08
6	6	-0.643015E 07	10	0.262566E 08	11	-0.224607E 08	12	-0.104869E 08	19	0.523558E 08
20	20	0.149588E 08	21	-0.590772E 08	22	0.261779E 08	23	-0.149588E 08	24	-0.858691E 01
4	1	-0.643016E 07	2	-0.224382E 08	3	-0.261779E 08	4	0.275072E 08	5	0.336909E 08
6	6	-0.786124E 08	7	-0.643017E 07	8	-0.112528E 08	9	-0.131283E 08	10	-0.447149E 08
11	11	-0.224382E 08	12	0.261779E 08	13	-0.262565E 08	14	0.224382E 08	15	0.523558E 08
22	22	-0.235959E 09	23	-0.336910E 08	24	0.262566E 08	25	0.269889E 01	26	0.336910E 08
27	27	0.131283E 08								
5	1	-0.112528E 08	2	-0.256821E 08	3	-0.149588E 08	4	0.336909E 08	5	0.136698E 08
6	6	0.224606E 08	7	-0.224382E 08	8	-0.256821E 08	9	-0.750187E 07	10	-0.112528E 08
11	11	0.758540E 01	12	0.224606E 08	13	0.112528E 08	14	-0.224607E 08	15	-0.224607E 08
22	22	-0.336909E 08	23	-0.590772E 08	24	0.750187E 07	25	0.336910E 08	26	0.140790E 01
27	27	-0.750187E 07								
6	1	-0.131283E 08	2	-0.750186E 07	3	-0.643015E 07	4	-0.786124E 08	5	0.224606E 08
6	6	0.176806E 09	7	-0.261779E 08	8	-0.149588E 08	9	-0.643014E 07	10	0.131283E 08
11	11	0.224606E 08	12	-0.981847E 01	13	0.262566E 08	14	-0.224607E 08	15	-0.104869E 09
22	22	0.523558E 08	23	0.149588E 08	24	-0.590772E 08	25	0.261779E 08	26	-0.149588E 08
27	27	0.120702E 02								
7	4	-0.643017E 07	5	-0.224382E 08	6	-0.261779E 08	7	0.172488E 08	8	0.336910E 08
9	9	-0.393062E 08	13	-0.593382E 01	14	-0.224382E 08	15	0.261779E 08	16	-0.875218E 07
17	17	0.224382E 08	18	0.261779E 08	25	-0.157303E 09	26	-0.112528E 08	27	0.131283E 08
8	4	-0.112528E 08	5	-0.256821E 08	6	-0.149588E 08	7	0.336910E 08	8	0.738192E 08
9	9	-0.133876E 02	13	-0.112528E 08	14	-0.119320E 02	15	0.224607E 08	16	0.112528E 08
17	17	-0.875218E 07	18	-0.750187E 07	25	-0.224382E 08	26	-0.393848E 08	27	-0.419769E-06
9	4	-0.131283E 08	5	-0.750186E 07	6	-0.643016E 07	7	-0.393062E 08	8	-0.133876E 02
9	9	0.807713E 08	13	0.131283E 08	14	0.224607E 08	15	-0.147051E 02	16	0.131283E 08
17	17	-0.149588E 08	18	-0.349563E 08	25	0.261779E 08	26	-0.837024E-04	27	-0.393848E 08

FIGURE III-C.17 STIFFNESS MATRIX OUTPUT - TETRAHEDRON ELEMENT, CANTILEVER BEAM

MATRIX STIFF / 1/

CUTOFF = 0.0

SIZE 36 BY 36

DISP	FORCE	FORCE	FORCE	FORCE	FORCE	FORCE	FORCE	FORCE	FORCE	
10	1	-0.262565E 08	2	0.112528E 08	3	0.262566E 08	4	-0.447149E 08	5	-0.112528E 08
	6	0.131283E 08	10	0.287932E 09	11	-0.328910E 08	13	-0.128603E 08	14	0.224382E 08
	19	-0.368896E 01	20	0.111854E 08	21	-0.786123E 08	22	0.195624E 01	23	0.112528E 08
	24	-0.131283E 08	28	-0.235955E 08	29	0.336910E 08	30	0.523558E 01		
11	1	0.224382E 08	2	-0.262565E 08	3	-0.224607E 08	4	-0.224382E 08	5	0.758540E 01
	6	0.224606E 08	10	-0.336910E 08	11	0.188062E 09	12	0.224607E 08	13	0.112528E 08
	14	-0.513643E 08	15	-0.149588E 08	19	-0.111854E 08	20	-0.147337E 02	21	0.745692E 07
	22	0.224382E 08	23	0.547051E 01	24	0.750187E 07	28	0.336910E 08	29	-0.990772E 08
	30	-0.149588E 08								
12	1	0.523558E 08	2	-0.224606E 08	3	-0.104869E 09	4	0.261779E 01	5	0.224606E 08
	6	-0.981847E 01	11	0.224607E 08	12	0.189667E 09	14	-0.750187E 07	15	-0.128603E 08
	19	-0.786123E 08	20	-0.745692E 07	21	-0.368896E 01	22	-0.261779E 08	23	0.149588E 08
	24	0.429909E 01	28	0.262566E 08	29	-0.750187E 07	30	-0.590772E 08		
13	4	-0.262565E 08	5	0.112528E 08	6	0.262566E 08	7	-0.533382E 01	8	-0.112528E 08
	9	0.131283E 08	10	-0.128603E 08	11	0.112528E 08	13	0.287932E 01	14	-0.336910E 08
	15	-0.117142E 01	16	-0.128603E 08	17	0.224382E 08	18	0.936195E 01	19	0.162090E 01
	20	-0.224382E 08	21	-0.261779E 08	22	-0.702849E 01	23	0.111854E 08	24	-0.786123E 08
	25	-0.368539E 01	26	0.112528E 08	27	-0.131283E 08	28	-0.203010E 02	29	-0.336910E 08
	30	0.261779E 08	31	-0.235955E 09	32	0.336909E 08	33	0.523558E 01		
14	4	0.224382E 08	5	-0.262565E 08	6	-0.224607E 08	7	-0.224382E 08	8	-0.119320E 02
	9	0.224607E 08	10	0.224382E 08	11	-0.513643E 08	12	-0.750187E 07	13	-0.336910E 08
	14	0.188062E 09	15	0.224606E 08	16	0.112528E 08	17	-0.513643E 08	18	-0.149588E 08
	19	-0.112528E 08	20	0.413108E 01	21	-0.149588E 08	22	-0.111854E 08	23	0.702849E 01
	24	0.745692E 07	25	0.224382E 08	26	-0.446945E 01	27	0.750187E 07	28	-0.336910E 08
	29	-0.653910E 01	30	0.149588E 08	31	0.336910E 08	32	-0.590772E 08	33	-0.149588E 08
15	4	0.523558E 08	5	-0.224607E 08	6	-0.104869E 09	7	0.261779E 01	8	0.224607E 08
	9	-0.147051E 02	11	-0.149588E 08	12	-0.128603E 08	13	-0.117142E 01	14	0.224606E 08
	15	0.189667E 09	16	0.669504E 01	17	-0.750187E 07	18	-0.128603E 08	19	-0.131283E 08
	20	-0.750187E 07	21	0.386373E 01	22	-0.786123E 08	23	-0.745692E 07	24	-0.238419E-08
	25	-0.261779E 08	26	0.149588E 08	27	-0.131283E 08	28	0.131283E 08	29	0.750187E 07
	30	-0.859891E 01	31	0.262566E 08	32	-0.750187E 07	33	-0.590772E 08		
16	7	-0.875218E 07	8	0.112528E 08	9	0.131283E 08	13	-0.128603E 08	14	0.112528E 08
	15	0.469504E 01	16	0.178916E 09	17	0.527137E 01	22	-0.346539E 01	23	-0.224382E 08
	24	-0.261779E 08	26	0.111854E 08	27	-0.393062E 08	31	0.269889E 01	32	-0.336910E 08
	33	0.261779E 08	34	-0.157303E 09	35	0.224382E 08	36	0.261779E 08		
17	7	0.224382E 08	8	-0.875218E 07	9	-0.149588E 08	13	0.224382E 08	14	-0.513643E 08
	15	-0.750187E 07	16	0.527137E 01	17	0.995013E 08	18	0.224606E 08	22	-0.112528E 08
	23	-0.446945E 01	24	-0.149588E 08	25	-0.111854E 08	31	-0.336910E 08	32	0.140790E 01
	33	0.149588E 08	34	0.112528E 08	35	-0.393062E 08	36	0.837024E-08		

FIGURE III-C-17 CONTINUED



CUTOFF = 0.0

DISP	FORCE	FORCE	FORCE	FORCE	FORCE	SIZE	SIZE	36 BY	36	FORCE	FORCE
18	7	0.261779E 08	8	-0.750187E 07	9	13	13	0.936195E 01	14	14	-0.149588E 08
	15	-0.128603E 08	17	0.224604E 08	18	22	22	-0.131283E 01	23	23	-0.750187E 07
	24	-0.128367E 02	25	-0.393062E 08	31	32	32	0.750187E 07	33	33	0.120702E 02
	34	0.131283E 08	35	0.419769E-06	36						
19	1	-0.235959E 09	2	-0.336910E 08	3	10	10	-0.368894E 01	11	11	-0.111854E 08
	12	-0.786123E 08	13	0.162090E 01	14	15	15	-0.131283E 01	19	19	0.267932E 09
	20	0.336909E 08	22	-0.128603E 08	23	28	28	-0.262565E 01	29	29	-0.112528E 08
	30	0.262566E 08	31	-0.782508E 00	32	33	33	0.131283E 01			
20	1	-0.336910E 08	2	-0.590772E 08	3	10	10	0.111854E 01	11	11	-0.147337E 02
	12	-0.745692E 07	13	-0.224382E 08	14	15	15	-0.750187E 07	19	19	0.336909E 08
	20	0.188062E 09	21	-0.224607E 08	22	23	23	-0.513643E 01	24	24	0.149588E 08
	28	-0.224382E 08	29	-0.262565E 08	30	31	31	0.224607E 08	32	32	0.624597E 01
	33	-0.224606E 08									
21	1	0.262566E 08	2	0.750187E 07	3	10	10	-0.786123E 01	11	11	0.745692E 07
	12	-0.368894E 01	13	-0.261779E 08	14	15	15	0.396373E 01	20	20	-0.224607E 08
	21	0.189467E 09	23	0.750187E 07	24	28	28	0.523558E 01	29	29	0.224607E 08
	30	-0.104869E 09	31	0.261779E 08	32	33	33	-0.101538E 02			
22	1	-0.203010E 02	2	0.336910E 08	3	4	4	-0.235959E 01	5	5	-0.336909E 08
	6	0.523558E 08	10	0.195626E 01	11	12	12	-0.261779E 01	13	13	-0.702849E 01
	14	-0.111854E 08	15	-0.786123E 08	16	17	17	-0.112528E 01	18	18	-0.131283E 08
	19	-0.128603E 08	20	-0.112528E 08	22	23	23	0.336909E 01	24	24	-0.117142E 01
	25	-0.128603E 08	26	-0.224382E 08	27	31	31	-0.262565E 01	32	32	-0.112528E 08
	33	0.262566E 08	34	-0.361748E 01	35	36	36	0.131283E 01			
23	1	0.336910E 08	2	-0.853910E 01	3	4	4	-0.336910E 01	5	5	-0.590772E 08
	6	0.149588E 08	10	0.112528E 08	11	12	12	0.149588E 01	13	13	0.111854E 08
	14	0.702849E 01	15	-0.745692E 07	16	17	17	-0.446945E 01	18	18	-0.750187E 07
	19	-0.224382E 08	20	-0.513643E 08	21	22	22	0.750187E 07	23	23	0.188062E 09
	24	-0.224606E 08	25	-0.112528E 08	26	27	27	-0.513643E 08	31	31	-0.224382E 08
	32	-0.262565E 08	33	0.224606E 08	34	35	35	-0.427811E 01	36	36	-0.224607E 08
24	1	0.131283E 08	2	-0.750187E 07	3	4	4	0.262566E 01	5	5	0.750187E 07
	6	-0.590772E 08	10	-0.131283E 08	11	12	12	0.429909E 01	13	13	-0.786123E 08
	14	0.745692E 07	15	-0.238419E-05	16	17	17	-0.149588E 01	18	18	-0.128367E 02
	20	0.149588E 08	21	-0.128603E 08	22	23	23	-0.224606E 01	24	24	0.189667E 09
	25	0.469504E 01	26	0.750187E 07	27	31	31	0.523558E 01	32	32	0.224606E 08
	33	-0.104869E 09	34	0.261779E 08	35	36	36	-0.127888E 02			
25	4	0.269885E 01	5	0.336910E 08	6	7	7	-0.157303E 01	8	8	-0.224382E 08
	9	0.261779E 08	13	-0.346539E 01	14	15	15	-0.261779E 01	17	17	-0.111854E 08
	18	-0.393062E 08	22	-0.128603E 08	23	24	24	0.469504E 01	25	25	0.178916E 09
	26	-0.527137E 01	34	-0.675219E 07	35	36	36	-0.112528E 08			

FIGURE III-C.17 CONTINUED



MATRIX STIFF / 1/

CUTOFF = 0.0

SIZE 36 BY 36

DISP	FORCE	FORCE	FORCE	FORCE	FORCE	FORCE	FORCE	FORCE	FORCE		
26	4	0.336910E 08	5	0.140790E 01	6	-0.149588E 08	7	-0.112528E 01	8	-0.393848E 08	
	9	-0.837024E-06	13	0.112528E 08	14	-0.446945E 01	15	0.149588E 08	16	0.111854E 08	
	22	-0.224382E 08	23	-0.513643E 08	24	0.750187E 07	25	-0.527137E 01	26	0.995013E 08	
	27	-0.224606E 08	34	-0.224382E 08	35	-0.875219E 07	36	0.149588E 08			
DISP	27	4	0.131283E 08	5	-0.750187E 07	6	0.120702E 02	7	0.131283E 08	8	-0.419769E-06
	9	-0.393848E 08	13	-0.131283E 08	14	0.750187E 07	15	-0.128367E 02	16	-0.393062E 08	
	22	0.936195E 01	23	0.149588E 08	24	-0.128603E 08	26	-0.224606E 01	27	0.872014E 08	
	34	0.261779E 08	35	0.750187E 07	36	-0.349563E 08					
DISP	28	10	-0.235935E 09	11	0.336910E 08	12	0.262566E 08	13	-0.203010E 02	14	-0.336910E 08
	15	0.131283E 08	19	-0.262565E 08	20	-0.224382E 08	21	0.523558E 08	28	0.275072E 09	
	29	-0.336910E 08	30	-0.786124E 08	31	-0.643016E 07	32	0.112528E 01	33	-0.131283E 08	
DISP	29	10	0.336910E 08	11	-0.590772E 08	12	-0.750187E 07	13	-0.336910E 08	14	-0.853910E 01
	15	0.750187E 07	19	-0.112528E 08	20	-0.262565E 08	21	0.224606E 08	28	-0.336910E 08	
	29	0.136698E 09	30	-0.224606E 08	31	0.224382E 08	32	-0.256821E 08	33	0.750186E 07	
DISP	30	10	0.523558E 08	11	-0.149588E 08	12	-0.590772E 08	13	0.261779E 08	14	0.149588E 08
	15	-0.858691E 01	19	0.262566E 08	20	0.224607E 08	21	-0.104869E 05	28	-0.786124E 08	
	29	-0.224606E 08	30	0.176806E 09	31	-0.261779E 08	32	0.149588E 08	33	-0.643015E 07	
DISP	31	13	-0.235935E 09	14	0.336910E 08	15	0.262566E 08	16	0.269885E 01	17	-0.336910E 08
	18	0.131283E 08	19	-0.782508E 00	20	0.224382E 08	21	0.261779E 08	22	-0.262565E 08	
	23	-0.224382E 08	24	0.523558E 08	28	-0.643016E 07	29	0.224382E 08	30	-0.261779E 08	
	31	0.275072E 09	32	-0.336909E 08	33	-0.786124E 08	34	-0.643017E 01	35	0.112528E 08	
	36	-0.131283E 08									
DISP	32	13	0.336909E 08	14	-0.590772E 08	15	-0.750187E 07	16	-0.336910E 08	17	0.140790E 01
	18	0.750187E 07	19	0.112528E 08	20	0.624597E 01	21	-0.224606E 08	22	-0.112528E 08	
	23	-0.262565E 08	24	0.224606E 08	28	0.112528E 08	29	-0.256821E 08	30	0.149588E 08	
	31	-0.336909E 08	32	0.136698E 09	33	-0.224606E 08	34	0.224382E 08	35	-0.256821E 08	
	36	0.750186E 07									
DISP	33	13	0.523558E 08	14	-0.149588E 08	15	-0.590772E 08	16	0.261779E 08	17	0.149588E 08
	18	0.120702E 02	19	0.131283E 08	20	-0.224606E 08	21	-0.101538E 02	22	0.262566E 08	
	23	0.224606E 08	24	-0.104869E 09	28	-0.131283E 08	29	0.750186E 07	30	-0.643015E 07	
	31	-0.786124E 08	32	-0.224606E 08	33	0.176806E 09	34	-0.261779E 08	35	0.149588E 08	
	36	-0.643014E 07									
DISP	34	14	-0.157303E 09	17	0.131283E 08	18	0.131283E 08	22	-0.341748E 01	23	0.224382E 08
	24	0.261779E 08	25	-0.875219E 07	28	-0.224382E 08	27	0.261779E 08	31	-0.643017E 07	
	32	0.224382E 08	33	-0.261779E 08	34	0.172486E 09	35	-0.336910E 01	36	-0.393062E 08	

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FIGURE III-C.17 CONTINUED

MATRIX STIFF / 1/

CUTOFF = 0.0

DISP	FORCE	35	16	0.224382E 08	17	-0.393848E 08	18	0.419769E-06	22	0.112528E 08	23	-0.427811E 01
		24	-0.224607E 08	25	-0.112528E 08	26	-0.875219E 07	27	0.750187E 07	31	0.112528E 08	
		32	-0.256821E 08	33	0.149588E 08	34	-0.336910E 08	35	0.738192E 06	36	0.669380E 01	
DISP		36	16	0.261779E 08	17	0.837034E-06	18	-0.393848E 08	22	0.131283E 06	23	-0.224607E 08
		24	-0.127888E 02	25	0.131283E 08	26	0.149588E 08	27	-0.349563E 08	31	-0.131283E 08	
		32	0.750186E 07	33	-0.643016E 07	34	-0.393848E 08	35	0.669380E 01	36	0.807713E 08	

FIGURE III-C.17 CONCLUDED

GPRINT OF MATRIX FT ELA (SET 1)

ROW	FX	FY	FZ
1	0.0	0.0	-0.18666650 E 03
2	0.0	0.0	-0.27999976 E 03
3	0.0	0.0	-0.27999976 E 03
4	0.0	0.0	-0.93333252 E 02
5	0.0	0.0	0.0
6	0.0	0.0	0.0
7	0.0	0.0	0.0
8	0.0	0.0	0.0
9	0.0	0.0	-0.93333252 E 02
10	0.0	0.0	-0.27999976 E 03
11	0.0	0.0	-0.27999976 E 03
12	0.0	0.0	-0.18666650 E 03
13	0.0	0.0	0.0
14	0.0	0.0	0.0
15	0.0	0.0	0.0
16	0.0	0.0	0.0

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FIGURE III-C.18 GPRINT OF MATRIX LOADS - TETRAHEDRON ELEMENT, CANTILEVER BEAM

MATRIX LOADS SIZE 48 BY 1  
NULL MATRIX

FIGURE III-C.19 MATRIX LOADS - TETRAHEDRON ELEMENT, CANTILEVER BEAM

DISPLACEMENT MATRIX FOR LOAD CONDITION 1

48 X 1

ROW	U	V	W
1	0.0	0.0	0.0
2	0.11979033E-04	-0.24384004E-04	-0.51589872E-04
3	0.30256560E-04	-0.33360440E-04	-0.12496015E-03
4	0.50328206E-04	-0.35343779E-04	-0.19549463E-03
5	0.0	0.0	0.0
6	0.91729371E-05	0.15122847E-04	-0.50923380E-04
7	0.27327595E-04	0.20157197E-04	-0.12367318E-03
8	0.47134061E-04	0.20220563E-04	-0.19335645E-03
9	0.0	0.0	0.0
10	0.10999482E-04	-0.15932798E-04	-0.55194657E-04
11	0.29959934E-04	-0.22020060E-04	-0.12696841E-03
12	0.50579853E-04	-0.23572582E-04	-0.19824316E-03
13	0.0	0.0	0.0
14	0.11155033E-04	0.24179215E-04	-0.50296454E-04
15	0.28405630E-04	0.31450443E-04	-0.12420131E-03
16	0.47979382E-04	0.31560735E-04	-0.19497190E-03

FIGURE III-C.20 DISPLACEMENT MATRIX - TETRAHEDRON ELEMENT, CANTILEVER BEAM

REACTIONS AND INVERSE CHECK FOR LOAD CONDITION 1

ROW	FX	FY	FZ
1	-0.34702160E 03	0.15006333E 04	0.10793909E 04
2	-0.17333984E-01	0.14648438E-01	0.25634766E-01
3	-0.40039063E-01	0.32226563E-01	0.57861328E-01
4	-0.25146484E-01	0.48828125E-03	0.20751953E-01
5	0.34203149E 03	-0.15005569E 04	-0.20247501E 03
6	-0.95214844E-02	-0.17822266E-01	0.51269531E-02
7	0.0	-0.19531250E-01	0.35156250E-01
8	0.78125000E-02	-0.78125000E-02	0.15625000E-01
9	0.32235352E 03	0.14390151E 04	-0.54902991E 02
10	0.29296875E-02	0.58593750E-02	0.23925781E-01
11	-0.39062500E-02	0.0	0.95214844E-02
12	0.20507813E-01	-0.63476563E-02	-0.18183994E-01
13	-0.31737866E 03	-0.14390593E 04	0.85943945E 03
14	0.18310547E-01	-0.12201031E-01	-0.10253906E-01
15	0.23437500E-01	-0.12695313E-01	-0.21728516E-01
16	0.39062500E-02	0.28432876E-02	-0.31250000E-01

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FIGURE III-C.21 REACTION MATRIX - TETRAHEDRON ELEMENT, CANTILEVER BEAM

STRESSES FOR THE TETRAHEDRON ELEMENT

(STRESSES EVALUATED AT THE ELEMENT CENTROID)

LOAD CONDITION NUMBER	ELEMENT NUMBER	ELEMENT TYPE	ELEMENT GRID POINTS
1	1	50	1 5 9 10
APPLIED ELEMENT STRESSES			
STRESS POINT	SIGMA-X	SIGMA-Z	SIGMA-YZ
1	0.49789597E 02	0.49789597E 02	0.85512802E 02
MEMBRANE STRESSES			
STRESS POINT	SIGMA-X	SIGMA-YZ	SIGMA-ZX
1	0.0	0.0	0.0
MEMBRANE STRESSES			
STRESS POINT	SIGMA-X	SIGMA-YZ	SIGMA-ZX
1	0.0	0.0	0.0
NET ELEMENT STRESSES			
STRESS POINT	SIGMA-X	SIGMA-YZ	SIGMA-ZX
1	0.49789597E 02	0.49789597E 02	0.85512802E 02

227 FIGURE III-C-22 STRESS OUTPUT, ELEMENT NO. 1 - TETRAHEDRON ELEMENT, CANTILEVER BEAM

STRESSES FOR THE TETRAHEDRON ELEMENT  
(STRESSES EVALUATED AT THE ELEMENT CENTROID)

LOAD CONDITION NUMBER	ELEMENT NUMBER	ELEMENT TYPE	ELEMENT GRID POINTS		
1	7	50	2 6 10 11		
APPARENT ELEMENT STRESSES					
STRESS POINT	SIGMA-X	SIGMA-Y	SIGMA-Z	SIGMA-XY	SIGMA-YZ
1	0.28019104E 01	0.31420776E 02	0.80636597E 01	-0.62224417E 01	0.44468930E 02
MEMBRANE STRESSES					
	SIGMA-X	SIGMA-Y	SIGMA-Z	SIGMA-XY	SIGMA-YZ
	0.0	0.0	0.0	0.0	0.0
ELEMENT APPLIED STRESSES					
STRESS POINT	SIGMA-X	SIGMA-Y	SIGMA-Z	SIGMA-XY	SIGMA-YZ
1	0.0	0.0	0.0	0.0	0.0
NET ELEMENT STRESSES					
STRESS POINT	SIGMA-X	SIGMA-Y	SIGMA-Z	SIGMA-XY	SIGMA-YZ
1	0.28019104E 01	0.31420776E 02	0.80636597E 01	-0.62224417E 01	0.44468930E 02
MEMBRANE STRESSES					
	SIGMA-X	SIGMA-Y	SIGMA-Z	SIGMA-XY	SIGMA-YZ
	0.0	0.0	0.0	0.0	0.0

FIGURE III-C-23 STRESS OUTPUT, ELEMENT NO. 7 - TETRAHEDRON ELEMENT, CANTILEVER BEAM



STRESSES FOR THE TETRAHEDRON ELEMENT  
(STRESSES EVALUATED AT THE ELEMENT CENTROID)

LOAD CONDITION NUMBER	ELEMENT ALBERP	ELEMENT TYPE	ELEMENT GRID POINTS				
1	18	50	12 8 16 11				
APPARENT ELEMENT STRESSES							
STRESS POINT 1	SIGMA-X 0.35231018E 01	SIGMA-Y 0.50445557E 01	SIGMA-Z -0.13503021E 02	SIGMA-XY -0.12101746E 00	SIGMA-YZ 0.97680841E 01	SIGMA-ZX 0.33227539E 00	
ELEMENT APPLIED STRESSES							
STRESS POINT 1	SIGMA-X 0.0	SIGMA-Y 0.0	SIGMA-Z 0.0	SIGMA-XY 0.0	SIGMA-YZ 0.0	SIGMA-ZX 0.0	
NET ELEMENT STRESSES							
STRESS POINT 1	SIGMA-X 0.35231018E 01	SIGMA-Y 0.50445557E 01	SIGMA-Z -0.13503021E 02	SIGMA-XY -0.12101746E 00	SIGMA-YZ 0.97680841E 01	SIGMA-ZX 0.33227539E 00	

FIGURE III-C.24 STRESS OUTPUT, ELEMENT NO. 18 - TETRAHEDRON ELEMENT, CANTILEVER BEAM

FORCES FOR THE TETRAHEDRON ELEMENT

LOAD CONDITION NUMBER 1 ELEMENT NUMBER 1 ELEMENT TYPE 50 ELEMENT GRID POINTS 1 5 9 10

APPARENT ELEMENT POINT	FX	FY	FZ
1	-0.34852686E 03	0.51833325E 03	0.23235138E 03
2	0.0	-0.39905933E 03	-0.23235138E 03
3	0.28037012E 03	0.27964063E 03	0.34205103E 03
4	0.68156555E 02	-0.39891479E 03	-0.34205103E 03

ELEMENT APPLIED POINT	FX	FY	FZ
1	0.0	0.0	-0.93333252E 02
2	0.0	0.0	0.0
3	0.0	0.0	-0.93333252E 02
4	0.0	0.0	-0.93333252E 02

NET ELEMENT POINT	FX	FY	FZ
1	-0.34852686E 03	0.51833325E 03	0.32560457E 03
2	0.0	-0.39905933E 03	-0.23235138E 03
3	0.28037012E 03	0.27964063E 03	0.43538428E 03
4	0.68156555E 02	-0.39891479E 03	-0.2487177E 03

FIGURE III-C.25 FORCE OUTPUT, ELEMENT NO. 1 - TETRAHEDRON ELEMENT, CANTILEVER BEAM

F O R C E S F C R T H E T E T R A H E D R O N E L E M E N T

LOAD CONDITION NUMBER 1 ELEMENT #LPREF 7 ELEMENT TYPE 50 ELEMENT GRID POINTS 2 6 10 11

APPARENT ELEMENT FORCES

POINT	FX	FY	FZ
1	-0.16121735E 02	0.25107715E 03	0.32393829E 02
2	-0.34907579E 01	-0.2075202E 03	-0.37628891E 02
3	-0.52778320E 01	0.82125488E 02	0.18310889E 03
4	0.24890640E 02	-0.12568311E 03	-0.17787378E 03

ELEMENT APPLIED FORCES

POINT	FX	FY	FZ
1	0.0	0.0	-0.93333252E 02
2	0.0	0.0	0.0
3	0.0	0.0	-0.93333252E 02
4	0.0	0.0	-0.93333252E 02

NET ELEMENT FORCES

POINT	FX	FY	FZ
1	-0.16121735E 02	0.25107715E 03	0.12572708E 03
2	-0.34907579E 01	-0.2075202E 03	-0.37628891E 02
3	-0.52778320E 01	0.82125488E 02	0.27644214E 03
4	0.24890640E 02	-0.12568311E 03	-0.84540527E 02

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FIGURE III-C.26 FORCE OUTPUT, ELEMENT NO. 7 - TETRAHEDRON ELEMENT, CANTILEVER BEAM

F O R C E S F O R T H E T E T R A H E D R O N E L E M E N T

LOAD CONDITION NUMBER 1 ELEMENT NUMBER 18 ELEMENT TYPE 50 ELEMENT GRID POINTS 12 8 16 11

APPARENT ELEMENT POINT	FX	FY	FZ
1	0.20361328E 01	0.25122803E 02	-0.10183960E 03
2	-0.24663574E 02	0.84652383E 00	-0.23251953E 01
3	0.23112753E 02	-0.46151123E 02	0.65326843E 02
4	-0.48358154E 00	0.20179155E 02	0.38832031E 02

ELEMENT APPLIED FORCES POINT	FX	FY	FZ
1	0.0	0.0	0.0
2	0.0	0.0	0.0
3	0.0	0.0	0.0
4	0.0	0.0	0.0

NET ELEMENT FORCES POINT	FX	FY	FZ
1	0.20361328E 01	0.25122803E 02	-0.10183960E 03
2	-0.24663574E 02	0.84652383E 00	-0.23251953E 01
3	0.23112753E 02	-0.46151123E 02	0.65326843E 02
4	-0.48358154E 00	0.20179155E 02	0.38832031E 02

FIGURE III-C 27 FORCE OUTPUT, ELEMENT NO. 18 - TETRAHEDRON ELEMENT, CANTILEVER BEAM

## D. Triangular Prism

A six element cantilever beam subjected to an end moment is shown in Figure III-D.1. This figure displays the loading, idealization, dimensions and material properties. The pre-printed input data forms associated with this example are given in Figures III-D.2 to III-D.9. No comments need to be made with respect to the input for this element since no peculiarities exist. The reader, however, should review the input data sheets and compare them to the previous examples for clarification purposes.

The output supplied by the MAGIC III System for this example is described below and shown in Figures III-D.10 to III-D.22. The matrix abstraction instructions are shown in Figure III-D.10. A complete description of these instructions is provided in Reference 5. Figures III-D.11 to III-D.14 show the output data obtained from the Structural Systems Monitor. These figures record the data pertinent to the problem being solved.

Figure III-D.11 displays the problem title and material data output. The gridpoint coordinates, temperatures and pressures are given in Figure III-D.12. Boundary condition information and finite element description is presented in Figure III-D.13. In the boundary condition portion of the figure, zeros ('0') represent degrees of freedom that are fixed (i.e., no motion) and ones ('1') represent degrees of freedom that are free (have unknown values of displacement). The second last column accumulates the number of ones which in this problem is 36. The second portion of Figure III-D.13 shows the finite element description. Each of the six elements is called out in turn with gridpoints, print options and material number. Note that neither extra gridpoint nor section properties are presented since they are not required for this element.

Figure III-D.14 presents the external load condition and transformed external assembled load column. This 48x1 vector is the total unreduced load which is read row-wise. The ordering of this vector is consistent with that of the boundary condition table given in Figure III-D.13. Note that a load of 66.66667 pounds

is applied at node point 4 in the negative global Y direction. This is position (11,1) in the load vector which corresponds to the eleventh entry in the boundary condition table which is the global V displacement node point 4. The other loads follow the same pattern.

MAGIC III System output of final results are displayed in Figures III-D.15 to III-D.22. Figure III-D.15 shows the stiffness matrix which is presented row-wise and it's ordering is consistent with that of the boundary condition table previously discussed. In this problem the ordering is

$$\left\{ \Delta^P \right\}^T = \left[ U_2, V_2, W_2, U_3, V_3, W_3, \dots, U_{16}, V_{16}, W_{16} \right] \text{ with}$$

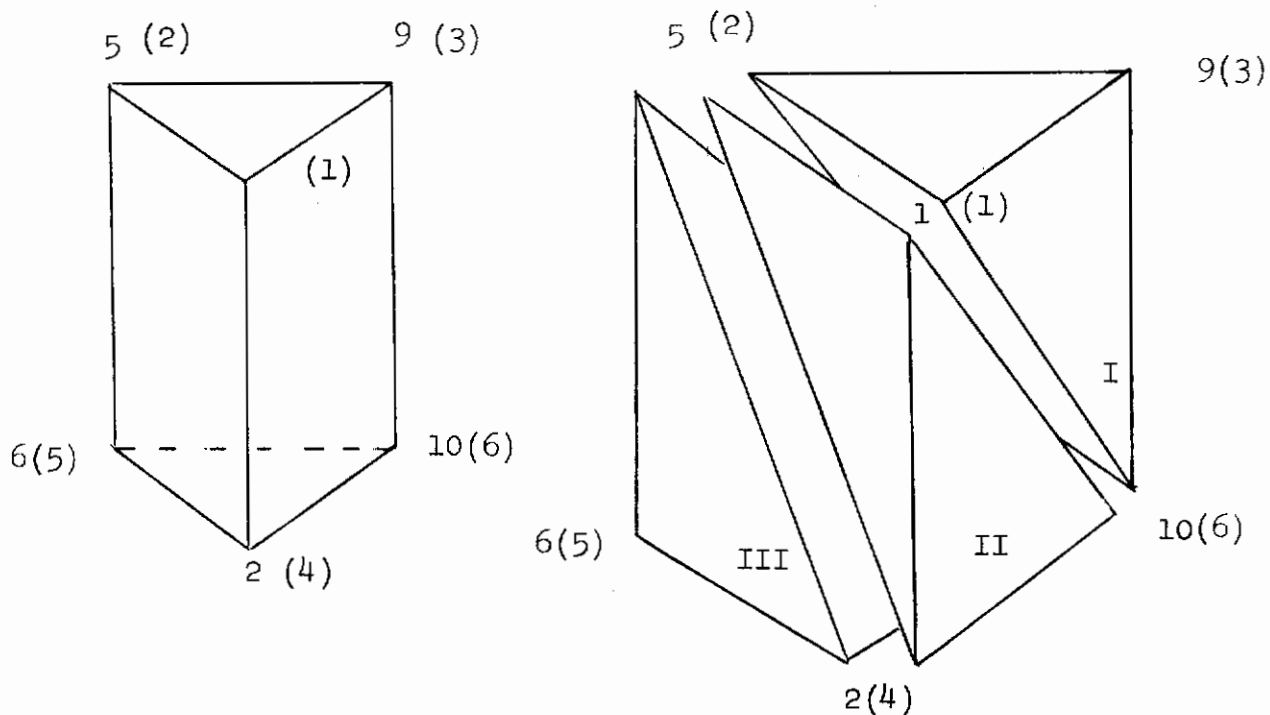
degrees of freedom  $U_1, V_1, W_1, U_5, V_5, W_5, U_9, V_9, W_9$ , and  $U_{13}, V_{13}, W_{13}$  fixed.

The externally applied load vector (GPRINT OF MATRIX LOADS) is presented in Figure III-D.16. This figure shows that forces ( $F_y$ ) are applied in the negative and positive global Y directions at node points 4, 8, 12 and 16. These forces are numerically equal to  $\pm 66.66667$  pounds and are directed to form a moment of  $M_x = 800$  in pounds applied to the tip of the cantilever.

The displacements of the cantilever beam resulting from the above loads are given in Figure III-D.17. It is noted that the displacements (U, V, W) are output corresponding to node point number and are referenced to the global axis. Figure III-D.18 shows the reactions ( $F_x, F_y, F_z$ ). These are output corresponding to node point number and are referenced to the global axes system.

The stresses arising in the structure are displayed in tabular form in Figures III-D.19 and III-D.20 for elements 1 and 6 for example. Stresses are defined at the centroid and are referenced to the global axes for each tetrahedron which makes up a particular triangular prism element. Three stress points are given under each stress category for each triangular prism. These stress points correspond to the stresses in particular tetrahedrons which are defined in the heading of the stress data. The tetrahedron nodes listed are the local node numbering system

and these must be correlated with the grid point numbering system. In the present case, element number one is defined as shown in the sketch below:

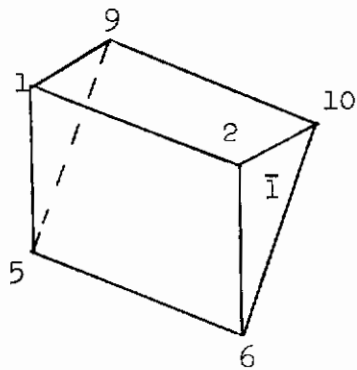
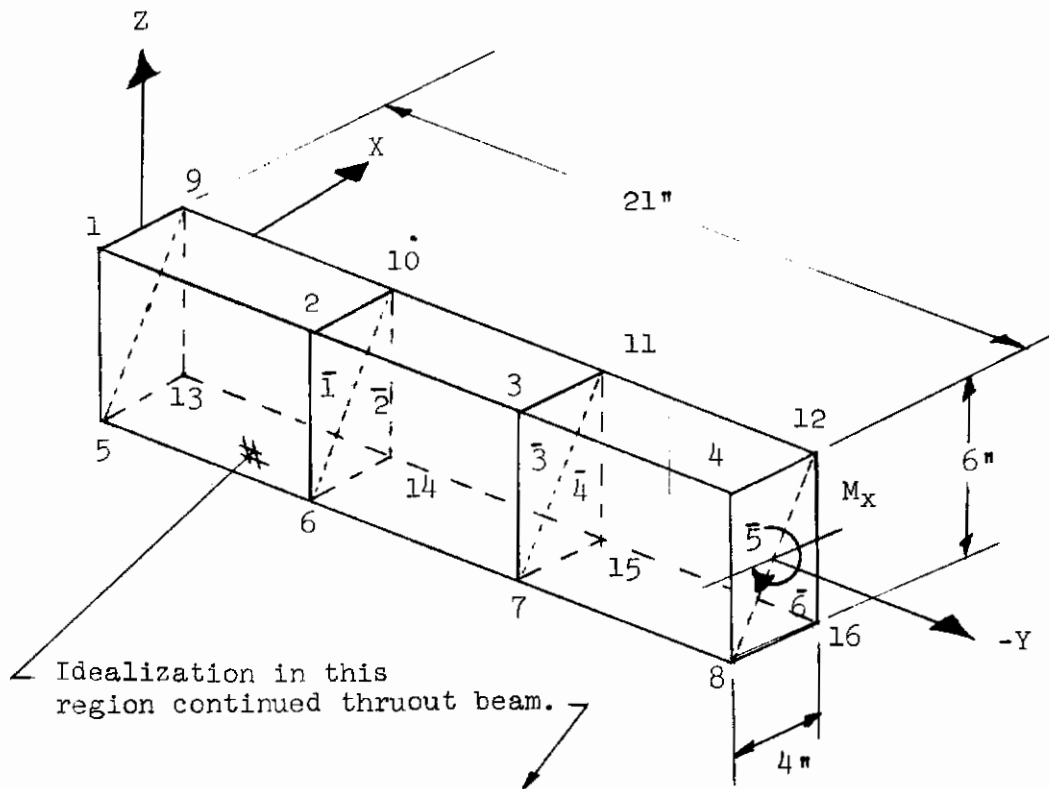


The numbers in parenthesis are the local element numbering system (See Reference 7) and the other numbers are global gridpoints. The Roman numerals on the right hand sketch are the tetrahedron numbering system. The remaining elements in the idealization are handled in the same fashion.

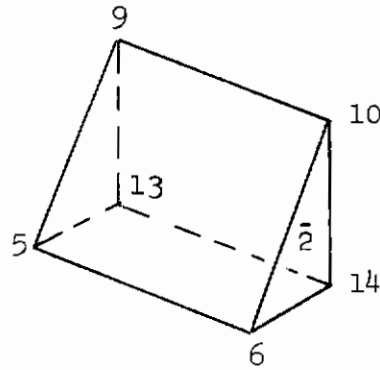
The last set of output is given in Figures III-D.21 to III-D.22 and consist of the global oriented element forces. Output labeling is analogous to the stress output except that the element forces are defined only at the six corner points of the triangular prism element. Six force points are given and for element number one for example, force points 1, 2, 3, 4, 5, 6 correspond to element grid point numbers 1, 5, 9, 2, 6, 10 respectively.



# Contrails



Prism #1



Prism #2

$$E_x = E_y = E_t = E = 30.0 \times 10^6 \text{ psi}$$

$$\nu_{xy} = \nu_{yx} = \nu_{yz} = \nu_{zy} = \nu_{zx} = \nu_{xz} = \nu = .333$$

$$E_x = E_y = E_z = 0, \quad T_1 = T_2 = T_{16} = 0.0$$

Figure III-D1 Triangular Prism Cantilever Beam With End Moment



MATER ( / )  
1 2 3 4 5 6

MAGIC STRUCTURAL ANALYSIS SYSTEM  
INPUT DATA FORMAT

( / )  
7 8 9  
No. of Requests

MATERIAL TAPE INPUT

Request Number	MATERIAL NUMBER	Lock Code	MATERIAL IDENTIFICATION												MASS DENSITY													
			1	2	3	4	5	6	7	8	9	0	1	2	3	4	5	6	7	8	9	0	1	2	3	4		
7	8	9	0	1	2	3	4	5	6	7	8	9	0	1	2	3	4	5	6	7	8	9	0	1	2	3	4	
1	1																											

MATERIAL PROPERTIES TABLE

TEMPERATURE		
1	2	
3	4	5
6	7	8
9	0	1
2	3	4
5	6	7
8	9	0
1	2	3
4	5	6
7	8	9
0	1	2

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YOUNGS MODULI		
2	3	
3	4	5
4	5	6
5	6	7
6	7	8
7	8	9
8	9	0
9	0	1
0	1	2
1	2	3
2	3	4
3	4	5
4	5	6
5	6	7
6	7	8
7	8	9
8	9	0
9	0	1
0	1	2
1	2	3
2	3	4
3	4	5
4	5	6
5	6	7
6	7	8
7	8	9
8	9	0
9	0	1
0	1	2
1	2	3
2	3	4
3	4	5
4	5	6
5	6	7
6	7	8
7	8	9
8	9	0
9	0	1
0	1	2
1	2	3
2	3	4
3	4	5
4	5	6
5	6	7
6	7	8
7	8	9
8	9	0
9	0	1
0	1	2
1	2	3
2	3	4
3	4	5
4	5	6
5	6	7
6	7	8
7	8	9
8	9	0
9	0	1
0	1	2
1	2	3
2	3	4
3	4	5
4	5	6
5	6	7
6	7	8
7	8	9
8	9	0
9	0	1
0	1	2
1	2	3
2	3	4
3	4	5
4	5	6
5	6	7
6	7	8
7	8	9
8	9	0
9	0	1
0	1	2
1	2	3
2	3	4
3	4	5
4	5	6
5	6	7
6	7	8
7	8	9
8	9	0
9	0	1
0	1	2
1	2	3
2	3	4
3	4	5
4	5	6
5	6	7
6	7	8
7	8	9
8	9	0
9	0	1
0	1	2
1	2	3
2	3	4
3	4	5
4	5	6
5	6	7
6	7	8
7	8	9
8	9	0
9	0	1
0	1	2
1	2	3
2	3	4
3	4	5
4	5	6
5	6	7
6	7	8
7	8	9
8	9	0
9	0	1
0	1	2
1	2	3
2	3	4
3	4	5
4	5	6
5	6	7
6	7	8
7	8	9
8	9	0
9	0	1
0	1	2
1	2	3
2	3	4
3	4	5
4	5	6
5	6	7
6	7	8
7	8	9
8	9	0
9	0	1
0	1	2
1	2	3
2	3	4
3	4	5
4	5	6
5	6	7
6	7	8
7	8	9
8	9	0
9	0	1
0	1	2
1	2	3
2	3	4
3	4	5
4	5	6
5	6	7
6	7	8
7	8	9
8	9	0
9	0	1
0	1	2
1	2	3
2	3	4
3	4	5
4	5	6
5	6	7
6	7	8
7	8	9
8	9	0
9	0	1
0	1	2
1	2	3
2	3	4
3	4	5
4	5	6
5	6	7
6	7	8
7	8	9
8	9	0
9	0	1
0	1	2
1	2	3
2	3	4
3	4	5
4	5	6
5	6	7
6	7	8
7	8	9
8	9	0
9	0	1
0	1	2
1	2	3
2	3	4
3	4	5
4	5	6
5	6	7
6	7	8
7	8	9
8	9	0
9	0	1
0	1	2
1	2	3
2	3	4
3	4	5
4	5	6
5	6	7
6	7	8
7	8	9
8	9	0
9	0	1
0	1	2
1	2	3
2	3	4
3	4	5
4	5	6
5	6	7
6	7	8
7	8	9
8	9	0
9	0	1
0	1	2
1	2	3
2	3	4
3	4	5
4	5	6
5	6	7
6	7	8
7	8	9
8	9	0
9	0	1
0	1	2
1	2	3
2	3	4
3	4	5
4	5	6
5	6	7
6	7	8
7	8	9
8	9	0
9	0	1
0	1	2
1	2	3
2	3	4
3	4	5
4	5	6
5	6	7
6	7	8
7	8	9
8	9	0
9	0	1
0	1	2
1	2	3
2	3	4
3	4	5
4	5	6
5	6	7
6	7	8
7	8	9
8	9	0
9	0	1
0	1	2
1	2	3
2	3	4
3	4	5
4	5	6
5	6	7
6	7	8
7	8	9
8	9	0
9	0	1
0	1	2
1	2	3
2	3	4
3	4	5
4	5	6
5	6	7
6	7	8
7	8	9
8	9	0
9	0	1
0	1	2
1	2	3
2	3	4
3	4	5
4	5	6
5	6	7
6	7	8
7	8	9
8	9	0
9	0	1
0	1	2
1	2	3
2	3	4
3	4	5
4	5	6
5	6	7
6	7	8
7	8	9
8	9	0
9	0	1
0	1	2
1	2	3
2	3	4
3	4	5
4	5	6
5	6	7
6	7	8
7	8	9
8	9	0
9	0	1
0	1	2
1	2	3
2	3	4
3	4	5
4	5	6
5	6	7
6	7	8
7	8	9
8	9	0
9	0	1
0	1	2
1	2	3
2	3	4
3	4	5
4	5	6
5	6	7
6	7	8
7	8	9
8	9	0
9	0	1
0	1	2
1	2	3
2	3	4
3	4	5
4	5	6
5	6	7
6	7	8
7	8	9
8	9	0
9	0	1
0	1	2
1	2	3
2	3	4
3	4	5
4	5	6
5	6	7
6	7	8
7	8	9
8	9	0
9	0	1
0	1	2
1	2	3
2	3	4
3	4	5
4	5	6
5	6	7
6	7	8
7	8	9
8	9	0
9	0	1
0	1	2
1	2	3
2	3	4
3	4	5
4	5	6
5	6	7
6	7	8
7	8	9
8	9	0
9	0	1
0	1	2
1	2	3
2	3	4
3	4	5
4	5	6
5	6	7
6	7	8
7	8	9
8	9	0
9	0	1
0	1	2
1	2	3
2	3	4
3	4	5
4	5	6
5	6	7
6	7	8
7	8	9
8		

## MAGIC STRUCTURAL ANALYSIS SYSTEM INPUT DATA FORMAT

### SYSTEM CONTROL INFORMATION

ENTER APPROPRIATE NUMBER, RIGHT ADJUSTED, IN BOX OPPOSITE APPLICABLE REQUESTS

S	Y	S	T	E	M
1	2	3	4	5	6

(/)

1. Number of System Grid Points

				1	6
1	2	3	4	5	6

2. Number of Input Grid Points

				1	6
7	8	9	10	11	12

3. Number of Degrees of Freedom/Grid Point

	3
13	14

4. Number of Load Conditions

	2
15	16

5. Number of Initially Displaced Grid Points

					0
17	18	19	20	21	22

6. Number of Prescribed Displaced Grid Points

					0
23	24	25	26	27	28

7. Number of Grid Point Axes Transformation Systems

	0
29	30

8. Number of Elements

					6
31	32	33	34	35	36

9. Number of Requests and/or Revisions of Material Tape.

	1
37	38

10. Number of Input Boundary Condition Points

					4
39	40	41	42	43	44

11.  $T_0$  For Structure (With Decimal Point)

				0	.	0	
45	46	47	48	49	50	51	52

(/)

FIGURE III-D.4 SYSTEM CONTROL INFORMATION -  
TRIANGULAR PRISM ELEMENT, CANTILEVER BEAM







1 2 3 4 5 6  
LOADS

(/)

BAC 1627

## MAGIC STRUCTURAL ANALYSIS SYSTEM INPUT DATA FORMAT

1 2 3 4 5 6 7 8 9 0 1 2 3 4 5 6 7 8 9 0 1 2  
LC ON D 1 0.0

(/)

Scaler

Condition Number

1 2 3 4 5 6  
MORAL

### EXTERNAL LOADS

FORCE VALUES											
F <sub>x</sub>				F <sub>y</sub>				F <sub>z</sub>			
1	2	3	4	1	2	3	4	1	2	3	4
0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0

MOMENT VALUES											
M <sub>x</sub>				M <sub>y</sub>				M <sub>z</sub>			
1	2	3	4	1	2	3	4	1	2	3	4
0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0

GENERALIZED VALUES											
1				2				3			
1	2	3	4	1	2	3	4	1	2	3	4
0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0

Grid Pt. Number	FORCE VALUES												MOMENT VALUES												GENERALIZED VALUES											
	F <sub>x</sub>				F <sub>y</sub>				F <sub>z</sub>				M <sub>x</sub>				M <sub>y</sub>				M <sub>z</sub>				1				2				3			
1	2	3	4	1	2	3	4	1	2	3	4	1	2	3	4	1	2	3	4	1	2	3	4	1	2	3	4	1	2	3	4	1	2	3	4	
7.8	9.0	1.2	3.4	5.6	7.8	9.0	1.2	3.4	5.6	7.8	9.0	1.2	3.4	5.6	7.8	9.0	1.2	3.4	5.6	7.8	9.0	1.2	3.4	5.6	7.8	9.0	1.2	3.4	5.6	7.8	9.0					
0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0					
0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0					

FIGURE III-D.7 EXTERNAL LOADS - TRIANGULAR PRISM ELEMENT, CANTILEVER BEAM





MAGIC STRUCTURAL ANALYSIS SYSTEM  
INPUT DATA FORMAT

CHECK OR END CARD

C	H	E	C	K
1	2	3	4	5

 (/)

E	N	D
1	2	3

 (/)

FIGURE III-D.9 END CARD - TRIANGULAR PRISM ELEMENT, CANTILEVER BEAM



MAGIC ABSTRACTION INSTRUCTION LISTING

TEST MAGIC

19	C	REACTS = KELA, NULL, XD	TESTE055
20		REACTP= REACTS, SLBT, TLOAD	TESTE056
21		IF (DPPF, NULL, ) GO TO 10	TESTE057
	C		TESTE058
	C	PRINT ELEMENT APPLIED LOADS, EXTERNAL LCADS, DISPLACEMENTS,	TESTE059
	C	REACTIONS AND INVERSE CHECK IN ENGINEERING FORMAT	TESTE060
	C		TESTE061
	C	ELEMENTS HAVE 1 OR 2 DEGREES OF FREEDOM	TESTE062
	C		TESTE063
22		GPR INT(4, , , FX, FY, FZ, MX, MY, MZ, SC, TR )FTELA	TESTE064
23		GPR INT(4, , , FX, FY, FZ, MX, MY, MZ, SC, )LCACS	TESTE065
24		GPR INT(2, , , U, V, W, THETA, X, THETAZ, SC, )X	TESTE066
25		GPR INT(1, , , FX, FY, FZ, MX, MY, MZ, SC, TR )REACTP	TESTE067
26		IF (I3, NULL, ) GO TO 600	TESTE068
	C		TESTE069
	C	ELEMENTS HAVE 3 DEGREES OF FREEDOM	TESTE070
	C		TESTE071
	C		TESTE072
27	10	GPR INT(4, , , FR, O, FZ, O, MBETA, O, FI, O, F3, SC, TR )FTELA	TESTE073
28		GPR INT(4, , , FR, O, FZ, O, MBETA, C, FI, O, F3, SC, )LCADS	TESTE074
29		GPR INT(2, , , U, W, O, THETA, X, O, W, O, W, SC, )X	TESTE075
30		GPR INT(1, , , FR, O, FZ, O, MBETA, O, FI, O, F3, SC, TR )REACTP	TESTE076
	C		TESTE077
	C	GENERATE STRESSES AND FORCES	TESTE078
	C		TESTE079
31	600	STRES=EM, XD , STRESS, (4, )	TESTE080
32		FORCE=EM, XD , FORCE, (4, )	TESTE081

FIGURE III-D.10 CONCLUDED

SIX ELEMENT CANTILEVER BEAM SUBJECTED TO AN END MOMENT.  
TRIANGULAR PRISM ELEMENT ICEN, NO. 51 STATICS ANALYSIS

REVISIONS OF MATERIAL TAPE

ASTERISK (\*) PRECEDING MATERIAL  
IDENTIFICATION INDICATES THAT INPUT  
ERROR RETURNS WILL NOT RESULT IN  
TERMINATION OF EXECUTION

\*\*\*\*\*

REVISION 1  
MATERIAL NUMBER 1  
MATERIAL IDENTIFICATION STEEL  
NUMBER OF MATERIAL PROPERTY POINTS . . . . 1  
NUMBER OF PLASTIC PROPERTY POINTS . . . . 0  
MASS DENSITY . . . . . 0.73394994E-03  
INPUT CODE 1

MATERIAL PROPERTIES

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	YOUNG'S MODULI		POISSON'S RATIOS	
	DIRECTIONS		DIRECTIONS	
TEMPERATURE	XX	YY	XY	ZZ
0.0	0.30000E C8	0.30000E 08	0.33300E 00	0.33300E 00
	TH. EXP.CDEF.		RIGIDITY MODUL I	
	DIRECTIONS		DIRECTIONS	
TEMPERATURE	XX	YY	XY	ZZ
0.0	0.65000E -C5	0.65000E -05	0.11252E 08	0.11252E 08

\*\*\*\*\*

FIGURE III-D.11 TITLE AND MATERIAL DATA OUTPUT - TRIANGULAR PRISM ELEMENT, CANTILEVER BEAM

16 REF. POINTS

NO. DIRECTIONS = 3 AC. DEGREES CF FREDCM = 1

GRIDPOINT DATA  
(IN RECTANGULAR COORDINATES)

POINT	X	Y	Z	TEMPERATURES	PRESSURES
1	-0.2000000E 01	0.0	0.30000000E 01	0.0	0.0
2	-0.2000000E 01	-0.7000000E 01	0.30000000E 01	0.0	0.0
3	-0.2000000E 01	-0.1400000E 02	0.30000000E 01	0.0	0.0
4	-0.2000000E 01	-0.2100000E 02	0.30000000E 01	0.0	0.0
5	-0.2000000E 01	0.0	-0.30000000E 01	0.0	0.0
6	-0.2000000E 01	-0.7000000E 01	-0.30000000E 01	0.0	0.0
7	-0.2000000E 01	-0.1400000E 02	-0.30000000E 01	0.0	0.0
8	-0.2000000E 01	-0.2100000E 02	-0.30000000E 01	0.0	0.0
9	0.2000000E 01	0.0	0.30000000E 01	0.0	0.0
10	0.2000000E 01	-0.7000000E 01	0.30000000E 01	0.0	0.0
11	0.2000000E 01	-0.1400000E 02	0.30000000E 01	0.0	0.0
12	0.2000000E 01	-0.2100000E 02	0.30000000E 01	0.0	0.0
13	0.2000000E 01	0.0	-0.30000000E 01	0.0	0.0
14	0.2000000E 01	-0.7000000E 01	-0.30000000E 01	0.0	0.0
15	0.2000000E 01	-0.1400000E 02	-0.30000000E 01	0.0	0.0
16	0.2000000E 01	-0.2100000E 02	-0.30000000E 01	0.0	0.0

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GRIDPOINT DATA  
OUTPUT - TRIANGULAR  
PRISM ELEMENT,  
CANTILEVER BEAM

FIGURE III.D.12

**BOUNDARY CONDITION INFORMATION**

NODES	DEGREES OF FREEDOM	NO. OF ONES	NO. OF TWOS
1	0	0	0
2	1	3	0
3	1	6	0
4	1	9	0
5	0	9	0
6	1	12	0
7	1	15	0
8	1	18	0
9	1	21	0
10	0	24	0
11	1	27	0
12	1	27	0
13	0	30	0
14	1	33	0
15	1	33	0
16	1	36	0

TOTAL NO. ELEMENTS = 6

ELEM TYPE	MAT. NO.	CODE	TEMP.	PRMT	NO.	GRID POINTS	EXTRA GRID PTS	SECTION PROPERTIES
1	51	1	6	0.0	0	1 5 9 2 6 10		0.0
2	51	1	6	0.0	0	5 13 9 6 14 10		0.0
3	51	1	6	0.0	0	2 6 10 3 7 11		0.0
4	51	1	6	0.0	0	6 14 10 7 15 11		0.0
5	51	1	6	0.0	0	3 7 11 4 8 12		0.0
6	51	1	6	0.0	0	7 15 11 8 16 12		0.0

FIGURE III-D.13 BOUNDARY CONDITION AND FINITE ELEMENT DESCRIPTION -  
TRIANGULAR PRISM ELEMENT, CANTILEVER BEAM



EXTERNAL LOAD CONDITIONS 1

LOAD NO. 1

ELEMENT LOAD SCALAR = 0.0

LOAD NO.	NUMBER OF LOADED NODES	16
4	0.0	0.0
12	0.0	-0.66667E 02
18	0.0	-0.66667E 02
16	0.0	0.66667E 02
1	0.0	0.0
2	0.0	0.0
3	0.0	0.0
5	0.0	0.0
6	0.0	0.0
7	0.0	0.0
9	0.0	0.0
10	0.0	0.0
11	0.0	0.0
13	0.0	0.0
14	0.0	0.0
15	0.0	0.0

TRANSFORMED EXTERNAL ASSEMBLED LOAD COLUMN

48 X 1

0.0	0.0	0.0	0.0	0.0	0.0
0.0	0.0	0.0	0.0	-0.6666992E 02	0.0
0.0	0.0	0.0	0.0	0.0	0.0
0.0	0.0	0.0	0.0	0.6666992E 02	0.0
0.0	0.0	0.0	0.0	0.0	0.0
0.0	0.0	0.0	0.0	-0.6666992E 02	0.0
0.0	0.0	0.0	0.0	0.0	0.0
0.0	0.0	0.0	0.0	0.6666992E 02	0.0

1-ZERO FOR STRUCTURE = 0.0

FIGURE III-D.14 TRANSFORMED EXTERNAL ASSEMBLED LOAD COLUMN - TRIANGULAR PRISM ELEMENT, CANTILEVER BEAM

CUTOFF = 0.0

SIZE 36 BY 36

DISP	FORCE	FORCE	FORCE	FORCE	FORCE	FORCE	FORCE	FORCE	FORCE	
1	1	0.275072E 09	2	0.336910E 08	3	-0.786124E 08	4	-0.643016E 07	5	-0.112528E 08
6	6	-0.131283E 08	10	-0.224382E 08	11	0.224382E 08	12	0.523558E 08	19	-0.235955E 09
20	20	-0.336910E 08	21	0.262566E 08	22	-0.203010E 02	23	0.336910E 08	24	0.131283E 08
2	1	0.336910E 08	2	0.136698E 09	3	0.224606E 08	4	-0.224382E 08	5	-0.256821E 08
6	6	-0.750187E 07	10	0.112528E 08	11	-0.262565E 08	12	-0.224606E 08	19	-0.336910E 08
20	20	-0.590772E 08	21	0.750187E 07	22	0.336910E 08	23	-0.853910E 01	24	-0.750187E 07
3	1	-0.786124E 08	2	0.224606E 08	3	0.176806E 09	4	-0.261779E 08	5	-0.149588E 08
6	6	-0.643015E 07	10	0.262566E 08	11	-0.224607E 08	12	-0.104865E 08	19	0.523558E 08
20	20	0.149588E 08	21	-0.590772E 08	22	0.261779E 08	23	-0.149588E 08	24	-0.853910E 01
4	1	-0.643016E 07	2	-0.224382E 08	3	-0.261779E 08	4	0.275072E 08	5	0.336910E 08
6	6	-0.786124E 08	7	-0.643017E 07	8	-0.112528E 08	9	-0.131283E 08	10	-0.212395E 01
11	11	-0.224382E 08	12	0.261779E 08	13	-0.262565E 08	14	0.224382E 08	19	0.523558E 08
22	22	-0.336910E 08	23	-0.336910E 08	24	0.262566E 08	25	0.582887E 01	26	0.336910E 08
27	27	0.131283E 08								
5	1	-0.112528E 08	2	-0.256821E 08	3	-0.149588E 08	4	0.336910E 08	5	0.136698E 09
6	6	0.224606E 08	7	-0.224382E 08	8	-0.256821E 08	9	-0.750187E 07	10	-0.112528E 08
11	11	0.88256E 00	12	0.224606E 08	13	0.112528E 08	14	-0.262565E 08	15	-0.224607E 08
22	22	-0.336910E 08	23	-0.590772E 08	24	0.750187E 07	25	0.336910E 08	26	0.483793E 01
27	27	-0.750187E 07								
6	1	-0.131283E 08	2	-0.750186E 07	3	-0.643015E 07	4	-0.786124E 08	5	0.224606E 08
6	6	0.176806E 09	7	-0.261779E 08	8	-0.149588E 08	9	-0.643017E 07	10	0.131283E 08
11	11	0.224607E 08	12	-0.114953E 02	13	0.262566E 08	14	-0.224606E 08	15	-0.104865E 09
22	22	0.523558E 08	23	0.149588E 08	24	-0.590772E 08	25	0.261779E 08	26	-0.149588E 08
27	27	0.245715E 02								
7	4	-0.643017E 07	5	-0.224382E 08	6	-0.261779E 08	7	0.172486E 08	8	0.336910E 08
9	9	-0.393062E 08	13	-0.220379E 01	14	-0.224382E 08	15	0.261779E 08	16	-0.875218E 07
17	17	0.224382E 08	18	0.261779E 08	25	-0.157303E 09	26	-0.112528E 08	27	0.131283E 08
8	4	-0.112528E 08	5	-0.256821E 08	6	-0.149588E 08	7	0.336910E 08	8	0.738192E 08
9	9	-0.535504E 01	13	-0.112528E 08	14	-0.880196E 01	15	0.224607E 08	16	0.112528E 08
17	17	-0.875218E 07	18	-0.750187E 07	25	-0.224382E 08	26	-0.393062E 08	27	-0.534968E 01
9	4	-0.131283E 08	5	-0.750187E 07	6	-0.643017E 07	7	-0.393062E 08	8	-0.535504E 01
9	9	0.807713E 08	13	0.131283E 08	14	0.224607E 08	15	-0.220379E 01	16	0.131283E 08
17	17	-0.149588E 08	18	-0.349563E 08	25	0.261779E 08	26	-0.266289E 01	27	-0.393062E 08

FIGURE III-D 15 STIFFNESS MATRIX OUTPUT - TRIANGULAR PRISM ELEMENT, CANTILEVER BEAM

MATRIX STIFF / 1/

CUTOFF = 0.0		FORCE		FORCE		FORCE		FORCE		FORCE		FORCE		FORCE	
DISP	FORCE	10	1	2	3	4	5	6	7	8	9	10	11	12	13
DISP	10	1	-0.262569E 08	2	0.112528E 08	3	0.262566E 08	4	-0.212399E 01	5	-0.112528E 08	6	0.131283E 08	7	-0.224382E 01
	6	14	0.131283E 08	10	0.305436E 09	11	-0.336909E 08	12	-0.393062E 08	13	-0.216125E 08	14	0.224382E 01	17	0.224382E 01
	22	30	0.875218E 07	23	0.261779E 08	24	-0.875218E 07	28	-0.336910E 08	29	-0.786123E 08	32	0.224382E 01	31	0.224382E 01
	30		0.916620E 08		0.336910E 08		-0.393062E 08		-0.244707E 05		0.336910E 08		0.224382E 01		0.336910E 08
DISP	11	1	0.224382E 08	2	-0.262565E 08	3	-0.224607E 08	4	-0.224382E 01	5	0.882556E 00	6	0.224607E 08	7	0.673820E 08
	6	14	0.224607E 08	10	0.336909E 08	11	0.205567E 09	12	0.673820E 08	13	0.112528E 09	14	-0.601164E 08	17	-0.875218E 07
	22	30	0.336910E 08	23	0.875218E 07	24	0.224606E 08	28	0.336910E 08	29	-0.678294E 08	32	0.224607E 08	31	0.224607E 08
	30		-0.149588E 08		0.149588E 08		0.875218E 07		0.336910E 08		0.149588E 08		0.224607E 08		0.224607E 08
DISP	12	1	0.523558E 08	2	-0.224606E 08	3	-0.104869E 09	4	0.261779E 08	5	0.224606E 08	6	0.112528E 08	7	0.259579E 08
	6	14	-0.114933E 02	10	-0.393062E 08	11	0.673820E 08	12	0.259579E 08	13	0.131283E 09	14	-0.299625E 08	17	0.259579E 08
	22	30	-0.299625E 08	23	-0.478166E 08	24	0.224606E 08	28	-0.224606E 08	29	-0.349563E 08	32	0.224606E 08	31	-0.349563E 08
	30		-0.940335E 08		0.224606E 08		0.349563E 08		0.655627E 08		0.349563E 08		0.224606E 08		0.224606E 08
DISP	13	4	-0.262565E 08	5	0.112528E 08	6	0.262566E 08	7	-0.220379E 01	8	-0.112528E 08	9	0.131283E 09	10	0.305436E 09
	9	14	0.131283E 08	10	-0.216125E 08	11	0.112528E 08	12	0.131283E 01	13	0.305436E 09	14	-0.299625E 08	17	0.224382E 01
	22	30	-0.336910E 08	23	-0.393061E 08	24	-0.786123E 08	25	0.675218E 07	26	0.224606E 08	29	0.224606E 08	31	0.224606E 08
	30		0.336910E 08		0.875218E 07		0.875218E 07		-0.131283E 01		0.875218E 07		0.224606E 08		0.224606E 08
DISP	14	4	0.224382E 08	5	-0.262565E 08	6	-0.224606E 08	7	-0.224382E 01	8	-0.860156E 01	9	0.224607E 08	10	0.336909E 08
	9	14	0.224607E 08	10	0.224382E 08	11	0.224382E 08	12	-0.299625E 08	13	-0.336909E 08	14	-0.478166E 08	17	0.224382E 01
	22	30	0.336910E 08	23	-0.875218E 07	24	-0.875218E 07	25	0.336910E 08	26	0.875218E 07	29	0.336910E 08	31	0.336910E 08
	30		0.224607E 08		-0.149588E 08		0.149588E 08		0.149588E 08		0.336910E 08		0.224607E 08		0.224607E 08
DISP	15	4	0.923558E 08	5	-0.224607E 08	6	-0.104869E 09	7	0.261779E 08	8	0.224607E 08	9	0.224607E 08	10	0.336909E 08
	9	14	-0.220379E 01	10	0.261779E 08	11	-0.374194E 08	12	-0.478166E 08	13	-0.393061E 08	14	-0.478166E 08	17	0.224607E 08
	22	30	0.875218E 07	23	0.259579E 08	24	0.131283E 08	25	-0.299625E 08	26	0.224606E 08	29	0.224606E 08	31	0.224606E 08
	30		0.750186E 07		-0.261779E 08		0.750186E 07		0.349563E 08		0.750186E 07		0.224606E 08		0.224606E 08
DISP	16	7	-0.875218E 07	8	0.112528E 08	9	0.131283E 08	10	-0.216125E 08	11	0.112528E 08	12	0.240977E 02	13	0.281140E 02
	15	26	0.112528E 08	16	0.109016E 09	17	0.240977E 02	18	0.281140E 02	19	0.281140E 02	20	0.281140E 02	21	0.281140E 02
	34		-0.786517E 08		0.336910E 08		0.336910E 08		0.336910E 08		0.336910E 08		0.336910E 08		0.336910E 08
DISP	17	7	0.224382E 08	8	-0.875218E 07	9	-0.149588E 08	10	0.224382E 08	11	0.149588E 08	12	0.224382E 08	13	0.224382E 08
	15	26	-0.299625E 08	16	0.240977E 02	17	0.240977E 02	18	0.240977E 02	19	0.240977E 02	20	0.240977E 02	21	0.240977E 02
	26		-0.875218E 07		0.149588E 08		0.149588E 08		0.336910E 08		0.336910E 08		0.336910E 08		0.336910E 08

FIGURE III-D.15 CONTINUED

MATRIX STIFF / I/

DISP	FORCE	CUTOFF = 0.0		FORCE	FCMC	SIZE		FORCE	FORCE
		35	8			36 BY	36		
17	34	0.112528E 08	-0.196924E 08	35	9	13	0.261779E 06	14	-0.374194E 08
DISP	7	0.261779E 08	-0.750187E 07	8	17	18	0.449213E 08	25	-0.393062E 08
DISP	15	-0.478166E 08	0.281140E 02	16	31	32	0.750187E 07	33	0.349562E 08
DISP	26	-0.750186E 07	-0.349563E 08	27	3	10	-0.875219E 07	11	-0.336910E 08
DISP	34	0.131283E 08	-0.196924E 08	36	20	22	-0.128603E 06	23	-0.224382E 08
DISP	1	-0.235949E 09	-0.336910E 08	2	30	10	0.336910E 06	11	-0.875220E 07
DISP	12	-0.786123E 08	0.287932E 09	19	3	21	0.224606E 06	22	-0.112528E 08
DISP	28	-0.175044E 08	0.112528E 08	29	20	29	-0.175044E 06	30	-0.224606E 08
DISP	1	-0.336910E 08	-0.590772E 08	2	3	10	0.149588E 06	11	-0.875220E 07
DISP	12	-0.224606E 08	0.336909E 08	19	20	21	0.224606E 06	22	-0.112528E 08
DISP	23	-0.513642E 08	-0.750186E 07	24	28	29	-0.175044E 06	30	-0.224606E 08
DISP	1	0.262566E 08	0.750187E 07	2	3	10	-0.590772E 06	11	-0.224607E 08
DISP	12	-0.349563E 08	0.224606E 08	20	21	23	0.149588E 06	24	-0.128603E 08
DISP	26	0.523959E 08	-0.224607E 08	29	30	4	-0.699125E 08	5	-0.336910E 08
DISP	1	-0.203010E 02	0.336910E 08	2	3	12	0.261779E 06	13	-0.875219E 07
DISP	6	0.523558E 08	0.875218E 07	10	11	20	-0.393062E 06	22	0.287932E 09
DISP	14	-0.336910E 08	-0.786123E 07	15	19	26	0.112528E 06	27	0.934195E 01
DISP	23	0.336910E 08	-0.351425E 01	24	25	31	-0.224382E 06	32	0.112528E 08
DISP	28	-0.875218E 07	-0.112528E 08	29	30	4	-0.175044E 06	5	-0.590772E 08
DISP	33	0.262566E 08	-0.224606E 08	33	3	12	0.224606E 06	13	-0.336910E 08
DISP	1	0.336910E 08	-0.853910E 01	2	11	20	-0.149588E 06	21	-0.149588E 08
DISP	6	0.149588E 08	0.336910E 08	10	19	25	0.875220E 07	26	-0.513642E 08
DISP	14	-0.875218E 07	-0.224606E 08	15	24	30	-0.224606E 06	31	0.513643E 08
DISP	22	0.336910E 08	0.188062E 09	23	29	4	0.224606E 06	5	0.224382E 08
DISP	27	-0.750187E 07	-0.224382E 08	28	3	12	-0.875220E 06	13	-0.336910E 08
DISP	32	-0.175044E 08	-0.224606E 08	33	11	20	0.262566E 06	21	-0.786123E 08
DISP	1	0.336910E 08	-0.853910E 01	2	19	26	0.224606E 06	27	-0.128603E 08
DISP	6	0.149588E 08	0.336910E 08	10	24	31	-0.175044E 06	32	-0.224606E 08
DISP	14	-0.875218E 07	-0.224606E 08	15	3	4	0.262566E 06	5	0.750187E 07
DISP	22	0.336910E 08	0.188062E 09	23	11	12	0.349563E 06	13	-0.786123E 08
DISP	27	-0.750187E 07	-0.224382E 08	28	20	21	-0.128603E 06	22	-0.391425E 01
DISP	32	-0.175044E 08	-0.224606E 08	33	26	27	-0.149588E 06	28	-0.128603E 08
DISP	1	0.336910E 08	-0.853910E 01	2	30	31	0.523558E 06	32	-0.224606E 08
DISP	6	0.149588E 08	0.336910E 08	10	3	4	0.262566E 06	5	0.750187E 07
DISP	14	-0.875218E 07	-0.224606E 08	15	11	12	0.349563E 06	13	-0.786123E 08
DISP	22	0.336910E 08	0.188062E 09	23	20	21	-0.128603E 06	22	-0.391425E 01
DISP	27	-0.750187E 07	-0.224382E 08	28	26	27	-0.149588E 06	28	-0.128603E 08
DISP	32	-0.175044E 08	-0.224606E 08	33	31	32	0.523558E 06	33	-0.224606E 08
DISP	4	0.582887E 01	0.336910E 08	5	6	7	-0.157303E 01	8	-0.224382E 08
DISP	9	0.261779E 08	0.875218E 07	13	14	15	0.336910E 06	16	-0.875218E 07
DISP	17	-0.324382E 08	-0.393062E 08	18	22	23	-0.112528E 06	24	0.469504E 01
DISP	25	0.187668E 09	-0.527137E 01	26	27	31	0.199140E 02	32	-0.112528E 08
DISP	33	0.131283E 08	-0.875218E 07	34	35	36	-0.264096E 01	37	0.131283E 06

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FIGURE III-D.15 CONTINUED



CUTOFF = 0.0

DISP	FORCE	FORCE	FORCE	FORCE	SIZE	FORCE	36 BY	36			
35	16	0.224382E 08	17	-0.196924E 08	25	-0.264096E 01	26	-0.875218E 07	27	-0.149588E 08	
31	31	0.112528E 08	32	-0.256821E 08	33	-0.750187E 07	34	-0.336910E 06	35	0.541267E 08	
36	36	0.224606E 08									
DISP	36	16	0.261779E 08	18	-0.196924E 08	25	0.131283E 08	26	-0.750187E 07	27	-0.349563E 08
	32	32	-0.149588E 08	33	-0.643017E 07	34	-0.393062E 08	35	0.224606E 06	36	0.610788E 08

FIGURE III-D.15 CONCLUDED

GPRINT CF MATRIX LOADS (SET 1)

ROW	FX	FY	FZ
1	0.0	0.0	0.0
2	0.0	0.0	0.0
3	0.0	0.0	0.0
4	0.0	-0.66666592E 02	0.0
5	0.0	0.0	0.0
6	0.0	0.0	0.0
7	0.0	0.0	0.0
8	0.0	0.66666592E 02	0.0
9	0.0	0.0	0.0
10	0.0	0.0	0.0
11	0.0	0.0	0.0
12	0.0	-0.66666592E 02	0.0
13	0.0	0.0	0.0
14	0.0	0.0	0.0
15	0.0	0.0	0.0
16	0.0	0.66666592E 02	0.0

FIGURE III-D.16 GPRINT OF MATRIX LOADS - TRIANGULAR PRISM ELEMENT, CANTILEVER BEAM



DISPLACEMENT MATRIX FOR LOAD CONDITION 1

48 X 1

ROW	U	V	W
1	0.0	0.0	0.0
2	0.5194209E-06	-0.18232240E-05	-0.17246730E-05
3	0.2285644E-05	-0.36443889E-05	-0.67784067E-05
4	0.54131560E-05	-0.54552580E-05	-0.14936108E-04
5	0.0	0.0	0.0
6	0.8228638E-06	0.10807307E-05	-0.16738813E-05
7	0.2843736E-05	0.22012910E-05	-0.67885239E-05
8	0.59999056E-05	0.38794566E-05	-0.16189276E-04
9	0.0	0.0	0.0
10	0.3834659E-06	-0.10807571E-05	-0.14056868E-05
11	0.21245487E-05	-0.22067861E-05	-0.42474273E-05
12	0.51698862E-05	-0.33873048E-05	-0.14530836E-04
13	0.0	0.0	0.0
14	0.93951201E-06	0.18882555E-05	-0.13954250E-05
15	0.29730098E-05	0.35991225E-05	-0.61594055E-05
16	0.64449150E-05	0.62887530E-05	-0.15540849E-04

FIGURE III-D.17 DISPLACEMENT MATRIX - TRIANGULAR PRISM ELEMENT, CANTILEVER BEAM

REACTIONS AND INVERSE CHECK FOR LOAD CONDITION 1

ROW	FX	FY	FZ
1	-0.1504763E 02	0.71572342E 02	0.50970963E 02
2	-0.16326904E-02	0.85449219E-03	0.07931480E-02
3	-0.28228760E-02	0.25482178E-02	0.25132437E-02
4	-0.35705566E-02	0.30517578E-04	0.29296875E-02
5	-0.17282297E 02	-0.71578503E 02	-0.57377411E 02
6	0.99162129E-03	-0.11596680E-02	-0.11444092E-02
7	0.17089844E-02	-0.27465820E-03	-0.17089844E-02
8	-0.73242188E-03	-0.36621094E-03	0.36621094E-02
9	0.19318710E 02	0.61742584E 02	0.34244446E 02
10	0.45776367E-03	0.41198730E-03	0.91552734E-04
11	-0.54931641E-03	-0.64086914E-03	0.48828125E-03
12	0.21057129E-02	-0.10681152E-02	0.73242188E-03
13	0.13012327E 02	-0.61735397E 02	-0.27844162E 02
14	-0.76293949E-03	0.12207031E-03	0.51879883E-03
15	0.48828125E-03	0.3505215E-03	-0.25939941E-03
16	0.17089844E-02	-0.97456250E-03	-0.39062500E-02

FIGURE III-D.18 REACTION MATRIX - TRIANGULAR PRISM ELEMENT, CANTILEVER BEAM

STRESSES FOR THE TRIANGULAR PRISM ELEMENT

(STRESS POINT ONE EQUALS ELEMENT STRESSES AT CENTROID OF TETRAHEDRON WITH NODES(1,2,3,4))  
 (STRESS POINT TWO EQUALS ELEMENT STRESSES AT CENTROID OF TETRAHEDRON WITH NODES(1,2,1,4))  
 (STRESS POINT THREE EQUALS ELEMENT STRESSES AT CENTROID OF TETRAHEDRON WITH NODES(2,6,5,4))

LOAD CONDITION NUMBER	ELEMENT NUMBER	ELEMENT TYPE	ELEMENT GRID POINTS					
			1	5	9	2	6	10
1	1	51						
APPARENT ELEMENT STRESSES			MEMBRANE STRESSES					
STRESS POINT			SIGMA-X	SIGMA-Y	SIGMA-Z	SIGMA-XY	SIGMA-YZ	SIGMA-ZX
1			0.34643211E 01	0.69390488E 01	0.34643211E 01	-0.61643881E 00	0.22597046E 01	0.0
2			0.43166809E 01	0.10943444E 02	0.50810103E 01	0.12537174E 01	0.27724876E 01	0.89737320E 00
3			-0.51817627E 01	-0.78914680E 01	-0.46073389E 01	0.76592064E 00	-0.27554369E 01	0.32827568E 00

ELEMENT APPLIED STRESSES			MEMBRANE STRESSES		
STRESS POINT	SIGMA-X	SIGMA-Y	SIGMA-Z	SIGMA-XY	SIGMA-YZ
1	0.0	0.0	0.0	0.0	0.0
2	0.0	0.0	0.0	0.0	0.0
3	0.0	0.0	0.0	0.0	0.0

NET ELEMENT STRESSES			MEMBRANE STRESSES		
STRESS POINT	SIGMA-X	SIGMA-Y	SIGMA-Z	SIGMA-XY	SIGMA-YZ
1	0.34643211E 01	0.69390488E 01	0.34643211E 01	-0.61643881E 00	0.22597046E 01
2	0.43166809E 01	0.10943444E 02	0.50810103E 01	0.12537174E 01	0.27724876E 01
3	-0.51817627E 01	-0.78914680E 01	-0.46073389E 01	0.76592064E 00	-0.27554369E 01

FIGURE III-D.19 STRESS OUTPUT, ELEMENT NO. 1 - TRIANGULAR PRISM ELEMENT, CANTILEVER BEAM

STRESSES FOR THE TRIANGULAR PRISM ELEMENT

(STRESS POINT ONE EQUALS ELEMENT STRESSES AT CENTROID OF TETRAHEDRON WITH NODES (2,3,4))  
 (STRESS POINT TWO EQUALS ELEMENT STRESSES AT CENTROID OF TETRAHEDRON WITH NODES (1,4))  
 (STRESS POINT THREE EQUALS ELEMENT STRESSES AT CENTROID OF TETRAHEDRON WITH NODES (2,5,4))

LOAD CONDITION NUMBER	ELEMENT NUMBER	ELEMENT TYPE	ELEMENT GRID POINTS					
1	6	51	7	15	11	8	14	12
APPARENT ELEMENT STRESSES								
POINT	SIGMA-X	SIGMA-Y	SIGMA-Z	MEMBRANE STRESSES			SIGMA-YZ	SIGMA-ZX
1	0.49076519E 01	0.79760122E 01	0.38501024E 01	-0.96312809E 00	-0.24272915E 01	0.17858338E 00	0.24272915E 01	0.17858338E 00
2	-0.37657166E 00	-0.64593439E 01	0.24569702E 01	-0.11413012E 01	-0.12466357E 01	-0.29281031E -01	-0.12466357E 01	-0.29281031E -01
3	0.15577698E 00	-0.10995422E 02	0.14404602E 01	0.11966228E 01	-0.30660515E 01	-0.79219318E 00	-0.30660515E 01	-0.79219318E 00
ELEMENT APPLIED STRESSES								
POINT	SIGMA-X	SIGMA-Y	SIGMA-Z	MEMBRANE STRESSES			SIGMA-YZ	SIGMA-ZX
1	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0
2	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0
3	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0
NET ELEMENT STRESSES								
POINT	SIGMA-X	SIGMA-Y	SIGMA-Z	MEMBRANE STRESSES			SIGMA-YZ	SIGMA-ZX
1	0.49076519E 01	0.79760122E 01	0.38501024E 01	-0.96312809E 00	-0.24272915E 01	0.17858338E 00	0.24272915E 01	0.17858338E 00
2	-0.37657166E 00	-0.64593439E 01	0.24569702E 01	-0.11413012E 01	-0.12466357E 01	-0.29281031E -01	-0.12466357E 01	-0.29281031E -01
3	0.15577698E 00	-0.10995422E 02	0.14404602E 01	0.11966228E 01	-0.30660515E 01	-0.79219318E 00	-0.30660515E 01	-0.79219318E 00

FIGURE III-D.20 STRESS OUTPUT ELEMENT NO. 6 - TRIANGULAR PRISM ELEMENT, CANTILEVER BEAM

F O R C E S F C P T H E T R I A N G U L A R P R I S M E L E M E N T

LOAD CONDITION NUMBER 1 ELEMENT NUMBER 1 ELEMENT TYPE 51 ELEMENT GRID POINTS 1 5 9 2 6 10

APPARENT ELEMENT POINT	FORCES		ELEMENT TYPE
	FX	FZ	
1	-0.15047638E 02	0.71572342E 02	0.50970963E 02
2	-0.11240540E 01	-0.55049362E 02	-0.50902725E 02
3	0.21784470E 02	0.23441101E 02	0.90388203E 01
4	0.25726776E 01	-0.70769852E 02	-0.41170395E 02
5	-0.45955963E 01	0.4424530E 02	0.32522629E 02
6	-0.35892285E 01	-0.13610750E 02	-0.45930481E 00

ELEMENT APPLIED FORCES POINT	FORCES	
	FX	FZ
1	0.0	0.0
2	0.0	0.0
3	0.0	0.0
4	0.0	0.0
5	0.0	0.0
6	0.0	0.0

NET ELEMENT FORCES POINT	FORCES	
	FX	FZ
1	-0.15047638E 02	0.71572342E 02
2	-0.11240540E 01	-0.55049362E 02
3	0.21784470E 02	0.23441101E 02
4	0.25726776E 01	-0.70769852E 02
5	-0.45955963E 01	0.4424530E 02
6	-0.35892285E 01	-0.13610750E 02

FIGURE III-D.21 FORCE OUTPUT, ELEMENT NO. 1 - TRIANGULAR PRISM ELEMENT, CANTILEVER BEAM

F O R C E S F O R T H E T R I A N G U L A R P R I S M E L E M E N T

LOAD CONDITION NUMBER 1 ELEMENT NUMBER 6 ELEMENT TYPE 51 ELEMENT GRID POINTS 7 15 11 8 14 12

APPARENT ELEMENT POINT	FORCES		ELEMENT TYPE
	FX	FZ	
1	-0.36415189E 02	0.53542480E 01	
2	0.35807129E 02	-0.40652100E 02	
3	-0.30191040E 01	0.27676498E 02	
4	0.36113281E 01	-0.85327148E 00	
5	0.97656250E-03	-0.31738281E-02	
6	0.19034155E-01	0.84785156E 01	

ELEMENT APPLIED POINT	FORCES	
	FX	FZ
1	0.0	0.0
2	0.0	0.0
3	0.0	0.0
4	0.0	0.0
5	0.0	0.0
6	0.0	0.0

NET ELEMENT POINT	FORCES	
	FX	FZ
1	-0.36415189E 02	0.53542480E 01
2	0.35807129E 02	-0.40652100E 02
3	-0.30191040E 01	0.27676498E 02
4	0.36113281E 01	-0.85327148E 00
5	0.97656250E-03	-0.31738281E-02
6	0.19034155E-01	0.84785156E 01

FIGURE III-D.22 FORCE OUTPUT, ELEMENT NO. 6 - TRIANGULAR PRISM ELEMENT, CANTILEVER BEAM

## E. Symmetric Triangular Prism

A six element cantilever beam subjected to an end moment is shown in Figure III-E.1. This figure depicts the loading, idealization, dimensions, and material properties. The preprinted input data forms for this example are given in Figures III-E.2 to III-E.9. Preparation of input data for this element is straight forward, however, a comment must be made on the 'Element Control Data' form, Figure III-E.8. Since we are using a symmetric triangular prism element only three (3) node points define the element. Although column 34 in this figure indicates that 6 input nodes are needed the user only inputs the three pertinent node points. Note also that the 'Plug No,' columns 11 and 12 is the same as used for the symmetric triangular prism element. It is also important to note that the user must define the global XY plane as the plane of symmetry for the symmetric triangular prism.

The output supplied by the MAGIC III System for this particular example is described below and shown in Figures III-E.10 to III-E.21. The matrix abstractions are shown in Figures III-E.10. A complete description of the instructions is provided in Reference 5. Output from the Structural Systems Monitor is given in Figures III-E.11 to III-E.13. These figures record the data pertinent to the problem being solved.

The problem title and material data output are shown in Figure III-E.11. Gridpoint coordinates, temperatures, and pressures are given in Figure III-E.12, as well as boundary condition information and element description. In the boundary condition portion of the figure, zeros ('0') represent degrees of freedom that are fixed (i.e. no motion) and ones ('1') represent degrees of freedom that are free or have unknown values of displacement. The second last column represents the cumulative number of degrees of freedom which actively participate in the equation solving process for displacements. The last column accumulates the number of twos ('2') which participate in the calculation of the reduced stiffness matrix. The third portion of Figure III-E.12



shows the finite element description. Each of the six elements is called out in turn with gridpoints, print options and material number. Note that no extra grid points are listed nor needed for this element. The same comment also holds for section properties since all pertinent data are calculated within the program.

Figure III-E.13 displays the external load condition and transformed external assembled load column. This  $24 \times 1$  vector is the total unreduced load which is read row-wise. The ordering of this vector is consistent with that of the boundary condition table given in Figure III-E.12. Note that a load of 66.66667 pounds is applied at node 4 in the negative global Y direction. This is position (11,1) in the load vector which corresponds to the eleventh entry in the boundary condition table which is the global V displacement for node point 4. The other loads follow the same pattern.

MAGIC III system output of final results are displayed in Figures III-E.14 to III-E.21. Figure III-E.14 shows the stiffness matrix for this problem. Note that only the non-zero terms are displayed. The stiffness matrix is presented row-wise and its ordering is consistent with that of the boundary condition table previously discussed. In this problem the ordering is

$$\{ \Delta^{SP} \}^T = [U_2, V_2, W_2, U_3, V_3, W_3, \dots, U_8, V_8, W_8]$$

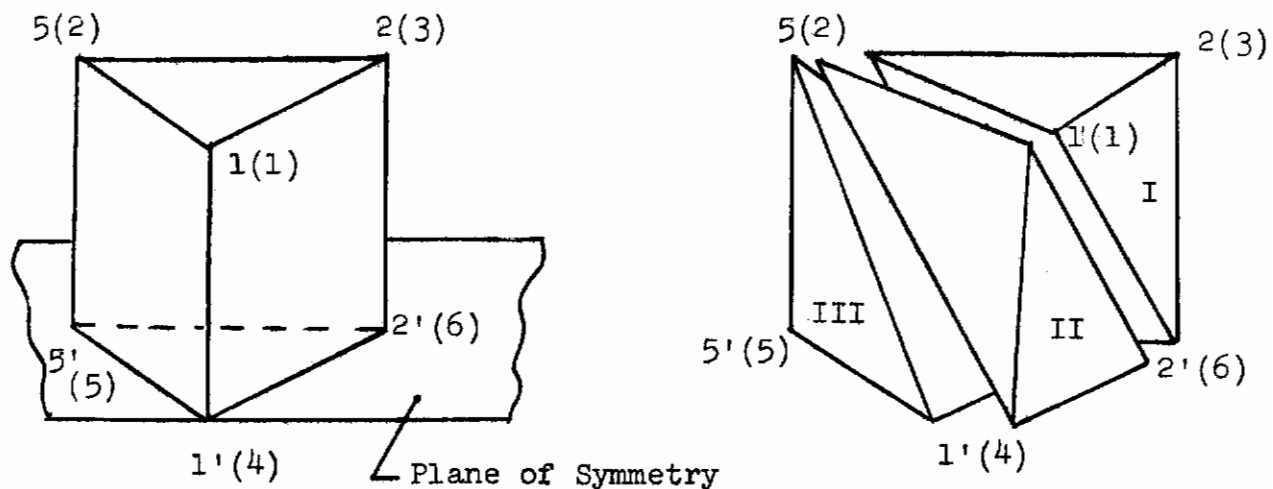
with displacements  $U_1, V_1, W_1$  and  $U_5, V_5$  and  $W_5$  fixed.

The externally applied load vector (GPRINT OF MATRIX LOADS) is presented in Figure III-E.15. This figure shows that forces ( $F_y$ ) are applied in the negative and positive global Y directions at nodes 4 and 8. These forces are numerically equal to  $\pm 66.66667$  pounds and are directed to form a moment of  $M_x = 800$  in pounds applied at the tip of the cantilever.

The displacements of the cantilever beam resulting from the above loads are given in Figure III-E.16. These displacements (U, V, W) are output corresponding to node point number and are referenced to the global axes unless otherwise noted. Figure III-E.17 shows the reactions ( $F_x, F_y, F_z$ ). These are also output corres-

ponding to node point number and are referenced to the global axes system unless otherwise specified.

The stresses arising in the structure are displayed in tabular form in Figures III-E.18 and III-E.19 for elements 1 and 6 for example. Stresses are defined at the centroid and are referenced to the global axes for each tetrahedron which makes up a particular symmetric triangular prism element. Three stress points are given under each stress category for each prism. These stress points correspond to the stresses in particular tetrahedrons which are defined in the heading of the stress data. The tetrahedron nodes listed are the local node numbering system and these must be correlated with the gridpoint numbering system. In the present case, element number one is defined as shown in the sketch below:

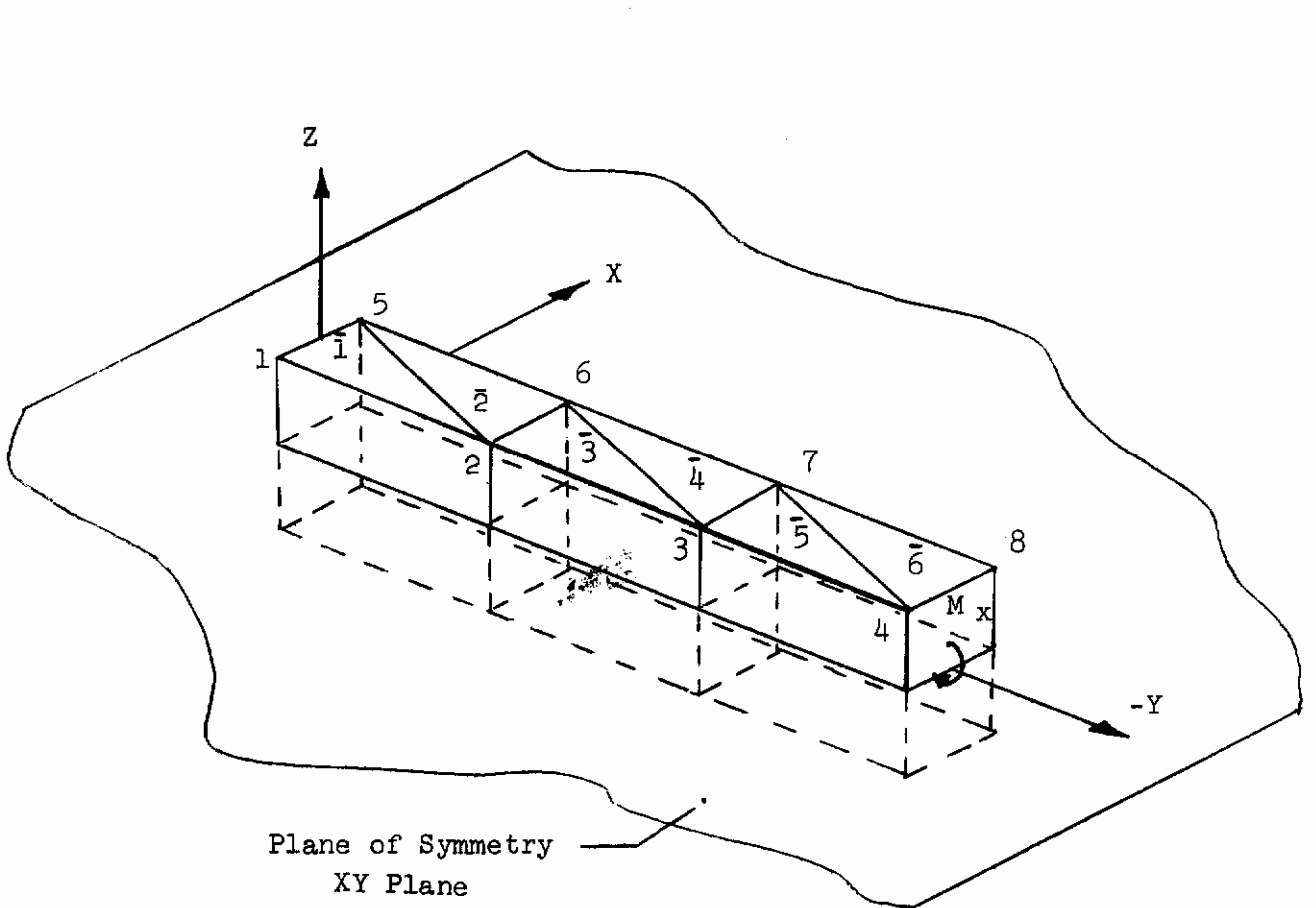


The numbers in parenthesis correspond to the local element numbering system (See Reference 7) and the other numbers are global gridpoints. The Roman numerals on the right hand sketch are the tetrahedron numbering system. The remaining elements in the idealization are handled in the same fashion.

The last set of output is given in Figures III-E.20 and III-E.21 and consist of the global oriented element forces.

# *Contrails*

Output labeling is analogous to the stress output except that the element forces are defined only at the three corner points of the symmetric triangular prism element. Three force points are given and for element number one for example, force points 1, 2, 3 correspond to element gridpoint numbers 5, 2, and 6 respectively.



$$E_x = E_y = E_z = E = 30.0 \times 10^6 \text{ psi}$$

$$\nu_{xy} = \nu_{yx} = \nu_{yz} = \nu_{zy} = \nu_{zx} = \nu_{xz} = \nu = .333$$

$$\bar{\epsilon}_x = \bar{\epsilon}_y = \bar{\epsilon}_z = 0, \quad T_1 = T_2 = \dots T_{16} = 0.0$$

$$M_{xz} = 800 \text{ in}\#$$

Figure III-E.1 Symmetrical Triangular Prism - Cantilevered Beam  
With End Moment





**MAGIC STRUCTURAL ANALYSIS SYSTEM  
INPUT DATA FORMAT**

SYSTEM CONTROL INFORMATION

ENTER APPROPRIATE NUMBER, RIGHT ADJUSTED, IN BOX OPPOSITE APPLICABLE REQUESTS

	<table border="1" style="margin: auto;"> <tr><td>S</td><td>Y</td><td>S</td><td>T</td><td>E</td><td>M</td></tr> <tr><td>1</td><td>2</td><td>3</td><td>4</td><td>5</td><td>6</td></tr> </table>	S	Y	S	T	E	M	1	2	3	4	5	6	( / )			
S	Y	S	T	E	M												
1	2	3	4	5	6												
1. Number of System Grid Points	<table border="1" style="margin: auto;"> <tr><td></td><td></td><td></td><td></td><td></td><td>8</td></tr> <tr><td>1</td><td>2</td><td>3</td><td>4</td><td>5</td><td>6</td></tr> </table>						8	1	2	3	4	5	6				
					8												
1	2	3	4	5	6												
2. Number of Input Grid Points	<table border="1" style="margin: auto;"> <tr><td></td><td></td><td></td><td></td><td></td><td>8</td></tr> <tr><td>7</td><td>8</td><td>9</td><td>10</td><td>11</td><td>12</td></tr> </table>						8	7	8	9	10	11	12				
					8												
7	8	9	10	11	12												
3. Number of Degrees of Freedom/Grid Point	<table border="1" style="margin: auto;"> <tr><td></td><td>3</td></tr> <tr><td>13</td><td>14</td></tr> </table>		3	13	14												
	3																
13	14																
4. Number of Load Conditions	<table border="1" style="margin: auto;"> <tr><td></td><td>1</td></tr> <tr><td>15</td><td>16</td></tr> </table>		1	15	16												
	1																
15	16																
5. Number of Initially Displaced Grid Points	<table border="1" style="margin: auto;"> <tr><td></td><td></td><td></td><td></td><td></td><td>0</td></tr> <tr><td>17</td><td>18</td><td>19</td><td>20</td><td>21</td><td>22</td></tr> </table>						0	17	18	19	20	21	22				
					0												
17	18	19	20	21	22												
6. Number of Prescribed Displaced Grid Points	<table border="1" style="margin: auto;"> <tr><td></td><td></td><td></td><td></td><td></td><td>0</td></tr> <tr><td>23</td><td>24</td><td>25</td><td>26</td><td>27</td><td>28</td></tr> </table>						0	23	24	25	26	27	28				
					0												
23	24	25	26	27	28												
7. Number of Grid Point Axes Transformation Systems	<table border="1" style="margin: auto;"> <tr><td></td><td>0</td></tr> <tr><td>29</td><td>30</td></tr> </table>		0	29	30												
	0																
29	30																
8. Number of Elements	<table border="1" style="margin: auto;"> <tr><td></td><td></td><td></td><td></td><td></td><td>6</td></tr> <tr><td>31</td><td>32</td><td>33</td><td>34</td><td>35</td><td>36</td></tr> </table>						6	31	32	33	34	35	36				
					6												
31	32	33	34	35	36												
9. Number of Requests and/or Revisions of Material Tape.	<table border="1" style="margin: auto;"> <tr><td></td><td>1</td></tr> <tr><td>37</td><td>38</td></tr> </table>		1	37	38												
	1																
37	38																
10. Number of Input Boundary Condition Points	<table border="1" style="margin: auto;"> <tr><td></td><td></td><td></td><td></td><td></td><td>2</td></tr> <tr><td>39</td><td>40</td><td>41</td><td>42</td><td>43</td><td>44</td></tr> </table>						2	39	40	41	42	43	44				
					2												
39	40	41	42	43	44												
11. $T_0$ For Structure (With Decimal Point)	<table border="1" style="margin: auto;"> <tr><td></td><td></td><td></td><td></td><td>0</td><td>.</td><td>0</td></tr> <tr><td>45</td><td>46</td><td>47</td><td>48</td><td>49</td><td>50</td><td>51</td><td>52</td></tr> </table>					0	.	0	45	46	47	48	49	50	51	52	( / )
				0	.	0											
45	46	47	48	49	50	51	52										

FIGURE III- E.4 SYSTEM CONTROL INFORMATION - SYMMETRIC TRIANGULAR PRISM, CANTILEVER BEAM







MAGIC STRUCTURAL ANALYSIS SYSTEM  
INPUT DATA FORMAT

1	2	3	4	5	6
LOADS					
1	2	3	4	5	6
LCOND					
1	2	3	4	5	6
1	0	0			

EXTERNAL LOADS																									
FORCE VALUES						MOMENT VALUES																			
Fx			Fy			Fz			Mx			My			Mz										
1	2	3	4	5	6	7	8	9	0	1	2	3	4	5	6	7	8	9	0	1	2	3	4	5	6
0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0

EXTERNAL LOADS																									
FORCE VALUES						MOMENT VALUES																			
Fx			Fy			Fz			Mx			My			Mz										
1	2	3	4	5	6	7	8	9	0	1	2	3	4	5	6	7	8	9	0	1	2	3	4	5	6
0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0

EXTERNAL LOADS																									
FORCE VALUES						MOMENT VALUES																			
Fx			Fy			Fz			Mx			My			Mz										
1	2	3	4	5	6	7	8	9	0	1	2	3	4	5	6	7	8	9	0	1	2	3	4	5	6
0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0

EXTERNAL LOADS																									
FORCE VALUES						MOMENT VALUES																			
Fx			Fy			Fz			Mx			My			Mz										
1	2	3	4	5	6	7	8	9	0	1	2	3	4	5	6	7	8	9	0	1	2	3	4	5	6
0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0

EXTERNAL LOADS																									
FORCE VALUES						MOMENT VALUES																			
Fx			Fy			Fz			Mx			My			Mz										
1	2	3	4	5	6	7	8	9	0	1	2	3	4	5	6	7	8	9	0	1	2	3	4	5	6
0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0

FIGURE III-E-7 EXTERNAL LOADS - SYMMETRIC TRIANGULAR PRISM, CANTILEVER BEAM



MAGIC STRUCTURAL ANALYSIS SYSTEM  
INPUT DATA FORMAT

CHECK OR END CARD

C	H	E	C	K
1	2	3	4	5

 (/)

E	N	D
1	2	3

 (/)

FIGURE III-E.9 END CARD - SYMMETRIC  
TRIANGULAR PRISM,  
CANTILEVER BEAM



TEST MAGIC

```

19 C REACTS = KELA.MLLT.XO
20 C REACTP= REACTS.SUBT.TLCAD
21 C IF (DIFF.NULL.) GO TO 10

22 C PRINT ELEMENT APPLIED LCADS, EXTERNAL LCADS, DISPLACEMENTS,
23 C REACTIONS AND INVERSE CHECK IN ENGINEERING FORMAT
24 C ELEMENTS HAVE 1 DP 2 DEGREES CF FREEDOM
25 C
26 C GPRINT(4,,,FX,FY,FZ,MX,MY,MZ,SC,TR )FTELA
27 C GPRINT(4,,,FX,FY,FZ,MX,MY,MZ,SC, )LCAES
28 C GPRINT(2,,,U,V,W,THE TAY,THE TAZ,SC, )X
29 C GPRINT(1,,,FX,FY,FZ,MX,MY,MZ,SC,TR )REACTP
30 C IF (I3.NULL.) GO TO 600

31 C ELEMENTS HAVE 3 DEGREES CF FREEDOM
32 C
33 C GPRINT(4,,,FR,O,FZ,O,FBETA,O,FL,O,F3,SC,TR )FTELA
34 C GPRINT(4,,,FR,O,FZ,O,FBETA,O,FL,O,F3,SC, )LCAES
35 C GPRINT(2,,,U,V,W,THE TAY,O,THE TAZ,SC, )X
36 C GPRINT(1,,,FR,O,FZ,O,FBETA,O,FL,O,F3,SC,TR )REACTP

37 C GENERATE STRESSES AND FORCES
38 C
39 C STRESP=EM,XD *STRESS.(4,)
40 C FORCEP=EM,XO *FORCE.(4,)

```

277

```

TESTC055
TESTC056
TESTC057
TESTC058
TESTC059
TESTC060
TESTC061
TESTC062
TESTC063
TESTC064
TESTC065
TESTC066
TESTC067
TESTC068
TESTC069
TESTC070
TESTC071
TESTC072
TESTC073
TESTC074
TESTC075
TESTC076
TESTC077
TESTC078
TESTC079
TESTC080
TESTC081

```

FIGURE III-E.10 MAGIC ABSTRACT INSTRUCTION LISTING - SYMMETRIC TRIANGULAR PRISM, CANTILEVER BEAM (CONCLUDED)



SIX ELEMENT CANTILEVERED BEAM SUBJECTED TO AN END MOMENT  
SYMMETRIC TRIANGULAR PRISM IDEN. NO.51  
STATICS ANALYSIS CONCENTRATED LOADS EQUIVALENT TO MOMENT.

REVISIONS OF MATERIAL TAPE

ASTERISK (\*) PRECEDING MATERIAL IDENTIFICATION INDICATES THAT INPUT ERROR RETURNS WILL NOT RESULT IN TERMINATION OF EXECUTION

\*\*\*\*\*

REVISION  
MATERIAL NUMBER 1  
MATERIAL IDENTIFICATION STEEL  
NUMBER OF MATERIAL PROPERTY POINTS . . . . 1  
NUMBER OF PLASTIC PROPERTY POINTS . . . . 0  
MASS DENSITY . . . . . 0.73394994E-03  
INPUT CODE I

MATERIAL PROPERTIES

YOUNG'S MODULI  
DIRECTIONS  
XX 0.30000E 08  
YY 0.30000E 08  
ZZ 0.30000E 08  
TH. EXP.CCEF.  
RIGIDITY MODULI  
DIRECTIONS  
XX 0.65000E-05  
YY 0.65000E-05  
ZZ 0.65000E-05  
POISSON'S RATIOS  
DIRECTIONS  
XY 0.33300E 00  
XZ 0.33300E 00  
YZ 0.33300E 00  
TEMPERATURE 0.0

\*\*\*\*\*

FIGURE III-E.11 TITLE AND MATERIAL DATA OUTPUT - SYMMETRIC TRIANGULAR PRISM, CANTILEVER BEAM

ELEM TYPE	NAT. NO.	CODE	TEMP.	PRAT	AC.	GRID POINTS								EXTRA GRID PTS	SECTION PROPERTIES		
1	51	1	3	0.0	C	6	1	2	5	0	0	0	0	0	C.0	0.0	0.0
2	51	1	3	0.0	C	6	5	2	6	0	0	0	0	0	C.0	0.0	0.0
3	51	1	3	0.0	0	6	2	3	6	0	0	0	0	0	C.0	0.0	0.0
4	51	1	3	0.0	0	6	6	3	7	0	0	0	0	0	C.0	0.0	0.0
5	51	1	3	0.0	0	6	3	4	7	0	0	0	0	0	C.0	0.0	0.0
6	51	1	3	0.0	C	6	7	4	8	0	0	0	0	0	C.0	0.0	0.0

FIGURE III-E.12 GRIDPOINT DATA, BOUNDARY CONDITION AND FINITE ELEMENT DESCRIPTION OUTPUT -  
 SYMMETRIC TRIANGULAR PRISM, CANTILEVER BEAM



CUTOFF = 0.0		MATRIX STIFF / I/				PAGE 1				
DISP	FORCE	FCRCE	FCRCE	FCRCE	SIZE	1P	1B			
					FORCE	RY	1B			
							FORCE			
DISP	1	0.163351E 09	2	-0.168455E 08	3	-0.262566E 08	4	-0.643016E 07	5	0.112191E 08
	10	-0.786516E 08	11	0.168455E 08	12	0.262566E 08				
DISP	2	0.168455E 08	2	0.133416E 09	3	-0.260000E 02	4	0.562640E 07	5	-0.256921E 09
	6	-0.750187E 07	10	0.168455E 08	11	-0.196924E 08	12	-0.750187E 07		
DISP	3	0.262566E 08	2	-0.260000E 02	3	0.783678E 08	4	-0.131283E 08	5	0.750187E 07
	6	-0.564524E 07	10	-0.262566E 08	11	0.750187E 07	12	-0.590773E 08		
DISP	4	0.643016E 07	2	0.562640E 07	3	-0.131283E 08	4	0.183351E 08	5	-0.168455E 08
	6	-0.262566E 08	7	-0.643014E 07	8	0.112191E 08	10	-0.393258E 08	11	-0.158455E 08
	12	0.131283E 08	13	-0.786516E 08	14	0.168455E 08	15	0.262566E 08		
DISP	5	0.112191E 08	2	-0.256821E 08	3	0.750187E 07	4	-0.168455E 08	5	0.133416E 09
	6	-0.180000E 02	7	0.562640E 07	8	-0.256821E 08	9	-0.750187E 07	10	-0.168455E 08
	11	-0.584622E 07	12	0.750187E 07	13	0.168455E 08	14	-0.196924E 08	15	-0.750187E 07
DISP	6	0.750187E 07	3	-0.964524E 07	4	-0.262566E 08	5	-0.180000E 02	6	0.783678E 08
	7	-0.131283E 08	8	0.750187E 07	9	-0.964524E 07	10	-0.131283E 08	11	-0.750187E 07
	12	-0.114253E 02	13	-0.262566E 08	14	0.750187E 07	15	-0.590772E 08		
DISP	7	0.643014E 07	5	0.562640E 07	6	-0.131283E 08	7	0.120091E 08	8	0.627544E 01
	9	-0.131283E 08	13	-0.393258E 08	14	-0.168455E 08	15	0.131283E 08	16	-0.393258E 08
	17	0.112191E 08	18	0.131283E 08						
DISP	8	0.112191E 08	5	-0.256821E 08	6	0.750187E 07	7	0.627544E 01	8	0.803633E 08
	9	-0.750187E 07	13	-0.168455E 08	14	-0.984622E 07	15	0.750187E 07	16	0.562641E 07
	17	-0.984622E 07	18	-0.750187E 07						
DISP	9	0.750187E 07	6	-0.964524E 07	7	-0.131283E 08	8	-0.750187E 07	9	0.391839E 08
	13	-0.131283E 08	14	-0.750187E 07	15	-0.170547E 02	16	-0.131283E 08	17	0.568981E-01
	18	-0.255386E 08								
DISP	10	0.786516E 08	2	0.168455E 08	3	-0.262566E 08	4	-0.393258E 08	5	-0.168455E 08
	6	-0.131283E 08	10	0.176921E 09	11	-0.168455E 08	12	0.262566E 08	13	-0.321506E 07
	14	0.562634E 07	15	0.131283E 08						
DISP	11	0.168455E 08	2	-0.196924E 08	3	0.750187E 07	4	-0.168455E 08	5	-0.584622E 07
	6	-0.750187E 07	10	-0.168455E 08	11	0.107734E 09	12	0.300000E 01	13	0.112191E 08
	14	-0.128411E 08	15	-0.750187E 07						
DISP	12	0.262566E 08	2	-0.750187E 07	3	-0.590773E 08	4	0.131283E 08	5	0.750187E 07
	6	-0.114253E 02	10	0.262566E 08	11	0.300000E 01	12	0.783678E 08	13	0.109410E 02
	14	0.750187E 07	15	-0.964524E 07						
DISP	13	0.786516E 08	5	0.168455E 08	6	-0.262566E 08	7	-0.393258E 08	8	-0.168455E 08

FIGURE III-E.14 STIFFNESS MATRIX OUTPUT - SYMMETRIC TRIANGULAR PRISM, CANTILEVER BEAM

MATRIX STIFF / 1/

CUTOFF = 0.0

DISP	FORCE	FORCE	FORCE	FORCE	SIZE	BY	FORCE	FORCE	DISP	
13	9	-0.131283E 08	10	-0.321506E 07	11	0.112191E 08	12	0.109410E 02	13	0.176921E 09
	14	-0.168455E 08	15	0.2622566E 08	16	-0.321507E 07	17	0.562635E 07	18	0.131283E 08
14	4	0.168455E 08	5	-0.196924E 08	6	0.750187E 07	7	-0.168455E 08	8	-0.584622E 07
	9	-0.750187E 07	10	0.562639E 07	11	-0.128411E 08	12	0.750187E 07	13	-0.168455E 08
	14	0.107734E 08	15	0.280000E 02	16	0.112191E 08	17	-0.128411E 08	18	-0.750187E 07
15	4	0.262565E 08	5	-0.750187E 07	6	-0.590772E 08	7	0.131283E 08	8	0.750187E 07
	9	-0.170547E 02	10	0.131283E 08	11	-0.750188E 07	12	-0.964525E 07	13	0.262566E 08
	14	0.280000E 02	15	0.783678E 08	16	0.109410E 02	17	0.750187E 07	18	-0.964525E 07
16	7	-0.393258E 08	8	0.562641E 07	9	-0.131283E 08	13	-0.321507E 07	14	0.112191E 08
	15	0.109410E 02	16	0.600452E 08	17	-0.168455E 08	18	0.131283E 08		
17	7	0.112191E 08	8	-0.984621E 07	9	0.568981E-01	13	0.562635E 07	14	-0.128411E 08
	15	0.750187E 07	16	-0.168455E 08	17	0.401916E 08	18	-0.750187E 07		
18	7	0.131283E 08	8	-0.750187E 07	9	-0.295386E 08	13	0.131283E 08	14	-0.750187E 07
	15	-0.964525E 07	16	0.131283E 08	17	-0.750187E 07	18	0.391839E 08		

FIGURE III-E.14 STIFFNESS MATRIX OUTPUT - SYMMETRIC TRIANGULAR PRISM, CANTILEVER BEAM (CONCLUDED)

PRINT OF MATRIX LOADS (SET 1)

ROW	FX	FY	FZ
1	0.0	0.0	0.0
2	0.0	0.0	0.0
3	0.0	0.0	0.0
4	0.0	-0.6666692E 02	0.0
5	0.0	0.0	0.0
6	0.0	0.0	0.0
7	0.0	0.0	0.0
8	0.0	-0.6666692E 02	0.0

FIGURE III-E.15 PRINT OF MATRIX LOADS - SYMMETRIC TRIANGULAR PRISM, CANTILEVER BEAM

DISPLACEMENT MATRIX FOR LOAD CONDITION 1

24 X 1

ROW	U	V	W
1	0.0	0.0	0.0
2	0.1645486E-04	-0.16483218E-05	-0.22326640E-05
3	-0.37463274E-07	-0.36883549E-05	-0.94028337E-05
4	-0.29041902E-06	-0.36827414E-05	-0.21379325E-04
5	0.0	0.0	0.0
6	-0.25490013E-04	-0.23282837E-05	-0.21421192E-05
7	-0.41523072E-06	-0.44505487E-05	-0.90750118E-05
8	-0.85801026E-06	-0.69309481E-05	-0.21089658E-04

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FIGURE III-E.16 DISPLACEMENT MATRIX - SYMMETRIC TRIANGULAR PRISM, CANTILEVER BEAM

REACTIONS AND INVERSE CHECK FOR LOAD CONDITION 1

ROW	FX	FY	FZ
1	-0.1959766E 02	0.60607462E 02	0.70089073E 01
2	-0.18910547E-03	0.12207631E-03	0.57983398E-03
3	-0.93078613E-03	0.16621724E-02	0.12207031E-02
4	-0.12207031E-02	0.85445214E-03	0.21972656E-02
5	0.10204174E 02	0.73314774E 02	-0.70104370E 01
6	-0.15258789E-03	0.9155734E-03	-0.21362305E-03
7	-0.5765625E-03	0.71716309E-03	-0.94604492E-03
8	-0.7324218E-03	0.27465820E-03	-0.17089844E-02

FIGURE III-E.17 REACTION MATRIX - SYMMETRIC TRIANGULAR PRISM, CANTILEVER BEAM

S T R E S S E S F O R T H E T R I A N G U L A R P R I S M E L E M E N T

(STRESS POINT ONE EQUALS ELEMENT STRESSES AT CENTROID OF TETRAHEDRON WITH NODES (1,2,3,1))  
 (STRESS POINT TWO EQUALS ELEMENT STRESSES AT CENTROID OF TETRAHEDRON WITH NODES (1,2,1,4))  
 (STRESS POINT THREE EQUALS ELEMENT STRESSES AT CENTROID OF TETRAHEDRON WITH NODES (2,6,5,4))

LOAD CONDITION NUMBER	ELEMENT NUMBER	ELEMENT TYPE	ELEMENT GRID POINTS							
1	1	51	1	2	5	0	1	0	0	
APPARENT ELEMENT STRESSES										
STRESS POINT	SIGMA-X	SIGMA-Y	SIGMA-Z	MEMBRANE STRESSES			SIGMA-YZ	SIGMA-ZX		
1	0.52836256E 01	0.10583124E 02	0.52836256E 01	SIGMA-XY	SIGMA-YZ	SIGMA-ZX	0.0	0.0		
2	0.52836256E 01	0.10583124E 02	0.52836256E 01	-0.26451939E 00	0.35891075E 01	0.0	0.0	0.0		
3	0.0	0.0	0.0	-0.26451939E 00	0.35891075E 01	0.0	0.0	0.0		
ELEMENT APPLIED STRESSES										
STRESS POINT	SIGMA-X	SIGMA-Y	SIGMA-Z	MEMBRANE STRESSES			SIGMA-YZ	SIGMA-ZX		
1	0.0	0.0	0.0	SIGMA-XY	SIGMA-YZ	SIGMA-ZX	0.0	0.0		
2	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0		
3	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0		
NET ELEMENT STRESSES										
STRESS POINT	SIGMA-X	SIGMA-Y	SIGMA-Z	MEMBRANE STRESSES			SIGMA-YZ	SIGMA-ZX		
1	0.52836256E 01	0.10583124E 02	0.52836256E 01	SIGMA-XY	SIGMA-YZ	SIGMA-ZX	0.0	0.0		
2	0.52836256E 01	0.10583124E 02	0.52836256E 01	-0.26451939E 00	0.35891075E 01	0.0	0.0	0.0		
3	0.0	0.0	0.0	-0.26451939E 00	0.35891075E 01	0.0	0.0	0.0		

FIGURE III-E.18 STRESS OUTPUT, ELEMENT NO. 1 - SYMMETRIC TRIANGULAR PRISM, CANTILEVER BEAM



STRESSES FOR THE TRIANGULAR PRISM ELEMENT

(STRESS POINT ONE EQUALS ELEMENT STRESSES AT CENTROID OF TETRAHEDRON WITH NODES (1,2,3,4))  
 (STRESS POINT TWO EQUALS ELEMENT STRESSES AT CENTROID OF TETRAHEDRON WITH NODES (1,2,1,4))  
 (STRESS POINT THREE EQUALS ELEMENT STRESSES AT CENTROID OF TETRAHEDRON WITH NODES (2,6,5,4))

LOAD CONDITION NUMBER	ELEMENT NUMBER	ELEMENT TYPE	ELEMENT GRID POINTS	SIGMA-X	SIGMA-Y	SIGMA-Z	SIGMA-XY	SIGMA-YZ	SIGMA-ZX
1	6	51	7 4 8 0 C C	0.15733185E C1	0.12741470E C2	0.47667999E 01	-0.28559265E 01	-0.66831665E C1	-0.24040375E 01
				-0.14023790E C1	-0.70013865E C0	-0.70014328E 00	0.34100904E 01	0.26205928E C1	-0.74319458E 00
				-0.94668394E-C6	-0.18962115E-C5	-0.94668394E-06	-0.24348736E-07	-0.19261760E C1	-0.27502641E 00

ELEMENT APPLIED STRESSES

STRESS POINT	SIGMA-X	SIGMA-Y	SIGMA-Z	SIGMA-XY	SIGMA-YZ	SIGMA-ZX
1	0.0	0.0	0.0	0.0	0.0	0.0
2	0.0	0.0	0.0	0.0	0.0	0.0
3	0.0	0.0	0.0	0.0	0.0	0.0

NET ELEMENT STRESSES

STRESS POINT	SIGMA-X	SIGMA-Y	SIGMA-Z	SIGMA-XY	SIGMA-YZ	SIGMA-ZX
1	0.15733185F C1	0.12741470E C2	0.47667999E 01	-0.28559265E 01	-0.66831665E C1	-0.24040375E 01
2	-0.14023790E C1	-0.70013865E C0	-0.70014328E 00	0.34100904E 01	0.26205928E C1	-0.74319458E 00
3	-0.94668394E-C6	-0.18962115E-C5	-0.94668399E-06	-0.24348736E-07	-0.19261760E C1	-0.27502641E 00

FIGURE III-E.19 STRESS OUTPUT, ELEMENT NO. 6 - SYMMETRIC TRIANGULAR PRISM, CANTILEVER BEAM

F O R C E S F O R T H E T R I A N G U L A R P R I S M E L E M E N T

LOAD CONDITION NUMBER 1 ELEMENT NUMBER 1 ELEMENT TYPE 51 ELEMENT GRID POINTS 1 2 5 0 C C

APPARENT ELEMENT FORCES

POINT	FX	FY	FZ
1	-0.19550766E C2	0.6007462E O2	0.70089073E O1
2	0.39383974E C1	-0.5436157E C2	-0.91691189E O1
3	0.18492691E C2	0.15823343E C2	0.21602411E O1
4	0.0	0.0	0.0
5	0.0	0.0	0.0
6	0.0	0.0	0.0

ELEMENT APPLIED FORCES

POINT	FX	FY	FZ
1	0.0	0.0	0.0
2	0.0	0.0	0.0
3	0.0	0.0	0.0
4	0.0	0.0	0.0
5	0.0	0.0	0.0
6	0.0	0.0	0.0

NET ELEMENT FORCES

POINT	FX	FY	FZ
1	-0.19550766E O2	0.6007462E O2	0.70089073E O1
2	0.39383974E C1	-0.5436157E C2	-0.91691189E O1
3	0.18492691E C2	0.15823343E C2	0.21602411E O1
4	0.0	0.0	0.0
5	0.0	0.0	0.0
6	0.0	0.0	0.0

FIGURE III-E.20 FORCE OUTPUT, ELEMENT No. 1 - SYMMETRIC TRIANGULAR PRISM, CANTILEVER BEAM

F O R C E S F O R T H E T R I A N G U L A R P R I S M E L E M E N T

LOAD CONDITION NUMBER 1 ELEMENT ALPHEA 6 ELEMENT TYPE 51 ELEMENT GRID POINTS 7 4 8 0 C 0

APPARENT ELEMENT FORCES

POINT	FX	FY	FZ
1	-0.14082379E C2	0.49647873E 02	-0.11977249E 02
2	-0.18815918E C1	-0.10528375E C2	0.11978271E 02
3	-0.48828125E-C3	-0.66666718E 02	-0.12207031E-02
4	0.0	0.0	0.0
5	0.0	0.0	0.0
6	0.0	0.0	0.0

ELEMENT APPLIED FORCES

POINT	FX	FY	FZ
1	0.0	0.0	0.0
2	0.0	0.0	0.0
3	0.0	0.0	0.0
4	0.0	0.0	0.0
5	0.0	0.0	0.0
6	0.0	0.0	0.0

NET ELEMENT FORCES

POINT	FX	FY	FZ
1	-0.14082379E C2	0.49647873E 02	-0.11977249E 02
2	-0.18815918E C1	-0.10528375E C2	0.11978271E 02
3	-0.48828125E-C3	-0.66666718E 02	-0.12207031E-02
4	0.0	0.0	0.0
5	0.0	0.0	0.0
6	0.0	0.0	0.0

FIGURE III-E.21 FORCE OUTPUT, ELEMENT No. 6 - SYMMETRIC TRIANGULAR PRISM, CANTILEVER BEAM

## F. SYMMETRIC SHEAR WEB

A two-bay cantilevered box beam is idealized by use of the symmetric shear web, axial force and quadrilateral shear panel finite elements and serves as the fifth example problem. This structure is shown in Figure III-F.1. with the attendant loading, idealization, dimensions and material properties. The preprinted input data forms associated with this example problem are given in Figure III-F.2 to III-F.10. Note the manner in which the boundary conditions Figure III-F.6 are imposed in this example. First, all degrees-of-freedom are fixed through use of the MODAL card, then the exceptions are designated on the following cards. The REPEAT option is used to advantage here also. A comment must be made here with respect to the symmetric shear web element. The plane of symmetry used for this element must always be the global XY plane thus the element is oriented perpendicular to this plane, and  $z = Z$ . That is, the local  $z$  coordinate of the grid points which define the element are identical to the global  $Z$  coordinates of these same points.

The following load data is evident by inspection of Figure III-F.7, External Loads Section.

- 1) One load condition is input
- 2) The external applied load scalar is zero
- 3) Grid point 6 is loaded with a force in the positive  $Z$  direction equal to 1000.0 pounds.

Note that no entries corresponding to External Moments are made since the elements used in the idealization do not accommodate such loadings.

Note that external element input data are needed for the finite elements used in this example. The axial force elements require cross-sectional area, the symmetric shear web and quadrilateral shear panel elements require thickness. These data are shown on Figure III-F.9.

# Contrails

The output supplied by the MAGIC III system for this particular example is described below and shown in Figures III-F.11 to III-F.12.

Figure III-F.11 shows the matrix abstraction instructions associated with this example. A complete description of these instructions is provided in Reference 5. Figures III-F.12 to III-F.15 display the output from the Structural Systems Monitor. These figures record the input data pertinent to the problem being solved.

Figure III-F.12 displays the problem title and material data output. The grid point coordinates, temperatures and pressures are given in Figure III-F.13. In addition, boundary condition information and finite element descriptions are also shown on this figure. In the boundary condition portion of the figure, zeros ('0') represent degrees of freedom that are fixed (i.e., no motion), ones ('1') represent degrees of freedom that are free or have unknown values of displacement, and two's ('2') represent degrees of freedom that are eliminated in the analysis procedure through the condensation technique. The second last column represents the cumulative number of degrees of freedom which actively participate in the equation solving process for displacements. The last column accumulates the number of twos which participate in the calculation of the reduced stiffness matrix which is not used in the present example. Figure III-F.14 shows the finite element description. Each of the elements is called out in turn with grid points, print options and material number. Note that extra grid points are needed for the axial element in order to define the orientation of the local axes system for this element. The section properties previously discussed are also listed in the right hand column of the figure.

Figure III-F.15 displays the external load condition and the transformed external assembled load column. This 42 x 1 vector is the total unreduced load which is read row-wise. The ordering of this vector is consistent with that of the boundary condition table

# Contrails

given in Figure III-F.13. Note that a load of 1000.0 pounds is applied at node point 6 in the positive global Z direction. This is position (33,1) in the load vector which corresponds to the thirty-third entry in the boundary condition table which is the global w displacement for node point 6.

MAGIC III system output of final results are displayed in Figures III-F.16 to III-F.24. Figure III-F.16 shows the stiffness matrix for this problem. It is noted that only the non-zero terms are displayed. The stiffness matrix is presented row-wise and its ordering is consistent with that of the boundary condition table previously discussed. In this problem the ordering is

$$\{\Delta\}^T = \left[ U_3, V_3, W_3, U_4, V_4, W_4, \dots, U_6, V_6, W_6 \right].$$

The externally applied load vector (GPRINT OF MATRIX LOADS) is presented in Figure III-F.17. This figure shows that a force ( $F_z$ ) is applied in the positive global Z direction at node point 6.

The displacements and reactions of the cantilever beam resulting from the above loads are given in Figure III-F.18. It is noted that the displacements (U, V, W) are output corresponding to node point number and are referenced to the global axes unless otherwise specified. The second portion of Figure III-F.19 shows the reactions ( $F_x, F_y, F_z$ ). These are also output corresponding to node point number and are referenced to the global axes system unless otherwise specified.

The stresses arising in the structure are displayed in tabular form in Figures III-F.19 to III-F.24. Stress data for the axial force elements, elements 1 to 6, are referenced to the element coordinate system and defined to be the axial force acting at the two grid point connections. Figure III-F.19 presents typical results wherein stress points 1 and 2 correspond to the element end grid points.



# Contrails

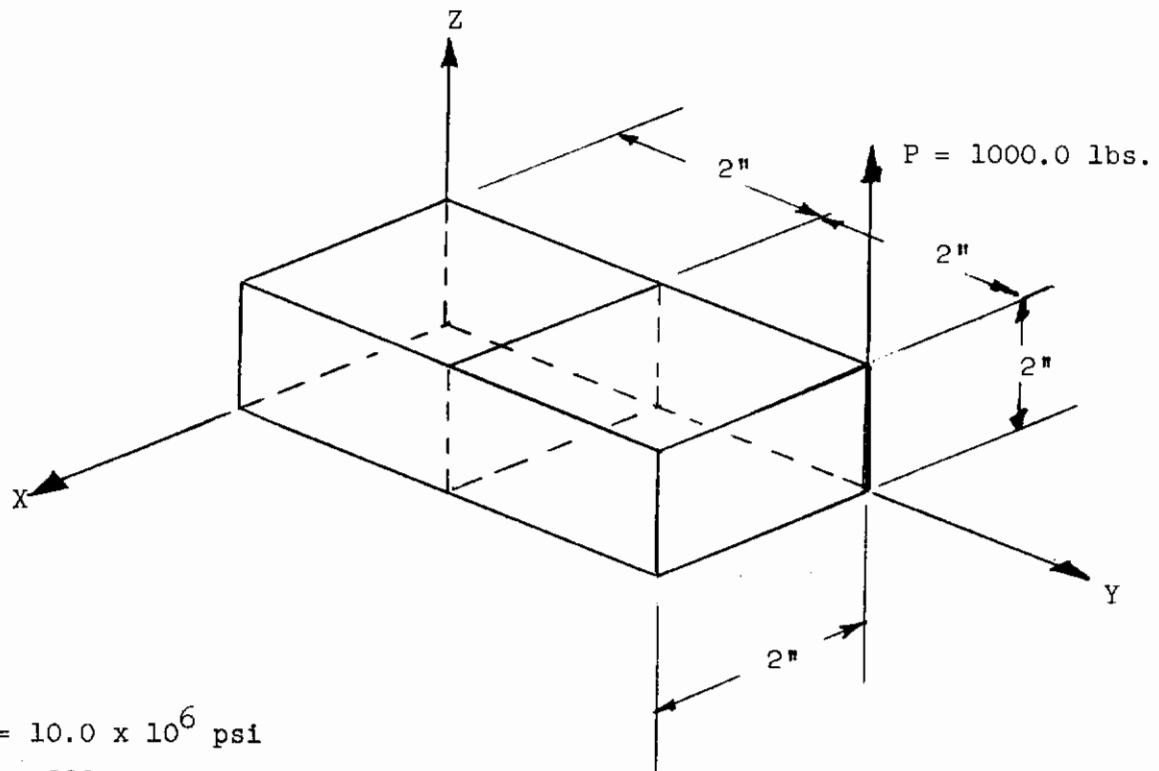
Six stress value headings are printed for the apparent element stress, element applied stress and net element stress categories. The apparent stress arises from element deformations and the applied stress arises from pre-strain and thermal effects. The net stress is the difference between the apparent and applied stress values. In this instance only the axial heading has entries, the remaining headings are used for the frame element. (See Reference 5 page 227.)

Stress values for the symmetric shear web element are typically presented for element one in Figure III-F.20. The membrane shear stress is listed for each of the three categories for one element stress point, that being the centroid of the element. These stresses are oriented to the local axes system. The final set of stresses are typically displayed for element thirteen in Figure III-F.28 for the quadrilateral shear panel elements and are tabulated in the same fashion as in the shear web element.

The last set of output is given in Figures III-F.22 to III-F.24 which displays the element forces for each of the three elements. These forces are given in the global system. The force points correspond to the end points of the axial elements. In element one, for example, force point 1 corresponds to grid point 1 and force point 2 to grid point 3. Data for the remaining element types is presented in the same fashion.



# Contrails



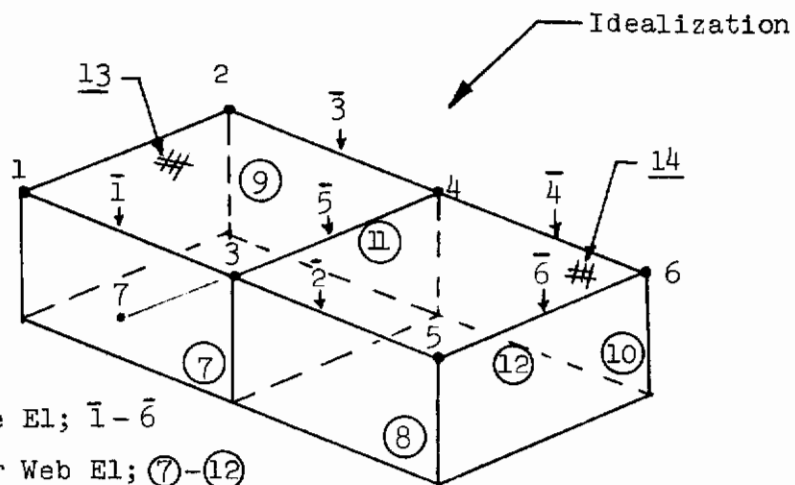
$$E = 10.0 \times 10^6 \text{ psi}$$

$$\nu = .300$$

$$E_x = E_y = E_z = 0.0$$

$$T_1 = T_2 = \dots = T_6 = 0.0$$

$$P = 1000.0 \text{ lbs.}$$



Note:

(1) Axial Force El;  $\bar{1}-\bar{6}$

(2) Symm. Shear Web El;  $\bar{7}-\bar{12}$

(3) Quad. Shear El;  $\bar{13}-\bar{14}$

Figure III-F.1 Box Beam With Symmetric Shear Web Element





MAGIC STRUCTURAL ANALYSIS SYSTEM  
INPUT DATA FORMAT

SYSTEM CONTROL INFORMATION

ENTER APPROPRIATE NUMBER, RIGHT ADJUSTED, IN BOX OPPOSITE APPLICABLE REQUESTS

- |     |   |    |    |    |    |    |     |     |    |
|-----|---|----|----|----|----|----|-----|-----|----|
|     |   | S  | Y  | S  | T  | E  | M   | (/) |    |
|     |   | 1  | 2  | 3  | 4  | 5  | 6   |     |    |
| 1.  | Number of System Grid Points                          |    |    |    |    |    | 7   |     |    |
|     |   | 1  | 2  | 3  | 4  | 5  | 6   |     |    |
| 2.  | Number of Input Grid Points                           |    |    |    |    |    | 7   |     |    |
|     |   | 7  | 8  | 9  | 10 | 11 | 12  |     |    |
| 3.  | Number of Degrees of Freedom/Grid Point               |    |    |    |    |    | 6   |     |    |
|     |   |    |    |    |    | 13 | 14  |     |    |
| 4.  | Number of Load Conditions                             |    |    |    |    |    | 1   |     |    |
|     |   |    |    |    |    | 15 | 16  |     |    |
| 5.  | Number of Initially Displaced Grid Points             |    |    |    |    |    | 0   |     |    |
|     |   | 17 | 18 | 19 | 20 | 21 | 22  |     |    |
| 6.  | Number of Prescribed Displaced Grid Points            |    |    |    |    |    | 0   |     |    |
|     |   | 23 | 24 | 25 | 26 | 27 | 28  |     |    |
| 7.  | Number of Grid Point Axes Transformation Systems      |    |    |    |    |    | 0   |     |    |
|     |   |    |    |    |    | 29 | 30  |     |    |
| 8.  | Number of Elements                                    |    |    |    |    |    | 14  |     |    |
|     |   | 31 | 32 | 33 | 34 | 35 | 36  |     |    |
| 9.  | Number of Requests and/or Revisions of Material Tape. |    |    |    |    |    | 1   |     |    |
|     |   |    |    |    |    | 37 | 38  |     |    |
| 10. | Number of Input Boundary Condition Points             |    |    |    |    |    | 4   |     |    |
|     |   | 39 | 40 | 41 | 42 | 43 | 44  |     |    |
| 11. | T <sub>0</sub> For Structure (With Decimal Point)     |    |    |    |    |    | 0.0 | (/) |    |
|     |   | 45 | 46 | 47 | 48 | 49 | 50  | 51  | 52 |

FIGURE III-F.4 SYSTEM CONTROL INFORMATION - SYMMETRIC SHEAR WEB CANTILEVERED BEAM















MAGIC STRUCTURAL ANALYSIS SYSTEM

INPUT DATA FORMAT

CHECK OR END CARD

C	H	E	C	K
1	2	3	4	5

 (/)

E	N	D
1	2	3

 (/)

FIGURE III-F.10 END CARD - SYMMETRIC SHEAR WEB, CANTILEVERED BEAM'

TEST MAGIC

\$INSTRUCTION	SOURCE	TEST0004
C	-----STATICS AGENDUM WITHOUT PRESCRIBED DISPLACEMENTS	TEST0005
C	* * * * *	TEST0006
C	* * * * *	TEST0007
C	* * * * *	TEST0008
C	STATICS INSTRUCTION SEQUENCE	TEST0009
C	* * * * *	TEST0010
C	* * * * *	TEST0011
C	* * * * *	TEST0012
C	GENERATE ELEMENT MATRICES	TEST0013
C	*MLIB,,XLD,TR, *MEL,FTEL,SEL,STEL,,SC,EM, ,,, *USER04,	TEST0014
C		TEST0015
C		TEST0016
C	FORM (1 X 1) UNIT AND (1 X 1) NULL MATRICES	TEST0017
C	DETERMINE PRINT FORMAT FOR TYPE OF ELEMENTS USED	TEST0018
C		TEST0019
C		TEST0020
C	I1 = SC *IDENTC.	TEST0021
C	I3 = I1 *NULL,SC	TEST0022
C	DIFF = I1 *SMULT, SC (5,1)	TEST0023
C		TEST0024
C		TEST0025
C	ASSEMBLE STIFFNESS MATRIX AND ELEMENT APPLIED LOADS	TEST0026
C		TEST0027
C	KELA = EM *ASSEM, SC, (1,C)	TEST0028
C	FTELA = EM *ASSEM, SC, (4,C)	TEST0029
C	LSCALE,LOADS = XLD *DEJCTN, (1,1)	TEST0030
C	REDUCE STIFFNESS MATRIX AND PRINT	TEST0031
C		TEST0032
C	K0,KNO = KELA *DEJCTN, ( SC(5,1),1)	TEST0033
C	KCO,STIFF = KNO *DEJCTN, ( SC(5,1),0)	TEST0034
C	PRINT(FORCE,DISP,,) STIFF	TEST0035
C		TEST0036
C	FORM REDUCED TOTAL LCAC COLUMN	TEST0037
C		TEST0038
C	MULTIPLY ELEMENT APPLIED LOADS BY LCAC SCALAR	TEST0039
C	FTELS = FTELA *MULT, LSCALE	TEST0040
C	TRANSFORM EXTERNAL LCACS TO 0-1-2 ASSEMBLED SYSTEM	TEST0041
C	LOADO = TR *MULT, LOADS	TEST0042
C	FORM TOTAL LOAD COLUMNS	TEST0043
C	TLJAD = FTELS *ADD, LOADO	TEST0044
C	TL, TLOADR = TLOAD *DEJCTN, ( SC(5,1),1)	TEST0045
C		TEST0046
C	SOLVE FOR DISPLACEMENTS	TEST0047
C		TEST0048
C	XX = STIFF *SEQL, TLOADR	TEST0049
C	TR0, TR12 = TR *DEJCTN, (SC(5,1),1)	TEST0050
C	X = TR12 *MULT, X	TEST0051
C		TEST0052
C		TEST0053
C	CALCULATE REACTIONS AND INVERSE CHECK	TEST0054

FIGURE III-F-11 FORMAT ABSTRACTION INSTRUCTION LISTING - SYMMETRIC SHEAR WEB, CANTILEVERED BEAM

TEST MAGIC

```

C      19      REACTS = KELA,MLLY,XC
C      20      REACTP= REACTS.SUBT.TLCAD
C      21      IF (DIFF.NULL.) GO TO 10

C      PRINT ELEMENT APPLIED LCALS, EXTERNAL LCADS, DIS PLACEMENTS,
C      REACTIONS AND INVERSE CHECK IN ENGINEERING FORMAT

C      ELEMENTS HAVE 1 OR 2 DEGREES CF FREEDCM
C
C      GPR INTI 4,,,FX,FY,FZ,MY,MZ,SC,TF )FTELA
C      GPR INTI 2,,,U,V,W,THE1AY,THE1AZ,SC,IX
C      GPR INTI 1,,,FX,FY,FZ,MY,MZ,SC,TF )REACTP
C      IF (I3.NULL.) GO TO 6CC

C      ELEMENTS HAVE 3 DEGREES CF FREEDCM
C
C      10      GPR INTI 4,,,FR,O,FZ,C,PBETA,C,FI,O,F3,SC,TR )FTELA
C      GPR INTI 2,,,U,C,h,C,THE1AY,C,h,O,h*1,SC,IX
C      GPR INTI 1,,,FR,O,FZ,O,PBETA,C,FI,O,F3,SC,TR )REACTP

C      GENERATE STRESSES AND FORCES
C
C      600     STRESP=EM,XO *STRESS.(4,)
C      FORCEP=EM,XO *FORCE.(4,)

```

```

TESTD055
TESTD056
TESTD057
TESTD058
TESTD059
TESTD060
TESTD061
TESTD062
TESTD063
TESTD064
TESTD065
TESTD066
TESTD067
TESTD068
TESTD069
TESTD070
TESTD071
TESTD072
TESTD073
TESTD074
TESTD075
TESTD076
TESTD077
TESTD078
TESTD079
TESTD080
TESTD081

```

FIGURE III-F.11 MAGIC ABSTRACTION INSTRUCTION LISTING - SYMMETRIC SHEAR WEB, CANTILEVERED BEAM, CONCL'D.

BOX BEAM CANTILEVERED WITH TIP LOAD.  
SYMMETRIC SHEAR WEB, EQUAL-SPEAK PANEL AND AXIAL FORCE ELEMENTS  
STATICS ANALYSIS.

REVISIONS OF MATERIAL TAPE

ASTERISK (\*) PRECEDING MATERIAL IDENTIFICATION INDICATES THAT INPUT CHECK RETURN WILL NOT RESULT IN TERMINATION OF EXECUTION.

\*\*\*\*\*

REVISION

MATERIAL NUMBER 77  
MATERIAL IDENTIFICATION ALLUMINUM  
NUMBER OF MATERIAL PROPERTY POINTS . . . . 1  
NUMBER OF PLASTIC PROPERTY POINTS . . . . 0  
MASS DENSITY . . . . . 0.0

MATERIAL PROPERTIES

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YOUNG'S MODULI		POISSON'S RATIOS	
DIRECTIONS		DIRECTIONS	
XX	0.10000E 08	XY	0.30000E 00
YY	0.10000E 08	YZ	0.30000E 00
ZZ	0.10000E 08	ZX	0.30000E 00
TH. EXP. CCEF.			
DIRECTIONS		RIGIDITY MODULI	
XX	0.65000E -05	XY	0.38461E 07
YY	0.65000E -05	YZ	0.38461E 07
ZZ	0.65000E -05	ZX	0.38461E 07

\*\*\*\*\*

FIGURE III-F.12 TITLE AND MATERIAL DATA OUTPUT - SYMMETRIC SHEAR WEB, CANTILEVERED BEAM

7 REF. POINTS

NO. DIRECTIONS = 3    NO. DEGREES OF FREEDOM = 2

GRIDPOINT DATA  
(IN RECTANGULAR COORDINATES)

POINT	X	Y	Z	TEMPERATURES	PRESSURES
1	0.2000000E 01	0.0	0.2000000E 01	0.0 0.0	0.0 0.0
2	0.0	0.0	0.2000000E 01	0.0 0.0	0.0 0.0
3	0.2000000E 01	0.2000000E 01	0.2000000E 01	0.0 0.0	0.0 0.0
4	0.0	0.2000000E 01	0.2000000E 01	0.0 0.0	0.0 0.0
5	0.2000000E 01	0.4000000E 01	0.2000000E 01	0.0 0.0	0.0 0.0
6	0.0	0.4000000E 01	0.2000000E 01	0.0 0.0	0.0 0.0
7	0.4000000E 01	0.2000000E 01	0.2000000E 01	0.0 0.0	0.0 0.0

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BOUNDARY CONDITION INFORMATION

NODES	DEGREES OF FREEDOM	NO. OF ONES	NO. OF TWOS
1	C	0	0
2	C	0	0
3	1	3	0
4	1	6	0
5	1	9	0
6	1	12	0
7	C	12	0

FIGURE III-F.13 GRIDPOINT DATA AND BOUNDARY CONDITIONS - SYMMETRIC SHEAR WEB, CANTILEVERED BEAM







GPRINT CF MATRIX LOADS (SET 1)

ROW	FX	FY	FZ	MX	MY	MZ
1	0.0	0.0	0.0	0.0	0.0	0.0
2	0.0	0.0	0.0	0.0	0.0	0.0
3	0.0	0.0	0.0	0.0	0.0	0.0
4	0.0	0.0	0.0	0.0	0.0	0.0
5	0.0	0.0	0.0	0.0	0.0	0.0
6	0.0	0.0	0.1000000E 04	0.0	0.0	0.0
7	0.0	0.0	0.0	0.0	0.0	0.0

FIGURE III-F.17 GPRINT OF MATRIX LOADS - SYMMETRIC SHEAR WEB, CANTILEVERED BEAM

DISPLACEMENT MATRIX FOR LOAD CONDITION 1

42 X 1

ROW	U	W	THETA X	THETA Y	THETA Z
1	0.0	0.0	0.0	C.C	0.0
2	0.0	0.0	0.0	C.C	0.0
3	0.16555938E-02	0.19294322E-02	0.0	C.C	0.0
4	0.16555972E-02	0.38705571E-02	0.0	C.C	0.0
5	0.30805431E-02	0.40019090E-02	0.0	C.C	0.0
6	0.30805506E-02	0.83991885E-02	0.0	C.C	0.0
7	0.0	0.0	0.0	C.C	0.0

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REACTIONS AND INVERSE CHECK FOR LOAD CONDITION 1

ROW	FX	FY	FZ	MX	MY	NZ
1	-0.15417577E 03	0.11235535E 04	-0.30833359E 03	0.0	C.C	0.0
2	-0.15417577E 03	0.87644556E 03	-0.69166138E 03	0.0	C.C	0.0
3	-0.61035156E-02	0.20294189E-02	-0.36621094E-02	0.0	C.C	0.0
4	0.34179688E-02	0.24414063E-03	0.0	0.0	C.C	0.0
5	-0.11718750E-01	0.32643457E-03	0.24414063E-03	0.0	C.C	0.0
6	0.53710538E-02	0.14648438E-02	0.19531250E-02	0.0	C.C	0.0
7	0.0	0.0	0.0	0.0	C.C	0.0

FIGURE III-F.18 DISPLACEMENT AND REACTIONS MATRICES - SYMMETRIC SHEAR WEB, CANTILEVERED BEAM

STRESSES FOR THE FRAME ELEMENT

LOAD CONDITION NUMBER	ELEMENT NUMBER	ELEMENT TYPE	ELEMENT GRID POINTS			FLEXURAL MOMENTS NORMAL (MY)	ACR PAL (MZ)
			1	3	7		
1	1	11					
APPARENT ELEMENT STRESSES							
STRESS POINT	AXIAL (FX)	FORCES SHEAR (FY)	SHEAR (FZ)	TORQUE (MX)	FLEXURAL MOMENTS NORMAL (MY)	ACR PAL (MZ)	
1	0.81521045E C3	C.C	0.0	0.0	C.0	0.0	
2	-0.81521045E C3	C.0	0.0	0.0	C.0	0.0	
ELEMENT APPLIED STRESSES							
STRESS POINT	AXIAL (FX)	FORCES SHEAR (FY)	SHEAR (FZ)	TORQUE (MX)	FLEXURAL MOMENTS NORMAL (MY)	ACR PAL (MZ)	
1	0.0	C.C	0.0	0.0	C.0	0.0	
2	0.0	C.0	0.0	0.0	C.0	0.0	
NET ELEMENT STRESSES							
STRESS POINT	AXIAL (FX)	FORCES SHEAR (FY)	SHEAR (FZ)	TORQUE (MX)	FLEXURAL MOMENTS NORMAL (MY)	ACR PAL (MZ)	
1	0.81521045E C3	0.0	0.0	0.0	C.0	0.0	
2	-0.81521045E C3	0.0	0.0	0.0	C.0	0.0	

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FIGURE III-F.19 STRESS OUTPUT, ELEMENT NO. 1 - SYMMETRIC SHEAR WEB, CANTILEVERED BEAM

STRESSES FOR THE SYMMETRIC SHEAR WEB ELEMENT

(STRESSES EVALUATED AT THE ELEMENT CENTROID)

LOAD CONDITION NUMBER	ELEMENT NUMBER	ELEMENT TYPE	ELEMENT GRID POINTS
1	7	29	1 3

APPARENT ELEMENT STRESSES

STRESS POINT 1  
MEMBRANE SHEAR STRESS  
0.30833616E 04

ELEMENT APPLIED STRESSES

STRESS POINT 1  
MEMBRANE SHEAR STRESS  
0.0

NET ELEMENT STRESSES

STRESS POINT 1  
MEMBRANE SHEAR STRESS  
0.30833616E 04

FIGURE III-F.20 STRESS OUTPUT, ELEMENT NO. 7 - SYMMETRIC SHEAR WEB, CANTILEVERED BEAM

STRESSES FOR THE CULVERT LATERAL SPRAY PANEL ELEMENT

(STRESSES EVALUATED AT ELEMENT CENTROID)

LOAD CONDITION NUMBER	ELEMENT NUMREF	ELEMENT TYPE	ELEMENT GRID POINTS
1	13	25	1 3 4 2

APPARENT ELEMENT STRESSES

STRESS POINT 1  
 MEMBRANE SHEAR STRESS  
 -0.30835161E 04

ELEMENT APPLIED STRESSES

STRESS POINT 1  
 MEMBRANE SHEAR STRESS  
 0.0

NET ELEMENT STRESSES

STRESS POINT 1  
 MEMBRANE SHEAR STRESS  
 -0.30835161E 04

FIGURE III-F.21 STRESS OUTPUT, ELEMENT NO. 13 - SYMMETRIC SHEAR WEB, CANTILEVERED BEAM



F C F C E S F C R T H E F R A M E E L E M E N T

LOAD CONDITION NUMBER	ELEMENT NUMBER	ELEMENT TYPE	ELEMENT GRID POINTS			FLEXURAL MOMENTS NORMAL (MY)	NORMAL (MZ)
			1	3	7		
<b>APPARENT ELEMENT FORCES</b>							
	1	11					
POINT	AXIAL (FX)	SHEAR (FZ)	TORQUE (MX)				
	FORCES						
	SHEAR (FY)						
1	0.0	0.0	0.0	0.0	0.0	0.0	0.0
2	0.0	0.0	0.0	0.0	0.0	0.0	0.0
3	0.0	0.0	0.0	0.0	0.0	0.0	0.0
<b>ELEMENT APPLIED FORCES</b>							
	1						
POINT	AXIAL (FX)	SHEAR (FZ)	TORQUE (MX)				
	FORCES						
	SHEAR (FY)						
1	0.0	0.0	0.0	0.0	0.0	0.0	0.0
2	0.0	0.0	0.0	0.0	0.0	0.0	0.0
3	0.0	0.0	0.0	0.0	0.0	0.0	0.0
<b>NET ELEMENT FORCES</b>							
	1						
POINT	AXIAL (FX)	SHEAR (FZ)	TORQUE (MX)				
	FORCES						
	SHEAR (FY)						
1	0.0	0.0	0.0	0.0	0.0	0.0	0.0
2	0.0	0.0	0.0	0.0	0.0	0.0	0.0
3	0.0	0.0	0.0	0.0	0.0	0.0	0.0

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FIGURE III-F.22 FORCE OUTPUT, ELEMENT NO. 1 - SYMMETRIC SHEAR WEB, CANTILEVERED BEAM



FORCES FOR THE QUADRILATERAL SHEAR PANEL ELEMENT

LOAD CONDITION NUMBER	ELEMENT NUMBER	ELEMENT TYPE	ELEMENT GRID POINTS				
1	13	25	1 3 4 2				
APPARENT ELEMENT FORCES							
POINT	FX	FY	FZ	MX	MY	MZ	
1	-0.15417577E 03	0.15417577E 03	0.0	0.0	0.0	0.0	
2	0.15417577E 03	0.15417577E 03	0.0	0.0	0.0	0.0	
3	0.15417577E 03	-0.15417577E 03	0.0	0.0	0.0	0.0	
4	-0.15417577E 03	-0.15417577E 03	0.0	0.0	0.0	0.0	
ELEMENT APPLIED FORCES							
POINT	FX	FY	FZ	MX	MY	MZ	
1	0.0	0.0	0.0	0.0	0.0	0.0	
2	0.0	0.0	0.0	0.0	0.0	0.0	
3	0.0	0.0	0.0	0.0	0.0	0.0	
4	0.0	0.0	0.0	0.0	0.0	0.0	
NET ELEMENT FORCES							
POINT	FX	FY	FZ	MX	MY	MZ	
1	-0.15417577E 03	0.15417577E 03	0.0	0.0	0.0	0.0	
2	0.15417577E 03	0.15417577E 03	0.0	0.0	0.0	0.0	
3	0.15417577E 03	-0.15417577E 03	0.0	0.0	0.0	0.0	
4	-0.15417577E 03	-0.15417577E 03	0.0	0.0	0.0	0.0	

FIGURE III-F.24 FORCE OUTPUT, ELEMENT NO. 13 - SYMMETRIC SHEAR WEB, CANTILEVERED BEAM

## G. MODIFIED QUADRILATERAL

A four element idealization of a structural joint is shown in Figure III-G.1. This figure shows the loading, idealization, dimension and material properties. The problem is one of those shown in Reference 9 page 329 wherein the effects of the modification of this element were evaluated. The preprinted input data forms associated with this example are given in Figure III-G.2 to III-G.10.

Of interest is the Boundary Condition Section, Figure III-G.6 which shows the use of the MODAL and REPEAT options. There are 8 exceptions to the MODAL card. Grid points 4, 6, 11, and 14 have the same boundary conditions as grid point 1, therefore the option is employed by placing an "X" in column 12 opposite the entry for grid points 4, 6, 11 and 14. The same procedure is followed for grid points 22 and 23. Note the use of symmetrical boundary conditions so that only one-half of the joint need be considered. Note that the eight exceptions to the MODAL card are called out on the System Control Information Data Form, Figure III-G.4.

The following load data is presented in Figure III-G.7, External Loads Section:

- (1) One load condition is input
- (2) Grid points 1 and 3 are loaded with a force in the +X direction equal to 16.67 pounds and grid point 2 is loaded with a force of 66.66 pounds in the +X direction.

Zero valued entries are made in the External Moments section since these do not exist in this problem.

The Element Control Data Form, Figure III-G.8, displays the use of the REPEAT option. This is used to advantage here since each of the four elements are identical. Although 8 input nodes define the element the User will note that 10 nodes are listed. The last two nodes "6" and "1" in locations 9 and 10 define the X direction for the material properties axes. This allows the User to effectively

define stress output direction. The same two points used for the reference element can also be used for the following elements so that output has a common reference.

The output supplied by the MAGIC III System for this illustrative problem is described below and shown on Figures III-G.11 to III-G.27. Figure III-G.11 shows the matrix abstraction instructions which are completely described in Reference 5. Figures III-G.11 to III-G.14 display the output from the Structural Systems Monitor. These figures record the input data pertinent to the problem being solved.

Problem title and material data are given in Figure III-G.12 whereas Figure III-G.13 displays the gridpoint coordinates, temperatures and pressures. Figure III-G.14 presents the boundary conditions and finite element description. In the boundary condition portion of the figure, zeros ('0') represent degrees of freedom that are fixed (i.e., no motion), ones ('1') represent degrees of freedom that are free or have unknown values of displacement, and twos ('2') represent degrees of freedom that are eliminated in the analysis procedure through the condensation technique. The second last column represents the cumulative number of degrees of freedom which actively participate in the equation solving process for displacements. The last column accumulates the number of twos which participate in the calculation of the reduced stiffness matrix. This procedure is not used in this example problem. The second portion of Figure III-G.14 depicts the finite element representation. Each of the four elements is called out in turn with grid points, print options and material number. The use of extra grid points "6" and "1" were explained above. The section properties listed represents the joint thickness.

Figure III-G.15 displays the external load condition and the transformed external assembled load column. This 138x1 vector is the total unreduced load which is read row-wise. The ordering of this vector is consistent with that of the boundary condition table, Figure III-G.14. A load of 16.67 pounds is applied at node point one in the positive X direction. This is position (1,1) in the load vector which corresponds to the first entry in the boundary condition table which is the global U displacement for node point 1. Likewise position (7,1) in the load vector corresponds to the seventh entry in the boundary condition table and the last position (13,1) corresponds to thirteenth entry.

MAGIC III system output of final results are displayed in Figures III-G.16 to III-G.27. The stiffness matrix is shown in Figure III-G.16 where only the non-zero terms are displayed. The stiffness matrix is presented row-wise and it's ordering is consistent with that of the boundary condition table previously discussed. In this problem the ordering is

$$\{\Delta\}^T = \left[ U_1, U_2, V_2, U_3, V_3, U_4, \dots, V_{21}, V_{22}, V_{23} \right].$$

The externally applied load vector (GPRINT of MATRIX LOADS) is presented in Figure III-G.17. The figure shows that forces ( $F_x$ ) are applied in the positive X direction at nodes 1, 2 and 3 as previously discussed.

The displacements of the joint are given in Figure III-G.18. These displacements (U,V,W) are output versus node point number and are referenced to the global axes unless otherwise specified. Figure III-G.19 shows the reactions ( $F_x, F_y, F_z$ ). These are also output versus node point number and are referenced to the global axes system unless otherwise specified.

# Contrails

Stresses arising in the structure are displayed in Figures III-G.20 to III-G.23. Eight stress resultants are evaluated at each corner point of the element and also at the intersection of the diagonals which connect the opposite corner points of the element. The stress resultants are defined as follows:

$$N_x = \int_x \sigma_x \, dz \quad ; \text{ units } \frac{\text{force}}{\text{length}}$$

$$N_y = \int_z \sigma_y \, dz \quad ; \text{ units } \frac{\text{force}}{\text{length}}$$

$$N_{xy} = \int_z \tau_{xy} \, dz \quad ; \text{ units } \frac{\text{force}}{\text{length}}$$

$$M_x = \int_z z \sigma_x \, dz \quad ; \text{ units } \frac{\text{force} \times \text{length}}{\text{length}}$$

$$M_y = \int_z z \sigma_y \, dz \quad ; \text{ units } \frac{\text{force} \times \text{length}}{\text{length}}$$

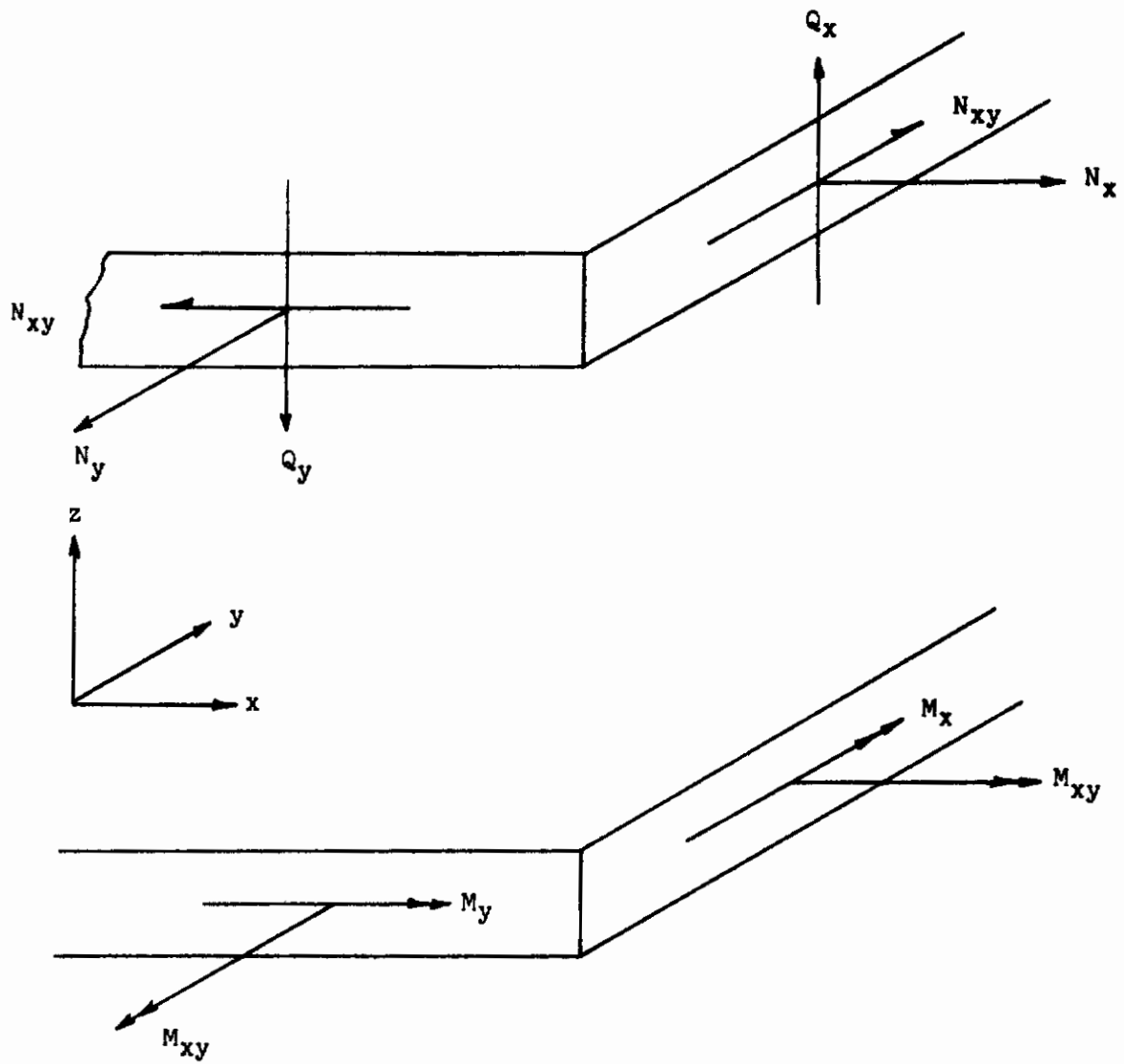
$$M_{xy} = \int_z z \tau_{xy} \, dz \quad ; \text{ units } \frac{\text{force} \times \text{length}}{\text{length}}$$

$$Q_x = \int_z z \left( \frac{\partial \sigma_x}{\partial x} \right) dz + \int_z z \left( \frac{\partial \tau_{xy}}{\partial y} \right) dz ; \text{ units } \frac{\text{force}}{\text{length}}$$

$$Q_y = \int_z z \left( \frac{\partial \sigma_y}{\partial y} \right) dz + \int_z z \left( \frac{\partial \tau_{xy}}{\partial x} \right) dz ; \text{ units } \frac{\text{force}}{\text{length}}$$

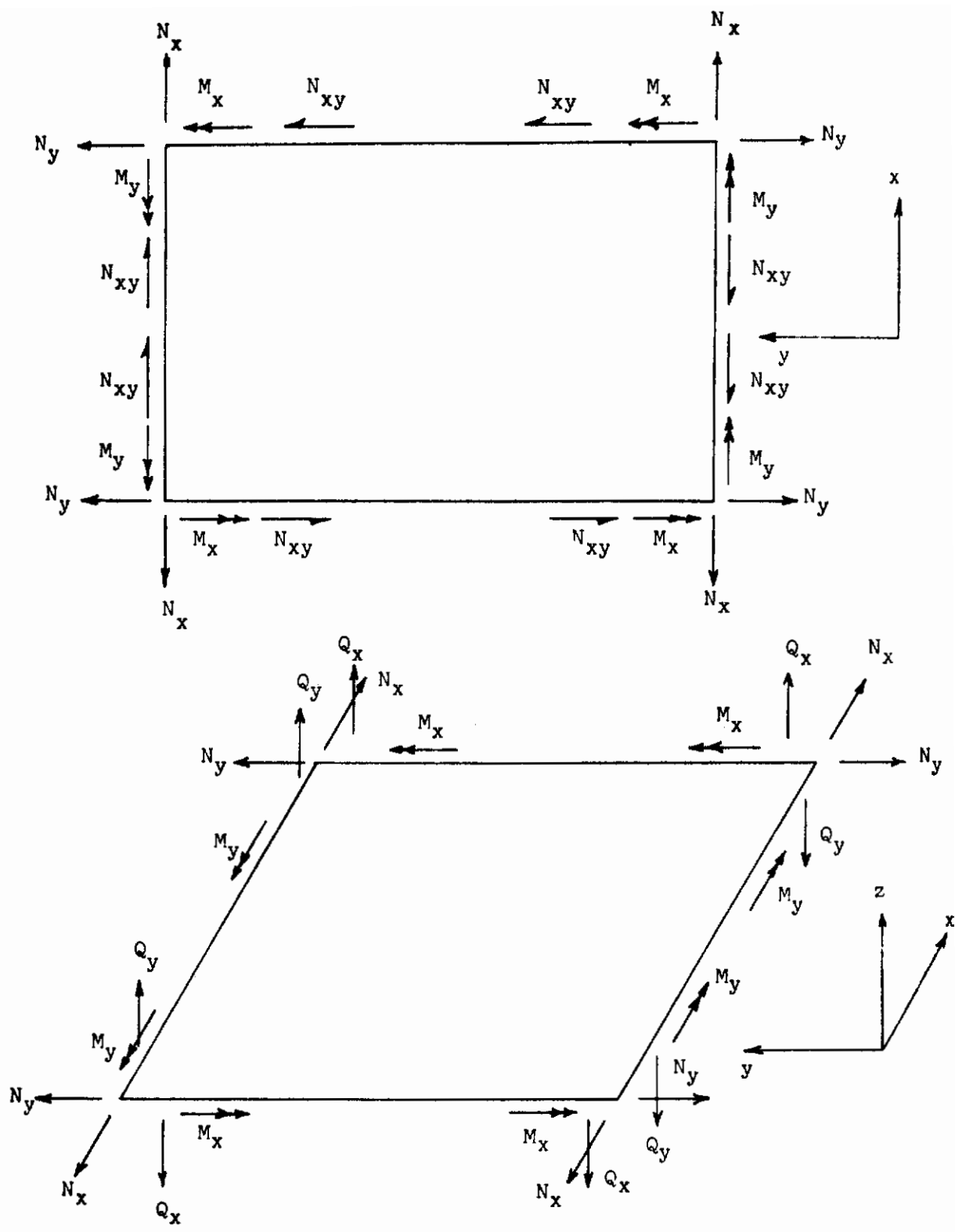
The following sketches show the proper manner in which to interpret the stress resultants.





**Stress Resultants**

# Contrails



# Contrails

Returning to Figure III-G.20 it is noted that there are five stress points at which the stress resultants are evaluated. These correspond to element grid points 1, 3, 8, and 6. The fifth stress point corresponds to the stresses evaluated at the element centroid. The stresses are in general referenced to the element coordinate system. For the quadrilateral or triangular thin shell elements, however, the User has the option of specifying material or stress axes in order to effectively define stress output direction. This is accomplished by utilizing locations 9 and 10 or 11 and 12 of the node point portion of the Element Control Section. In this particular problem the numbers '6' and '1' were entered in locations 9 and 10 of the node point portion of the Element Control Section. These two points define the X direction of the material properties axes. (Positive X from node point 6 to node point 1.) This axis of reference then becomes the reference axis for the stress output.

The element forces for the Modified Quadrilateral Thin Shell Element are displayed in Figures III-G.24 to III-G.27. The forces ( $F_X$ ,  $F_Y$ ,  $F_Z$ ,  $M_X$ ,  $M_Y$ ,  $M_Z$ ) are defined with respect to the Global coordinate system. The forces are defined at eight points on the element. The first four points are corner points, element grid points 1, 3, 8, and 6, and the last four points are mid-points, element grid points 2, 5, 7, 4 for element 1, for example.

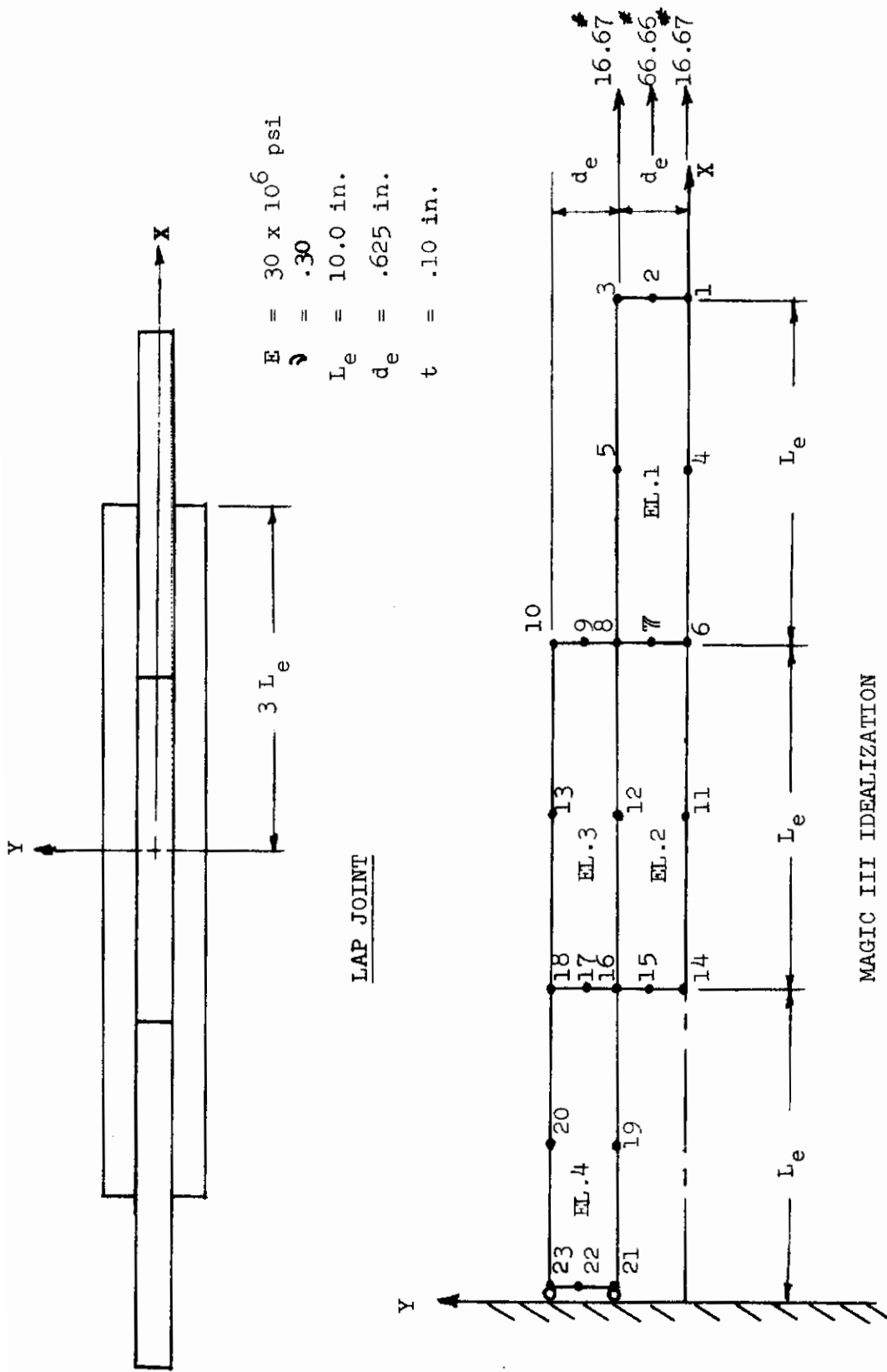


FIGURE III-G.1 MODIFIED QUADRILATERAL THIN SHELL ELEMENT - LAP JOINT PROBLEM





## MAGIC STRUCTURAL ANALYSIS SYSTEM INPUT DATA FORMAT

### SYSTEM CONTROL INFORMATION

ENTER APPROPRIATE NUMBER, RIGHT ADJUSTED, IN BOX OPPOSITE APPLICABLE REQUESTS

	<table border="1" style="margin: auto;"> <tr><td>S</td><td>Y</td><td>S</td><td>T</td><td>E</td><td>M</td></tr> <tr><td>1</td><td>2</td><td>3</td><td>4</td><td>5</td><td>6</td></tr> </table>	S	Y	S	T	E	M	1	2	3	4	5	6	( / )			
S	Y	S	T	E	M												
1	2	3	4	5	6												
1. Number of System Grid Points	<table border="1" style="margin: auto;"> <tr><td></td><td></td><td></td><td></td><td>2</td><td>3</td></tr> <tr><td>1</td><td>2</td><td>3</td><td>4</td><td>5</td><td>6</td></tr> </table>					2	3	1	2	3	4	5	6				
				2	3												
1	2	3	4	5	6												
2. Number of Input Grid Points	<table border="1" style="margin: auto;"> <tr><td></td><td></td><td></td><td></td><td>1</td><td>0</td></tr> <tr><td>7</td><td>8</td><td>9</td><td>10</td><td>11</td><td>12</td></tr> </table>					1	0	7	8	9	10	11	12				
				1	0												
7	8	9	10	11	12												
3. Number of Degrees of Freedom/Grid Point	<table border="1" style="margin: auto;"> <tr><td></td><td>6</td></tr> <tr><td>13</td><td>14</td></tr> </table>		6	13	14												
	6																
13	14																
4. Number of Load Conditions	<table border="1" style="margin: auto;"> <tr><td></td><td>1</td></tr> <tr><td>15</td><td>16</td></tr> </table>		1	15	16												
	1																
15	16																
5. Number of Initially Displaced Grid Points	<table border="1" style="margin: auto;"> <tr><td></td><td></td><td></td><td></td><td></td><td>0</td></tr> <tr><td>17</td><td>18</td><td>19</td><td>20</td><td>21</td><td>22</td></tr> </table>						0	17	18	19	20	21	22				
					0												
17	18	19	20	21	22												
6. Number of Prescribed Displaced Grid Points	<table border="1" style="margin: auto;"> <tr><td></td><td></td><td></td><td></td><td></td><td>0</td></tr> <tr><td>23</td><td>24</td><td>25</td><td>26</td><td>27</td><td>28</td></tr> </table>						0	23	24	25	26	27	28				
					0												
23	24	25	26	27	28												
7. Number of Grid Point Axes Transformation Systems	<table border="1" style="margin: auto;"> <tr><td></td><td>0</td></tr> <tr><td>29</td><td>30</td></tr> </table>		0	29	30												
	0																
29	30																
8. Number of Elements	<table border="1" style="margin: auto;"> <tr><td></td><td></td><td></td><td></td><td>4</td></tr> <tr><td>31</td><td>32</td><td>33</td><td>34</td><td>35</td><td>36</td></tr> </table>					4	31	32	33	34	35	36					
				4													
31	32	33	34	35	36												
9. Number of Requests and/or Revisions of Material Tape.	<table border="1" style="margin: auto;"> <tr><td></td><td>1</td></tr> <tr><td>37</td><td>38</td></tr> </table>		1	37	38												
	1																
37	38																
10. Number of Input Boundary Condition Points	<table border="1" style="margin: auto;"> <tr><td></td><td></td><td></td><td></td><td>8</td></tr> <tr><td>39</td><td>40</td><td>41</td><td>42</td><td>43</td><td>44</td></tr> </table>					8	39	40	41	42	43	44					
				8													
39	40	41	42	43	44												
11. $T_0$ For Structure (With Decimal Point)	<table border="1" style="margin: auto;"> <tr><td></td><td></td><td></td><td></td><td>0</td><td>.</td><td>0</td></tr> <tr><td>45</td><td>46</td><td>47</td><td>48</td><td>49</td><td>50</td><td>51</td><td>52</td></tr> </table>					0	.	0	45	46	47	48	49	50	51	52	( / )
				0	.	0											
45	46	47	48	49	50	51	52										

FIGURE III-G.4 SYSTEM CONTROL INFORMATION - LAP JOINT PROBLEM













MAGIC STRUCTURAL ANALYSIS SYSTEM

INPUT DATA FORMAT

CHECK OR END CARD

C	H	E	C	K
1	2	3	4	5

 (/)

E	N	D
1	2	3

 (/)

FIGURE III-G.10 END CARD - LAP JOINT PROBLEM





TEST MAGIC

```

19 C          REACTS = KELA.MULT.XO
20 C          REACTP= REACTS.SUBT.TLCCAD
21 C          IF (DIFF.NULL.) GO TC 10

22 C          PRINT ELEMENT APPLIED LCACS, EXTERNAL LCACS, DISPLACEMENTS,
23 C          REACTIONS AND INVERSE CHECK IN ENGINEERING FORMAT
24 C          ELEMENTS HAVE 1 OR 2 DEGREES CF FREEDCM
25 C          GPR INT(4,,,FX,FY,FZ,MX,MY,MZ,SC,TF )FTELA
26 C          GPR INT(4,,,FX,FY,FZ,MX,MY,MZ,SC, )LCACS
27 C          GPR INT(2,,,U,V,W,THE TAY,THE TAZ,SC, )X
28 C          GPR INT(1,,,FX,FY,FZ,MX,MY,MZ,SC,TF )REACTP
29 C          IF (I3.NULL.) GO TO 600
30 C          ELEMENTS HAVE 3 DEGREES CF FREEDCM
31 C          GPR INT(4,,,FR,0,FZ,0,MBETA,0,F1,0,F3,SC,TR )FTELA
32 C          GPR INT(4,,,FR,0,FZ,0,MBETA,0,F1,0,F3,SC, )LCADS
33 C          GPR INT(2,,,U,0,0,THE TAY,C,0,0,0,0,0,0,SC, )X
34 C          GPR INT(1,,,FR,0,FZ,0,MBETA,C,F1,0,F3,SC,TR )REACTP
35 C          GENERATE STRESSES AND FCFCEs
36 C          STRESP=EM,XO .STRESS.(4, )
37 C          FORCEP=EM,XO .FORCE.(4, )

```

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FIGURE III-G.11 CONCLUDED



23 REF. POINTS

NO. DIRECTIONS = 3      NO. DEGREES OF FREEDOM = 2

GRIDPOINT DATA  
(IN RECTANGULAR COORDINATES)

POINT	X	Y	Z	TEMPERATURES	PRESSURES
1	0.3000000E 02	0.0	0.0	0.0	0.0
3	0.3000000E 02	0.6250000E 00	0.0	0.0	0.0
6	0.2000000E 02	0.0	0.0	0.0	0.0
8	0.2000000E 02	0.6250000E 00	0.0	0.0	0.0
10	0.2000000E 02	0.1250000E 01	0.0	0.0	0.0
14	0.1000000E 02	0.0	0.0	0.0	0.0
16	0.1000000E 02	0.6250000E 00	0.0	0.0	0.0
18	0.1000000E 02	0.1250000E 01	0.0	0.0	0.0
21	0.0	0.6250000E 00	0.0	0.0	0.0
23	0.0	0.1250000E 01	0.0	0.0	0.0

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FIGURE III-G.13 GRIDPOINT DATA OUTPUT - LAP JOINT PROBLEM

BOUNDARY CONDITIONAL INFORMATION

NODES	DEGREES OF FREEDOM	NO. OF UNES	NO. OF TIME
1	C	1	C
2	C	3	C
3	C	5	C
4	C	6	C
5	C	8	C
6	C	9	C
7	C	11	C
8	C	13	C
9	C	15	C
10	C	17	C
11	C	18	C
12	C	20	C
13	C	22	C
14	C	23	C
15	C	25	C
16	C	27	C
17	C	29	C
18	C	31	C
19	C	33	C
20	C	35	C
21	C	36	C
22	C	37	C
23	C	38	C

TOTAL NO. ELEMENTS = 4

ELEM TYPE	MAT. NO.	CODE	TEMP.	PRNT NO.	GRID POINTS	EXTRA GRID PTS	SECTION PROPERTIES
1	38	99398	0 0.0	0 8	1 3 8 6 2 5 7 4	6 1	C. LOGOE CI 0.0 0.0
2	38	99398	0 0.0	0 8	6 8 16 14 7 12 15 11	6 1	ELEMENT MATRICES REPEATED
					PRE-STRAIN = 0.0		C.C 0.0
					PRE-STRESS = 0.0		C.C 0.0
3	38	99398	0 0.0	0 8	8 10 18 16 9 13 17 12	6 1	ELEMENT MATRICES REPEATED
					PRE-STRAIN = 0.0		C.C 0.0
					PRE-STRESS = 0.0		C.C 0.0
4	38	99398	0 0.0	0 8	16 18 23 21 17 20 22 19	6 1	ELEMENT MATRICES REPEATED
					PRE-STRAIN = 0.0		C.C 0.0
					PRE-STRESS = 0.0		C.C 0.0

FIGURE III-G.14 BOUNDARY CONDITIONS AND FINITE ELEMENT DESCRIPTION - LAP JOINT PROBLEM









MATRIX STIFF / 1/

CUTOFF = 0.0

SIZE 38 8Y 3P

DISP	FORCE	FORCE	FORCE	FORCE	FORCE	FORCE	FORCE	FORCE	FORCE	FORCE	FORCE	FORCE	
19	29	0.552376E 07	30	-0.132234E 08	31	0.238095E 07							
20	9	0.238094E 07	10	0.952376E 07	11	-0.308750E 02	12	-0.400000E 01	13	0.690473E 08			
14	14	-0.952377E 07	15	-0.289938E 03	16	-0.238094E 07	17	-0.354850E 08	18	0.700000E 01			
19	19	-0.320000E 02	20	0.565198E 09	21	-0.280676E 09	22	-0.238094E 07	23	-0.552375E 07			
25	25	0.417188E 03	26	-0.100000E 01	27	0.690471E 08	28	0.952376E 07	29	-0.299125E 03			
30	30	0.238095E 07	31	-0.354849E 08									
21	12	-0.132234E 08	13	0.238094E 07	14	-0.606250E 01	15	0.952376E 07	16	0.104762E 08			
17	17	-0.421242E 07	18	-0.966297E 08	19	0.102124E 09	20	-0.320000E 02	21	-0.132235E 08			
27	27	-0.238095E 07	28	0.114750E 03	29	-0.952375E 07	30	0.104761E 08	31	0.421243E 07			
22	12	0.238094E 07	13	-0.354851E 08	14	0.952376E 07	15	-0.308750E 02	16	-0.531133E 07			
17	17	0.345234E 08	18	0.700000E 01	19	-0.280676E 09	20	-0.320000E 02	21	0.282599E 09			
26	26	-0.238096E 07	27	-0.354853E 08	28	-0.952375E 07	29	0.417188E 03	30	0.531133E 07			
31	31	0.345233E 08											
23	9	0.355126E 08	10	-0.821883E 08	11	-0.238096E 07	12	0.477050E 08	13	0.416665E 07			
18	18	0.104762E 08	19	-0.132235E 08	20	-0.238096E 07	21	0.107857E 09	22	-0.163965E 09			
25	25	-0.421245E 07	26	0.578248E 08	27	-0.137348E 06							
24	9	-0.821883E 08	10	0.163003E 09	11	0.280000E 02	12	-0.821883E 08	13	-0.238095E 07			
18	18	-0.605625E 02	19	0.114750E 03	20	-0.952375E 07	21	-0.163965E 09	22	0.329302E 09			
25	25	0.144000E 03	26	-0.163964E 09	27	0.531132E 07							
25	9	0.238095E 07	10	0.160000E 02	11	0.469476E 09	12	-0.238096E 07	13	-0.234479E 09			
18	18	0.552376E 07	19	-0.952375E 07	20	0.417188E 03	21	-0.421242E 07	22	0.144000E 03			
25	25	0.438108E 09	26	0.421242E 07	27	-0.468813E 09							
26	9	0.477050E 08	10	-0.821883E 08	11	0.238092E 07	12	0.710252E 08	13	0.300000E 01			
14	14	-0.821883E 08	15	-0.238096E 07	16	0.477050E 08	17	0.416665E 07	18	-0.132234E 08			
19	19	0.209523E 08	20	-0.100000E 01	21	-0.132235E 08	22	-0.238096E 07	23	0.578248E 08			
24	24	-0.163964E 09	25	0.421242E 07	26	0.323570E 09	27	-0.101190E 08	28	-0.327925E 09			
29	29	-0.320000E 02	30	0.115650E 09	31	0.150000E 02	32	0.104762E 08	33	0.531132E 07			
34	34	-0.132234E 08	35	0.238094E 07	36	-0.137361E 06	37	0.238095E 07	38	-0.416665E 07			
27	9	-0.416665E 07	10	0.238094E 07	11	-0.234479E 09	12	0.195715E 09	13	-0.238096E 07			
15	15	-0.234479E 09	16	0.416665E 07	17	0.134982E 09	18	0.238095E 07	19	0.200000E 01			
20	20	0.690471E 08	21	-0.238096E 07	22	-0.354853E 08	23	-0.137348E 06	24	0.531132E 07			
25	25	-0.468813E 09	26	-0.101190E 08	27	0.915527E 09	28	-0.220000E 02	29	-0.537626E 09			
30	30	0.130000E 02	31	0.328475E 09	32	0.421242E 07	33	0.345237E 08	34	0.238095E 07			
35	35	-0.354851E 08	36	0.998573E 08	37	-0.234479E 09	38	0.134982E 09					
29	12	-0.821883E 08	13	0.238093E 07	14	0.163003E 09	15	0.280000E 02	16	-0.821883E 08			
17	17	-0.238095E 07	18	-0.605625E 02	19	0.114750E 03	20	0.952376E 07	21	0.114750E 03			

FIGURE III-G.16 CONTINUED

MATRIX STIFF / 1/

CUTOFF \* 0.C

DISP	FORCE	FORCE			FC/CE			FORCE			FORCE		
		28	31	36	27	32	37	28	33	38	29	34	39
		26	29	30	12	17	26	13	19	27	14	20	28
		31	36	38	17	26	31	19	27	32	20	28	33
		36	38	39	26	31	36	27	32	37	28	33	38
		38	39	40	31	36	41	32	37	42	33	38	43
		41	46	47	36	41	46	37	42	47	38	43	48
		46	51	52	41	46	51	42	47	52	39	44	49
		51	56	57	46	51	56	47	52	57	44	49	54
		56	61	62	51	56	61	52	57	62	49	54	59
		61	66	67	56	61	66	57	62	67	54	59	64
		66	71	72	61	66	71	62	67	72	59	64	69
		71	76	77	66	71	76	67	72	77	64	69	74
		76	81	82	71	76	81	72	77	82	69	74	79
		81	86	87	76	81	86	77	82	87	74	79	84
		86	91	92	81	86	91	82	87	92	79	84	89
		91	96	97	86	91	96	87	92	97	84	89	94
		96	101	102	91	96	101	88	93	102	89	94	99
		101	106	107	96	101	106	93	98	107	94	99	104
		106	111	112	101	106	111	98	103	112	99	104	109
		111	116	117	106	111	116	103	108	117	104	109	114
		116	121	122	111	116	121	108	113	122	109	114	119
		121	126	127	116	121	126	113	118	127	114	119	124
		126	131	132	121	126	131	118	123	132	119	124	129
		131	136	137	126	131	136	123	128	137	124	129	134
		136	141	142	131	136	141	128	133	142	129	134	139
		141	146	147	136	141	146	133	138	147	134	139	144
		146	151	152	141	146	151	138	143	152	139	144	149
		151	156	157	146	151	156	143	148	157	144	149	154
		156	161	162	151	156	161	148	153	162	149	154	159
		161	166	167	156	161	166	153	158	167	154	159	164
		166	171	172	161	166	171	158	163	172	159	164	169
		171	176	177	166	171	176	163	168	177	164	169	174
		176	181	182	171	176	181	168	173	182	169	174	179
		181	186	187	176	181	186	173	178	187	174	179	184
		186	191	192	181	186	191	178	183	192	179	184	189
		191	196	197	186	191	196	183	188	197	184	189	194
		196	201	202	191	196	201	188	193	202	189	194	199
		201	206	207	196	201	206	193	198	207	194	199	204
		206	211	212	201	206	211	198	203	212	199	204	209
		211	216	217	206	211	216	203	208	217	204	209	214
		216	221	222	211	216	221	208	213	222	209	214	219
		221	226	227	216	221	226	213	218	227	214	219	224
		226	231	232	221	226	231	218	223	232	219	224	229
		231	236	237	226	231	236	223	228	237	224	229	234
		236	241	242	231	236	241	228	233	242	229	234	239
		241	246	247	236	241	246	233	238	247	234	239	244
		246	251	252	241	246	251	238	243	252	239	244	249
		251	256	257	246	251	256	243	248	257	244	249	254
		256	261	262	251	256	261	248	253	262	249	254	259
		261	266	267	256	261	266	253	258	267	254	259	264
		266	271	272	261	266	271	258	263	272	259	264	269
		271	276	277	266	271	276	263	268	277	264	269	274
		276	281	282	271	276	281	268	273	282	269	274	279
		281	286	287	276	281	286	273	278	287	274	279	284
		286	291	292	281	286	291	278	283	292	279	284	289
		291	296	297	286	291	296	283	288	297	284	289	294
		296	301	302	291	296	301	288	293	302	289	294	299
		301	306	307	296	301	306	293	298	307	294	299	304
		306	311	312	301	306	311	298	303	312	299	304	309
		311	316	317	306	311	316	303	308	317	304	309	314
		316	321	322	311	316	321	308	313	322	309	314	319
		321	326	327	316	321	326	313	318	327	314	319	324
		326	331	332	321	326	331	318	323	332	319	324	329
		331	336	337	326	331	336	323	328	337	324	329	334
		336	341	342	331	336	341	328	333	342	329	334	339
		341	346	347	336	341	346	333	338	347	334	339	344
		346	351	352	341	346	351	338	343	352	339	344	349
		351	356	357	346	351	356	343	348	357	344	349	354
		356	361	362	351	356	361	348	353	362	349	354	359
		361	366	367	356	361	366	353	358	367	354	359	364
		366	371	372	361	366	371	358	363	372	359	364	369
		371	376	377	366	371	376	363	368	377	364	369	374
		376	381	382	371	376	381	368	373	382	369	374	379
		381	386	387	376	381	386	373	378	387	374	379	384
		386	391	392	381	386	391	378	383	392	379	384	389
		391	396	397	386	391	396	383	388	397	384	389	394
		396	401	402	391	396	401	388	393	402	389	394	399
		401	406	407	396	401	406	393	398	407	394	399	404
		406	411	412	401	406	411	398	403	412	399	404	409
		411	416	417	406	411	416	403	408	417	404	409	414
		416	421	422	411	416	421	408	413	422	409	414	419
		421	426	427	416	421	426	413	418	427	414	419	424
		426	431	432	421	426	431	418	423	432	419	424	429
		431	436	437	426	431	436	423	428	437	424	429	434
		436	441	442	431	436	441	428	433	442	429	434	439
		441	446	447	436	441	446	433	438	447	434	439	444
		446	451	452	441	446	451	438	443	452	439	444	449
		451	456	457	446	451	456	443	448	457	444	449	454
		456	461	462	451	456	461	448	453	462	449	454	459
		461	466	467	456	461	466	453	458	467	454	459	464
		466	471	472	461	466	471	458	463	472	459	464	469
		471	476	477	466	471	476	463	468	477	464	469	474
		476	481	482	471	476	481	468	473	482	469	474	479
		481	486	487	476	481	486	473	478	487	474	479	484
		486	491	492	481	486	491	478	483	492	479	484	489
		491	496	497	486	491	496	483	488	497	484	489	494
		496	501	502	491	496	501	488	493	502	489	494	499
		501	506	507	496	501	506	493	498	507	494	499	504
		506	511	512	501	506	511						

MATRIX STIFF / I/

CUTCFF = 0.0

DISP	FORCE	38	31	0.598574E 08	32	0.238095E 07	33	-0.354849E 08	34	0.421243E 07	35	0.345233E 08
		36		0.164238E 09	37	-0.468813E 09	38	0.305175E 09				

FIGURE III-G.16 CONCLUDED

GPRINT OF MATRIX LOADS (SET 1)

ROW	FX	FY	FZ	MX	MY	MZ
1	0.16669950E 02	0.0	0.0	0.0	0.0	0.0
2	0.66659980E 02	0.0	0.0	0.0	0.0	0.0
3	0.16669950E 02	0.0	0.0	0.0	0.0	0.0
4	0.0	0.0	0.0	0.0	0.0	0.0
5	0.0	0.0	0.0	0.0	0.0	0.0
6	0.0	0.0	0.0	0.0	0.0	0.0
7	0.0	0.0	0.0	0.0	0.0	0.0
8	0.0	0.0	0.0	0.0	0.0	0.0
9	0.0	0.0	0.0	0.0	0.0	0.0
10	0.0	0.0	0.0	0.0	0.0	0.0
11	0.0	0.0	0.0	0.0	0.0	0.0
12	0.0	0.0	0.0	0.0	0.0	0.0
13	0.0	0.0	0.0	0.0	0.0	0.0
14	0.0	0.0	0.0	0.0	0.0	0.0
15	0.0	0.0	0.0	0.0	0.0	0.0
16	0.0	0.0	0.0	0.0	0.0	0.0
17	0.0	0.0	0.0	0.0	0.0	0.0
18	0.0	0.0	0.0	0.0	0.0	0.0
19	0.0	0.0	0.0	0.0	0.0	0.0
20	0.0	0.0	0.0	0.0	0.0	0.0
21	0.0	0.0	0.0	0.0	0.0	0.0
22	0.0	0.0	0.0	0.0	0.0	0.0
23	0.0	0.0	0.0	0.0	0.0	0.0

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FIGURE III-G.17 GPRINT OF MATRIX LOADS - LAP JOINT PROBLEM

DISPLACEMENT MATRIX FOR LOAD CONDITION 1

138 X 1

ROW	U	V	W	THETA X	THETA Y	THETA Z
1	0.13501084E-03	0.0	0.0	0.0	0.0	0.0
2	0.13487609E-03	-0.47325757E-06	0.0	0.0	0.0	0.0
3	0.13477041E-03	-0.54640353E-06	0.0	0.0	0.0	0.0
4	0.10810344E-03	0.0	0.0	0.0	0.0	0.0
5	0.10821772E-03	-0.10245358E-05	0.0	0.0	0.0	0.0
6	0.81977443E-04	0.0	0.0	0.0	0.0	0.0
7	0.81662176E-04	-0.38511168E-06	0.0	0.0	0.0	0.0
8	0.81290957E-04	-0.77687406E-06	0.0	0.0	0.0	0.0
9	0.80524303E-04	-0.10398508E-05	0.0	0.0	0.0	0.0
10	0.79940874E-04	-0.12878181E-05	0.0	0.0	0.0	0.0
11	0.67464265E-04	0.0	0.0	0.0	0.0	0.0
12	0.67667250E-04	-0.44560721E-06	0.0	0.0	0.0	0.0
13	0.67976827E-04	-0.91653559E-06	0.0	0.0	0.0	0.0
14	0.55488752E-04	0.0	0.0	0.0	0.0	0.0
15	0.54964505E-04	-0.30050563E-06	0.0	0.0	0.0	0.0
16	0.54258853E-04	-0.61322083E-06	0.0	0.0	0.0	0.0
17	0.53487762E-04	-0.10038639E-05	0.0	0.0	0.0	0.0
18	0.52761654E-04	-0.14030793E-05	0.0	0.0	0.0	0.0
19	0.27219023E-04	-0.94516899E-05	0.0	0.0	0.0	0.0
20	0.26472917E-04	-0.10488505E-04	0.0	0.0	0.0	0.0
21	0.0	-0.12465570E-04	0.0	0.0	0.0	0.0
22	0.0	-0.12547585E-04	0.0	0.0	0.0	0.0
23	0.0	-0.13410809E-04	0.0	0.0	0.0	0.0

FIGURE III-G-18 DISPLACEMENT MATRIX - LAP JOINT PROBLEM

REACTIONS AND INVERSE CHECK FOR LOAD CONDITION 1

ROW	FX	FY	FZ	MX	MY	MZ
1	0.21563721E 00	0.88856506E 00	0.00	0.00	0.00	0.00
2	-0.40031433E 00	0.23086548E-01	0.00	0.00	0.00	0.00
3	0.19642639E 00	-0.62408447E-02	0.00	0.00	0.00	0.00
4	0.26968765E 00	-0.37375546E 01	0.00	0.00	0.00	0.00
5	-0.27252253E 00	-0.10223389E-01	0.00	0.00	0.00	0.00
6	-0.34629154E 00	-0.72068787E 00	0.00	0.00	0.00	0.00
7	0.60245800E 00	0.49468594E-01	0.00	0.00	0.00	0.00
8	-0.80131531E-01	-0.17074585E-01	0.00	0.00	0.00	0.00
9	-0.29767759E 00	-0.78125000E-02	0.00	0.00	0.00	0.00
10	0.14369744E 00	-0.23956299E-02	0.00	0.00	0.00	0.00
11	-0.10095556E-01	-0.35350342E 01	0.00	0.00	0.00	0.00
12	0.54603577E-01	0.25527954E-01	0.00	0.00	0.00	0.00
13	-0.15834808E-01	-0.21667480E-01	0.00	0.00	0.00	0.00
14	-0.20264059E 00	0.70785980E 01	0.00	0.00	0.00	0.00
15	0.43830109E 00	-0.23525781E-01	0.00	0.00	0.00	0.00
16	-0.28027344E 00	0.51513672E-01	0.00	0.00	0.00	0.00
17	0.71563721E-02	-0.31250000E-01	0.00	0.00	0.00	0.00
18	0.43360710E-01	0.12939453E-01	0.00	0.00	0.00	0.00
19	-0.10238647E-01	-0.17089844E-02	0.00	0.00	0.00	0.00
20	0.11032104E-01	-0.41503506E-02	0.00	0.00	0.00	0.00
21	-0.15227541E 02	0.36621054E-02	0.00	0.00	0.00	0.00
22	-0.70409378E 02	-0.19531250E-01	0.00	0.00	0.00	0.00
23	-0.14896011E 02	0.75683594E-02	0.00	0.00	0.00	0.00

FIGURE III-G-19 REACTION MATRIX - LAP JOINT PROBLEM

STRESSES FOR THE HIGH ASPECT RATIO QUADRILATERAL ELEMENT  
(STRESS POINT FIVE EQUALS ELEMENT STRESSES EVALUATED AT THE CENTROID)

LOAD CONDITION NUMBER	ELEMENT NUMBER	ELEMENT TYPE	ELEMENT GRID POINTS				ELEMENT GRID POINTS						
1	1	38	1	3	2	5	7	4					
APPARENT ELEMENT STRESSES													
STRESS POINT	MEMBRANE	STRESS	RESULTS	SHEAR (AXY)		NORPMAL (MX)		FLEXURAL MOMENTS		TORQUE (MXY)		SHEAR	
	NORMAL (NX)	NORMAL (NY)						NORMAL (MY)	TORQUE (MXY)	NORMAL (QX)			NORMAL (QY)
1	0.165003E 03	0.405542E 01	0.173172E 03	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0
2	0.158868E 03	0.225170E 01	0.165301E 03	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0
3	0.166522E 03	0.127961E 02	0.163610E 03	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0
4	0.157348E 03	0.978467E 01	0.158817E 03	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0
5	0.159217E 03	-0.141223E 01	0.174836E 03	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0
ELEMENT APPLIED STRESSES													
STRESS POINT	MEMBRANE	STRESS	RESULTS	SHEAR (AXY)		NORPMAL (MX)		FLEXURAL MOMENTS		TORQUE (MXY)		SHEAR	
	NORMAL (NX)	NORMAL (NY)						NORMAL (MY)	TORQUE (MXY)	NORMAL (QX)			NORMAL (QY)
1	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0
2	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0
3	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0
4	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0
5	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0
NET ELEMENT STRESSES													
STRESS POINT	MEMBRANE	STRESS	RESULTS	SHEAR (AXY)		NORPMAL (MX)		FLEXURAL MOMENTS		TORQUE (MXY)		SHEAR	
	NORMAL (NX)	NORMAL (NY)						NORMAL (MY)	TORQUE (MXY)	NORMAL (QX)			NORMAL (QY)
1	0.165003E 03	0.405542E 01	0.173172E 03	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0
2	0.158868E 03	0.225170E 01	0.165301E 03	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0
3	0.166522E 03	0.127961E 02	0.163610E 03	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0
4	0.157348E 03	0.978467E 01	0.158817E 03	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0
5	0.159217E 03	-0.141223E 01	0.174836E 03	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0

FIGURE III-G-20 STRESS OUTPUT, ELEMENT NO. 1 - LAP JOINT PROBLEM



STRESSES FOR THE HIGH ASPECT RATIO QUADRILATERAL ELEMENT  
(STRESS POINT FIVE EQUALS ELEMENT STRESSES EVALUATED AT THE CENTROID)

LOAD CONDITION NUMBER		ELEMENT ALPREF		ELEMENT TYPE		ELEMENT GRID POINTS										
1		2		38		6	8	16	14	7	12	15	11			
APPARENT ELEMENT STRESSES		STRESS RESULTANTS		NORMAL(MX)		FLEXURAL MOMENTS		TORQUE(MXY)		NORMAL(QX)		SHEAR				
STRESS POINT	MEMBRANE	NORMAL(NY)	SHEAR(NXY)	NORMAL(MX)	SHEAR(NXY)	NORMAL(MY)	TORQUE(MXY)	NORMAL(QX)	SHEAR	NORMAL(QY)						
1	0.917195E 02	-0.990356E 01	0.935541E 02	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0
2	0.782852E 02	-0.136744E 02	0.760711E 02	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0
3	0.776077E 02	-0.732472E 01	0.631229E 02	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0
4	0.612771E 02	-0.987964E 01	0.564712E 02	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0
5	0.809624E 02	0.285975E 01	0.895001E 02	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0
ELEMENT APPLIED STRESSES																
STRESS POINT	MEMBRANE	NORMAL(NY)	SHEAR(NXY)	NORMAL(MX)	SHEAR(NXY)	NORMAL(MY)	TORQUE(MXY)	NORMAL(QX)	SHEAR	NORMAL(QY)						
1	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0
2	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0
3	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0
4	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0
5	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0
NET ELEMENT STRESSES																
STRESS POINT	MEMBRANE	NORMAL(NY)	SHEAR(NXY)	NORMAL(MX)	SHEAR(NXY)	NORMAL(MY)	TORQUE(MXY)	NORMAL(QX)	SHEAR	NORMAL(QY)						
1	0.917195E 02	-0.990356E 01	0.935541E 02	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0
2	0.782852E 02	-0.136744E 02	0.760711E 02	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0
3	0.776077E 02	-0.732472E 01	0.631229E 02	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0
4	0.612771E 02	-0.987964E 01	0.564712E 02	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0
5	0.809624E 02	0.285975E 01	0.895001E 02	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0

FIGURE III-G-21 STRESS OUTPUT, ELEMENT NO. 2 - LAP JOINT PROBLEM

STRESSES FOR THE HIGH ASPECT RATIO QUADRILATERAL ELEMENT  
(STRESS POINT FIVE EQUALS ELEMENT STRESSES EVALUATED AT THE CENTROID)

LOAD CONDITION NUMBER		ELEMENT PLPREF		ELEMENT TYPE		ELEMENT GRID POINTS							
1		3		38		8	10	18	16	5	13	17	12
APPARENT ELEMENT STRESSES													
STRESS POINT	MEMBRANE	STRESS		RESULTS		FLEXURAL MOMENTS		TORQUE		NORMAL (QX)		SHEAR	
	NORMAL (NX)	NORMAL (NY)	SHEAR (NXY)	NORMAL (MX)	NORMAL (MY)	TORQUE (MXY)	TORQUE (MXY)	NORMAL (QX)	NORMAL (QY)	NORMAL (QX)	NORMAL (QY)	SHEAR	NORMAL (QY)
1	0.819719E 02	-0.138233E 01	0.621751E 02	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0
2	0.605571E 02	-0.490854E 01	0.507861E 02	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0
3	0.982681E 02	-0.525603E 01	0.895258E 02	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0
4	0.754702E 02	-0.144455E 02	0.631821E 02	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0
5	0.816750E 02	0.185908E 01	0.928693E 02	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0
ELEMENT APPLIED STRESSES													
STRESS POINT	MEMBRANE	STRESS		RESULTS		FLEXURAL MOMENTS		TORQUE		NORMAL (QX)		SHEAR	
	NORMAL (NX)	NORMAL (NY)	SHEAR (NXY)	NORMAL (MX)	NORMAL (MY)	TORQUE (MXY)	TORQUE (MXY)	NORMAL (QX)	NORMAL (QY)	NORMAL (QX)	NORMAL (QY)	SHEAR	NORMAL (QY)
1	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0
2	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0
3	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0
4	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0
5	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0
NET ELEMENT STRESSES													
STRESS POINT	MEMBRANE	STRESS		RESULTS		FLEXURAL MOMENTS		TORQUE		NORMAL (QX)		SHEAR	
	NORMAL (NX)	NORMAL (NY)	SHEAR (NXY)	NORMAL (MX)	NORMAL (MY)	TORQUE (MXY)	TORQUE (MXY)	NORMAL (QX)	NORMAL (QY)	NORMAL (QX)	NORMAL (QY)	SHEAR	NORMAL (QY)
1	0.819719E 02	-0.138233E 01	0.621751E 02	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0
2	0.605571E 02	-0.490854E 01	0.507861E 02	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0
3	0.982681E 02	-0.525603E 01	0.895258E 02	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0
4	0.754702E 02	-0.144455E 02	0.631821E 02	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0
5	0.816750E 02	0.185908E 01	0.928693E 02	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0

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FIGURE III-G.22 STRESS OUTPUT, ELEMENT NO. 3 - LAP JOINT PROBLEM

STRESSES FOR THE HIGH ASPECT RATIO QUADRILATERAL ELEMENT  
(STRESS POINT FIVE EQUALS ELEMENT STRESSES EVALUATED AT THE CENTROID)

LOAD CONDITION NUMBER	ELEMENT NUMBER	ELEMENT TYPE	ELEMENT GRID POINTS					ELEMENT GRID POINTS							
1	4	38	16	18	23	21	17	20	22	19					
<b>APPARENT ELEMENT STRESSES</b>															
STRESS POINT	MEMBRANE NORMAL(NX)	STRESS NORMAL(NY)	STRESS RESULTANTS SHEAR(NXY)	MEMBRANE NORMAL(NX)	STRESS NORMAL(NY)	STRESS RESULTANTS SHEAR(NXY)	FLEXURAL MOMENTS NORMAL(MY)	TORQUE(MXY)	NORMAL(QX)	SHEAR ACRMAL(QY)					
1	0.165465E 03	0.125452E 02	0.173666E 03	0.0	0.0	0.0	0.0	0.0	0.0	0.0					
2	0.159954E 03	0.925061E 01	0.172278E 03	0.0	0.0	0.0	0.0	0.0	0.0	0.0					
3	0.160728E 03	0.445972E 01	0.172626E 03	0.0	0.0	0.0	0.0	0.0	0.0	0.0					
4	0.164694E 03	0.280769E 01	0.177967E 03	0.0	0.0	0.0	0.0	0.0	0.0	0.0					
5	0.159926E 03	-0.178857E 01	0.174074E 03	0.0	0.0	0.0	0.0	0.0	0.0	0.0					
<b>ELEMENT APPLIED STRESSES</b>															
STRESS POINT	MEMBRANE NORMAL(NX)	STRESS NORMAL(NY)	STRESS RESULTANTS SHEAR(NXY)	MEMBRANE NORMAL(NX)	STRESS NORMAL(NY)	STRESS RESULTANTS SHEAR(NXY)	FLEXURAL MOMENTS NORMAL(MY)	TORQUE(MXY)	NORMAL(QX)	SHEAR ACRMAL(QY)					
1	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0					
2	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0					
3	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0					
4	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0					
5	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0					
<b>NET ELEMENT STRESSES</b>															
STRESS POINT	MEMBRANE NORMAL(NX)	STRESS NORMAL(NY)	STRESS RESULTANTS SHEAR(NXY)	MEMBRANE NORMAL(NX)	STRESS NORMAL(NY)	STRESS RESULTANTS SHEAR(NXY)	FLEXURAL MOMENTS NORMAL(MY)	TORQUE(MXY)	NORMAL(QX)	SHEAR ACRMAL(QY)					
1	0.165465E 03	0.125452E 02	0.173666E 03	0.0	0.0	0.0	0.0	0.0	0.0	0.0					
2	0.159954E 03	0.925061E 01	0.172278E 03	0.0	0.0	0.0	0.0	0.0	0.0	0.0					
3	0.160728E 03	0.445972E 01	0.172626E 03	0.0	0.0	0.0	0.0	0.0	0.0	0.0					
4	0.164694E 03	0.280769E 01	0.177967E 03	0.0	0.0	0.0	0.0	0.0	0.0	0.0					
5	0.159926E 03	-0.178857E 01	0.174074E 03	0.0	0.0	0.0	0.0	0.0	0.0	0.0					

FIGURE III-G-23 STRESS OUTPUT, ELEMENT NO. 4 - LAP JOINT PROBLEM

FORCES FOR THE HIGH ASPECT RATIO QUADRILATERAL ELEMENT  
 (THE FIRST FOUR POINTS ARE CORNER POINTS AND THE LAST FOUR POINTS ARE MID-POINTS)

LOAD CONDITION NUMBER	ELEMENT ALPHEF	ELEMENT TYPE	ELEMENT GRID POINTS						
			1	3	6	2	5	7	
1	38								
APPARENT ELEMENT FORCES			FORCES	MOMENTS					
POINT	FX	FY	FZ	MX	MY	MZ			
1	0.16864990E 02	0.88540430E 00	0.0	0.0	0.0	0.0	0.0	0.0	
2	0.16852295E 02	-0.70800781E-02	0.0	0.0	0.0	0.0	0.0	0.0	
3	-0.35993652E 02	0.16681412E 02	0.0	0.0	0.0	0.0	0.0	0.0	
4	-0.57014160E 01	-0.98017578E 01	0.0	0.0	0.0	0.0	0.0	0.0	
5	0.66263809E 02	0.22216757E-01	0.0	0.0	0.0	0.0	0.0	0.0	
6	-0.28125000E 00	-0.99036507E-02	0.0	0.0	0.0	0.0	0.0	0.0	
7	-0.58522217E 02	-0.40375577E 01	0.0	0.0	0.0	0.0	0.0	0.0	
8	0.28125000E 00	-0.37381645E 01	0.0	0.0	0.0	0.0	0.0	0.0	

ELEMENT APPLIED FORCES			FORCES	MOMENTS				
	POINT	FX	FY	FZ	MX	MY	MZ	
1	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0
2	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0
3	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0
4	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0
5	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0
6	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0
7	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0
8	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0

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NET ELEMENT FORCES			FORCES	MOMENTS				
	POINT	FX	FY	FZ	MX	MY	MZ	
1	0.16864990E 02	0.88540430E 00	0.0	0.0	0.0	0.0	0.0	0.0
2	0.16852295E 02	-0.70800781E-02	0.0	0.0	0.0	0.0	0.0	0.0
3	-0.35993652E 02	0.16681412E 02	0.0	0.0	0.0	0.0	0.0	0.0
4	-0.57014160E 01	-0.98017578E 01	0.0	0.0	0.0	0.0	0.0	0.0
5	0.66263809E 02	0.22216757E-01	0.0	0.0	0.0	0.0	0.0	0.0
6	-0.28125000E 00	-0.99036507E-02	0.0	0.0	0.0	0.0	0.0	0.0
7	-0.58522217E 02	-0.40375577E 01	0.0	0.0	0.0	0.0	0.0	0.0
8	0.28125000E 00	-0.37381645E 01	0.0	0.0	0.0	0.0	0.0	0.0

FIGURE III-G.24 FORCE OUTPUT, ELEMENT NO. 1 - LAP JOINT PROBLEM

FORCES FOR THE HIGH ASPECT RATIO QUADRILATERAL ELEMENT  
(THE FIRST FOUR POINTS ARE CORNER POINTS AND THE LAST FOUR POINTS ARE MID-POINTS)

LOAD CONDITION NUMBER	ELEMENT NUMBER	ELEMENT TYPE	ELEMENT GRID POINTS				ELEMENT TYPE	38	ELEMENT GRID POINTS				ELEMENT TYPE	38
			1	2	6	8			16	14	7	12		
APPARENT ELEMENT FORCES														
POINT	FX	FY	FZ	MX	MY	MZ		FX	FY	FZ	MX	MY	MZ	
1	0.53361816E 01	0.90810547E 01	0.0	0.0	0.0	0.0		0.0	0.0	0.0	0.0	0.0	0.0	0.0
2	-0.11475586E 02	-0.13504669E 02	0.0	0.0	0.0	0.0		0.0	0.0	0.0	0.0	0.0	0.0	0.0
3	-0.46917236E 02	-0.30122070E 01	0.0	0.0	0.0	0.0		0.0	0.0	0.0	0.0	0.0	0.0	0.0
4	-0.21386719E 00	0.70776367E 01	0.0	0.0	0.0	0.0		0.0	0.0	0.0	0.0	0.0	0.0	0.0
5	0.59134613E 02	0.40854452E 01	0.0	0.0	0.0	0.0		0.0	0.0	0.0	0.0	0.0	0.0	0.0
6	-0.64804688E 01	-0.16737771E 00	0.0	0.0	0.0	0.0		0.0	0.0	0.0	0.0	0.0	0.0	0.0
7	0.44508564E 00	-0.24165522E-01	0.0	0.0	0.0	0.0		0.0	0.0	0.0	0.0	0.0	0.0	0.0
8	0.0	-0.35355539E 01	0.0	0.0	0.0	0.0		0.0	0.0	0.0	0.0	0.0	0.0	0.0
ELEMENT APPLIED FORCES														
POINT	FX	FY	FZ	MX	MY	MZ		FX	FY	FZ	MX	MY	MZ	
1	0.0	0.0	0.0	0.0	0.0	0.0		0.0	0.0	0.0	0.0	0.0	0.0	0.0
2	0.0	0.0	0.0	0.0	0.0	0.0		0.0	0.0	0.0	0.0	0.0	0.0	0.0
3	0.0	0.0	0.0	0.0	0.0	0.0		0.0	0.0	0.0	0.0	0.0	0.0	0.0
4	0.0	0.0	0.0	0.0	0.0	0.0		0.0	0.0	0.0	0.0	0.0	0.0	0.0
5	0.0	0.0	0.0	0.0	0.0	0.0		0.0	0.0	0.0	0.0	0.0	0.0	0.0
6	0.0	0.0	0.0	0.0	0.0	0.0		0.0	0.0	0.0	0.0	0.0	0.0	0.0
7	0.0	0.0	0.0	0.0	0.0	0.0		0.0	0.0	0.0	0.0	0.0	0.0	0.0
8	0.0	0.0	0.0	0.0	0.0	0.0		0.0	0.0	0.0	0.0	0.0	0.0	0.0
NET ELEMENT FORCES														
POINT	FX	FY	FZ	MX	MY	MZ		FX	FY	FZ	MX	MY	MZ	
1	0.53361816E 01	0.90810547E 01	0.0	0.0	0.0	0.0		0.0	0.0	0.0	0.0	0.0	0.0	0.0
2	-0.11475586E 02	-0.13504669E 02	0.0	0.0	0.0	0.0		0.0	0.0	0.0	0.0	0.0	0.0	0.0
3	-0.46917236E 02	-0.30122070E 01	0.0	0.0	0.0	0.0		0.0	0.0	0.0	0.0	0.0	0.0	0.0
4	-0.21386719E 00	0.70776367E 01	0.0	0.0	0.0	0.0		0.0	0.0	0.0	0.0	0.0	0.0	0.0
5	0.59134613E 02	0.40854452E 01	0.0	0.0	0.0	0.0		0.0	0.0	0.0	0.0	0.0	0.0	0.0
6	-0.64804688E 01	-0.16737771E 00	0.0	0.0	0.0	0.0		0.0	0.0	0.0	0.0	0.0	0.0	0.0
7	0.44508564E 00	-0.24169922E-01	0.0	0.0	0.0	0.0		0.0	0.0	0.0	0.0	0.0	0.0	0.0
8	0.0	-0.35355539E 01	0.0	0.0	0.0	0.0		0.0	0.0	0.0	0.0	0.0	0.0	0.0

FIGURE III-G.25 FORCE OUTPUT, ELEMENT NO. 2 - LAP JOINT PROBLEM

FORCES FOR THE HIGH ASPECT RATIO QUADRILATERAL ELEMENT  
 (THE FIRST FOUR POINTS ARE CORNER POINTS AND THE LAST FOUR POINTS ARE MID-POINTS)

LOAD CONDITION NUMBER	ELEMENT NUMBER	ELEMENT TYPE	ELEMENT GRID POINTS	ELEMENT GRID POINTS				ELEMENT GRID POINTS				
1	3	38	8 10 18 12	8 10 18 12	8 10 18 12	8 10 18 12	8 10 18 12	8 10 18 12	8 10 18 12	8 10 18 12	8 10 18 12	
APPARENT ELEMENT FORCES			FORCES				MOMENTS					
POINT	FX	FY	FZ	MX	MY	MZ	FX	FY	FZ	MX	MY	MZ
1	0.47354660E 02	-0.31923780E 01	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0
2	0.12687969E 00	-0.31375885E -02	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0
3	-0.20855637E 02	-0.95370646E 01	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0
4	0.25125061E 02	0.18111542E 02	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0
5	-0.29732704E 00	-0.84157214E -02	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0
6	-0.23437500E -01	-0.21057125E -01	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0
7	-0.58130890E 02	-0.55389250E 01	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0
8	0.65273438E 01	0.19206238E 00	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0

ELEMENT APPLIED FORCES			FORCES				MOMENTS					
POINT	FX	FY	FZ	MX	MY	MZ	FX	FY	FZ	MX	MY	MZ
1	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0
2	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0
3	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0
4	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0
5	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0
6	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0
7	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0
8	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0

NET ELEMENT FORCES			FORCES				MOMENTS					
POINT	FX	FY	FZ	MX	MY	MZ	FX	FY	FZ	MX	MY	MZ
1	0.47354660E 02	-0.31923780E 01	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0
2	0.12687969E 00	-0.31375885E -02	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0
3	-0.20855637E 02	-0.95370646E 01	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0
4	0.25125061E 02	0.18111542E 02	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0
5	-0.29732704E 00	-0.84157214E -02	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0
6	-0.23437500E -01	-0.21057125E -01	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0
7	-0.58130890E 02	-0.55389250E 01	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0
8	0.65273438E 01	0.19206238E 00	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0

FIGURE III-G.26 FORCE OUTPUT, ELEMENT NO. 3 - LAP JOINT PROBLEM

FORCES FOR THE HIGH ASPECT RATIO QUADRILATERAL ELEMENT  
 (THE FIRST FOUR POINTS ARE CORNER POINTS AND THE LAST FOUR POINTS ARE MID-POINTS)

LOAD CONDITION NUMBER	ELEMENT NUMBER	ELEMENT TYPE	ELEMENT GRID POINTS	FORCES	MOMENTS
1	4	30	16 18 23 21 17 20 22 19	FX FY FZ	MX MY MZ
APPARENT ELEMENT FORCES					
POINT					
1				0.21474731E 02	0.0
2				0.20874466E 02	0.0
3				-0.14901428E 02	0.0
4				-0.15232101E 02	0.0
5				0.58146286E 02	0.0
6				0.10498047E-01	0.0
7				-0.70403564E 02	0.0
8				-0.97656250E-02	0.0

ELEMENT APPLIED FORCES	FORCES	MOMENTS
POINT	FX FY FZ	MX MY MZ
1	0.0	0.0
2	0.0	0.0
3	0.0	0.0
4	0.0	0.0
5	0.0	0.0
6	0.0	0.0
7	0.0	0.0
8	0.0	0.0

NET ELEMENT FORCES	FORCES	MOMENTS
POINT	FX FY FZ	MX MY MZ
1	0.21474731E 02	0.0
2	0.20874466E 02	0.0
3	-0.14901428E 02	0.0
4	-0.15232101E 02	0.0
5	0.58146286E 02	0.0
6	0.10498047E-01	0.0
7	-0.70403564E 02	0.0
8	-0.97656250E-02	0.0

FIGURE III-G.27 FORCE OUTPUT, ELEMENT NO. 4 - LAP JOINT PROBLEM



## H. TRIANGULAR RING ASYMMETRIC LOADING (THICK WALLED DISC)

A thick walled disc was analyzed to determine its response to typical asymmetric pressure and thermal loadings. The dimensions of the disc, its pertinent material properties and the subsequent three element idealization are pictured in Figure III-H.1.

Individual analyses of the disc were carried out for the pressure and thermal loadings respectively. The input and output for the pressure loading will be discussed first. Changes in the input and output brought about by application of the thermal loading will be discussed later in this section. Both the applied pressure and thermal loads chosen possessed the same variation  $(1 + \cos 2\theta)$  in the circumferential coordinate  $\theta$ . This variation was chosen because it could be described exactly by the MAGIC III program utilizing the (0) and (+2) harmonics.

### Asymmetric Pressure Loading

The preprinted input data forms associated with the asymmetric pressure load problem are shown in Figures III-H.2 through III-H.9. The input illustrated in Figures III-H.2 through III-H.7 is completed in a similar manner as that provided for the Axisymmetric Triangular Ring Element (See Reference 5). The only notable difference between the two elements (Axi and Asymmetric Triangular Ring) being in the input linked to the external loading conditions.

As has been previously indicated (See Section II.C) the difference in manner of input for external loads is quite large between the Axisymmetric and Asymmetric Triangular Ring Elements. Input options specialized and linked to the former must be abandoned when utilizing the Asymmetric Triangular Ring Element. Examples of the options to be ignored in this instance are the Temperature Interpolate Option and Pressure Suppression Options of the Axisymmetric Triangular Ring (See Sections II.C.8).

The only element of the Thick Walled Disc assumed loaded (See Figures III-H.1 and III-H.8) is element number 1. This loading was assumed acting radially outward and possessed a circumferential  $(1 + \cos 2\theta)$  variation.

# Contrails

The Asymmetric Pressure Load Input is accomplished through the form illustrated in Figure III-H.8. The first entry on the form is pre-labeled HARM, and requires no input from the user. The second entry on the form contains the information that our problem includes:

- a) one loaded element,
- b) a maximum of two harmonics will be chosen to represent the loading on the element,
- c) and a maximum of two harmonics are to be used for the analysis of the thick walled disc.

The third entry on the form provides that:

- a) the loaded element is Element Number One,
- b) it is loaded in the radial direction only and (36) values of the loading at equally spaced intervals are being provided,
- c) and finally the actual values at these 36 intervals are input.

Designation of points about the structure, in this case the Thick Walled Disc, where output of stresses and displacements are to be provided is accomplished by the form illustrated in Figure III-H.9. The first entry on the form is pre-labeled HSDC and requires no input from the user. The second entry on the form provides that output of stresses and displacements will be provided over the entire circumference of the Thick Walled Disc ( $360^\circ$ ) at ( $30^\circ$ ) intervals.

A sampling of the output derived from the analysis of the previously described Thick Walled Disc under the Asymmetric Radial Pressure Loading is presented and discussed. Reference should be made to Figures III-H.10 through III-H.19.

Figures III-H.10, III-H.11 and III-H.12 present typical element data output of pertinent material data, gridpoint coordinates, boundary conditions and element definitions (for Elements 1 and 2). This output is consistent with that presented for the Axisymmetric ring element.

The output presented in Figures III-H.13 and III-H.14 describes the asymmetric loading applied to the thick walled disc. Figure III-H.13 confirms that a radial loading has been placed on Element No. (1), that a limit of two harmonics describing the loading has been set and also presents the 36 circumferential values of

the radial load used to describe the loading. Figure III-H.14 presents the harmonic loads which result from the Fourier decomposition, carried out automatically by the MAGIC III Program, of the loading defined in Figure III-H.13. In the question of the Thick Walled Disc under consideration, the program has determined that the radial loading on Element (1) for a given circumferential location ( $\theta$ ) can be expressed as follows (with reference to Figure H.14)

$$P_r(\theta) = - \{ 100.027 + 98.9766 \cos 2 \theta \} \quad (1)$$

Referencing Sections III-C.8f, it is evident that complete two dimensional analyses for the ( $m = 0$ ) and ( $m = +2$ ) harmonics are required to carry out the analysis of the Thick Walled Disc. This involves the MAGIC III program assembling structure stiffness matrices for the ( $m = 0$ ) and ( $m = +2$ ) harmonics. Figures III-H.15 a and b provide the element (for Element #1) harmonic stiffness and load matrices for harmonics ( $m = 0$ ) and ( $m = +2$ ) which are used in assembling the structure (Disc) stiffness and load matrices.

Figures III-H.16a and b present the harmonic stresses (for  $m = 0$  and  $m = +2$ ) for Element #1 which result from the above analyses. The harmonic stresses presented in these two figures can be combined as shown below to evaluate the stress in Element #1 at a centroidal location (cross-section) and at an arbitrary circumferential location  $\theta$ .

$$\begin{bmatrix} \sigma_{rr}(\theta) \\ \sigma_{\theta\theta}(\theta) \\ \sigma_{\theta z}(\theta) \\ \sigma_{rz}(\theta) \\ \sigma_{rz}(\theta) \\ \sigma_{\theta z}(\theta) \end{bmatrix} = \begin{bmatrix} -75.384766 \\ 3.0233459 \\ -163.20439 \\ -7.5593872 \\ 0 \\ 0 \end{bmatrix} + [C_2(\theta)] \begin{bmatrix} -111.55794 \\ -10.439285 \\ -458.28198 \\ 25.463989 \\ 101.23071 \\ -3.4822311 \end{bmatrix} \quad (2)$$

The matrix  $C_2(\theta)$  is the diagonal matrix

$$[C_2(\theta)] = [\cos 2\theta, \cos 2\theta, \cos 2\theta, \cos 2\theta, \sin 2\theta, \sin 2\theta]. \quad (3)$$

Expressions similar to that given by Equation (2) above can be obtained for the displacements and reactions at the nodes of Element #1 for an arbitrary circumferential location  $\theta$ .

The expressions for the circumferentially varying displacements, reactions and stresses of the nodes (and consequently elements) which define the Thick Walled Disc were evaluated in accordance with the information provided on the HSDC form (Figure III-H.9) by the MAGIC III Program. Displacements, reactions and stresses were consequently provided for all elements at  $30^\circ$  intervals completely around the structure. Figures III-H.17 and III-H.18 provide the displacements and reactions for all five of the structures nodes for two selected circumferential positions ( $\theta = 0^\circ$  and  $\theta = 60^\circ$ ). Figure III-H.19 provides the stresses at the centroid of Element 1 for all 12 specified locations.

### Asymmetric Thermal Loading

The Thick Walled Disc was analyzed to determine the effects of an applied asymmetric thermal loading. The loading possessed an  $(1 + \cos 2\theta)$  circumferential variation in magnitude and varied non-linearly through the cross-section.

The asymmetric temperature load input for this test case is accomplished through the form illustrated in Figures III-H.20 a and b. The first entry on the form is pre-labeled HTEM, and requires no input from the user. The second entry on the form contains the information that the test case includes:

- a) three elements loaded by asymmetric temperature distributions,
- b) a maximum of two harmonics to be chosen to represent the thermal loadings on the elements,
- c) and a maximum of two harmonics to be used for the analysis of the thick walled disc.

The following three entries in Figures III-H.20 a and b provide

- a) the numbers of the three loaded elements,
- b) the information that (36) values of the loadings will be provided for each of the three elements,
- c) and the values of these loadings at (36) intervals for the three loaded elements.



# Contrails

The thermal run is accomplished by substituting the HTEM input for the HARM input provided earlier (Figure III-H.8) and providing the remainder of the input as before. The input for the case of the asymmetrically loaded Thick Walled Disc is reviewed by Figure III-H.21. Selected output from the MAGIC III Program is provided for this analysis in Figure III-H.22 through Figure III-H.26.

Figure III-H.22 describes the asymmetric thermal loading applied to Element (1). The values provided in this figure which comprise the loading must be interpreted as changes in temperature to which the element is subjected at varying circumferential locations. These temperature changes can be imagined as occurring at the centroid of the element cross-section. Figure III-H.23 presents the harmonic loads (coefficients) which result from the Fourier decomposition, carried out automatically by the MAGIC III program, of the loading defined in Figure III-H.22.

Figures III-H.24 a and b present the net harmonic stresses (coefficients in the Fourier series which represent the net stresses on Element 1) for harmonics  $m = 0$  and  $m = +2$ . The net stress of Element 1 can be expressed in the following Fourier series form

$$\{\sigma\} = \{\sigma_0\} + \sum_m \{C_{0,m}\} + \sum_m \{\bar{C}_{0,m}\} \{\bar{\sigma}_m\} \quad (4)$$

where the diagonal matrices  $\{C_{0,m}\}$  and  $\{\bar{C}_{0,m}\}$  appear as

$$\{C_{0,m}\} = \begin{bmatrix} \cos m\theta & \cos m\theta & \cos m\theta & \cos m\theta & \sin m\theta & \sin m\theta \end{bmatrix} \quad (5)$$

and

$$\{\bar{C}_{0,m}\} = \begin{bmatrix} \sin m\theta & \sin m\theta & \sin m\theta & \sin m\theta & \cos m\theta & \cos m\theta \end{bmatrix}.$$

The net harmonic stress for the A-series,  $m^{\text{th}}$  harmonic can be expressed as

$$\{\sigma_m\} = [E] \{\epsilon_m\} - \{SZAEL(m)\} \quad (6)$$

where

$$[E] \{\epsilon_m\} = \text{harmonic apparent element stress,} \quad (7)$$

and

$$\{SZAEL(m)\} = \text{harmonic element applied stress.} \quad (8)$$

# Contrails

The vector  $\{SZAEL(m)\}$  is a harmonic stress coefficient correction vector for any element possessing an applied asymmetric (or axisymmetric) temperature load.  $\{SZAEL(m)\}$  is calculated as follows (for the A series,  $m^{th}$  harmonic):

$$\{SZAEL(m)\} = T(m) [E] \{\alpha\} \quad (9)$$

where  $[E]$  is the material property matrix which has the form

$$[E] = \frac{1}{\Delta} \begin{bmatrix} E_n(1-\nu_{03}\nu_{30}), E_n(\nu_{3n}+\nu_{30n}), E_n(\nu_{0n}+\nu_{3n0}), & 0 & , & 0 & , & 0 \\ E_n(1-\nu_{n0}\nu_{0n}), E_n(\nu_{30}+\nu_{n03}), & 0 & , & 0 & , & 0 \\ E_n(1-\nu_{n3}\nu_{3n}), & 0 & , & 0 & , & 0 \\ & \Delta G_{n3} & , & 0 & , & 0 \\ & & & \Delta G_{r0} & , & 0 \\ & & & & & \Delta G_{30} \end{bmatrix} \quad (10)$$

and where  $\Delta = (1 - \nu_{n0}\nu_{0n} - \nu_{03}\nu_{30} - \nu_{3n}\nu_{n3} - \nu_{n0}\nu_{03}\nu_{3n} - \nu_{n3}\nu_{0n}\nu_{30})$ .

The matrix  $[E]$  for the Thick Walled Disc (which is constructed using an isotropic material) is

# Contrails

$$[E] = \frac{E}{\Delta} \begin{bmatrix} 1-\nu^2 & ; & \nu(1+\nu) & ; & \nu(1+\nu) & ; & 0 & ; & 0 & ; & 0 \\ & & (1-\nu^2) & ; & \nu(1+\nu) & ; & 0 & ; & 0 & ; & 0 \\ & & & & (1-\nu^2) & ; & 0 & ; & 0 & ; & 0 \\ & & & & & & \frac{\Delta}{2(1+\nu)} & ; & 0 & ; & 0 \\ & & & & & & & & \frac{\Delta}{2(1+\nu)} & ; & 0 \\ & & & & & & & & & & \frac{\Delta}{2(1+\nu)} \end{bmatrix} \quad (12)$$

where  $\Delta = 1 - 3\nu^2 - 2\nu^3$ . (13)

The thermal coefficient vector for the Thick Walled Disc (isotropic material) is

$$\{\alpha\}^T = [ \alpha, \alpha, \alpha, 0, 0, 0 ] \quad (14)$$



# Contrails

The properties utilized in the analysis (Figure III-H.21) are defined below

$$\begin{aligned}
 E &= 30 \times 10^6 && \text{a)} \\
 \nu &= 0.3 && \text{b)} \\
 \alpha &= 6 \times 10^{-6} && \text{c)}
 \end{aligned}
 \tag{15}$$

The scalar  $T(m)$  is the harmonic temperature (coefficient in the Fourier series representing the applied asymmetric temperature loading on the element) and assumes the following values for harmonics  $m = 0$  and  $m = +2$  for Element of the Thick Walled Disc (see Figure III-H.23).

$$\begin{aligned}
 T(0) &= 353.526 \\
 T(+2) &= 349.965
 \end{aligned}
 \tag{16}$$

The vector  $\{SZAEL(m)\}$  for an isotropic material can be expressed as

$$\{SZAEL(m)\} = \begin{bmatrix} \sigma_{\theta m} \\ \sigma_{\theta m} \\ \sigma_{\theta m} \\ 0 \\ 0 \\ 0 \end{bmatrix}
 \tag{17}$$

where

$$\sigma_{\theta m} = \frac{(1 + \nu)^2}{1 - 3\nu^2 - 2\nu^3} E \alpha T_m
 \tag{18}$$

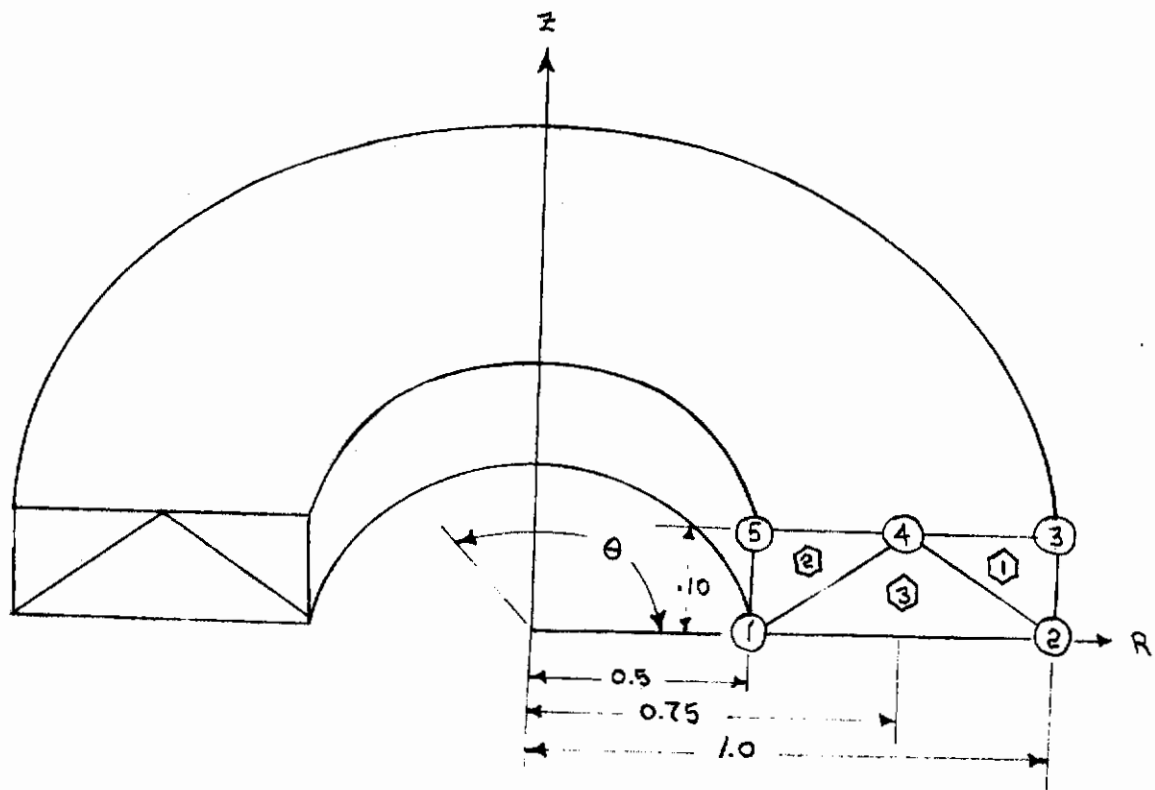
Evaluating Equation III-H.18 for harmonics  $m = 0$ ,  $m = +2$  for Element Number 1 of the Thick Walled Disc:

$$\begin{aligned}
 \sigma_{\theta}(0) &= .15908656 \text{ E } 06 && \text{a)} \\
 \sigma_{\theta}(+2) &= .15748388 \text{ E } 06 && \text{b)}
 \end{aligned}
 \tag{19}$$

The quantities  $\sigma_{\theta}(0)$  and  $\sigma_{\theta}(+2)$  appear as harmonic element applied stresses in Figures III-H.24 a and b.

# Contrails

The displacements for the 5 nodes of the Thick Walled Disc are provided for  $\theta = 0^\circ$  and  $\theta = 60^\circ$  (Figures III-H.25 a and b). The net stress distribution in Element 1 is provided in Figure III-H.26.



$$E = 3.0 \times 10^7 \text{ PSI}$$

$$\mu = 0.3$$

$$\alpha = 6.0 \times 10^{-6}$$

FIGURE III-H.1 IDEALIZED THICK WALLED DISC





BAC 1618

## MAGIC STRUCTURAL ANALYSIS SYSTEM INPUT DATA FORMAT

### SYSTEM CONTROL INFORMATION

	ENTER APPROPRIATE NUMBER, RIGHT ADJUSTED, IN BOX OPPOSITE APPLICABLE REQUESTS																	
		<table border="1" style="margin: auto;"> <tr> <td>S</td><td>Y</td><td>S</td><td>T</td><td>E</td><td>M</td> </tr> <tr> <td>1</td><td>2</td><td>3</td><td>4</td><td>5</td><td>6</td> </tr> </table> (/)	S	Y	S	T	E	M	1	2	3	4	5	6				
S	Y	S	T	E	M													
1	2	3	4	5	6													
1. Number of System Grid Points		<table border="1" style="margin: auto;"> <tr> <td></td><td></td><td></td><td></td><td></td><td>5</td> </tr> <tr> <td>1</td><td>2</td><td>3</td><td>4</td><td>5</td><td>6</td> </tr> </table>						5	1	2	3	4	5	6				
					5													
1	2	3	4	5	6													
2. Number of Input Grid Points		<table border="1" style="margin: auto;"> <tr> <td></td><td></td><td></td><td></td><td></td><td>5</td> </tr> <tr> <td>7</td><td>8</td><td>9</td><td>10</td><td>11</td><td>12</td> </tr> </table>						5	7	8	9	10	11	12				
					5													
7	8	9	10	11	12													
3. Number of Degrees of Freedom/Grid Point		<table border="1" style="margin: auto;"> <tr> <td></td><td>3</td> </tr> <tr> <td>13</td><td>14</td> </tr> </table>		3	13	14												
	3																	
13	14																	
4. Number of Load Conditions		<table border="1" style="margin: auto;"> <tr> <td></td><td>0</td> </tr> <tr> <td>15</td><td>16</td> </tr> </table>		0	15	16												
	0																	
15	16																	
5. Number of Initially Displaced Grid Points		<table border="1" style="margin: auto;"> <tr> <td></td><td></td><td></td><td></td><td></td><td>0</td> </tr> <tr> <td>17</td><td>18</td><td>19</td><td>20</td><td>21</td><td>22</td> </tr> </table>						0	17	18	19	20	21	22				
					0													
17	18	19	20	21	22													
6. Number of Prescribed Displaced Grid Points		<table border="1" style="margin: auto;"> <tr> <td></td><td></td><td></td><td></td><td></td><td>0</td> </tr> <tr> <td>23</td><td>24</td><td>25</td><td>26</td><td>27</td><td>28</td> </tr> </table>						0	23	24	25	26	27	28				
					0													
23	24	25	26	27	28													
7. Number of Grid Point Axes Transformation Systems		<table border="1" style="margin: auto;"> <tr> <td></td><td>0</td> </tr> <tr> <td>29</td><td>30</td> </tr> </table>		0	29	30												
	0																	
29	30																	
8. Number of Elements		<table border="1" style="margin: auto;"> <tr> <td></td><td></td><td></td><td></td><td></td><td>3</td> </tr> <tr> <td>31</td><td>32</td><td>33</td><td>34</td><td>35</td><td>36</td> </tr> </table>						3	31	32	33	34	35	36				
					3													
31	32	33	34	35	36													
9. Number of Requests and/or Revisions of Material Tape.		<table border="1" style="margin: auto;"> <tr> <td></td><td>1</td> </tr> <tr> <td>37</td><td>38</td> </tr> </table>		1	37	38												
	1																	
37	38																	
10. Number of Input Boundary Condition Points		<table border="1" style="margin: auto;"> <tr> <td></td><td></td><td></td><td></td><td></td><td>5</td> </tr> <tr> <td>39</td><td>40</td><td>41</td><td>42</td><td>43</td><td>44</td> </tr> </table>						5	39	40	41	42	43	44				
					5													
39	40	41	42	43	44													
11. $T_0$ For Structure (With Decimal Point)		<table border="1" style="margin: auto;"> <tr> <td></td><td></td><td></td><td></td><td></td><td>0</td><td>.</td><td>0</td> </tr> <tr> <td>45</td><td>46</td><td>47</td><td>48</td><td>49</td><td>50</td><td>51</td><td>52</td> </tr> </table> (/)						0	.	0	45	46	47	48	49	50	51	52
					0	.	0											
45	46	47	48	49	50	51	52											

FIGURE III-H.4 SYSTEM CONTROL INFORMATION, THICK WALLED DISC











BAC 2060

MAGIC STRUCTURAL ANALYSIS SYSTEM  
INPUT DATA FORMAT

TRIANGULAR RING ELEMENT (ASYMMETRIC LOADING)  
HARMONIC INCREMENTS

H	S	D	C	
1	2	3	4	5

( / )

REF. VALUE	INC. VALUE	FINC. VALUE
1		2
7	8	9
0	1	2
3	4	5
6	7	8
9	0	1
0	3	6
0	3	6

( / )

FIGURE III-H.9 DESIGNATION OF STRESS AND DISPLACEMENT  
OUTPUT LOCATIONS; THICK WALLED DISC

THICK WALLED DISC SUBJECTED TO NON-AXISYMMETRIC LOADING

REVISIONS OF MATERIAL TAPE

ASTERISK (\*) PRECEEDING MATERIAL IDENTIFICATION INDICATES THAT INPUT ERROR RETURNS WILL NOT RESULT IN TERMINATION OF EXECUTION

\*\*\*\*\*

REVISION 12  
MATERIAL IDENTIFICATION STEEL - E=30E6, MU=0.3 INPUT CODE I  
NUMBER OF MATERIAL PROPERTY POINTS . . . . 1  
NUMBER OF PLASTIC PROPERTY POINTS . . . . 0  
MASS DENSITY . . . . . 0.49999985E-03

MATERIAL PROPERTIES

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YOUNG'S MODULI

POISSON'S RATIOS

TEMPERATURE	0.0	0.300000E 08	0.300000E 08	0.300000E 08	0.300000E 08	0.300000E 00	0.300000E 00	0.300000E 00	0.300000E 00
		XX	YY	ZZ	XY	YZ	ZX		
		DIRECTIONS							

TH. EXP.COEFF.

RIGIDITY MODULI

TEMPERATURE	0.0	0.600000E-05	0.600000E-05	0.600000E-05	0.600000E-05	0.600000E-05	0.600000E-05	0.600000E-05	0.600000E-05
		XX	YY	ZZ	XY	YZ	ZX		
		DIRECTIONS							

FIGURE III-H.10 TITLE AND MATERIAL DATA OUTPUT, THICK WALLED DISC

5 REF. POINTS

NO. DIRECTIONS = 3 NO. DEGREES OF FREEDOM = 1

GRIDPOINT DATA  
(IN RECTANGULAR COORDINATES)

POINT	X	Y	Z	TEMPERATURES	PRESSURES
1	0.5000000E 00	0.0	0.0	0.0 0.0 0.0 0.0 0.0	0.0 0.0 0.0 0.0 0.0
2	0.1000000E 01	0.0	0.0	0.0 0.0 0.0 0.0 0.0	0.0 0.0 0.0 0.0 0.0
3	0.1000000E 01	0.0	0.99999964E-01	0.0 0.0 0.0 0.0 0.0	0.0 0.0 0.0 0.0 0.0
4	0.7500000E 00	0.0	0.99999964E-01	0.0 0.0 0.0 0.0 0.0	0.0 0.0 0.0 0.0 0.0
5	0.5000000E 00	0.0	0.99999964E-01	0.0 0.0 0.0 0.0 0.0	0.0 0.0 0.0 0.0 0.0

BOUNDARY CCNDITION INFORMATION

NODES	DEGREES OF FREEDOM	NO. OF JNES	NO. OF TWDS
1	1	3	0
2	1	6	0
3	1	9	0
4	1	12	0
5	1	14	0

374

FIGURE III-H.11 GRIDPOINT DATA AND BOUNDARY CONDITION OUTPUT, THICK WALLED DISC

```

ELEM TYPE MAT.NO. CODE TEMP. PRINT NO. 2 3 4 5 6 7 8 9 10 11 12 13 14 15 16 17 18 19 20 21 22 23 24 25 26 27 28 29 30 31 32 33 34 35 36 37 38 39 40 41 42 43 44 45 46 47 48 49 50 51 52 53 54 55 56 57 58 59 60 61 62 63 64 65 66 67 68 69 70 71 72 73 74 75 76 77 78 79 80 81 82 83 84 85 86 87 88 89 90 91 92 93 94 95 96 97 98 99 100
1 31 12 0 0.0 2 3 4 5 6 7 8 9 10 11 12 13 14 15 16 17 18 19 20 21 22 23 24 25 26 27 28 29 30 31 32 33 34 35 36 37 38 39 40 41 42 43 44 45 46 47 48 49 50 51 52 53 54 55 56 57 58 59 60 61 62 63 64 65 66 67 68 69 70 71 72 73 74 75 76 77 78 79 80 81 82 83 84 85 86 87 88 89 90 91 92 93 94 95 96 97 98 99 100
-----SECTION PROPERTIES-----
MATERIAL NUMBER . . . . . 12
MATERIAL IDENTIFICATION . . . . . STEEL - E3086, M=0.3
ANALYSIS CAPABILITY . . . . . ISOTROPIC
INPUT PRINT CODE . . . . . 1
ELEMENT PRINT CODE . . . . . -1

INTERPOLATED MATERIAL PROPERTIES
TEMPERATURE - 0.0
YOUNG'S MODULI 0.3000000E 08 0.3000000E 08 0.3000000E 08 0.3000000E 08
POISSON'S RATIO 0.2999999E 00 0.2999999E 00 0.2999999E 00 0.2999999E 00
TM. EXP. COEF. 0.1133846E 05 0.1133846E 05 0.1133846E 05 0.1133846E 05
RIGIDITY MODULI 0.1133846E 05 0.1133846E 05 0.1133846E 05 0.1133846E 05

INTERPOLATED PLASTIC PROPERTIES
NONE

PRE-STRAIN INPUT
NONE

PRE-STRESS INPUT
NONE

EXTERNAL INPUT
NONE
    
```

```

ELEM TYPE MAT.NO. CODE TEMP. PRINT NO. 2 3 4 5 6 7 8 9 10 11 12 13 14 15 16 17 18 19 20 21 22 23 24 25 26 27 28 29 30 31 32 33 34 35 36 37 38 39 40 41 42 43 44 45 46 47 48 49 50 51 52 53 54 55 56 57 58 59 60 61 62 63 64 65 66 67 68 69 70 71 72 73 74 75 76 77 78 79 80 81 82 83 84 85 86 87 88 89 90 91 92 93 94 95 96 97 98 99 100
2 31 12 0 0.0 2 3 4 5 6 7 8 9 10 11 12 13 14 15 16 17 18 19 20 21 22 23 24 25 26 27 28 29 30 31 32 33 34 35 36 37 38 39 40 41 42 43 44 45 46 47 48 49 50 51 52 53 54 55 56 57 58 59 60 61 62 63 64 65 66 67 68 69 70 71 72 73 74 75 76 77 78 79 80 81 82 83 84 85 86 87 88 89 90 91 92 93 94 95 96 97 98 99 100
-----SECTION PROPERTIES-----
MATERIAL NUMBER . . . . . 12
MATERIAL IDENTIFICATION . . . . . STEEL - E3086, M=0.3
ANALYSIS CAPABILITY . . . . . ISOTROPIC
INPUT PRINT CODE . . . . . 1
ELEMENT PRINT CODE . . . . . -1

INTERPOLATED MATERIAL PROPERTIES
TEMPERATURE - 0.0
YOUNG'S MODULI 0.3000000E 08 0.3000000E 08 0.3000000E 08 0.3000000E 08
POISSON'S RATIO 0.2999999E 00 0.2999999E 00 0.2999999E 00 0.2999999E 00
TM. EXP. COEF. 0.1133846E 05 0.1133846E 05 0.1133846E 05 0.1133846E 05
RIGIDITY MODULI 0.1133846E 05 0.1133846E 05 0.1133846E 05 0.1133846E 05

INTERPOLATED PLASTIC PROPERTIES
NONE

PRE-STRAIN INPUT
NONE

PRE-STRESS INPUT
NONE

EXTERNAL INPUT
NONE
    
```

```

ELEM TYPE MAT.NO. CODE TEMP. PRINT NO. 2 3 4 5 6 7 8 9 10 11 12 13 14 15 16 17 18 19 20 21 22 23 24 25 26 27 28 29 30 31 32 33 34 35 36 37 38 39 40 41 42 43 44 45 46 47 48 49 50 51 52 53 54 55 56 57 58 59 60 61 62 63 64 65 66 67 68 69 70 71 72 73 74 75 76 77 78 79 80 81 82 83 84 85 86 87 88 89 90 91 92 93 94 95 96 97 98 99 100
3 31 12 0 0.0 2 3 4 5 6 7 8 9 10 11 12 13 14 15 16 17 18 19 20 21 22 23 24 25 26 27 28 29 30 31 32 33 34 35 36 37 38 39 40 41 42 43 44 45 46 47 48 49 50 51 52 53 54 55 56 57 58 59 60 61 62 63 64 65 66 67 68 69 70 71 72 73 74 75 76 77 78 79 80 81 82 83 84 85 86 87 88 89 90 91 92 93 94 95 96 97 98 99 100
-----SECTION PROPERTIES-----
MATERIAL NUMBER . . . . . 12
MATERIAL IDENTIFICATION . . . . . STEEL - E3086, M=0.3
ANALYSIS CAPABILITY . . . . . ISOTROPIC
INPUT PRINT CODE . . . . . 1
ELEMENT PRINT CODE . . . . . -1

INTERPOLATED MATERIAL PROPERTIES
TEMPERATURE - 0.0
YOUNG'S MODULI 0.3000000E 08 0.3000000E 08 0.3000000E 08 0.3000000E 08
POISSON'S RATIO 0.2999999E 00 0.2999999E 00 0.2999999E 00 0.2999999E 00
TM. EXP. COEF. 0.1133846E 05 0.1133846E 05 0.1133846E 05 0.1133846E 05
RIGIDITY MODULI 0.1133846E 05 0.1133846E 05 0.1133846E 05 0.1133846E 05

INTERPOLATED PLASTIC PROPERTIES
NONE

PRE-STRAIN INPUT
NONE

PRE-STRESS INPUT
NONE

EXTERNAL INPUT
NONE
    
```

375 FIGURE III-H.12 FINITE ELEMENT DESCRIPTION OUTPUT, THICK WALLED DISC



HARMONIC LOADS (PRESSURE)

ELEMENT NO	DIRECTION OF LOADING	NUMBER OF POINTS	NUMBER OF HARMONIC REQUESTS		LOADS					
			1	2	POINT	POINT	POINT	POINT	POINT	POINT
1	RADIAL	36	4	7	-0.200000E 03	-0.193970E 03	-0.176600E 03	-0.150000E 03	-0.117370E 03	-0.826200E 02
			7	10	-0.500000E 02	-0.234000E 02	-0.603000E 01	-0.603000E 02	-0.603000E 01	-0.603000E 01
			10	13	0.0	-0.826300E 02	-0.117370E 03	-0.176600E 02	-0.117370E 03	-0.234000E 02
			13	16	-0.500000E 02	-0.176600E 03	-0.193970E 03	-0.150000E 03	-0.193970E 03	-0.193970E 03
			16	19	-0.150000E 03	-0.117370E 03	-0.176600E 02	-0.200000E 03	-0.176600E 03	-0.176600E 03
			19	22	-0.200000E 03	-0.117370E 03	-0.176600E 02	-0.150000E 03	-0.117370E 03	-0.836300E 02
			22	25	-0.150000E 03	-0.234000E 02	-0.603000E 01	-0.500000E 02	-0.234000E 02	-0.603000E 01
			25	28	0.0	-0.603000E 01	-0.603000E 02	0.0	-0.603000E 01	-0.234000E 02
			28	31	-0.500000E 02	-0.826300E 02	-0.117370E 03	-0.500000E 02	-0.826300E 02	-0.117370E 03
			31	34	-0.150000E 03	-0.176600E 03	-0.176600E 03	-0.150000E 03	-0.117370E 03	-0.193970E 03

FIGURE III-H.13 ASYMMETRIC LOAD DATA OUTPUT, THICK WALLED DISC

RESULT OF HARMONIC ANALYSIS

ELEMENT NO	HARMONIC COEFFICIENT	FOURIER COEFFICIENT
1	0.0	-0.100027E 03
	0.200000E 01	-0.989766E 02

FIGURE III-H.14 HARMONIC LOAD OUTPUT, THICK WALLED DISC







STRESSES FOR THE ASYMMETRIC TRIANGULAR CROSS SECTION RING ELEMENT  
 (STRESSES EVALUATED AT THE ELEMENT CENTROID)

LOAD CONDITION NUMBER	ELEMENT NUMBER	ELEMENT TYPE	ELEMENT GRID POINTS
1	1	31	2 3 4
APPARENT ELEMENT STRESSES			
STRESS POINT	AXIAL (ZZ)	CIRCUMFERENTIAL (THETA-THETA)	SHEAR STRESSES (R-THETA) (Z-THETA)
1	-0.10439285E 02	-0.45828198E 03	0.25463989E 02 0.10123071E 03 -0.34822311E 01
ELEMENT APPLIED STRESSES			
STRESS POINT	AXIAL (ZZ)	CIRCUMFERENTIAL (THETA-THETA)	SHEAR STRESSES (R-THETA) (Z-THETA)
1	0.0	0.0	0.0 0.0 0.0
NET ELEMENT STRESSES			
STRESS POINT	AXIAL (ZZ)	CIRCUMFERENTIAL (THETA-THETA)	SHEAR STRESSES (R-THETA) (Z-THETA)
1	-0.10439285E 02	-0.45828198E 03	0.25463989E 02 0.10123071E 03 -0.34822311E 01

FIGURE III-H.16b HARMONIC STRESS COEFFICIENTS FOR ELEMENT NO. (1), HARMONIC (n = +2), THICK WALLED DISC

DISPLACEMENTS

THETA	GRID POINT	U	V	W
0	1	-0.296247E-04	0.0	0.544077E-06
0	2	-0.299754E-04	0.0	-0.498346E-05
0	3	-0.289156E-04	0.0	-0.419975E-05
0	4	-0.289300E-04	0.0	-0.193829E-05
0	5	-0.285588E-04	0.0	0.0

FIGURE III-H.17a NODAL CIRCLE DISPLACEMENTS AT  $\theta = 0^\circ$ , THICK WALLED DISC

DISPLACEMENTS

THETA	GRID POINT	U	V	W
60	1	0.787113E-05	0.192114E-04	-0.698266E-06
60	2	0.776971E-05	0.252034E-05	-0.299463E-05
60	3	0.816699E-05	0.234586E-05	-0.301347E-05
60	4	0.851580E-05	0.108076E-04	-0.158066E-05
60	5	0.820817E-05	0.191946E-04	0.0

FIGURE III-H.17b NODAL CIRCLE DISPLACEMENTS AT  $\theta = 60^\circ$ , THICK WALLED DISC

TMETA	GRID POINT	REACTIONS	U	V	W
0	1	0.109502E 01	0.106812E-03	-0.146484E-02	
0	2	-0.137329E-03	0.122070E-03	-0.103474E-02	
0	3	-0.427246E-03	-0.152588E-04	-0.619888E-04	
0	4	0.247955E-04	-0.915527E-04	0.117493E-02	
0	5	-0.991821E-03	0.0	0.976563E-03	

FIGURE III-H.18a NODAL CIRCLE REACTIONS AT  $\theta = 0^\circ$ ,  
THICK WALLED DISC

TMETA	GRID POINT	REACTIONS	U	V	W
60	1	-0.546215E 00	0.120364E-02	0.732376E-03	
60	2	-0.480645E-03	0.137509E 02	0.517336E-03	
60	3	0.946017E-03	0.137500E 02	0.309925E-04	
60	4	0.102042E-03	-0.109588E-02	-0.587427E-03	
60	5	0.519767E-03	-0.106627E-02	-0.488251E-03	

FIGURE III-H.18b NODAL CIRCLE REACTIONS AT  $\theta = 60^\circ$ ,  
THICK WALLED DISC



KEY STRESSES FOR THE TRIANGULAR RING ELEMENT  
(STRESSES EVALUATED AT ELEMENT CENTROID)

ELEMENT NUMBER	ELEMENT TYPE	CIRCUMFERENTIAL	NORMAL RADIAL	NORMAL CIRCUMFERENTIAL	NORMAL AXIAL	SHEAR R-Z	SHEAR R-THE TA	SHEAR Z-THE TA
1	31	0	-0.1869427E C3	-0.7415939E 01	-0.6214863E 03	0.1790460E 02	0.0	0.0
1	31	30	-0.1311643E 03	-0.2196350E 01	-0.3923477E 03	0.5172737E 01	0.8766806E 02	-0.3015690E 01
1	31	60	-0.1960695E 02	0.8242881E 01	0.6593187E 02	-0.2029111E 02	0.8766898E 02	-0.3015720E 01
1	31	90	0.3617317E 02	0.1346263E 02	0.2950774E 03	-0.3302338E 02	0.1704763E-02	-0.5864208E-04
1	31	120	-0.1960361E 02	0.8243194E 01	0.6594560E 02	-0.2029187E 02	-0.8766721E 02	0.3015660E 01
1	31	150	-0.1311609E 03	-0.2196036E 01	-0.3923337E 03	0.5171972E 01	-0.8766982E 02	0.3015750E 01
1	31	180	-0.1869427E C3	-0.7415939E 01	-0.6214863E 03	0.1790460E 02	-0.3409527E-02	0.1172842E-03
1	31	210	-0.1311676E 03	-0.2196654E 01	-0.3923611E 03	0.5173480E 01	0.8766635E 02	-0.3015631E 01
1	31	240	-0.1961011E 02	0.8242585E 01	0.6591888E C2	-0.2029039E 02	0.8767062E 02	-0.3015778E 01
1	31	270	0.3617317E 02	0.1346263E 02	0.2950774E 03	-0.3302338E 02	0.5114272E-02	-0.1759257E-03
1	31	300	-0.1960036E C2	0.8243498E 01	0.6595897E 02	-0.2029262E 02	-0.8766551E 02	0.3015602E 01
1	31	330	-0.1311578E C3	-0.2195741E 01	-0.3923208E 03	0.5171251E 01	-0.8767146E 02	0.3015807E 01

FIGURE III-H.19 STRESSES IN ELEMENT NO. (1), THICK WALLED DISC

# Contrails

1 2 3 4  
H T E M

(/)

NUMBER OF ELEMENTS				NO OF HARMONIC PER ELEMENT	
7	8	9	10	11	
		3	2	2	
NUMBER OF ELEMENTS				NO OF HARMONIC REQUESTS	
7	8	9	10	11	
		3	2	2	

(/)

ELEMENT ID			REPEAT ELEMENT LOADING OPTION				NUMBER OF LOADING POINTS	
7	8	9	10	11	12	13	3	6
		1						

HARMONIC THERMAL LOAD VALUES						
17	27	37	47	57	67	
707.1	685.8	624.4	530.2	414.9	292.1	
176.8	82.7	21.3	0.0	21.3	82.7	
176.8	292.1	414.9	530.2	624.4	685.8	
707.1	685.8	624.4	530.2	414.9	292.1	
176.8	82.7	21.3	0.0	21.3	82.7	
176.8	292.1	414.9	530.2	624.4	685.8	

(/)

FIGURE III-H.20a HARMONIC THERMAL LOAD INPUT, THICK WALLED DISK

# Contrails

ELEMENT ID			REPEAT ELEMENT	LOADING OPTION	NUMBER OF LOADING POINTS			HARMONIC THERMAL LOAD VALUES					
7	8	9			11	12	13	17	27	37	47	57	67
		2			3	6	218.0	211.4	192.5	163.5	127.9	90.0	
							54.5	25.5	6.6	0.0	6.6	25.0	
							54.5	90.0	127.9	163.5	192.5	211.4	
							218.0	211.4	192.5	163.5	127.9	90.0	
							54.5	25.5	6.6	0.0	6.6	25.0	
							54.5	90.0	127.9	163.5	192.5	211.4	

(-)

ELEMENT ID			REPEAT ELEMENT	LOADING OPTION	NUMBER OF LOADING POINTS			HARMONIC THERMAL LOAD VALUES					
7	8	9			11	12	13	17	27	37	47	57	67
		3			3	6	976.0	946.0	861.8	732.0	572.8	403.2	
							244.0	114.2	29.4	0.0	29.4	114.2	
							244.0	403.2	572.8	732.0	861.8	946.6	
							976.0	946.6	861.8	732.0	572.8	403.2	
							244.0	114.2	29.4	0.0	29.4	114.2	
							244.0	403.2	572.8	732.0	861.8	946.2	

(-)

FIGURE III-H.206 HARMONIC THERMAL LOAD INPUT,  
THICK WALLED DISK (CONTINUED)

1 2 3 4 5 6 7 8  
 1234567890123456789012345678901234567890123456789012345678901234567890

REPORT  
 TITLE

1  
 THICK WALLED DISC SUBJECTED TO NON-AXISYMMETRIC LOADING

SYSTEM

5 5 3 1 0 0 0 3 1 5 0.0

MATER

1 121M STEEL - E=30E6, MU=0.3 X 1 0.0005  
 0.0 30.0 E60.30 6.00 E-6

COORD

10.5 0.0 0.0  
 201.0 0.0 0.0  
 301.0 0.0 0.1  
 40.75 0.0 0.1  
 500.5 0.0 0.1

BOUNDS

1 111  
 2 111  
 3 111  
 4 111  
 5 11C

ELEM

131 12 X 3 002003004  
 331 12 X 3 001002004  
 231 12 X 3 001004005

HTEM

HTEM	ELEM	BOUNDS	COORD	HTEM	ELEM	BOUNDS	COORD
01	36	322		01	36	322	
			707.1				685.8
			176.8				82.7
			707.1				292.1
			176.8				685.8
			176.8				82.7
			218.0				292.1
			54.5				211.4
			54.5				25.5
			218.0				90.0
			54.5				127.9
			54.5				192.5
			244.0				211.4
			244.0				25.5
			976.0				90.0
			244.0				127.9
			244.0				192.5
			976.0				211.4
			244.0				25.5
			244.0				90.0
			976.0				127.9
			244.0				192.5
			244.0				211.4
			976.0				25.5
			244.0				90.0
			244.0				127.9
			976.0				192.5
			244.0				211.4
			244.0				25.5
			976.0				90.0
			244.0				127.9
			244.0				192.5
			976.0				211.4
			244.0				25.5
			244.0				90.0
			976.0				127.9
			244.0				192.5
			244.0				211.4
			976.0				25.5
			244.0				90.0
			244.0				127.9
			976.0				192.5
			244.0				211.4
			244.0				25.5
			976.0				90.0
			244.0				127.9
			244.0				192.5
			976.0				211.4
			244.0				25.5
			244.0				90.0
			976.0				127.9
			244.0				192.5
			244.0				211.4
			976.0				25.5
			244.0				90.0
			244.0				127.9
			976.0				192.5
			244.0				211.4
			244.0				25.5
			976.0				90.0
			244.0				127.9
			244.0				192.5
			976.0				211.4
			244.0				25.5
			244.0				90.0
			976.0				127.9
			244.0				192.5
			244.0				211.4
			976.0				25.5
			244.0				90.0
			244.0				127.9
			976.0				192.5
			244.0				211.4
			244.0				25.5
			976.0				90.0
			244.0				127.9
			244.0				192.5
			976.0				211.4
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			244.0				90.0
			976.0				127.9
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			244.0				211.4
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			244.0				90.0
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			976.0				192.5
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			976.0				90.0
			244.0				127.9
			244.0				192.5
			976.0				211.4
			244.0				25.5
			244.0				90.0
			976.0				127.9
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			244.0				90.0
			244.0				127.9
			976.0				192.5
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			976.0				90.0
			244.0				127.9
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			976.0				211.4
			244.0				25.5
			244.0				90.0
			976.0				127.9
			244.0				192.5
			244.0				211.4
			976.0				25.5
			244.0				90.0
			244.0				127.9
			976.0				192.5
			244.0				211.4
			244.0				25.5
			976.0				90.0
			244.0				127.9
			244.0				192.5
			976.0				211.4
			244.0				25.5
			244.0				90.0
			976.0				127.9
			244.0				192.5
			244.0				211.4
			976.0				25.5
			244.0				90.0
			244.0				127.9
			976.0				192.5
			244.0				211.4
			244.0				25.5
			976.0				90.0
			244.0				127.9
			244.0				192.5
			976.0				211.4
			244.0				25.5
			244.0				90.0
			976.0				127.9
			244.0				192.5
			244.0				211.4
			976.0				25.5
			244.0				90.0
			244.0				127.9
			976.0				192.5
			244.0				211.4
			244.0				25.5
			976.0				90.0
			244.0				127.9
			244.0				192.5
			976.0				211.4
			244.0				25.5
			244.0				90.0
			976.0				127.9
			244.0				192.5
			244.0				211.4
			976.0				25.5
			244.0				90.0
			244.0				127.9
			976.0				192.5
			244.0				211.4
			244.0				25.5
			976.0				90.0
			244.0				127.9
			244.0				192.5
			976.0				211.4
			244.0				25.5
			244.0				90.0
			976.0				127.9
			244.0				192.5
			244.0				211.4
			976.0				25.5
			244.0				90.0
			244.0				127.9
			976.0				192.5
			244.0				211.4
			244.0				25.5
			976.0				90.0
			244.0				127.9
			244.0				192.5
			976.0				211.4
			244.0				25.5
			244.0				90.0
			976.0				127.9
			244.0				192.5
			244.0				211.4
			976.0				25.5
			244.0				90.0
			244.0				127.9
			976.0				192.5
			244.0				211.

H A R M O N I C L C A D S ( T H E R M A L )

ELEMENT NO	DIRECTION OF LOADING	NUMBER OF POINTS	NUMBER OF HARMONIC REQUESTS		LOADS																																	
			1	2	POINT	POINT																																
1		36	4	7	10	13	16	19	22	25	28	31	34	5	8	11	14	17	20	23	26	29	32	35	6	9	12	15	18	21	24	27	30	33	36			
			0.530200E 03	0.176800E 03	0.0	0.176800E 03	0.530200E 03	0.707100E 03	0.530200E 03	0.176800E 03	0.0	0.176800E 03	0.530200E 03	0.707100E 03	0.530200E 03	0.176800E 03	0.0	0.176800E 03	0.530200E 03	0.707100E 03	0.530200E 03	0.176800E 03	0.0	0.176800E 03	0.530200E 03	0.707100E 03	0.530200E 03	0.176800E 03	0.0	0.176800E 03	0.530200E 03	0.707100E 03	0.530200E 03	0.176800E 03	0.0	0.176800E 03	0.530200E 03	0.707100E 03

FIGURE III-H.22 ASYMMETRIC LOAD DATA OUTPUT, THICK WALLED DISC

R E S U L T O F H A R M O N I C A N A L Y S I S

ELEMENT NO	HARMONIC COEFFICIENT	FOURIER COEFFICIENT
1	0.0	0.353526E 03
	0.200000E 01	0.349965E 03

FIGURE III-H.23 HARMONIC LOAD OUTPUT, THICK WALLED DISC

STRESSES FOR THE ASYMMETRIC TRIANGULAR CROSS SECTION RING ELEMENT  
 (STRESSES EVALUATED AT THE ELEMENT CENTROID)

LOAD CONDITION NUMBER	ELEMENT NUMBER	ELEMENT TYPE	ELEMENT GRID POINTS
1	1	31	2 3 4
APPARENT ELEMENT STRESSES			
STRESS POINT	AXIAL (ZZ)	CIRCUMFERENTIAL (THETA-THETA)	SHEAR (R-THETA) (Z-THETA)
1	0.13969919E C6	0.16607738E 06	0.74993750E 04 0.0 0.0
ELEMENT APPLIED STRESSES			
STRESS POINT	AXIAL (ZZ)	CIRCUMFERENTIAL (THETA-THETA)	SHEAR (R-THETA) (Z-THETA)
1	0.15908656E 06	0.15908656E 06	0.0 0.0 0.0
NET ELEMENT STRESSES			
STRESS POINT	AXIAL (ZZ)	CIRCUMFERENTIAL (THETA-THETA)	SHEAR (R-THETA) (Z-THETA)
1	-0.19387375E 05	0.69908125E 04	0.74993750E 04 0.0 0.0

FIGURE III-H.24 a HARMONIC STRESSES FOR ELEMENT NO. (1), HARMONIC ( $n = 0$ ), THICK WALLED DISC

STRESSES FOR THE ASYMMETRIC TRIANGULAR CROSS SECTION RING ELEMENT  
 (STRESSES EVALUATED AT THE ELEMENT CENTROID)

LOAD CONDITION NUMBER	ELEMENT NUMBER	ELEMENT TYPE	ELEMENT GRID POINTS
1	1	31	2 3 4
APPARENT ELEMENT STRESSES			
STRESS POINT	AXIAL (ZZ)	CIRCUMFERENTIAL (THETA-THETA)	SHEAR (R-THETA) (Z-THETA)
1	0.12593919E 06	0.11695213E 06	0.13904594E 05 -0.25779180E 04 0.38851470E 04
ELEMENT APPLIED STRESSES			
STRESS POINT	AXIAL (ZZ)	CIRCUMFERENTIAL (THETA-THETA)	SHEAR (R-THETA) (Z-THETA)
1	0.15748388E 06	0.15748388E 06	0.0 0.0 0.0
NET ELEMENT STRESSES			
STRESS POINT	AXIAL (ZZ)	CIRCUMFERENTIAL (THETA-THETA)	SHEAR (R-THETA) (Z-THETA)
1	-0.52789375E 04	-0.40531750E 05	0.13904594E 05 -0.25779180E 04 0.38851470E 04

FIGURE III-H.24 b HARMONIC STRESSES FOR ELEMENT NO. (1), HARMONIC (n = +2), THICK WALLED DISC



DISPLACEMENTS

THETA	GRID POINT	U	V	W
0	1	-0.322093E-02	0.0	-0.672627E-03
0	2	-0.174501E-02	0.0	-0.281899E-01
0	3	0.377694E-02	0.0	-0.277109E-01
0	4	0.304158E-02	0.0	-0.143697E-01
0	5	0.222443E-02	0.0	0.0

FIGURE III-H.25 a NODAL CIRCLE DISPLACEMENTS AT  $\theta = 0^\circ$ , THICK WALLED DISC

DISPLACEMENTS

THETA	GRID POINT	U	V	W
60	1	-0.135652E-02	0.116087E-02	-0.142783E-03
60	2	-0.976223E-03	0.656639E-03	-0.281772E-01
60	3	0.462233E-02	0.690500E-03	-0.280949E-01
60	4	0.445186E-02	0.856080E-03	-0.141104E-01
60	5	0.422915E-02	0.116352E-02	0.0

FIGURE III-H.25 b NODAL CIRCLE DISPLACEMENTS AT  $\theta = 60^\circ$ , THICK WALLED DISC

NET STRESSES FOR THE TRIANGULAR RING ELEMENT  
(STRESSES EVALUATED AT ELEMENT CENTROID)

ELEMENT NUMBER	ELEMENT TYPE	CIRCUMFERENTIAL	RADIAL	NORMAL CIRCUMFERENTIAL	AXIAL	NORMAL AXIAL	R-Z	R-THETA	Z-THETA	SHEAR
1	31	0	-0.5093206E 05	-0.8280563E 04	-0.3354094E 05	0.2140397E 05	0.0	0.0	0.0	0.0
1	31	30	-0.3515988E 04	-0.5641117E 04	-0.1327527E 05	0.1445174E 05	-0.2232535E 04	-0.3364624E 04	0.3364624E 04	0.3364624E 04
1	31	60	-0.3615359E 04	-0.3622107E 03	0.2725627E 05	0.5472227E 03	-0.2232558E 04	-0.3364659E 04	0.3364659E 04	0.3364659E 04
1	31	90	0.1215731E 05	0.2277313E 04	0.4752256E 05	-0.6405219E 04	-0.4341311E-01	0.6542730E-01	0.6542730E-01	0.6542730E-01
1	31	120	-0.3614414E 04	-0.3620525E 03	0.2725748E 05	0.5468047E 03	0.2232513E 04	-0.3364592E 04	-0.3364592E 04	-0.3364592E 04
1	31	150	-0.3515893E 05	-0.5640961E 04	-0.1327405E 05	0.1445132E 05	0.2232579E 04	-0.3364692E 04	-0.3364692E 04	-0.3364692E 04
1	31	180	-0.5093206E 05	-0.8280563E 04	-0.3354094E 05	0.2140397E 05	0.0	0.0	0.0	0.0
1	31	210	-0.3516080E 05	-0.5641273E 04	-0.1327645E 05	0.1445214E 05	-0.2232491E 04	-0.3364559E 04	0.3364559E 04	0.3364559E 04
1	31	240	-0.3616250E 04	-0.3623604E 03	0.2725512E 05	0.5476172E 03	-0.2232600E 04	-0.3364722E 04	0.3364722E 04	0.3364722E 04
1	31	270	0.1215731E 05	0.2277313E 04	0.4752256E 05	-0.6405219E 04	-0.4341311E-01	0.6542730E 00	0.6542730E 00	0.6542730E 00
1	31	300	-0.3613492E 04	-0.3618987E 03	0.2725866E 05	0.5464023E 03	0.2232470E 04	-0.3364526E 04	-0.3364526E 04	-0.3364526E 04
1	31	330	-0.3515804E 05	-0.5640813E 04	-0.1327290E 05	0.1445093E 05	0.2232621E 04	-0.3364755E 04	-0.3364755E 04	-0.3364755E 04

FIGURE III-H.26 STRESSES IN ELEMENT NO. (1), THICK WALLED DISC

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## APPENDIX A USER MANUAL UPDATES

The following presents updated User instructions to the MAGIC User's Manual. The updates are referenced to the MAGIC II User's Manual (Reference 5) by page number.

1. Page 36 The EPRINT abstraction instruction does not have dots around it. It should read EPRINT(a,b,c)D.
2. Page 37 The following additional options are available for the .ASSEM. structural abstraction instruction:  
d = 1 , to assemble the reduced element stiffness matrices  
d = 2 , to assemble the reduced element mass matrices  
d = 3 , to assemble the reduced element incremental matrices  
d = 4 , to assemble the reduced element applied load matrices

where for d = 1, 2 and 3 [C] will have the order (N x N) where N = NS x S - (the number of retained degrees of freedom). If d = 4, then C will have the order (N x 1).

$$C = \begin{bmatrix} C_{11} & C_{12} \\ C_{21} & C_{22} \end{bmatrix} \quad \text{or} \quad C = \begin{bmatrix} C_1 \\ C_2 \end{bmatrix}$$

3. Page 37 The GPRINT abstraction instruction does not have dots around it. It should read GPRINT(a,b,c,C1.C2.C3.etc)F,G.
4. Page 38 Explanation of Matrix E.  
E. This matrix is optional. It may be suppressed if input matrix F is in unreduced form, i.e., contains all system degrees of freedom. If matrix F is reduced, then E must be a transformation matrix (generated from OMP5) used to unreduce F for printing. If a = 3, then this matrix must be present if the eigenvector matrix is reduced, which is usually the case.

## APPENDIX A (CONT)

5. Page 103 Item Number 4
4. Number of Load Conditions - (Cols. 15-16)  
The number of load conditions is equal to the number of external load conditions that are applied to the system. Note that external loads are not to be confused with element applied loading such as temperature and pressure.  
If there are no external loads applied to the system, then the number of load conditions should be set to zero and no LOADS section need be input. An element applied load scalar of 1.0 will automatically be generated.  
At the present time, the maximum number of external load conditions allowed is one hundred (100).
6. Page 103 Item Number 6
6. Number of Prescribed Displacement Condition - (Cols. 23-28)  
Applied loading may be prescribed in terms of non-zero displacement values. Either one prescribed displacement condition or NL prescribed displacement conditions can be accommodated per execution, where NL is defined in item number (4) above. Therefore, the number of prescribed displacement conditions should be equal only to 1 or NL. If there are no prescribed displacement conditions, then this entry is ignored by the User.
7. Page 105 Item Number 6  
This item should read as follows:  
6. Number of prescribed displacement conditions.
8. Page 131 Item Number 12  
This item should read as follows:  
12. Prescribed Displacement Condition Section (Figure II-11)
9. Page 134 Condition Number - (Cols. 7-11)  
The condition number is a fixed point number. In the present MAGIC System either 1 or NL prescribed displacement conditions can be accommodated per execution. NL is defined as the total number of loading conditions in a given analysis. If the User specifies NL prescribed displacement conditions then the corresponding prescribed displacement condition will be used with the appropriate external load condition. If you specify 1 prescribed displacement condition, then the same set of values will be generated NL times to be used with each external load condition.

## APPENDIX A (CONT.)

10. Page 136 Item Number 5
5. The number of prescribed displacement conditions must be specified on the System Control Information Data Form (Figure II-3). This value is equal to 1 or NL, where NL is defined to be the number of external load conditions.
11. Page 138 Last Paragraph should read as follows:
- The first entry on the External Grid Point Loads Form is pre-labeled LOADS and requires no information from the User. If there are no External Loads acting on the system, then the User does not have to input a LOADS section. The MAGIC system will automatically generate one zero load condition with an element applied load scalar of 1.0 for the User.
12. Page 138 Delete Item Number 3 under Condition Number.
13. Page 140 Item Number 1 under REMEMBER heading should read:
1. The External Grid Point Loads Section may be omitted if there is no external grid point loads acting on the structure. Enter a zero on the System Control Information Data Form (Figure II-3) if this is the case. An applied element load scalar of 1.0 will automatically be generated for the user.

APPENDIX B

MAGIC INPUT DATA FORMS

This Appendix compiles all the MAGIC structural analysis input data forms. The use of these forms is explained in detail in Reference 5 and this report. They are placed here to serve the User as "tear-outs".





BAC 1616-1

MATERIAL (//)  
1 2 3 4 5 6

(//)  
7 8 9  
No. of Requests

MAGIC STRUCTURAL ANALYSIS SYSTEM  
INPUT DATA FORMAT

MATERIAL TAPE INPUT

Request Number	MATERIAL NUMBER	Lock Code	MATERIAL IDENTIFICATION										Orthotropic	Plastic Isotropic	Plastic Orthotropic	Add Plastic	Delete Material	Print Tape	Print Mat'l. Table	Print Mat'l. Summary	Number of Mat'l. Pts.	Number of Plastic Pts.	MASS DENSITY					
			1	2	3	4	5	6	7	8	9	0											1	2	3	4	5	6
7	8	0	1	2	3	4	5	6	7	8	9	0	1	2	3	4	5	6	7	8	9	0	1	2	3	4		

(//)

MATERIAL PROPERTIES TABLE

TEMPERATURE						YOUNGS MODULI						POISSONS RATIOS						COEF OF THERMAL EXPANSION						RIGIDITY MODULI																	
1	2	3	4	5	6	7	8	9	0	1	2	3	4	5	6	7	8	9	0	1	2	3	4	5	6	7	8	9	0	1	2	3	4	5	6	7	8	9	0	1	2
Direction						Direction						Direction						Direction																							
E <sub>x</sub>						E <sub>y</sub>						E <sub>z</sub>						G <sub>xy</sub>						G <sub>yz</sub>						G <sub>zx</sub>											



BAC 1817

MAGIC STRUCTURAL ANALYSIS SYSTEM  
INPUT DATA FORMAT

TEMPERATURE										PLASTIC PROPERTIES TABLE																													
1	2	3	4	5	6	7	8	9	0	1	2	3	4	5	6	7	8	9	0	1	2	3	4	5	6	7	8	9	0	1	2	3	4	5	6	7	8	9	0

## MAGIC STRUCTURAL ANALYSIS SYSTEM INPUT DATA FORMAT

### SYSTEM CONTROL INFORMATION

ENTER APPROPRIATE NUMBER, RIGHT  
ADJUSTED, IN BOX OPPOSITE  
APPLICABLE REQUESTS

1. Number of System Grid Points

S	Y	S	T	E	M
1	2	3	4	5	6

(/)

2. Number of Input Grid Points

1	2	3	4	5	6

7	8	9	10	11	12

3. Number of Degrees of Freedom/Grid Point

13	14

4. Number of Load Conditions

15	16

5. Number of Initially Displaced Grid Points

17	18	19	20	21	22

6. Number of Prescribed Displaced Grid Points

23	24	25	26	27	28

7. Number of Grid Point Axes Transformation  
Systems

29	30

8. Number of Elements

31	32	33	34	35	36

9. Number of Requests and/or Revisions of  
Material Tape.

37	38

10. Number of Input Boundary  
Condition Points

39	40	41	42	43	44

11.  $T_0$  For Structure (With Decimal Point)

45	46	47	48	49	50	51	52

(/)

**MAGIC STRUCTURAL ANALYSIS SYSTEM  
INPUT DATA FORMAT**

**CALCULATION CONTROL**

C	A	L	C		(/)
1	2	3	4	5	

PLACE 'X' IN BOX OPPOSITE  
DESIRED OPERATIONS

1. Revise Material Tape

1

2. Inverse Solution

2

3. Choleski Decomposition

3

4. Linear Function Minimization Solution

4

5. Nonlinear Function Minimization Solution

5

6. Plastic Analysis

6

7. Grid Point Axes Transformation

7

8. Stress Calculations

8

9. Reactions

9

10. Structure Plot

10

11. Dynamics Analysis

11

(/)

**MAGIC STRUCTURAL ANALYSIS SYSTEM  
INPUT DATA FORMAT**

**PRINT OPTIONS**

P	R	I	N	T		( / )
1	2	3	4	5	6	

PLACE 'X' IN BOX OPPOSITE  
DESIRED PRINT

1. Assembly – Stiffness

1

2. Inverse – Stiffness

2

3. Triangularized – Stiffness

3

4. Displacements

4

5. Intermediate Function Minimization

5

( / )



## MAGIC STRUCTURAL ANALYSIS SYSTEM INPUT DATA FORMAT

### EIGENVALUE INFORMATION

FOR USE IN ALL  
EIGENVALUE  
PROBLEMS

**D Y N A M** (/)  
1 2 3 4 5 6

1. Number of Eigenvalues Requested  
(Less Than or Equal to 20)

1	2

2. Convergence Criteria (Floating Point)  
(Default Option - 0.001)

3	4	5	6	7	8	9	10	11	12	13	14			

3. Maximum Number of Iterations  
(Default Option - 500 Iterations)

15	16	17

4. Debug Iteration Print  
Iteration Print ON = 1  
Iteration Print OFF = 0  
(Default Option - Print OFF)

18

5. First Normalizing Element for Print  
(Default Option - No First Normalization)

19	20	21	22

6. Second Normalizing Element for Print  
(Default Option - No Second Normalization)

23	24	25	26

7. Control for Guess Vector Iteration Start  
Column Iteration Start = 0  
Row Iteration Start = 1  
(Default Option - Column Iteration Start)

27 (/)

































MAGIC STRUCTURAL ANALYSIS SYSTEM  
INPUT DATA FORMAT

1 2 3 4 5 6  
PARDISP (/)

1 2 3 4 5 6 7 8 9 0 1 1  
PCOND (/)

Condition Number

1 2 3 4 5 6  
MCPAL

PRESCRIBED DISPLACEMENTS

TRANSLATIONS				ROTATIONS				GENERALIZED													
U		V		W		$\theta_x$	$\theta_y$	$\theta_z$	1		2		3								
1	2	3	4	5	6	7	8	9	0	1	2	3	4	1	2	3	4				
3	4	5	6	7	8	9	0	1	2	3	4	5	6	7	8	9	0	1	2	3	4

Grid Pt. Number		TRANSLATIONS				ROTATIONS				GENERALIZED															
1	2	3	4	5	6	7	8	9	0	1	2	3	4	1	2	3	4	1	2	3	4				
7	8	9	0	1	2	3	4	5	6	7	8	9	0	1	2	3	4	1	2	3	4	1	2	3	4
3	4	5	6	7	8	9	0	1	2	3	4	5	6	7	8	9	0	1	2	3	4	1	2	3	4

























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<b>13. ABSTRACT</b> An automated general purpose system for analysis is presented. This system identified by the acronym, "MAGIC III" for Matrix Analysis via Generative and Interpretive Computations, is an extension of the structural analysis capability available in the initial MAGIC System. MAGIC III provides a powerful framework for implementation of the finite element analysis technology and provides diversified capability for displacement, stress, vibration, and stability analyses.  Additional elements have been added to the MAGIC element library in this phase of MAGIC development. These are the solid elements; rectangular prism, tetrahedron, triangular prism, symmetric triangular prism, and triangular ring (asymmetrical loading). Also included are the symmetric shear web element and a revised quadrilateral thin shell element. The finite elements listed include matrices for stiffness, mass, prestrain load, thermal load, distributed mechanical load, pressure and stress.  Documentation of the MAGIC III System is presented in three parts; namely, Volume I: Engineer's Manual, Volume II: User's Manual and Volume III: Programmer's Manual.		

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	ROLE	WT	ROLE	WT	ROLE	WT
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2. Matrix Methods						
3. Matrix Abstraction						
4. Digital Computer Methods						
5. Finite Element Techniques						

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